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Transient Dynamics Sector

EUROPLEXUS

A Computer Program for the Finite Element Simulation
of Fluid-Structure Systems under Transient Dynamic Loading

USER'S MANUAL

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1 FOREWORD

1.1 ABOUT THIS DOCUMENTATION

This documentation is produced starting from a \LaTeX source file that is continuously updated and maintained. This file contains the information in a simply tagged format. A formatted version of the manual, such as the present one, can be automatically obtained either in printable (PostScript) form or for graphic display (DVI), or for on-line consulting either via the Adobe Acrobat Reader (PDF), or via an Internet browser such as Netscape or Internet Explorer (HTML).

For more information about source file writing and formatting, consult the documentation listed in the references.

A table of contents and an index can be found at the beginning and at the end of this documentation, respectively, and may help to rapidly retrieve information.

1.1.1 PAGE NUMBERING CONVENTIONS

Each page of the present manual contains information in both an header (at the page top) and a footer (at the page bottom).

The header contains in its middle part a ‘page identification’ used to identify each page or sequence of pages dealing with a specific topic. This is composed by a letter or group of letters, indicating a ‘section’ of the manual, and a number. The same information is also repeated in the left part of the footer. The number is not the actual page number (this is given in the right part of the footer, instead), since this is subject to frequent changes as text is added or reformatting is performed. Cross references in the text are generally done by referring to the page identification.

The numbers appearing in the page identification are not incremented by 1, but have larger increments (usually 10) in order to leave space for future pages.

The page header contains in its right part a date indicating when a certain page has been inserted or modified in the manual.

The date in the center of the footer corresponds to the manual edition date.

2 PROGRAMMING GUIDELINES

The present document (User's Manual) deals primarily with a detailed description of all the input commands of the EUROPLEXUS code, in the order in which they appear in a typical input file. It is meant to be an exhaustive *reference* for Users of the code, but it is *not a tutorial*.

Before presenting the input commands (see following Sections), some short indications are given in this Section, which might be useful to EUROPLEXUS code *developers* (i.e. to *programmers*), in the form of simple programming guidelines. Developers are assumed to have access to the EUROPLEXUS Consortium web site, which contains among other things all the available technical documentation.

Only an overview is presented here for brevity. Reference is made to several reports and other documents, listed in the bibliography and available on-line on the Consortium web site, which contain all the details.

2.1 Programming Language

EUROPLEXUS is entirely written in the Fortran language. For historical reasons some parts of the code (the oldest ones) are still coded in Fortran 77 (F77). All new developments from 1999 onwards use the Fortran 90 (F90) language, mainly based upon F90 Modules.

A great effort of converting F77 parts to F90 is being performed, but the process is not yet complete. An example of successful conversion is the materials data structure, which has been completely ported to F90.

2.2 F77 Programming

An introduction to F77 programming style in EUROPLEXUS is given in reference [150].

2.3 F90 Programming

An introduction to F90 programming style and some tentative guidelines for the development of EUROPLEXUS are given in reference [175].

2.4 Code Profiling

A code profiler is **not** readily available under Windows. A simple mechanism is set up to profile selected parts of the code (in a platform-independent way) by using the standard system routine `CPU_TIME`, which returns the job's CPU time in seconds.

This profiling cannot be totally automatic as e.g. under Linux, but it is set up in such a way:

- to have **virtually no impact** on the code standard version, i.e. when no profiling is activated.
- to require only a **very small** intervention on routines (or any code parts) to be profiled.

The mechanism is very simple. Assume we want to profile a routine SUB1. Then the following standard code is inserted at the beginning of the routine (before the first executable statement) and at the end of the routine (before the RETURN or the END SUBROUTINE):

```
SUBROUTINE SUB1
...
USE M_PROFILE
...
CIF PROFILE
* Profiling
    LOGICAL, SAVE :: LPROF
    INTEGER, SAVE :: IPROF
    REAL(4) :: TPROF1, TPROF2, TPROFD
    DATA LPROF, IPROF /.TRUE., 0/
    IF (LPROF) THEN
        CALL GET_PROFILE_INDEX ('SUB1', IPROF)
        LPROF = .FALSE.
    ENDIF
    CALL CPU_TIME (TPROF1)
CENDIF

... routine body (executable statements)

CIF PROFILE
    CALL CPU_TIME (TPROF2)
    TPROFD = TPROF2 - TPROF1
    CALL CUMULATE_PROF_TIME (IPROF, TPROFD)
CENDIF
END SUBROUTINE SUB1
```

Note that the two code portions to be inserted can be just copied (e.g. from LOOPELM.FF) and pasted as such. Only the name of the subroutine (SUB1) in the CALL GET_PROFILE_INDEX has to be changed.

To activate profiling of the routine, compile it with the PROFILE keyword activated for filtering. Under Windows the command is:

```
epx_cmp -o -k PROFILE
```

Then link normally and run the code normally. At the end of the listing the profile table is printed, reporting the CPU time spent in each profiled portion of the code (both in absolute terms and in % of total CPU time), and the number of calls of each profiled routine.

The CIF PROFILE ... CENDIF code portions can be safely left in the routines, since they have no impact on normal code efficiency thanks to the standard EUROPLEXUS filtering mechanism.

Obviously, this type of profiling is not only limited to entire subroutines. The same mechanism can also be used to profile other code parts, e.g. a suspect DO loop. The name passed

to `GET_PROFILE_INDEX` can be freely chosen by the user, it is not necessarily a subroutine name (maximum length is 32 characters, including any inter-word blanks).

Profiling instructions (filtered) will be gradually inserted in at least the most important routines of the code (CELEM, LOOPELM etc.) so that first-attempt profiling can become as simple as the following commands (under Windows):

```
eplx_modtree M_PROFILE      (search routines already containing M_PROFILE)
eplx_get_users              (copy all such routines in current directory)
<possibly add some other routines for profiling>
eplx_cmp -o -k PROFILE      (compile all local routines for profiling)
eplx_lk -o                  (link to produce a local executable eplx.exe)
eplx_bench -e eplx.exe ... (run tests and obtain profile)
```

Based on the results of this first-level profiling, other routines can then be added to the list for a refinement of profiling results.

The implementation uses a very small module `M_PROFILE` containing some very small **static** tables (no allocation / deallocation), with space for up to `NPROFMAX` (currently 100) routines to be profiled.

A permanent (unfiltered) profiling based upon the above mechanism has been added in the time loop routines: `CALCUL` for the normal case or `TLOOPP` for the case with spatial partitioning. Thus, by default the following main “blocks” of code are profiled:

- VFCC (cell-centred Finite Volumes)
- Finite Elements
- External Forces
- Coupled links (LINK COUP)
- Coupled links by the old “liaison” implementation (LINK LIAI)
- Decoupled links (LINK DECO)
- Outputs

This should give a first-level (and fully automatic) overview of where in the code the majority of CPU time is spent, for a given application. Then, refinement of the profiling can be performed if necessary by the technique explained above on the affected code portion.

3 INTRODUCTION

3.1 ABOUT EUROPLEXUS

EUROPLEXUS is a computer code being jointly developed since 1999 by CEA (CEN Saclay, DMT) and EC (JRC Ispra, IPSC) under a collaboration contract. It stems from CEA's CASTEM-PLEXUS (a program belonging to the CASTEM system) and the previous CEA-EC joint product PLEXIS-3C.

The code analyses 1-D, 2-D or 3-D domains composed of solids (continua, shells or beams) and fluids. Fluid-structure interaction is also taken into account.

The program uses an explicit algorithm (central-difference) for the discretization in time and therefore it is best adapted to rapid dynamic phenomena (fast transient dynamics) such as explosions, impacts, crashes etc. Geometric non linearity (large displacements, large rotations, large strains), and the non-linearity of materials (plasticity, viscoplasticity, etc) are fully taken into account.

The spatial discretization is mainly based on the Finite Element or Finite Volume method. Other formulations such as SPH (Smoothed Particle Hydrodynamics), Spectral Elements, Diffuse Elements etc. are also available or under development. Numerous element types and a comprehensive library of material types for solids, fluid and special media (e.g. impedances) are available.

Three main descriptions are available in the code: the Lagrangian description which is well suited for the structural domain, the Eulerian description useful for purely fluid problems, and the Arbitrary Lagrangian Eulerian (ALE) description which is typically used in fluid-structure interaction problems.

EUROPLEXUS is interfaced to various pre- and post-processing programs that enable the meshing of the studied domain (e.g. CEA's Cast3m) and the visualization of the results (e.g. Cast3m, ParaView or EUROPLEXUS itself).

A large bibliography concerning EUROPLEXUS as well as its ancestors is provided at the end of the present manual (see Section BIB). Many of the cited documents are available to EUROPLEXUS developers in electronic form on the EUROPLEXUS Consortium web site (<https://europlexus.jrc.ec.europa.eu/>). The up-to-date version of the present User's Manual may also be found on the site.

3.2 PREPARATION OF THE INPUT DATA

Rule 1:

All the data may be coded in FREE format (unless a fixed format is explicitly chosen by the user for specific data, e.g. the geometry). One or more blanks and/or an end of line act as a separator between data items.

Only columns 1 to 72 of each 'data card' are analysed. A data card is a record, i.e. a sequence of up to 72 characters terminated by a new line character.

Rule 2:

The data consist in a sequence of instructions or DIRECTIVES, (each one possibly including a number of OPTIONS and SUBOPTIONS) that can be specified using keywords.

Only the FIRST FOUR LETTERS ARE COMPULSORY for the coding of keywords, since these are unique in each situation. EUROPLEXUS ignores characters beyond the fourth one when decoding a keyword.

Rule 3:

Numeric values are coded with or without a decimal point:

Example:

12 24. .3 1.3E-4 1E4 .5D+2 -.4E 00

Rule 4:

Comment 'cards' (records, in the sense defined above) can be freely interspersed with the data, except that the first record of a data file cannot be a comment (it is the problem title, see below).

A comment card begins with a dollar (\$) or an asterisk (*) in column 1. Alternatively, it is possible to use an exclamation point (!) which may be placed at any position in the 'card' and not only in column 1.

Any characters that follow one of these special characters on an input card are ignored by EUROPLEXUS. By using an exclamation point, it is therefore possible to add comments on the same line as the data.

Note that the program does include in the input data echo comments written at the beginning of the output listing.

A semicolon is considered as the end of a card.

Example:

```
"TRID"  "ALE"      !  3D ALE computation
```

3.3 WRITING CONVENTIONS

Object:

To ensure that the necessary keywords, optional key-words as well as the numerical values to be entered show up clearly in the syntax of the instructions, the following conventions have been adopted.

Description of language conventions:

Keywords are enclosed in quotation marks: "TRIDIM".

Anything not enclosed in quotation marks represents numerical values or something else to be described. Names representing character strings are enclosed in apostrophes:

```
< "SAUVE"  nb  ifreq  < "PROT"  'mykey'  > >
```

When a sequence is optional, it is enclosed in angle brackets: (<...>).

If there is a choice between several sequences among which ONE ONLY is compulsory, the sequences are enclosed between (| [...] |) symbols, each of them separated by semicolones (;). The list of sequences can be written on several lines, for example:

```
| [ "DEPL" ; "VITE" ; "ACCE" ] |
```

or

```
| [  
    "DEPL" ;  
    "VITE" ;  
    "ACCE"  
] |
```

If there is a choice between several sequences among which ONE AT MOST is compulsory, the sequences are enclosed between (\$[...]\$) symbols, each of them separated by semicolones (;). Again, the list of sequences can be written on several lines, for example:

```
$[ "ECHO" ; "NOEC" ]$
```

or

```
$ [  
    "ECHO" ;  
    "NOEC"  
] $
```

If a sequence can be repeated as many times as wanted, it is enclosed in parentheses: (...).

If a sequence must be repeated, for example *nf* times, it is framed by: **nf*(...)**. The reading of *nf* must precede immediately the first sequence:

```
( "VMIS" "ISOT" "RO" rho "YOUNG" e "NU" nu ...
... "TRAC" nf*( sig eps ) /LECTURE/ )
```

In this case, the input data should contain an integer (**nf**), immediately followed by *nf* couples of values. For example, if *nf*=3, the sequence **nf*(sig eps)** could be written as:

```
3      0. 0.      1.E8 1.E-3      1.E9 1.E-1
```

In order to simplify the writing, a symbolic name is sometimes assigned to frequently used sequences, that are described only once and then referred to simply by their name enclosed in /, as in:

```
"MASSE" ( /LECDDL/ xm /LECTURE/ )
```

These named sequences are sometimes called procedures. It should be stressed that these names should not appear as such in an input data, but have to be replaced by the appropriate sequence of keywords and values. Named sequences should not be confused with keywords or directives. Their names therefore do not obey to the 4-character rule like keywords.

3.4 USE OF LITERAL VARIABLES

Object :

It is possible to replace any data (a number, a keyword or a text) by a literal variable.

The name of this variable must start by % (per cent), followed by at most 16 alphanumeric characters.

Before being used for the first time, a literal variable must be assigned a value.

In order to assign a value to a literal variable, type its name followed by the equals sign (=) and then by the value, like in the Fortran programming language.

Example :

```
. . .
%diameter_1 = 0.0548      %diameter_2 = 0.2027
%tube_1 = lig_ent        %tube_2 = lig_tot
. . .
COMPLEMENT
  DIAMETRE DROIT %diameter_1  LECT %tube_1 TERM
  DIAMETRE DROIT %diameter_2  LECT %tube_2 TERM
. . .
```

Comments :

It is possible to re-define a literal variable (i.e., change its ‘contents’), at any place in the input data set.

The assignment sign ‘=’ must lie on the same input line as the variable name in the input data set.

A literal variable represents JUST ONE data: in the previous example, %tube_1 and %tube_2 represent each one a single word, that is an object name.

3.5 PROCEDURE /LECTURE/

Object:

Procedure used to define a set of integers. Most of the time it is used to specify a list of nodes or of elements.

Syntax:

Explicit definition (direct list of values):

```
|  "LECT"  n1  n2  . . .  nk  "TERM"                |
```

Implicit definition (by using an increment npas):

```
|  "LECT"  ndeb  "PAS"  npas  nfin  "TERM"            |
```

Definition using the objects created by the mesh generator "GIBI":

```
|  "LECT"    ( 'nomobjet' )  "TERM"                  |
```

Definition using the "permanent groups" created by the mesh generator "I-DEAS":

```
|  "LECT"    ( 'nomgroup' )  "TERM"                  |
```

All the elements or all the nodes are concerned (special keyword TOUS):

```
|  "LECT"  "TOUS"  "TERM"                            |
```

None of the elements or none of the nodes are concerned (special keyword NONE). This can be useful to avoid an error message in directives where the specification of elements or nodes is optional:

```
|  "LECT"  "NONE"  "TERM"                            |
```

Difference, intersection or symmetric difference between two sets (each set being defined by one of the above syntaxes):

```
|  "LECT" <first_set> "DIFF" <second_set> "TERM"  |
|  "LECT" <first_set> "INTR" <second_set> "TERM"  |
|  "LECT" <first_set> "SDIF" <second_set> "TERM"  |
```

Comments:

The explicit and implicit syntaxes can be linked together. For example, to obtain the integers 3, 5, 2, 4, 6, 8, 10, 14, 15, 18, 21, 24, write:

```
LECT 3 5 2 PAS 2 10 14 15 PAS 3 24 TERM
```

For the implicit syntax, the step `npas` can be negative.

In the case of GIBI objects, the procedure extracts, if necessary, the indexes of the nodes or of the elements that constitute the objects/groups defined by the user. The directive where the procedure `/LECTURE/` appears determines if the indexes indicate nodes or elements.

If the `DIFF`, `INTR` or `SDIF` keywords are used, they must be at the end of the directive, as shown in the examples below. In other words, first the basic set must be defined, followed by one of these three keywords and then by the definition of the second set.

From now on this procedure will be called `/LECTURE/` or `/LECT/`.

Remarks:

EUROPLEXUS systematically checks the coherence of the indexes taking into account the expected type (nodes or elements).

Once they have been read, the indexes are classified in ascending order (if this does not harm the concerned directive).

If GIBI object names are mixed up with EUROPLEXUS numbers, separate the sequences by the keyword `SUIT`.

I-DEAS permanent group names can be freely mixed up with EUROPLEXUS indexes.

The keywords `DIFF`, `INTR` and `SDIF` may only be used in `/LECT/ures` that return lists of elements or nodes *ordered* in growing sequence and *without repeated items*. These are the vast majority in the code.

In any case, the finale result of the `/LECT/ure` may not be the empty set. An error is issued in this case.

To indicate **all** the elements or all the nodes in the model, the advised syntax is, as indicated above: `LECT TOUS TERM`. Other (obsolete) forms of the same directive are also accepted sometimes, for example the short syntax `TOUS` (i.e. without the keyword `LECT`). These alternative syntaxes are still accepted for compatibility with old input files, but might become unsupported

in the short future. Note that the full syntax (LECT TOUS TERM) is the only supported syntax for use in conjunction with operations introduced by the keywords DIFF, INTR or SDIF.

Examples:

```

LECT 3 5 2 4 6 8 10 14 15 18 21 24 TERM
LECT 3 5 2 PAS 2 10 14 15 PAS 3 24 TERM
LECT 3 5 10 PAS -2 2 14 15 PAS 3 24 TERM
LECT objet1 objet2 SUIT 25 27 TERM
LECT objet1 SUIT 25 27 SUIT objet2 TERM
LECT toto tata DIFF titi tutu TERM
LECT TOUS DIFF titi tutu TERM
LECT 1 PAS 3 25 INTR titi tutu TERM
LECT toto tata SDIF 1 5 PAS 2 28 TERM

```

Warning concerning the DIFF operator:

Pay attention in the use of the DIFF operator in /LECT/ures where a set of **nodes** is required. For example, the expression:

```
LECT coco DIFF caca TERM
```

used in a context where a set of nodes is expected, and when **coco** is an object composed of elements, is evaluated as follows:

- First, the nodes of all the elements belonging to the **coco** object are evaluated.
- Next, the nodes belonging to the object **caca** are evaluated. If **caca** contains only nodes, these are directly available. If **caca** contains only elements, then all the nodes belonging to such elements are evaluated.
- Finally, the nodes of the second set belonging also to the first set are removed from the first set.

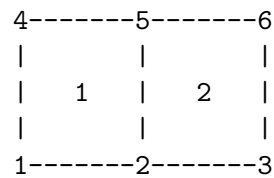
Note that this might not be the result you want or expect. In particular, in the case that both **coco** and **caca** contain elements, the resulting set of nodes is different from (smaller than) the set of nodes belonging to the difference between the two element sets. If the latter is what you want, proceed as follows: first, define a named group of elements (say **edif**) containing the difference of the elements, by using the COMP GROU directive; then, use directly the name **edif** in the /LECT/ expecting the nodes:

```

COMP GROU 1 'edif' LECT coco DIFF caca TERM ! element group
. . .
!LINK COUP BLOQ 1 LECT coco DIFF caca TERM ! wrong !!!
LINK COUP BLOQ 1 LECT edif TERM ! ok : nodes of the
! element group

```

The following example may also help clarify the matter. Assume the following simple mesh:



and assume that the Cast3m object `mesh` contains elements 1 and 2, while the object `right` contains element 2.

Then, in a `/LECT/` directive looking for nodes (e.g. the following blockage directive), the expression:

```
LINK COUP ... BLOQ 1 LECT mesh DIFF right TERM
```

will return nodes 1 and 4 (but *not* nodes 2 and 5). This is because: first the nodes of `mesh` are extracted (nodes 1 to 6); then the nodes of `right` are extracted (nodes 2, 3, 5 and 6); and finally these are subtracted from the first set, thus leaving only nodes 1 and 4.

If one wants instead to select all the nodes of the difference between the element sets (i.e. nodes 1, 2, 4 and 5), one can proceed as follows:

```
COMP GROU 1 'edif' LECT mesh DIFF right TERM
. . .
LINK COUP ... BLOQ 1 LECT edif TERM
```

The first directive builds up the element group containing the difference between the two element sets (difference between *elements*), which results into element 1. The second directive then extracts all the nodes of such element set, i.e. nodes 1, 2, 4 and 5.

3.6 PROCEDURE /PROGRESSION/

Object:

Procedure used to prescribe a group of real numbers. For example, it can be used to specify the values of the physical times at which the solution has to be printed or stored.

Syntax:

Explicit definition :

```
|  "PROG"  r1  r2  . . .  rk  "TERM"          |
```

Implicit definition :

```
|  "PROG"  rdeb  "PAS"  rpas  rfin  "TERM"      |
```

Comments:

The explicit and implicit syntaxes can be linked together. For example, to obtain the values 3., 5., 2., 4., 6., 8., 10., 14., 15., 18., 21., 24. write:

```
"PROG"  3.  5.  2.  "PAS"  2.  10.  14.  15.  "PAS"  3.  24.  "TERM"
```

For the implicit syntax, the step rpas can be negative.

From now on this procedure will be called /PROG/.

Remark:

Once they have been read, the values are classified in ascending order (if this does not harm the instruction concerned).

3.7 PROCEDURE /CTIME/

Object:

Procedure used to choose time values in the form of equidistant values (in time steps or in time values) or user-defined values. Typically, this can be used to specify the times at which output operations such as printouts, storage of data for restart or post-processing, etc., should take place during a computation.

Syntax:

```
< "FREQ" ifreq > < "TFRE" tfreq >  
< "NUPA" /LECT/ > < "TIME" /PROG/ >
```

FREQ

A fixed frequency in time steps is chosen.

ifreq

Value of the frequency in time steps, starting from step 0.

TFRE

A fixed frequency in time is chosen.

tfreq

Value of the frequency in time units, starting from the initial time.

NUPA

A series of time steps is specified by the user.

/LECT/

Definition of the series of time step numbers.

TIME

A series of time values is specified by the user.

/PROG/

Definition of the series of time values.

Comments:

The above optional forms of specifying time values can be freely combined. For example:


```
FREQ 10 NUPA LECT 10 15 35 TERM TFRE 1.E-3  
TIME PROG 0.5E-3 PAS 1.E-3 3.5E-3 TERM
```

is a valid specification and prescribes that the concerned event (e.g., a printout) will take place each 10 steps, and at steps 15 and 35, and each 1.E-3 time units, and at time values of 0.5E-3, 1.5E-3, 2.5E-3 and 3.5E-3.

If repeated values occur (such as in the above example for step number 10), they are automatically dealt with, i.e. only one event takes place for repeated values.

Note that if a time frequency is specified (keyword **TFRE**), the resulting time values take into account the **initial time** of the calculation declared in directive **CALC** by the **TINI** keyword. For example, if **TFRE 2.E-3** and **TINI 0.0**, then the corresponding event will take place at times 2.E-3, 4.E-3, 6.E-3 etc. But if **TFRE 2.E-3** and **TINI 1.0E-3**, then the corresponding event will take place at times 3.E-3, 5.E-3, 7.E-3 etc.

In a few instances, the keyword **"TEMPS"** can be used as an alias for **"NUPA"**. This keyword has been maintained for backwards compatibility, but should no longer be used in newly-written input data.

Warning

Be aware that values given in time units with the above **"TFRE"** and **"TIME"** keywords are rounded to the closest number of 'time normalization units', as described below in the directive **"OPTION"**.

3.8 PROCEDURE /LCHP/

Object:

Procedure used to read a field of values ('champoint') generated by CASTEM2000.

Syntax:

```
"LCHP" nomobjet "TERM"
```

Reads a CASTEM2000 object called `nomobjet`, of the type 'champoint', previously stored in CASTEM2000 with the directive "SAUV". The object is read and stored for successive use by other directives (see e.g. "EPAI").

3.9 PROCEDURE /LECDDL/

Object:

It often occurs in a set of EUROPLEXUS input data that one has to define, for some nodes, the degrees of freedom according to which it is necessary to add a mass, prescribe the direction of a velocity, set a displacement to zero, and so on.

These degrees of freedom are identified by the numbers from 1 to 7 as described hereafter. See also the description of the available elements.

Meaning of the numbers:**AXISYMMETRIC CASE (see AXIS):**

Solid elements:

- 1 : d.o.f. along R;
- 2 : d.o.f. along Z.

Shell elements:

- 1 : d.o.f. along R;
- 2 : d.o.f. along Z;
- 3 : rotation d.o.f.

TWO DIMENSIONAL CASES (see CPLA and DPLA):

Same as for the axisymmetric case (but with X and Y instead of R and Z).

THREE DIMENSIONAL CASE (see TRID):

Solid elements:

- 1 : d.o.f. along X;
- 2 : d.o.f. along Y;
- 3 : d.o.f. along Z.

Shell, beam and pipe elements:

- 1 : d.o.f. along X;

- 2 : d.o.f. along Y;
- 3 : d.o.f. along Z;
- 4 : d.o.f. of rotation around X;
- 5 : d.o.f. of rotation around Y;
- 6 : d.o.f. of rotation around Z;
- 7 : fluid d.o.f. in coupled 1-D (for pipes only).

Conventions:

The degrees of freedom are specified by the corresponding numbers, one immediately after the other, i.e. without blanks.

Examples:

```
346   ==>  d.o.f.  3, 4, and 6;
1426  ==>  d.o.f.  1, 4, 2, and 6.
```

The number of degrees of freedom described must correspond to the problem type (plane, axisymmetric, tridimensional) and to the element type.

In the following this procedure is called /LECDDL/.

3.10 GENERAL ORGANIZATION OF THE INPUT DATA

There are three types of EUROPLEXUS runs:

- 1/ Normal (first run) EUROPLEXUS computation.
- 2/ Restart computation of a previous computation.
- 3/ Editing of EUROPLEXUS computation results.

For the restarts see the pages prefixed with SR.

For the editions see the pages prefixed with ED.

Some examples of input data are described in the pages prefixed with EX.

Normal (first run) computation:

For a normal (first run) computation, the data that is necessary for a EUROPLEXUS run can be classified into 9 groups which are described in the pages prefixed with A, B, C, D, E, F, G, H and I, respectively.

Group A:

Keywords giving indications about the general type of problem to be solved and defining the problem size.

Group B:

Keywords describing the geometry of the structure (geometry, mesh).

Group C:

Keywords assigning the properties of the materials and completing the geometry (e.g. shell thicknesses).

Group D:

Definition of boundary conditions and couplings.

Group E:

Definition of initial values.

Group F:

Definition of applied loads.

Group G:

Keywords determining the output of computed results.

Group H:

Definition of additional options.

Group I:

Run of the program.

Comments:

In this manual, page numbers are preceded by the name of the treated group (from A to I).

The keywords of groups A, B and I must be used in the same order in which they appear in the descriptions given below in this manual.

The keywords belonging to the other groups (groups C to H) can be inserted in any order between the groups B and I. Therefore, their proposed order may be modified.

The complete input data set is printed at the beginning of the EUROPLEXUS output listing.

3.11 MPI PARALLEL CALCULATIONS

Parallel calculations on distributed memory clusters yield the following requirements:

- Calculations must be launched through a working MPI library installed on the system (for questions about hardware or compatibility between EUROPLEXUS executable and MPI library, please contact CEA).
- Since data distribution is achieved through domain decomposition, subdomains must be used (see **STRUCTURE** and **INTERFACE** directives on page I.15). If these directives are ignored, subdomains and interfaces are automatically created using default parameters corresponding to the following input:

```
STRUCTURE AUTO ROB  
INTERFACE LINK NOMU
```

One subdomain is attached to each thread of the parallel simulation. Data between subdomains are exchanged through MPI messages.

For details about parallel algorithms implemented in EUROPLEXUS, please consult:

- Parallel development planification for EUROPLEXUS [\[710\]](#)
- Progress report [\[715\]](#)
- Description and evaluation of the first available parallel version [\[717\]](#)

4 SPATIAL DISCRETIZATION

This Section gives a short description of all the “element” types available in EUROPLEXUS for the spatial discretization of the problem to be solved.

The code contains various formulations, including:

- Finite Elements, which may be used for both structural and fluid parts of the model;
- Spectral Elements, which can be used for the discretization of continuum-like solid parts which behave linearly (e.g. small-strain wave propagation);
- Finite Volumes, which are suited for the fluid parts;
- SPH particles, often used for high-speed impact problems;
- Diffuse Elements.

4.1 AVAILABLE ELEMENTS

Object:

We describe now the different elements available for a one-dimensional, two-dimensional (plane or axisymmetric), or three-dimensional problem, by specifying, for each problem type:

- the number associated to the element type (num.),
- the element name for EUROPLEXUS (Name),
- the element name for the CASTEM meshing (Gibi),
- the number of nodes (npt),
- the number of degrees of freedom per node (dof),
- the number of integration points where the stresses will be computed (ngp).

Specific information relative to each particular element type follows these tables.

Remark:

Stresses are computed at the integration points.

In the output (on the listing or on the result file) all the stresses for a given element and for each of its integration points will be printed. See chapter G as well.

4.1.1 1-D ELEMENTS

The various elements available in 1D are presented below. For practical reasons, they are subdivided into two groups, i.e. models developed at CEA and models developed at JRC.

The element models developed at CEA are :

num.	Name	Gibi	npt	dof	ngp	Remarks
22	TUBE	SEG2	2	1	1	fluid only (rigid tube)
23	TUYA	SEG2	2	7	2	tube coupled with FSI
24	CL1D	POI1	1	1	1	fluid boundary condition
25	BIFU	SUPE	1:9	7	1	bifurcation junction
26	CAVI	SUPE	1:9	1	1	cavity junction
31	CLTU	POI1	1	7	1	boundary condition with FSI
146	BREC	SEG2	2	7	1	pipeline rupture

For these elements, the "EULER" option is mandatory (see page A.30). Furthermore, the mesh nodes must have three coordinates. Thus, the directives for the type of problem treated by these elements will always be of the form:

"TRIDIM" "EULER"

These elements are specified hereafter:

TUBE

This element allows to model the fluid contained within a fixed pipeline. It is assumed that fluid properties are the same in all points of a give cross-section of the pipe (monodimensional calculation).

CL1D

This element allows to introduce the different boundary conditions or localised singularities, in a pipeline meshed by elements of type "TUBE".

CAVI

To assemble the different branches of a pipeline. The junction is done by a finite volume with an attached constitutive law, unlike in the case of a simple bifurcation.

BIFU

This element allows to simply specify the relationships between the inputs and outputs of different branches which are joined together. The conservation of mass flow rate and pressure is ensured.

TUYA

This element allows to treat the motions of pipelines in the presence of an internal flow. It results from the superposition of elements "TUBE" and "POUTRE".

CLTU

Similarly to "CL1D", this element further allows to introduce coupled boundary conditions of the fluid-structure type (e.g. a closed pipe end).

BREC

This element allows to model a pipeline rupture. Before the optional rupture instant, this element behaves like a bifurcation.

The element models developed at JRC are :

num.	Name	Gibi	npt	dof	ngp	Remarks
44	ED1D	SEG2	2	1	1	1D/2D-3D structural coupling

Note that, unlike the other 1-D elements presented above, the ED1D is used to perform coupled 1-D/multi-d calculations, therefore the problem must be declared either "AXIS" or "CPLA" or "DPLA" or "TRID" and the "EULE" is not needed.

These elements are specified hereafter:

ED1D

A 2-node element used as an interface between a 1D structure and a multi-D structure.

This element can be used to couple a 2D or 3D model to a 1D model to be calculated by the EURDYN-1D code, developed at JRC Ispra (this “code” is now embedded within EUROPLEXUS, so that its usage is transparent to the EUROPLEXUS user). Of the two nodes, one is used to define the location of the interface (i.e. the point at which forces are transmitted between the two structures), while the other is only used to indicate the direction in space along which coupling is enforced.

This direction remains unchanged during deformation. The distance between the two nodes is therefore irrelevant. The 1D structure has to be separately modelled by EURDYN-1D, which is seen by EUROPLEXUS as a standard element module. A special set of input data for EURDYN-1D (ED1D “input deck”) has to be prepared. This must be included within the normal EUROPLEXUS input file, immediately after the CALCUL directive and before any additional EUROPLEXUS directives (see page I.23 and the EURDYN-1D manual listed in the References: ([33])).

The ED1D input deck must be immediately preceded by a line containing "ED1D START" (capitals, starting in column 1, followed only by blanks if any) and be immediately followed by a line containing "ED1D END" (capitals, starting in column 1, followed only by blanks if any).

The following (fixed) logical unit numbers are used by the EURDYN-1D module:

- 01 (formatted) used to copy the ED1D input deck, thus “extracting” it from the normal EUROPLEXUS input file. At run end it contains the list of 1D space plots storages;
- 04 (unformatted) used to store data for TPLOT (time plots);
- 33 (unformatted) used to store data for TPLOT (space plots);
- 34 (unformatted) used to store data for restart.

Note that units 33 and 34 are used by default to store data for TPLOT (space plots) and for a restart, respectively. They are both unformatted. These values, however, can be changed from the input.

Since EURDYN-1D is a specialised module, its usage can lead to important savings in the overall computation cost in large, complex multi-D problems when one or more portions of the structure can be conveniently represented by a 1D model.

In setting up a coupled 1-D/multi-D model, the following guidelines should be followed:

1. The multi-D part of the model is meshed as usual and an ED1D interface element is placed at each point of connection between the multi-D domain and the 1-D domain. An arbitrary number of such interface elements can be used in a calculation, according to user needs.
2. Each ED1D element must be so oriented that its first node coincides with the interface (attachment) point, while the second node only defines the orientation of the 1-D structure (i.e., of the interaction force) in multi-D space. The length of such elements is therefore irrelevant.
3. All 1-D parts of the model are represented in a single 1-D model, for which a separate input data set has to be provided.
4. When more than one 1-D part is present, each one of these will form a separate 'level' in the 1-D model (see the EURDYN-1D manual for the definition of level).
5. In setting up the 1-D model, the abscissa of each level should be oriented from the interface point towards the outside of the multi-D body. Thus, the interface node is always the first one (or the 'left' node, according to EDURDYN-1D conventions) of each level.
6. Each ED1D element should be assigned a different VM1D material (see page C.220) and the "PT1D" directive of this material should be used to assign the associated node index in the 1-D model.

Furthermore, note that a few directives, most notably saving for restart, are not available in coupled 1-D/multi-D computations.

Some care should also be taken in specifying the final times for the calculation and times for printouts and data storages. The following procedure is suggested in order to minimize potential problems:

- Assume that the initial and final physical times of interest for the calculations be TI and TF, respectively.
- Then, in the 1-D input file choose a final time (see EURDYN-1D input manual) FINTIM larger than TF, say:

$$\text{FINTIM} = \text{TF} + (\text{TF} - \text{TI}) * K$$

with : $0.01 < K < 0.1$

- Choose a final printout time:

`TPRINT(NPRINT) = TF`

If the final 1-D results should appear on the listing, and choose a final space plots storage time:

`TSTOR(NSSTOR) = TF`

If space plots in the final 1-D configuration are desired.

- In the multi-D input file (EUROPLEXUS), choose in the `CALCUL` directive a final time `TEND` larger than `TF`, but smaller than the 1-D value (`FINTIM`) specified above:

`TF < TEND < FINTIM`

- Choose printout and storage times as appropriate, by including the value `TF` if desired.

In this way, the coupled calculation will be stopped at the end by the multi-D part of the code at time `TEND`, and the printouts and storages at `TF` should be correctly produced for both the 1-D and the multi-D models.

The only drawback of this method is that the 'space plots summary' table in the 1-D listing is not produced at the end of the run because the 1-D calculation is stopped before its declared final time is actually reached. However, this is not a real problem, since the 1-D space plots file is nevertheless correctly generated.

4.1.2 2-D ELEMENTS

The various elements available in 2D are presented below. For practical reasons, they are subdivided into two groups, i.e. models developed at CEA and models developed at JRC.

The element models developed at CEA are :

num.	Name	Gibi	npt	dof	ngp	Remarks
1	COQU	SEG2	2	3	2	thin shell
2	TRIA	TRI3	3	2	1	triangle
3	BARR	SEG2	2	2	1	bar (membrane only)
4	PONC	POI1	1	2	1	circular bar(axisym.)
5	MEMB	SEG2	2	2	1	'virole' in membrane (axisym.)
7	CL2D	SEG2	2	2	1	boundary conditions
8	CAR1	QUA4	4	2	1	quadrangle with 1 Gauss pt.
9	CAR4	QUA4	4	2	4	quadrangle with 4 Gauss pt.
10	COQC	SEG2	2	3	1	thin shell with shear
15	FS2D	RAC2	4	2	1	F.S. coupling
28	PMAT	POI1	1	2	1	material point
45	TVL1	TRI3	3	2	1	Van Leer triangle
46	CVL1	QUA4	4	2	1	Van Leer quadrangle
114	BSHT	SEG2	2	2	-	bushing with translational dof
118	MAP2	—	3	2	-	Point on solid edge
121	MAP5	—	3	2	-	Point on 2D shell
124	INT4	—	4	2	2	2D interface element
131	T3VF	TRI3	3	2	1	triangle finite volume
132	Q4VF	QUA4	4	2	1	quadrangle finite volume

The specifications for these elements are given hereafter:

COQU

Reference element for all calculations with 2D thin shells.

TRIA

This element can equally well represent solids or fluids. However, if the mesh is not very regular, it may give rise to fluctuations in the distribution of masses, especially near the axis of revolution.

BARR

This element is intended for the modeling of steel reinforcement in concrete structures, or of bars that work only in traction-compression.

PONC

This element should only be used to model circular steel reinforcement in axisymmetric concrete structures.

MEMB

This element is similar to COQU, but has a purely membrane behaviour.

CL2D

This element allows to specify a condition of absorbing medium or an impedance.

CAR1

This element is especially used to represent fluids. However, due to its under-integration, it is strongly advised to add some anti-hourglass numerical damping, unless the boundary conditions themselves prevent the appearance of hourglass motions.

CAR4

This element is recommended for elastoplastic solids.

COQC

Simpler than "COQUE", it also enables to evaluate the shear, if this is not too large.

FS2D

Incompressible fluid elements which ensure the transmission of forces.

The first face (1-2) is in the fluid, the other (3-4) in the solid. To ensure a proper fluid-structure connection, it is useful that the sides 2-3 and 1-4 be as short as possible, possibly of zero length. In this last case, only the forces normal to faces 1-2 and 3-4 will be transmitted.

These elements are defined in the mesh but they work only through the directive "LIAISON". It is therefore possible to "activate" or "deactivate" this particular connection depending on the problem to be treated, by modifying the "LIAISON" directive during a "REPRISE" (see page SR.40).

PMAT

This element is particularly aimed at modeling rigid projectiles, in connection with the directive "IMPACT". It can also be used to specify added masses.

TVL1

This element has been developed for Van Leer fluids.

CVL1

Similarly to TVL1, this element has been developed for Van Leer fluids.

BSHT

The availability of the bushing element family allows to define generalized stiffness and damping between two nodes. The implemented model provides in 2D the element BSHT, with only translation degrees of freedom. All the characteristics of the bushing element are defined using "JOINT PROPERTIES" material type.

MAP2

This element is used in order to glue one slave node to a master side. The slave node should be on the side. 2 kinematic constraints are introduced in order to impose the translation dof of the slave node. These elements are defined in the topology but they work only through the directive "LIAISON".

MAP5

This element is used in order to glue one slave node to a master shell side. The slave node should be on the side. 2 kinematic constraints are introduced in order to impose the translation dof of the slave node and a kinematic constraint is added on the rotational dof of the slave node. These elements are defined in the topology but they work only through the directive "LIAISON".

INT4

The INT4 element is pure displacement interface element dedicated to the modeling of interlayers, separating "standard" structural elements. In the particular case of a composite model, this element can be considered as representing a homogeneous resin layer ensuring the interlaminar stress transfer between adjacent plies. This approach is most often referred to as "mesoscopic" laminate modeling.

T3VF

2D triangle finite volume element. The finite volume is defined as cell centred. Several options for the calculation can be chosen with OPTI VFCC.

Q4VF

2D quadrangle finite volume element. The finite volume is defined as cell centred. Several options for the calculation can be chosen with OPTI VFCC.

The element models developed at JRC are :

num.	Name	Gibi	npt	dof	ngp	Remarks
38	Q92	QUA8	9	2	4	9-node quadrilateral
39	Q93	QUA8	9	2	9	9-node quadrilateral
42	CL23	SEG3	3	2	2	3-node b.c.s
43	ED01	SEG2	2	3	10	beam/conical-shell
49	Q92A	QUA8	9	2	4	Q92 on symmetry axis
52	FLU1	QUA4	4	2	1	fluid quadrilateral
54	PFEM	POI1	1	2	1	particle finite element
56	ED41	SEG2	4	3	10	old version of ED01
58	ADQ4	QUA4	4	2	1	advection-diffusion quadrilateral
64	FL23	TRI3	3	2	1	fluid triangle
65	FL24	QUA4	4	2	1	fluid quadrilateral
70	CL22	SEG2	2	2	2	2-node b.c.s
71	Q41	QUA4	4	2	1	ALE structural quadrilateral
72	Q42	QUA4	4	2	4	ALE structural quadrilateral
73	Q41N	QUA4	4	2	1	ALE structural quadrilateral
74	Q42N	QUA4	4	2	4	ALE structural quadrilateral
75	Q41L	QUA4	4	2	1	Lagr. structural quadrilateral
76	Q42L	QUA4	4	2	4	Lagr. structural quadrilateral
77	Q95	QUA8	9	2	4	test version of Q92
97	MC23	TRI3	3	2	1	finite volume fluid triangle
98	MC24	QUA4	4	2	1	finite volume fluid quadrilateral
98	MS24	QUA4	4	2	1	spectral "macro" quadrilateral
100	Q42G	QUA4	4	2	4	ALE structural quadrilateral
105	S24	QUA4	4	2	1	spectral "micro" quadrilateral
109	FUN2	SEG2	2	2	1	cable element
140	DEBR	POI1	1	2	-	debris particle

The specifications for these elements are given hereafter:

Q92

This element can be used for precise modelling of continua. It can undergo arbitrarily large deformations. Since it is underintegrated, it is locking-free, but it may occasionally suffer from mechanisms if boundary conditions are too loose. In such cases, use of the Q93 element (which, however, is more expensive) is recommended.

Q93

This is the fully-integrated version of the Q92 element. Its use is only recommended in plane cases, when mechanisms might occur.

CL23

This element is mainly used to specify uniform pressure conditions acting on the boundaries of quadratic elements of type Q92, Q93 or Q92A.

ED01

This element is integrated through the thickness (5 points at each of 2 longitudinal stations) and offers accurate modelling in highly nonlinear cases (spreading of plasticity through the thickness). The effect of arbitrarily large membrane strains over the element thickness is taken into account, unless option "EDSS" is used (see "OPTION").

Q92A

This element should be used in place of Q92 in axisymmetric problems for those elements that have one side along the axis of symmetry (y-axis). It does not suffer from mechanisms.

FLU1

This element offers a specialised treatment which is thought to be particularly effective for fluids, in conjunction with the A.L.E. formulation. An implicit phase for the calculation of pressure is introduced during time integration. The element can be degenerated to represent a triangle by simply repeating one of the nodes in the description of topology.

PFEM

This element is used to represent a 2D (or 3D) continuum (usually a fluid) by means of the Particle Finite Element method (PFEM).

ED41

A 4-node version of the ED01 element that facilitates fluid-structure interaction for certain problems. Two nodes are used at each extremity of the element in order to define the element thickness. However, these are really one physical node since displacements, velocities etc. are coincident.

In the element numbering, the first two nodes must define an 'external side' of the element. In other words, they must not be along the element thickness.

ADQ4

4-node quadrilateral for advection-diffusion problems.

This element is used to model advection-diffusion problems in incompressible fluids with heat transfer, according to JRC's models.

FL23

3-node triangle for compressible fluids. This is an alternative to the degeneratable FLU1 quadrilateral.

FL24

4-node quadrilateral for compressible fluids. This is an alternative to the degeneratable FLU1 quadrilateral.

CL22

2-node boundary condition. This is recommended for use with 2-D Ispra models. This element automatically recognizes the element to which it is attached, and uses the most appropriate pressure discretization.

Q41

4-node quadrilateral for structural ALE calculations with reduced integration.

Q42

4-node quadrilateral for structural ALE calculations with full integration.

Q41N

4-node quadrilateral for structural ALE calculations with reduced integration. Uses Godunov algorithm.

Q42N

4-node quadrilateral for structural ALE calculations with full integration. Uses Godunov algorithm. This element doesn't work well in some test cases, so it is advisable to use Q42G instead.

Q41L

4-node quadrilateral for Lagrangian calculations with reduced integration.

Q42L

4-node quadrilateral for Lagrangian calculations with full integration.

Q95

9-node isoparametric quadrilateral with curved sides. This is a special version of the Q9 element under test, that should avoid mechanisms.

Its use is not recommended for the moment.

MC23

3-node finite volume triangle for multicomponent flows.

MC24

4-node finite volume quadrilateral for multicomponent flows.

MS24

4-node quadrilateral MACRO spectral element.

The integration points coincide with the Gauss-Lobatto-Legendre points and are determined by specifying the MICRO spectral elements S24.

Q42G

4-node quadrilateral for structural ALE calculations with full integration. Uses Godunov algorithm.

S24

4-node quadrilateral MICRO spectral element This element is used only to specify 'internal' nodes of an MS24.

FUN2

2-node cable element.

This is a specialized element for the representation of cables in 2D space, in conjunction with the FUNE material. The element is large-strain.

DEBR

1-node debris particle element.

This is a specialized element for the representation of flying debris, as e.g. resulting from an explosion or an impact, by means of spherical particles. It may be used both in 2D and in 3D.

4.1.3 3-D ELEMENTS

The various elements available in 3D are presented below. For practical reasons, they are subdivided into two groups, i.e. models developed at CEA and models developed at JRC.

The element models developed at CEA are :

num.	Name	Gibi	npt	dof	ngp	Remarks
11	CUBE	CUB8	8	3	1	brick with 1 Gauss pt
12	COQ3	TRI3	3	6	1	triangular thin shell
13	CUB6	CUB8	8	3	6	brick with 6 Gauss pt
14	COQ4	QUA4	4	6	4	quadrangular thin shell
16	FS3D	RAC3	8	3	1	F.S. connection (4-node face)
17	POUT	SEG2	2	6	2	beam
18	CL3D	QUA4	4	3	1	bound. cond. (4-node face)
19	BR3D	SEG2	2	3	1	spring or bar
20	PR6	PRI6	6	3	6	prism with 6 Gauss pt
21	TETR	TET4	4	3	1	tetrahedron with 1 Gauss pt
27	PRIS	PRI6	6	3	1	prism with 1 Gauss pt
28	PMAT	POI1	1	3	1	material point
29	CL3T	TRI3	3	3	1	bound. cond. (3-node face)
30	CUB8	CUB8	8	3	8	brick with 8 Gauss pt
32	APPU	POI1	1	6	1	support
33	MECA	SEG2	2	6	1	mechanism (articulated systems)
41	TUBM	SUPE	1*	3*	1	connection 3D-1D
47	CMC3	TRI3	3	6	2	multilayer shell
48	FS3T	PRI6	6	3	1	F.S. connection (3-node face)
51	T3GS	TRI3	3	6	5	shell (Reissner-Mindlin)
79	BILL	POI1	1	3	1	particle element (NABOR and SPH)
80	ELDI	POI1	1	6	1	discrete element
81	CUVL	CUB8	8	3	1	Van Leer cube
82	PRVL	PRI6	6	3	1	Van Leer prism
83	DST3	TRI3	3	6	15	shell (Discrete Shear Triangle)
84	DKT3	TRI3	3	6	15	shell (Mindlin)
85	SHB8	CUB8	8	3	5	thick shell
89	SPHC	POI1	1	6	1	particle element (thick shell)
90	Q4G4	QUA4	4	6	4	shell (Batoz)
111	Q4GR	QUA4	4	6	5	idem Q4G4 (simplified : 1 pt)
112	Q4GS	QUA4	4	6	20	idem Q4G4 (simplified : 4 pts)
114	BSHT	—	2	3	—	bushing with translational dof
115	BSHR	—	2	6	—	bushing with trans. and rot. dof
116	TUYM	SUPE	1*	3*	1	connection 3D-1D
117	SH3D	—	3	6	—	node to shell connector
119	MAP3	—	4	3	—	point on a triangular facet
120	MAP4	—	5	3	—	point on a quadrangular facet

num.	Name	Gibi	npt	dof	ngp	Remarks
122	MAP6	—	4	6	—	point on a triangular shell facet
123	MAP7	—	5	6	—	point on a quadrangular shell facet
125	INT6	—	6	3	1	3D triangular element interface
126	INT8	—	8	3	4	3D quadrilateral element interface
127	SH3V	—	8	3	4	node to element connector
128	MOY4	—	4	3	—	node to element mean connector
129	MOY5	—	5	3	—	node to element mean connector
130	ASHB	—	8	3	5	assumed strain thick shell
133	CUVF	CUB8	8	3	1	cube finite volume
134	PRVF	PRI6	6	3	1	prism finite volume
135	TEVF	TET4	4	3	1	tetrahedron finite volume
136	PYVF	PYR4	5	3	1	pyramid finite volume
138	Q4MC	QUA4	4	6	-	multilayered quadrangular shell
139	T3MC	TRI3	4	6	-	multilayered triangular shell

The specifications for these elements are given hereafter:

CUBE

This element is used especially with fluid materials. However, because of its under-integration, "hourglassing" phenomena may appear, if they are not prevented by the boundary conditions. Such phenomena may be contrasted by using the HOURG option (anti-hourglass artificial viscosity).

COQ3

This element must be used with care. In order to obtain good results make sure to use a symmetric mesh (ex: British flag).

CUB6

Solid element for elasto-plastic computations. This element should be used with caution, because being underintegrated it may lead to hourglassing. Use preferently CUB8 elements.

COQ4

This element is recommended for computations on 3-D shells if the deformations are small. In reality, this element is composed of 4 triangular COQ3 plates which are superimposed and symmetrized.

FS3D

Same remarks as for FS2D.

POUT

This element enables the modelling of complex profile beams submitted to a tension or bending stress. The default stability time step for this element is quite conservative (optimized for tubes?). Much larger values for different cross sections may be obtained by using the option DTML (see H 20).

CL3D

Same remarks as for CL2D.

BR3D

This element is mainly used to model concrete rebars or any other beam submitted to a simple tension.

PR6

As it is the case of CUB8, this element is recommended for elasto-plastic computations.

TETR

This element may be used for fluids or for elasto-plastic computations.

PRIS

This prismatic element is especially used for fluids (see CUBE).

PMAT

Primarily, this element enables the modelling of rigid missiles in connection with the keyword "IMPACT"; but it also enables the introduction of added masses.

CL3T

Same remarks as for CL2D.

CUB8

Solid element recommended for elasto-plastic computations (no hourglassing phenomena).

APPU

This element has one node and six degrees of freedom. It allows to use constitutive equations of type non-linear support in a chosen direction. See also page C2.108.

MECA

This element has two nodes with six degrees of freedom. It allows to use constitutive equations for mechanisms (articulations, etc.).

TUBM

Connection 3D-1D. Consult the corresponding 'liaison' (connection).

TUYM

Connection 3D-1D for moving meshes (ALE). Consult the corresponding 'liaison' (connection).

CMC3

This element is used for the modelling of an eccentric layer in relation to the average plane defined by its 3 nodes. The layer is associated with an orthotropic behaviour in the given plane. Several CMC3 elements are supported by the same nodes, but they are differently eccentric; they represent a multilayer structure.

The geometric and mechanical characteristics of the element (eccentricity, orthotropy associated to a local system) can be defined either when CASTEM2000 generates the mesh (see option CASTEM page A.30) or directly by EUROPLEXUS (see page C.95) in a normal mesh generated by COCO or GIBI.

The local reference of the element is as follows: the first axis is formed by side 1-2, the second is such that the 3rd node lies in the half-plane ($Y > 0$).

FS3T

Same remarks as for FS2D.

T3GS

3-node thick shell (Reissner-Mindlin) element with 1 integration point in the plane. It has the same local frame as COQ3.

There are 8 stress components: σ_x , σ_y , σ_{xy} , σ_{yx} , σ_{yz} , σ_{zy} , τ_{xz} , τ_{yz} .

It is a predecessor of the Q4G family, uses the same approach for representing the shear strain and is thus the best suited among T3 shell elements to be combined with Q4G shell elements.

BILL

This element is primarily aimed at the modelization of fluids or structures by using the method of particles and forces.

ELDI

This element has one node with six degrees of freedom. The size distribution obtained by using a particular growing technique (Donze 2002) may be described in the mesh file generated by Cast3M. Interactions between these elements allow to model cohesive nature of materials or contact.

CUVL

Specific element (hexahedron) for Van Leer fluids in 3D.

PRVL

Specific element (prism) for Van Leer fluids in 3D.

DST3

3-node shell element (Discrete Shear Triangle).

It is a thick shell element (Mindlin). Same local frame as COQ3.

DKT3

3-node shell element (Discrete Kirchhoff Triangle). It is a thick shell element (Mindlin). It has the same local frame as COQ3.

There are 6 stress components: σ_x , σ_y , σ_{xy} , σ_{yx} , σ_{yz} , σ_{zy} .

SHB8

8-node thick shell element obtained starting from the 8-noded brick. The 2 faces of this element are formed by the nodes: 1, 2, 3, 4 for the first face and 5, 6, 7, 8 for the second face.

SPHC

This thick shell (Mindlin-Reissner) particle element has one node with five degrees of freedom: 3 translations and 2 rotations.

Q4G4

4-node shell element (Batoz), with 4 integration points in the plane and 5 integration points through the thickness for plasticity.

There are 8 stress components: σ_x , σ_y , σ_{xy} , σ_f^x , σ_f^y , σ_{fy} , τ_{xz} , τ_{yz} .

It is a thick shell element with 4 nodes (BATOZ formulation) which accounts for the non-coplanarity of the four nodes. It is a complete but expensive version of Batoz's element.

A local frame is defined at each Gauss point: the first vector is tangent to the line ($\text{csi}=\text{cst}$.) in the sense from node 1 to node 2, the second vector is the vector product of the first by the vector tangent to the line ($\text{eta}=\text{cst}$.) in the sense from node 1 to node 4. The frame is completed so as to be right-handed.

Q4GR

4-node shell element (BATOZ) with 1 integration point in the plane and 5 integration points through the thickness for plasticity.

There are 8 stress components: σ_x , σ_y , σ_{xy} , σ_f^x , σ_f^y , σ_{fy} , τ_{xz} , τ_{yz} .

It is a simplified version of Q4G4 with a single integration point in the plane. An incomplete anti-hourglass stiffness (only in rotation) is implemented; an adjusting coefficient for anti-hourglass can be set using the following syntax:

```
"OPTI"  "HGQ4"  hgq4ro
```

The default value of hgq4ro is 0.018.

Q4GS

4-node shell element (Batoz), with 4 integration points in the plane and 5 integration points through the thickness for plasticity.

There are 8 stress components: σ_x , σ_y , σ_{xy} , σ_f^x , σ_f^y , σ_{fy} , τ_{xz} , τ_{yz} .

It is a simplified version of Q4G4 with 4 integration points in the plane.

BSHT and BSHR

The availability of the bushing element family allows to define generalized stiffness and damping between two nodes. The implemented model provides a first type of element, BSHT, with only translation degrees of freedom (available both in 2D and in 3D), and a second type, BSHR, with rotational degrees of freedom too.

All the characteristics of the bushing element are defined using "JOINT PROPERTIES" material type.

SH3D

This element is used to connect a slave node to a master edge of shell. Three kinematic constraints are introduced on the translational and rotational degrees of freedom of the slave node. The displacements and rotations of the slave node are linearly interpolated between the two master nodes. These elements are defined in the topology but they work only through the "LIAISON" directive.

SH3V

This element is used to connect a slave node to a master edge of element. It is the same as for the SH3D element except that there is no constraint on rotations. These elements are defined in the topology but they work only through the "LIAISON" directive.

MAP3 and MAP4

This element is used in order to glue one slave node to a master face. The master face is triangular in the case of the MAP3 and quadrangular in the case of the MAP4. 3 kinematic constraints are introduced in order to impose the translation dof of the slave node. These elements can be used in order to glue 2 volumic meshes. These elements are defined in the topology but they work only through the "LIAISON" directive.

MAP6 and MAP7

This element is used in order to glue one slave node to a master shell face. The master face is triangular in the case of the MAP6 and quadrangular in the case of the MAP7. 3 kinematic constraints are introduced in order to impose the translation dof of the slave node and 3 kinematic constraints are added on the rotational dof. These elements can be used in order to glue 2 shell meshes. These elements are defined in the topology but they work only through the "LIAISON" directive.

INT6 and INT8

The INT6 and INT8 elements are pure displacement interface elements dedicated to the modeling of interlayers, separating "standard" structural elements. In the particular case of a composite model, this element can be considered as representing a homogeneous resin layer ensuring the interlaminar stress transfer between adjacent plies. This approach is most often referred to as "mesoscopic" laminate modeling.

ASHB

8-node thick shell element obtained starting from the 8-noded brick. This element is identical as SHB8 but follows the assumed strain formulation. The 2 faces of this element are formed by the nodes: 1, 2, 3, 4 for the first face and 5, 6, 7, 8 for the second face.

Q4MC

4-node multilayered shell element which is a generalization of the Q4GS element. This element is also multi-material. The number of Gauss point in the thickness depends on the number of plies. The user has to define the total number of Gauss points in the thickness using the parameter NGPZ in COMP (resp. SAND).

T3MC

3-node multilayered shell element which is a generalization of the DST3 element. This element is also multi-material. The number of Gauss point in the thickness depends on the number of plies. The user has to define the total number of Gauss points in the thickness using the parameter NGPZ in COMP (resp. SAND).

CUVF

3D cubic finite volume element. The finite volume is defined as cell centred. Several options for the calculation can be chosen with OPTI VFCC.

PRVF

3D prism finite volume element. The finite volume is defined as cell centred. Several options for the calculation can be chosen with OPTI VFCC.

TEVF

3D tetrahedral finite volume element. The finite volume is defined as cell centred. Several options for the calculation can be chosen with OPTI VFCC.

PYVF

3D pyramid finite volume element. The finite volume is defined as cell centred. Several options for the calculation can be chosen with OPTI VFCC.

The element models developed at JRC are :

num.	Name	Gibi	npt	dof	ngp	Remarks
40	COQI	TRI3	3	6	15	triangular shell
53	FLU3	CUB8	8	3	1	fluid brick
54	PFEM	POI1	1	3	1	particle finite element
57	ADC8	CUB8	8	3	1	advection-diffusion brick
62	CL32	QUA4	4	6	4	b.c.s for CQD4
63	CL33	QUA9	9	6	9	b.c.s for CQD9
66	FL34	TET4	4	3	1	fluid tetrahedron
67	FL35	PYR5	5	3	1	fluid pyramid
68	FL36	PRI6	6	3	1	fluid prism
69	FL38	CUB8	8	3	1	fluid hexahedron
78	CL3I	TRI3	3	3	1	3-node b.c.s
91	CQD4	QUA4	4	6	20	degenerated shell (Hughes-Liu)
92	CQD9	QUA9	9	6	45	degenerated shell (Hughes-Liu)
93	CQD3	TRI3	3	6	15	degenerated shell (Hughes-Liu)
94	CQD6	TRI6	6	6	20	degenerated shell (Hughes-Liu)
95	CLD3	TRI3	3	6	3	b.c.s for CQD3
96	CLD6	TRI6	6	6	4	b.c.s for CQD6
99	CL3Q	QUA4	4	3	1	4-node b.c.s
101	MC34	TET4	4	3	1	finite volume tetrahedron
102	MC35	PYR5	5	3	1	finite volume pyramid
103	MC36	PRI6	6	3	1	finite volume prism
104	MC38	CUB8	8	3	1	finite volume hexahedron
107	MS38	CUB8	8	3	1	spectral "macro" quadrilateral
108	S38	CUB8	8	3	1	spectral "micro" quadrilateral
110	FUN3	SEG2	2	3	1	cable element
140	DEBR	POI1	1	3	-	debris particle
144	C272	CU27	27	3	8	27-node cube
145	C273	CU27	27	3	27	27-node cube
151	CL92	QUA9	9	3	4	9-node (3D) b.c. for C272
152	CL93	QUA9	9	3	9	9-node (3D) b.c. for C273

The specifications for these elements are given hereafter:

COQI

3 node triangular plate element.

This element can be used to model 3D plates or shells (by plane facet approximation). It is integrated through the thickness.

FLU3

8 node specialised element for compressible fluids.

The same remarks apply as for FLU1 in 2D. The element can be degenerated to represent a prism (6 nodes), a pyramid (4 nodes), or a tetrahedron (4 nodes) by suitable repetition of node numbers in the topology.

PFEM

This element is used to represent a 2D (or 3D) continuum (usually a fluid) by means of the Particle Finite Element method (PFEM).

ADC8

8 node brick for advection-diffusion problems.

This element is used to solve advection-diffusion problems in incompressible fluids with heat transfer according to JRC models.

CL32

4-node boundary condition for the CQD4.

These elements must be attached directly to the CQD4, i.e., they share the same nodes.

CL33

9-node boundary condition for the CQD9.

These elements must be attached directly to the CQD9, i.e., they share the same nodes.

FL34

4-node tetrahedron for compressible fluids. Is an alternative to the degeneratable FLU3 hexahedron.

FL35

5-node pyramid for compressible fluids. Is an alternative to the degeneratable FLU3 hexahedron.

FL36

6-node prism for compressible fluids. Is an alternative to the degeneratable FLU3 hexahedron.

FL38

8-node hexahedron for compressible fluids. Is an alternative to the degeneratable FLU3 hexahedron.

CL3I

Boundary conditions of 3 nodes.

Recommended for use with COQI triangular shell elements and in general with all 3D Ispra models. This element automatically recognizes the element to which it is attached and uses the most appropriate pressure discretization.

CQD4

4-node quadrilateral degenerated shell element (Hughes-Liu).

CQD9

9-node quadrilateral degenerated shell element (Hughes-Liu).

CQD3

3-node triangular degenerated shell element (Hughes-Liu).
Similar to CQD4 but with a triangular shape.

CQD6

6-node triangular degenerated shell element (Hughes-Liu).
Similar to CQD9 but with a triangular shape.

CLD3

3-node boundary condition element for CQD3.

CLD6

6-node boundary condition element for CQD6.

CL3Q

Boundary conditions of 4 nodes.

Recommended for use with 3D Ispra models. This element automatically recognizes the element to which it is attached and uses the most appropriate pressure discretization.

MC34

Finite-volumes: 4-node tetrahedron for multicomponent flows. This element is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC. For more information on this element, see reference [\[135\]](#).

MC35

Finite-volumes: 5-node pyramid for multicomponent flows. This element is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC. For more information on this element, see reference [\[135\]](#).

MC36

Finite-volumes: 6-node prism for multicomponent flows. This element is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC. For more information on this element, see reference [\[135\]](#).

MC38

Finite-volumes: 8-node hexahedron for multicomponent flows. This element is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC. For more information on this element, see reference [\[135\]](#).

MS38

Finite-volumes: 8-node hexahedral MACRO spectral element. This element is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC.

The integration points coincide with the Gauss-Lobatto-Legendre points and are determined by specifying the MICRO spectral elements S38.

S38

8-node hexahedral MICRO spectral element.

This element is used only to specify 'internal' nodes of an MS38.

FUN3

This is a specialized element for the representation of cables in 3D space, in conjunction with the FUNE material. The element is large-strain.

DEBR

1-node debris particle element.

This is a specialized element for the representation of flying debris, as e.g. resulting from an explosion or an impact, by means of spherical particles. It may be used both in 2D and in 3D.

C272

This element can be used for precise modelling of continua. It can undergo arbitrarily large deformations. Since it is underintegrated, it is locking-free, but it may occasionally suffer from mechanisms if boundary conditions are too loose. In such cases, use of the C273 element (which, however, is more expensive) is recommended.

C273

This is the fully-integrated version of the C272 element. Its use is only recommended when mechanisms might occur.

CL92

9-node boundary condition element for C272.

CL93

9-node boundary condition element for C273.

4.2 SANDWICH (MULTI-LAYER) ELEMENTS

Some shell elements developed at Ispra may be defined as a sandwich (an assembly) composed of several layers, each one having its own material. The usual hypothesis that fibers (straight lines across the thickness of an undeformed shell) remain straight during deformation is retained. The fiber may or may not be/remain normal to some ‘mean’ or ‘reference’ shell surface depending on the theory (Kirchhoff or Mindlin) assumed, i.e. on the fact that transverse shear strains are taken into account or not. As a consequence of fibers remaining straight, the deformation assumes a simple pattern through the thickness. In sandwich elements the state of stress may be discontinuous at layer interfaces because the different materials have in general different properties. No detachment (delamination) of the various layers is modelled at present.

These models are useful e.g. for representing reinforced concrete structures, or other composite materials (sandwich structures).

For the moment, this feature is available for elements of type ED01 in 2D and elements of type COQI, CQD3, CQD4, CQD6, CQD9, T3MC, Q4MC, Q4GS, Q4GR, QPPS, T3GS, DKT3 in 3D.

In order to use these models, see the **SAND** directives in the Geometry (page C.45) and in the Materials (page C.1110) Sections of the manual.

4.2.1 LOCATION AND NUMBER OF THE INTEGRATION POINTS

When using sandwich elements, the number of layers and of integration points through the thickness in each layer is specified by the user and may therefore vary from test case to test case. In order to facilitate the use of these elements, the following rule has been chosen:

For sandwich elements, the numbering of the integration points proceeds along each fiber (through the thickness) first, and from the lower to the upper part of the fiber

The lower and upper element surfaces are defined by element numbering and the right-hand rule, as usual in EUROPLEXUS. The above numbering scheme is called ‘fiber-first’, as opposed to ‘lamina-first’ numbering schemes.

As an example, consider an element with two fibers, i.e. two integration stations in the element’s plane (sometimes called lamina) and 5 integration points through the thickness. Then, the points numbered 1 to 5 belong to the first fiber, while points 6 to 10 belong to the second fiber. Furthermore, points 1 and 6 are the bottom ones, 3 and 8 the middle ones and 5 and 10 the top ones, and so on.

For ease of reference, the precise numbering schemes for elements susceptible of being multi-layered is given below.

ED01 element

The numbering scheme is fiber-first (i.e. identical) for both the old (until August 1995) and the new (homogeneous or multilayered) element.

COQI element

The unlayered element used until August 1995 an unusual numbering rule where the outer integration points were numbered first, then the intermediate points and finally a (single) point in the mean surface (see the Technical Note: “A Triangular Plate Element for the Nonlinear Dynamic Analysis of Thin 3D Structural Components”, reference [87]). The element had 13 points altogether.

The new numbering rule is fiber-first, and is the same for both the unlayered and the layered element. For the unlayered element, 3 fibers of 5 points each (15 points altogether) are assumed, while in the multilayer element the fibers are still 3 but the number of points through the thickness may vary.

CQDx elements

In the versions before August 1995, (unlayered element) a ‘lamina-first’ numbering rule was assumed. Along each lamina, points were numbered along the η direction first, then along the ξ direction (these directions as well as the lower and upper faces of the elements are uniquely defined by the numbering of the element nodes). The number of points through the thickness was chosen by the user.

In the current version, for both the homogeneous and the multilayer elements, integration points are numbered fiber-first and of course the number of points through the thickness is still variable.

Q4MC and T3MC elements

Those element are a generalization of the Q4GS and DST3 elements. The number of total Gauss point through the thickness must be defined with the NPGZ parameter in the dimensioning section.

5 INTRODUCTION TO FLUID-STRUCTURE INTERACTION

EUROPLEXUS offers a rich variety of models for Fluid-Structure Interactions (FSI). The following is a short introduction to FSI and a tentative classification of the models available in the code, in order to guide the user in the choice of the most appropriate FSI models for the applications of interest. For a more detailed overview of the available FSI models see e.g. [303].

Fluid-structure interaction (FSI) phenomena play an important role in many areas, ranging from aeronautical and space applications, to civil and marine/offshore engineering and to the transport industry, to name just a few. The EUROPLEXUS development team has been involved for many years in the development of numerical methods for FSI modeling applied to safety studies—initially for the nuclear industry and more recently for conventional power plants (electrical machinery)—to civil engineering (vulnerability of buildings and other critical infrastructures to terrorist attacks) and to land mass transports (blast effects in railway stations, metro lines, rolling stock).

All these studies are characterized by the violent blast loading, resulting either from an accident or from an intentional attack, and by the very short time scale (fast transient dynamics). Strong pressure waves propagate in the fluid and load the surrounding structures, which typically undergo large deformations and in some cases reach complete failure and fragmentation.

For this class of problems, an explicit time marching algorithm is usually adopted, where the fluid is modelled as compressible and inviscid (Euler equations). An Arbitrary Lagrangian Eulerian (ALE) formulation is adopted for the fluid sub-domain, while the structure is Lagrangian.

Three different discretization approaches are available in the code for the fluid sub-domain: finite elements (FE), node-centred finite volumes (NCFV) and cell-centred finite volumes (CCFV):

- In the FE case, kinematic variables (such as the velocity v) are discretized at the element nodes, while state variables (such as the fluid pressure p) are discretized at Gauss points, typically located at the element centroid.
- In the NCFV case, a virtual FV (dual) mesh centred on fluid nodes is automatically built up starting from the FE-like (primal) mesh provided in input, and all variables are discretized at the nodes.
- In the CCFV case, the FV mesh looks similar to the FE case, but all variables are discretized at the volume centres. Note that in this case the ‘nodes’ carry no relevant information other than their position, used to compute the volume.

The coupling between the fluid (ALE) and the structure (Lagrangian) is realized by suitable FSI algorithms. Two broad classes of algorithms are available in the code. The first class uses a **strong** approach, based on constraints imposed on the (velocity of) fluid and structure nodes at the F-S interface. The second class uses a **weak** approach, based on direct application of fluid pressure forces to the structure. This terminology (strong/weak) is tentatively adopted here in an attempt to characterize the different nature of the two approaches, but it should not be confused with other uses of the same terms in the literature, in particular with weak (i.e. integral) forms in continuum mechanics. Traditionally, strong FSI algorithms are mainly used in FE, while weak FSI algorithms are mainly used in FV.

Yet another classification of FSI algorithms concerns the degree of deformation/damage that the structure can undergo (and thus the type of application). One class of **basic** algorithms

is suitable for large motion and large deformation of structures, but only provided these do not fail. Another class of algorithms can go up to complete failure, and fragmentation, of the loaded structures. Finally, FSI algorithms can be classified in three types according to spatial discretization: (nodally) conforming, non-conforming, and embedded (or immersed). The first two types are mostly used in applications without structural failure (but there are exceptions), while embedded algorithms are the only ones capable of dealing with extreme loading cases where the structure fails and breaks up in pieces.

The following Table summarizes the architecture of a typical FSI model, consisting of a *detection* module and of an *enforcement* module. The various types of approaches (*basic* / *embedded* or *strong* / *weak*) are briefly summarized.

Table 1: A classification of FSI algorithms

FSI Algorithm	FSI Detection	Basic	No structural failure, moderate rotations.
		Embedded	Structure can fail, arbitrary rotations.
	FSI Enforcement	Strong	Constraints on F and S velocities are imposed, e.g. by Lagrange multipliers (implicit).
		Weak	Pressure forces are transmitted from the fluid to the structure; structure motion provides weak feedback on fluid ($S = \text{master}$ / $F = \text{slave}$).

The following Table completes the classification of the available FSI models, by showing the type of spatial discretization (*conforming*, *non-conforming* or *embedded*), the name of the input directive (when applicable/needed), and the associated fluid discretization(s).

Table 2: The available FSI algorithms

FSI Algorithm	Detection Strategy	Spatial Discretization	Enforcement Strategy	Name / Command	Use with
	Basic (no structural failure)	Conforming F - S meshes	Strong	FSA	FE, NCFV
			Weak	Merge F - S nodes	CCFV
		Non- conforming F - S meshes	Strong	FSA	FE, NCFV
			Weak	Declare non-matching F -nodes	CCFV
	Embedded (structure can fail)	S -mesh is immersed in the F -mesh	Strong	FLSR	FE, NCFV
			Weak	FLSW	CCFV

6 GROUP A—PROBLEM TYPE AND DIMENSIONING

Object:

The keywords of this group enable the definition of the problem, and the dimensioning (memory allocation) of the computation.

Comments:

These keywords are described in detail in the following pages.

6.1 TITLE AND PRELIMINARY INFORMATION

6.1.1 TITLE

The title, composed of a maximum of 72 characters, is the first card of the data set and is compulsory.

The contents of this data card is printed at the top of each page edited by EUROPLEXUS together with the date of the run.

6.1.2 INPUT DATA ECHO AND INPUT CHECK UP

Object:

These keywords are used to obtain an echo on the terminal or console window of the input data being precessed and to check up the syntactical correctness and the coherence of the data.

Syntax:

`< $["ECHO" ; "NOEC"]$ >`

Comments:

The "ECHO" optional keyword produces an echo on the screen or terminal of the input file directives as they are being processed. If the input file is very long, this may be annoying. By default no echo is produced. See also the option OPTI ECHO, Page H.50, which may be used at any point of the input file after the dimensioning.

The "NOEC" optional keyword disables the echo on the screen or terminal of the input file directives as they are being processed. By default no echo is produced. See also the option OPTI NOEC, Page H.50, which may be used at any point of the input file after the dimensioning.

6.2 INTERACTIVE (FOREGROUND) EXECUTION

Object:

The **CONV** directive can be used to execute EUROPLEXUS **interactively**, i.e. in the foreground (as opposed to the default **batch** or background execution). In this execution mode, the user pilots the advancement of the computation, and the results at selected time instants can be either visualized on graphic screens of various types (**on-screen** rendering), or be stored in graphic files of various types (**off-screen** rendering), including animation files.

Syntax:

```
< "CONV"    $[ "TEKT" ; "WIN" ; "PS" ; "MIF" ]$ >
```

TEKT

Tektronix 4014 screen (PLOT-10 graphics language).

WIN

MS-Windows graphics (QuickWin or OpenGL). Note that OpenGL may be supported also on non-Windows platforms, e.g. on Linux.

PS

PostScript (but see also the TRAC PS interactive command).

MIF

FrameMaker MIF (but see also the TRAC MIF interactive command).

Comments:

When interactive execution is chosen, EUROPLEXUS reads the input data-set as usual, performs step 0 to initialise the computation, then prompts the user for commands from the keyboard with the phrase: **COMMANDE ?**.

The user can then issue various commands and subcommands **typically from the keyboard** in order to pilot the computation. For example, he can ask the program to perform a certain number of steps, then to pause again for further commands. Each time the calculation is paused, the current computational model can be visualized (e.g. by means of the built-in

OpenGL-based visualization module) and information concerning the computation (time step, CPU time, etc.) can be printed. Furthermore, the current time step can be varied by the user.

As an alternative to typing commands by hand from the keyboard, such commands may be included in the regular EUROPLEXUS input file by enclosing them into a special directive `PLAY ... ENDPLAY` as described in Section 15.6 (Page I.24) and in Section 16.3.4 (Page ED.140).

All EUROPLEXUS interactive commands are described in detail in Section 17 (Group O).

6.3 FILE MANAGEMENT

This Section gives some information about the management of files related to the EUROPLEXUS program.

6.3.1 DEFAULT FILE NAMES

The code tries whenever possible to use default values both for the file names that it needs during the calculation, and for the associated logical unit numbers.

The idea is that logical unit numbers (a concept specific only to FORTRAN) are irrelevant for the user and should be totally transparent to him or her. What *does* matter for users is file *names*.

In many cases the code helps users by providing default values for such names (the behaviour may depend on the platform, though, see below). Of course, users are free to choose file names (and even unit numbers) if for any reason they find such defaults inconvenient.

Default names under MS-Windows

Under the MS-Windows platform default file names are built automatically by the code by using the base name of the (main) input file used in the run.

For example, suppose that the main input file is called `test01.epx` and resides on a directory `D:\Work`. The user may launch the program for example by opening a console window on the directory `D:\Work` (i.e., such that `D:\Work` is the current directory) and by typing a command such as:

```
epx_bench -l test01
```

The actual command may vary depending on the implementation.

The code interpretes the name passed on the command line (after removing any options such as the `-l` above) as the name of the main input file. It removes the extension `.epx` from this name, if present, and uses the resulting string as the base name for default file names. Thus, the user may give a full file name, complete with its path, if preferred.

Suppose now that for run `test01` the user needs to read a CASTEM 2000 mesh and wants to produce results for ALICE TEMPS and CASTEM 2000.

The first task may be accomplished by the CAST directive (see page A.30). If the mesh file is formatted, and the global (main) mesh object is called `model`, this directive may take any of the following forms:

1. CAST FORM 'test01.msh' model
2. CAST FORM 9 model
3. CAST FORM model

In the first form, the file name containing the mesh is given explicitly. The code associates this name to the default unit number, which for CASTEM mesh reading is number 9.

In the second form, the unit number is given explicitly. The code uses the given number as unit number and opens a file without specifying its name. This results in different behaviour depending on the platform. Under MS-Windows, a file `fort.9` is opened.

In the third form neither the name nor the unit number are specified. The code uses the default file name (`test01.msh` in this case) and the default unit number (9).

It is clear that, whenever possible, the third form is preferable. This requires, however, that the mesh file name matches the main input file name (and is on the same directory).

When either of these conditions is impractical, the first form should be used by preference (name chosen by the user, unit number chosen by default by the code).

The second syntax is obsolete and should be avoided in new inputs.

The task of producing results files may be accomplished similarly, by the ECRI directive (see page G.70). For example, for the ALICE TEMPS output, this directive may take any of the following forms:

1. ECRI FICH ALIC TEMP 'test01.alt' /CTIME/ . . .
2. ECRI FICH ALIC TEMP 11 /CTIME/ . . .
3. ECRI FICH ALIC TEMP /CTIME/ . . .

Default names under Unix

Under the Unix platform(s), the same concepts apply as seen above.

Let us assume for exemple that the main input file is called `test01.epx` and resides on a directory `/u/user/My_dir`. The user may run the program starting from his work directory (`My_dir`) by the command:

```
europlexus test01.epx
```

The mesh file will be sought in this directory under the name `test01.msh`. If some other input files are required, they will also be sought in this directory, under the same name but a different extension. The same rule applies to output files, in particular to the listing file.

However, for reasons related to efficient exploitation of disk space, output files may of course be directed to special directories. It is therefore recommended to contact your system administrator to learn about local disk space policies.

Alternatives under MS-Windows

Under the MS-Windows platform, there exist other ways of launching the program, alternative to the one (command line) assumed in the above example.

For example, a user might double click on the EUROPLEXUS executable or on an icon (shortcut) to this executable, or on the input file `test01.epx` (provided file association is used), etc.

Whenever a file name is not directly available (like in the first two alternative methods listed above), the program prompts interactively the user for such a name. The behaviour is thereafter identical to the case of command-line execution.

Comments:

Currently, default file names are available for the following directives:

- Reading a CASTEM 2000 or GIBI mesh, **GIBI** and **CAST** keywords (see page A.30);
- Writing result files, **ECRI FICH** keywords (see page G.70);
- Reading a results file (post-treatment by EUROPLEXUS), keyword **RESU** (see page ED.10).
- Reading a modal basis file in a multi-domain computation (see **STRU** directive on page I.15).

6.3.2 EXPLICIT FILE OPENING

Object :

This directive allows to open a file directly from the input data, by specifying its logical unit number and its name.

It may be used whenever either the user wants to override a file choice made by default by the code (example: a very long calculation requires its results file to be written to a remote disk, due to space problems), or to force opening of a unit that is otherwise not opened by the code.

Note, however, that including this directive in input files generally renders them not portable across different platforms or even different machines belonging to the same platform. In fact, absolute (full) file names are usually required.

For this reason, it is preferable to use the short file name syntax (or even the default name syntax) described in the previous sections whenever possible.

Syntax

```
"OPNF" < "FORMAT" ; "WXDR" ; "RXDR" > nfic 'nomfic'
```

FORMAT

Specify that formatted file is required. By default, the file is unformatted.

WXDR

Specify that XDR writable file is required. For the output "K2000" file for example.

RXDR

Specify that XDR readable file is required. For the input mesh file for example.

nfic

Logical unit number of the file.

'nomfic'

File name.

Comments :

It is forbidden to open explicitly the logical unit numbers 0, 5, 6 and 7, that are reserved in most operating systems. Other unit numbers reserved by EUROPLEXUS are 15, 16 and from 91 onwards.

The way of coding the file name depends upon the operating system:

- Under Unix, the full name must be given, for example: `/u/user/ssrep/fich.sortplex`.
- Under MS-Windows the full path name is recommended, like under Unix. For example: `D:\Users\Epx\myfile.abc`.

For other operating systems, please contact your system administrator.

6.3.3 EXPLICIT OUTPUT DIRECTORY DEFINITION

Object :

This directive allows to define a directory for output files.

Like for EXPLICIT FILE OPENING, the rules for the name of the given directory may vary from one platform to another.

Syntax

```
"OPNF"  "DRST"  |[ 'nomdir' ; "PWD0" ]|
```

DRST

Following by the name of directory used for named result files (ALICE, ALICE TEMPS, K2000, INP, VTK-PARAVIEW, ...). Alias : DPRV or PATH.

'nomdir'

Name of directory used for result files.

PWD0

Directory name is the current directory.

Comments :

In the case of an UNIX system, the defined directory is an absolute path. In the case of a WINDOWS system, the directory is given as a relative path.

In both cases, if given directory does not exist, it is created.

6.4 TYPE OF MESH, PROBLEM AND LISTING

Object:

1/ To define the mesh type that will be used:

- Mesh in free format.
- Mesh generated by GIBI or CAST3M (uses objects).
- Mesh generated by I-DEAS Version 6 or Master Series (uses so-called “permanent groups”).
- Mesh described in a MED file

2/ To define the general type of computation:

- Axisymmetric, 2-D, 3-D, 1-D, etc.
- Lagrangian or Eulerian or A.L.E. formulation.
- Presence of mechanical rezoning for ALE computations.

3/ To define some general options about the form of printed results (useful to reduce the size of the output listing in extremely large test cases)

Syntax:

```

$[
  $[ "GIBI" ; "CASTEM" ]$ <[$[ "FORM" ; "XDR" ; "BINA" ]$>
                                <[$[ ndis ; 'file_name' ]$> 'nomobjet' ;
  "IDEA" <[$[ ndidea ; 'file_name' ]$> <"REWR" > <"MAPP" > ;
  "MEDL" 'file_name' ;
]$

|[ "DPLA" ; "CPLA" ; "AXIS" <"HOLE" hole> ; "TRID" ]|

< $[ "LAGR" ; "EULE" ; "ALE" ]$ >

< "NAVIER" > < "HOMO" nhtube >
< "MBETON" nssc >
< "FRQR" nfrqr >
< "LAGC" >
< "MBACON" < "POST" > >
< "SNECMA" < "FANTOME" < coefan > > >

```



```

< "SAUVEGARDE" ... >      < "REPRISE" ... >

< "ADDF" < "NAVS" > < "TEMP" > < "TURB" > >
< "MECA" >
< "MEDE" >
< "EROS" <ldam> <CROI> <LIMI>      >
< "RISK" < "PROB" |[ "FERR" ; "YETP" ]| >
      < "LUNG" |[ "BAKE" ; "LEES" ]| >
      < "SPLI" > >

< "BMPI" >

```

GIBI

Mesh generated by GIBI (GIBI objects will be read to define the mesh) and stored with the 'SORT' directive (see Comments below).

CASTEM

Mesh and other characteristics (geometrical, material, champoints) generated by CASTEM2000 and stored with the 'SAUV' directive (see Comments below).

FORM

The CAST3M generated data are to be read in formatted (ascii) mode (this is the default).

XDR

The CAST3M generated data are to be read in XDR mode.

BINA

The CAST3M generated data are to be read in binary mode.

ndis

Number of the logical unit of the mesh file.

'file_name'

Complete path localising the mesh file, under Unix operating systems. If both this and the unit number ndis are omitted, the code chooses a name and a unit number by default (see page A.27).

'nomobjet'

Name of the whole (main) object meshed by GIBI.

IDEA

Mesh generated by I-DEAS. At the moment, only formatted IDEAS files are allowed in input.

ndidea

Number of the logical unit of the mesh file.

'file_name'

Complete path localising the mesh file, under Unix operating systems. If both this and the unit number `ndis` are omitted, the code chooses a name and a unit number by default (see page A.27).

REWR

Write a new I-DEAS file with re-ordered numbering (no holes), see details below.

MAPP

Stop run after re-writing the new I-DEAS file with re-ordered numbering, see below for details.

MEDL

Mesh described in a MED file.

'file_name'

Full path of the med file (this file contains the mesh). The key word OPEN followed by a unit number is not available to open a MED file.

AXIS

Axisymmetric computation.

HOLE

Optionally, for axisymmetric cases a central 'hole' may be specified by this keyword: in this case the given hole radius is automatically added to the mesh radial coordinates given in input (this is a shorthand alternative to providing a mesh with the actual hole in it).

DPLA

Two-dimensional plane strain computation.

CPLA

Two-dimensional plane stress computation.

TRID

Three-dimensional computation.

LAGR

Computation with the Lagrangian formulation (default). All nodes are Lagrangian. THE GRIL directive is not required.

EULE

Computation with the Eulerian formulation. All nodes are Eulerian. THE GRIL directive is not required.

ALE

Computation with the A.L.E. formulation. THE GRIL directive may be used to specify the motion of nodes.

NAVIER

One-dimensional uncoupled calculation (fixed pipelines), with incompressible or nearly incompressible fluids. The calculation may only be done in Eulerian. Therefore, it is mandatory to specify also the directive "EULE".

nhtube

Maximum number of tubes per unit cell (homogenised material).

nssc

Number of layers for the CMC3 element with the BETO material. By default, there is just one layer per element. This number must be between 1 and 20.

nfrqr

Frequency of searching neighbour nodes. By default = 1. This option, valid only for calculations with the NABOR or SPH methods, allows the user to considerably reduce the calculation time.

LAGC

This keyword specifies that the contact forces due to impact and sliding will be computed implicitly by the Lagrange multipliers method. This enables the coupling with the permanent connections (relations, boundary conditions, etc.)

MBACON

This keyword specifies that the characteristics of multi-layer homogenised elements will be read from a BACON file (see page C.165).

POST

This option of MBACON indicates that the rupture criteria will be evaluated in the multi-layer elements during the direct calculation. During post-treatment, this option allows to compute the maximum deformations (upper and lower "skin"), and also the deformations along the directions of strain gauges.

SNEC

This option indicates that one does a calculation of impact on turbine blades, with special treatment of the matter front.

FANT

This option of the preceding keyword allows to declare void elements as "phantoms": they do not take part in the calculation.

coefan

This is the ratio between the current densities and the nominal ones below which the element is declared "phantom". By default this is = 1.E-3.

SAUV

The directives for saving and restart are described in Section "SR" (see page SR.10 and following).

ADDF

Advection-diffusion computation. The Eulerian description is used, therefore the "EULE" keyword is compulsory in this case. See description on page A.31.

NAVS

Solution of Navier-Stokes equations (for fluid velocities) has to be performed in the advection-diffusion computation. See description on page A.31.

TEMP

Solution of the temperature equation has to be performed in the advection-diffusion computation. See description on page A.31.

TURB

Solution of the turbulence equations (k, eps) has to be performed in the advection-diffusion computation (this option is still under development).

MECA

ALE computation with mechanical rezoning model.

MEDE

to create a med file. See description on page G.70.

EROS

This keyword activates the "erosion" algorithm of the code. The algorithm has the following characteristics:

1- Those elements whose failure criterion (damage, principal strain, minimum pressure, ...) is beyond a certain level are considered as eroded and are ignored during the rest of the calculation (GHOST status). The concerned materials are at the moment: **LEM1** (see

page C.147, Section 9.6.5), ZALM (see page C.148, Section 9.6.6), LMC2 (see page C.149, Section 9.6.7), VM23 (see page C.241, Section 9.6.31), MINT (see page C.262, Section 9.6.48), GLAS (see page C.280, Section 9.6.53), and LSGL (see page C.290, Section 9.6.55). Erosion can also be defined using displacement erosion (see page C.67, Section 8.1.15) or using a minimum time step size of an element (see page I.20, Section 15.4).

2- In the case of a calculation with contact by sliding surfaces in 3D (see page D.180, Section 10.21), the contact surfaces are updated by eliminating the eroded elements.

3- The list of the eroded elements is stored in the results file. This allows to remove them from the visualization (if so desired) during the post-treatment.

4- The erosion can also be activated for a part of the elements (see page C.69, Section 8.1.18).

5- Obsolete are the keywords FAIL and GHOS.

ldam

Optional parameter indicating the number of failed Gauss points that cause erosion of the element, in proportion to the total number of Gauss points of an element. It should lie between 0 and 1. The default value is 1. The value 1.0 indicates that an element is eroded when *all* its Gauss points have failed. The value 0.5 indicates that an element is eroded whenever approximately half of its Gauss points have failed. The special value 0.0 may be used to indicate that an element is eroded when any one of its Gauss points fails. This value is global and is used (as a default) for all elements in the current calculation. However, specific material types (see e.g. LEM1) may contain parameters that allow to override this value. In this way the user may set different values of the erosion threshold in different parts of the model (e.g. low values or even 0.0 for very brittle materials such as glass, and high values for ductile materials such as metals).

CROI

This option allows the erosion of crossed elements. The calculation is not stopped in such case.

LIMI

Special limitation for the element erosion: 2D: only elements, which have not 3 nodes connected to already failed elements can fail. This avoids large zone with eroded elements.

BMPI

The simulation will only be executed using a parallel MPI version of EUROPLEXUS. Any sequential version running the case will produce a clean stop just after the keyword is read, declaring any qualification as valid.

RISK

This keyword activates the calculation of risk analysis related to explosive events. Note that to perform this type of analysis, it is mandatory to use standard measurement units. In particular, the pressures must be expressed in Pa. This is because the model internally uses some non-dimensionless constants. Furthermore the model assumes that the

atmospheric pressure has the standard value of 1.D5 Pa, i.e. 1 bar. The risk estimation is performed according to the two references listed below. In order to compute the total probability from the probit functions, two different approaches can be chosen using the keyword **PROB**. A very conservative probability function from Yet-Pole can be activated with **YETP**. The more realistic probability function of Ferradás can be chosen with **FERR** (this is the default in case no **PROB** is defined). Two different formulations for risk of lung hemorrhage can be activated using the keyword **LUNG**. The default is equation (7, Baker, **BAKE**) from Ferradas paper. Equation (9, Lees, **LEES**) doesn't consider the impulse and is conservative. Note that thre optional sub-directives **PROB ...** and **LUNG ...** must be re-defined in case of results reading from an Alice file. This allows to perform a calculation, say, with the default values (i.e. by specifying only the **RISK** directive, possibly followed by **SPLI** if so desired), and then to do several post-processing of the results each time computing (and visualizing) the risk by using a different set of sub-options (e.g. once by using the Ferradas probits and another time using the Yet-Pole probits), without having to run again the main calculation (which may be very time-consuming). Therefore, be aware that in calculations with risk it is mandatory to re-define the entire **RISK** directive before the **RESU** directive (which typically reads the results from an Alice file).

SPLIT

The risk of death is splitted in risk of head impact, risk of body impact, and risk of lung haemorrhage. The result is only usable for **.pvtk** files.

References

The “probit” functions for risk estimation are taken from:

- E. González Ferradás, F. Díaz Alonso, M. Doval Miñarro, A. Miñana Aznar, J. Ruiz Gimeno and J.F. Sánchez Pérez: *Consequence analysis by means of characteristic curves to determine the damage to humans from bursting spherical vessels*. Process safety and environmental protection **86** (2008), 121–129.
- I. Yet-Pole, Cheng Te-Lung: *The development of a 3D risk analysis method*. Journal of hazardous materials **153** (2008), 600–608.

Comments:

The **CASTEM** directive is meant to read data produced by **CAST3M** and saved by the directive **”SAUV”**. It will read also objects of type 'champoint' explicitly stored, together with the mesh, by **CAST3M**.

On the other hand, the **GIBI** directive is meant to read only mesh objects (maillage) produced by **CAST3M** and saved by the directive **”SORT”**. No champoints may be stored by **CAST3M** with this directive, therefore they may not be transmitted to **EUROPLEXUS**.

The syntax in **CAST3M** is:

(SORT)

Assume we have an object 'mymesh' of type 'maillage' to write on the file 'myfile.msh' on the current directory. Then :

```
. . .  
OPTI SORT 'myfile.msh' ;  
SORT mymesh ;  
. . .
```

(SAUV)

Assume we have an object 'mymesh' of type 'maillage' and an object 'mychampnt' of type 'champoin' to write on the file 'myfile.msh' on the current directory. Then :

(formatted mode)

```
. . .  
OPTI SAUV FORM 'myfile.msh' ;  
SAUV FORM mymesh mychampnt ;
```

or:

(XDR mode)

```
. . .  
OPTI SAUV 'myfile.msh' ;  
SAUV mymesh mychampnt ;
```

or:

(binary mode)

```
. . .  
OPTI SAUV BINA 'myfile.msh' ;  
SAUV BINA mymesh mychampnt ;
```

Note that, if the data have been produced by CAST3M in formatted mode, then the keyword FORM may optionally be used in the EUROPLEXUS directive, for clarity, but it is not necessary, since this is the default reading mode for this directive.

On the other hand, if the data have been produced by CASTEM in "XDR" (rep. "BINA") mode, then the keyword XDR (resp. BINA) is mandatory in the EUROPLEXUS directive.

Note also, the XDR mode is the default mode for CAST3M output "SAUVER" file.

In the case of the "GIBI" directive, the mesh file is always formatted.

If neither the keyword "GIBI" nor "CASTEM" appear, the program assumes that the mesh is directly given in the main input file under the form of a list of coordinate values, and as many lists of node index (topology) values as there are element zones. This format is also known as the 'COCO' type format.

This form allows also to use meshes issued from mesh generators other than CASTEM-GIBI or even, in case for example the mesh is quite simple, to enter this data directly, or to generate them by an independent software. For more information, consult the 'GEOM' directive.

The type of problem directive must be specified and contain at least one keyword (at least 2 for non-Lagrangian cases), for example:

"AXIS"

or

"TRID" "ALE"

The various options are mutually incompatible.

I-DEAS mesh

When the IDEA directive is used, the program reads the mesh from an I-DEAS 'universal file'. This file may contain other information besides the mesh, but the extra information is ignored.

The program interprets the following information: 1) nodal coordinates (dataset 781 or 2411) ; 2) element topology (dataset 780 or 2412); 3) permanent groups (dataset 752 or 2429 or 2430). Permanent groups are identified by a name, which can then be used in the EUROPLEXUS input file (like in the case of CASTEM mesh) to identify the correspondig list of nodes or elements.

Note that normally I-DEAS universal files do not have consecutive node or element numbers, while EUROPLEXUS requires consecutive numbering (and starting from 1). In order to solve

this problem, use the optional **REWR** directive: the code reads the universal file (use extension ‘.unv’), re-orders the mesh numbering and writes the result in a new universal file (same name but with the characters ‘new’ appended). Nodes are re-ordered simply by eliminating ‘holes’ in the numbering. For the elements, however, a subdivision into homogeneous ‘blocks’ of the same element type, material and physical properties (geometrical complements) is performed. A summary of the resulting blocks is printed on the listing.

Elements are re-ordered subdividing them into homogeneous ‘blocks’ of the same element type, material and physical properties (geometrical complements). This permits the mapping of the I-DEAS element library onto the EUROPLEXUS one, for the relation between I-DEAS and EUROPLEXUS element libraries is not unique. The ordering criterion followed by the procedure is: [material property number] - [element type number] - [physical property number]; as an example, if some elements have to be declared as the last ones (i.e. CLxx elements when present), they must be associated to the highest material number in the I-DEAS mesh.

A list of the resulting blocks is printed on the listing, containing the number of elements, the element type in I-DEAS mesh and a list of possible choices for the corresponding EUROPLEXUS element type; these informations are useful in order to set up the declaration of the geometry in the EUROPLEXUS input file (see 5.3).

The **MAPP** optional directive may be used in order to stop the code right after writing the re-ordered universal file.

Example 1: EUROPLEXUS input file:

```
$-----  
Example of use of IDEAS universal file: 1. file re-writing  
IDEA 'myfile.unv' REWR MAPP  
DIME TERM  
FIN
```

This input reads in universal file ‘myfile.unv’, re-orders the mesh and produces a new universal file ‘myfile.unvnew’.

It is important to note that in order to post-process the EUROPLEXUS results with I-DEAS, use should be made of the reordered universal file (myfile.unvnew in the above example).

Example 2: EUROPLEXUS input file:

```
$-----  
Example of use of IDEAS universal file: 2. actual computation  
IDEA 'myfile.unvnew'  
DIME  
...  
(problem definition, using 'permanent group' names)  
FIN
```

In this second example, the geometry is read from the re-ordered universal file, and in any successive input directive the names of permanent groups may be used to define element or node lists. The syntax is the same as with CAST3M objects.

MED mesh

When the MEDL directive is used, the program reads the mesh from a MED file. This file may contain other information besides the mesh. Displacements could be read and used for the initialization (See description on page E.180). Other extra informations are ignored.

From the elements families and from the nodes families the program reconstructs the elements groups and the nodes groups. Groups are identified by a name, which can then be used in the EUROPLEXUS input file (like in the case of CASTEM mesh) to identify the corresponding list of nodes or elements. The elements groups described in a MED file are homogeneous ‘blocks’ of the same geometric support.

Note that the MED elements numbers are not necessarily the EUROPLEXUS elements numbers.

A summary of the resulting blocks is printed on the listing.

Warning

For an axisymmetric computation, the program considers a sector of ONE RADIAN. Therefore, all the forces, added masses, etc. must be defined correspondingly.

This has to be taken into account in particular when defining the mass associated to a material point: the “true” mass shall be divided by 2π .

Example:

If the whole force is F_{tot} , the force to be introduced in EUROPLEXUS is:

$$F_{\text{plex}} = \frac{F_{\text{tot}}}{2\pi}.$$

6.4.1 MODELING OF ADVECTION-DIFFUSION PHENOMENA

EUROPLEXUS includes a module for the modeling of advection-diffusion phenomena. This module stems from the TRAFLU-2D and TRAFLU-3D codes developed at JRC Ispra in the late eighties [53, 54].

The module is activated by using elements of type ADQ4 (in 2D) or ADC8 (in 3D), the ADFM material, specific generalised “loads” (see page F.320) and initial conditions (see page E.85), and specific options (see page H.70). Here is a synthetic description of these models, borrowed from [54].

The model uses a quasi-explicit finite element algorithm for the solution of the basic equations describing combined conductive and convective transfer of heat in a liquid. The presence of enclosing solid (rigid) structures is accounted for.

The governing equations in the fluid region are the incompressible Navier-Stokes equations and the thermal energy equation. These equations are treated in an Eulerian frame of reference and they are expressed in terms of primitive variables: velocity, pressure and temperature.

The flow is assumed to be laminar and the fluid Newtonian and incompressible within the Boussinesq approximation. Either the velocity components or the total surface stress are specified as boundary conditions for the Navier-Stokes equations.

The governing equation in the solid is the transient heat conduction equation. Boundary conditions are of prescribed temperature, imposed normal heat flux and heat transfer by convection or radiation.

Spatial discretization is achieved by means of four-node quadrilateral elements in 2D (ADQ4) or eight-node hexahedral elements in 3D (ADC8) with multi-linear velocity and temperature fields. The pressure is assumed uniform over each fluid element.

A fractional step method is employed for time integration of the Navier-Stokes and thermal energy equations. This consists of three distinct steps dealing, respectively, with the advective terms, the viscous/diffusion terms and the pressure/incompressibility terms.

A second-order explicit Taylor-Galerkin method is used in the advection step, where the mass matrix is retained in its consistent form to improve phase accuracy.

A first-order explicit Euler method is used in the viscous-diffusion phase. Here the mass matrix is put into diagonal form.

Finally, a first-order implicit method is used in the pressure phase for the momentum equations. The pressure field itself is obtained as solution of a linear algebraic system arising from the discrete form of the incompressibility condition.

6.4.2 TYPE OF OUTPUT LISTING

Object:

To define the type of printed output listing. If nothing is specified, in extremely large model computations the standard listing could be very large. Therefore, it may be useful to selectively reduce the printed information via the following directives.

Syntax:

```
< $ "LIST"  
  
    | "COOR" ; "ELEM" ; "GIBI" ; "GRIL" ; "EPAI" ; "NORM" ; "NONE" |  
  
    "TERM" $ >
```

COOR

the initial nodal coordinates will be printed on the output listing; furthermore, the principal directions of inertia of COQI element nodes will also be printed

ELEM

the mesh topology (element nodes) will be printed on the output listing

GIBI

the composition of CASTEM2000 objects will be printed on the output listing

GRIL

the characteristics of ALE grid motion will be printed on the output listing

EPAI

the initial element thicknesses will be printed on the output listing

NORM

the FSA and FSR normals will be printed on the output listing

NONE

none of the above quantities will be printed on the output listing

Remarks

By default, i.e. in the absence of the `LIST` directive, all the above quantities are printed out in the normal way. When the `LIST` directive is encountered, all the above printouts are inhibited, i.e. the effect is the same as with `LIST NONE`. Any of the keywords `COORD ... NORM` may then be used to re-activate the printing of selected quantities. In this case, however, printing of sequences of integer numbers occurs in a “compact” way, in the sense that any sequence of four or more **consecutive** numbers n_1, n_2, \dots, n_n is listed simply as ' n_1 to n_n '. In many cases this allows important savings in the quantity of output data.

For example, the directive:

```
LIST NONE ELEM EPAI TERM
```

would print only the mesh topology and element thicknesses.

To obtain the most compact listing, use `LIST NONE TERM`.

Another way of obtaining a compact listing is the option `OPTI NOPR`, see Page H.50. However, that directive does not allow selective printout.

6.5 MPI GLOBAL OPTIONS

Object:

Optional global options to set for MPI calculations:

- SCLM toggles aggressive memory distribution, coming with restrictions upon output and qualification options (**still under strong development**),
- BMPI prevents current dataset to be run without MPI.

Syntax:

```
< "SCLM" <"DTUN"> <"PMET"> <"ROB"> <"CINI">  
    <"WFIL" <ndwfil>> <"DACT" /LECDDL/> <"DPRE" ipre>  
    <"IOPT" iopt> >  
  
< "BMPI">
```

DTUN

Multiple time scales treatment (one per subdomain) is deactivated. Every subdomain has the same time scale (see comment below).

PMET

ParMetis library is used to perform domain decomposition.

ROB

Recursive Orthogonal Bisection algorithm is used to perform domain decomposition (see comment below).

CINI

Automatic domain decomposition with ROB after a restart is performed using initial coordinates instead of current coordinates.

WFIL

Use of an element weight file for automatic domain decomposition (see comment below).

ndwfil

Number of the logical unit of the weight file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is .wgt.

DACT

Selection of active directions (from 1 to 2 in 2D, from 1 to 3 in 3D) for automatic domain decomposition using ROB.

ipre

Number of the first cutting direction for automatic domain decomposition using ROB (see comment below).

iopt

Level of memory optimization (see comment below).

BMPI

When present, the current dataset can only be run using MPI.

Comments:

Keywords to be used with **SCLM** option are very close to the ones dedicated to the **STRUCTURE** directive with automatic domain decomposition activated (keyword **AUTO**, see page I.15). Indeed, to provide an optimized memory distribution, the domain decomposition has to be defined and performed before the global data structure is built and initialized, which is not the case when using the **STRUCTURE** directive just before launching the calculation. Only automatic domain decomposition can be defined this way. See comments on page I.15 for a complete description of keywords **DTUN**, **PMET**, **ROB**, **CINI**, **WFIL**, **DACT**, **DPRE**.

When activating memory optimization for MPI calculations with **SCLM** option, **the STRUCTURE directive must not be used**, since domain decomposition has already been defined.

IOPT keyword is used to define the level of memory optimization: the more aggressive the optimization is, the more restrictions upon output and qualification there are.

- Level 0: all centralized outputs are forbidden (**ECRI** directive), except listing printouts, **ALICE** and **ALICE TEMPS** files (without time splitting), distributed **PVTk** files (**MPI** keyword).
- Level 1: same as level 0, plus immediate qualification **QUAL** directive just after **CALCUL** directive also forbidden.

TIP: The right way to deal with restrictions imposed by **SCLM** option is to write an **ALICE** file during the parallel calculation within a series of time-steps of interest and then to generate the desired output files and perform the desired qualifications from this file.

6.6 DIMENSIONING

Object:

Allocation of memory for the problem variables.

Syntax:

"DIMENSION"

Comments:

The dimensioning of variables data is specified by keywords, which enable the user to reserve for a given problem only the memory that will be really necessary to perform the computation. All the dimensions are maximum values, their value by default is 0 (unless a different value is specified in the description).

On the following pages, the keywords have been classified according to the data they affect. Actually, they can be provided in any order. These keywords together form the directive "DIMENSION".

6.6.1 DIMENSIONS RELATIVE TO GROUP B (GEOMETRY)

These dimensions concern the geometry (elements) and, in ALE computations, the motion of grid nodes.

Overview:

The overall syntax is as follows:

```
< NPOI np    > < NDDL  nd >
< "typ1" n1 "typ2" n2 ... >

< ADAP NPOI np <NIND ni> <NVFI nvfi>
    "typ1" n1 "typ2" n2 ... ENDA >

< NALE nale > < NBLE nble >
< SLPC slpc > < SLPN slpn > < NDVC ndvc >

< NGPZ mxngpz >
```

NODES

Syntax:

```
< "NPOI" np > < "NDDL" nd >
```

np

Maximum number of mesh points. This parameter **is not necessary**, except some special cases. (Cf. comment below).

nd

Total number of degrees of freedom. This parameter **is not necessary**, except some special cases. (Cf. comment below).

Comments:

Normally EUROPLEXUS detects automatically the exact number of nodes (and the number of degrees of freedom), the exact number of eaches element types required, from the input file or from the associated mesh file. Therefore, the directives NPOI, NDDL and TYPi are usually not necessary.

These directives are needed only in special cases, whereby EUROPLEXUS has to create additional nodes (not specified in the input nor in the associated mesh file) after the reading of the geometry: for example a pipeline circuit with a bifurcation, a rigid body, or in case of remeshing.

NUMBER OF ELEMENTS

Object:

The number of the different elements that will be used in the problem is specified (if necessary).

Syntax:

```
| "typ1" n1 "typ2" n2 ..... |
```

typi

name of an element type (see page INT.80).

ni

maximum total number of corresponding elements.

Comments:

Normally EUROPLEXUS detects automatically the exact number of each element type required, from the input file or from the associated mesh file. Therefore, these directives are usually not necessary. These directives are needed only in special cases, whereby EUROPLEXUS has to create additional elements (not specified in the input nor in the associated mesh file) after the reading of the geometry: for example a pipeline circuit with a bifurcation, a rigid body, in case of remeshing, or in case of flying debris.

The various elements are described on page INT 80.

Warning :

If you use 1-D elements (except ED1D), the directives:

```
"TRID" "EULE"
```

are mandatory in the definition of the problem type (page A.30).

ADAPTIVITY (Automatic Mesh Refinement)

Purpose:

This optional sub-directive allows to set the dimensions for the automatic mesh refinement during a computation, as required e.g. in adaptivity. The directive syntax is similar to that described in the previous pages for the “base” mesh. The user must define the maximum number of nodes, of degrees of freedom, and of elements (for each element type which can be refined) that are allowed to be “created” during the transient calculation. This is referred to as the “extension” zone as opposed to the “base” zone containing the base (normal) mesh. The optional directive must be terminated by the keyword **ENDA**.

Syntax:

```
< ADAP
    NPOI np <NIND ni> <NVFI nvfi>
    "typ1" n1 "typ2" n2 ...
    ENDA >
```

np

Maximum number of mesh points in the extension zone.

ni

Total number of indicator variable types used in the adaptive calculation. For example, if one wants to use both displacement and velocity as indicators, then it must be **NIND 2**. By default (i.e. if this keyword is omitted) only one indicator variable is allowed. This quantity is used only in adaptive calculations with the error indicator.

nvfi

Maximum number of cell-centred finite volume (VFCC) interfaces in the extension zone. This quantity is used only in adaptive calculations with VFCC fluids.

n1, n2 ...

Total number of elements of type “typ1”, “typ2” etc. in the extension zone. For the names of the element types see pages INT.80, INT.90 and INT.100.

Comments:

The number of degrees of freedom in the extension memory zone (i.e. the dofs relative to the adaptive nodes) is automatically computed by the code as the number of nodes in the extension zone (**np**) multiplied by the space dimension (2D or 3D). This implies of course that only elements whose nodes do not have any rotational dofs can be used in adaptivity, for the moment.

GRID MOTION (A.L.E.)**Syntax:**

```
< "NALE" nal > < "NBLE" nbl >  
< "SLPC" slpc > < "SLPN" slpn > < "NDVC" ndvc >
```

nale

Maximum number of ALE nodes subjected to manual (i.e., non-automatic) rezoning. To be used only in ALE computations. The nodes are specified by the **GRIL** directive. By default (i.e., if not specified) the code assumes **nale** = 0.

nble

Maximum number of ALE nodes subjected to automatic rezoning. To be used only in ALE computations. The nodes are specified by the **GRIL** directive. By default (i.e., if not specified) the code assumes **nble** = **NPTL**, i.e. the total number of nodes.

slpc

Maximum number of slipping curves

slpn

Maximum number of slipping nodes (total for all slipping curves)

ndvc

Maximum number of neighbour nodes (total) for: a) the rezoning directive **MEAN** or b) a computation with multicomponent fluid elements (**MCxx**) using a second-order scheme. In case a) it can only be used in an ALE computation (See also the **GRIL** directive).

SPACE INTEGRATION FOR SHELL AND BEAM ELEMENTS**Syntax:**

```
< "NGPZ" mxngpz >
```

mxngpz

Maximum number of Gauss Points through the thickness for shell, plate or beam elements which are integrated through the thickness. This value overrides the default value set in INICO1 for each of these element types. This value is global and affects *all* the concerned element types. To set the “true” number of integration points through the thickness for each element (possibly a different value for each element), see the **COMP** directive (Geometrical Complements) on page C.42.

6.6.2 DIMENSIONS RELATIVE TO GROUP C (MATERIALS)

Object :

Dimensions relative to the materials used.

Syntax :

```
< "LMAS" lmas >  
< "ECRO" lecr >  
< "PYRO" mxpyro >
```

lmas

Size of the consistent mass matrix.

lecr

Maximum length of the vector of parameters associated to materials (ECR). This parameter **is not necessary**, except some special cases. (Cf. comment below).

mxpyro

Maximum number of distinct oil pyrolysis bubbles (material FLUT ... PYRO, see page C.530). This material is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC.

Comments :

Normally EUROPLEXUS detects automatically the exact length of the ECR vector associated to materials, from the input file. Therefore, the directive **ECRO** is usually not necessary. This directive is needed only in special cases, whereby EUROPLEXUS has to create additional elements (not specified in the input nor in the associated mesh file) after the reading of the geometry: for example a pipeline circuit with a bifurcation, a rigid body, or in case of remeshing.

It is compulsory to enter the size of the consistent mass matrix, when the computation includes the material "MHOM".

6.6.3 DIMENSIONS RELATIVE TO GROUP D (CONNECTIONS)

Object:

Dimensions relative to the couplings.

Syntax:

```

< "MXLI" maxlie >          < "LNOD" maxnod>      < "LCOF" maxcof >
< "GLIS" nslid nemax >    < "JONC" njonc >
< "NPEF" nmpef      "NPTS" nomax >          < "SOLI" nsol >
< "MECA" nmeca >
< "FSSA" mxfssa >          < "FSSL" mxfssl >      < "FSSF" mxfssf >
< "NBJE" nbjeux >
< "FSA"  mxfsa >          < "IFSA" mxifsa >      < "VCON" mxvcon >

```

maxlie

Total number of connections in the 'LIAISONS'. This parameter **is not necessary**, except some special cases. (Cf. comment below).

maxnod

Total number of nodes involved in the 'LIAISONS'. This parameter **is not necessary**, except some special cases. (Cf. comment below).

maxcof

Total number of coefficients used in the 'LIAISONS'. This parameter **is not necessary**, except some special cases. (Cf. comment below).

nslid

Number of couples of sliding lines.

nemax

Total number of nodes defining these lines (master AND slave).

njonc

Total number of nodes involved in a "TUBM" or "TUYM" type of junction.

nmpef

Number of "particle-structure" couples.

nomax

Total number of nodes defining these "particle-structure" couples.

nsol

Number of rigid solids.

nmeca

Total number of mechanisms.

mxfssa

Maximum number of nodes or element side couples subjected to fluid-structure sliding of the ALE type according to JRC's model (see directive "FSS" "ALE").

mxfssl

Maximum number of nodes or element side couples subjected to fluid-structure sliding of the Lagrangian type according to JRC's model (see directive "FSS" "LAGR").

.

mxfssf

Maximum number of nodes or element side couples subjected to fluid-structure sliding of the fixed type according to JRC's model (see directive "FSS" "FIXE").

nbjeux

Number of couples of nodes to which an impact with gap is associated.

mxfsa

Maximum number of fluid nodes subjected to FSA sliding (see directive "LIAI FSA").

mxifsa

Maximum total number of (element, face) values forming the influence domain of FSA sliding nodes (see directive "LIAI FSA", see page D.155). Each (element, face) couple of values occupies 2 places. This directive is optional, in the sense that the program will try to estimate the required space automatically. However, since in 3D there is no way to estimate the exact value, the program will issue an error message if the dimensioning is not sufficient. It will then be possible to solve the problem by specifying at least the value printed by the program.

mxvcon

Maximum total number of parameters used to define bilateral constraints (CONT SPHE, CYLI, CONE, TORE) with variable coefficients (OPTI CONT VARI). Each sphere requires 3 parameters, each cylinder or cone requires 6 parameters, and each torus requires 9 parameters).

Comment:

Normally EUROPLEXUS detects automatically the exact number of LIAISONS parameters required, from the input file. Therefore, the directives MXLI, LNOD, LCOF, ... **are usually not necessary**. These directives are needed only in special cases, whereby EUROPLEXUS has to create additional nodes or additional elements.

6.6.4 DIMENSIONS RELATIVE TO GROUP F (LOADS)**Object:**

Europlexus no longer needed for the dimension parameters of loads.

6.6.5 DIMENSIONS RELATIVE TO GROUP G (PRINTOUTS)

Object:

Dimensions relative to printout and storage keywords.

Syntax:

```
< "MTTI" mtime >    < "MNTI" mntime >
< "ELVC" mxelvc >
< "NFR0" nfront >    < "NPFR" npfron >
< "NEPE" nepedi >
```

mtime

Maximum number of times for which the printing/storage of the results is requested.

mntime

Maximum number of time steps for which the printing/storage of the results is requested.

mxelvc

Maximum number of neighbour elements (total) for the storage of stresses and hardening quantities for XPLOT. The value of a stress component in a node is estimated, for example, as the mean of the mean values in all neighbour elements (elements that have that node).

nfront

Number of borders ('frontiere') for which the calculation of resultants is requested.

npfron

Total number of nodes involved (by putting all borders together).

nepedi

Length of the memory reserved for the vector NEPEDI which is constructed in subroutine edit1. The code normally computes this automatically.

6.6.6 DIMENSIONS RELATIVE TO GROUP I (CALCULATION)**Object:**

Dimensions relative to the calculation run.

Syntax:

```
< "TTHI" mtthis >
```

mtthis

Maximum number of time values for which the solution has to be computed, in case of "PAS UTIL" option and "CALC" ... "HIST" (the time marching is imposed by the user).

6.6.7 DIMENSIONS RELATIVE TO ADVECTION-DIFFUSION

Object:

Dimensions relative to advection-diffusion problems as declared by keyword "ADDF" above in this section.

Syntax:

```
< "ELSN" mxelsn    "BWDt" mxbwdt    "TPOI" mxtpoi  
  "ELGR" mxelgr    "CVEL" mxcvel    "GRPS" mxgrps >
```

mxelsn

Maximum number of elements surrounding (i.e., connected to) any given node.

mxbwdt

Maximum bandwidth of pressure matrix (for direct solution) or maximum number of elements surrounding an element (for iterative solution).

mxtpoi

Maximum number of time points for prescribed time-dependent 'charges' in advection-diffusion problems (temperatures, heat flux, heat generation, heat convection, heat radiation, external pressure, velocities).

mxelgr

Maximum number of elements in each group with prescribed time-dependent 'charges'.

mxcvel

Maximum number of nodes with constrained velocities.

mxgrps

Maximum number of groups with prescribed time-dependent 'charges'.

6.6.8 END OF DIMENSIONING

Syntax:

"TERM"

Comments:

The word "TERM" marks the end of the dimension, it must appear.

7 GROUP B—MESH AND GRID MOTION

Object:

The following directives enable to define the mesh.

Syntax:

1/ Mesh generated by COCO or in free format:

```
"GEOM" <optional mesh manipulation commands> ... "TERM"  
      ... COCO data or free format data ...
```

2/ Mesh generated by GIBI:

```
"GEOM" <optional mesh manipulation commands>  
      ("nomelm" ('nomobjet') ) "TERM"
```

3/ Mesh generated by I-DEAS:

```
"GEOM" <optional mesh manipulation commands>  
      (... zone declaration list ...) "TERM"
```

Comments:

These directives are described in detail on the following pages.

The **GEOM** directive accepts some simple optional mesh manipulation commands that can be used to scale, shift, etc. the mesh read from an external mesh generator before starting the transient calculation. These commands affect only the nodal coordinates, but not the mesh connectivity. They are described below on page B.15.

7.1 OPTIONAL MESH MANIPULATION COMMANDS

Object:

To manipulate the mesh coordinates read from an external mesh generator before starting the transient calculation. For example, the mesh can be scaled, translated, centred, etc. Note that these commands only affect the (initial) nodal coordinates, they do not affect the elements (i.e. the connectivity).

Syntax:

```
< SCAL $[ FACT fc ;  
          FACX fx ; FACY fy ; <FACZ fz> ]$ >  
< SHIF $[ CENT ;  
          SHIX sx SHIY sy <SHIZ sz> ]$ >
```

SCAL

Scale the coordinates of the mesh to be subsequently read in input either isotropically (i.e. by the same factor **fc** along all axes), or anisotropically.

fc

Isotropic scaling factor.

fx

Scaling factor along the x direction (by default 1.0).

fy

Scaling factor along the y direction (by default 1.0).

fz

Scaling factor along the z direction (by default 1.0).

SHIF

Shift the coordinates of the mesh to be subsequently read in input either in such a way that it is centered around the origin, or by specified amounts in each spatial direction.

CENT

Shift the coordinates of the mesh to be read in input in such a way that it is centered around the origin.

sx

Shift along the x direction (by default 0.0).

sy

Shift along the y direction (by default 0.0).

sz

Shift along the z direction (by default 0.0).

Comments:

The above commands, in particular the scaling commands, can be useful e.g. in case the geometry has been produced by an external mesh generator in some non-standard units. For example, assume the mesh has been generated in millimetres rather than metres. To convert to metres use the command `GEOM SCAL FACT 0.001 ...`

Note that the mesh manipulation occurs immediately after reading the nodal coordinates, so that the coordinates printed on the listing are the corrected ones, not the ones read from the input file.

Note that in case of simultaneous mesh scaling and shifting, the scaling occurs first, then the shifting is applied. Therefore, the shift amounts should be given in the corrected (scaled) mesh units, not in the original mesh units.

7.2 MESH IN COCO-LIKE OR IN FREE FORMAT

7.2.1 GEOMETRY

Object:

To define the logical number on which the geometry (mesh) will be read.

To define an input format, if need be.

To read the coordinates of the nodes and the topology of the elements.

Syntax:

```
"GEOM"  $[          < "LIBR" >   < "POLA" >          ;
           < nl > < '(format1)'  '(format2)' > ]$
```

```
"POIN"  npoin
```

nl

Logical number of input unit where the geometry will be read. By default, nl=5 (15 at JRC).

format1

Reading format of node coordinates. By default format1=6E12.5.

format2

Reading format of the numbers of the nodes composing the elements. By default format2=18I4.

LIBR

The file describing the geometry will be read in free format.

POLA

Nodes are specified by their polar coordinates (by default Cartesian coordinates). In 2D, first enter the radius (R), then the angle (θ) in degrees for each node. In 3D, the third coordinate is interpreted as the elevation Z , thus the coordinate system is cylindrical. The corresponding Cartesian coordinates are computed according to: $x = R \cos(\theta)$, $y = R \sin(\theta)$ and $z = Z$ (3D only).

npoin

Exact number of mesh nodes.

Comments:

If nothing is specified after the word "GEOM", the file describing the geometry is assumed to be in COCO format, it is read from logical unit 5 (15 at JRC) with the formats: format1 and format2.

If the formats are modified, they must be enclosed in parentheses AND in apostrophes.

Example:

```
"GEOM" '(5E20.12)' '(16I5)' "POIN" 123
```

The option "LIBR" is particularly useful for a simple mesh when the user himself prepares the coordinates and the topology.

In the case of polar coordinates, EUROPLEXUS transforms them into Cartesian coordinates for the following computations. If outputs in polar coordinates are desired, see keyword "OPTION".

The number of nodes npoin must not be greater than the number declared for the dimension (page A.40).

7.2.2 ELEMENT ZONES

Object:

Each of the following keywords defines a zone of elements of the given type, that are sequentially numbered.

Syntax:

```
| "typ1" n1 "typ2" n2 ... |  
  
"TERM"  
  
...   COCO data set with its title  
      (title is optional if free format, see "LIBR") ...
```

typi

Name of an element type (see page I.80).

ni

Number of elements in the zone

TERM

Marks the end of the directive GEOMETRY.

Comments:

The various elements are described on page INT.80.

The number of the elements announced for a zone must correspond exactly to the elements defined in the COCO data set.

The same type of element can occupy several zones.

The number of zones must be less than or equal to the one given during the dimensioning (p. A.40).

In the COCO data set, the topology of the elements must be read by zones, and these zones are arranged in the order of their definition in the directive "GEOM".

The word "TERM" is compulsory to indicate the end of the keyword GEOMETRY.

The title appearing before the coordinates of the points is not compulsory when reading in FREE format (see "LIBR").

In order to become acquainted with the keyword "GEOM" the user may have a look at the examples on pages EX 10 and on the following ones.

Warning

There is a mandatory logical order for the 1-D elements (except ED1D). These elements are to be subdivided in 3 groups, which are respectively:

- 1st group: TUBE and TUYA,
- 2nd group: CL1D and CLTU,
- 3rd group: CAVI and BIFU.

Further information allowing to completely define the properties of "CAVI" and "BIFU" elements are given by the "RACCORD" sub-directive of the "COMPLEMENT" directive (page C.80).

The elements of type "BIFU" cause the automatic generation of connections between the concerned d.o.f.s. It is therefore mandatory to list them again in the "LIAISON" directive: see this directive.

The junction elements "CAVI" and "BIFU" must possess the same materials as the neighbour elements of which they ensure the continuity.

7.3 MESH BY MEANS OF GIBI OR CASTEM2000

Object:

To define the objects associated to each element type.

Syntax:

```
"GEOM" ( "nomelm" ( 'nomobjet' ) ) "TERM"
```

nomelm

Name defining the type of element to be taken from the list of available elements.

'nomobjet'

Name of the GIBI object(s)

TERM

End of the directive "GEOM"

Comments:

The names of the available elements may be found on page INT.80, the same keywords are used as in the case of COCO or free-format data (see the chapter ELEMENT ZONES).

The names of the elements cannot be used for other purposes. This explains why the names of the GIBI objects cannot begin with the same first four letters as the name of an element.

Several objects may be associated to a certain type of element. In order to obtain, on the GIBI drawings, the same node and element numbers as on EUROPLEXUS, write in GIBI:

```
"TRAC" ('objet1' "ET" 'objet2' "ET" 'objet3') ...;
```

in the same order as for the instruction "GEOM" of the EUROPLEXUS program.

There is a zone each time a name of an element is specified. Do not forget to sufficiently dimension the number of zones (page A.40).

Warning

The elements with variable number of nodes such as "CAVI" and "BIFU" may be simply generated by GIBI by means of so-called super-elements:

```
my_junction = 'MANU' 'SUPE' pt1 pt2 ... ptn ;
```

It is also possible to use topologically equivalent elements:

```
my_junct_1 = 'MANU' 'POI1' pt1 ;  
my_junct_2 = 'MANU' 'SEG2' pt1 pt2 ;  
my_junct_3 = 'MANU' 'TRI3' pt1 pt2 pt3 ;  
my_junct_4 = 'MANU' 'QUA4' pt1 pt2 pt3 pt4 ;  
my_junct_5 = 'MANU' 'PYR5' pt1 pt2 pt3 pt4 pt5 ;  
etc...
```

However, in order to generate a K2000 file correct for post-treatment, it is necessary that such objects be formed by one single element.

The elements of type "BIFU" cause the automatic generation of connections between the concerned d.o.f.s. It is therefore mandatory to list them again in the "LIAISON" directive: see this directive.

7.4 MESH BY MEANS OF I-DEAS

Object:

To read the coordinates of the nodes and the topology of the elements from an I-DEAS universal file. The elements are declared through a list of keywords defining zones of elements of the given type, that are sequentially numbered.

Syntax:

```
"GEOM" | "typ1" n1 "typ2" n2 ... | "TERM"
```

(same as "Mesh by means of GIBI or CASTEM 2000")

Comments:

Even when reading from an I-DEAS mesh, the topology of the elements has to be read by zones, as defined in the directive **GEOM**. The user has to be sure that the list of zones declared is consistent with the data contained in the I-DEAS universal file used. In order to make this easy, a first run using the **REWR** option (see Page A.30) can be carried out; the informations to be set into the **GEOM** list can be obtained from the table printed on the listing file.

The word **TERM** is compulsory to indicate the end of the keyword **GEOM**.

7.5 GRID MOTION IN AN A.L.E. COMPUTATION

Object:

These keywords enable the user to impose the motion of the mesh under the Arbitrary Lagrangian Eulerian (ALE) formulation. Therefore, this directive can only be used in an ALE computation (see keyword "ALE" on page A.30).

Attention!

If you use any "RACCORDS" 1D ("CAVT" and "BIFU"), the "GRIL" directive must be placed **after** the directive "COMPLEMENT" (see page C.20).

Syntax:

```
"GRILLE"  < "LAGRANGE"  /LECTURE/      >
           < "EULE"      /LECTURE/      >
           < "FS"        /LECTURE/      >
           < "BFIxE"     /LECTURE/      >
           < "GRFS"      /LECTURE/      >

           < "SUIVRE"    ... >
           < "LIGNE"     ... >
           < "CONTOUR"   ... >
           < "PLAN"      ... >
           < "TETR"      ... >
           < "HEXA"      ... >
           < "PRIS"      ... >
           < "PYRA"      ... >
           < "SLIP"      ... >
           < "AUTO"      ... >
           < "MEAN"      ... >
           < "DIRE"      ... >
           < "QUAD"      ... >
           < "SPEC"      ... >
           < "MECA"      ... >
```

LAGRANGE

The following nodes are Lagrangian: the mesh is fixed to material particles.

EULE

List of nodes explicitly declared Eulerian, they are fixed in space but they correspond to different material particles at different times, in general.

FS

The user has to mention all the elements of the fluid-structure type in contact with ALE continuum (fluid) elements. In this case, the keyword "SUIVRE", to ensure the continuity of the mesh, is redundant: the keyword "FS" will do it automatically.

"BFIxE"

The following nodes will be considered as fixed (purely Eulerian). This directive allows thus to specify all the fixed nodes that will serve as base points for manual rezoning options to be entered successively.

"GRFS"

The following elements must be of the CLxx type and their nodes must be geometrically coincident with structural nodes belonging to shell elements. During the calculation, the fluid nodes will be piloted by the corresponding structural nodes like if the "SUIVRE" directive would have been specified. These elements must always be associated to the "IMPE" "GRFS" material.

Comments:

If the motion of a node is not specified, then it is supposed to be Eulerian (fixed mesh).

Several options may be set for the fine-tuning of the automatic rezoning algorithms. For more information, please see the OPTI REZO directive in Section 12.

Warning:

Do not repeat the fluid-structure couplings in the instruction "LIAISON" (except in very specific cases: perforated plates described on page C.330).

Do not forget to dimension sufficiently: "NALE" described on page A.40 (number of A.L.E. or eulerian nodes).

The order in which the different directives appear (LAGRANGE, BFIxE, FS, SUIVRE, CONTOUR...) is important: EUROPLEXUS follows the same order during the remeshing operations.

The following rule holds:

- 1) A node that has to be used as base (master) point for the motion of other points must have a defined motion, else it will be considered as fixed.
- 2) When a point is already used as base point, its motion may no longer be re-defined.

The order of instructions use more often is:

- 1) First, the LAGRANGIAN nodes are defined;
- 2) Then, one passes to a first manual rezoning directive among (SUIVRE, LIGNE, ...) by respecting the following rule: a node may be used as base point only if its motion is already defined previously.

The points defined by this directive may then be used as base points for the following directives.

Restrictions for 1-D problems:

In the presence of bifurcations or cavities (elements "BIFU" and "CAVI"), their junctions must be defined BEFORE specifying grid motions. For further details see the directive "COMPLEMENT", sub-directive "RACCORD".

It is useless to define a grid motion for elements "TUYA". In fact, they result from the assembly of an element of type "POUT" and one of type "TUBE", and the "grid" for the internal fluid is nothing but the set of nodes that define the tube walls. EUROPLEXUS automatically ensures their motion.

In the case of "TUBE" elements, as they have just one d.o.f., this will necessarily be the first one, and only the first component will be driven. Therefore, make sure that these elements are parallel to the Ox axis.

Furthermore, for "TUBE" elements for which the length must change, one should first make sure that the orientation of the global Ox axis and the local orientations of the nodes belonging to the "TUBE" elements to be driven are coherent. In fact the velocity computed by EUROPLEXUS in these nodes has a sign which depends from this orientation.

7.5.1 AUXILIARY FILE

Object :

This directive allows to read the grid remeshing data from an auxiliary file.

Syntax :

```
"GRILLE"      < "FICHIER"      'nom.fic'  >
```

In certain cases these data may be bulky. Then it is advisable to store them on an auxiliary file in order to shorten the main data file. The auxiliary file is activated by means of the directive "FICHIER", followed by the name (complete under Unix) of the file. Then, in the main data file remains only the keyword "GRILLE", followed by "FICHIER".

The auxiliary file (in free format) will contain all grid rezoning data, except the "GRILLE" keyword. In order to return to reading from the main input file, the auxiliary file must terminate by the keyword "RETOUR".

7.5.2 “SUIVRE”

Object:

To force one or more A.L.E. mesh nodes to follow the motion of a "base" node.

Syntax:

```
"SUIVRE"      "BASE"    /LECTURE/  
              "LIST"    /LECTURE/
```

```
BASE /LECTURE/
```

Number of the "base" node to be followed.

```
LIST /LECTURE/
```

Numbers of the A.L.E. nodes with an imposed motion.

Comments:

The particle which is present at the node is changing all the time. This instruction is therefore very different from imposing a node to be Lagrangian.

Warning:

Please read the rule for defining the base points, page B.60.

7.5.3 “LIGNE”

Object:

To impose the motion of several nodes so that they remain aligned between two “base” nodes . The initial subdivision is maintained: the segments remain in the same relation.

Syntax:

```
"LIGNE"      "BASE"    /LECTURE/  
              "LIST"    /LECTURE/
```

"BASE" /LECTURE/

Numbers of the 2 base nodes which will impose the motion.

"LIST" /LECTURE/

Numbers of A.L.E. nodes lying between the two preceeding points and following the motion. The list can safely include also the two base nodes: if present, they are automatically discarded from the list.

Comments:

It is possible to have roughly aligned points, but if the basic points are very distant from each other, the computation tends to realign the points (and vice versa).

Warning:

Please read the rule for defining the base points, page B.60.

7.5.4 “PLAN”

Object:

To impose a homeomorphic motion to several nodes of the grid describing a triangle or a quadrangle. This command is available both in 2D and in 3D. In the 3D case, all slave points should lie at least approximately on the plane defined by the triangle or quadrangle.

Syntax:

```
"PLAN"      "BASE"    /LECTURE/  
            "LIST"    /LECTURE/
```

```
"BASE" /LECTURE/
```

Numbers of the base points composing a triangle (3 points) or a quadrangle (4 points).

```
"LIST" /LECTURE/
```

Numbers of A.L.E. points submitted to homeomorphic motion. These nodes must be located inside the basic triangle or quadrangle, or on their boundaries. The list can safely include also the base nodes: if present, they are automatically discarded from the list.

Comments:

It is strongly recommended to use quadrangles. Triangles are only useful if the initial mesh already has a triangular shape.

Warning:

Please read the rule for defining the base points, page B.60.

7.5.5 “TETR”

Object:

To impose a homeomorphic motion to several nodes of the grid describing a tetrahedron. This command is available only in 3D.

Syntax:

```
"TETR"      "BASE"    /LECTURE/  
            "LIST"    /LECTURE/
```

```
"BASE" /LECTURE/
```

Numbers of the 4 (usually Lagrangian) base points defining the tetrahedron. These points should not be coplanar.

```
"LIST" /LECTURE/
```

Numbers of A.L.E. points submitted to homeomorphic motion. These nodes must be located inside the basic tetrahedron or along its boundaries. The list can safely include also the base nodes: if present, they are automatically discarded from the list.

Warning:

Please read the rule for defining the base points, page B.60.

7.5.6 “HEXA”

Object:

To impose a homeomorphic motion to several nodes of the grid describing a hexahedron. This command is available only in 3D.

Syntax:

```
"HEXA"      "BASE"    /LECTURE/  
              "LIST"   /LECTURE/
```

```
"BASE" /LECTURE/
```

Numbers of the 8 (usually Lagrangian) base points defining the hexahedron.

```
"LIST" /LECTURE/
```

Numbers of A.L.E. points submitted to homeomorphic motion. These nodes must be located inside the basic hexahedron or along its boundaries. The list can safely include also the base nodes: if present, they are automatically discarded from the list.

Warning:

Please read the rule for defining the base points, page B.60.

7.5.7 “PRIS”

Object:

To impose a homeomorphic motion to several nodes of the grid describing a prism. This command is available only in 3D.

Syntax:

```
"PRIS"      "BASE"    /LECTURE/  
            "LIST"    /LECTURE/
```

```
"BASE" /LECTURE/
```

Numbers of the 6 (usually Lagrangian) base points defining the tetrahedron.

```
"LIST" /LECTURE/
```

Numbers of A.L.E. points submitted to homeomorphic motion. These nodes must be located inside the basic prism or along its boundaries. The list can safely include also the base nodes: if present, they are automatically discarded from the list.

Warning:

Please read the rule for defining the base points, page B.60.

7.5.8 “PYRA”

Object:

To impose a homeomorphic motion to several nodes of the grid describing a pyramid. This command is available only in 3D.

Syntax:

```
"PYRA"      "BASE"    /LECTURE/  
              "LIST"    /LECTURE/
```

```
"BASE" /LECTURE/
```

Numbers of the 6 (usually Lagrangian) base points defining the pyramid.

```
"LIST" /LECTURE/
```

Numbers of A.L.E. points submitted to homeomorphic motion. These nodes must be located inside the basic pyramid or along its boundaries. The list can safely include also the base nodes: if present, they are automatically discarded from the list.

Warning:

Please read the rule for defining the base points, page B.60.

7.5.9 “CONTOUR”

Object:

To impose a homeomorphic motion to the grid nodes inside a given bounded area.

Syntax:

```
"CONT"      "BASE"    /LECTURE/  
              "LIST"    /LECTURE/
```

```
"BASE" /LECTURE/
```

Numbers of the base nodes defining the boundary.

```
"LIST" /LECTURE/
```

Numbers of the A.L.E. nodes submitted to the homeomorphic motion. These nodes have to be located inside the bounded area, they may not be on the boundary. The list can safely include also the base nodes: if present, they are automatically discarded from the list.

Comments:

It is recommended to use the facilities offered in GIBI by the keywords "CONTOUR" and "ENVELOPPE".

For ease of use, EUROPLEXUS accepts that among the points defined by "LIST" there be also the base points. These points will be then removed by a special treatment within the code.

Otherwise, one can also separate the internal points from those on the contour, as shown in the following example.

Given an object "LIQ", the user wants to distinguish its internal nodes (ALE) from the nodes defining the outline of the surface (Lagrangian).

In GIBI:

```
TLIQ = LIQ changer POI1 ;  
CLIQ = contour LIQ      ;  
CLIQ = CLIQ changer POI1 ;  
ILIQ = TLIQ differ CLIQ ;
```

In EUROPLEXUS:

CONTOUR BASE LECTURE CLIQ TERM
LIST LECTURE ILIQ TERM

7.5.10 "SLIP"

Object:

Define 2D curves consisting of nodes that are allowed to slip tangentially to the curve itself. This only applies to ALE models in structures or fluids.

Syntax:

```
"SLIP"      |[ "NORM" /LECTURE/  ;  
              "EQUI" /LECTURE/ ]|
```

"SLIP"

The following nodes belong to a curve (in 2D) that is Lagrangian in the normal direction but ALE in the tangential direction. Examples are free surfaces in fluids, free boundaries in solids treated by the ALE method for structures, or interfaces between different materials that should not be mixed. Sliding in the tangential direction can be of two types: a) no sliding (only normal motion) or b) slide tangentially so as to keep nodes nearly equidistant.

"NORM" /LECTURE/

The specified nodes will only move along the normal direction to the curve.

"EQUI" /LECTURE/

The specified nodes will move both along the normal and along the tangential direction, so as to remain nearly equidistant from each other.

Comments

The nodes have to be listed in /LECTURE/ in the order in which they appear along the curve, and by leaving the body on the left side (for free surfaces).

Each list must include an initial and a final node, not subject to the imposed "SLIP" motion, whose positions are used to evaluate the normal direction for each triple of nodes. Therefore, each /LECTURE/ must contain at least three numbers.

Restrictions

The algorithms implemented here are only valid for "nearly straight" curves, in the sense that the angle between successive segments of the curve (element sides) must be close to 180 degrees. If this is not the case, then the mesh should be refined locally.

Warning

Do not forget to dimension adequately "SLPC" and "SLPN" (see Page A.40).

7.5.11 “AUTO”

Object:

Use Giuliani’s (automatic) rezoning algorithm to determine the motion of the nodes specified next.

Syntax:

```
"AUTO"      | [ "AUTRES"                ;
                "NOEUDS" /LECTURE/ ] |
```

"AUTRES"

All the ‘remaining’ A.L.E. nodes (i.e. those which have not been forced by a ‘manual’ command such as SUIVRE, LIGNE, ...) will be automatically rezoned.

"NOEUDS" /LECTURE/

Allows to list explicitly the nodes which have to be automatically rezoned.

Comments:

Use of the automatic rezoning technique is encouraged when tackling a new problem or in cases when the node pattern and the deformation process cannot be described by simple laws such as those provided by the ‘manual’ rezoning commands. However, compatibility is ensured so that manual commands can still be used in conjunction with the automatic option in case some nodes (usually few) are not properly treated by the automatic technique.

The algorithm starts by estimating node by node, and on the basis of purely geometric criteria, the best grid velocity W that would bring to an optimal rezoning in just one step.

Then, this velocity is projected onto the fluid velocity V at the node concerned: this in order to take automatically into account possible boundary conditions imposed at the node.

Finally, the resulting module is limited in order to avoid too high remeshing velocities. This limitation is done by a coefficient γ_0 , i.e :

$$-\gamma_0 V < W < (1 + \gamma_0) V$$

The coefficient γ_0 may be defined by the option OPTI REZO GAM0, see page H.150. This parameter does not have a large influence on remeshing, but in any case with small values of γ_0

one should get a slightly more effective remeshing. Suggested values are between 0.1 and 0.8, the default is 0.2.

Note that **GAM0** is a global parameter, and hence it is the same for the whole mesh. Consequently, it is recommended to "help" the remeshing algorithm, in case of need, by the manual remeshing directives such as **SUIVRE**, **LIGNE**, etc.

Note that ALE nodes lying on fluid-structure sliding lines of the ALE type (see directive **FSS ALE**) have to be declared as automatically rezoned. The program then automatically applies the correct sliding conditions.

The list of nodes can be given either explicitly, by a **/LECTURE/**, introduced by keyword **NOEU**, or implicitly, by keyword **AUTR**. In the latter case, the nodes considered are all nodes that have not been assigned any rezoning method up to the current point in the input file. The **GAM0** parameter should be specified in the **OPTI** directive, since it applies not only to Giuliani's but also to the other automatic remeshing methods (in fact, it is used in the mesh velocity restriction algorithm).

Several options may be set for the fine-tuning of the automatic rezoning algorithms. For more information, please see the **OPTI REZO** directive in Section 12.

Warning:

To date, the automatic rezoning facility is only implemented in 2D for nodes belonging to elements of type **TRIA**, **CAR1**, **CAR4**, **FLU1**, **FL23**, **FL24**, **Q41**, **Q41N**, **Q42**, **Q42N**, **TRIA**, **CVL1**, **TVL1**, **MC23**, **MC24**. In 3D, it is implemented for nodes belonging to elements of type **FLU3**, **FL34**, **FL35**, **FL36**, **FL38**, **TETR**, **PRIS**, **CUBE**.

7.5.12 "MEAN"

Object:

Use the mean algorithm to determine the motion of the nodes specified next. This rezoning method is available for all ALE element types.

Syntax:

```
"MEAN"      $[ "NOEU" /LECTURE/  ;  
              "AUTR"              ]$
```

"NOEU"

The following (ALE) nodes will be rezoned by the "mean position" algorithm. The position of each node will tend to become the mean of the position of neighbour nodes. For a generic node I, neighbour nodes are those connected to it in the mesh by a straight (two-noded) element side.

"AUTR"

All other 'remaining' ALE nodes, i.e. those that have not been forced to move by a 'manual' command such as "SUIVRE", "LIGNE", etc., will be rezoned by the "MEAN" algorithm.

Comments:

Several options may be set for the fine-tuning of the automatic rezoning algorithms. For more information, please see the OPTI REZO directive in Section 12.

Warning

Do not forget to dimension adequately for the neighbouring nodes table, see "NDVC" on page A.60. (see Page A.40).

7.5.13 “DIRE”

Object:

Use the direct algorithm to determine the motion of the nodes specified next. Note, however, that this algorithm is experimental and is currently implemented only for 2D quadrilateral ALE finite elements and finite volumes.

Syntax:

```
"DIRE"      $[ "NOEU" /LECTURE/  ;  
              "AUTR"              ]$
```

"NOEU"

The following (ALE) nodes will be rezoned by the "direct" algorithm.

"AUTR"

All other 'remaining' ALE nodes, i.e. those that have not been forced to move by a 'manual' command such as "SUIVRE", "LIGNE", etc., will be rezoned by the "DIRE" algorithm.

Comments:

Several options may be set for the fine-tuning of the automatic rezoning algorithms. For more information, please see the OPTI REZO directive in Section 12.

Warning

This algorithm is experimental and is currently implemented only for 2D quadrilateral ALE finite elements and finite volumes.

7.5.14 “QUAD”

Object:

Use the specific quadrilateral algorithm to determine the motion of the nodes specified next. Note, however, that this algorithm is experimental and is currently implemented only for 2D quadrilateral ALE finite elements and finite volumes.

Syntax:

```
"QUAD"      $[ "NOEU" /LECTURE/  ;  
              "AUTR"              ]$
```

"NOEU"

The following (ALE) nodes will be rezoned by the "quadrilateral" algorithm.

"AUTR"

All other 'remaining' ALE nodes, i.e. those that have not been forced to move by a 'manual' command such as "SUIVRE", "LIGNE", etc., will be rezoned by the "QUAD" algorithm.

Comments:

Several options may be set for the fine-tuning of the automatic rezoning algorithms. For more information, please see the OPTI REZO directive in Section 12.

Warning

This algorithm is experimental and is currently implemented only for 2D quadrilateral ALE finite elements and finite volumes.

7.5.15 “SPEC”

Object:

Use the element-specific algorithm to determine the motion of the nodes specified next. Note, however, that this algorithm is experimental and is currently implemented only for 2D quadrilateral and triangular ALE finite elements and finite volumes.

Syntax:

```
"SPEC"    $[ "NOEU" /LECTURE/  ;  
             "AUTR"              ]$
```

"NOEU"

The following (ALE) nodes will be rezoned by the "specific" algorithm.

"AUTR"

All other 'remaining' ALE nodes, i.e. those that have not been forced to move by a 'manual' command such as "SUIVRE", "LIGNE", etc., will be rezoned by the "SPEC" algorithm.

Comments:

Several options may be set for the fine-tuning of the automatic rezoning algorithms. For more information, please see the OPTI REZO directive in Section 12.

Warning

This algorithm is experimental and is currently implemented only for 2D quadrilateral and triangular ALE finite elements and finite volumes.

7.5.16 “MECA”

Object:

Use the mechanical algorithm to determine the motion of the nodes specified next.

Syntax:

```
"MECA"      $[ "NOEU" /LECTURE/  ;  
              "AUTR"              ]$
```

"NOEU"

The following (ALE) nodes will be rezoned by the "mechanical" algorithm.

"AUTR"

All other 'remaining' ALE nodes, i.e. those that have not been forced to move by a 'manual' command such as "SUIVRE", "LIGNE", etc., will be rezoned by the "MECA" algorithm.

Comments:

Several options may be set for the fine-tuning of the automatic rezoning algorithms. For more information, please see the OPTI REZO directive in Section 12.

Warning

This algorithm is experimental and is currently implemented only for 2D quadrilateral ALE finite elements and finite volumes. Finally, in order to use this model the problem type keyword MECA must be specified in the initial part of the input file (see Section 4.2).

7.5.17 “ELAS”**Object:**

Affects a fictitious elastic material to grid elements to control the motion of the ALE nodes.

Syntax:

```
"ELAS"  "RO" ro "YOUNG" young "NU" nu "DAMP" damp /LECTURE/  
"ro"  
    Fictitious material's density.  
"young"  
    Fictitious material's Young modulus.  
"nu"  
    Fictitious material's Poisson ratio.  
"damp"  
    Inertial damping.  
/LECT/  
    List of the fluid elements concerned.
```

Comments:

At the moment, this type of rezoning is available for the following element types: TRIA, CAR1, TETR, PRIS, CUBE, T3VF, Q4VF, TEVF, PRVF, CUVF, FL23, FL24, FL34, FL36, FL38. Note that the model is not yet available for the pyramid elements (FL35 for example).

Be aware that a critical time step is computed for the explicit elastic rezoning problem to remain stable. A soft fictitious material should be used so that this time step is not smaller than the "physical" critical time step.

An large inertial damping coefficient (i.e. from 1.E3 to 1.E4) should be used to prevent vibrating oscillations of the deformed grid.

Do not forget to set option OPTI REZO LIAI (see Page H.150, Section [14.14](#)), so that nodes subjected to kinematic links are rezoned accordingly.

7.5.18 USER'S ROUTINE "COOGRI"**Object:**

This routine can be written by the user and exploited in order to specify the motion of fluid nodes when using the ALE description. Its use should be only needed in exceptional cases, because normally the automatic and manual rezoning directives are perfectly appropriate.

Since this routine is called last by routine NVCOOR any motion specified in it will overwrite any other motion, either automatic or by means of manual rezoning directives, specified by the user.

Normally, the routine does nothing.

The listing of the sample routine is included hereafter.

```

      SUBROUTINE COOGRI(V,WG,postp,mvgril)
C-----
C    ---  ROUTINE UTILISATEUR ( vitesses DE LA GRILLE )
C-----
c v      : fluid velocities
c wg      : mesh grid velocities
c postp   : pointer in both v and wg
C MVGRIL(I,1)
c          -1=LAGRANGIAN
c           0=EULERIAN
C          1=A.L.E.,AUTOMATICALLY REZONED (JRC);
C          2=A.L.E.,MANUALLY REZONED (CEA),
c          3=A.L.E, rezoned by FSS ALE (ALE sliding JRC)
c          4=A.L.E, "MEAN" rezoned (JRC)
C MVGRIL(I,2)  NODAL INDEXES IN THE FOLLOWING ORDER :
C               - FIRST LAGRANGIAN NODES (GROWING ORDER)
C               - THEN NON-LAGRANGIAN BASE NODES (USED AS MASTER
C                 NODES FOR MOTION OF SLAVE A.L.E. NODES)
C               - FINALLY ALL OTHER NODES
C MVGRIL(I,3)  IAD (ADDRESS IN <NBALE> AND <CBALE>) IF MVGRIL(I,1)=2
c              -1  IF NODE IS SUBJECT TO "LIAISON" AND MVGRIL(I,1)=1
C              0   IN ALL OTHER CASES
C-----
c
      implicit none
      include 'CONTRO.INC'
C
      double precision V(*), WG(*)
      integer postp, mvgril

```

```
C
      dimension posp(*), mvgril(*)
c
c Insert hereafter the user's definition of the appropriate
c grid velocities:
c
C
      RETURN
      END
```

7.6 MESH REFINEMENT FOR WAVEFRONT TRACKING

Object:

This directive enables the user to impose the refinement of the computational mesh grid to follow the propagation of one or more prescribed wave fronts. It should be used in conjunction with mesh adaptivity dimensioning directive **ADAP**, see page A.62. However, note that this directive is incompatible with “true” adaptivity (piloted by an error indicator) which is activated by the **ADAP** directive of page B.210.

Note that this model does not represent a true implementation of adaptivity, since the mesh refinement and de-refinement is entirely piloted by the user with the present directive. However, it may be useful to check the mesh refinement and de-refinement processes in simple test cases, where the propagation of wave fronts is known a priori.

For a true implementation of adaptivity (piloted by an error indicator) see the **ADAP** directive on page B.210.

Syntax:

```
WAVE  nwav * ($ SPHE ; PLAN ; CYLI $
              X x Y y <Z z> <NX nx NY ny <NZ nz>>
              T0 t0 <T1 t1> C c MAXL m H1 h1 H2 h2)
```

WAVE

Prescribe one or more wave fronts to be tracked.

nwav

Number of wave fronts to be defined.

SPHE

The wave being defined is a spherical wave.

PLAN

The wave being defined is a plane wave.

CYLI

The wave being defined is a cylindrical wave.

x

X-coordinate of the wave source point.

y

Y-coordinate of the wave source point.

z

Z-coordinate of the wave source point (0 by default).

nx

X-component of the normal vector to the plane (for **PLAN** waves). X-component of the cylinder axis vector (for **CYLI** waves). Note that the vector need not be normalized to unit lenght.

ny

Y-component of the normal vector to the plane (for **PLAN** waves). Y-component of the cylinder axis vector (for **CYLI** waves).

nz

Z-component of the normal vector to the plane (for **PLAN** waves). Z-component of the cylinder axis vector (for **CYLI** waves). This is 0 by default.

t0

Time instant at which wave propagation starts from the corresponding source point.

t1

Time instant at which wave propagation terminates. The mesh is completely un-refined (thus returning to the base mesh) at times greater than this value.

c

Propagation speed of the wavefront. For spherical waves, propagation is assumed isotropic in all space directions.

m

Maximum level of mesh refinement at the wave front.

h1

Thickness of maximum refined mesh layer normally to the wave front.

h2

Thickness of refined mesh layer normally to the wave front. Refinement passes from level **m** to level 0 (no refinement) linearly when passing from a distance **h1** to a distance **h2** from the wave front.

7.7 ADAPTIVITY

Object:

This directive enables the user to impose the refinement or un-refinement of the computational mesh grid (**adaptivity**) in accordance to some chosen **error indicator**. It should be used in conjunction with mesh adaptivity dimensioning directive **ADAP**, see page A.62. However, note that this directive is incompatible with the wave front tracking directive **WAVE** of page B.200.

In contrast to the **WAVE** directive of page B.200, the present model represents a “true” implementation of adaptivity.

Syntax:

```
ADAP  INDI | DEPL ; VITE ; PRES ; DENS ; CONT icon ; ECRO iecr |
      < TYPE | CURV ; GRAD | >
      STRA $ PERR perr ; PELE pele ALFA alfa ; PEMA pema $
      CERR ( cerr /LECT/ )
```

ADAP

Activates true adaptivity according to the error indicator(s) chosen next.

INDI

Introduces the variable(s) used as error indicator(s). Note that exactly **ni** variables must be specified next, where **ni** is the number given in the dimensioning (**NIND ni**), see page A.62. If the keyword **NIND** has been omitted in the dimensioning, then by default **ni** = 1.

DEPL

Use nodal displacement (norm) as error indicator.

VITE

Use nodal velocity (norm) as error indicator.

PRES

Use (fluid) element pressure as error indicator. Only **GRAD** type of indicator can be used in this case.

DENS

Use (fluid) element density as error indicator. Only **GRAD** type of indicator can be used in this case.

CONT icon

Use element stress component **icon** as error indicator. Only **GRAD** type of indicator can be used in this case.

ECRO iecr

Use element hardening component **iecr** as error indicator. Only **GRAD** type of indicator can be used in this case.

TYPE

Introduces the types of error indicator(s). Note that exactly **ni** variables must be specified next, where **ni** is the number given in the dimensioning (**NIND ni**), see page A.62. The types must be entered in the same order as the indicator variables, i.e. the first type corresponds to the first indicator variable, and so on. If the keyword **NIND** has been omitted in the dimensioning, then by default **ni** = 1. If the **TYPE** sub-directive is omitted, all indicators are assumed to be of the curvature type.

CURV

Error indicator is of curvature type, i.e. the curvature of the indicator variable is used to compute the error indicator. This type of error indicator can be used only for the node-based indicator variables listed above (i.e. only for **DEPL** or **VITE**).

GRAD

Error indicator is of gradient type, i.e. the gradient of the indicator variable is used to compute the error indicator. This type of error indicator can be used for all indicator variables listed above.

STRA

Introduces the strategy used for the error indicator. At the moment, two strategies are available: prescribing the error or prescribing (approximately) the number of used elements.

PERR

Introduces the prescribed error **perr** ($\tilde{\epsilon}$). This is then used to compute the prescribed element size \tilde{h}_k , see formula below.

PELE

Introduces the prescribed number of adaptive elements **pele** (\tilde{n}).

ALFA

Coefficient α used in the formula to estimate the predicted number of elements in memory (see below). This is an empirical value. The suggested value is 4 for 2D calculations.

PEMA

Introduces the prescribed number of adaptive elements **pema** (\tilde{n}). This version of the command is suited for use in conjunction with the **OPTI ADAP MAXL** option, see page H.180. In fact when this option is specified, the **PELE** strategy respects very badly the prescribed number of elements. Note that the **PEMA** strategy requires additional calculations with respect to **PELE** and is therefore more expensive. Note also that the **PEMA** strategy makes no use of the **ALFA** coefficient.

CERR

Introduces the choice of the constant C (**cerr**) appearing in the expression of the error indicator (see below). The value may vary from element to element (e.g., due to different element types). Each (parent) element in the adaptive mesh must receive a value. Descendent elements inherit the value from their own parent element.

/LECT/

List of the elements to which the value **cerr** is assigned.

Comments:

For a **curvature-based indicator**, the expression used to compute the error indicator is:

$$|e| \approx Ch^2 \max(|k_1|, |k_2|)$$

where C is the constant **cerr** given in input for the current element, h is the local mesh size (i.e. the characteristic length of the element under consideration), k_1 and k_2 are the principal curvatures of the variable chosen as error indicator on a patch composed by the element itself and by all its direct neighbors.

For a **gradient-based indicator**, the expression used to compute the error indicator is:

$$|e| \approx Ch ||G||$$

where C is the constant **cerr** given in input for the current element, h is the local mesh size (i.e. the characteristic length of the element under consideration), $||G||$ is the norm of the gradient of the variable chosen as error indicator on a patch composed by the element itself and by all its direct neighbors.

The expression used to compute the prescribed element size \tilde{h}_k is:

$$\tilde{h}_k = \sqrt{\frac{\tilde{e}}{e_k}} h_k, \quad k = 1, \dots, N$$

where N is the current number of active elements in the mesh, and e_k is the estimated error in the k -th element.

The expression used to estimate the number of elements \tilde{n} in memory is:

$$\tilde{n} \approx \frac{\alpha}{\tilde{e}} \sum_{k=1}^N e_k$$

This formula is actually used to compute \tilde{e} :

$$\tilde{e} \approx \frac{\alpha}{\tilde{n}} \sum_{k=1}^N e_k$$

and then the above formula gives \tilde{h}_k .

8 GROUP C—GEOMETRIC COMPLEMENTS

Object:

These directives enable the user to complete the geometry.

Syntax:

"COMPLEMENT"

Comments:

These directives are described in detail on the following pages.

Do not forget the corresponding dimensioning (page A.70).

8.1 ADDITIONAL MESH-RELATED DATA

Object:

This directive enables the user to input complementary geometric (or mesh-related) data.

Syntax:

"COMPLEMENT"

Comments:

The keyword "COMPLEMENT" and the associated data are compulsory only if required by the elements (shells, beams and 1-D elements) or if the user enters added masses.

8.1.1 AUXILIARY FILE

Object :

This directive allows to read complementary data from an auxiliary file.

Syntax :

```
< "FICHIER"      'nom.fic'  >
```

In certain cases the data may be bulky. It is then advisable to store the data on an auxiliary file in order to shorten the main input data file. The auxiliary file is activated by the keyword "FICHIER" that precedes the full name of the file (under Unix). Then, only the keyword "COMPLEMENT" preceding the keyword "FICHIER" remains in the main input file.

The auxiliary file (in free format) will contain the whole set of geometry complement data, with the exception of the keyword "COMPLEMENT" itself. To resume reading from the main input data file, the auxiliary file must be terminated by the keyword "RETOUR".

8.1.2 ADDED MASSES

Object:

This instruction enables the user to define the masses which are added to certain nodes, along certain degrees of freedom.

Syntax:

```
< "MASS" ( /LECDDL/ xm /LECTURE/ ) >
```

LECDDL

List of the degrees of freedom concerned.

xm

Value of the added mass.

LECTURE

List of the nodes concerned.

Comments:

1/ Several added masses may be defined without repeating the key-word "MASS".

Example:

```
"MASS" /LECDDL/ xm1 /LECTURE/  
        /LECDDL/ xm2 /LECTURE/  
        . . .
```

2/ Use:

```
"MASS" 1342 xm "SUIT" 1 2 3 "TERM"
```

The degrees of freedom 1,3,4,2 of the nodes 1,2,3 are modified by the mass xm (this mass is added to the initial one).

In axisymmetric cases, do not forget to divide the "true" mass by 2π .

Remark:

Material points ("PMAT" elements) may be used too, in order to enter added masses (see page C.200).

8.1.3 THICKNESS OR SECTION

Object:

1/ Thickness:

By means of this directive the user specifies the thickness of two- and three-dimensional shells. A thickness must also be specified for elements of types Q92, Q93, Q92A, ED01, ED41, COQI, Q41, Q41N, Q42, Q42N, Q41L, Q42L, Q95, CQD4, CQD9, CQD3, CQD6, FUN2, FUN3. The directive is mandatory if the mesh contains any of these elements.

2/ Section:

This directive is similar to the preceeding one, but it is only applied to beams and bars.

Syntax:

```
< |[ "EPAI" ( ep /LECTURE/ ) ;  
      "EPAI" ( "CQDX" /LCHP/ /LECTURE/ ) ]| >
```

Or:

```
< "SECT" ( ep /LECTURE/ ) >
```

ep

Thickness or section.

LECTURE

List of the elements concerned.

Comments:

Various thicknesses can be defined for different elements without repeating the key-word "EPAI". It is the same for "SECT".

Example:

```
"EPAI" ep1 /LECTURE/  
      ep2 /LECTURE/  
      ep3 /LECTURE/
```

A special syntax is foreseen to define the thickness of degenerated shell elements CQDx. For these elements, the thickness should be defined at the nodes. The most accurate way of doing this is to prepare a 'champoint' object with CASTEM2000 and store it together with the mesh in the CASTEM2000 file (see directive "SAUV" in CASTEM2000). This file is then read by EUROPLEXUS using the directive "CASTEM" (see page A.30) and can then be referred to from other directives.

The keyword "CQDX" introduces this kind of syntax: the reference to the CASTEM2000 champoint object is read by the /LCHP/ procedure (see page INT.57) and the associated geometrical support (object of type 'maillage' containing the shell elements) is indicated by the following /LECT/. Note that /LCHP/ and /LECT/ must be given in this order.

A simpler, but not as precise, way of specifying the thicknesses of these shells is to assign a single value to each element by the standard "EPAI" directive, without using the CASTEM2000 objects for this purpose. In this case, the program itself estimates values for the thicknesses (and fiber orientations) at each node based on the values of the surrounding elements.

8.1.4 GEOMETRICAL PARAMETERS FOR SHELL ELEMENTS

Object:

This directive allows to choose some geometrical parameters for shell, plate and beam elements. For example, the type of spatial integration through the thickness or in the lamina directions, for certain types of shell/plate elements.

Syntax:

```
<"NGPZ"  ngpz  /LECT/ > <"INTE"  typ1  /LECT/>
<"ALPH"  alpha /LECT/ > <"BETA"  beta  /LECT/>
<"SK"    sk    /LECT/ > <"REFE"  refe /LECT/>
```

ngpz

Number of gauss points in the thickness for shell, plate or beam elements. This value must not exceed the maximum value specified in the dimensioning (see DIME ... NGPZ on page A.66). The default value for the CQDx elements is 3.

typ1

Type of lamina integration for 3D degenerated shell elements (CQD3, CQD4, CQD6, CQD9), **SELE** means selective reduced, **REDU** means reduced, **FULL** means full, **SELM** means selectively reduced with ‘mean tau’ procedure and **FULM** means full with ‘mean tau’ procedure; the default is reduced.

alpha

Participation to bending (only for some shell elements). Default is 2/3. This parameter is only used by elements which adopt a global model: elements integrated through the thickness ignore the value of this parameter.

beta

Participation to membrane (only for some shell elements). Default is 1. This parameter is only used by elements which adopt a global model: elements integrated through the thickness ignore the value of this parameter.

sk

Shear correction factor for 3D degenerated shell elements (CQD3, CQD4, CQD6, CQD9), default value is 5/6.

refe

Location of the reference surface. `refe = -1, 0, +1` indicates that the surface is located at the bottom, middle, and top surface of the shell, respectively. The shell element is moved in the positive direction of the element normal.

`/LECT/`

List of the concerned elements.

Comments:

The number of gauss points through the thickness for a sandwich element is defined by `COMP SAND NGPZ`, see Page C.45. Therefore, an additional `COMP NGPZ` for the *same* element should be avoided.

The ‘mean tau’ procedure may be applied to CQD3, CQD4, CQD6, CQD9 degenerated shell elements. However, it simply sets the transverse shear values to a mean value, not to a linearly variable pattern. This is likely to be too simplistic for the 9-node element CQD9 (and also for the 6-node CQD6).

The parameter `alpha` is used to modify the bending coefficient for the global shell models. The program will use the following criterion:

$$\text{sig*} = \text{SQRT} (\text{sigm} ** 2 + (\text{alph} * \text{sigf}) ** 2)$$

In this formula, `sig*`, `sigm` and `sigf` represent the Von Mises equivalent stress, the membrane stress and the bending stress, respectively. By default, $\alpha = 0.666$ (i.e. $2/3$). Of course, this parameter only makes sense for shell elements that use a global model (i.e. which are **not** integrated through the thickness).

Each of the above directives may be repeated as needed to associate appropriate values of the parameters to each concerned element.

8.1.5 EXCENTRICITY FOR SHELL ELEMENTS

Object:

This (optional) directive allows to specify excentricity for thick shell elements.

Syntax:

```
<"EXCE" exce /LECT/ >
```

exce

Distance between the shell mean surface and the reference surface used in the calculation.
The default value is 0 (no excentricity).

/LECT/

List of the elements concerned.

Comments:

For the moment, this feature concerns only T3GS and Q4GS shells [753].

8.1.6 SANDWICHES AND LAYERS

Object:

To define sandwiches, each composed of several layers, for use with some types of shell elements. Each **SAND** directive defines a new sandwich composed of several layers and associates it with a group of elements.

Syntax:

```
( "SAND"    nl "FRAC" nl*fracl "NGPZ" nl*ngpzl  /LECTURE/  )
```

nl

Number of layers in the sandwich, which is associated with each of the elements given in the following /LECT/.

fracl

Thickness fraction of each layer: this is the ratio of the layer thickness to the total thickness of the element.

ngpzl

Number of integration points through the thickness in each layer.

LECTURE

Elements concerned.

Comments:

For the moment, only the elements of type ED01, COQI, CQD3, CQD4, CQD6 and CQD9 may be chosen to be multi-layered sandwiches.

The directive **SAND** may be repeated as necessary (by repeating each time also the **SAND** keyword itself) in order to define all the geometrical information related to sandwiches. There may e.g. be a sandwich (and therefore elements) with, say, 3 layers, and another (other elements) with, say, 5 layers, in the same calculation.

Each sandwich stores all the geometrical information related to the layered structure. However, elements within the same sandwich may be made of different materials (or material combinations in the various layers).

Remember to assign a material to each layer, see page C.750.

The total number of integration points through the thickness of each element is defined by the sum of the `ngpz1` values over the layers of its sandwich. This value should not exceed the maximum value available for each element type. The value can be increased by using `DIME ... NGPZ`.

The number of Gauss points through the thickness for each layer is defined by `ngpz1`. Therefore, an additional `COMP NGPZ` for the *same* elements should be avoided.

The set of sandwiches defined in the `SAND` directive(s) is stored in an array. Each layer receives an index corresponding to its definition order within the corresponding sandwich. This index is then used in order to assign a material (see page C.1110) or a set of orthotropy directions (see page C.97) to each layer.

For example, suppose that we define two sandwiches, the first with 3 layers and the second with 5 layers. The layers of the first sandwich will be identified by indexes 1 to 3 while those of the second sandwich by indexes 1 to 5. These are the indexes to be used in successive directives to assign materials and/or orthotropy characteristics to each layer within the corresponding sandwich (i.e., within the associated element).

8.1.7 GEOMETRY OF THE BEAMS

Object:

Description of the characteristics of beam elements.

There are four possible shapes:

- arbitrary section **QUEL**;
- rectangular section **RECT**;
- circular section **CIRC**;
- annular section (pipe) **TUYA**.

Moreover, in the case of pipes, it is possible to enter a curvature in order to model the elbows.

References:

For an example of use of the various cross sections see e.g. reference [598]. For an example of use of annular sections see also reference [662]. Finally, reference [666] gives an overview of pipelines.

Syntax:

```
"GEOP" |[ "QUEL"  "VX" vx  "VY" vy  "VZ" vz  "AIRE" aire
           "IY" iy  "IZ" iz  "HY" hy  "HZ" hz
           < "J" j >  "R" r  < "EXCE" ex >           ;

"RECT"  "VX" vx  "VY" vy  "VZ" vz  "AY" ay
        "AZ" az  < "GAUC" gauch >  < "J" j >
        < "EXCE" ex >                               ;

"CIRC"  "VX" vx  "VY" vy  "VZ" vz  "DEXT" diam
        < "EXCE" ex >                               ;

"TUYA"  "VX" vx  "VY" vy  "VZ" vz  "DEXT" diam
        "EP" ep  < "COUR" co >  < "RAYC" rayco >
        < "SFY" sfy  "SFZ" sfz  "SFT" sft >
        < "EXCE" ex >                               ; ]|
```

```
< "VMIS"  "APRS" aprs  "AMMB" ammb  "ATRS" atrs  "AFLX" aflx >  
  
/LECTURE/
```

vx vy vz

Global coordinates X, Y, Z of a vector **v** defining the local system (**oxyz**), see comments below.

aire

Area of the cross section of the beam.

iy iz

Bending inertias around the local axes **y** and **z**.

hy

Distance along **y** used to estimate an “equivalent bending stress” around the local axis **z**.

hz

Distance along **z** used to estimate an “equivalent bending stress” around the local axis **y**.

j

Torsional inertia, if different from **iy + iz**.

r

Distance used to estimate an “equivalent torsional stress”.

ex

Excentricity along **y** (optional).

ay az

Length of the sides for a rectangular beam section (along **y** and **z**, respectively)

diam

External diameter for a beam with a circular section or for a pipe.

ep

Thickness of the pipe making up the beam.

co

Curvature of the elbows. It is the inverse of the curvature radius. In the case of a straight pipe: $co = 0.0$ This curvature radius stays constant during a calculation.

rayco

Curvature radius for the elbows. It is infinite for straight pipes.

sfy sfz sft

Coefficients of “surflexion” of the elbow: in the elbow plane (**sfy**), out of the plane (**sfz**) and in torsion (**sft**). The moment of inertia corresponding to the straight pipe is divided respectively by **sfy** (or **sfz** or **sft**) in order to account for the increased flexibility of the elbows (see also the comments below).

gauch

Coefficient of “gauchissement” for the cross-section, allowing to compute the torsional inertia starting from $J_o = I_y + I_z$ by means of a multiplicative coefficient: $J_p = \text{gauch} * J_o$.

VMIS

This keyword states that the weighting coefficients that allow to compute the Von Mises criterion starting from the different “equivalent stresses” are given by the user.

aprs

Weighting coefficient in the Von Mises criterion for the internal pressure of pipes.

ammb

Weighting coefficient in the Von Mises criterion for the membrane stress (normal stress).

atrs

Weighting coefficient in the Von Mises criterion for the equivalent torsional stress.

aflx

Weighting coefficient in the Von Mises criterion for the equivalent bending stress.

LECTURE

List of the elements concerned.

Comments:

A local reference system **oxyz** is attached to each beam element. The origin **o** of the system is in node 1 of the element. The second node (node 2) of the element then defines the longitudinal direction of the beam, which is assumed to correspond to the local **x** axis.

Then, to complete the definition of the local reference frame, another direction corresponding to the local **y** axis must be specified. This is done by giving a vector **v** (by means of its global components **vX**, **vY** and **vZ**), which is located in the **oxy** plane and completely defines this plane. Thus, the **v** vector may **not** coincide with **x**. The length of the **v** vector is irrelevant (but of course it may not be zero).

The **y** vector is then computed as the vector normal to **x** and lying in the plane defined by **x** and **v**. Finally, the **z** vector is computed as the vector normal to **x** and **y**.

In the case of an elbow, the vector **v** must be in the elbow plane and directed to the inner side of the elbow, however it is not compulsory that **v** is radial (directed exactly towards the “center” of the elbow). Bending around the **y** axis is therefore outside the elbow plane, while bending around the **z** axis is in the plane of the elbow.

In case of arbitrary cross section, the equivalent stress corresponding to the moment around **y** (respectively **z**) is computed for a distance h_z from the axis (respectively h_y) according to the following formula:

$$\sigma_y = M_y \frac{h_z}{I_y}$$

In the other cases, the formula is identical but the distance h becomes:

- for a rectangle: the corresponding half-side,
- for a cylinder or a tube: the external radius.

In the case of torsion, it is again the same formula, and the h distance is then:

- for an arbitrary cross-section: the **r** parameter,
- for a rectangle: the semi-diagonal,
- for a cylinder or a tube: the external radius.

In the case of a rectangular cross-section, it is possible to give either a “gauchissement” coefficient allowing to compute the torsional inertia starting from the inertia terms **iy** and **iz**, or to give directly the torsional inertia **j**.

For a square cross section this “gauchissement” coefficient has the value 0.844.

The bending inertias are modified in the case of elbows in order to account for the flexibility produced by the curvature as :

$$I_{coude} = I_{droit} / k$$

where k is **sfy**, **sfz** or **sft**.

ATTENTION: these coefficients are associated with the parameters of the von Mises criterion. A modification of the default values of **sfy**, **sfz** and **sft** imposes a different set of input values for **aprs**, **ammb**, **atrs** and **aflx**.

By default, these coefficients are computed as follows, according to RCCMR 3644.31: *i*) it is assumed that there is no change for the torsion ($\mathbf{sft} = 1$). *ii*) it is also assumed that the flexibility is the same in the plane

$$\mathbf{sfy} = \mathbf{sfz} = k$$

.

The coefficient k is a function of the elbow parameter λ , defined as follows: $k = 1.65/\lambda$. By definition, the elbow parameter λ is of the form :

$$\lambda = e R_c / R_m^2$$

where e is the thickness and R_m the mean radius of the tube, and R_c the curvature radius of the elbow. In practice, EUROPLEXUS computes and prints the inverse of this value, that vanishes for a straight tube.

The used Von Mises criterion σ^* is of the form : $\sigma^* = \sqrt{\alpha_p P^2 + \alpha_n \sigma_n^2 + \alpha_t \sigma_t^2 + \alpha_f \sigma_f^2}$

In this formula, coefficients α_p , α_n , α_t and α_f are respectively the parameters **aprs**, **ammb**, **atrs** and **aflx** defined above. By default, these coefficients assume the following values (which are those for a pipeline): $\alpha_p = 0.75$, $\alpha_n = 1$, $\alpha_t = 3$ and $\alpha_f = \pi^2/16$.

In the case of elbows, the coefficients α_t and α_f are modified. The default values are: $\alpha_t = 0.75$ and $\alpha_f = (\gamma \pi/4)^2$.

The coefficient γ has the expression: $\gamma = \max(1; \frac{8}{9}\lambda^{-2/3})$

Important: please consult also Chapter G.

8.1.8 DIAMETERS

Object :

Diameters of the monodimensional elements "TUBE", "TUYA", "CL1D" and "CLTU".

The elements "TUBE" and "TUYA" may be straight (constant diameter) or conical, but the elements "CL1D" et "CLTU" are always straight.

For practical reasons, there are always two diameters per element (at the inlet and at the outlet, respectively).

Syntax:

```
"DIAM" $[ "ELAS" ;
           "ELAT" "EPAI" epai "YOUN" youn "NU" nu ]$
|[ "DROI" d1 /LECTURE/ ;
   "CONE" "D1" d1 "D2" d2 "ORIG" /LECTURE/
           "LIST" /LECTURE/ ]|
```

"ELAS"

This optional key-word activates an elastic correction of the speed of sound in the fluid of the straight (DROI) TUYA elements listed in the followed /LECT/ sequence. By default, the fluid vena section is rigid. For information about this correction see reference [741].

"ELAT"

This optional key-word activates an elastic correction of the speed of sound in the fluid of the TUBE elements listed in the followed /LECT/ sequence. When specified, this key-word must be associated with the key-words "EPAI", "YOUN" and "NU" defining the Allievi correction (characteristics of an equivalent pipe structure). By default, the fluid vena section is rigid. For information about this correction see reference [754].

epai

Pipe thickness (optional key-word associated with "ELAT" key-word)

youn

Young modulus (optional key-word associated with "ELAT" key-word)

nu

Poisson coefficient (optional key-word associated with "ELAT" key-word)

"DROI"

Introduces the characteristics of a straight tube.

d1

Diameter of the straight tube.

LECTURE

List of the concerned elements.

"CONE"

Introduces the characteristics of a conical tube.

d1

Inlet diameter, corresponding to node "ORIGINE".

d2

Outlet diameter.

"ORIG"

The following directive /LECTURE/ allows to specify the origin node, where the diameter is d1.

"LIST"

The following directive /LECTURE/ lists the elements concerned.

Comments :

If the element is straight: $d1 = d2$. If the element is conical, d1 is different from d2.

If a conical pipe is composed of several elements, EUROPLEXUS automatically computes the inlet and outlet diameters of each of them.

If some friction (material PAROI) is associated to the fluid material, only cones with a vertex angle α less than 20 degrees are allowed.

In the last case, EUROPLEXUS computes the friction correction factor C_{frot} , according to the formulas of IDEL'CIK (diagram 5.2), where R is the ratio of the areas at the inlet and outlet of the cone ($R < 1$), and λ the loss coefficient for a straight tube:

$$C_{frot} = \frac{\lambda}{8 \sin(\alpha)} (1 - R^2)$$

According to IDEL'CIK, the loss coefficients are then, C_{div} for a divergent pipe and C_{conv} for a convergent pipe (diagrams 3.7 and 5.2) :

$$C_{div} = C_{frot} + 3.2 \tan(\alpha)^{\frac{5}{4}} (1 - R)^2$$

$$C_{conv} = C_{frot} + 0.45 (1 - R)$$

8.1.9 NAMED ELEMENT GROUPS

Object:

To define named groups of elements.

Syntax:

```
"GROU"  ngro * ('nom_grou' |[ /LECT/ ; STFL FLUI ; STFL CLXS ; DEBR ] |  
                <conditions>  
                <"CENT" cx cy cz>  
                <"SHRI" sh>  
                <"SHFT" sx sy sz>)
```

ngro

Total number of groups that will be defined.

nom_grou

Name associated with the group, enclosed in quotes.

/LECT/

List of the concerned elements.

STFL FLUI

Instead of the explicit elements list **/LECT/**, all structured fluid elements (defined using **STFL** command, see page C.68) are taken. Of course, if present this command must be specified *after* the definition of the structured fluid mesh.

STFL CLXS

Instead of the explicit elements list **/LECT/**, all **CLxS** elements attached to structured fluid elements (defined using **STFL CLxx** command, see page C.68) are taken. Of course, if present this command must be specified *after* the definition of the structured fluid mesh.

DEBR

Instead of the explicit elements list **/LECT/**, all **DEBR** elements are taken. Of course, if present this command must be specified *after* the definition of the debris particle.

<conditions>

An optional set of conditions that allow to restrict the chosen elements to a subset of the list specified in the preceding `/LECT/`. See below for the syntax of conditional statements.

CENT

Introduces the optional definition of a centerpoint for the group, of coordinates `cx`, `cy`, `cz`, to be used in graphical rendering. In particular, this point is used for shrinking operations (see **SHRI** below). If omitted, the code computes it automatically as the (unweighted) average of the center points of the elements belonging to the group.

SHRI

Introduces the optional definition of a shrinkage factor for the group, of value `sh`, to be used in graphical rendering. If omitted, a factor 1.0 is assumed.

SHFT

Introduces the optional definition of a shift vector for the group, of components `sx`, `sy`, `sz`, to be used in graphical rendering. If omitted, zero shift is assumed.

Comments:

Object names are not case-sensitive: they are converted internally to upper-case before being used.

After their definition, group names may be used to specify in input directives lists of elements (or of the associated nodes), in exactly the same way as GIBI object names are, within a `/LECT/` directive. The set of nodes ‘associated’ with a group of elements is the union of all nodes belonging to the elements in the group.

GIBI object names, or universal format groups or I-DEAS groups have the precedence over the present element groups, in case they are present (and in case of homonymy).

Note that, if element groups are to be passed to the OpenGL-based visualization module, they should preferably be disjoint, i.e. such that each element belongs to (at most) one group. This would ensure independence of rendering from the order in which group selection/unselection operations are performed. However, the code does not enforce this requirement, so that the graphical results are under full control (and responsibility) of the user.

The optional center, shrink and shift definitions may be used e.g. to obtain special graphical rendering effects such as an “exploded” view of a geometrical model. Be aware that the code applies shrinkage by the chosen factor around the centerpoint first, then followed by the chosen shift, if any.

Conditional statements

Various types of conditions may be imposed. The first one compares the position of the element's barycenter to a given value. Another one selects the (single) element whose barycenter is nearest to a given node or point. Other directives allow to identify all elements within a certain geometric shape (a box, a sphere, a cylinder, a cone). The last one allows to build up the complement (symmetric difference) of the chosen object with respect to a second object.

```
(COND | $ XB ; YB ; ZB $ $ LT ; GT $ val |
      | NEAR $ NODE /LECT/ ; POIN x y <z> $ |
      | BOX <X0 x0> <Y0 y0> <Z0 z0> DX dx DY dy <DZ dz> |
      | SPHE <XC xc> <YC yc> <ZC zc> R r |
      | CYLI <X1 x1> <Y1 y1> <Z1 z1> <X2 x2> <Y2 y2> <Z2 z2> R r |
      | CONE <X1 x1> <Y1 y1> <Z1 z1> <X2 x2> <Y2 y2> <Z2 z2> R1 r1 R2 r2 |
      | COMP /LECT/ |)
```

COND

Introduces a condition. This keyword may be repeated as many times as necessary to specify multiple conditions, which are applied in sequence.

XB

X-coordinate of the element's barycenter.

YB

Y-coordinate of the element's barycenter.

ZB

Z-coordinate of the element's barycenter.

LT

Less than operator.

GT

Greater than operator.

val

Value.

NEAR

Selects the (single) element whose centroid is nearest to a given node or point. If there is more than one element at the minimum distance, then only the first one found is retained.

NODE

Specify the node by the following /LECT.

POIN

Specify the point by its coordinates **x**, **y** and **z**. The last coordinate is needed only in 3D calculations.

BOX

Introduces the definition of a “box”, (a quadrilateral in 2D or a parallelepiped in 3D) with the sides aligned with the global axes.

x0, **y0**, **z0**

Coordinates of the ‘origin’ of the box.

dx, **dy**, **dz**

Lengths of the box sides.

SPHE

Introduces the definition of a sphere (in 3D, or a circle in 2D).

xc, **yc**, **zc**

Coordinates of the centre of the sphere (or of the circle).

r

Radius of the sphere or of the circle.

CYLI

Introduces the definition of a cylinder (3D only). The cylinder is defined by the two extremities of its axis (**P1**, **P2**) and its radius.

x1, **y1**, **z1**

Coordinates of the first extremity **P1** of the cylinder axis.

x2, **y2**, **z2**

Coordinates of the second extremity **P2** of the cylinder axis.

r

Radius of the cylinder.

CONE

Introduces the definition of a (truncated) cone (3D only). The cone is defined by the two extremities of its axis (**P1**, **P2**) and its radii.

x1, **y1**, **z1**

Coordinates of the first extremity P1 of the cone axis.

x2, y2, z2

Coordinates of the second extremity P2 of the cone axis.

r1

Radius of the cone at the first extremity.

r2

Radius of the cone at the second extremity.

COMP

Introduces a second object to be used for the symmetric difference (complement) operation.

/LECT/

List of the concerned elements.

Comments:

If any of the above coordinates (x0, y0 etc.) is omitted, it is assumed to be 0.

Example:

Suppose that we want to select all the elements of a 2D object **ob1** that lie in the first quadrant. The syntax would be:

```
COMP ... GROU 1 'firqua' LECT ob1 TERM
          COND XB GT 0
          COND YB GT 0
```

The group is from now on accessible under the name **firqua**.

Suppose then that we want to do the same thing as in the previous example, but also get access to the parts of **ob1** in the other three quadrants, under the name **others**. The syntax would be:

```
COMP ... GROU 2 'firqua' LECT ob1 TERM
                COND XB GT 0
                COND YB GT 0
        'others' LECT ob1 TERM
                COND COMP LECT firqua TERM
```

Note that the `firqua` object becomes available immediately after its definition, and may therefore be used in the definition of the `others` group.

8.1.10 NAMED NODE GROUPS

Object:

To define named groups of nodes.

Syntax:

```
"NGRO"  nngr * ('nom_grou' /LECT/ <conditions>)
```

nngr

Total number of node groups that will be defined.

nom_grou

Name associated with the group, enclosed in quotes.

/LECT/

List of the concerned nodes.

<conditions>

An optional set of conditions that allow to restrict the chosen nodes to a subset of the list specified in the preceding **/LECT/**. See below for the syntax of conditional statements.

Comments:

Object names are not case-sensitive: they are converted internally to upper-case before being used.

After their definition, group names may be used to specify in input directives lists of nodes (or of the associated elements), in exactly the same way as GIBI object names are, within a **/LECT/** directive. An element is considered 'associated' with a group of nodes if and only if all its nodes belong to the group.

GIBI object names, or universal format groups or I-DEAS groups have the precedence over the present node groups, in case they are present (and of homonymy). Moreover, named groups of elements (see page C.61) also have the precedence over the present node groups, in case they are present (and of homonymy).

Note that, if node groups are to be passed to the OpenGL-based visualization module, they should preferably be disjoint, i.e. such that each node belongs to (at most) one group. This

would ensure independence of rendering from the order in which group selection/unselection operations are performed. However, the code does not enforce this requirement, so that the graphical results are under full control (and responsibility) of the user.

Conditional statements

Various types of conditions may be imposed. The first one compares the node position to a given value. Other directives allow to identify all nodes within a certain geometric shape (a box, a sphere, a cylinder, a cone). The last one allows to build up the complement (symmetric difference) of the chosen object with respect to a second object.

```
(COND | $ X ; Y ; Z $ $ LT ; GT $ val |
| BOX <X0 x0> <Y0 y0> <Z0 z0> DX dx DY dy <DZ dz> |
| SPHE <XC xc> <YC yc> <ZC zc> R r |
| CYLI <X1 x1> <Y1 y1> <Z1 z1> <X2 x2> <Y2 y2> <Z2 z2> R r |
| CONE <X1 x1> <Y1 y1> <Z1 z1> <X2 x2> <Y2 y2> <Z2 z2> R1 r1 R2 r2 |
| LINE X1 x1 Y1 y1 <Z1 z1> X2 x2 Y2 y2 <Z2 z2> TOL tol <DIST d> |
| COMP /LECT/ |)
```

COND

Introduces a condition. This keyword may be repeated as many times as necessary to specify multiple conditions, which are applied in sequence.

X

X-coordinate of the node.

Y

Y-coordinate of the node.

Z

Z-coordinate of the node.

LT

Less than operator.

GT

Greater than operator.

val

Value.

BOX

Introduces the definition of a “box”, (a quadrilateral in 2D or a parallelepiped in 3D) with the sides aligned with the global axes.

x_0, y_0, z_0

Coordinates of the ‘origin’ of the box.

dx, dy, dz

Lengths of the box sides.

SPHE

Introduces the definition of a sphere (in 3D, or a circle in 2D).

xc, yc, zc

Coordinates of the centre of the sphere (or of the circle).

r

Radius of the sphere or of the circle.

CYLI

Introduces the definition of a cylinder (3D only). The cylinder is defined by the two extremities of its axis (P1, P2) and its radius.

x_1, y_1, z_1

Coordinates of the first extremity P1 of the cylinder axis.

x_2, y_2, z_2

Coordinates of the second extremity P2 of the cylinder axis.

r

Radius of the cylinder.

CONE

Introduces the definition of a (truncated) cone (3D only). The cone is defined by the two extremities of its axis (P1, P2) and its radii.

x_1, y_1, z_1

Coordinates of the first extremity P1 of the cone axis.

x_2, y_2, z_2

Coordinates of the second extremity P2 of the cone axis.

r1

Radius of the cone at the first extremity.

r2

Radius of the cone at the second extremity.

LINE

Introduces the definition of a straight line. The line is defined by the two extremities (P1, P2), a tolerance (TOL) and an optional spacing (DIST) between the nodes to be retained. Nodes on the line are kept in the order in which they occur passing from P1 to P2 (included).

In 3D space, the line passing through 2 points P1(x_1, y_1, z_1), P2(x_2, y_2, z_2) has the following parametric equations:

$$x = x_1 + dx t, \quad dx = x_2 - x_1$$

$$y = y_1 + dy t, \quad dy = y_2 - y_1$$

$$z = z_1 + dz t, \quad dz = z_2 - z_1$$

From each equation a value for t can be defined

$$t_x = \frac{x - x_1}{dx}$$

$$t_y = \frac{y - y_1}{dy}$$

$$t_z = \frac{z - z_1}{dz}$$

These 3 quantities are calculated for each point of the applied model and if the 3 following conditions are satisfied then the point is valid.

- t_x, t_y, t_z are equal or at least their difference is smaller or equal to the defined tolerance (in order the point to be on the line defined by P1, P2).
- The common value of t_x, t_y, t_z is between 0.0 and 1.0 (in order the point to be between P1 and P2).
- The value of t divided by (DIST) (if it is not zero) should be in the vicinity of an integer number (the length of the vicinity is defined by the input tolerance).

x1, y1, z1

Coordinates of the first extremity P1 of the line.

x2, y2, z2

Coordinates of the second extremity P2 of the line.

tol

Tolerance for searching the nodes on the line.

d

Parametric distance between two consecutive nodes retained on the line. If omitted, all nodes on the line are retained. If specified, then the parametric distance between two consecutive retained nodes will be greater or equal to d (and so possibly some nodes on the line will be skipped).

COMP

Introduces a second object to be used for the symmetric difference (complement) operation.

/LECT/

List of the concerned nodes.

Comments:

If any of the above coordinates (x_0 , y_0 etc.) is omitted, it is assumed to be 0.

Example:

Suppose that we want to select all the nodes of a 2D object **ob1** that lie (strictly) within the first quadrant. The syntax would be:

```
COMP ... NGRO 1 'firqua' LECT ob1 TERM
      COND X GT 0
      COND Y GT 0
```

The group is from now on accessible under the name **firqua**.

Suppose then that we want to do the same thing as in the previous example, but also get access to the nodes of **ob1** in the other three quadrants, under the name **others**. The syntax would be:

```
COMP ... NGRO 2 'firqua' LECT ob1 TERM
      COND X GT 0
      COND Y GT 0
      'others' LECT ob1 TERM
      COND COMP LECT firqua TERM
```

Note that the `firqua` object becomes available immediately after its definition, and may therefore be used in the definition of the `others` group.

8.1.11 ELEMENT COLORS

Object:

To define or re-define (e.g. in the case of a mesh generated by Cast3m) the colors of elements, for visualization purposes.

Syntax:

```
"COUL" (nom_coul /LECT/)
```

`nom_coul`

Name of the color (**not** enclosed in quotes). The valid names are those of Cast3m, i.e. `bleu`, `roug`, `rose`, `vert`, `turq`, `jaun`, `blan` or `noir`, plus the following nine gray levels: `gr10` (almost black), `gr20`, `gr30`, `gr40`, `gr50`, `gr60`, `gr70`, `gr80` and `gr90` (almost white). Elements not assigned a color have a default color.

`/LECT/`

List of the concerned elements.

Comments:

Repeat as many times as necessary to define all the desired colors.

This directive is particularly useful in conjunction with the definition of element groups (see `GROU`) to assign colors to groups of elements.

If there are several colors to be defined, be sure **not** to repeat the keyword `COUL`, but only the color name `nom_coul` followed by the corresponding `/LECT/`. In fact, each time the keyword `COUL` is encountered, all colors defined so far are reset to the default color (black). For example:

```
COUL roug LECT explosive TERM
      vert LECT structure TERM
```

is correct, while:

```
COUL roug LECT explosive TERM
COUL vert LECT structure TERM
```

would be wrong (the `explosive` object would appear black and not red).

8.1.12 COMPLEMENTS FOR RTM COMPOSITE MATERIALS

Object:

This directive allows to define values to used by a composite material made by a RTM process ie. the CRTM material (page C264).

Syntax:

"RTMVF" *vf* /LECTURE/

"RTMRCT" *rct* /LECTURE/

"RTMANGL" *angle* /LECTURE/

vf

Value of the volumic fraction

rct

Value of the ratio between warp and weft

angle

Value of the angle between warp and weft directions.

LECTURE

List of the elements concerned.

8.1.13 PFEM METHOD

Object:

Warning: the present model is under development at JRC and not all the directives described below are available yet!

This directive sets some parameters used by the PFEM method.

Syntax:

```
PFEM  H h ALPHA alpha
```

h

Expected distance between nodes in the Bowyer-Watson triangulation.

alpha

Alpha coefficient for the Alpha-shape method to determine the contour of the triangulation.

8.1.14 FLYING DEBRIS MODEL

Object:

Warning: the present model is under development at JRC and not all the directives described below are available yet!

This directive allows to model flying debris resulting from an explosion or an impact. Each piece of debris is modeled by a particle, optionally embedded in the surrounding fluid and optionally subjected to the gravity force. This latter force, if present, must be specified via the CHAR CONS GRAV directive, see page F.30.

The fluid surrounding the debris particles may be modeled either as a uniform field with constant properties (velocity, density), or as an evolving fluid field, discretized by Finite Elements or Finite Volumes, or even as a combination of the two (e.g., FE fluid field near the explosive source, uniform field far away).

The debris particles may either be active from the very beginning of the calculated transient (thus assuming that they result from a fragmentation process that occurred at a previous time), or they may be associated with certain finite elements representing a structure, and be activated automatically by the code only when the element undergoes complete failure.

In any case, since particles are represented by the specialized DEBR elements, and since the topology is basically constant in time in EUROPLEXUS, **all particles must be declared** (and thus they are present in the model) **from the beginning of the calculation**. However, some of them may be already active at the initial time, some not.

The model includes the optional treatment of the impact of debris particles against the surrounding structure. This is accomplished by the pinball method.

Syntax:

```
DEBR <ROF rof>
      <VFX vfx> <VFY vfy> <VFZ vfz>
      <FLUI /LECT/ <DGRI> $[ HGRI hgri ; NMAX nmax ; DELE dele ]$ >
      (PART particle_description)
      (FILL fill_description)
```

rof

Density of the uniform fluid field in which the particle is embedded. This value is 0.0 by default, meaning that the particle moves in vacuum.

vfx, vfy, vfz

Components of the velocity of the surrounding uniform fluid field. These values are 0.0 by default.

FLUI

Introduces the **/LECT/** of the discretized fluid domain with which the particles motion should be coupled. When a particle traverses this domain, the (local) fluid velocity and density are automatically computed by the code, instead of using the constant user-given values **rof, vfx, vfy, vfz** described above. A fast search algorithm based on a grid of cells (as in bucket sorting) is used to compute the fluid element (if any) encompassing each debris particle.

DGRI

Dump out the initial grid of cells used for fast searching on the listing (only at step 0).

HGRI

Specifies the size of the grid cell. Each cell has the same size in all spatial directions and is aligned with the global axes. Note that the size of this grid is related to the size of the **fluid** elements specified in **FLUI**, not of the structural elements producing the flying debris.

NMAX

Specifies the maximum number of cells along one of the global axes.

DELE

Specifies the size of the grid cell as a multiple of the diameter of the largest coupled **fluid** element. Element “diameters” are computed only along each global spatial direction and the maximum is taken. For example, by setting **DELE 4** the size of the cell is four times the diameter of the largest coupled fluid element. By default, i.e. if neither **HGRI**, nor **NMAX**, nor **DELE** are specified, the code takes **DELE 3** (this value is probably too large, a value of 1.1 or so should be more appropriate in most cases).

PART

Introduces the description of a single particle, see details below. Such particles are active from the very beginning of the calculation. This directive may be repeated any number of times.

FILL

Fill by particles a single finite element or a finite element mesh. Such particles may either be active from the very beginning of the calculation, or be activated upon failure of the associated element(s). See below for the details of this directive. This directive may be repeated any number of times.

Dimensioning for the flying debris:

Dimensioning for the flying debris cannot be made fully automatic, because of the `FILL` command which generates a variable number of particles depending upon which finite element type it is applied to. The following semi-automatic procedure is suggested:

- Prepare an input file containing the `COMP DEBR ...` directive but without any dimensioning for the debris. The code will stop with an error message and it will print the total number of `DEBR` needed, say `n_debr`.
- Add to the input file the directive `DIME DEBR n_debr TERM`. Now the calculation should run smoothly.

Comments:

Debris particles may be subjected to the gravity force. This latter force, if present, must be specified via the `CHAR CONS GRAV` directive, see page F.30.

The drag force acting on a particle is $\underline{F}_d = -C_d \rho_f A \|\underline{w}\|^2 \frac{\underline{w}}{\|\underline{w}\|}$, where C_d is the particle's drag coefficient (see below), ρ_f is the fluid's density, $A = \frac{\pi}{4} d^2$ is the particle's cross-section, d is the particle's diameter and \underline{w} is the particle's velocity \underline{v} relative to the fluid velocity \underline{v}_f : $\underline{w} = \underline{v} - \underline{v}_f$.

The total number of particles described by the `PART` and `FILL` sub-directives must be less than or equal to the number of elements of type `DEBR` that has been reserved in the dimensioning of the problem.

Describing a single particle

Object:

To describe a single particle of debris. The particle is already active at the beginning of the transient calculation. Therefore, it results from the fragmentation of a structure which has occurred at a previous time.

Syntax:

```
PART <X x> <Y y> <Z z>    <VX vx> <VY vy> <VZ vz>
      RO ro  D d  DRAG drag
      <COUP> <IMPA> <TRAJ>
```

x, y, z

Coordinates of the particle at the initial time. These values are 0.0 by default.

vx, vy, vz

Velocity components of the particle at the initial time. These values are 0.0 by default.

ro

Density of the particle.

d

Diameter of the particle.

drag

Drag coefficient of the particle. This is a number usually between 0.3 and 1.1 for a sphere. In the supersonic region the value is almost constant and close to 1.0, while it drops rapidly in the transonic region.

COUP

Couple the particle's motion with the surrounding fluid domain defined by the FLUI directive above.

IMPA

Treat the impact of the particle against surrounding structures (by the pinball method). One (parent) pinball is embedded in the particle, of diameter equal to that of the particle. Pinballs for the potentially impacted structures must be defined separately by the PINB directive, see page D.480.

TRAJ

Save the particle's trajectory, e.g. for visualization purposes.

Filling an element or a mesh by particles

Object:

To fill by debris particles an element or a mesh. Particles are automatically generated uniformly within the volume of the element or mesh (element by element).

The particles may either: 1) be already active at the beginning of the transient calculation (in this case they result from the fragmentation of a structure which has occurred at a previous time), or 2) be activated automatically by the code when the associated element(s) undergo complete failure.

In case 1) above, the associated element or mesh is defined at the geometric level only as a geometric support for the particles generation. This element or mesh must be associated with a **FANT** material so as to exclude it from the transient computation.

In case 2) above, the associated element or mesh must be assigned a structural material with a failure model (thus **not** the **FANT** material). When the element(s) fail, the associated particles are suddenly activated while at the same time the element is deactivated, so it no longer contributes to the model.

Syntax:

```
FILL  $<VX vx> <VY vy> <VZ vz> ; <VR vr <CX cx> <CY cy> <CZ cz>>$
      PLEV plev R0 ro DRAG drag <AFly afly>
      <COUP> <IMPA> <TRAJ>
      OBJE /LECT/
```

vx, vy, vz

Components of the velocity of the particles at the initial time. This value is 0.0 by default.

vr

Radial velocity of the particles at the initial time. This value is 0.0 by default.

cx, cy, cz

Coordinates of the particles centroid with respect to which the radial velocity is expressed. By default this is the centroid of the geometrical object defined by the **/LECT/** directive given below.

plev

Level of hierarchic subdivision of the parent element(s) along each spatial direction in order to generate the particles. The particles' diameter is automatically determined so as to conserve the element's total volume. For example, a level of 3 means that $2^3 = 8$ subdivisions along each spatial direction to generate the particles. A 2D quadrilateral element would in this case be filled by $8 \cdot 8 = 64$ particles.

ro

Density of the particles. This should be equal to the density of the parent element. The user is left responsible for this, since material densities are read in only later on in the input file.

drag

Drag coefficient of the particles. This is a number usually between 0.3 and 1.1 for a sphere. In the supersonic region the value is almost constant and close to 1.0, while it drops rapidly in the transonic region.

afly

The drag forces and the **AIRB** forces depend on the area of the spherical particle, which can be different from the “true” debris cross section. For each shell element and beam element a minimum and a maximum area are estimated. By using the keyword **AFly** the minimum (**afly** = 0.0) or the maximum (**afly** = 1.0) value is used. Values of **afly** between 0.0 and 1.0 interpolate linearly between these two values. The default value is 0.5. For solid elements, the cross section of the spherical particle is used and so **afly** is ignored.

COUP

Couple the particles’ motion with the surrounding fluid domain defined by the **FLUI** directive above.

IMPA

Treat the impact of the particles against surrounding structures (by the pinball method). One (parent) pinball is embedded in each particle, of diameter equal to that of the particle. Pinballs for the potentially impacted structures must be defined separately by the **PINB** directive, see page D.480.

TRAJ

Save the particle’s trajectories, e.g. for visualization purposes.

OBJE

Introduces the list of the elements to be filled.

/LECT/

List of the elements to be filled.

Comments:

The initial velocity of each group of particles defined by the **FILL** directive described above may be defined in two ways:

- Either it is assumed as uniform for all particles: in this case specify the necessary Cartesian components **vx**, **vy** and **vz**;
- Or, it is assumed to be oriented radially from a point: in this case specify the velocity modulus **vr**. By default, the assumed point is the centroid of the geometric object being filled. However, the user may specify a different point by giving **cx**, **cy** and **cz**.

8.1.15 DISPLACEMENT EROSION

Object:

This directive allows to define an erosion (failure) criterion, which uses a maximum displacement of a given node.

The model can be used for calculations of laminated windows. The criterion of the complete erosion of a laminated window can be set to 30% of the span.

The model can be combined with any other erosion criterion.

Syntax:

```
FAIL ( DISP disp NODE /LECT/ OBJE /LECT/ )  
      ( AUTO rati DIRE disp /LECT/ )
```

disp

Displacement of the node given by the keyword **NODE**, which results in an erosion (failure) of the elements given by the keyword **OBJE**.

node

Node used for the displacement criterion. The following **/LECT/** must contain just one node index.

OBJE

Introduces the **/LECT/** of the elements which are eroded, if the criterion is reached.

AUTO

The keyword **AUTO** introduces an automatic development of the nodes used for the displacement criterion and the element which should eroded. The elements given by **/LECT/** are separated to several subsets, the node near the barycentre is used for the criterion, the full subset is used for the elements which could be eroded.

rati

Ratio of the minimum span which should be used for the displacement criterion.

dire

The maximum span is defined in this direction.

Remarks:

The set of keywords **DISP ... OBJE** may be repeated as many times as needed to define all the desired displacement-based erosion criteria.

8.1.16 STRUCTURED FLUID GRID MODEL

Object:

This directive allows to define a structured, Eulerian fluid grid that is added to the mesh specified in the **GEOM** directive. The grid has the form of a rectangular parallelepiped, is aligned along the global axes, and has a uniform spacing in each of the three global directions.

Using a structured fluid grid may substantially speed up the numerical calculations because many operations (especially those related to searching) can be highly optimized. In particular, this model is useful in conjunction with the **FLSR** model for fluid-structure interaction, see page D2.143.

Special fluid elements of type **FL2S** (in 2D) or **FL3S** (in 3D) are automatically built up and used to discretize the structured grid. The former is a simplified version of **FL24** while the latter is a simplified version of **FL38**.

All nodes of the structured fluid grid (which are also generated automatically) must be declared Eulerian in the **GRIL** directive, see page B.60. Note that nodes not mentioned in the **GRIL** directive are indeed considered Eulerian.

This directive may only be used in ALE or purely Eulerian calculations.

Although the elements and nodes of the structured fluid grid are not contained in the mesh read in by the **GEOM** directive, adequate dimensioning must be provided. In 2D, the number of elements generated automatically is $nx*ny$ and the number of nodes is $(nx+1)*(ny+1)$. In 3D, the number of elements generated automatically is $nx*ny*nz$ and the number of nodes is $(nx+1)*(ny+1)*(nz+1)$. These dimensions must be added to the dimensions for the other elements and nodes which are explicitly defined in the mesh.

In addition to the fluid elements, special boundary condition elements of type **CL2S** (in 2D) or **CL3S** (in 3D) may be optionally generated along the appropriate faces of the fluid domain (see **CLij** input directives below). These may be used, for example, to specify non-absorbing boundary conditions. If such elements are generated, their number must be estimated *a priori* by the user and appropriate dimensionings (only for the elements, and not for the nodes, which are in common with the fluid) must be specified.

Syntax:

```
STFL X0 x0 Y0 y0 <Z0 z0>
      LX lx LY ly <LZ lz>
      NX nx NY ny <NZ nz>
      <CLX1> <CLX2> <CLY1> <CLY2> <CLZ1> <CLZ2>
```

x0, y0, z0

Coordinates of the origin of the structured fluid grid. The z -coordinate **z0** is only needed in 3D calculations.

lx, ly, lz

Total lengths of the sides of the structured fluid grid. The z -length **lz** is only needed in 3D calculations.

nx, ny, nz

Number of cells of the structured fluid grid in each direction. The z -number of cells **nz** is only needed in 3D calculations. Cells have a uniform length in each direction.

CLX1

Automatically generate CL2S elements (CL3S in 3D) along the face of the fluid domain of equation $x = x_0$.

CLX2

Automatically generate CL2S elements (CL3S in 3D) along the face of the fluid domain of equation $x = x_0 + l_x$.

CLY1

Automatically generate CL2S elements (CL3S in 3D) along the face of the fluid domain of equation $y = y_0$.

CLY2

Automatically generate CL2S elements (CL3S in 3D) along the face of the fluid domain of equation $y = y_0 + l_y$.

CLZ1

Useful only in 3D. Automatically generate CL3S elements along the face of the fluid domain of equation $z = z_0$.

CLZ2

Useful only in 3D. Automatically generate CL3S elements along the face of the fluid domain of equation $z = z_0 + l_z$.

Comments:

Each cell (element) of the grid is a rectangle (rectangular parallelepiped in 3D) with sides of length l_x/n_x , l_y/n_y (and l_z/n_z in 3D).

Nodes and elements in the grid are numbered progressively starting from the chosen origin (x_0, y_0, z_0) , first along the global X -direction, then along the Y -direction (in 3D, finally along the Z -direction).

Once the additional elements and nodes have been generated by the **STFL** directive, they are considered like any other elements and nodes, in particular as concerns the rest of the input file and the post-processing.

Appropriate materials must be assigned, in the usual way, to all the automatically generated elements. For example, a low-pressure gas to all fluid elements except those in a bubble zone, representing an explosion, in which a high-pressure gas is assigned. In order to identify the concerned elements, use may be made e.g. of directives for the definition of element groups, see page C.61. A special command to choose the **STFL** elements is provided, see **STFL FLUI** or **STFL CLXS** on page C.61.

In the frequent case of absorbing boundaries of the fluid domain, the concerned **CL2S** or **CL3S** elements must be identified in order to assign an adequate impedance material to them. The rule for automatic numbering of the generated elements is as follows: first, all fluid elements are generated (their number may be computed as specified above). Next, any specified **CL2S** or **CL3S** elements are generated, in the following order: **CLX1**, **CLX2**, **CLY1**, **CLY2**, **CLZ1**, **CLZ2**.

Appropriate boundary conditions may also be specified (e.g. via **LINK**) as appropriate to the boundary nodes (e.g. to block a certain face of the fluid domain).

8.1.17 AUTOMATIC GENERATION OF SPECTRAL MICRO MESH

Object:

This directive allows to automatically generate a Spectral Element (SE) “micro” mesh starting from an SE “macro” mesh and a given degree (N) of the interpolation polynomial. The degree of the polynomial is the same for all spectral elements, and along each of the spatial directions.

The “macro” spectral element mesh is composed of either MS24 4-node quadrilateral elements (in 2D) or of MS38 8-node hexahedral elements (in 3D), and must have been specified in the previous GEOM directive. The generated micro SE mesh will be composed of S24 4-node quadrilaterals in 2D or of S38 8-node hexahedra in 3D.

Syntax:

SPEC GMIC NSPE nspe

GMIC

Introduces the automatic generation of micro SE elements according to the parameters given in the following.

nspe

Degree N of the interpolation polynomial for the SE mesh.

Comments:

Each macro SE generates exactly N^2 micro SE in 2D or N^3 micro SE in 3D.

The number of micro SE nodes generated is roughly (by excess) $(N + 1)^2$ in 2D or $(N + 1)^3$ in 3D, for each macro SE. The exact number of generated nodes is difficult to determine *a priori* because it depends upon the connectivity of the macro SE mesh (coincident nodes of adjacent micro SE and coincident nodes of micro and macro SE are eliminated). After the calculation of the exact number of nodes (and elements) required, the code prints out this information in case the user wants to keep the memory to a minimum (by giving minimum dimensioning commands).

The generated micro SE are available in an automatically created element group named `_S24` if the calculation is 2D, or `_S38` if the calculation is 3D.

Note that, like for other directives which change the mesh topology (by adding new elements and new nodes), the dimensioning related to geometrical data cannot be fully automatic. The user *must* in this case dimension the total number of nodes, the total number of degrees of freedom and the total number of micro SE generated elements (S24 in 2D or S38 in 3D), like in the following example:

```
. . .  
DIME NPOI 9 NDDL 18 S24 4 TERM  
. . .  
GEOM . . .  
COMP SPEC GMIC NSPE 2  
. . .
```

8.1.18 ELEMENT-SPECIFIC EROSION

Object:

This directive allows to define an erosion criterion for a specific subset only of the elements. A global definition of the erosion criterion is given in the definition of the problem (see directive EROS <ldam> on page A.30). The global value given there can be overridden for one or more subsets of the elements by using the present directive.

Syntax:

```
EROS  $[ eros ; NOER ]$ /LECT/
```

eros

Erosion criterion for the elements given by /LECT/. If no erosion limit is needed for a set of elements the keyword NOER can be taken.

Remarks:

The set of keywords EROS ... /LECT/ may be repeated as many times as needed to define all the desired element-based erosion criteria.

8.1.19 MESH ORIENTATION

Object:

To orient or re-orient those elements of the mesh for which a specific orientation is important. Typically, these are 3D shell elements without a topological thickness. Normally, proper orientation should be done in the mesh generator, but the present directive may be useful to correct any problems in case one uses a mesh whose generator is not available.

This sub-directive should be used only in emergency cases, e.g. when the mesh used in a calculation (especially flat 3D shell elements) has the wrong orientation and comes from a mesh generator that is not available. This command has the last word on the orientation of the elements, since it comes after the automatic re-orientation which is done in the SENS routine (called from the geometry reading routine). The user is therefore fully responsible of the use of this command.

Syntax:

```
"ORIE"    < "OBJE" /LEC1/ $[ "POIN" x y z ; "NODE" /LECN/ ]$ >
          < "INVE" /LEC2/ >
```

OBJE

The elements in object /LEC1/ have to be oriented so that their outwards normal direction points towards a certain point or node in space, to be specified next. By “pointing” we intend here simply that the scalar product of the unit normal with the line joining the element’s center to the given point or node should be positive.

POIN

Introduces the coordinates of the point.

x y z

Coordinates of the point. Note that three coordinates should always be given even in 2D cases (but the ORIE directive is only useful in 3D cases anyway).

NODE

Introduces the index of the node.

/LECN/

One node index or the name of an object with just one node (e.g. a Cast3m point name if the mesh has been produced by Cast3m).

INVE

The orientation of elements in object /LEC2/ has to be inverted without any checking.

Comments:

Only some element types admit re-orienting: typically, these are 3-node or 4-node “thin” elements in 3D, such as shell, membrane or CLxx elements.

Note that the **ORIE** sub-directive may be repeated any number of times, if needed. For example, this may be useful to re-orient a randomly oriented closed surface so that it points outwards. Use a first **ORIE** sub-directive to orient the all the surface elements consistently towards an internal point (e.g. its barycenter). Then, use a second **ORIE** sub-directive to invert the orientation:

```
COMP ... ORIE OBJE LECT toto TERM POIN x y z
        ORIE INVE LECT toto TERM
```

8.1.20 AUTOMATICALLY GENERATED SPH PARTICLES

Object:

To generate automatically SPH particles within user-defined volumes.

Syntax:

```
"GBIL"  ngen * (RBIL r <RESE rese>
(INSI | BOX <X0 x0> <Y0 y0> <Z0 z0> DX dx DY dy <DZ dz> |
      | SPHE <XC xc> <YC yc> <ZC zc> R r |
      | CYLI <X1 x1> <Y1 y1> <Z1 z1> <X2 x2> <Y2 y2> <Z2 z2> R r |
      | CONE <X1 x1> <Y1 y1> <Z1 z1> <X2 x2> <Y2 y2> <Z2 z2> R1 r1 R2 r2 |)
(OUTS | BOX <X0 x0> <Y0 y0> <Z0 z0> DX dx DY dy <DZ dz> |
      | SPHE <XC xc> <YC yc> <ZC zc> R r |
      | CYLI <X1 x1> <Y1 y1> <Z1 z1> <X2 x2> <Y2 y2> <Z2 z2> R r |
      | CONE <X1 x1> <Y1 y1> <Z1 z1> <X2 x2> <Y2 y2> <Z2 z2> R1 r1 R2 r2 |))
```

ngen

Total number of groups of SPH particles that will be generated.

r

Radius for the particles of this group.

rese

Type of spheres packing: 0 means compact hexagonal (default), 1 means compact cubic (to be implemented), 2 means trivial (non-compact) cubic (normally to be used only for tests and debugging).

INSI

Introduces an “inside” condition: all particles “within” a certain geometrical shape are to be generated. This keyword can be repeated as many times as necessary to specify multiple conditions, which are applied in sequence. As a result, all the particles “inside” the union of the specified geometrical shapes will be generated.

OUTS

Introduces an “outside” condition: from all particles in the set generated by the previously specified INSI condition(s), only those “external” to a certain geometrical shape are to be retained. This keyword can be repeated as many times as necessary to specify multiple conditions, which are applied in sequence. As a result, only the particles “outside” the union of the specified geometrical shapes will be retained.

BOX

Introduces the definition of a “box”, (a quadrilateral in 2D or a parallelepiped in 3D) with the sides aligned with the global axes.

x_0, y_0, z_0

Coordinates of the ‘origin’ of the box.

dx, dy, dz

Lengths of the box sides.

SPHE

Introduces the definition of a sphere (in 3D, or a circle in 2D).

x_c, y_c, z_c

Coordinates of the centre of the sphere (or of the circle).

r

Radius of the sphere or of the circle.

CYLI

Introduces the definition of a cylinder (3D only). The cylinder is defined by the two extremities of its axis (P1, P2) and its radius.

x_1, y_1, z_1

Coordinates of the first extremity P1 of the cylinder axis.

x_2, y_2, z_2

Coordinates of the second extremity P2 of the cylinder axis.

r

Radius of the cylinder.

CONE

Introduces the definition of a (truncated) cone (3D only). The cone is defined by the two extremities of its axis (P1, P2) and its radii.

x_1, y_1, z_1

Coordinates of the first extremity P1 of the cone axis.

x_2, y_2, z_2

Coordinates of the second extremity P2 of the cone axis.

r1

Radius of the cone at the first extremity.

r2

Radius of the cone at the second extremity.

Comments:

If any of the above coordinates (x0, y0 etc.) is omitted, it is assumed to be 0.

Example:

Suppose that we want to generate SPH particles within a cylinder representing a pipe full of fluid. Then the syntax would be simply:

```
GBIL 1 RBIL 0.001
      INSI CYLI X0 0 Y0 0 Z0 0 X1 0 Y1 0 Z1 10 R 0.1
```

The group of particles is from now on accessible under the name `_gbil001`.

Suppose then that the pipe of the previous example is submerged in the sea. To generate also the particles in a prismatic sea region around the pipe, the syntax would be:

```
GBIL 2 RBIL 0.001
      INSI CYLI X0 0 Y0 0 Z0 0 X1 0 Y1 0 Z1 10 R 0.1
      RBIL 0.001
      INSI BOX  X0 -1 Y0 -1 Z0 0 DX 2 DY 2 DZ 10
      OUTS CYLI X0 0 Y0 0 Z0 0 X1 0 Y1 0 Z1 10 R 0.1
```

The two group of particles are from now on accessible under the names `_gbil001` and `_gbil002`, respectively.

The presence of the (mandatory) **RBIL** keyword starts a new group of particles. Each group must contain at least one **INSI** condition. All **INSI** conditions must be specified before the **OUTS** conditions (if any).

8.1.21 WATER TABLES

Object :

These directives create tables containing the physical properties of water, according to one of the following textbooks:

- 1) Directive "TEAU" : Properties of water and steam in SI - units

(E.Schmidt Springer Verlag, Berlin 1979)

or

- 2) Directive "TH2O" (letter O) : NBS/NRC Steam Tables 1984

(Extended tables)

Syntax :

```
$[ "TEAU" ; "TH2O" ]$    "TMIN"  tmin  "TMAX"  tmax
                        "PMIN"  pmin  "PMAX"  pmax
                        "UNIL"  cl    "UNIM"  cm
                        "DBTE"  nbte  "DSAT"  nsat    "DHTE"  nhte
                        < "DPHY"  nhhy >
```

For the tests:

```
< "DESS" >  < "PERF" >  < "TEST" ( "CAS" ... ) "FINT" >
```

With :

```
"CAS"  num    "P1" p1 $ "T1" t1 ; "X1" x1 $
          $    "P2" p2 $ "T2" t2 ; "X2" x2 $ $
          $    "DVS" dvs    "DH" dh          $
```

tmin

Minimum temperature in the tables (this must be lower than the saturation temperature corresponding to **pmin**).

tmax

Maximum temperature in the tables (this must be higher than the saturation temperature corresponding to **pmax**, in the case of a sub-critical domain, or to 374 degrees Celsius in the case of a hyper-critical domain).

pmin

Minimum pressure in the tables.

pmax

Maximum pressure in the tables.

cl

Conversion factor of the length units adopted, towards metres.

cm

Conversion factor of the mass units adopted, towards kilograms.

nbte

Number of intervals into which the low-temperature domain. is subdivided.

nhte

Number of intervals into which the high-temperature domain. is subdivided.

nsat

Number of intervals into which the pressure in the saturation curve is subdivided.

nhy

Number of intervals into which the pressure in the hypercritical domain is subdivided.

For the tests:

"DESS"

Allows to draw a cross-section of the tables in the plane (temperature, pressure).

"PERF"

Allows to output on a file (logical unit 7) the tables of water properties.

"TEST"

This keyword provides a trace of the work performed by the algorithm that searches the thermodynamic parameters of the one-phase or two-phase water, by giving an initial and a final state.

"FINT"

End of the sequence opened by keyword **TEST**.

"CAS" nbr

Number of the treated case. **nbr** is a simple identification number.

p1,t1 or p1,x1

Initial state of the water. If this state is two-phase, it is sufficient to specify **p1** and **x1**. Attention: temperatures are expressed in degrees Celsius, pressures in bar and concentrations are per unit mass.

p2,t2 or p2,x2

Final state of the water. If this state is two-phase, it is sufficient to specify **p2** and **x2**. Attention: temperatures are expressed in degrees Celsius, pressures in bar and concentrations are per unit mass.

dvs,dh

Instead of **p2** and **t2** (or **p2** and **x2**), it is possible to specify the variation of specific volume (in m³/kg) and the variation of specific enthalpy (**dh**) in J/kg.

Warning :

If one intends just to run a test without a real EUROPLEXUS transient calculation, it is preferable to put the keyword "FIN" immediately after the directive "COMPLEMENT". This test is recommended, because it allows to verify the initial conditions (pressure, temperature, void fraction) and to check the composition of the tables.

Comments :

A portion of the saturation curve must be included in the Pressure-Temperature domain chosen.

The **nbte** temperature intervals start from the minimum water temperature to the saturation temperature (for the minimum pressure). It is the same for the **nhte** intervals between the saturation temperature and the maximum temperature, for the maximum pressure.

The **nsat** pressure intervals lie between minimum pressure and maximum pressure, if this is in the sub-critic domain ; else, they go from the minimum pressure to the critical pressure.

In the case of a maximum pressure above 221 bars (hypercritical domain), a further parameter is needed : the number **nhy** of intervals between the critical pressure and the maximum pressure.

For low temperatures ("DBTE"), the subdivision is linear in the temperature. Along the saturation curve ("DSAT"), the subdivision is initially linear in the temperature, then linear in the pressure, in order to obtain a regular subdivision along this curve. Beyond the critical point ("DHTE" and "DPHY"), the subdivision is logarithmic in temperature and in pressure.

When the "TEAU" directive is used, the pressure must be between 0.0062 bar and 1000 bar, and the temperature between 0 and 800 degrees Celsius.

If the extended tables are used (directive "TH2O"), the pressure must be between 0.0062 bar and 30000 bar, and the temperature between 0 and 2000 degrees Celsius.

If the user enters his data in the SI unit system, it is `c1=cm=1`. Otherwise, `c1` or `cm` represent the value of the SI unit expressed in the user's unit. For example, if the lengths are in mm, then `c1=1000`.

8.1.22 HELIUM TABLES

Object :

These directives create tables containing the physical properties of helium, according to CEA/IRF/DPC (1985).

Syntax :

```
"THEL"      "UNIL"  c1      "UNIM"  cm
```

For the tests:

```
< "TEST" <"DETA" > ( "CAS" ... ) "FINT" >
```

With :

```
"CAS"  num      "P1" p1 $ "T1" t1 ; "X1" x1 $
          $ "P2" p2 $ "T2" t2 ; "X2" x2 $ $
          $ "DVS" dvs      "DH" dh          $
```

c1

Conversion factor of the length units adopted, towards metres.

cm

Conversion factor of the mass units adopted, towards kilograms.

For the tests:

"TEST"

This keyword provides a trace of the work performed by the algorithm that searches the thermodynamic parameters of the one-phase or two-phase helium, by giving an initial and a final state.

"FINT"

End of the sequence opened by keyword TEST.

"CAS" nbr

Number of the treated case. **nbr** is a simple identification number.

p1,t1 or p1,x1

Initial state of helium. If this state is two-phase, it is sufficient to specify **p1** and **x1**. Attention: temperatures are expressed in degrees Kelvin, pressures in bar and concentrations are per unit mass.

p2,t2 or **p2,x2**

Final state of helium. If this state is two-phase, it is sufficient to specify **p2** and **x2**. Attention: temperatures are expressed in degrees Kelvin, pressures in bar and concentrations are per unit mass.

dvs,dh

Instead of **p2** and **t2** (or **p2** and **x2**), it is possible to specify the variation of specific volume (in m^3/kg) and the variation of specific enthalpy (**dh**) in J/kg.

Warning :

If one intends just to run a test without a real EUROPLEXUS transient calculation, it is preferable to put the keyword "FIN" immediately after the directive "COMPLEMENT". This test is recommended, because it allows to verify the initial conditions (pressure, temperature, void fraction).

Comments :

The pressure must be above 0.042 bar, and the temperature above 2.1 Kelvin.

The critical point of helium is at 2.27 bars and 5.19 Kelvin. Beyond these values there is only one phase.

Only the gas and the liquid are considered. In the case of high pressures and very low temperatures, the latter must be higher than the melting temperature: solid \rightarrow liquid.

Unlike the water tables (page C.74), the thermodynamic parameters of helium are directly calculated starting from the interpolation polynomials. Therefore, calculations may at times become very time-consuming.

If the user enters his data in the SI unit system, it is **c1=cm=1**. Otherwise, **c1** or **cm** represent the value of the SI unit expressed in the user's unit. For example, if the lengths are in mm, then **c1=1000**.

8.1.23 PIPE JUNCTIONS

Object :

Join together different branches of a pipeline.

Syntax:

```
"RACC" |[ "BIFU" ;
          "CAVI" ;
          "BREC" ]|

          $[          n1 ... nk                      ;
          "LECT" racname "TERM" "LECT" P1 ....PK "TERM" ]$

          ....    "DSOR" d1...dk < "VOLU" v >
```

n1 ... nk

Numbers of the nodes connected by the junction (the order is irrelevant).

racname

LECTURE of the name of the junction element in GIBI.

P1....PK

LECTURE of the names of the connected points as given in GIBI.

d1 ... dk

Internal diameters of the pipelines joined together (the order must correspond to that of the node numbers or names given above).

v

Volume of the junction: mandatory for a cavity, useless for a bifurcation or a pipeline rupture.

Comments:

One may distinguish two cases:

”BIFU” or ”BREC” : bifurcation with a small volume (acoustic continuity)

”CAVI” : cavity with a large volume (with a law describing its behaviour)

In the case of a bifurcation or a pipeline rupture, EUROPLEXUS re-computes a fictitious volume, which corresponds to the sphere having the same area as the sum of the areas of the branches that arrive at the bifurcation.

The exact number of junction elements must be specified in the "GEOM" directive (see page B.30), and the order of junctions is the same as in the "GEOM" directive if GIBI is not used.

Example:

```
"GEOM"      . . .  "CAVI"  2  "BIFU"  1  "BREC"  1  "TERM"
```

corresponds to:

```
"CAVI"  ....  )   2 cavities
"CAVI"  ....  )
"BIFU"  .....  1 bifurcation
"BREC"  .....  1 pipeline rupture
```

If GIBI is used, the number of junction elements specified may be larger than the exact number, and the order is not compulsory, because the name of each junction is specified in the directive.

Example:

```
"GEOM"  ...  "CAVI" cav_one "CAVI" cav_two "BIFU" my_bif "BREC" my_bre ... "TERM"
```

corresponds to (in the RACC directive) :

```
"CAVI"  "LECT" cav_one "TERM" "LECT" P1      "TERM" ....
"CAVI"  "LECT" cav_two "TERM" "LECT" P2 P3    "TERM" ....
"BIFU"  "LECT" my_bif  "TERM" "LECT" P4 P5 P6 "TERM" ....
"BREC"  "LECT" my_bre  "TERM" "LECT" P7 P8    "TERM" ....
```

8.1.24 TUBM (3D-1D JUNCTION)

Object:

To connect, by means of a "TUBM" element, a pipeline meshed in 1-D with a fluid meshed in 3-D.

Syntax:

```
"RACC" ( "TUBM" /LECTURE/  "NTUB" /LECTURE/  "DTUB" dtub  ...  
        "FACE" /LECTURE/  "COEF" coef  )
```

"TUBM" /LECTURE/

The /LECTURE/ procedure allows to specify the name of the GIBI object associated with the junction element.

"NTUB" /LECTURE/

The /LECTURE/ procedure allows to specify the name of the GIBI object associated with the 1D node of the tube.

dtub

Internal diameter of the connected tube.

"FACE" /LECTURE/

The /LECTURE/ procedure allows to specify the name of the GIBI object associated with nodes of the 3D face.

coef

This coefficient allows to take into account of possible symmetries in the 3D mesh. The area of the face meshed in 3D is multiplied by **coef** in order to find out the same area as that of a non-symetrised face.

Comments :

These elements are created by CASTEM, by means of the following syntax:

```
mon_tubm = MANU SUPERELEMENT (p_tube ET s_face) ;
```

where **p_tube** is the object corresponding to the 1D point, and **s_face** the object corresponding to the nodes of the 3D face. All nodes of the 3D face must be coplanar.

"TUBM" connects the fluid of the continuum elements (3D) with the fluid of a "TUBE" element (continuity of the mass flow rate). The velocities of nodes belonging to the 3D face are all equal and normal to the face itself.

The type of elements whose face(s) participate in forming the 3D face is irrelevant: therefore it is possible to use cubes, prisms or even tetrahedra for the mesh.

A material must be associated to the "TUBM" element, although this has no behaviour law.

Warning:

It is mandatory to specify in the dimensioning the parameter "JONC", in order to reserve the space indispensable for the relations associated to the junction (see page A.80).

Do not forget to mention "TUBM" also in the "LIAISON" directive (page D.200).

8.1.25 TUYM (3D-1D JUNCTION)

Object:

To connect, by means of a "TUYM" element, a pipeline meshed ("TUYA" element) in 1-D with a fluid meshed in 3-D for moving meshes (A.L.E computation).

Syntax:

```
"RACC" ( "TUYM" /LECTURE/ "NTUB" /LECTURE/ "DTUB" dtub ...  
        "FACE" /LECTURE/ "COEF" coef )
```

"TUYM" /LECTURE/

The /LECTURE/ procedure allows to specify the name of the GIBI object associated with the junction element.

"NTUB" /LECTURE/

The /LECTURE/ procedure allows to specify the name of the GIBI object associated with the 1D node of the tube ("TUYA" element).

dtub

Internal diameter of the connected tube ("TUYA" element).

"FACE" /LECTURE/

The /LECTURE/ procedure allows to specify the name of the GIBI object associated with nodes of the 3D face.

coef

This coefficient allows to take into account of possible symmetries in the 3D mesh. The area of the face meshed in 3D is multiplied by `coef` in order to find out the same area as that of a non-symetrised face.

Comments :

These elements are created by CASTEM, by means of the following syntax:

```
mon_tuym = MANU SUPERELEMENT (p_tuya ET s_face) ;
```


where `p_tuya` is the object corresponding to the 1D point, and `s_face` the object corresponding to the nodes of the 3D face. All nodes of the 3D face must be coplanar.

"TUYM" connects the fluid of the continuum elements (3D) with the fluid of a "TUYA" element (continuity of the mass flow rate). The velocities of nodes belonging to the 3D face are all equal and normal to the face itself.

The type of elements whose face(s) participate in forming the 3D face is irrelevant: therefore it is possible to use cubes, prisms or even tetrahedra for the mesh.

A material must be associated to the "TUYM" element, although this has no behaviour law.

Warning:

It is mandatory to specify in the dimensioning the parameter "JONC", in order to reserve the space indispensable for the relations associated to the junction (see page A.80).

Do not forget to mention "TUYM" also in the "LIAISON" directive (page D.200).

8.1.26 CORRESPONDENCE BETWEEN NODES

Object:

The purpose of this directive is to define a one-to-one correspondence between couples of nodes. This user-defined correspondence may be useful in various situations, in which the code needs to find a one-to-one correspondence between nodes in the mesh and the automatic determination of such a correspondence is impossible. For example, this might happen under exceptional circumstances in the following cases:

- In the search for structural (Lagrangian) nodes correspondent to fluid nodes in the FSA fluid-structure interaction directive, see page D.450.
- In the search for structural nodes corresponding to fluid nodes in the model for perforated plates IMPE PPLT, see page C.760.
- In the search for structural nodes corresponding to fluid nodes in the model for rupture disks IMPE RUDI, see page C.770.
- In the search for structural nodes corresponding to fluid nodes in the model for rupture disks for the MC formulation, IMPE RDMC, see page C.790.

In such cases, the code tries to automatically determine the structural (or other Lagrangian) node “corresponding” to a certain fluid node. This node is defined as the Lagrangian node having the same initial coordinates as the fluid node under consideration, within a certain small tolerance (that may be changed via the OPTI TOLC, page H.40). If there is no such node or if more than one candidate node is found (e.g. because there are several superposed structures in the mesh), then the automatic search would fail. In this case, the user may assume control by explicitly specifying the corresponding Lagrangian node to each “ambiguous” fluid node.

It is advised to use this directive only in case of necessity. First, an input without this directive should be prepared. Then, in case the code produces some error messages related to the impossibility of automatically determining the node correspondence, the present directive may be added to resolve the identified conflicts.

Syntax:

```
"CNOD"    "NODF"  /LECT1/    "NODS"    /LECT2/

/LECT1/

List of first nodes of each node couple. Typically, these are fluid nodes.

/LECT2/
```

List of second nodes of each node couple. These nodes must be Lagrangian. Typically, these are structure nodes, but Lagrangian fluid nodes are also accepted.

Comments:

The order in which nodes are listed in `/LECT1` or `/LECT2` is retained. To the i -th node of `/LECT1` corresponds the i -th node of `/LECT2`. The number of nodes in `/LECT1` and `/LECT2` must be the same.

Note that the directive `CNOD` may be specified only once in each calculation (i.e. it should not be repeated). In other words, all correspondent nodes should be specified in just one `/LECT1/` and `/LECT2/`.

In case of problems with the `FSA` directive, please note that another way of resolving node conflicts, alternative to the present `CNOD` directive, is the `STRU` sub-directive of `FSA`, see page D.450, which is more practical in case there is a large number of conflicting nodes.

8.1.27 SPH SHELL ELEMENT (SPHC)

Object :

This instruction introduces characteristics for the SPH shell elements (SPHC) which allow discretizing shell structures with a single layer of particles.

Syntax:

```
"CSPH"   "RAYO" rbille  "EPAI" ep
          "ORX" orx  "ORY" ory  "ORZ" orz
          < "LINE" c1 >   < "QUAD" cq >
          < "RLIM" rlim > < "RESEAU" ires >
          < "VOIS" nvoi >
          ( "STRP" istrp /LECT/ )
```

rbille

Radius of the SPH shell particles.

ep

Thickness of the shell particles.

orx, ory, orz

x,y,z co-ordinates of a point used to orient normals of the SPH shell particles.

c1

Linear damping coefficient.

cq

Quadratic damping coefficient.

rlim

Multiplicative coefficient for the search radius.

ires

Type of particles lattice (1: cubic, 0: hexagonal).

nvoi

Number of neighbouring particles sought.

istrp

Type of stress points (1: free, 2: clamped) read in the following /LECTURE/ sequence (see comment below).

Comments :

For the quadratic damping, it is advised to take $cq=4$.

To damp out the high-frequency oscillations it is advisable to use a value of cl between 0.1 and 0.5.

At least one set of stress points must be entered. Several sets can be entered by repeating the STRP keyword.

Two types of particle lattice are possible: for $ires = 1$ a cubic lattice is adopted; in the case $ires = 0$ (default value), a compact hexagonal lattice is adopted.

The number of sought neighbouring particles is by default 12. This number may not be changed for the PEF algorithm. Its modification is accepted only for the SPH method.

For a given particle, the search considers the neighbours whose center is within a distance of $r_{lim} \cdot r_{bille}$ from its center. By default, $r_{lim}=1.3$.

8.1.28 DISCRETE ELEMENT (ELDI)

Object :

This instruction is mandatory in the input file when using discrete elements (ELDI). It allows printing out to the output listing the value of the radius of each discrete element and to impose the correct masses of different parts of the discrete element model (element density will be corrected).

Syntax:

```
"CELDI"  < "IMPR" >
          < "MASSE" nval  nval*(val /LECTURE/ ) >
          < "ARMA"  /LECTURE/ >
          < "LTM"   nbse  nbse*(beta plas /LECTURE/) >
          < "CSTE"   coef  >
          < "EDEF"  nbcoup
                        nbcoup*("NCOU"  ncouches
                                "ELDI"   /LECTURE/
                                "FRON"   /LECTURE/ ) >
```

"IMPR"

This optional keyword allows printing out in the output listing the value of the radius of each discrete element.

"MASSE"

This optional keyword enables the user to impose the masses of discrete elements lists.

nval

Number of imposed masses.

val

Value of the imposed mass.

"ARMA"

Indicates the presence of steel reinforcement modeled with aligned elements. Caculates the main direction of the reinforcement used

LECTURE

List of the discrete elements concerned.

"LTM"

Indicates the presence of bending properties (rotation stiffnesses for discrete elements).

nbse

Number of sequences with different bending properties.

beta

Coefficient used to calculate the bending stiffness: $K_r = \beta EI/R$.

plas

Coefficient used to calculate the plastic torque: $M_p = \text{plas} \cdot \sigma \cdot I/R$. In elastic calculations, one should use $\text{plas}=0$ (no test on M_p) or put $\text{plas} \gg 1$ to guarantee M_p is very high.

LECTURE

List of the discrete elements concerned.

"CSTE"

This optional keyword enables the user to define the security coefficient of the time step

coef

Security coefficient (by default 0.1) : $\text{dt} = \text{coef} \cdot \text{dtcrit}$

Comments :

To guarantee the masses of different parts of the discrete element model are correct, each discrete element should belong to only one group.

8.1.29 CHARACTERISTICS OF A MULTILAYER ELEMENT CMC3

Object:

The characteristics of CMC3 elements are described when they have not been defined by CASTEM2000.

Syntax:

```
"CORTHO"    "EPAISSEUR"  ep      "EXCENTREMENT"  ex
              $[  "ANGLE"    angle      ;
                  "VECTEUR"   vx  vy  vz  ]$          /LECTURE/
```

ep

Thickness of the element.

ex

Element eccentricity with respect to the plane defined by the 3 nodes of the mesh.

angle

Angle (in degrees) formed by the first side of the element and the first axis of the orthotropic system.

vx,vy,vz

The 3 components in the global frame of the vector that defines the first orthotropy direction.

LECTURE

List of the elements concerned.

Comments:

The sign of the excentricity is defined by the orientation of the normal. This depends on the numbering of the nodes of the CMC3 element (see Maxwell's cork-screw rule).

The first side of the element is the one formed by the first 2 nodes.

8.1.30 CHARACTERISTICS OF ORTHOTROPY

Object:

Description of the orthotropy directions for continuum elements in 2D and 3D.

Syntax:

```
"MORTHO"  $[ "ALPHA" angle1                      ;
            "TETHA" angle2                          ;
            "AXE1"  e11 e12 e13  "AXE2" e21 e22 e23  ;
            "COCY"  "POINT" $[ /LECTURE1/  ;
                        xx yy zz  ]$ "VECT" v1 v2 v3  ;
            "V1LC"  v1x v2x v3x  "V2LC" v2x v2y v2z  ]$
            /LECTURE/
```

angle1

Angle (in degrees) formed by the Ox axis (in 2D plane cases or 3D) or the Or axis (in axisymmetric) and the first axis of the orthotropy reference frame.

angle2

Angle (in degrees) formed by the first side of the element and the first orthotropy axis (in 2D or axisymmetric). The first side of the element is the segment connecting the first two nodes declared for the element in the GEOM directive.

e11, e12, e13

First vector defining the orthotropy plane of the material.

e21, e22, e23

Second vector defining the orthotropy plane of the material.

COCY

This directive allows to define a “cylindrical” type of orthotropy, that may be used for example by the BOIS (wood) material. The first axis of orthotropy is parallel to the vector defined by the VECT directive described above. The second axis of orthotropy (perpendicular to the first one) lies on the plane formed by a straight line passing through the POINT defined below and parallel to VECT, and the barycenter of the element.

POINT

This directive allows to define a point which is either a node of the mesh (option /LECTURE1/), or a geometric point defined by its three coordinates (xx yy zz).

v1 v2 v3

First vector defining the orthotropy reference frame of the COCY directive.

v1x, v1y, v1z

First vector defining the orthotropy plane of the material in the local repere of the element.

v2x, v2y, v2z

Second vector defining the orthotropy plane of the material in the local repere of the element.

LECTURE

List of the elements concerned.

Comments:

One can define several orthotropy directions by repeating each time the keyword MORT. It is also possible to repeat it starting from different items.

The ALPHA or TETHA keywords are used in 2D, the AXE1 ... AXE2 or COCY directive are used in 3D.

The vectors V1(e11,e12,e13 or v1x,v1y,v1z) and V2(e21,e22,e23 or v1x,v1y,v1z) are not necessarily unit vectors, and V2 is not necessarily normal to V1.

Starting from these inputa data, EUROPLEXUS computes and stores the values in the local reference frames relatives to each element. These local values will be utilised during the transient calculation. For this reason, the calculation remains valid also for large rotations.

8.1.31 ORTHOTROPY CHARACTERISTICS FOR 3D SHELLS (JRC)

Object:

Description of the orthotropy directions for 3D (layered) shell elements (JRC models). The parameter **ANGLE** can only be used with Q4GS, DST3, Q4MC and DST3 elements associated with HILL or ORTS material.

Syntax:

```
"ORTS" |[ vx vy vz ; "ANGLE" alpha ]| /LECT/    < "LAYE" /LECT_LAY/>
```

vx, vy, vz

Components, in the global reference frame, of a vector whose projection on the lamina (local) coordinate system of the 3D shell element indicates the orthotropy direction (one such direction is thus sufficient, for shell elements).

alpha

Angle between the first direction of the shell element and the first direction of the orthotropic frame.

/LECT/

Concerned elements.

/LECT LAY/

Concerned layers. Layers are identified by their indexes, as described in the **SAND** directive on page C.45.

Comments:

Note that the directive **COMP ORTS** must be specified *after* the definition of the material characteristics (**MATE** directive). If other quantities (e.g. thickness, etc.) are to be specified via the **COMP** directive, then *two* **COMP** directives should be used: the first one, immediately after the **GEOM** directive, and the second one (**COMP ORTS**) immediately after the **MATE** directive.

8.1.32 FRICTION CHARACTERISTICS

Object :

Characterisation of friction for sliding lines and sliding surfaces.

Syntax:

```
"FROT"  iglis  "MU0"  mu0  "MU1"  mu1  "GAMMA" gamma
```

iglis

Number of the couple of sliding lines or surfaces for which the friction is being specified.

mu0

Static friction coefficient.

mu1

High-velocity friction coefficient.

gamma

Coefficient of the friction law.

Comments :

Friction law of the form:

$$\begin{array}{c} \text{-----} \\ | \quad -> \qquad \qquad \qquad -> \quad | \\ | \quad F = MU \cdot N \cdot T \quad | \\ | \qquad \qquad \qquad \qquad \qquad | \\ \text{-----} \end{array}$$

$$\text{with} \quad MU = MU1 + (MU0 - MU1) \cdot E^{-GAMMA \cdot V}$$

V : modulus of the relative velocity
N : contact force along the normal
-> : tangent vector
T : tangent vector

One may define several friction laws for different couples of sliding lines or surfaces, by repeating for each couple the keyword "FROT".

8.1.33 PARTICLE ELEMENT (BILLE)**Object :**

Description of the characteristics of the BILLE element (particle element).

Syntax:

```
"CBILLE"  "RAYON"  rbille  < "LINEAIRE"  cl  >  ...  
... < "QUADRATIQUE" cq > < "RESEAU"    ires >  ...  
... < "VOISIN"      nvoi >  
... < "RLIM"        rlim >
```

rbille

Radius of the particles.

cl

Linear damping coefficient.

cq

Quadratic damping coefficient.

ires

Type of particles lattice.

nvoi

Number of neighbouring particles sought.

rlim

Multiplicative coefficient for the search radius.

Comments :

For the quadratic damping, it is advised to take $cq=4$.

To damp out the high-frequency oscillations it is advisable to use a value of cl between 0.1 and 0.5.

Two types of particle lattice are possible: for $ires = 1$ a cubic lattice is adopted; in the case $ires = 0$ (default value), a compact hexagonal lattice is adopted.

The number of sought neighbouring particles is by default 12. This number may not be changed for the PEF algorithm. Its modification is accepted only for the SPH method.

For a given particle, the search considers the neighbours whose center is within a distance of $r_{lim} * Diameter$ from its center. By default, $r_{lim}=1.3$.

9 GROUP C1—MATERIALS

Object:

These directives enable the user to specify the materials.

Syntax:

"MATE" . . .

Comments:

These directives are described in detail on the following pages.

Do not forget the corresponding dimensioning (page A.70).

9.1 GENERALITIES ABOUT MATERIALS

Object:

This instruction enables the user to enter the properties of various materials.

Syntax:

```
"MATE"    (  < "LOI" numldc > . . . )
```

LOI

This keyword announces that a number will be assigned to the constitutive law whose definition follows.

numldc

Number of the constitutive law.

Comments:

The word "MATE" is compulsory and may only be used once, at the beginning of the data sequence relative to the instruction MATERIALS.

The numbers introduced by the "LOI" directive may be in arbitrary order, and some numbers may be missing. This is very useful in the case of multiple materials: one can add or move material data in the input file without changing the number of the corresponding material law (see "MULT", page C.380).

If the "LOI" directive is absent, the number automatically attributed to the law by EUROPLEXUS is the index of the material in the order its constitutive law is listed in the input data.

The material models are:

number	name	law of behaviour
1	LINE	linear elasticity
2	PARF	perfectly plastic Von Mises
3	HILL	Isotropic plasticity associated with a HILL criterion and with a orthotropic elastic behavior
4	ISOT	isotropic Von Mises
5	TETA	Von Mises dependent upon temperature
6	POST	post-rupture (beton)
7	FLUI	isothermal fluid (c = cte)
8	CAVI	isothermal fluid with cavitation
9	GAZP	perfect gas
10	NAH2	sodium-water reaction
11	BETO	concrete (NAHAS model)
12	DRUC	Drucker-Prager
13	LIBR	free (material defined by the user)
16	IMPE	impedance
17	FANT	phantom: ignore the associated elements (see note)
18	ODMS	Onera Damage Mechanic (composite)
19	DYNA	dynamic Von Mises isotropic rate-dependent
20	BLMT	DYNAR LMT Concrete
22	EAU	two-phase water (liquid + vapour)
23	LIQU	incompressible (or quasi-) fluid
24	SOUR	imposed pressure in a continuum element
25	MULT	multiple materials (coupled monodim.)
26	MASS	mass of a material point
27	FLFA	rigit tube bundles
28	HELI	helium
29	BL3S	reinforced concrete for discretes elements
30	STGN	Steinberg - Guinan
31	MOTE	motor force or couple (meca)
32	ASSE	motor "asservissement" (meca)
33	MHOM	homogeneization
34	ADCR	containment accident (fast neutrons)
35	VM23	Von Mises elasto-plastic radial return
36	FLUT	fluid, to be specified by the user
37	VM1D	material for elements of type "ED1D"
38	DONE	viscoplastic material
39	PUFF	material of type "PUFF"
40	GZPV	perfect gas for Van Leer
41	ORTH	linear orthotropic in user system
42	ORTE	elastic damage orthotropy
43	IMPV	impedance Van Leer

45	MECA	mechanism associated to articulated systems
46	ORTS	linear orthotropic with local reference frame
47	ADFM	advection-diffusion fluid
48	GVDW	Van Der Waals gas
49	EXVL	hydrogen explosion Van Leer
50	JWL	explosion (Jones-Wilkins-Lee model)
51	CHOC	shock waves, Rankine-Hugoniot equation
52	GPDI	diffusive perfect gas Van Leer
53	ADCJ	hypothetical core disruptive accident with law of type JWL for the bubble
54	RSEA	reaction sodium-water with three constituents
56	PARO	friction and heat exchange for pipeline walls
57	BILL	LIBRE (user's free particle material), or FLUIDE (isothermal fluid particle: $c = cte$)
58	CRIT	damage criteria calculation : PY (damage of type P/Y) DUCTile (ductile damage)
59	BUBB	
60	MCOU	Linear multi-layer homogenised through the thickness
61	BIRD	fluid bird
64	CHAN	Multi-layer with the CHANG-CHANG criterion
66	JWLS	Explosion (Jones-Wilkins-Lee for solids)
67	ZALM	Zerilli-Armstrong with damage Lemaitre-Chaboche
69	PRGL	Porous jelly for the particles
70	LMC2	Von Mises isotropic coupled with damage (Lemaitre) with strain-rate sensitivity
71	APPU	Material for elements of type PPUI
72	LEM1	Von Mises isotropic coupled with damage (type Lemaitre)
74	ABSE	
75	BOIS	Materiau BOIS (WOOD) servant d'amortir les chocs (utilise en compression)
76	VMJC	Johnson-Cook
77	VMZA	Zerilli-Armstrong
78	VMLP	Ludwig-Prandtl
79	VMLU	Ludwik
80	FUNE	specialized cable material (no compression resistance)
81	MCGP	multicomponent fluid material (perfect gas)
82	MCFF	multicomponent fluid material (far-field)
83	DRPR	Drucker Prager Ispra model
84	VMSF	Von Mises with softening and viscoplastic regularization
85	MAZA	Mazars (under development)

86		FLMP		Fluid multi-phase
87		DPSF		Drucker Prager with softening and viscoplastic
				regularization
88		COMM		Composite material (linear orthotropic), Ispra
				implementation
89		CAMC		Modified Cam-clay material
90		CLAY		Modified Cam-clay material (backward fully
				implicit algorithm (radial return), and
				viscoplastic regularization)
91		JPRP		for bushing elements
92		GLRC		Plasticity with kinematic softening for
				orthotropic shells. Global plastic criterion.
93		FOAM		Aluminium foam (for crash simulations)
94		SUPP		support
95		HYPE		Hyperelastic material (Model of Mooney-Rivlin,
				Hart-Smith and Ogden)
96		PBED		particle bed
97		MINT		Material for interface element
98		TVMC		elastoplastic short fibers with damage
99		SLZA		Steinberg-Lund-Zerilli-Armstrong
100		CRTM		Composite manufactured by RTM process
101		TAIT		
102		STIF		
103		SG2P		
104		SGMP		
105		EOBT		
106		SMAZ		Mazars Damage for SPHC
107		SLIN		Linear Damage for SPHC
108		JCLM		Johnson-Cook with Damage Lemaitre-Chaboche for SPHC
109	!	DADC	!	Dynamic Anisotropic Damage Concrete
----- ----- -----				

The "FANT" material may be allocated to any element, with the effect of 'eliminating' it from the mesh, as far as mechanical resistance is concerned.

The different elements may use the following materials (defined by their numbers):

AVAILABLE MATERIALS FOR EACH ELEMENT

=====

NO. | ELEMENT | AVAILABLE MATERIALS

----|-----|-----

1		COQU		LINE PARF ISOT TETA DYNA ORTH
2		TRIA		LINE PARF ISOT TETA POST FLUI CAVI GAZP NAH2 BETO
				DRUC DYNA EAU LIQU SOUR MULT FLFA STGN ADCR VM23

			PUFF ORTH JWL CHOC ADCJ RSEA CRIT JWLS ZALM LMC2
			LEM1 BOIS VMJC VMZA VMLP VMLU VMSF DPSF CAMC CLAY
3		BARR	LINE PARF ISOT DYNA
4		PONC	LINE PARF ISOT
5		MEMB	LINE
6		LAYR	VM23
7		CL2D	IMPE CLVF IMPV
8		CAR1	LINE PARF ISOT TETA POST FLUI CAVI GAZP NAH2 BETO
			DRUC DYNA EAU LIQU SOUR MULT FLFA STGN ADCR VM23
			DONE PUFF ORTH JWL CHOC ADCJ RSEA CRIT BIRD JWLS
			ZALM LMC2 LEM1 BOIS VMJC VMZA VMLP VMLU DRPR VMSF
			DPSF CAMC CLAY
9		CAR4	LINE PARF ISOT TETA POST FLUI CAVI GAZP BETO DRUC
			DYNA EAU MULT FLFA STGN VM23 DONE PUFF ORTH GLAS
			CHOC ADCJ CRIT LSGL ZALM LMC2 LEM1 BOIS VMJC VMZA
			VMLP VMLU DRPR VMSF DPSF CAMC CLAY HYPE
10		COQC	LINE PARF ISOT ORTH
11		CUBE	LINE ISOT TETA FLUI GAZP NAH2 DRUC DYNA BLMT
			EAU LIQU SOUR MULT FLFA STGN ADCR VM23 PUFF ORTH
			ORTE GLAS ORTS JWL CHOC ADCJ RSEA ORPE CRIT BUBB
			BIRD LSGL JWLS ZALM LMC2 LEM1 BOIS VMJC VMZA VMLP
			VMLU DRPR VMSF MAZA DPSF CAMC CLAY FOAM HYPE PBED
			TVMC SLZA CRTM EOBT DADC
12		COQ3	LINE ISOT TETA DYNA MCOU CHAN
13		CUB6	LINE ISOT TETA FLUI GAZP DRUC DYNA BLMT MULT STGN
			VM23 PUFF ORTH ORTE GLAS ORTS CHOC ORPE CRIT BUBB
			LSGL BOIS VMJC DRPR VMSF DPSF CAMC CLAY FOAM HYPE
			SLZA CRTM
14		COQ4	LINE ISOT TETA DYNA MCOU CHAN
15		FS2D	
16		FS3D	
17		POUT	LINE ISOT
18		CL3D	IMPE CLVF
19		BR3D	LINE PARF ISOT DYNA
20		PR6	LINE HILL ISOT TETA FLUI GAZP BETO DRUC DYNA
			BLMT MULT VM23 PUFF ORTH ORTE ORTS ORPE CRIT BUBB
			ZALM LMC2 LEM1 BOIS VMJC DRPR VMSF MAZA DPSF CAMC
			CLAY FOAM HYPE SLZA CRTM EOBT DADC
21		TETR	LINE HILL ISOT TETA FLUI GAZP NAH2 DRUC DYNA
			BLMT EAU LIQU SOUR MULT FLFA ADCR VM23 PUFF ORTH
			ORTE ORTS JWL CHOC ADCJ RSEA ORPE CRIT BUBB BIRD
			JWLS ZALM BOIS VMJC VMSF MAZA DPSF CAMC CLAY FOAM
			HYPE SLZA CRTM EOBT DADC
22		TUBE	FLUI GAZP NAH2 EAU LIQU SOUR MULT HELI ADCR GVDW
			RSEA PARO
23		TUYA	LINE ISOT FLUI GAZP NAH2 EAU LIQU SOUR MULT HELI

[illegible]

55		FL2S		FLUT BUBB FLMP
56		ED41		VM23 DONE VMSF DPSF
57		ADC8		ADFM
58		ADQ4		ADFM
59		FL3S		FLUT BUBB FLMP
60		CL2S		IMPE FLUT
61		CL3S		IMPE FLUT
62		CL32		IMPE
63		CL33		IMPE
64		FL23		FLUT BUBB FLMP
65		FL24		FLUT BUBB FLMP
66		FL34		FLUT BUBB FLMP
67		FL35		FLUT BUBB FLMP
68		FL36		FLUT BUBB FLMP
69		FL38		FLUT BUBB FLMP
70		CL22		IMPE FLUT IMPV MCFF
71		Q41		VM23 DONE LSGL VMSF DPSF
72		Q42		VM23 DONE LSGL VMSF DPSF
73		Q41N		VM23 DONE LSGL VMSF DPSF
74		Q42N		VM23 DONE LSGL VMSF DPSF
75		Q41L		VM23 DONE GLAS VMJC VMZA VMLP DRPR VMSF DPSF CAMC
				CLAY
76		Q42L		VM23 DONE GLAS LSGL VMJC VMZA VMLP DRPR VMSF DPSF
				CAMC CLAY
77		Q95		VM23 DONE
78		CL3I		IMPE FLUT MCFF
79		BILL		LINE ISOT PUFF BILL PRGL MAZA
80		ELDI		BL3S
81		CUVL		GZPV GVDW EXVL GPDI
82		PRVL		GZPV GVDW EXVL GPDI
83		DST3		LINE HILL ISOT TETA DYNA VM23 DONE GLAS ORTS MCOU
				LSGL CHAN LEM1 VMJC VMZA VMLP VMLU VMSF DPSF HYPE
				SLZA
84		DKT3		LINE ISOT DYNA VM23 DONE GLAS MCOU LSGL CHAN LEM1
				VMJC VMZA VMLP VMLU VMSF DPSF GLRC SLZA
85		SHB8		LINE ISOT DYNA ZALM LMC2 LEM1 SLZA
86		XCUB		LINE PARF ISOT ODSM ORTE VMJC VMSF
87		XCAR		LINE PARF ISOT VMJC VMSF
88		PROT		LINE ISOT
89		SPHC		LINE ISOT LEM1 SMAZ SLIN JCLM
90		Q4G4		LINE ISOT
91		CQD4		VM23 LSGL VMSF DPSF COMM
92		CQD9		VM23 LSGL VMSF DPSF COMM
93		CQD3		VM23 LSGL VMSF DPSF COMM
94		CQD6		VM23 LSGL VMSF DPSF COMM
95		CLD3		IMPE

96		CLD6		IMPE	
97		MC23		MCGP	
98		MC24		MCGP	
99		CL3Q		IMPE FLUT MCFF	
100		Q42G		VM23 DONE	
101		MC34		MCGP	
102		MC35		MCGP	
103		MC36		MCGP	
104		MC38		MCGP	
105		MS24		LINE	
106		S24		LINE	
107		MS38		LINE	
108		S38		LINE	
109		FUN2		VM23 VMJC VMZA VMLP VMLU FUNE	
110		FUN3		VM23 FUNE	
111		Q4GR		LINE HILL ISOT TETA DYNA VM23 DONE GLAS MCOU LSGL	
				CHAN LEM1 VMJC VMZA VMLP VMLU VMSF DPSF GLRC SLZA	
112		Q4GS		LINE HILL ISOT TETA DYNA VM23 DONE GLAS ORTS MCOU	
				LSGL CHAN LEM1 VMJC VMZA VMLP VMLU VMSF DPSF GLRC	
				HYPE SLZA	
113		(free)			
114		BSHT		JPRP	
115		BSHR		JPRP	
116		TUYM		FLUI GAZP NAH2 EAU LIQU ADCR RSEA	
117		SH3D		JPRP	
118		MAP2			
119		MAP3			
120		MAP4			
121		MAP5			
122		MAP6			
123		MAP7			
124		INT4		LINE MINT	
125		INT6		LINE MINT	
126		INT8		LINE MINT	
127		SH3V			
128		MOY4			
129		MOY5			
130		ASHB		LINE ISOT DYNA ZALM LMC2 LEM1 SLZA	
131		T3VF		FLUI GAZP EAU ADCR GVDW JWL ADCJ JWLS CDEM GAZD	
				TAIT STIF SG2P SGMP	
132		Q4VF		FLUI GAZP EAU ADCR GVDW JWL ADCJ JWLS CDEM GAZD	
				TAIT STIF SG2P SGMP	
133		CUVF		FLUI GAZP EAU ADCR GVDW JWL ADCJ JWLS CDEM GAZD	
				TAIT STIF SG2P SGMP	
134		PRVF		FLUI GAZP EAU ADCR GVDW JWL ADCJ JWLS CDEM GAZD	
				TAIT STIF SG2P SGMP	

135		TEVF		FLUI	GAZP	EAU	ADCR	GVDW	JWL	ADCJ	JWLS	CDEM	GAZD
				TAIT	STIF	SG2P	SGMP						
136		PYVF		FLUI	GAZP	EAU	ADCR	GVDW	JWL	ADCJ	JWLS	CDEM	GAZD
				TAIT	STIF	SG2P	SGMP						
137		COQ2		LINE	ISOT	TETA							
138		Q4MC		LINE	HILL	ISOT	DYNA	VM23	ORTS	ORPE	LSGL		
139		T3MC		LINE	HILL	ISOT	DYNA	VM23	ORTS	ORPE	LSGL		
140		DEBR											
141		INS6											
142		INS8											
143		P3ZT		LINE	HILL	ISOT	DYNA	ORTS	ORPE	PIEZ			
144		C272		VM23									
145		C273		VM23									
146		BREC		EAU									
147		TUVF		FLUI	GAZP	EAU	ADCR	GVDW	JWL	ADCJ	JWLS	GAZD	TAIT
				STIF	SG2P	SGMP							
148		TYVF		FLUI	GAZP	EAU	ADCR	GVDW	JWL	ADCJ	JWLS	GAZD	TAIT
				STIF	SG2P	SGMP							
149		BIVF		FLUI	GAZP	EAU	ADCR	GVDW	JWL	ADCJ	JWLS	GAZD	TAIT
				STIF	SG2P	SGMP							
150		CAVF		FLUI	GAZP	EAU	ADCR	GVDW	JWL	ADCJ	JWLS	GAZD	TAIT
				STIF	SG2P	SGMP							

AVAILABLE ELEMENTS FOR EACH MATERIAL

=====

NO.		MATERIAL		AVAILABLE ELEMENTS
----		-----		-----
1		LINE		COQU TRIA BARR PONC MEMB CAR1 CAR4 COQC CUBE COQ3
				CUB6 COQ4 POUT BR3D PR6 TETR TUYA PRIS CUB8 QPPS
				CMC3 T3GS BILL DST3 DKT3 SHB8 XCUB XCAR PROT SPHC
				Q4G4 MS24 S24 MS38 S38 Q4GR Q4GS INT4 INT6 INT8
				ASHB COQ2 Q4MC T3MC P3ZT
2		PARF		COQU TRIA BARR PONC CAR1 CAR4 COQC BR3D XCUB XCAR
3		HILL		PR6 TETR PRIS CUB8 DST3 Q4GR Q4GS Q4MC T3MC P3ZT
4		ISOT		COQU TRIA BARR PONC CAR1 CAR4 COQC CUBE COQ3 CUB6
				COQ4 POUT BR3D PR6 TETR TUYA PRIS CUB8 QPPS CMC3
				BILL DST3 DKT3 SHB8 XCUB XCAR PROT SPHC Q4G4 Q4GR
				Q4GS ASHB COQ2 Q4MC T3MC P3ZT
5		TETA		COQU TRIA CAR1 CAR4 CUBE COQ3 CUB6 COQ4 PR6 TETR
				PRIS CUB8 DST3 Q4GR Q4GS COQ2
6		POST		TRIA CAR1 CAR4
7		FLUI		TRIA CAR1 CAR4 CUBE CUB6 PR6 TETR TUBE TUYA BIFU
				CAVI PRIS CUB8 QAX1 TUBM PFEM TUYM T3VF Q4VF CUVF
				PRVF TEVF PYVF TUVF TYVF BIVF CAVF
8		CAVI		TRIA CAR1 CAR4

9	GAZP		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	TUBE	TUYA	BIFU
			CAVI	PRIS	CUB8	QAX1	TUBM	TUYM	T3VF	Q4VF	CUVF	PRVF
			TEVF	PYVF	TUVF	TYVF	BIVF	CAVF				
10	NAH2		TRIA	CAR1	CUBE	TETR	TUBE	TUYA	BIFU	CAVI	PRIS	TUBM
			TUYM									
11	BETO		TRIA	CAR1	CAR4	PR6	CMC3					
12	DRUC		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	
14	IFS											
16	IMPE		CL2D	CL3D	CL1D	CL3T	CLTU	CL23	CL2S	CL3S	CL32	CL33
			CL22	CL3I	CLD3	CLD6	CL3Q					
18	ODMS		CUB8	XCUB								
19	DYNA		COQU	TRIA	BARR	CAR1	CAR4	CUBE	COQ3	CUB6	COQ4	BR3D
			PR6	TETR	PRIS	CUB8	QPPS	DST3	DKT3	SHB8	Q4GR	Q4GS
			ASHB	Q4MC	T3MC	P3ZT						
20	BLMT		CUBE	CUB6	PR6	TETR	PRIS	CUB8				
21	CLVF		CL2D	CL3D	CL1D	CL3T						
22	EAU		TRIA	CAR1	CAR4	CUBE	TETR	TUBE	TUYA	BIFU	CAVI	PRIS
			TUBM	TUYM	T3VF	Q4VF	CUVF	PRVF	TEVF	PYVF	BREC	TUVF
			TYVF	BIVF	CAVF							
23	LIQU		TRIA	CAR1	CUBE	TETR	TUBE	TUYA	BIFU	CAVI	TUBM	TUYM
24	SOUR		TRIA	CAR1	CUBE	TETR	TUBE	TUYA	BIFU	CAVI	PRIS	
25	MULT		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	TUBE	TUYA	CAVI
			PRIS	CUB8								
26	MASS		PMAT									
27	FLFA		TRIA	CAR1	CAR4	CUBE	TETR	PRIS				
28	HELI		TUBE	TUYA								
29	BL3S		ELDI									
30	STGN		TRIA	CAR1	CAR4	CUBE	CUB6	CUB8				
31	MOTE		MECA									
32	ASSE		MECA									
33	MHOM		FHQ2	FHT2								
34	ADCR		TRIA	CAR1	CUBE	TETR	TUBE	TUYA	BIFU	CAVI	PRIS	TUBM
			TUYM	T3VF	Q4VF	CUVF	PRVF	TEVF	PYVF	TUVF	TYVF	BIVF
			CAVF									
35	VM23		TRIA	LAYR	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8
			QPPS	Q92	Q93	COQI	ED01	Q92A	T3GS	ED41	Q41	Q42
			Q41N	Q42N	Q41L	Q42L	Q95	DST3	DKT3	CQD4	CQD9	CQD3
			CQD6	Q42G	FUN2	FUN3	Q4GR	Q4GS	Q4MC	T3MC	C272	C273
36	FLUT		FLU1	FLU3	FL2S	FL3S	CL2S	CL3S	FL23	FL24	FL34	FL35
			FL36	FL38	CL22	CL3I	CL3Q					
37	VM1D		ED1D									
38	DONE		CAR1	CAR4	QPPS	Q92	Q93	COQI	ED01	Q92A	T3GS	ED41
			Q41	Q42	Q41N	Q42N	Q41L	Q42L	Q95	DST3	DKT3	Q42G
			Q4GR	Q4GS								
39	PUFF		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	BILL
40	GZPV		TVL1	CVL1	CUVL	PRVL						

41		ORTH		COQU	TRIA	CAR1	CAR4	COQC	CUBE	CUB6	PR6	TETR	PRIS
				CUB8	CMC3								
42		ORTE		CUBE	CUB6	PR6	TETR	PRIS	CUB8	XCUB			
43		IMPV		CL2D	CL22								
44		GLAS		CAR4	CUBE	CUB6	CUB8	QPPS	COQI	ED01	T3GS	Q41L	Q42L
				DST3	DKT3	Q4GR	Q4GS						
45		MECA		MECA									
46		ORTS		CUBE	CUB6	PR6	TETR	PRIS	CUB8	DST3	Q4GS	Q4MC	T3MC
				P3ZT									
47		ADFM		ADC8	ADQ4								
48		GVDW		TUBE	TUYA	BIFU	CAVI	TVL1	CVL1	CUVL	PRVL	T3VF	Q4VF
				CUVF	PRVF	TEVF	PYVF	TUVF	TYVF	BIVF	CAVF		
49		EXVL		TVL1	CVL1	CUVL	PRVL						
50		JWL		TRIA	CAR1	CUBE	TETR	PRIS	T3VF	Q4VF	CUVF	PRVF	TEVF
				PYVF	TUVF	TYVF	BIVF	CAVF					
51		CHOC		TRIA	CAR1	CAR4	CUBE	CUB6	TETR	PRIS	CUB8		
52		GPDI		TVL1	CVL1	CUVL	PRVL						
53		ADCJ		TRIA	CAR1	CAR4	CUBE	TETR	PRIS	T3VF	Q4VF	CUVF	PRVF
				TEVF	PYVF	TUVF	TYVF	BIVF	CAVF				
54		RSEA		TRIA	CAR1	CUBE	TETR	TUBE	TUYA	BIFU	CAVI	PRIS	TUBM
				TUYM									
55		ORPE		CUBE	CUB6	PR6	TETR	PRIS	CUB8	Q4MC	T3MC	P3ZT	
56		PARO		TUBE	TUYA	CAVI							
57		BILL		BILL									
58		CRIT		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	
59		BUBB		CUBE	CUB6	PR6	TETR	PRIS	CUB8	FLU3	FL2S	FL3S	FL23
				FL24	FL34	FL35	FL36	FL38					
60		MCOU		COQ3	COQ4	QPPS	DST3	DKT3	Q4GR	Q4GS			
61		BIRD		CAR1	CUBE	TETR	PRIS						
62		PIEZ		P3ZT									
63		LSGL		CAR4	CUBE	CUB6	CUB8	QPPS	COQI	ED01	T3GS	Q41	Q42
				Q41N	Q42N	Q42L	DST3	DKT3	CQD4	CQD9	CQD3	CQD6	Q4GR
				Q4GS	Q4MC	T3MC							
64		CHAN		COQ3	COQ4	QPPS	DST3	DKT3	Q4GR	Q4GS			
65		MORI											
66		JWLS		TRIA	CAR1	CUBE	TETR	PRIS	T3VF	Q4VF	CUVF	PRVF	TEVF
				PYVF	TUVF	TYVF	BIVF	CAVF					
67		ZALM		TRIA	CAR1	CAR4	CUBE	PR6	TETR	PRIS	CUB8	SHB8	ASHB
68		CDEM		T3VF	Q4VF	CUVF	PRVF	TEVF	PYVF				
69		PRGL		BILL									
70		LMC2		TRIA	CAR1	CAR4	CUBE	PR6	PRIS	CUB8	SHB8	ASHB	
71		APPU		APPU									
72		LEM1		TRIA	CAR1	CAR4	CUBE	PR6	PRIS	CUB8	QPPS	COQI	T3GS
				DST3	DKT3	SHB8	SPHC	Q4GR	Q4GS	ASHB			
73		GAZD		T3VF	Q4VF	CUVF	PRVF	TEVF	PYVF	TUVF	TYVF	BIVF	CAVF
74		ABSE		MECA									

75		BOIS		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	
76		VMJC		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	QPPS
				Q92	Q93	Q92A	T3GS	Q41L	Q42L	DST3	DKT3	XCUB	XCAR
				FUN2	Q4GR	Q4GS							
77		VMZA		TRIA	CAR1	CAR4	CUBE	CUB8	QPPS	Q92	Q93	Q92A	T3GS
				Q41L	Q42L	DST3	DKT3	FUN2	Q4GR	Q4GS			
78		VMLP		TRIA	CAR1	CAR4	CUBE	CUB8	QPPS	Q92	Q93	Q92A	T3GS
				Q41L	Q42L	DST3	DKT3	FUN2	Q4GR	Q4GS			
79		VMLU		TRIA	CAR1	CAR4	CUBE	CUB8	QPPS	Q92	Q93	Q92A	T3GS
				DST3	DKT3	FUN2	Q4GR	Q4GS					
80		FUNE		FUN2	FUN3								
81		MCGP		MC23	MC24	MC34	MC35	MC36	MC38				
82		MCFE		CL22	CL3I	CL3Q							
83		DRPR		CAR1	CAR4	CUBE	CUB6	PR6	PRIS	Q92	Q93	Q92A	Q41L
				Q42L									
84		VMSF		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	QPPS
				Q92	Q93	COQI	ED01	Q92A	T3GS	ED41	Q41	Q42	Q41N
				Q42N	Q41L	Q42L	DST3	DKT3	XCUB	XCAR	CQD4	CQD9	CQD3
				CQD6	Q4GR	Q4GS							
85		MAZA		CUBE	PR6	TETR	CUB8	BILL					
86		FLMP		FL2S	FL3S	FL23	FL24	FL34	FL35	FL36	FL38		
87		DPSF		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	QPPS
				Q92	Q93	COQI	ED01	Q92A	T3GS	ED41	Q41	Q42	Q41N
				Q42N	Q41L	Q42L	DST3	DKT3	CQD4	CQD9	CQD3	CQD6	Q4GR
				Q4GS									
88		COMM		COQI	CQD4	CQD9	CQD3	CQD6					
89		CAMC		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	Q92
				Q93	Q92A	Q41L	Q42L						
90		CLAY		TRIA	CAR1	CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	Q92
				Q93	Q92A	Q41L	Q42L						
91		JPRP		BSHT	BSHR	SH3D							
92		GLRC		QPPS	T3GS	DKT3	Q4GR	Q4GS					
93		FOAM		CUBE	CUB6	PR6	TETR	PRIS	CUB8				
94		SUPP		APPU									
95		HYPE		CAR4	CUBE	CUB6	PR6	TETR	PRIS	CUB8	DST3	Q4GS	
96		PBED		CUBE									
97		MINT		INT4	INT6	INT8							
98		TVMC		CUBE									
99		SLZA		CUBE	CUB6	PR6	TETR	CUB8	QPPS	DST3	DKT3	SHB8	Q4GR
				Q4GS	ASHB								
100		CRTM		CUBE	CUB6	PR6	TETR	PRIS	CUB8				
101		TAIT		T3VF	Q4VF	CUVF	PRVF	TEVF	PYVF	TUVF	TYVF	BIVF	CAVF
102		STIF		T3VF	Q4VF	CUVF	PRVF	TEVF	PYVF	TUVF	TYVF	BIVF	CAVF
103		SG2P		T3VF	Q4VF	CUVF	PRVF	TEVF	PYVF	TUVF	TYVF	BIVF	CAVF
104		SGMP		T3VF	Q4VF	CUVF	PRVF	TEVF	PYVF	TUVF	TYVF	BIVF	CAVF
105		EOBT		CUBE	PR6	TETR	PRIS	CUB8					

106		SMAZ		SPHC					
107		SLIN		SPHC					
108		JCLM		SPHC					
109		DADC		CUBE	PR6	TETR	CUB8		

To print out (on the log file!) an up-to-date version of the above element and material tables, just run EUROPLEXUS with any input data file by adding the option `OPTI DPEM` (see also page H.90).

9.2 AUXILIARY FILE

Object:

This directive allows to read the material data from an auxiliary file.

Syntax:

```
"MATERIAUX"      < "FICHIER"      'nom.fic'  >
```

In certain cases the data may be bulky. It is then advised to store the data on an auxiliary file in order to shorten the main input data file. The auxiliary file is activated by means of the keyword "FICHIER", followed by the full name (under Unix) of the file. Therefore, only the words "MATERIAUX" "FICHIER" 'nom.fic' remain in the main input file.

The auxiliary file (in free format) will contain the whole set of material data, with the exception of the "MATERIAUX" keyword. To return to the main input data file, the auxiliary file must be terminated by the keyword "RETOUR".

9.3 LOCALISED DAMPING

Object:

This directive allows to add a localised damping on some d.o.f.s of some particular nodes.

Syntax:

```
"AMORTISSEMENT" ( /LECDDL/ "BETA" beta "FREQ" freq /LECTURE/ )
/LECDDL/
    Concerned degrees of freedom.
beta
    Reduced damping  $\beta$ .
freq
    Frequency  $f$  of the global mode to be damped out.
/LECTURE/
    List of the concerned nodes.
```

Comments:

The value $\beta = 1$ corresponds to the critical damping for the frequency f . All frequencies are damped. The components with a frequency lower than the cut-off frequency: $f_c = \beta f$ will be damped in a pseudo-periodic manner while those having higher frequencies will be damped in an aperiodic manner.

This damping is proportional to the mass M and to the particles velocity v , and may be used in order to damp out preferably the structures without influence on the internal fluid, for example.

One adds an external force F_{amort} of the form:

$$F_{\text{amort}} = -2\beta\omega Mv$$

where $\omega = 2\pi f$.

It is evident that the work of external forces will be modified by the damping forces.

This directive differs from the global damping directive (`OPTI AMOR` . . . , see page H.30) mainly by the fact that here the region to which damping is applied may be specified by the user, while in the other case the damping applies to the whole model (but limitedly to some element types, see page H.30).

9.4 NON-LINEAR SUPPORTS : "APPU"

Object :

This directive allows to model non-linear supports of type spring or damper. It may be used only for the elements of type "APPUI" (material points with 6 d.o.f.s). The user gives the evolution curve of the force applied by the support as a function of its displacement (for the springs) or of its velocity (for the dampers). These supports work in translation or in rotation.

Syntax :

```
"APPUI" |[ "RESS" ; "AMOR" ] | |[ "TRAN" ; "ROTA" ] |
          "CMPX" cmpx      "CMPY" cmpy      "CMPZ" cmpz
          "COEF" coef      "NUFO" nufo      <"MASS" mass>
          <"INCR" incr>    <"DECX" decx>    <"DECY" decy> /LECTURE/
```

"RESS"

The support is of type spring.

"AMOR"

The support is of type damper.

"TRAN"

The support works in translation.

"ROTA"

The support works in rotation.

cmpx

Component in X of the translation or rotation axis of the support.

cmpy

Component in Y of the translation or rotation axis of the support.

cmpz

Component in Z of the translation or rotation axis of the support.

coef

Multiplicative coefficient of the function.

nufo

Number of the function.

mass

Inertia of the support along its working direction.

incr

Increment of the velocity or displacement for the calculation of the local stiffness.

decx

Offset of the abscissas of the force/displacement or force/velocity curve.

decy

Offset of the ordinates of the force/displacement or force/velocity curve.

/LECTURE/

List of the concerned nodes.

Comments :

The user must define a vector corresponding to the rotation axis or translation axis of the support. This vector does not need to be normalised, just its direction matters. This direction defines the local reference frame of the support: it is the projection of the displacement (or of the velocity) of the concerned node onto this axis that allows to determine the reaction force.

An APPUI element may not work simultaneously as a spring AND as a damper, nor in translation AND in rotation. Therefore it will be sometimes necessary to define several APPUI elements, geometrically coincident, in order to correctly define the local stiffness.

The function defining the force generated by the support in response to displacement or velocity of its application point on the supported structure is of the form:

$$F = coef f(D) \quad or \quad F = coef f(V)$$

with f(D) or f(V) given by the user. Warning: these values have a sign. Do not forget to give the force with the opposite sign as the displacement (this is a reminder).

For the estimation of the stability step, it is necessary to know the local slope of the behaviour curve. To this end, the user must specify the keyword "INCR". The computation of the local stiffness will then be (by default, incr=1.E-4):

$$K = (F(D + incr) - F(D))/incr \quad or \quad C = (F(V + incr) - F(V))/incr$$

In the case that the structure is not in equilibrium for a zero displacement at the beginning of the calculation, the user may impose a translation of vector (decx, decy) of the behaviour curve. The computed force will then be (by default, decx and decy are zero):

$$F = (coef\ f(D + decx)) - decy$$

(in fact : $decy = coef\ f(decx)$)

Outputs :

The components of the ECR vector are:

ECR(1): Force (resp. moment) along X.

ECR(2): Force (resp. moment) along Y.

ECR(3): Force (resp. moment) along Z.

ECR(4): Current stiffness.

ECR(5): Current velocity (or angular velocity).

ECR(6): Total displacement (or rotation).

ECR(7): Applied force (or moment) to the node (reaction force).

9.5 NON-LINEAR SUPPORTS : "SUPP"

Object :

This directive allows to model a complex non-linear support, having arbitrary stiffness and damping values along the 6 dofs of the concerned node. It may only be used in conjunction with elements of type "APPUI" (material point with 6 dofs). The user gives the evolution curve of the reaction force generated by the support as a function of the displacement or of the velocity of the associated node.

Syntax :

```
"SUPP"  "MASS" m
        <| [ "KX" kx ; "KY" ky ; "KZ" kz ] | "NFKT" nufo1>
        <| [ "AX" ax ; "AY" ay ; "AZ" az ] | "NFAT" nufo2>
        <"IRX" irx> <"IRY" iry> <"IRZ" irz>
        <| [ "KRX" krx ; "KRY" kry ; "KRZ" krz ] | "NFKR" nufo3>
        <| [ "ARX" arx ; "ARY" ary ; "ARZ" arz ] | "NFAR" nufo4>
        /LECTURE/
```

m

Additional translational mass (optional).

kx, ky, kz

Translationl stiffnesses along the global axes.

nufo1

Index of the function associated with translational stiffnesses.

ax, ay, az

Translationl dampings along the global axes.

nufo2

Index of the function associated with translational dampings.

irx, iry, irz

Additional rotational inertias along the global axes (optional).

krx, kry, krz

Rotational stiffnesses along the global axes.

nufo3

Index of the function associated with rotational stiffnesses.

arx, ary, arz

Rotational dampings along the global axes.

nufo4

Index of the function associated with rotational dampings.

/LECTURE/

List of the concerned nodes.

Comments :

The stiffnesses and the dampings are given along the global (fixed) axes of the problem. Each of the 4 associated functions applies to the 3 corresponding stiffnesses (or dampings).

If a key-word is missing, the corresponding value is zero, and the order in which the parameters are specified is irrelevant.

The reaction force generated by the support has the form (e.g., assuming translation along Ox):

$$F_x = k_x f_1(D_x) + a_x f_2(V_x)$$

If the displacement (or the velocity) is positive, the function f1 (or f2) must be negative in order to obtain a correct reaction.

Outputs :

The components of the ECR vector are:

ECR(1): Reaction of the support along X.

ECR(2): Reaction of the support along Y.

ECR(3): Reaction of the support along Z.

ECR(4): Reaction of the support along RX.

ECR(5): Reaction of the support along RY.

ECR(6): Reaction of the support along RZ.

9.6 SOLID MATERIALS

9.6.1 LINEAR ELASTICITY

Object:

This option enables materials with a linear elastic behaviour to be used.

Syntax:

```
"LINE"    "RO" rho  "YOUN" young  "NU" nu  
          <"VISC" visc "KRAY" kray "MRAY" mray>  /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

visc

Viscosity coefficient (decay factor), used only by spectral elements (MS24, MS38) and finite elements of the following types: TRIA, CAR1, CAR4, CUBE, CUB6, CUB8, TETR, PR6, PRIS.

kray, mray

Rayleigh's stiffness and mass proportional damping coefficients, used only by finite elements of the following types: POUT, TUYA, DKT3, T3GS, Q4GS. Default values: kray=0, mray=0. For information about Rayleigh's damping see reference [\[738\]](#).

LECTURE

List of the elements concerned.

Comments:

This option may be repeated as many times as necessary.

Outputs:

The components of the ECR table are as follows:

Solid elements:

ECR(1): pressure

ECR(2): Von Mises criterion

Shells:

ECR(1): Von Mises criterion (membrane)

ECR(2): Von Mises criterion (membrane + bending)

Beams (3D):

ECR(1): Von Mises criterion (bending)

ECR(2): Von Mises criterion (membrane + bending + torsion)

Bars (BARR, PONC, BR3D):

ECR(1): elastic strain

ECR(2): Von Mises criterion

9.6.2 DRUCKER-PRAGER

Object:

This option enables to specify materials with a perfect elasto-plastic behaviour (Drucker-Prager criterion).

Syntax:

```
"DRUC"  "RO"  rho  "YOUNG"  young  "POISSON"  nu  ...  
...  "TRACTION"  sigt  "COMPRESSION"  sigc  ...  
...  < "FRACTURE"  pf  >  /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

sigt

Maximum stress under tension (without confinement).

sigc

Maximum stress under compression (without confinement).

FRACTURE pf

The material is fractured (does no longer resist tension) as soon as the criterion is reached for the first time. Then, the domain changes and the new D.P. criterion corresponds to vanishing cohesion whereas the slope is equivalent to the one in the previous case. The parameter 'pf' is compulsory and represents the maximum pressure of fracturing under compression. If the criterion is reached for the first time when the pressure is superior to pf, the domain does not change and the criterion is the same as initially.

/LECTURE/

Numbers of the elements concerned.

Comments:

The values of sigt, sigc and pf are absolute values.

If P defines the pressure (positive under tension) and SIG* the Von Mises criterion, the Drucker-Prager criterion is defined by:

$$\text{Criterion} = \text{SIG*} - \text{cohe} + P * \text{pente} \quad (\text{always} \leq 0)$$

The 2 parameters : cohe and pente (slope), are calculated from sigt and sigc values, they are printed after the reading of the data. The parameter cohe (cohesion) corresponds to a maximum Von Mises under non-existent pressure. The slope is the straight line limiting the domain, in the coordinate system (P,SIG*).

In the space of the principal stresses the criterion determines a cone the axis of which is the straight line of equation: $\text{sig}(1) = \text{sig}(2) = \text{sig}(3)$.

The maximum stresses: sigt and sigc correspond to the values observed during uniaxial tests without confinement. These two points enable the Drucker-Prager domain to be defined.

The value of the parameter "FRACTURE" enables the behaviour of concrete to be represented in a very simplified way. Two domains may be distinguished:

- Brittle rupture
- Ductile rupture (strong compressions)

Most often one may take $\text{pf} = \text{sigc}/3$. A great value for pf delay the fracturation.

Outputs:

The different components of the ECR table are as follows:

ECR(1): pressure

ECR(2): Von Mises

ECR(3): equivalent plastic strain

ECR(4): D.P. criterion (always ≤ 0)

ECR(5): cohesion (becomes non-existent in the case of brittle rupture)

9.6.3 VON MISES MATERIAL

Object:

This sub-directive enables materials with an elasto-plastic behaviour to be used. There are four options:

- "VMIS" "PARF" : perfectly plastic Von Mises material;
- "VMIS" "ISOT" : isotropic Von Mises material;
- "VMIS" "DYNA" : isotropic Von Mises material depending on strain rate;
- "VMIS" "TETA" : isotropic Von Mises material depending on temperature.

Syntax:

```
"VMIS"  
  $ [  
    "PARF" . . . ;  
    "ISOT" . . .  
    "DYNA" . . .  
    "TETA" . . .  
  ]$
```

Comments:

This sub-instruction may be repeated as many times as necessary with different options each time (if need be). The word "VMIS" cannot be separated from the option which follows.

PERFECTLY PLASTIC VON MISES**Object:**

Perfectly plastic Von Mises material.

Syntax:

```
"VMIS"  "PARF"  "RO" rho  "YOUN" young  "NU" nu  "ELAS" sige  ...
          ...  /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

sige

Elastic limit.

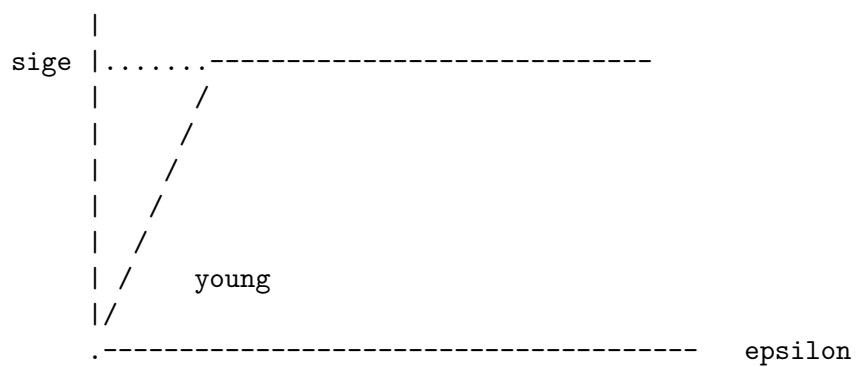
ncrit

LECTURE

List of the elements concerned.

Comments:

The law of behaviour is described by the following diagram of stresses and strains:



Outputs:

The components of the ECR table are as follows:

Solid elements:

ECR(1): pressure
ECR(2): Von Mises criterion
ECR(3): plastic strain

Shells integrated through the thickness:

ECR(1): pressure
ECR(2): Von Mises criterion
ECR(3): plastic strain

Global model shells:

ECR(1): Von Mises criterion (membrane)
ECR(2): Von Mises criterion (membrane + bending)
ECR(3): plastic strain

Bars (BARR, PONC, BR3D):

ECR(1): elastic strain
ECR(2): Von Mises criterion
ECR(3): plastic strain

ISOTROPIC VON MISES**Object:**

Isotropic Von Mises material.

Syntax:

```
"VMIS" "ISOT" "RO" rho "YOUN" young "NU" nu "ELAS" sige ...  
      <FAIL fail LIMI limi>  
      ... "TRAC" npts*( sig eps ) /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

sige

Elastic limit.

FAIL

Optional keyword: introduces an element failure model represented by a failure criterion and a by failure limit value. Two failure criteria only available for POUT elements are:
fail = 1 for a criterion based upon Von Mises stress (membrane + bending + torsion),
fail = 2 for a criterion based upon plastic strain.

limi

Optional parameter, indicates the failure limit for the chosen criterion.

"TRAC"

This key-word introduces the yield curve.

npts

Number of points (except the origin) defining the yield curve.

sig

Stress.

eps

Total strain (elastic + plastic).

/LECTURE/

List of the elements concerned.

Comments:

1/ - The young parameter defines Young's modulus during an elastic phase.

2/ - The points (sig,eps) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.

Outputs:

The components of the ECR table are as follows:

Solid elements

ECR(1): pressure

ECR(2): Von Mises criterion

ECR(3): plastic strain

ECR(7): new elastic limit

Shells integrated through the thickness:

ECR(1): pressure

ECR(2): Von Mises criterion

ECR(3): plastic strain

ECR(7): new elastic limit

Global model shells:

ECR(1): Von Mises criterion (membrane)

ECR(2): Von Mises criterion (membrane + bending)

ECR(3): plastic strain

ECR(7): new elastic limit

Beams (3D):

ECR(1): Von Mises criterion (bending)
ECR(2): Von Mises criterion (membrane + bending + torsion)
ECR(3): plastic strain
ECR(7): new elastic limit
ECR(10): failure flag (0=virgin Gauss Point, 1=failed Gauss Point)

Bars (BARR, PONC, BR3D):

ECR(1): elastic strain
ECR(2): Von Mises criterion
ECR(3): plastic strain
ECR(7): new elastic limit

DYNAMIC VON MISES**Object:**

Isotropic Von Mises material depending on strain rate.

Syntax:

```
"VMIS"  "DYNA"  "RO" rho  "YOUN" young  "NU" nu  ...  
...  "TRAC"  npts*( sig  eps )  ...  
  
...  $[ "SYMO"  "D" d  "P" p  ;  
      "ISPR" "VITE"  a  b  c  d  e  f  ;  
      "LIBR"  num  "PARA"  /LECPARA/  ]$  /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

"TRAC"

This key-word introduces the yield curve.

npts

Number of points (except the origin) defining the yield curve.

sig

Stress

eps

Total strain (elastic + plastic).

"SYMO"

Constitutive relation of Symonds and Cowper.

d

First coefficient of the Symonds and Cowper law.

P

Second coefficient of the Symonds and Cowper law.

"VITE"

This key-word introduces the parameters of the dynamic yield curve.

a,b,c,d,e,f

The 6 parameters of the dynamic yield curve.

"LIBRE"

Introduces the utilisation of a user's subroutine to compute the dynamic amplification coefficient.

num

Identification number of the free material.

"PARA"

Keyword that can be used to introduce a series of parameters for the free material. The number of parameters is arbitrary, because the /LECTURE/ procedure signals the termination of the list.

/LECTURE/

List of the elements concerned.

Comments:

For the Symonds and Cowper law, the dynamic traction curve is derived from the static one through a multiplicative coefficient which depends upon the strain rate (EPSP):

$$\text{SIG(dyna)} = \text{SIG(stat)} * (1 + (\text{EPSP} / D) ** (1/P))$$

Indicatively, for the stainless steel 304 L, experimental results suggest: $D = 100 \text{ s}^{-1}$ and $P = 10$ (Forrestal and Sagartz 1978). For ordinary steel, it is usually assumed: $D = 40 \text{ s}^{-1}$ and $P = 5$ (Symonds 1965).

For titanium TI-50A, the values suggested are: $D = 120 \text{ s}^{-1}$ and $P = 9$ (Symonds et Chon 1974).

For aluminum alloys, some authors use $D = 6500 \text{ s}^{-1}$ and $P = 4$ (Symonds 1965).

In the case of the ISPRA law, the formulation is similar, but the multiplying coefficient depends upon the strain (EPS) as well as on the strain rate (EPSP):

$$\text{SIG(dyna)} = \text{SIG(stat)} * (1 + (\text{EPSP} / K) **M)$$

with the K and M coefficients of the form:

$$K = \text{EXP}((A + B * \text{EPS}) / (1 + C * \text{EPS}))$$

$$M = (D + E * \text{EPS}) / (1 + F * \text{EPS})$$

Examples of data (source ISPRA-CADARACHE):

Material	a	b	c	d	e	f
Steel 304	5.82	168.76	9.62	0.242	2.263	12.77
Steel 316	6.388	86.215	6.457	0.233	0.0	0.0

Outputs:

The components of the ECR table are as follows:

Solid elements:

- ECR (1): pressure
- ECR (2): Von Mises criterion in dynamics
- ECR (3): equivalent plastic strain
- ECR (7): new elastic limit in statics
- ECR (8): equivalent strain rate
- ECR (9): total equivalent deformation
- ECR(11): elastic limit in dynamics

Shells integrated through the thickness:

- ECR (1): pressure
- ECR (2): Von Mises criterion in dynamics
- ECR (3): equivalent plastic strain
- ECR (7): new elastic limit in statics
- ECR (8): equivalent strain rate
- ECR (9): total equivalent deformation
- ECR(11): elastic limit in dynamics

Global model Shells:

- ECR (1): Von Mises criterion (membrane)
- ECR (2): global Von Mises criterion (membrane + bending)
- ECR (3): equivalent plastic strain
- ECR (7): new elastic limit
- ECR (8): equivalent strain rate
- ECR (9): total equivalent deformation
- ECR(11): elastic limit in dynamics

TEMPERATURE-DEPENDENT VON MISES**Object:**

Von Mises isotropic material dependent upon the temperature.

Syntax:

```
"VMIS"  "TETA"  "RO" rho < "NU" nu >...  
...  "NBCOURBE" nc*( "TETA" ti "YOUNG" yg <"NUT"> nut ...  
...  "TRAC" npts*( sig eps ) ) /LECTURE/
```

rho

Density.

nu

Poisson coefficient. Only if NU does not depend on the temperature.

nc

Number of traction curves that allow the interpolation as a function of temperature.

ti

Temperature associated with the following traction curve.

yg

Young's modulus.

nut

Poisson Poisson. If NU depend on temperature.

"TRAC"

Introduces the traction curve.

npts

Number of points (excluding the origin) which define the traction curve.

sig

Stress.

eps

Total strain (elastic + plastic).

/LECTURE/

List of the elements concerned.

Comments:

Each element is isothermal, i.e. its temperature remains constant during the whole calculation.

Depending upon temperature, the Young's modulus, the poisson coefficient and the traction curve are interpolated starting from the values associated to the known temperatures.

Note that it is possible to define either a temperature-dependant Poisson coefficient or not which can be sufficient in case of steels for example.

Outputs:

The components of the ECR table are as follows:

Continuum elements:

- ECR(1) : pressure
- ECR(2) : Von Mises criterion
- ECR(3) : equivalent plastic strain

Shells integrated through the thickness:

- ECR(1) : pressure
- ECR(2) : Von Mises criterion
- ECR(3) : equivalent plastic strain

Global model shells:

- ECR(1) : Von Mises criterion (membrane)
- ECR(2) : global Von Mises criterion (membrane + bending)
- ECR(3) : equivalent plastic strain

9.6.4 STEINBERG-GUINAN

Object:

This is a Von Mises isotropic material whose Young's modulus and elastic limit are a function of hydrostatic pressure, temperature increase and strain rate.

Syntax:

```
"STGN"  "RO"  rhoz  "YOUN"  youngz  "NU"    nu    ...
...     "SIGE" sigez "SIGD"  sigd   "CHSP"  cv    ...
...     "TF"   tfus  "TINI"  tini   "B"     b    ...
...     "H"   h     "BETA"  beta   "N"     n    /LECTURE/
```

rhoz

Density at the initial temperature.

youngz

Young's modulus at the initial temperature.

nu

Poisson coefficient (constant).

sigez

Static elastic limit at the initial temperature.

sigd

Dynamic elastic limit at the initial temperature.

cv

Specific heat capacity of the solid.

tfus

Melting temperature of the material.

tini

Initial temperature of the material.

b,h,beta,n

Coefficients of the STEINBERG and GUINAN law.

/LECTURE/

List of the elements concerned.

Comments:

The STEINBERG and GUINAN law uses the Young's modulus E , and an elastic limit Y , which vary according to the following expressions:

$$E = \text{youngz} * P1$$

$$Y = \text{yield} * P1$$

with:

$$P1 = 1 + b * P / K^{1/3} + h * dteta$$

$$\text{yield} = \text{MIN} (\text{sigd} , \text{sigez} * P2)$$

$$P2 = (1 + \text{beta} * \text{EPSP})^n$$

where:

P is the hydrostatic pressure;

K is the compression ratio (ratio between the current density and the initial density);

EPSP is the total equivalent strain rate;

$dteta$ is the temperature increase with respect to the initial temperature.

On the other hand, when the current temperature ($teta = t_{ini} + dteta$) exceeds the melting temperature of the material (t_{fus}), it is assumed that the material is liquefied: the Young's modulus and the elastic limit are then taken as zero.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : hydrostatic pressure

ECR(2) : Von Mises

ECR(3) : equivalent plastic strain

ECR(4) : temperature increase (dteta)

ECR(5) : current elastic limit

ECR(6) : current Young's modulus

ECR(7) : equivalent plastic strain rate

9.6.5 LEM1

Object :

This directive allows to describe the behaviour of an elasto-plastic material that may undergo some damage, according to the Lemaitre model. There is coupling between damage and plasticity, represented by the Von Mises criterion. The damage evolution rate is a function of the triaxiality ratio of stresses and of the equivalent plastic strain rate. A failure criterion is implicitly contained within the model: rupture occurs when the damage exceeds a critical value. Two optional parameters allow to introduce a limitation of the damage rate (thanks to the delayed damage model) in order to avoid the mesh dependency.

Syntax:

```
"LEM1"  "RO" rho "YOUN" young "NU" nu "ELAS" sige ...
        "EPSD" epsd "S0" s0 "DC" dc ...
        <"CSTA" csta "TAUC" tauc "NOCO" noco> ...
        "TRAC" npts*( sig eps ) /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's coefficient.

sige

Elastic limit.

epsd

Damage threshold (i.e. equivalent plastic strain, weighted by a function of stress triaxiality, within which damage vanishes).

s0

Parameter driving the damage evolution rate.

dc

Critical damage defining the rupture criterion.

csta

Parameter of the delayed damage model

tauc

Characteristic time of the delayed damage model. $(1/\text{tauc})$ represents the maximum damage rate.

noco

Optional parameter indicating what to do when no convergence is reached in the material routine. The value 0 is the default and means that an error message is issued and the calculation is stopped. The value 1 indicates that the element (or more precisely, the element's current Gauss point) is made to fail (eroded). The value -1 indicates that subcycling is activated in an attempt to reach convergence, by subdividing the load step into smaller sub-cycles.

"TRAC"

Introduces the traction curve.

npts

Number of points (except the origin) defining the traction curve.

sig

Stress.

eps

Total strain (elastic + plastic).

LECTURE

List of concerned elements.

Comments:

A detailed description of the model can be found in the report DMT/98-026A, available on request.

Outputs:

The components of the ECR table are as follows for **Continuum elements**:

ECR(1) : pressure
ECR(2) : Von Mises criterion
ECR(3) : equivalent plastic strain
ECR(4) : plasticity multiplier
ECR(5) : damage
ECR(7) : new elastic limit

When the “erosion” algorithm is activated (see page A.30, Section 6.4, keyword FAIL), an element is considered as failed if `damage` \geq `dc`.

9.6.6 ZALM

Object :

This directive allows to describe the behaviour of an Zirelli-Armstrong material that may undergo some damage, according to the Lemaitre model. There is coupling between damage and plasticity, represented by the Von Mises criterion. The damage evolution rate is a function of the triaxiality ratio of stresses and of the equivalent plastic strain rate. A failure criterion is implicitly contained within the model: rupture occurs when the damage exceeds a critical value. Two optional parameters allow to introduce a limitation of the damage rate (thanks to the delayed damage model) in order to avoid the mesh dependency.

Syntax:

```
"ZALM"  "RO" rho "YOUN" young "NU" nu "ELAS" sige ...
        "EPSD" epsd "S0" s0 "DC" dc ...
        "ZAC0" zac0 "ZAC1" zac1 "ZAC2" zac2 "ZAC3" zac3 ...
        "ZAC4" zac4 "ZAC5" zac5 "ZAN" zan ...
        <"CSTA" csta "TAUC" tauc> ...
        "TRAC" npts*( sig eps ) /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's coefficient.

sige

Elastic limit.

epsd

Damage threshold (i.e. equivalent plastic strain, weighted by a function of stress triaxiality, within which damage vanishes).

s0

Parameter driving the damage evolution rate.

dc

Critical damage defining the rupture criterion.

csta

Parameter of the delayed damage model

zac0

Parameter of zerilli-armstrong model c0

zac1

Parameter of zerilli-armstrong model c1

zac2

Parameter of zerilli-armstrong model c2

zac3

Parameter of zerilli-armstrong model c3

zac4

Parameter of zerilli-armstrong model c4

zac5

Parameter of zerilli-armstrong model c5

zan

Parameter of zerilli-armstrong model n

tauc

Characteristic time of the delayed damage model. $(1/\text{tauc})$ represents the maximum damage rate.

"TRAC"

Introduces the traction curve.

npts

Number of points (except the origin) defining the traction curve.

sig

Stress.

eps

Total strain (elastic + plastic).

LECTURE

List of concerned elements.

Outputs:

The components of the ECR table are as follows for **Continuum elements**:

ECR(1) : pressure

ECR(2) : Von Mises criterion

ECR(3) : equivalent plastic strain

ECR(4) : plasticity multiplier

ECR(5) : damage

ECR(7) : new elastic limit

When the “erosion” algorithm is activated (see page A.30, Section 6.4, keyword **FAIL**), an element is considered as failed if **damage** \geq **dc**.

9.6.7 LMC2

Object:

This directive allows to describe the behaviour of an elasto-plastic material that may undergo some damage, according to the Lemaitre-Chaboche model. There is coupling between damage and plasticity, represented by the Von Mises criterion. The damage evolution rate is a function of the triaxiality ratio of stresses and of the equivalent plastic strain rate. A failure criterion is implicitly contained within the model: rupture occurs when the damage exceeds a critical value. Unlike model LEM1, the material properties may depend upon the strain rate. Two optional parameters allow to introduce a limitation of the damage rate (thanks to the delayed damage model) in order to avoid the mesh dependency

Syntax:

```
"LMC2" "RO" rho ...
      "YOUN" young < "FONC" nfyoun ...
          $[ "TABL" nptyou*( para vyou ) ; "ROUT" ; "DONE" ]$ > ...
      "NU" nu      < "FONC" nfnu ...
          $[ "TABL" nptnu*( para vnu ) ; "ROUT" ; "DONE" ]$ > ...
      "ELAS" sigE < "FONC" nfela ...
          $[ "TABL" nptela*( para vela ) ; "ROUT" ; "DONE" ]$ > ...
      "EPSD" epsd < "FONC" nfepd ...
          $[ "TABL" nptepd*( para vepd ) ; "ROUT" ; "DONE" ]$ > ...
      "S0" s0      < "FONC" nfs0 ...
          $[ "TABL" npts0*( para vs0 ) ; "ROUT" ; "DONE" ]$ > ...
      "DC" dc      < "FONC" nfdc ...
          $[ "TABL" nptdc*( para vdc ) ; "ROUT" ; "DONE" ]$ > ...
      <"CSTA" csta "TAUC" tauc> ...
```

If the traction curve is given by a table:

```
"TRAC" ctra "FTRA" nftra ...
      $ "TABL" npt*( sig eps ) ; "ROUT" ; "DONE" $ ...
```

If the traction curve is given by an abaque:

```
"TRAC" ctra "ATRA" natra $ "SET" npara ...
      "NPTM" nptm*( "PARA" para "TABL" npt*( sig eps )); ...
      "DONE" $ ...
```

/LECTURE/

rho

Density.

young

Young's modulus if it is constant or multiplicative coefficient of Young's modulus if it is defined by a function.

nfyou

Number of the function defining the variation of the Young's modulus with the strain rate.

nptyou

Number of point defining the variation of the Young's modulus with the strain rate.

para

Parameter (here the strain rate).

vyou

Value of the Young's modulus corresponding to the parameter.

nu

Poisson's coefficient if it is constant or multiplicative coefficient of Poisson's coefficient if it is defined by a function.

nfnu

Number of the function defining the variation of the Poisson's coefficient with the strain rate.

nptnu

Number of point defining the variation of the Poisson's coefficient with the strain rate.

vnv

Value of the Poisson's coefficient corresponding to the parameter.

sige

Elastic limit if it is constant or multiplicative coefficient of the elastic limit if it is defined by a function.

nfela

Number of the function defining the variation of the elastic limit with the strain rate.

nptela

Number of point defining the variation of the elastic limit with the strain rate.

vela

Value of the elastic limit corresponding to the parameter.

epsd

Damage threshold (i.e. equivalent plastic strain, weighted by a function of triaxiality rate of stresses, below which the damage is zero) if it is constant or multiplicative coefficient of the damage threshold if it is defined by a function.

nfepd

Number of the function defining the variation of the damage threshold with the strain rate.

nptepd

Number of point defining the variation of the damage threshold with the strain rate.

vepd

Value of the damage threshold corresponding to the parameter.

s0

Parameter driving the evolution rate of damage if it is constant or multiplicative coefficient of the parameter driving the evolution rate of damage if it is defined by a function.

nfs0

Number of the function defining the variation of the parameter driving the evolution rate of damage with the strain rate.

npts0

Number of point defining the variation of the parameter driving the evolution rate of damage with the strain rate.

vs0

Value of the dparameter driving the evolution rate of damage corresponding to the parameter.

dc

Critical damage defining the rupture criterion if it is constant or multiplicative coefficient of critical damage if it is defined by a function.

nfdc

Number of the function defining the variation of the critical damage with the strain rate.

nptdc

Number of point defining the variation of the critical damage with the strain rate.

vdc

Value of the critical damage corresponding to the parameter.

csta

Parameter of the delayed damage model

tauc

Characteristic time of the delayed damage model. $(1/\text{tauc})$ represents the maximum damage rate.

"TRAC"

Introduces the traction curve.

ctra

Multiplicative coefficient of the stress in the traction curve or curves.

"FTRA"

Introduces the single traction curve for all strain rates.

nftra

Number of the function defining the traction curve.

npt

Number of point (except the origin) defining the traction curve.
item[**sig**]

Stress. item[**eps**]

Strain (elastic+plastic).

"ATRA"

Introduces an abaque giving the traction curve for different strain rates.

natra

Number of the abaque defining the traction curves.

npara

Number of the set of parametrised functions that associate to each strain rate the corresponding traction curve.

nptm

Maximum number of point (except the origin) defining the traction curve amongst the set of parametrised functions.

LECTURE

List of the concerned elements.

Comments:

In the case of traction curve, parametrised or not, the origin is always omitted.

If both the Young's modulus and the traction curve are parametrised, the strain rate parameter should be identical.

Dans le cas de la courbe de traction paramtr, il faudra fournir les vitesses de dformation de manire croissante.

In the case of a component dependent upon strain rate, it is mandatory to give its value for a zero velocity (static case) and for a very large velocity.

A detailed description of the model may be found in the report DMT/98-036A, available on request.

Outputs:

The components of the ECR table are as follows for **Continuum elements** :

ECR(1) : pressure

ECR(2) : Von Mises criterion

ECR(3) : equivalent plastic strain

ECR(4) : plasticity multiplier

ECR(5) : damage

ECR(7) : new elastic limit

ECR(8) : strain rate

ECR(11): = 1 critical damage reached, otherwise < 1

When the "erosion" algorithm is activated (see page A.30, Section 6.4, keyword FAIL), an element is considered as failed if $ECR(11) > 0.99$.

9.6.8 CONCRETE: Old version

Object:

This option is used to define materials such as concrete, soil, rock, etc.

Comments:

The law of behaviour used in this model is based on plasticity; it takes into account three modes of damaging the material:

- 1) Damage due to traction;
- 2) Damage due to shear;
- 3) Damage due to hydrostatic pressure.

A material of this type possesses 38 input parameters; however, only some of them are compulsory. Each parameter is entered into the input file by means of a key-word, these words can be entered in any order. Just remember that the data placed between angle brackets are not compulsory, for example: <"PREC" prec>.

The numerical values of the different parameter are entered in absolute value. Moreover the following conventions have been adopted for the outputs:

- positive values: tension stresses;
- negative values: compression stresses.

The option "BETON" can be repeated as many times as necessary.

Syntax:

The data can be classified in 4 groups.

- 1) Generic data

```
"BETON"  "RO" rho  "YOUN" young  "NU" nu  
          < "ALPH" alph >  < "PREC" prec >
```

rho

Density of the material.

young

Elasticity modulus.

nu

Poisson's ratio.

alpha

Coefficient of thermal expansion.

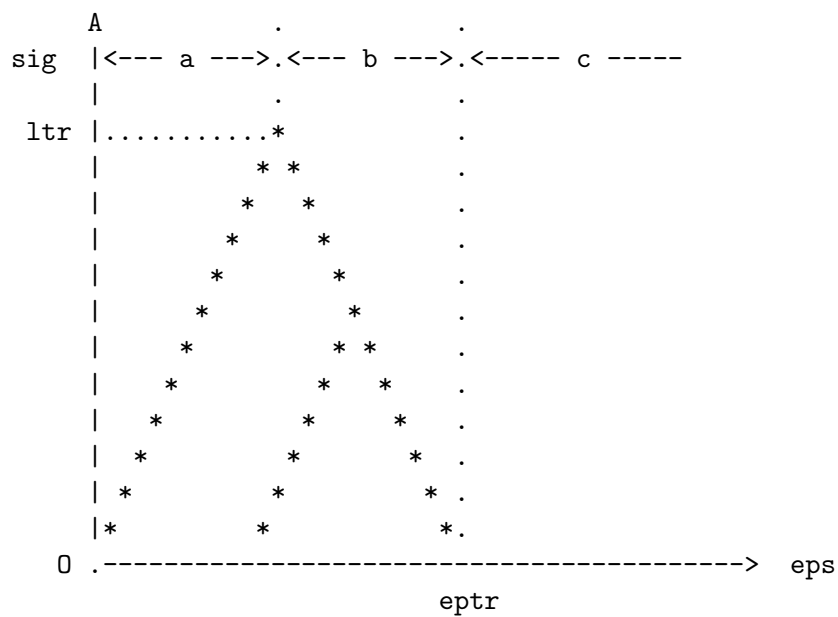
prec

Precision of the computation on the internal iterations.

- 2) Data concerning the damage due to traction:

This kind of damage occurs in 3 phases:

- a) elastic behaviour;
- b) cracked elastic behaviour;
- c) perfectly plastic behaviour.



```
< "BETA"  cisail >
```

```
* initially isotropic material:
```

```
"LTR"  ltr
"EPTR" eptr
```

```
* initially anisotropic material:
```

```
< "IFIS"  ifis >
```

```
< "LT1"  lt1 > < "LT2"  lt2 > < "LT3"  lt3 >
< "EPT1" ept1 > < "EPT2" ept2 > < "EPT3" ept3 >
```

```
< "OUV1" ouv1 > < "OUV2" ouv2 > < "OUV3" ouv3 >
```

```
< "ANGL" angle >  or  < "V1X" v1x    "V1Y" v1y    "V1Z" v1z >
                        < "V2X" v2x    "V2Y" v2y    "V2Z" v2z >
                        < "V3X" v3x    "V3Y" v3y    "V3Z" v3z >
```

cisail

Value of residual shear after cracking, in comparison with the initial status (value between 0 and 1).

ltr

Limit in traction in the case of an initially isotropic material.

eptr

Rupture strain in the case of an initially isotropic material.

ifis

Cracking index (0: no cracking, 1: one crack only, 2: two cracks, 3: three cracks).

lt1, lt2, lt3

Traction limits along the directions 1, 2 and 3 in the case of an initially anisotropic material.

ept1, ept2, ept3

Rupture strains along the directions 1, 2 and 3 in the case of an initially anisotropic material.

ouv1, ouv2, ouv3

Opening of the cracks along the directions 1, 2 and 3 in the case of an initially cracked material (deformations).

angle

Crack angle in the (X-Y) plane, in degrees, in the case of a plane stress analysis.

v1x,v1y,v1z

Components of the vector defining direction 1.

v2x,v2y,v2z

Components of the vector defining direction 2.

v3x,v3y,v3z

Components of the vector defining direction 3.

The model takes into account the anisotropy induced by the cracking.

The opening and closing of cracks is managed by the model.

For an axisymmetric or three-dimensional analysis, the user can enter different characteristics for the three directions.

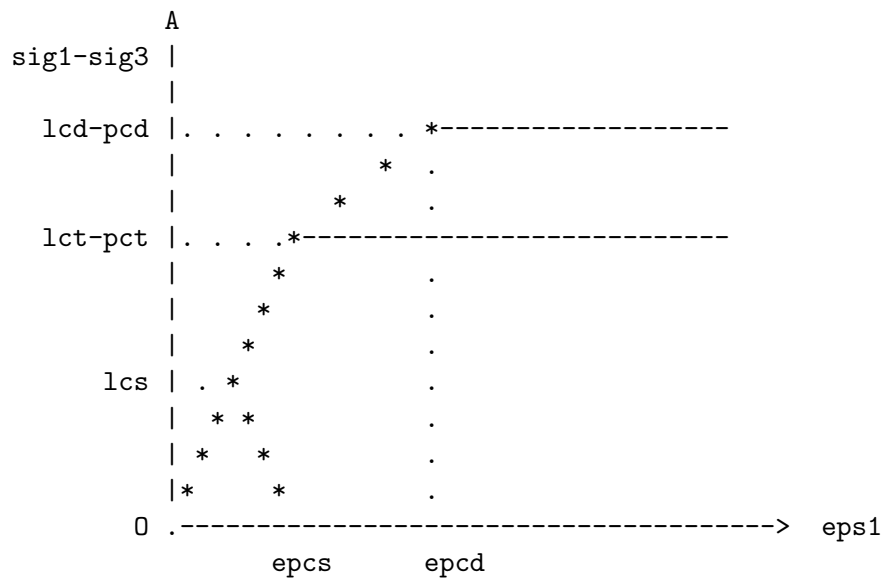
In the case of an initially cracked material, one can input the opening of cracks by means of initial deformations along the cracked direction.

- 3) Data relative to shear damage:

Triaxial tests, carried out at different confinement levels, are necessary to determine the various parameters of the model. The results are then linearized and entered onto the diagram (sig1-sig3, eps1). The user may distinguish two different domains:

- a) brittle behaviour corresponding to the confinement levels, i.e. low sig3. This behaviour can be schematized by a decreasing branch and a negative work-hardening.
- b) ductile behaviour corresponding to the high confinement levels, i.e. high sig3. They can be schematized by a decrease in the elastic modulus and the appearance of irreversible strains and work-hardening.

Hence, the existence of a threshold stress of confinement, sig3, has been assumed. It corresponds to the border between the two domains: sig3 = PCT.



"LCS" lcs "EPCS" epscs

< "LBIC" lbic > or < "LCT" lct "PCT" pct >
 < "LCD" lcd "PCD" pcd "EPCD" epscd >

lcs

Uniaxial compression limit.

epcs

Strain at rupture in uniaxial compression.

lbic

Limit in biaxial compression in the case of a plane stress analysis.

lct

Compression limit under a confinement pressure equal to the threshold confinement value (sig3).

pct

Threshold confinement pressure.

lcd

Compression limit under the pressure of ductile confinement.

ECR(1): hydrostatic pressure
 ECR(2): Von Mises criterion
 ECR(3): equivalent plastic strain
 ECR(4): crack angle in the (X-Y) plane (in degrees)
 ECR(5): yield limit in traction along direction 1
 ECR(6): yield limit in traction along direction 2
 ECR(7): yield limit in traction along direction 3
 ECR(8): crack opening in direction 1
 ECR(9): crack opening in direction 2
 ECR(10): crack opening in direction 3
 ECR(11): X component of the vector defining direction 1
 ECR(12): Y component of the vector defining direction 1
 ECR(13): Z component of the vector defining direction 1
 ECR(14): lambda(1) damage due to hydrostatic pressure
 ECR(15): lambda(2) damage due to the steady ductile Drucker criterion
 ECR(16): lambda(3) damage due to Von Mises criterion with hardening
 ECR(17): lambda(4) damage due to the steady brittle Drucker criterion
 ECR(18): lambda(5) damage due to the brittle Drucker criterion with hardening
 ECR(19): index of the damage criterion (0: no shear, 1: ductile shear, 2: brittle shear, 3: both).
 ECR(19): crack index (0: no crack, 1: one crack only, 2: two cracks, 3: three cracks).

Default values for an ordinary concrete:

All values are given in S.I. units.

- 1) Generic data:

RO	=	2.400E+03	Kg / m3
YOUN	=	37000E+06	Pa
NU	=	0.2100000	
ALPH	=	1.200E-05	
PREC	=	1.000E-03	

- 2) Data for the traction damage:

BETA	=	0.1000000	
LTR	=	4.440E+06	Pa
EPTR	=	3.600E-04	

- 3) Data for shear damage:

LCS	=	44.400E+06	Pa
EPCS	=	1.200E-02	
LBIC	=	111.000E+06	Pa
LCT	=	243.312E+06	Pa
PCT	=	71.040E+06	Pa
LCD	=	255.406E+06	Pa
PCD	=	79.920E+06	Pa
EPCD	=	6.000E-02	

- 4) Data for the hydrostatic pressure damage:

LPH	=	134.887E+06	Pa
PENT	=	7088.120E+06	Pa

9.6.9 CONCRETE: DYNAR LMT (BLMT)

Object:

Isotropic visco-damage and viscoplastic concrete material.

References:

- Gatuingt F. and Pijaudier-Cabot G., **Coupled damage and plasticity modelling in transient dynamic analysis of concrete**, *Int. J. Numer. Anal. Meth. Geomec.*, Vol 26, pp 1–24, 2002.
- Gatuingt F., **”Modèle de comportement BETON DYNAR LMT”**. Internal Report.

Syntax:

```
"BLMT"  "RO" rho  "YOUN" young  "NU" nu  "F0" f0
        "Q1" q1  "Q2" q2  "Q3" q3  "SGM0" sigM0  "XN" n
        "NVP" nvp  "MVP" mvp  "K" k  "MDT" mDt  "NDT" nDt
        "MDC" mDc  "NDC" nDc  "ED0" epsD0
        "AC" ac  "BC" bc  "AT" at  "BT" bt  /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

f0

Initial porosity of the concrete (0.3)

q1

Parameter of the modified Gurson plasticity criterion (0.5 to 2.)

q2

Parameter of the modified Gurson plasticity criterion (0.5 to 2.)

q3

Parameter of the modified Gurson plasticity criterion (0.5 to 2.)

sigM0

Resistance of the cement paste without pores (70 Mpa)

n

Exponent of the viscoplasticity threshold (15.)

nvp

Parameter of the Perzyna type viscoplasticity (1.5)

mvp

Parameter of the Perzyna type viscoplasticity (1.D-2)

k

Influence the porosity evolution (15 60)

mDt

Tension damage viscosity parameter (0.5D-4)

nDt

Tension damage viscosity parameter (5.)

mDc

Compression damage viscosity parameter (0.5D-3)

nDc

Compression damage viscosity parameter (20.)

epsD0

Strain tension threshold (1.D-04)

ac

Parameter for the compression (3000)

bc

Parameter for the compression (4.)

at

Parameter for the tension (20000)

bt

Parameter for the tension (1.6)

Comments:

1/ - BE CAREFUL the initial porosity influence the real young modulus

$$K_m = YOUNG / (3 * (1 - 2 * \nu))$$

$$G_m = YOUNG / (2 * (1 + \nu))$$

2/ - Compressibility and shear moduli with porosity f (Mori-Tanaka)

$$K_{poro} = 4 * X_{K_m} * X_{G_m} * (1 - f) / (4 * X_{G_m} + 3 * X_{K_m} * f)$$

$$G_{poro} = X_{G_m} * (1 - f) / (1 + f * (6 * X_{K_m} + 12 * X_{G_m}) / (9 * X_{K_m} + 8 * X_{G_m}))$$

3/ - Plasticity criterion FNT:

$$F = 3 * J_2(\text{SIG}) / \text{SGM}^{**2} + 2 * Q_1 * f * \cosh(Q_2 * I_1 / 2 * \text{SGM}) - (1 + (Q_3 * f)^{**2})$$

4/ - Plastic strain evolution:

$$\text{EPSP} = 1 / (1 - D) * (\text{FNT} / \text{MVP})^{**\text{NVP}} * d\text{FNT} / d\text{SIG}$$

5/ - Porosity evolution:

$$Df = K * f / (1 - f) * (\text{FNT} / \text{MVP})^{**\text{NVP}}$$

$$f(t + dt) = f(t) + df$$

6/ - Damage threshold function in tension and compression:

$$FD_i = (\text{EPSE} - \text{ED}_0 - 1 / A_i * (D_i / (1 - D_i))^{**2} / B_i)$$

7/ - Damage evolution in tension and compression:

$$D_i = (FD_i / MD_i)^{**ND_i}$$

Outputs:

The components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : Von Mises criterion

ECR(3) : Isotropic damage variable

ECR(4) : Material porosity

ECR(5) : xx plastic strain

ECR(6) : yy plastic strain

ECR(7) : zz plastic strain

ECR(8) : xy plastic strain

ECR(9) : yz plastic strain

ECR(10): zx plastic strain

ECR(11): Stress in the matrix without pores

ECR(12): Tension damage variable

ECR(13): Compression damage variable

ECR(14): Mazars threshold

9.6.10 CONCRETE: MAZARS-LINEAR ELASTIC LAW WITH DAMAGE

Object:

Isotropic linear elastic with a modified Mazars damage for concrete and brittle rupture materials.

References:

1- Jacky MAZARS, "Application de la mécanique de l'endommagement au comportement non linéaire et la rupture du béton de structure", Thèse de doctorat, Université Pierre et Marie Curie - Paris 6, 1984.

2- Yann CHUZEL-MARMOT, "Caractérisation expérimentale et simulation numérique d'impacts de glace haute vitesse", Thèse de doctorat, Université MEGA de Lyon - INSA Lyon, 2009.

Syntax:

```
"MAZA" "RO"    rho  "YOUN" young  "NU"    nu    "EPSD" epsd
      "DCRI" dcri "AT"    at      "AC"    ac    "BT"    bt
      "BC"    bc  "LCAR" lcar    "CSTA" csta "DCOE" dcoe
      "VCRI" vcri "VIMP" vimp                                /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

epsd

Initial strain threshold.

dcri

Critical value of damage (=1 per default).

at

Parameter of the tension law (asymptote of the curve stress-strain)

ac

	Parameter of the compression law (asymptote of the curve stress-strain)
bt	
	Parameter of the tension law (shape of the curve stress-strain)
bc	
	Parameter of the compression law (shape of the curve stress-strain)
lcar	
	Length parameter of the delay-damage
csta	
	Parameter of the delay-damage (=1 per default)
dcoe	
	Exponent of the sensitivity to the strain rate in tension ($=\frac{1}{3}$ per default)
vcrit	
	Critical velocity in tension (=1 per default)
vimp	
	Velocity impact of the body (or strain rate if it's not un'impact modeled)

Comments:

You can deactivate the delay effect with a negative value for the parameter **lcar**.

You can also deactivate the damage (so you obtain a linear material) with a negative value for the parameter **epsd**.

Outputs:

The components of the ECR table are as follows:

- ECR(1) : Pressure
- ECR(2) : Von Mises criterion
- ECR(3) : equivalent deformation
- ECR(4) : Global Damage
- ECR(5) : Level "traction/compression"
- ECR(6) : Strain rate
- ECR(7) : Threshold damage
- ECR(8) : Damage in traction
- ECR(9) : Damage in compression
- ECR(10): Factor of dynamic amplification in traction
- ECR(11): Bc parameter eventually corrected

9.6.11 DADC: Dynamic Anisotropic Damage Concrete

Object:

Concrete material with induced anisotropic damage represented by one damage variable and modeling biaxial behavior.

Reference:

Armand Leroux, *Modèle multiaxial d'endommagement anisotrope: Gestion numerique de la rupture et application à la ruine des structures en béton armé sous impacts*. Thèse LaMSID-UMR EDF/CNRS/CEA (2012)[[724](#)]

Syntax:

```
"DADC"  "RO" rho      "YOUN" young  "NU" nu      "SIGT" sigyt
        "SIGC" sigyc  "SGBC" sigybc  "ALPH" alpha
        "BETA" beta   "BT"  bt      "DC"  dc
        <"XINF" xinf>   <"BV" bv>     <"DTFI" dtfi>
        <"TCS" tcs>     /LECTURE/
```

rho

Density

young

Young's modulus

nu

Poisson's ratio

sigyt

Elastic limit for the tension

sigyc

Elastic limit for the compression in absolute value

sigybc

Elastic limit for the bi-compression in absolute value

alpha

Damage parameter ALPH. This parameter allows to modify peak values in tension and compression strengths.

beta

Damage parameter BETA. This parameter allows to modify the post-peak behavior in compression and bi-compression

bt

Parameter of the function $b(T_x)$. It is used for Hillerborg regularization.

dc

Critical value of the damage for the numerical control of rupture. (0.9 to 1)

*Optional parameters***dinf**

Delay damage parameter (suggested value: 50000. s-1)

bv

Delay damage parameter (suggested value: 1.)

dtfi

Activating calculation of time step in the behavior law with a value of first time step (recommended value, 1E-8 s.). The parameter

tcs

Formulation of the selected function $b(T_x)$ (1: Formulation TCS1(default), 2: Formulation TCS2)

Outputs:

The components of the ECR table are as follows:

- ECR(1) : pressure
- ECR(2) : Von Mises criterion
- ECR(3) : Damage Dxx
- ECR(4) : Damage Dyy
- ECR(5) : Damage Dzz
- ECR(6) : Damage Dxy
- ECR(7) : Damage Dyz
- ECR(8) : Damage Dzx
- ECR(9) : Rotation matrix for the eigenvector basis damage matrix (xx)
- ECR(10): Rotation matrix for the eigenvector basis damage matrix (xy)
- ECR(11): Rotation matrix for the eigenvector basis damage matrix (xz)
- ECR(12): Rotation matrix for the eigenvector basis damage matrix (yx)
- ECR(13): Rotation matrix for the eigenvector basis damage matrix (yy)

- ECR(14): Rotation matrix for the eigenvector basis damage matrix (yz)
- ECR(15): Rotation matrix for the eigenvector basis damage matrix (zx)
- ECR(16): Rotation matrix for the eigenvector basis damage matrix (zy)
- ECR(17): Rotation matrix for the eigenvector basis damage matrix (zz)
- ECR(18): Critical state damage flag
- ECR(19): Damage rate
- ECR(20): Equivalent effective stress
- ECR(21): 1st eigen value basis damage matrix
- ECR(22): 2nd eigen value basis damage matrix
- ECR(23): 3rd eigen value basis damage matrix
- ECR(24): The biggest three eigen values basis damage matrix
- ECR(25): Proposed time step
- ECR(26): Filtered stress tensor after five time steps
- ECR(27): Time step first flag
- ECR(28): Estimation error flag (0=ok,1=error)
- ECR(29): stress triaxiality
- ECR(30): Component of filtered stress tensor (xx)
- ECR(31): Component of filtered stress tensor (yy)
- ECR(32): Component of filtered stress tensor (zz)
- ECR(33): Component of filtered stress tensor (xy)
- ECR(34): Component of filtered stress tensor (yz)
- ECR(35): Component of filtered stress tensor (zx)
- ECR(36): Number of times that the damage criterion (for the calculation of the time step in the behavior law) is not respected
- ECR(37): Largest components (absolute values) of the strain rate tensor

9.6.12 DPDC: Dynamic Plastic Damage Concrete

Object:

DPDC A three-invariant cap model with mixed hardening and isotropic damage for concrete material.

Reference:

Rapport (à sortir)

Comments:

All values must be given in SI units.

Syntax:

```
"DPDC"  "RO" rho      "YOUN" young  "NU" nu      "FC" fc
        "DAGG" dagg   "VERS" vers
        <"GFT" gft      "GFC" gfc      "GFS" gfs>
        <"NH" nh>       <"CH" ch>       <"PWR" pwrc>
        <"PWR" pwrt>    <"B" b>        <"D" d>
        <"MAXC" maxc>   <"MAXT" maxt>   <"STR" str>
        <"R" r>         <"XO" xo>      <"W" w>
        <"D1" d1>       <"D2" d2>      <"TXCA" txca
        "TXCT" txct     "TXCL" txcl     "TXCB" txcB>
        <"TXEA" txea     "TXET" txet     "TXEL" txel
        "TXEB" txeB>    <"PMOD" pmod>   <"NC" nc
        "NOC" noc>      <"NT" nt       "NOT" not>
        <"REPW" repw>   <"RECO" reco>   <"EROD" erod>
        <"PRED" pred>
```

/LECTURE/

rho

Density

youn

Young's modulus

nu

Poisson's ratio

fc

Unconfined compression stress

dagg

Maximum aggregate size

vers

Version number (1 = plastic behavior with isotropic hardening only; 2 = plastic behavior with mixed hardening: isotropic and kinemat

Optional parameters

gft

Tensile fracture energy at $f_c = 10\text{MPa}$ (interpolated as a function of the maximum aggregate size) (J)

gfc

Compressive fracture energy at $f_c = 10\text{MPa}$ (default value: 100 gft) (J)

gfs

Shear fracture energy at $f_c = 10\text{MPa}$ (default value: gft) (J)

nh

Hardening initiation (default value: 0.8) (without unit)

ch

Kinematic hardening constant (default value: $1\text{E}+09$ Pa)

pwr c

Shear-to-compression transition parameter (default value: 5.) (without unit)

pwr t

Shear-to-tension transition parameter (default value: 1.) (without unit)

b

Ductile shape softening parameter (default value: 100) (without unit)

d

Brittle shape softening parameter (default value: 0.1) (without unit)

max c

Maximum overstress allowed in compression (interpolated as a function of the material strength in compression) (Pa)

maxt

Maximum overstress allowed in tension (interpolated as a function of the material strength in compression) (Pa)

str

Ratio of effective shear stress to tensile stress fluidity parameter (default value: 1.) (without unit)

r

Cap aspect ratio (default value: 5.) (without unit)

xo

Cap initial location (interpolated as a function of the material strength in compression) (Pa)

w

Maximum plastic volume compaction (default value: 0.05) (without unit)

d1

Linear shape parameter of the cap (default value: $2.5\text{E-}10 \text{ Pa}^{-1}$)

d2

Quadratic shape parameter of the cap (default value: $3.49\text{E-}19 \text{ Pa}^{-2}$)

txca

TXC surface constant term (TXC: triaxial compression) (interpolated as a function of the material strength in compression) (Pa)

txct

TXC surface linear term (interpolated as a function of the material strength in compression) (without unit)

txcl

TXC surface nonlinear term (interpolated as a function of the material strength in compression) (Pa)

txcb

TXC surface exponent (interpolated as a function of the material strength in compression) (Pa^{-1})

txea

TXE surface constant term (TXE: triaxial extension) (interpolated as a function of the material strength in compression) (without unit)

txet

TXE surface linear term (interpolated as a function of the material strength in compression) (Pa^{-1})

txel

TXE surface nonlinear term (interpolated as a function of the material strength in compression) (without unit)

txeb

TXE surface exponent (interpolated as a function of the material strength in compression) (Pa^{-1})

pmod

Modify moderate pressure softening parameter (default value: 0.) (without unit)

nc

Rate effects power for uniaxial compressive stress (default value: 0.78) (without unit)

noc

Rate effects parameter for uniaxial compressive stress (interpolated as a function of the material strength in compression). Default

nt

Rate effects power for uniaxial tensile stress (default value: 0.48) (without unit)

not

Rate effects parameter for uniaxial tensile stress (interpolated as a function of the material strength in compression). Default unit

repw

Power which increases fracture energy with rate effects (default value: 1.) (without unit)

reco

Option to recover stiffness in compression from tensile damage (default value: 0.) (without unit)

erod

Option to erode with strain at which erosion initiates: 0 = no erosion allowed ; 1 = erosion allowed (default value: 0)

pred

Damage level for predamaged concrete (default value: 0.) (without unit)

Outputs:

The components of the ECR table are as follows:

- ECR(1) : Pressure
- ECR(2) : Von Mises criterion
- ECR(3) : Equivalent plastic strain
- ECR(4) : Third deviatoric stress invariant
- ECR(5) : Lode angle
- ECR(6) : Total variation of the isotropic hardening parameter
- ECR(7) : Volumetric strain
- ECR(8) : Plastic volumetric strain
- ECR(9) : Ductile damage parameter
- ECR(10): Brittle damage parameter
- ECR(11): Ductile damage threshold
- ECR(12): Brittle damage threshold
- ECR(13): Effective strain rate

9.6.13 DAMAGE

Object:

This option allows to associate to the materials VON MISES ISOTROPE and VON MISES PARFAIT different damage laws, and to request the calculation of several fracture criteria. Now, only one criterion (Tuler-Butcher) is available.

Syntax:

```
"CRIT"  $[ "TULE" <"SIGL" sigl > <"EPSL" epsl > <"TAUL" taul >
          "SIGS" sigs      "LAMB" lamb  <"KER"  ker  > ]$
/LECTURE/
```

"CRIT"

Indicates that the calculation of different damage criteria is required.

"TULE"

The TULER-BUTCHER's criterion is selected.

sigl

Maximum principal stress criterion.

epsl

Maximum volumetric deformation criterion.

taul

Octahedral shear stress criterion.

sigs

First parameter of the Tuler-Butcher law.

lamb

Second parameter of the Tuler-Butcher law.

ker

Third parameter of the Tuler-Butcher law.

LECTURE

List of the concerned elements.

Comments:

The element types accepting these materials are: in 2D elements TRIA, CAR1 and CAR4 and in 3D elements CUBE, CUBE6, CUBE8, PRIS, TETR and PRI6.

Currently the damage model is only available in association with the materials Isotropic Von Mises, Steinberg-Guinan and dynamic Von Mises.

The isotropic Von Mises material must appear first in the input file, before the calculation of the damage criteria, if any, and one of the two damage laws, if necessary.

The Tuler-Butcher is given by the following expression where $\sigma_1, \sigma_2, \sigma_3$ representing the principal stresses:

$$\int_0^t (Max(\sigma_1, \sigma_2, \sigma_3) - \sigma_s)^\lambda dt < \text{ker}$$

The results stored in the ECR table (5 values) are, according to the material type:

Tuler-Butcher :

- ECR(1) = Maximum principal stress
- ECR(2) = Maximum principal deformation
- ECR(3) = Octahedral shear stress
- ECR(4) = Volume deformation
- ECR(5) = Tuler-Butcher criterion

Example:

A criterion "LOI 3", is associated with the principal material "LOI 1".

The corresponding data will be for example:

```

LOI  1      VMIS ISOT RO 7800.  YOUN 74020E6  NU .3  ELAS 350E6
                ENDO 3
                TRAC 4
                        350.E6          .472845E-2
                        476.26E6        7.2835E-2
                        518.51E6        15.700E-2
                        538.03E6        21.607E-2
                        550.24E6        26.083E-2
                LECT TOUS

LOI  3      CRIT TULE SIGS 1E7 LAMB 1.0
                LECT TOUS

```

9.6.14 EOBT: ANISOTROPIC DAMAGE OF CONCRETE (EDF)**Object:**

Concrete material with induced anisotropic damage.

Reference:

V. Godard, Modélisation de l'endommagement anisotrope du béton avec prise en compte de l'effet unilatéral : Application à la simulation numérique des enceintes de confinement, Thèse de l'Université Paris VI, 2005.

M. Bottoni, Loi de comportement ENDO_ORTH_BETON, Manuel de référence de Code_Aster, R7.01.09.

Syntax:

```
"EOBT"  "RO" rho    "YOUN" young  "NU" nu    "K0" k0
        "K1" k1    "K2" k2    "ECRB" ecrb    "ECD" ecd
        < "DC" dc > < "DM" dm >    /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

k0

Threshold in stress for the tension.

k1

Parameter for the threshold in stress in compression.

k2

Parameter for the threshold in stress in compression.

ecrb

Parameter driving the evolution of the loading surface while the damage tensor B is growing.

ecrd

Parameter driving the evolution of the loading surface while the damage scalar d is growing.

Optional parameters:**dc**

Limit value for the eigenvalues of the damage tensor B and for the damage scalar d . When this limit is reached, the material is cons

dm

Imposed value for damage when it reaches its limit value. By default, dm is set to 0.999

Outputs:

The components of the ECR table are as follows:

ECR(1) : pressure
ECR(2) : Damage in compression D
ECR(3) : Damage D_{xx}
ECR(4) : Damage D_{yy}
ECR(5) : Damage D_{zz}
ECR(6) : Damage D_{xy}
ECR(7) : Damage D_{yz}
ECR(8) : Damage D_{zx}
ECR(9) : Rotation matrix for the eigenvector basis damage matrix (xx)
ECR(10): Rotation matrix for the eigenvector basis damage matrix (xy)
ECR(11): Rotation matrix for the eigenvector basis damage matrix (xz)
ECR(12): Rotation matrix for the eigenvector basis damage matrix (yx)
ECR(13): Rotation matrix for the eigenvector basis damage matrix (yy)
ECR(14): Rotation matrix for the eigenvector basis damage matrix (yz)
ECR(15): Rotation matrix for the eigenvector basis damage matrix (zx)
ECR(16): Rotation matrix for the eigenvector basis damage matrix (zy)
ECR(17): Rotation matrix for the eigenvector basis damage matrix (zz)
ECR(21): 1st eigen value of the damage tensor B
ECR(22): 2nd eigen value of the damage tensor B
ECR(23): 3rd eigen value of the damage tensor B

9.6.15 LINEAR MULTI-LAYER

Object

This directive allows to define materials obtained by homogenisation through the thickness of different layers (or plies) each having a linear orthotropic behaviour.

Syntax:

```
"MCOU"      | [    "BACON"      ibacon      ;
               "NBCOUCHE"  ...           /LECTURE/  ] |
```

For the user data option (NBCOUCHE) :

```
... "NBCOUCHE"  nbcouche  "ZMIN"  zmin
```

```
nbcouche times  :
```

```
|  "ZMAX"  zmax  "TETA"  teta  "ROCO"  roco  "YG1"  yg1  |
|  "YG2"   yg2   "NU12"  nu12  "G12"   g12                |
| < "G13"  g13 > < "G23"  g23 > "TERM"                    |
```

ibacon

Logical unit number of the BACON file from which the characteristics of this material will be read. Using this option implies the necessity to introduce the keyword "MBACON" in part A of the input file (see page A.30) in order to dimension the arrays used by this model.

NBCOUCHE

The characteristics will be listed below.

nbcouche

Numbers of layers of the composite.

zmin

Minimum side of the first layer.

zmax

Maximum side of the current layer.

teta

Angle (in degrees) of the first vector of the orthotropy frame of the current layer with respect to the first side of the element.

roco

Density of the current layer.

yg1

Young's modulus along direction 1 of the current layer.

yg2

Young's modulus along direction 2 of the current layer.

nu12

Poisson's coefficient among directions 1-2.

g12

Shear modulus among directions 1-2.

g13

Shear modulus among directions 1-3.

g23

Shear modulus among directions 2-3.

TERM

Indicates that the data for layer i are terminated.

LECTURE

List of the concerned elements.

Comments:

When the BACON option is used, EUROPLEXUS reads the numbers of the elements associated with this material directly from the BACOn file: the /LECTURE/ procedure is redundant. Currently, one may read only one type of laminated material per calculation. On the other hand, EUROPLEXUS will write on the logical unit (ibacon+1):

- 1) the element number (1 value)
- 2) the angle (in degrees) between the first side and the first direction of the laminated (1 value)
- 3) the components of the symmetric matrices A, B and D
(A(1,1), A(2,1),A(2,2), A(3,1),A(3,2),A(3,3)...) (3x6= 18 values).

For the NBCOUCHE option, the various layers must be described in growing order of z. In particular, $z_{\max}(\text{couche}_i) = z_{\min}(\text{couche}_{i+1})$.

The value $z=0$ corresponds to the neutral fiber of the element. This material allows to take excentricity into account.

For the shells that consider transverse shears, i.e. DST3, Q4G4, Q4GR, Q4GS, it is necessary to give the values of G23 and G13.

Outputs:

The various components of the ECR table (values computed in the local reference of the shell element) are as follows:

Element COQ3:

- ECR(1) : Von Mises on the lower face of the shell
- ECR(2) : Von Mises on the upper face of the shell
- ECR(3) : $-d^3w/dx^3$ at the integration point
- ECR(4) : $-d^3w/dy^3$ at the integration point
- ECR(5) : $-d^3w/dx^2dy$ at the integration point
- ECR(6) : $-d^3w/dxdy^2$ at the integration point

Elements DKT3 and DST3:

- ECR(1) : Von Mises on the lower face of the shell
- ECR(2) : Von Mises on the upper face of the shell
- ECR(3) : $d^2\beta_{a_x}/dx^2$ at the first integration point
- ECR(4) : $d^2\beta_{a_x}/dy^2$ at the first integration point
- ECR(5) : $d^2\beta_{a_x}/dxdy$ at the first integration point
- ECR(6) : $d^2\beta_{a_y}/dx^2$ at the first integration point
- ECR(7) : $d^2\beta_{a_y}/dy^2$ at the first integration point
- ECR(8) : $d^2\beta_{a_y}/dxdy$ at the first integration point

Recall that the table of deformations EPST is composed by the following parameters (computed at the integration point):

EPST(1) : du/dx (membrane deformation e_{xx})
 EPST(2) : dv/dy (membrane deformation e_{yy})
 EPST(3) : $du/dy + dv/dx$ (membrane deformation $2e_{xy}$)
 EPST(4) : $dbeta_x/dx$ ($= -d^2w/dx^2$ if thin shell)
 EPST(5) : $dbeta_y/dy$ ($= -d^2w/dy^2$ if thin shell)
 EPST(6) : $dbeta_y/dx + dbeta_x/dy$
 EPST(7) : $2*epsi_{xz}$ (eventually)
 EPST(8) : $2*epsi_{yz}$ (eventually)

Example:

We assume a composite formed by 6 layers regularly spaced on a thickness of 0.12 m. The corresponding data will be:

```

MCOUCH      NBCOUCHE  6          ZMIN -0.06
      ZMAX -0.04  TETA  5.  ROC0 2.5E3  YG1 40E9 YG2 20E9
              NU12 0.2  G12 16.6666667E9 TERM
      ZMAX -0.02  TETA 36.  ROC0 2.5E3  YG1 40E9 YG2 25E9
              NU12 0.2  G12 16.6666667E9 TERM
      ZMAX -0.00  TETA 48.  ROC0 2.5E3  YG1 40E9 YG2 20E9
              NU12 0.2  G12 16.6666667E9 TERM
      ZMAX  0.02  TETA 135 ROC0 2.5E3  YG1 40E9 YG2 20E9
              NU12 0.2  G12 16.6666667E9 TERM
      ZMAX  0.04  TETA 33.  ROC0 2.5E3  YG1 40E9 YG2 20E9
              NU12 0.2  G12 16.6666667E9 TERM
      ZMAX  0.06  TETA 15.  ROC0 2.5E3  YG1 40E9 YG2 40E9
              NU12 0.2  G12 16.6666667E9 TERM
                                LECT 3  4 5 TERM

```


9.6.16 CHANG-CHANG MULTI-LAYER MODEL

Object:

This directive allows to define composite materials using the CHANG-CHANG criterion, as described in:

A Progressive Damage Model of Laminated Composites
 Containing Stress Concentrations
 by F.-K. CHANG and K.-Y. CHANG
 in Journal of Composite Materials, Vol. 21, Sept. 1987.

Syntax:

```
"CHANG"      |[  "BACON"      ibacon      ;
                $  "NBCOUCHE"  ... PBASE .... /LECTURE/      $
```

For the user data option (NBCOUCHE) :

```
... "NBCOUCHE" nbcouche "ZMIN" zmin
```

```
nbcouche times :
```

```
| "ZMAX" zmax "TETA"  tet  "ROCO" roco "YG1"  yg1 |
| "YG2"  yg2  "NU12" nu12 "G12"  g12  |
| "XT"   xt   "XC"   xc   "YT"   yt   "YC"   yc   |
| "SC"   sc   "A0"   a0   "BETA" beta   |
| "TERM" |
```

Once described the layers, one gives 2 points in order to define a reference direction:

```
"PBASE"      "LECTURE" nod1 nod2 "TERM"
```

ibacon

Logical unit number of the BACON file from which the characteristics of this material will be read. Using this option implies the necessity to introduce the keyword "MBACON" in part A of the input file (see page A.30) in order to dimension the arrays used by this model.

NBCOUCHE

The characteristics will be listed below in the main input file.

nbcouche

Number of layers in the composite.

zmin

Minimum side of the first layer.

zmax

Maximum side of the current layer.

teta

Angle (in degrees) of the first vector of the orthotropy frame of the current layer with respect to the reference direction.

roco

Density of the current layer.

yg1

Young's modulus along direction 1 of the current layer.

yg2

Young's modulus along direction 2 of the current layer.

nu12

Poisson's coefficient among directions 1-2.

g12

Shear modulus among directions 1-2.

xt

Traction limit along direction 1 of the orthotropy frame.

xc

Compression limit along direction 1 of the orthotropy frame.

yt

Traction limit along direction 2 of the orthotropy frame.

yc

Compression limit along direction 2 of the orthotropy frame.

sc

Shear limit 1-2 of the orthotropy frame.

a0

Critical area a0 of the CHANG-CHANG criterion.

beta

Weibull coefficient.

TERM

Indicates that the data for layer i are terminated.

nod1,nod2

Numbers of 2 nodes defining the reference direction.

LECTURE

List of the concerned elements.

Comments:

When the BACON option is used, EUROPLEXUS reads the numbers of the associated elements directly from the BACON file: the procedure /LECTURE/ is redundant. For this material, the number of laminas is unlimited. However, one should give the adequate numbers in the dimensioning section of the input file: see the key-words MATE and ECRO in the DIMENSIONS directive.

For the NBCOUCHE option, the various layers must be described along increasing order of z. In particular, $z_{\max}(\text{couche}_i) = z_{\min}(\text{couche}_{i+1})$.

The value $z=0$ corresponds to the neutral fiber of the element. This material allows to account for excentricity.

Outputs:

This constitutive law computes the damages appearing in each plie of the laminated structure. To this end, it is necessary to define the damage parameters in each layer. In each ply, the ECR table is dimensioned at 10, and the main parameters are:

DIMENSION ECR(10,NPLIS)

ECR(2,ipli) : Von Mises of the ply

ECR(3,ipli) : Rupture criterion of the matrix in traction

ECR(4,ipli) : Rupture criterion of the matrix in compression

ECR(5,ipli) : Rupture criterion of the fiber, or fiber-matrix delamination.

9.6.17 LINEAR ORTHOTROPY

Object:

The directive is used to enter materials with a linear orthotropic behaviour into a coordinate system defined by the user. The model is described in: *Mécanique des Matériaux Solides* (J-Lemaitre, L-Chaboche. Ed: Dunod, 1986).

Syntax:

```
"ORTH"  "R0"  rho    "YG1"  yg1    "YG2"  yg2    "YG3"  yg3
         "NU12" nu12  "NU13" nu13  "NU23" nu23
         "G12"  g12   "G13"  g13   "G23"  g23      /LECTURE/
```

rho

Density of the material.

yg1

Young's modulus along direction 1.

yg2

Young's modulus along direction 2.

yg3

Young's modulus along direction 3.

nu12

Poisson's ratio between directions 1 and 2.

nu13

Poisson's ratio between directions 1 and 3.

nu23

Poisson's ratio between directions 2 and 3.

g12

Shear modulus between directions 1 and 2.

g13

Shear modulus between directions 1 and 3.

g23

Shear modulus between directions 2 and 3.

LECTURE

List of the elements concerned.

Comments:

This option may be repeated as many times as necessary.

The associated coordinate system is defined by the option "CORTHO" (see page C1.95) for the multilayer element CMC3.

The associated coordinate system is defined by the option "MORTHO" (see page C1.96) for the continuum elements in 3D and in plane strain.

The associated coordinate system is defined by the directive "COMPLEMENT" (pages C1.95 and C1.96):

- "COMPLEMENT" "CORTHO" for the shells;
- "COMPLEMENT" "MORTHO" for the continuum elements 3D and 2D plane strain and axisymmetric.

Verify that this material is available for your elements, by means of the tables of page C.100.

Outputs:

The different components of the ECR table are as follows, for the CMC3 element:

ECR(1): Von mises criterion on the lower face of the multilayer element CMC3.

ECR(2): Vom mises criterion on the upper face of the multilayer element CMC3.

The different components of the ECR table are as follows, for the continuum elements:

ECR(1): pressure.

ECR(2): Vom mises criterion.

9.6.18 ORTS : LINEAR ORTHOTROPY (Local basis)**Object:**

The directive is used to enter materials with a linear orthotropic behaviour into a coordinate system defined by the user. The model is described in: *Mécanique des Matériaux Solides* (J-Lemaitre, L-Chaboche. Ed: Dunod, 1986).

Stress and strain are explained in user coordinate system.

Syntax:

```
"ORTS"  "R0"  rho    "YG1"  yg1    "YG2"   yg2  "YG3"  yg3
         "NU12" nu12  "NU13" nu13  "NU23" nu23
         "G12"  g12   "G13"  g13   "G23"  g23      /LECTURE/
```

rho

Density of the material.

yg1

Young's modulus along direction 1.

yg2

Young's modulus along direction 2.

yg3

Young's modulus along direction 3.

nu12

Poisson's ratio between directions 1 and 2.

nu13

Poisson's ratio between directions 1 and 3.

nu23

Poisson's ratio between directions 2 and 3.

g12

Shear modulus between directions 1 and 2.

g13

Shear modulus between directions 1 and 3.

g23

Shear modulus between directions 2 and 3.

LECTURE

List of the elements concerned.

Comments:

This option may be repeated as many times as necessary.

The associated coordinate system is defined by the option "CORTHO" (see page C1.95) for the multilayer element CMC3.

The associated coordinate system is defined by the option "MORTHO" (see page C1.96) for the continuum elements in 3D and in plane strain.

The associated coordinate system is defined by the directive "COMPLEMENT" (pages C1.95 and C1.96):

- "COMPLEMENT" "CORTHO" for the shells;
- "COMPLEMENT" "MORTHO" for the continuum elements 3D and 2D plane strain and axisymmetric.

Verify that this material is available for your elements, by means of the tables of page C.100.

Outputs:

The different components of the ECR table are as follows, for the CMC3 element:

ECR(1): Von mises criterion on the lower face of the multilayer element CMC3.

ECR(2): Vom mises criterion on the upper face of the multilayer element CMC3.

The different components of the ECR table are as follows, for the continuum elements:

ECR(1): pressure.

ECR(2): Vom mises criterion.

9.6.19 ORTE : ELASTIC DAMAGE ORTHOTROPY (only in 3D)**Object:**

The directive is used to enter materials with a orthotropic (local) behaviour into a coordinate system defined by the user, coupling with damage. There is coupling between damage (as material LEM1) and orthotropy (as material ORTS). There are 6 damages : $d_1, d_2, d_3, d_{12}, d_{13}, d_{23}$. Each damage evolution rate is a function of strain tensor. A failure criterion is implicitly contains within the model: rupture occurs when a damage exceeds a critical value.

Two parameters (for each damage) allow to introduce a limitation of the damage rate (thanks to the delayed damage effect) in order to avoid the mesh dependency.

$$D_{nc} = dc < \frac{\epsilon - ep}{s0 - ep} >$$

$$\dot{d} = \frac{1}{to} \left(1 - e^{-a < D_{nc} - d >} \right)$$

Syntax:

"ORTE"	"R0"	rho	"YG1"	yg1	"YG2"	yg2	"YG3"	yg3
	"NU12"	nu12	"NU13"	nu13	"NU23"	nu23	"G12"	g12
	"G13"	g13	"G23"	g23	"EL1"	e11	"EL2"	e12
	"EL3"	e13	"EL12"	e112	"EL13"	e113	"EL23"	e123
	"EP1"	ep1	"EP2"	ep2	"EP3"	ep3	"EP12"	ep12
	"EP13"	ep13	"EP23"	ep23	"DC1"	dc1	"DC2"	dc2
	"DC3"	dc3	"DC12"	dc12	"DC13"	dc13	"DC23"	dc23
	"A1"	a1	"A2"	a2	"A3"	a3	"A12"	a12
	"A13"	a13	"A23"	a23	"T01"	to1	"T02"	to2
	"T03"	to3	"T012"	to12	"T013"	to13	"T023"	to23
	"S01"	s01	"S02"	s02	"S03"	s03	"S012"	s012
	"S013"	s013	"S023"	s023	END0	endo	/LECTURE/	

rho

Density of the material.

yg1

Young's modulus along direction 1.

yg2

Young's modulus along direction 2.

yg3

Young's modulus along direction 3.

nu12

Poisson's ratio between directions 1 and 2.

nu13

Poisson's ratio between directions 1 and 3.

nu23

Poisson's ratio between directions 2 and 3.

g12

Shear modulus between directions 1 and 2.

g13

Shear modulus between directions 1 and 3.

g23

Shear modulus between directions 2 and 3.

e11

Elastic limit in direction 1.

e12

Elastic limit in direction 2.

e13

Elastic limit in direction 3.

e112

Elastic limit in direction 12.

e113

Elastic limit in direction 13.

e123

Elastic limit in direction 23.

ep1

Damage threshold in direction 1.

ep2

Damage threshold in direction 2.

ep3

Damage threshold in direction 3.

ep12

Damage threshold in direction 12.

ep13

Damage threshold in direction 13.

ep23

Damage threshold in direction 23.

dc1

Critical damage defining the rupture criterion, in direction 1.

dc2

Critical damage defining the rupture criterion, in direction 2.

dc3

Critical damage defining the rupture criterion, in direction 3.

dc12

Critical damage defining the rupture criterion, in direction 12.

dc13

Critical damage defining the rupture criterion, in direction 13.

dc23

Critical damage defining the rupture criterion, in direction 23.

a1

Parameter of the delayed damage model, for the direction 1.

a2

Parameter of the delayed damage model, for the direction 2.

a3

Parameter of the delayed damage model, for the direction 3.

a12

Parameter of the delayed damage model, for the direction 12.

a13

Parameter of the delayed damage model, for the direction 13.

a23

Parameter of the delayed damage model, for the direction 23.

to1

Characteristic time of the delayed damage model in direction 1. $(1/to)$ represents the maximum damage rate.

to2

Characteristic time of the delayed damage model in direction 2. $(1/to)$ represents the maximum damage rate.

to3

Characteristic time of the delayed damage model in direction 3. $(1/to)$ represents the maximum damage rate.

to12

Characteristic time of the delayed damage model in direction 12. $(1/to)$ represents the maximum damage rate.

to13

Characteristic time of the delayed damage model in direction 13. $(1/to)$ represents the maximum damage rate.

to23

Characteristic time of the delayed damage model in direction 23. $(1/to)$ represents the maximum damage rate.

s01

Parameter driving the damage evolution rate in direction 1.

s02

Parameter driving the damage evolution rate in direction 2.

s03

Parameter driving the damage evolution rate in direction 3.

s012

Parameter driving the damage evolution rate in direction 12.

s013

Parameter driving the damage evolution rate in direction 13.

s023

Parameter driving the damage evolution rate in direction 23.

endo

Parameter driving the post damage evolution : case 0. : nothing special happens (default case is 0.) ; case 1. : if a damage is critical at a Gauss point of the element, then the element becomes Ghost ; case 2. : if a damage is critical at all Gauss points of the element, then the element becomes Ghost.

LECTURE

List of the elements concerned.

Comments:

This option may be repeated as many times as necessary.

The associated coordinate system is defined by the option "MORTH0" (see page C1.96) for the continuum elements in 3D and in plane strain.

The associated coordinate system is defined by the directive "COMPLEMENT" (pages C1.95 and C1.96):

- "COMPLEMENT" "CORTH0" for the shells;
- "COMPLEMENT" "MORTH0" for the continuum elements 3D and 2D plane strain and axisymmetric.

Verify that this material is available for your elements : pr6, cub8, tetr, prism.

Outputs:

The different components of the ECR table are as follows, for the continuum elements:

ECR(2): Vom mises criterion.

ECR(8) : d1 - damage in direction 1.

ECR(9) : d2 - damage in direction 2.

ECR(10): d3 - damage in direction 3.

ECR(11): d12 - damage in direction 12.

ECR(12): d13 - damage in direction 13.

ECR(13): d23 - damage in direction 23.

ECR(14): d1 - damage not corrected (before delay effect) in direction 1.

ECR(15): d2 - damage not corrected (before delay effect) in direction 2.

ECR(16): d3 - damage not corrected (before delay effect) in direction 3.

ECR(17): d12 - damage not corrected (before delay effect) in direction 12.

ECR(18): d13 - damage not corrected (before delay effect) in direction 13.

ECR(19): d23 - damage not corrected (before delay effect) in direction 23.

ECR(20): T - time.

9.6.20 ODMS : ONERA DAMAGE MODEL (only in 3D)

Object:

The directive is used to enter materials with a orthotropic (local) behaviour into a coordinate system defined by the user, coupling with damage. There is coupling between damage (as material LEM1) and orthotropy (as material ORTS). There are 3 matrix damages : d_1^m, d_2^m, d_3^m (direction 1, 2 and 3), and 4 fiber damages : $d_1^f, d_2^f, d_3^f, d_4^f$ (traction direction 1, compression direction 1, traction direction 2, compression direction 2). Each damage evolution rate is a function of strain tensor following the ONERA Damage Mechanics law.

Cf. Maire J.F., Chaboche J.L. "A New Formulation of Continuum Damage Mechanics (ODM) for Composite material". Aerospace Science and Technology vol 4 (1997) 247-257

Two parameters (for each damage) allow to introduce a limitation of the damage rate (thanks to the delayed damage effect) in order to avoid the mesh dependency.

$$\dot{d} = \frac{1}{\tau} \left(1 - e^{-a \langle D_{nc} - d \rangle} \right)$$

The constitutive law is :

$$C_{eff}^{-1} = S_{eff} = S_0 + \sum_{i=1}^3 \eta_i d_i^m H_{0i}^m + \sum_{j=1}^4 d_j^f H_{0j}^f = C_{matrix}^{-1} + \sum_{j=1}^4 d_j^f H_{0j}^f$$

$$[\sigma] = C_{matrix} \cdot [\epsilon] - C_{eff} \cdot [\epsilon_{residual}]$$

where the matrix 6x6 S_0 and H_0^i are defined as:

Compliance S_0 : $S_0(1,1) = 1/E_1^0$, $S_0(2,2) = 1/E_2^0$, $S_0(3,3) = 1/E_3^0$; $S_0(1,2) = -\nu_{12}/E_1^0$, $S_0(1,3) = -\nu_{13}/E_1^0$, $S_0(2,3) = -\nu_{23}/E_2^0$; $S_0(4,4) = 1/G_{12}^0$, $S_0(5,5) = 1/G_{23}^0$, $S_0(6,6) = 1/G_{13}^0$, and 0 otherwise.

Matrix damage effect tensors $H_0^{m1}, H_0^{m2}, H_0^{m3}$:

H_0^{m1} : $H_0^{m1}(1,1) = h_1^n/E_1^0$, $H_0^{m1}(4,4) = h_1^p/G_{12}^0$, $H_0^{m1}(6,6) = h_1^{pn}/G_{13}^0$, and 0 otherwise.

H_0^{m2} : $H_0^{m2}(2,2) = h_2^n/E_2^0$, $H_0^{m2}(4,4) = h_2^p/G_{12}^0$, $H_0^{m2}(5,5) = h_2^{pn}/G_{23}^0$, and 0 otherwise.

H_0^{m3} : $H_0^{m3}(3,3) = h_3^n/E_3^0$, $H_0^{m3}(5,5) = h_3^p/G_{23}^0$, $H_0^{m3}(6,6) = h_3^{pn}/G_{13}^0$, and 0 otherwise.

Fiber damage effect tensors $H_0^{f1}, H_0^{f2}, H_0^{f3}, H_0^{f4}$:

H_0^{f1} : $H_0^{f1}(1,1) = hf_1^i/E_1^0$, $H_0^{f1}(2,2) = hf_2^i/E_2^0$, $H_0^{f1}(3,3) = hf_3^i/E_3^0$, $H_0^{f1}(1,2) = H_0^{f1}(2,1) = hf_4^i.S_0(1,2)$, $H_0^{f1}(1,3) = H_0^{f1}(3,1) = hf_5^i.S_0(1,3)$, $H_0^{f1}(2,3) = H_0^{f1}(3,2) = hf_6^i.S_0(2,3)$, $H_0^{f1}(4,4) = hf_7^i/G_{12}^0$, $H_0^{f1}(5,5) = hf_8^i/G_{23}^0$, $H_0^{f1}(6,6) = hf_9^i/G_{13}^0$.

Then, the matrix thermodynamic forces y_i^n, y_i^t are computed in function of positive strain as following: $y_i^n = \frac{1}{2} C_{ii}^0 \cdot \epsilon_i^+ \cdot \epsilon_i^+$ for $i \in 1, 2, 3$ and $y_1^t = (b_1 \cdot C_{66}^0 \cdot \epsilon_{13}^+ \cdot \epsilon_{13}^+ + b_2 \cdot C_{44}^0 \cdot \epsilon_{12}^+ \cdot \epsilon_{12}^+)$, $y_2^t = (b_3 \cdot C_{55}^0 \cdot \epsilon_{23}^+ \cdot \epsilon_{23}^+ + b_4 \cdot C_{44}^0 \cdot \epsilon_{12}^+ \cdot \epsilon_{12}^+)$, $y_3^t = (b_5 \cdot C_{66}^0 \cdot \epsilon_{13}^+ \cdot \epsilon_{13}^+ + b_6 \cdot C_{55}^0 \cdot \epsilon_{23}^+ \cdot \epsilon_{23}^+)$.

The fiber thermodynamic forces y_f^j are computed in function of strain as following: $y_f^1 = \frac{1}{2} C_0(1,1) \cdot \epsilon_1^+ \cdot \epsilon_1^+$, $y_f^2 = \frac{1}{2} C_0(1,1) \cdot \epsilon_1^- \cdot \epsilon_1^-$, $y_f^3 = \frac{1}{2} C_0(2,2) \cdot \epsilon_2^+ \cdot \epsilon_2^+$, $y_f^4 = \frac{1}{2} C_0(2,2) \cdot \epsilon_2^- \cdot \epsilon_2^-$.

The damage law is the following :

$$d_i = \max \left(g_i(y_i), d_i^0 \right)$$

where

$$g_i = d_{ci} \left(1 - e^{-\left(\frac{<\sqrt{y_i} - \sqrt{y_{0i}}>}{\sqrt{y_{ci}}} \right)^{p_i}} \right)$$

and if : $\Delta\epsilon_i^f \leq \bar{\epsilon}_i$,

$$\eta_i = 1$$

if : $-\Delta\epsilon_i^f \leq \bar{\epsilon}_i \leq \Delta\epsilon_i^f$,

$$\eta_i = \frac{1}{2} \left(1 - \cos \left(\frac{\Pi \bar{\epsilon}_i + \Delta\epsilon_i^f}{\Delta\epsilon_i^f} \right) \right)$$

if : $\bar{\epsilon}_i \leq -\Delta\epsilon_i^f$,

$$\eta_i = 0$$

where

$$\Delta\epsilon_i^f = (1 + a_{if} d_i^m) \Delta\epsilon_i^0$$

Syntax:

"ODMS"	"RO"	rho	"YG1"	yg1	"YG2"	yg2	"YG3"	yg3
	"NU12"	nu12	"NU13"	nu13	"NU23"	nu23	"G12"	g12
	"G13"	g13	"G23"	g23	"DCN1"	dcn1	"DCN2"	dcn2
	"DCN3"	dcn3	"DCT1"	dct1	"DCT2"	dct2	"DCT3"	dct3
	"YON1"	yon1	"YON2"	yon2	"YON3"	yon3	"YCN1"	ycn1
	"YCN2"	ycn2	"YCN3"	ycn3	"YOT1"	yot1	"YOT2"	yot2
	"YOT3"	yot3	"YCT1"	yct1	"YCT2"	yct2	"YCT3"	yct3
	"PN1"	pn1	"PN2"	pn2	"PN3"	pn3	"PT1"	pt1
	"PT2"	pt2	"PT3"	pt3	"HN1"	hn1	"HN2"	hn2
	"HN3"	hn3	"HP1"	hp1	"HP2"	hp2	"HP3"	hp3
	"HHP1"	hhp1	"HHP2"	hhp2	"HHP3"	hhp3	"XSI1"	xsi1
	"XSI2"	xsi2	"XSI3"	xsi3	"AIF1"	aif1	"AIF2"	aif2
	"AIF3"	aif3	"DE01"	deo1	"DE02"	deo2	"DE03"	deo3
	"B1"	b1	"B2"	b2	"B3"	b3	"B4"	b4
	"B5"	b5	"B6"	b6	"TAU1"	tau1	"TAU2"	tau2
	"TAU3"	tau3	"A1"	a1	"A2"	a2	"A3"	a3
	"GHOS"	ghos	"LATE"	late				
<	"DCF1"	dcf1	"DCF2"	dcf2	"DCF3"	dcf3	"DCF4"	dcf4
	"YF01"	yfo1	"YF02"	yfo2	"YF03"	yfo3	"YF04"	yfo4
	"PF1"	pf1	"PF2"	pf2	"PF3"	pf3	"PF4"	pf4
	"HF11"	hf11	"HF21"	hf21	"HF31"	hf31	"HF41"	hf41
	"HF51"	hf51	"HF61"	hf61	"HF71"	hf71	"HF81"	hf81
	"HF91"	hf91	"HF12"	hf22	"HF32"	hf32	"HF42"	hf42
	"HF52"	hf52	"HF62"	hf62	"HF72"	hf72	"HF82"	hf82
	"HF92"	hf92	"HF13"	hf23	"HF33"	hf33	"HF43"	hf43
	"HF53"	hf53	"HF63"	hf63	"HF73"	hf73	"HF83"	hf83

"HF93"	hf93	"HF14"	hf14	"HF34"	hf34	"HF44"	hf44
"HF54"	hf54	"HF64"	hf64	"HF74"	hf74	"HF84"	hf84
"HF94"	hf94	"HF24"	hf24	"AF1"	af1	"AF2"	af2
"AF3"	af3	"AF4"	af4	"TOF1"	tof1	"TOF2"	tof2
"TOF3"	tof3	"TOF4"	tof4	"RDC1"	rdc1	"RDC2"	rdc2
"RDC3"	rdc3	"RDC4"	rdc4	"EPS1"	eps1	"EPS2"	eps2
"EPS3"	eps3	"EPS4"	eps4	>			

/LECTURE/

rho

Density of the material.

yg1

Young's modulus along direction 1.

yg2

Young's modulus along direction 2.

yg3

Young's modulus along direction 3.

nu12

Poisson's ratio between directions 1 and 2.

nu13

Poisson's ratio between directions 1 and 3.

nu23

Poisson's ratio between directions 2 and 3.

g12

Shear modulus between directions 1 and 2.

g13

Shear modulus between directions 1 and 3.

g23

Shear modulus between directions 2 and 3.

dcn1

Normal critical damage defining the rupture criterion, in direction 1.

dcn2

Normal critical damage defining the rupture criterion, in direction 2.

dcn3

Normal critical damage defining the rupture criterion, in direction 3.

dct1

Tangential critical damage defining the rupture criterion, in direction 1.

dct2

Tangential critical damage defining the rupture criterion, in direction 2.

dct3

Tangential critical damage defining the rupture criterion, in direction 3.

yon1

Normal damage threshold in direction 1.

yon2

Normal damage threshold in direction 2.

yon3

Normal damage threshold in direction 3.

ycn1

Critical normal damage threshold in direction 1.

ycn2

Critical normal damage threshold in direction 2.

ycn3

Critical normal damage threshold in direction 3.

yot1

Tangential damage threshold in direction 1.

yot2

Tangential damage threshold in direction 2.

yot3

Tangential damage threshold in direction 3.

yct1

Critical tangential damage threshold in direction 1.

yct2

Critical tangential damage threshold in direction 2.

yct3

Critical tangential damage threshold in direction 3.

pn1

Parameter in normal damage law for direction 1.

pn2

Parameter in normal damage law for direction 2.

pn3

Parameter in normal damage law for direction 3.

pt1

Parameter in tangential damage law for direction 1.

pt2

Parameter in tangential damage law for direction 2.

pt3

Parameter in tangential damage law for direction 3.

hn1

Parameter in damaged constitutive law for normal direction 1.

hn2

Parameter in damaged constitutive law for normal direction 2.

hn3

Parameter in damaged constitutive law for normal direction 3.

hp1

Parameter in damaged constitutive law for tangential direction 1.

hp2

Parameter in damaged constitutive law for tangential direction 2.

hp3

Parameter in damaged constitutive law for tangential direction 3.

hhp1

Parameter in damaged constitutive law for tangential direction 1.

hhp2

Parameter in damaged constitutive law for tangential direction 2.

hhp3

Parameter in damaged constitutive law for tangential direction 3.

xsi1

Parameter to take into account residual deformation.

xsi2

Parameter to take into account residual deformation.

xsi3

Parameter to take into account residual deformation.

aif1

Parameter to evaluate limit deformation for activation index 1 calculus.

aif2

Parameter to evaluate limit deformation for activation index 2 calculus.

aif3

Parameter to evaluate limit deformation for activation index 3 calculus.

deo1

Parameter to evaluate limit deformation for activation index 1 calculus.

deo2

Parameter to evaluate limit deformation for activation index 2 calculus.

deo3

Parameter to evaluate limit deformation for activation index 3 calculus.

b1

Parameter for thermodynamic forces calculus.

b2

Parameter for thermodynamic forces calculus.

b3

Parameter for thermodynamic forces calculus.

b4

Parameter for thermodynamic forces calculus.

b5

Parameter for thermodynamic forces calculus.

b6

Parameter for thermodynamic forces calculus.

tau1

Characteristic time of the delayed damage model in direction 1. $(1/\tau)$ represents the maximum damage rate.

tau2

Characteristic time of the delayed damage model in direction 2. $(1/\tau)$ represents the maximum damage rate.

tau3

Characteristic time of the delayed damage model in direction 3. $(1/\tau)$ represents the maximum damage rate.

a1

Parameter of the delayed damage model, for the direction 1.

a2

Parameter of the delayed damage model, for the direction 2.

a3

Parameter of the delayed damage model, for the direction 3.

ghos

Parameter driving the post damage evolution: case 0. : nothing special happens (default case is 0.); case 1. : if a fiber damage is almost critical (ratio $rdc1$, $rdc2$, $rdc3$ $rdc4$) at a Gauss point of the element, then the element beco case 2. : if a strain reaches its given limit value at a Gauss point of the element, then the element becomes Ghost; case 3. : if case 1 or case 2 is reached, then the element becomes Ghost; case 4. : if there is an almost

critical damage at each Gauss points of the element, then the element becomes Ghost;
case 5. : if there is a critical strain at each Gauss points of the element, then the element
becomes Ghost; case 6. : if case 4 or case 5, then the element becomes Ghost;

late

Parameter driving the delay effect : case 0. : delay effect not active (default case is 0.)
; case 1. : delay effect is active in calculus, for the 3 matrix damages (and the 4 fiber
damages).

dcf1

Normal critical damage defining the rupture criterion of fiber in tension, in direction 1.

dcf2

Normal critical damage defining the rupture criterion of fiber in compression, in direction
1.

dcf3

Normal critical damage defining the rupture criterion of fiber in tension, in direction 2.

dcf4

Normal critical damage defining the rupture criterion of fiber in compression, in direction
2.

yfo1

Damage threshold in tension, in direction 1.

yfo2

Damage threshold in compression, in direction 1.

yfo3

Damage threshold in tension, in direction 2.

yfo4

Damage threshold in compression, in direction 2.

yfc1

Critical damage threshold in tension, in direction 1.

yfc2

Critical damage threshold in compression, in direction 1.

yfc3

Critical damage threshold in tension, in direction 2.

yfc4

Critical damage threshold in compression, in direction 2.

pf1

Parameter in tangential fiber damage law for direction 1, in tension.

pf2

Parameter in tangential fiber damage law for direction 1, in compression.

pf3

Parameter in tangential fiber damage law for direction 2, in tension.

pf4

Parameter in tangential fiber damage law for direction 2, in compression.

hf11

Fiber parameter in damaged constitutive law for tensile direction 1.

hf21

Fiber parameter in damaged constitutive law for tensile direction 1.

hf31

Fiber parameter in damaged constitutive law for tensile direction 1.

hf41

Fiber parameter in damaged constitutive law for tensile direction 1.

hf51

Fiber parameter in damaged constitutive law for tensile direction 1.

hf61

Fiber parameter in damaged constitutive law for tensile direction 1.

hf71

Fiber parameter in damaged constitutive law for tensile direction 1.

hf81

Fiber parameter in damaged constitutive law for tensile direction 1.

hf91

Fiber parameter in damaged constitutive law for tensile direction 1.

hf12

Fiber parameter in damaged constitutive law for compressive direction 1.

hf22

Fiber parameter in damaged constitutive law for compressive direction 1.

hf32

Fiber parameter in damaged constitutive law for compressive direction 1.

hf42

Fiber parameter in damaged constitutive law for compressive direction 1.

hf52

Fiber parameter in damaged constitutive law for compressive direction 1.

hf62

Fiber parameter in damaged constitutive law for compressive direction 1.

hf72

Fiber parameter in damaged constitutive law for compressive direction 1.

hf82

Fiber parameter in damaged constitutive law for compressive direction 1.

hf92

Fiber parameter in damaged constitutive law for compressive direction 1.

hf13

Fiber parameter in damaged constitutive law for tensile direction 2.

hf23

Fiber parameter in damaged constitutive law for tensile direction 2.

hf33

Fiber parameter in damaged constitutive law for tensile direction 2.

hf43

Fiber parameter in damaged constitutive law for tensile direction 2.

hf53

Fiber parameter in damaged constitutive law for tensile direction 2.

hf63

Fiber parameter in damaged constitutive law for tensile direction 2.

hf73

Fiber parameter in damaged constitutive law for tensile direction 2.

hf83

Fiber parameter in damaged constitutive law for tensile direction 2.

hf93

Fiber parameter in damaged constitutive law for tensile direction 2.

hf14

Fiber parameter in damaged constitutive law for compressive direction 2.

hf24

Fiber parameter in damaged constitutive law for compressive direction 2.

hf34

Fiber parameter in damaged constitutive law for compressive direction 2.

hf44

Fiber parameter in damaged constitutive law for compressive direction 2.

hf54

Fiber parameter in damaged constitutive law for compressive direction 2.

hf64

Fiber parameter in damaged constitutive law for compressive direction 2.

hf74

Fiber parameter in damaged constitutive law for compressive direction 2.

hf84

Fiber parameter in damaged constitutive law for compressive direction 2.

hf94

Fiber parameter in damaged constitutive law for compressive direction 2.

tof1

Characteristic time of the delayed damage model in tensile direction 1. ($1/\tau$) represents the maximum damage rate.

tof2

Characteristic time of the delayed damage model in compressive direction 1. ($1/\tau$) represents the maximum damage rate.

tof3

Characteristic time of the delayed damage model in tensile direction 2. ($1/\tau$) represents the maximum damage rate.

tof4

Characteristic time of the delayed damage model in compressive direction 2. ($1/\tau$) represents the maximum damage rate.

af1

Parameter of the delayed damage model, for the tensile direction 1.

af2

Parameter of the delayed damage model, for the compressive direction 1.

af3

Parameter of the delayed damage model, for the tensile direction 2.

af4

Parameter of the delayed damage model, for the compressive direction 2.

rdc1

Fiber damage (tensile direction 1) ratio for which element becomes ghost.

rdc2

Fiber damage (compressive direction 1) ratio for which element becomes ghost.

rdc3

Fiber damage (tensile direction 2) ratio for which element becomes ghost.

rdc4

Fiber damage (compressive direction 2) ratio for which element becomes ghost.

eps1

Strain limit (tensile direction 1) for which element becomes ghost.

eps2

Strain limit (tensile direction 2) for which element becomes ghost.

eps3

Strain limit (compressive direction 1) for which element becomes ghost.

eps4

Strain limit (compressive direction 2) for which element becomes ghost.

LECTURE

List of the elements concerned.

Comments:

This option may be repeated as many times as necessary.

The associated coordinate system is defined by the option "MORTHO" (see page C1.96) for the continuum elements in 3D and in plane strain.

The associated coordinate system is defined by the directive "COMPLEMENT" (pages C1.95 and C1.96):

- "COMPLEMENT" "CORTHO" for the shells;
- "COMPLEMENT" "MORTHO" for the continuum elements 3D and 2D plane strain and axisymmetric.

Verify that this material is available for your elements : pr6, cub8, tetr, prism.

Outputs:

The different components of the ECR table are as follows, for the continuum elements:

ECR(1): Pression.

ECR(2): Vom mises criterion.

ECR(3) : d1 - damage in direction 1.

ECR(4) : d2 - damage in direction 2.

ECR(5): d3 - damage in direction 3.

ECR(6): eta1 - activation damage index in direction 1.

ECR(7): eta2 - activation damage index in direction 2.

ECR(8): eta3 - activation damage index in direction 3.

- ECR(9) : epsilon s 11 - residual deformation 11.
ECR(10): epsilon s 22 - residual deformation 22.
ECR(11): epsilon s 33 - residual deformation 33.
ECR(12): epsilon s 12 - residual deformation 12.
ECR(13): epsilon s 23 - residual deformation 23.
ECR(14): epsilon s 13 - residual deformation 13.
ECR(15): epsilon r 11 - residual deformation 11.
ECR(16): epsilon r 22 - residual deformation 22.
ECR(17): epsilon r 33 - residual deformation 33.
ECR(18): epsilon r 12 - residual deformation 12.
ECR(19): epsilon r 23 - residual deformation 23.
ECR(20): epsilon r 13 - residual deformation 13.
ECR(21) : df1t - fiber tensile damage in direction 1.
ECR(22) : df1c - fiber compressive damage in direction 1.
ECR(23): df2t - fiber tensile damage in direction 2.
ECR(24): df2c - fiber compressive damage in direction 2.
ECR(25): T - time.

9.6.21 WOOD**Object:**

This directive allows to define the BOIS (wood) material, that is used for example for packaging and transportation as a shock absorber. Only the compressive behaviour of the material is considered, while the material response in traction is approximated as perfectly linear (or linear perfectly plastic) because of the lack of experimental data.

Syntax:

```
"BOIS"  "R0"  rho    "YG1"  yg1    "YG2"  yg2  "YG3"  yg3
        "NU12" nu12  "NU13" nu13  "NU23" nu23
        "G12"  g12   "G13"  g13   "G23"  g23
        "SY_1" sy1  "SY_2" sy2   "SY_3" sy3
        "ED_1" ed1  "ED_2" ed2   "ED_3" ed3
        < "TR_1" tr1 > < "TR_2" tr2 > < "TR_3" tr3 >
        < "COE1" coe1 > < "COE2" coe2 > < "COE3" coe3 >
        < "EDCV" edcv > < "DIRF" idir >
        /LECTURE/
```

rho

Density of the material.

yg1

Young's modulus along direction 1.

yg2

Young's modulus along direction 2.

yg3

Young's modulus along direction 3.

nu12

Poisson's ratio between directions 1 and 2.

nu13

Poisson's ratio between directions 1 and 3.

nu23

Poisson's ratio between directions 2 and 3.

g12

Shear modulus between directions 1 and 2.

g13

Shear modulus between directions 1 and 3.

g23

Shear modulus between directions 2 and 3.

sy1

Elastic limit along the first orthotropy direction.

sy2

Elastic limit along the second orthotropy direction.

sy3

Elastic limit along the third orthotropy direction.

ed1

Limit deformation before reconsolidation along the first orthotropy direction.

ed2

Limit deformation before reconsolidation along the second orthotropy direction.

ed3

Limit deformation before reconsolidation along the third orthotropy direction.

tr1

Limit stress in traction along the first orthotropy direction. If omitted, an elastic behaviour is assumed along this direction.

tr2

Limit stress in traction along the second orthotropy direction. If omitted, an elastic behaviour is assumed along this direction.

tr3

Limit stress in traction along the third orthotropy direction. If omitted, an elastic behaviour is assumed along this direction.

coe1

Scale factor between initial Young's modulus and Young's modulus after reconsolidation in direction 1.

coe1

Scale factor between initial Young's modulus and Young's modulus after reconsolidation in direction 2.

coe1

Scale factor between initial Young's modulus and Young's modulus after reconsolidation in direction 3.

edcv

Threshold deformation in all directions starting convergence of the material towards a consolidated linear elastic material (see comment below).

idir

Spatial direction of wood fibers, if not the first orthotropy direction (see comment below).

LECTURE

List of the elements concerned.

Comments:

This material model is taken from the thesis of P. François: "Plasticité du bois en compression multi-axiale : Application à l'absorption d'énergie mécanique". Doctoral Thesis of the Bordeaux I University (October 1992).

The associated coordinate system is defined by the option **MORTH0** (see page C.96) for the continuum elements in 2D and 3D.

Verify that this material is available for your elements, by means of the tables of page C.100.

Compression instability may be encountered after reconsolidation, since the material becomes very stiff in the consolidated direction and is still very soft in the other directions. To overcome this problem, it may be considered that the orthotropy of the material is lost when reconsolidation is achieved, since the microstructure has been completely crushed and all the voids filled. This assumption is taken from the observations made in the PhD Thesis of C. Adalian: "The behaviour of wood under multiaxial dynamic compression - Use for the modelling of crashes of containers", PhD Thesis of Bordeaux I University (1998)

Such a behaviour is activated using **EDCV** keyword. The floating value associated to this keyword defines the level of deformation above which the process of convergence towards an

elastic isotropic material is started. It should be less than the reconsolidation limit given in each orthotropy direction.

The convergence process is such that material characteristics of the element converge continuously towards the isotropic consolidated values, whatever the first direction is in which reconsolidation is achieved.

Isotropic elastic parameters of the consolidated materials are deduced from elastic parameters given in the wood fibers direction. If this direction is not the first orthotropy direction, it can be specified using "DIRF" keyword. Isotropic consolidated parameters are then obtained by the formulae (assuming the fibers direction is direction 1):

$$E_c = E_1.coe1$$

$$\nu_c = \frac{\nu_{12} + \nu_{13}}{2}$$

$$G_c = \frac{E_c}{2.(1 + \nu)}$$

Outputs:

The different components of the ECR table are as follows, for the continuum elements:

- ECR(1) : Pressure.
- ECR(2) : Von mises criterion.
- ECR(3) : Plastic strain along the first orthotropy direction.
- ECR(4) : Plastic strain along the second orthotropy direction.
- ECR(5) : Plastic strain along the third orthotropy direction.
- ECR(6) : Principal stress along the first orthotropy direction.
- ECR(7) : Principal stress along the second orthotropy direction.
- ECR(8) : Principal stress along the third orthotropy direction.
- ECR(9) : Total strain along the first orthotropy direction.
- ECR(10): Total strain along the second orthotropy direction.
- ECR(11): Total strain along the third orthotropy direction.
- ECR(12): Flag for isotropic elastic converged material (0: not fully converged, 1: fully converged)
- ECR(13): Convergence level in the first orthotropy direction.
- ECR(14): Convergence level in the second orthotropy direction.
- ECR(15): Convergence level in the third orthotropy direction.

9.6.22 MASS**Object:**

This directive enables the masses of the material points "PMAT" to be entered.

Syntax:

```
"MASSE"  xm  /LECTURE/
```

xm

Mass.

LECTURE

Numbers of the elements concerned.

Comments:

If the node corresponding to the material point belongs to another element, its mass may be zero. This is very useful in the case of unilateral junctions.

In this way, added masses may be entered too.

In axisymmetric, the real masses must be divided by 2π .

Outputs:

The different components of the ECR table are as follows:

ECR(1): integrated impulse from the origin

ECR(2): sum of the instantaneous reaction forces.

9.6.23 PHANTOM

Object:

This directive enables the elimination of elements in a mesh.

Syntax:

```
"FANT"  rho  /LECTURE/
```

rho

Density.

LECTURE

List of the elements concerned.

Comments:

The EUROPLEXUS program considers that all these elements do not exist.

However, the nodes are always present; as their masses may not be zero, it is necessary to give (very low) densities to "FANT" elements.

9.6.24 FREE (USER'S MATERIAL)**Object:**

The directive enables the user to enter his own constitutive laws.

Syntax:

```

          $ "STRU" ... $
          $ "FLUI" ... $
"LIBR"  $ "PMAT" ... $   < "PARA"  a b c ... > /LECTURE/
          $ "MECA" ... $
          $ "CLIM" ... $

```

"STRU" ...

Indicates that the free material is of type "STRUCTURE".

"FLUI" ...

Indicates that the free material is of type "FLUIDE".

"PMAT" ...

Indicates that the free material is of type "POINT MATERIEL".

"MECA" ...

Indicates that the free material is of type "MECANISME".

"CLIM" ...

Indicates that the free material is of type "CONDITION AUX LIMITES".

"PARA" ...

Key-word used to introduce a series of additional parameters.

LECTURE

List of the elements concerned.

Comments:

The distinction between structure and fluid is due to the processing differences in A.L.E. In fact, there are transport terms for the fluid, whereas the structure is always Lagrangian.

Similarly, the cases of material points, mechanisms and boundary conditions are so peculiar that a dedicated syntax is provided.

These directives are described in detail on the following pages.

Remarks:

In the examples proposed in the following pages, there are some calls to utility routines that are available within EUROPLEXUS:

1. **ERRMSS**(STRING1,STRING2) :

Subroutine named 'STRING1' generates the error message 'STRING2', increments the error counter and triggers the calculation stop. 'STRING1' and 'STRING2' are two character strings.

2. **TILT** :

This subroutine without arguments triggers the calculation stop at the end of the current time step, and passes control to the next input directive in the input data set, which is normally either "SUIT" or "FIN", following the directive "CALCUL".

3. **QUIDNE**(LOOP,NUM,LON,VAL) :

This subroutine extracts values relative to a node or to an element (of index "NUM"), and places them in the array "VAL".

If the quantity to be extracted is a vector (e.g. a velocity), the length of the extracted vector is in "LON", and the array "VAL" must be dimensioned sufficiently ($\text{DIM}(\text{VAL}) \geq \text{LON}$).

Argument "LOOP" allows to select the values to be extracted.

For a node :

LOOP = 0 : Coordinates of node "NUM",

LOOP = 1 : Displacements,

LOOP = 2 : Velocities,

LOOP = 5 : Nodal masses.

For an element :

LOOP = 21 : Stresses in element "NUM",

LOOP = 22 : Total deformations,

LOOP = 23 : Internal variables (ECR),

LOOP = 24 : Internal energy.

9.6.25 FREE MATERIAL OF TYPE STRUCTURE

Object:

This directive introduces a user-defined constitutive behaviour of structural type (“STRUCTURE”).

Syntax:

```
"LIBR"  "STRU" num  "RO" rho  "YOUN" young  "NU" nu  ...  
...    < "PARA" /LECPARA/ >  /LECTURE/
```

"STRU" num

Indicates that the free material of type “STRUCTURE” has the user-specified index **num**.

"RO" rho

Density. This value is mandatory in order to compute the element mass.

"YOUN" young

Young’s modulus.

"NU" nu

Poisson’s ratio.

"PARA" ...

Key-word used to introduce a series of additional parameters.

LECTURE

List of the elements concerned.

Comments:

The number **num** enables several materials chosen by the user to be recognized. The three parameters **rho**, **young** and **nu** are compulsory.

The user specifies his material’s parameters after the keyword “PARAM”. When EUROPLEXUS finds the keyword “LECTURE”, it considers that the list of parameters is terminated, whatever the number of values that have been read.

However, the total number of parameters for this material may not exceed 100, including the three mandatory values (`rho`, `young` and `nu`).

If there are no additional parameters besides the three mandatory ones, the keyword “PARAM” may be omitted.

The parameters are used within the subroutine “MSLIBR” that must be written by the user, compiled and linked with the code libraries to produce a special code executable before launching the run.

The elements that accept the free material of type “STRUCTURE” are the following:

2D : TRIA, CAR1, CAR4.

3D : CUBE, CUB6, CUB8, PRIS, PR6, TETR.

Be careful to respect the conventions chosen to rank the tensor components according to the 2-D plane, 2-D axisymmetric or 3-D cases. See page G.20 for further explanations.

The user can store for each element (and each integration point), the values he wants (up to 10) in the ECR table. For homogeneity with the other materials, the following data will be stored in the first two locations of the ECR table :

ECR(1) = Pressure

ECR(2) = Von Mises

The eight other locations are free.

Examples:

The following example, taken from the standard benchmark “bm_str_2d.lib”, concerns the traction of an axisymmetric cylinder.

The material data are as follows:

```
MATERIAUX  LIBRE  STRUCTURE 901
              RO 7800. YOUNG 210E9 NU 0.    TOUS
```

In this particular case there is just one material, identified by the user-supplied index 901. There are no additional parameters besides the three mandatory ones, and the Poisson coefficient is zero. All the elements in the mesh possess this material (keyword “TOUS”).

Programming example relative to MSLIBR:

```

SUBROUTINE MSLIBR(NLGEOM,NUM,TT,XMAT,SIG,DEPS,EDOT,RO,PI,
&                CSON,ECR,X,IEL,IDIM,NBN)
*
* -----
*
*      materiau libre (structure)                m.lepareux 11.86
*
* -----
*
* entree :
*      num      = numero de repereage du materiau utilisateur
*      tt       = temps du calcul
*      xmat(1)  = masse volumique initiale
*      xmat(2)  = young
*      xmat(3)  = poisson
*      xmat(4: ) = autres parametres du materiau
*      sig      = contraintes au debut du pas
*      deps     = accroissement des deformations
*      edot     = vitesse de deformation
*      ro       = masse volumique courante
*      x        = coordonnees des nbn noeuds
*      iel      = numero de l'element
*      idim     = dimension (2=2d ou axis , 3=3d)
*
* sortie :
*      nlgeom   = 0 (dans inico1.ff matal(13)=1)
*      pi       = contraintes a la fin du pas (a calculer)
*      cson     = vitesse du son (a calculer pour la stabilite)
*      ecr(1)   = pression (a calculer)
*      ecr(2)   = critere de von mises (a calculer)
*      ecr(3:7) = emplacements libres
*
* attention ! le materiau 901 est utilise par le benchmark
*              "bm_str_2d_libr.epx"
*
* IMPLICIT NONE
*
* --- variables globales :
* INTEGER, INTENT(IN)  :: NUM,NBN,IDIM
* INTEGER, INTENT(OUT) :: NLGEOM
* REAL(8), INTENT(IN)  :: TT,XMAT(*),SIG(*),DEPS(*),EDOT(*),
&                      RO,X(IDIM,NBN)
* REAL(8), INTENT(OUT) :: CSON,PI(*)
* REAL(8), INTENT(INOUT) :: ECR(*)
*
* --- variables locales :
* REAL(8) :: AMU,ALAMB,AUX,YOUNG,POISS,C1
*
*
* ----- on active les nonlinearites geometriques : nlgeom=0
* NLGEOM = 0
*
* SELECT CASE (NUM)
* CASE(901)
* ----- l'exemple qui suit concerne un mat. elastique (en 2d axis.)
* YOUNG=XMAT(2)
* POISS=XMAT(3)
* --- coefficients de lame :
* C1=YOUNG/(1.+POISS)
* AMU=0.5D0*C1
* ALAMB=C1*POISS/(1-2*POISS)
* AUX=ALAMB*(DEPS(1)+DEPS(2)+DEPS(4))
* --- nouveau tenseur des contraintes :
* PI(1) = SIG(1) + AUX + C1*DEPS(1)
* PI(2) = SIG(2) + AUX + C1*DEPS(2)
* PI(3) = SIG(3) + AMU*DEPS(3)
* PI(4) = SIG(4) + AUX + C1*DEPS(4)
* --- pression :
* ECR(1)=(PI(1)+PI(2)+PI(4))/3.D0
* --- critere de von mises :
* ECR(2)=SQRT(PI(1)*(PI(1)-PI(2))+PI(2)*(PI(2)-PI(4))
&          +PI(4)*(PI(4)-PI(1))+3*PI(3)*PI(3))
* --- vitesse du son (stabilite) :
* CSON=DSQRT(YOUNG/RO)
*
* CASE DEFAULT
* CALL ERRMSS('MSLIBR','ROUTINE UTILISATEUR NON PROGRAMMEE')
* STOP 'MSLIBR' ABSENT'
* END SELECT
*
* END SUBROUTINE MSLIBR

```

9.6.26 FREE MATERIAL OF TYPE FLUID

Object:

This directive introduces a user-defined constitutive behaviour of fluid type (“FLUIDE”).

Syntax:

```
"LIBR" "FLUI" num  "RO" rho  "PINI" pini  "PREF" pref  "EINT" ei ...  
      ...  < "PARA"  /LECPARA/ >  /LECTURE/
```

"FLUI" num

Indicates that the free material of type “FLUIDE” has the user-specified index **num**.

"RO" rho

Density. This value is mandatory in order to compute the element mass.

"PINI" pini

Initial pressure.

"PREF" pref

Reference pressure.

"EINT" ei

Initial internal energy per unit mass.

"PARA" ...

Key-word used to introduce a series of additional parameters.

LECTURE

List of the elements concerned.

Comments:

The number `num` enables several materials chosen by the user to be recognized. The four parameters `rho`, `pini`, `pref` and `ei` are compulsory.

The user specifies his material's parameters after the keyword "PARAM". When EUROPLEXUS finds the keyword "LECTURE", it considers that the list of parameters is terminated, whatever the number of values that have been read.

However, the total number of parameters for this material may not exceed 100, including the four mandatory values (`rho`, `pini`, `pref` and `ei`).

If there are no additional parameters besides the four mandatory ones, the keyword "PARAM" may be omitted.

The parameters are used within the subroutine "MFLIBR" that must be written by the user, compiled and linked with the code libraries to produce a special code executable before launching the run.

The elements that accept the free material of type "FLUIDE" are the following:

1D : TUBE, TUYA, CAVI.

2D : TRIA, CAR1.

3D : CUBE, PRIS, TETR.

Be careful to respect the conventions chosen to rank the tensor components according to the 2-D plane, 2-D axisymmetric or 3-D cases. See page G.20 for further explanations.

The user can store for each element (and each integration point), the values he wants (up to 10) in the ECR table. For homogeneity with the other materials, the following data will be stored in the first two locations of the ECR table :

ECR(1) = Pressure

ECR(2) = Density.

The eight other locations are free.

Examples:

The following example, taken from the standard benchmark "bm_flu_1d_libr", concerns a shock tube with a perfect gas.

The material data are as follows:

MATERIAUX

```
LIBRE FLUIDE 903 RO 13. PINI 1e6 PREF 1e5 EINT 192.3077e3
              PARAM 1.4 640.          LECT tub_1  TERM
```

```
LIBRE FLUIDE 903 RO 1.3 PINI 1e5 PREF 1e5 EINT 192.3077e3
              PARAM 1.4 640.          LECT tub_2  TERM
```

In this case there are two materials, whose user index is the same (903), but which have different initial conditions.

Note the value of the initial internal energy, which is mandatory because the behaviour of the perfect gas depends both on the density ρ and on the specific internal energy e :

$$P = (\gamma - 1)\rho e$$

These two variables ρ and e change during the transient, as a function of mass and energy transfer among the neighbouring elements. EUROPLEXUS computes these transfers automatically.

There are two additional parameters besides the four mandatory ones. These are respectively the ratio of specific heats (γ), and the specific heat at constant volume (C_v), which allow to obtain the temperature (θ).

$$\theta = \frac{e}{C_v}$$

Programming example relative to MFLIBR:

```
SUBROUTINE MFLIBR(NUM,TT,XMAT,SIG,DEPS,EDOT,RO,EINT,DSIG,CSO,
&               ECR,X,IEL,IDIM,NBN)
*
* -----
*
*      materiau libre (fluide)          m.lepareux 11.86
*
* -----
*
*  entree :
*      num      = numero de repereage du materiau utilisateur
*      tt       = temps du calcul
*      sig      = contraintes au debut du pas
*      deps     = accroissement des deformations
*      edot     = vitesse de deformation
*      ro       = masse volumique courante
*      eint     = energie interne massique courante
*      x        = coordonnees des nbn noeuds
*      iel      = numero de l'element
*      idim     = dimension (2=2d ou axis , 3=3d)
*      nbn      = nombre de noeuds de l'element
*      xmat(1)  = masse volumique initiale
*      xmat(2)  = pression initiale
*      xmat(3)  = pression de reference
*      xmat(4)  = energie interne massique initiale
*      xmat(5:) = parametres de l'utilisateur
*
*  sortie :
*      dsig     = increments de contraintes
```

```

*          cson      = vitesse du son (pour la stabilite)
*          ecr(1)    = pression
*          ecr(2)    = masse volumique
*          ecr(3:7)  = emplacements libres
*
* attention ! le materiau 903 est utilise par le benchmark
*              "bm_flu_1d_libr.epx"
*
      IMPLICIT NONE
*
*---  variables globales :
      INTEGER, INTENT(IN) :: NUM,IEL,NBN,IDIM
      REAL(8), INTENT(IN) :: TT,XMAT(*),SIG(*),DEPS(*),EDOT(*),RO,EINT,
&
      REAL(8), INTENT(OUT) :: DSIG(*),CSON
      REAL(8), INTENT(INOUT) :: ECR(*)
*
*---  variables locales :
      REAL(8) :: ROZR,PZER,PREF,PABS,PR,CV,GAMA,TRE
*
*
      SELECT CASE (NUM)
      CASE(903)
*--      cas d'un gaz parfait :
          PREF = XMAT(3)          ! PRESSION DE REFERENCE
          GAMA = XMAT(5)          ! GAMMA DU GAZ
          CV = XMAT(6)           ! CHALEUR MASSIQUE
          PABS = RO * (GAMA -1D0) * EINT ! PRESSION ABSOLUE
          CSON = SQRT(GAMA*PABS/RO) ! VITESSE DU SON
          PR = PABS - PREF        ! PRESSION RELATIVE
          TRE = EINT/CV - 273.15D0 ! TEMPERATURE
*--      increments de contrainte (1d)
          DSIG(1) = -PR -SIG(1)
*--      remplissage des "ecr" :
          ECR(1) = PABS
          ECR(2) = RO
          ECR(3) = CSON
          ECR(4) = TRE
*
      CASE DEFAULT
*----      routine utilisateur crire
          CALL ERRMSS('MFLIBR','ROUTINE UTILISATEUR NON PROGRAMMEE')
          STOP ' "MFLIBR" ABSENT'
      END SELECT
*
      END

```

9.6.27 FREE MATERIAL OF TYPE MATERIAL POINT

Object:

This directive introduces a user-defined constitutive behaviour of type material point (“POINT MATERIEL”).

Syntax:

```
"LIBR" "PMAT" num "MASS" m < "PARA" a b c ... > /LECTURE/
```

"PMAT" num

Indicates that the free material of type “POINT MATERIEL” has the user-specified index num.

"MASS" m

Mass of the material point element.

"PARA" ...

Key-word used to introduce a series of additional parameters.

LECTURE

List of the elements concerned.

Comments:

The number num enables several materials chosen by the user to be recognized. The single parameter “MASS” is mandatory.

The user specifies his material’s parameters after the keyword “PARAM”. When EUROPLEXUS finds the keyword “LECTURE”, it considers that the list of parameters is terminated, whatever the number of values that have been read.

However, the total number of parameters for this material may not exceed 100, including the single mandatory value (m).

If there are no additional parameters besides the mandatory one, the keyword “PARAM” may be omitted.

The parameters are used within the subroutine “MPLIBR” that must be written by the user, compiled and linked with the code libraries to produce a special code executable before launching the run.

The only element that accepts the free material of type “POINT MATERIEL” is “PMAT”, that is always 3-D.

Be careful to respect the conventions chosen to rank the tensor components according to the 3-D cases. See page G.20 for further explanations.

The user can store for each element (and each integration point), the values he wants (up to 10) in the ECR table. The ten locations are free.

Examples:

The following example, taken from the standard benchmark “bm_str_terlun”, treats the case of two pointwise masses that attract each other according to the universal gravitation law.

The material data are as follows:

MATERIAUX

```
!--
LIBRE  PMAT 101  MASS 1.00    PARAM 1.14e4  1  pt_lune
          LECT  terre TERM
!--
LIBRE  PMAT 101  MASS 0.0123  PARAM 1.14e4  1  pt_terre
          LECT  lune  TERM
!
!          param(1) = constante de gravitation
!          param(2) = nbr de noeuds attires par cet element
!          param(3) = liste des noeuds (ici un seul)
```

In this case there are two materials, whose user index is the same (101), but which have different parameters. Lines starting by a “!” are comments.

The used units are adapted to the treated problem. For masses, the reference is the earth mass, for lengths the earth radius and for times the day.

There are three additional parameters besides the mandatory one, that are respectively the gravity constant, the number of nodes subjected to gravity (here just one) and the index of the concerned node.

Programming example relative to MPLIBR:

```

SUBROUTINE MPLIBR(NUM,T,PARAM,AMAS,ECR,X,U,F,V,DTSTAB)
*
* -----
*
*      matieriau libre pour les points mat.      m.lepareux 08-95
*
* -----
*
* entree :
*      num      : numero de reperage pour l'utilisateur
*      t        : temps
*      param    : tableau des parametres du mat. libre
*      amas     : masse de l'element
*      x        : coordonnees
*      u        : déplacements
*      v        : vitesses
*
* sortie :
*      ecr      : tableau des parametres du matieriau
*      f        : forces internes
*
* attention ! le matieriau 101 est utilise par le benchmark
*              "bm_str_terlun.epx"
*
*      IMPLICIT NONE
*
* --- variables globales :
*      INTEGER, INTENT(IN) :: NUM
*      REAL(8), INTENT(IN) :: T,PARAM(*),AMAS,X(3),U(3),V(3)
*      REAL(8), INTENT(OUT) :: F(3)
*      REAL(8), INTENT(INOUT) :: ECR(*),DTSTAB
*
* --- variables locales :
*      INTEGER LON,NOD,NBR,II
*      REAL(8) GG,FF,R(3),R2,R3,AMB,XB(3),AUX,EPSI,BM(3)
*      REAL(8) CX,SS,ROL,V2,VV,COEF
*
*
*      SELECT CASE (NUM)
*      CASE (101)
*
*      !--- loi de gravitation universelle :
*      !-- (l'element affecte de ce matieriau agit sur les noeuds designes)
*      !--      param(2+1:2+nbr) ! numeros des noeuds designes
*      GG = PARAM(1) ! CONSTANCE G DE GRAVITATION
*      NBR = PARAM(2) ! NOMBRE DE NOEUDS SOUMIS A L'ATTRACTION
*      DTSTAB = 1000.
*      EPSI = 0.01
*
*      F(:) = ODO
*      DO II=1,NBR
*      NOD = PARAM( 2 + II )
*      CALL QUIDNE( 0,NOD,LON,XB ) ! XB = POSITION DU NOEUD CIBLE
*      CALL QUIDNE( 5,NOD,LON,BM )
*      AMB = BM(1) ! MASSE DU NOEUD CIBLE
*
*      R(:) = X(:) - XB(:)
*      R2 = R(1) * R(1) + R(2) * R(2) + R(3) * R(3)
*      R3 = R2 * SQRT(R2)
*      FF = GG * AMAS * AMB / R3
*      AUX = SQRT( R3 / ( GG * AMB ) )
*      DTSTAB = MIN( DTSTAB , AUX * EPSI )
*
*      F(:) = F(:) + R(:) * FF
*      END DO
*      CASE DEFAULT
*      CALL ERRMSS('MPLIBR','NUMERO DE MATERIAU LIBRE INCONNU')
*      STOP ' MPLIBR'
*
*      !
*      END SELECT
*
*      END

```

9.6.28 FREE MATERIAL OF TYPE MECHANISM

Object:

This directive introduces a user-defined constitutive behaviour of mechanism type (“MECANISME”).

Syntax:

```
"LIBR" "MECA" num    < "PARA"  a b c ... > /LECTURE/
```

"MECA" num

Indicates that the free material of type “MECANISME” has the user-specified index num.

"PARA" . . .

Key-word used to introduce a series of additional parameters.

LECTURE

List of the elements concerned.

Comments:

The number num enables several materials chosen by the user to be recognized. There are no mandatory parameters.

The user specifies his material’s parameters after the keyword “PARAM”. When EUROPLEXUS finds the keyword “LECTURE”, it considers that the list of parameters is terminated, whatever the number of values that have been read.

However, the total number of parameters for this material may not exceed 100.

If there are no parameters the keyword “PARAM” may be omitted.

The parameters are used within the subroutine “MMLIBR” that must be written by the user, compiled and linked with the code libraries to produce a special code executable before launching the run.

The only element that accepts the free material of type “MECANISME” is ”MECA”, that is an element with two nodes and 6 degrees of freedom per node.

The main interest of this free material is to allow the user to specify arbitrary relations (in matricial form) between the displacements of the two nodes and the applied forces. For example, it is possible to enter a symmetric stiffness matrix in order to model a complicated support (78 values). However, it must be noted that the relations must be specified in (or converted to) the global reference frame, and that they stay constant during the whole transient calculation.

The user can store for each element (and each integration point), the values he wants (up to 10) in the ECR table. The ten locations are free.

Examples:

The following example, taken from the standard benchmark "bm_str_meca_lbr", treats the very simple case of springs in translation and rotation.

The 78 values are specified in the global reference frame, and the principal axes are parallel to the global ones. The translational stiffness is $K_T = 1E3$, and the rotational one is $K_R = 4E6$. The free material has the user-specified index 905.

The material data are as follows:

MATERIAUX

```
LIBRE  MECA  905  PARAM
      1e3
      0.0  1e3
      0.0  0.0  1e3
      0.0  0.0  0.0  4e6
      0.0  0.0  0.0  0.0  4e6
      0.0  0.0  0.0  0.0  0.0  4e6
    -1e3  0.0  0.0  0.0  0.0  0.0  1e3
      0.0 -1e3  0.0  0.0  0.0  0.0  0.0  1e3
      0.0  0.0 -1e3  0.0  0.0  0.0  0.0  0.0  1e3
      0.0  0.0  0.0 -4e6  0.0  0.0  0.0  0.0  0.0  4e6
      0.0  0.0  0.0  0.0 -4e6  0.0  0.0  0.0  0.0  0.0  4e6
      0.0  0.0  0.0  0.0  0.0 -4e6  0.0  0.0  0.0  0.0  0.0  4e6
      LECT  L_meca  TERM
```

Programming example relative to MMLIBR:

```
SUBROUTINE MMLIBR(NUM,TT,NBPAR,XMAT,X,DU,F,XMA,V,DFX,
&
      ECR,SIG,DEPS,PI,DWINT,IEL,DTSTAB)
*
* -----
*
```

```

*          materiau libre mecanisme          m.lepareux 12-00
*
* -----
*
* entree :
*   num      = numero de repereage du materiau utilisateur
*   tt       = temps du calcul
*   nbpar    = nombre de parametres utilisateur
*   xmat(1:) = parametres du materiau
*   x(1:3,1:2) = coordonnees des 2 noeuds
*   du(1:12) = déplacements des 2 noeuds
*   xma(1:12) = masses des 2 noeuds
*   v(1:12) = vitesses des 2 noeuds
*   dfx(1:12) = déplacements cumules des 2 noeuds
*   sig(1:6) = forces internes au debut du pas
*              en fait : sig = 0.5 * (f2 - f1)
*   deps(1:6) = déplacement relatif : deps = (u2 - u1)
*   iel      = numero de l'element
*
* sorties :
*   f(1:12) = forces internes appliquees aux 2 noeuds
*   ecr(:)  = variables internes (emplacements libres)
*   pi(1:6) = forces internes a la fin du pas
*   dwint   = travail des forces internes
*   dtstab  = pas de stabilite
*
* attention ! le materiau 905 est utilise par le benchmark
*              "bm_str_meca_lbr.epx"
*
* IMPLICIT NONE
*
* --- variables globales :
* INTEGER, INTENT(IN)      :: NUM,IEL,NBPAR
* REAL(8), INTENT(IN)     :: TT,XMAT(*),X(3,2),DU(12),XMA(12),
* &                        V(12),DFX(12),SIG(6),DEPS(6)
* REAL(8), INTENT(OUT)    :: F(12),PI(6),DWINT
* REAL(8), INTENT(INOUT)  :: ECR(*),DTSTAB
*
* --- variables locales :
* INTEGER :: K,I,II
* REAL(8) :: R_K(12,12),RT(3),RR(3),DT(12)
*
*
* SELECT CASE (NUM)
* CASE(905)
*   IF(NBPAR /= 78) THEN
*     CALL ERMSS('MMLIBR','IL FAUT 78 VALEURS')
*     STOP ' MMLIBR'
*   ENDIF
*
* -- construction de la matrice de raideur :
* II = 0
* DO K=1,12
*   DO I=1,K
*     II = II+1
*     R_K(K,I) = XMAT(II)
*     R_K(I,K) = R_K(K,I)
*   END DO
* END DO
*
*
* -- calcul direct des forces internes :
* F(:) = ODO
* DO K=1,12
*   DO I=1,12
*     F(K) = F(K) + R_K(K,I)*DFX(I)
*   END DO
* END DO
*
*
* -- nouvelles forces internes (pour calculer wint)
* DO K=1,6
*   PI(K) = 0.5DO * (F(K+6) - F(K))
* END DO
*
*
* -- travail des forces internes (pendant le pas de temps)
* DWINT = ODO
* DO K=1,6
*   DWINT = DWINT + 0.5DO*(SIG(K)+PI(K))*DEPS(K)
* END DO
*
*
* -- ECR : variables internes (allongement)
* DO K=1,3
*   RT(K) = DFX(K+6) - DFX(K)
*   RR(K) = DFX(K+9) - DFX(K+3)
* END DO
* ECR(1) = SQRT(RT(1)*RT(1) + RT(2)*RT(2) + RT(3)*RT(3))
* ECR(2) = SQRT(RR(1)*RR(1) + RR(2)*RR(2) + RR(3)*RR(3))
*
*
* -- calcul du pas de stabilite :
* DTSTAB = 1000DO

```



```
      DO K=1,12
        DT(K) = SQRT(XMA(K)/R_K(K,K))
        DTSTAB = MIN(DTSTAB,2*DT(K))
      END DO
*
      CASE DEFAULT
        CALL ERRMSS('MMLIBR','ROUTINE UTILISATEUR NON PROGRAMMEE')
        STOP ' "MMLIBR" ABSENT'
      END SELECT
*
      END
```

9.6.29 FREE MATERIAL OF TYPE BOUNDARY CONDITIONS

Object:

This directive introduces a user-defined constitutive behaviour of the boundary condition type (“CONDITION AUX LIMITES”).

Syntax:

```
"LIBR" "CLIM" num "PREF" pref < "PARA" a b c ... > /LECTURE/
```

"CLIM" num

Indicates that the free material of type “CONDITION AUX LIMITES” has the user-specified index **num**.

"PREF" pref

Reference pressure.

"PARA" ...

Key-word used to introduce a series of additional parameters.

LECTURE

List of the elements concerned.

Comments:

The number **num** enables several materials chosen by the user to be recognized. The only mandatory parameter is **pref**.

The user specifies his material’s parameters after the keyword “PARAM”. When EUROPLEXUS finds the keyword “LECTURE”, it considers that the list of parameters is terminated, whatever the number of values that have been read.

However, the total number of parameters for this material may not exceed 100, including the single mandatory value **pref**.

If there are no additional parameters besides the mandatory one, the keyword “PARAM” may be omitted.

The parameters are used within the subroutine "CLIBRE" that must be written by the user, compiled and linked with the code libraries to produce a special code executable before launching the run.

The elements that accept the free material of type "CONDITION AUX LIMITES" are "CL1D" and "CLTU", which are respectively an element with one node and one dof and an element with one node and 7 dofs.

The main interest of this free material is to allow the user to specify boundary conditions applied to the 'fluid' degree of freedom of these elements, e.g. in order to model a special device mounted along a pipeline. In the case of "CLTU", only the 7-th dof is affected. The first 6 dofs concern the structure and are not affected.

The user can store for each element (and each integration point), the values he wants (up to 10) in the ECR table. For homogeneity with the other materials, the following data will be stored in the first two locations of the ECR table :

ECR(1) = DP : variation of pressure due to the device

ECR(2) = Density of the donor element

The eight other locations are free.

Examples:

The following example, taken from the standard benchmark "bm_cir_conteneur_eau", concerns the case of the brutal opening of a pressure container.

The container top is detached from the body and the opening (assumed circumferential) grows as the top moves away. The motion is parallel to the axis O_x . As a consequence of the detachment, the cross-section of the diaphragm across which the internal fluid passes is gradually increased, and the corresponding pressure drop is modified accordingly, as a function of the top distance.

The free material is identified by the index 906. Lines starting by a "!" are comments.

The material data are as follows:

MATERIAUX

LIBRE	CLIM	906	PREF	10E5	
	PARAM	1	25	22	0.1857
!--		ptfond	ptcouv	eldon	diam
		90e5	10e5	0.0	1.0

```

!--          pamon    pext    tau    ksi idel'cik (sortie)
          LECT  esort TERM
!
!    materiau libre 906 (ouverture circonferentielle) :
!          param(1) = numero du premier point
!          param(2) = numero du second point
!          param(3) = numero de l'element donneur
!          param(4) = diametre du tube
!          param(5) = pression amont initiale
!          param(6) = pression externe
!          param(7) = constante de temps pour l'ouverture
!          param(8) = perte de charge en sortie (idel'cik)
!

```

Programming example relative to CLIBRE:

```

      SUBROUTINE CLIBRE(NUNU,PREF,PARAM,AIRE,RHO,PAMON,VN,T,ECR,DP)
*
* -----
*          materiau libre pour el. clld          m.lepareux 08-95
* -----
*
*    attention !
*
*    le materiau nunu = 906 est utilise par "bm_cir_conteneur_eau.epx"
*
*    entree (ne pas les modifier) :
*          nunu      : numero de repereage pour l'utilisateur
*          pref      : pression de reference (obligatoire)
*          param     : tableau des parametres du mat. libre
*          aire      : section de la tuyauterie
*          rho       : masse volumique amont
*          pamon     : pression amont
*          vn        : vitesse du fluide dans la tuyauterie
*          t         : temps
*
*    sortie :
*          dp        : variation de pression due a l'appareil
*          ecr(1)    : affecte a dp
*          ecr(2)    : affecte a rho amont
*          ecr(3:9)  : selon utilisateur
*
*
*    les ecr libres permettent la sortie graphique
*    des grandeurs qui leur sont affectees
*
*    IMPLICIT NONE
*
*--  variables globales :
      INTEGER, INTENT(IN) :: NUNU
      REAL(8), INTENT(IN) :: PREF,AIRE,RHO,PAMON,VN,T,PARAM(*)
      REAL(8), INTENT(OUT) :: ECR(*),DP
*
      REAL(8), PARAMETER :: ZERO=1D-6, RMIN=1D-3, RS2MIN=1.005D0
      INTEGER, PARAMETER :: LON1=7
*
*--  variables locales :
      INTEGER NP1,NP2,LON,NELDON,KAS
      REAL*8 DIAM,PZERO,PAVAL,TAU,PEXT,VAL1(LON1),VAL2(LON1),DIST,SECT,
&          RAP,RS2,XKZ,PSEUIL,P0,XK,Q0,PP,ZMACH,
&          DPE,DPK,ROVK,DPMAX,ROVN,XKSI,CSON
      LOGICAL OUVERT
!
!
      SELECT CASE (NUNU)
*
      CASE (906)
!---      diaphragme pour une ouverture progressive
          NP1 = NINT(PARAM(1)) ! NUMERO DU PREMIER POINT
          NP2 = NINT(PARAM(2)) ! NUMERO DU DEUXIEME POINT

```

```

        NELDON = NINT(PARAM(3))      ! NUMERO DE L'ELEMENT DONNEUR
        DIAM   = PARAM(4)            ! DIAMETRE DU TUBE
        PZERO  = PARAM(5)            ! PRESSION AMONT INITIALE
        PAVAL  = PARAM(6)            ! PRESSION EXTERNE
        TAU    = PARAM(7)            ! CONSTANTE DE TEMPS POUR L'OUVERTURE
        XKSI   = PARAM(8)            ! PERTE DE CHARGE EN SORTIE (IDEL'CIK)

!
        OUVERT = .TRUE.

!
!--      on va chercher les déplacements des 2 noeuds :
        CALL QUIDNE (1,NP1,LON,VAL1)
        CALL QUIDNE (1,NP2,LON,VAL2)
        IF(LON > LON1) STOP ' CLIBRE : DIM INSUFFISANTES'
        DIST = ABS( VAL2(1) + VAL1(1) )
        SECT = 3.1416 * DIAM * DIST
        RAP = SECT / AIRE

!
!--      le rapport des sections (rap) est limite a RMIN
!              (cas des petites ouvertures) :
        IF( RAP .LT. RMIN ) THEN
            RAP = RMIN
            OUVERT = .FALSE.
        ENDIF
        RS2 = 1 / ( RAP*RAP )

!
!--      rs2 est limite a rs2min (cas des grandes ouvertures) :
        IF(RS2 < RS2MIN) THEN
            RS2 = RS2MIN
            RAP = SQRT(1/RS2MIN)
        ENDIF

!
!--      perte de charge (idel'cik) :
        XK = RS2 * XKSI

!
!-----      si tau .ne. 0 la pression aval chute progressivement :
!              (a condition que pamon > paval)
!
        PSEUIL = PZERO - PAVAL
        IF(TAU.GT.ZERO .AND. PSEUIL.GT.ZERO*PZERO) THEN
            PEXT = PAVAL + PSEUIL*EXP( -T / TAU)
        ELSE
            PEXT = PAVAL
        ENDIF

!
        DPMAX = PAMON - PREF
        IF(OUVERT) THEN
            DPE = PEXT - PREF
            DPK = 0.5*XK * RHO * VN*VN
        ELSE
            DPE = PAMON - PREF
            DPK = 0
        ENDIF
        DP = DPE + DPK
        IF(DP > DPMAX) DP = DPMAX

!
        ECR(1) = DP + PREF
        ECR(2) = RHO
        ECR(3) = RHO*VN      ! PRODUIT RHO*VN (DEBIT MASSIQUE UNITAIRE)
        ECR(4) = PEXT        ! PRESSION DE SORTIE ( PEXT OU PCRIT )
        ECR(5) = DIST        ! DISTANCE ENTRE LES FRAGMENTS
        ECR(6) = XK          ! COEF. DE PERTE DE CHARGE

!
!-----      coefficient de stabilite ( xk * rho * vn ) :
        ROVK = ABS(ECR(6)*ECR(3))

!
!--      on arrete le calcul quand pamon < pext :
        IF( PAMON < PEXT ) CALL TILT

!
        CASE DEFAULT
            CALL ERRMSS('CLIBRE','ROUTINE UTILISATEUR NON PROGRAMMEE')
            STOP ' "CLIBRE" ABSENT'
        END SELECT

!
        RETURN
    END

```

9.6.30 FREE PARTICLE MATERIAL**Object:**

This directive allows users to define their own constitutive laws for the particle elements (BILLE).

Syntax:

```
"BILLE" $ "LIBR" num "RO" rho $  
      ... < "FONC" numfon >  
      ... < "PARA" a b c ... > /LECTURE/
```

num

Number of the material used for the particle elements.

rho

Density.

numfon

Number of the function used.

"PARA" ...

Introduces a series of complementary parameters.

LECTURE

List of the concerned elements.

Comments:

The number (num) allows to distinguish between several user-defined materials.

The rho parameter is mandatory.

The number (numfon) allows to identify the function used for the interaction.

The complementary parameters introduced by "PARA" may be as many as needed. EUROPLEXUS recognizes the end of the parameters when the "LECTURE" keyword is encountered.

The subroutine "MBLIBR", to be written by the user, computes the interaction forces between neighbouring particles of the "BILLE" element considered, starting from the quantities at the beginning of the step, which are known. Consult the following example for a list of the available variables.

The only element type accepting this material is "BILLE".

The user may store for each element the variables of his choice within the ECR table (up to 7 values). However, for uniformity with the other materials, it is advised to use the first two slots as follows:

Fluid :

```
ECR(1) = Pressure
ECR(2) = Density
```

Continuum structure:

ECR(1) = Pressure
ECR(2) = Von Mises

Example:

Two new materials of the fluid type are defined: a material of type acoustic fluid, and the other depending upon the distance between two neighbouring particles.

The corresponding data will be, for example:

"BILL" "LIBR" 1 "RO" 1000 "PARA" 1000 /LECTURE/

"BILL" "LIBR" 2 "RO" 800 "FONC" 1 /LECTURE/

Programming example for routine MBLIBR:

```

SUBROUTINE MBLIBR(XMAT,DINI,DIST,A,B,C,NVOIS,NUMVOI,DX,DY,DZ,
* ROCOUR,IEL,INOE,INOE,IPFONC,TABFON,T,DT1,FORCE,SIG,ECR)
-----
MATERIAU "BILLE" "LIBRE"
-----
R.GALON 02/91
-----
ENTREE :
-----
XMAT(1) = MASSE VOLUMIQUE INITIALE
XMAT(2) = NUMERO DE REPERAGE DU MATERIAU UTILISATEUR
XMAT(3) = NUMERO DE LA FONCTION ASSOCIEE

```

```

C      XMAT(4: ) = AUTRES PARAMETRES DU MATERIAU
C      DINI      = DIAMETRE INITIAL DE LA BILLE
C      DIST      = DISTANCE SEPARANT LES 2 BILLES EN INTERACTION
C      A         = COSINUS DIRECTEUR SUIVANT X DE LA LIAISON
C      B         = COSINUS DIRECTEUR SUIVANT Y DE LA LIAISON
C      C         = COSINUS DIRECTEUR SUIVANT Z DE LA LIAISON
C      NVOIS     = NOMBRE DE BILLES VOISINES DE LA BILLE TRAITEE
C      NUMVOI    = NUMVOI-IEME BILLE EN INTERACTION
C      DVX       = VITESSE RELATIVE DES 2 BILLES DE LA LIAISON
C               SUIVANT X
C      DVY       = VITESSE RELATIVE DES 2 BILLES DE LA LIAISON
C               SUIVANT Y
C      DVZ       = VITESSE RELATIVE DES 2 BILLES DE LA LIAISON
C               SUIVANT Z
C      ROCOUR    = MASSE VOLUMIQUE ASSOCIEE A LA LIAISON
C      IEL       = NUMERO DE L ELEMENT TRAITE
C      INOE      = NUMERO DU NOEUD ASSOCIE A L ELEMENT IEL
C      INOE      = NUMERO DU NOEUD VOISIN DE LA BILLE
C      IPFONC    = POINTE SUR LA TABLE DE FONCTION
C      TABFON    = TABLE DE FONCTION ASSOCIEE AU MATERIAU
C      T         = TEMPS DE CALCUL
C      DT1       = INCREMENT DE TEMPS DE CALCUL
C
C
C      SORTIE :
C      -----
C      FORCE(1:3) = FORCES A APPLIQUER A LA BILLE TRAITEE
C      SIG(1:6)  = CONTRAINTES A LA FIN DU PAS (FACULTATIF)
C      ECR(1:10) = EMBLEMENTS LIBRES
C
C      REMARQUE : - DEUX BILLES SEPARÉES DE PLUS DE 1.3 * DINI SONT
C      ----- SUPPOSEES NE PAS POUVOIR INTERAGIR ENTRE ELLES.
C
C      - ON CUMULE TOUJOURS LES FORCES CAR ELLES PROVIENNENT
C      DE L INTERACTION DE TOUTES LES BILLES VOISINES DE LA
C      BILLE TRAITEE.
C
C      IMPLICIT REAL*8(A-H,O-Z)
C
C      DIMENSION XMAT(*),ECR(*),FORCE(*),TABFON(*),SIG(*),IPFONC(2,*)
C
C
C      NUM = XMAT(2)
C
C      IF(NUM.NE.1) GOTO 20
C
C      ----- CAS D UN MATERIAU DE TYPE FLUIDE
C      =====
C      RO = XMAT(1)
C      CSON = XMAT(4)
C      ----- POUR LA PREMIERE BILLE EN INTERACTION ON INITIALISE PAR EXEMPLE
C      LA MASSE VOLUMIQUE ET LA PRESSION MOYENNE DE L ELEMENT BILLE IEL
C      IF(NUMVOI.EQ.1)THEN
C          ECR(1)=0.
C          ECR(2)=0.
C          SIG(1)=0.
C          SIG(2)=0.
C          SIG(3)=0.
C      ENDIF
C      ----- DRO = VARIATION DE LA MASSE VOLUMIQUE
C      DRO = ROCOUR - RO
C      P = DRO * CSON * CSON
C      DVOL = (ROCOUR/RO) -1.DO
C      DP2 = DINI**3 / (DIST*(1.DO + DVOL))
C
C      ----- COEFFICIENT DE PONDERATION POUR UN RESEAU HEXAGONAL DE BILLES
C      COEF = SQRT(2.DO)/4.DO
C
C      ----- COEFFICIENT DE PONDERATION POUR UN RESEAU CUBIQUE DE BILLES
C      COEF = 1.DO
C      ----- FORCE DANS LA DIRECTION DE LA LIAISON APPLIQUEE A LA BILLE
C      FN = - DP2 * COEF * P
C      ----- ON PROJETTE LA FORCE DANS LE REPERE GLOBAL
C      FORCE(1) = FORCE(1) + A * FN
C      FORCE(2) = FORCE(2) + B * FN
C      FORCE(3) = FORCE(3) + C * FN
C      ----- CONTRAINTES DANS L ELEMENT (PRESSIONS)
C
C      SIG(1) = SIG(1) + P/NVOIS
C      SIG(2) = SIG(2) + P/NVOIS
C      SIG(3) = SIG(3) + P/NVOIS
C      SIG(4) = 0.
C      SIG(5) = 0.
C      SIG(6) = 0.
C      ----- MASSE VOLUMIQUE MOYENNE
C      ECR(1) = ECR(1) + ROCOUR/NVOIS
C      ----- PRESSION MOYENNE
C      ECR(2) = ECR(2) + P/NVOIS

```



```
      RETURN
C
C
20  CONTINUE
C
C ----- FORCE DEFINIE PAR UNE FONCTION
C      =====
C
C  REMARQUE : ON SUPPOSE ICI QUE LA FORCE AGISSANT SUR L ELEMENT BILLE
C             EST FONCTION UNIQUEMENT DE LA DISTANCE SEPARANT LES 2
C             BILLES EN INTERACTION (LA FONCTION EST DEFINIE PAR LA
C             DIRECTIVE "FONC".
C
C      IFONC = XMAT(3)
C
C ----- FN EST LA FORCE CORRESPONDANT A UNE DISTANCE DIST SEPARANT LES
C             2 BILLES EN INTERACTION (ELLE EST APPLIQUEE A L ELEMENT IEL)
C
C      CALL FFONCT(IFONC,DIST,FN,IPFONC,TABFON)
C
C ----- ON PROJETTE LA FORCE DANS LE REPERE GLOBAL
C
C      FORCE(1) = FORCE(1) + A * FN
C      FORCE(2) = FORCE(2) + B * FN
C      FORCE(3) = FORCE(3) + C * FN
C
C ----- DISTANCE MOYENNE DANS ECR(1) PAR EXEMPLE OU TOUTE AUTRE VALEUR
C             QUE L ON DESIRE CONSERVER
C
C      ECR(1) = ECR(1) + DIST/NVOIS
C
C      RETURN
C      END
```

9.6.31 VON MISES (ISPRA IMPLEMENTATION)

Object:

This option enables to choose the Von Mises material with the implementation developed at Ispra. Elasto-plasticity is implemented via a radial return algorithm. Only isotropic hardening is activated to date. There is no dependency on temperature nor on strain rate.

Syntax:

```
"VM23"  "R0" rho  "YOUN" young  "NU" nu  "ELAS" sige ...  
        <"FAIL" $[ VMIS ; PEPS ; PRES ; PEPR ]$ "LIMI" limit>  
        "TRAC" npts*(sig eps)  
        /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

sige

Elastic limit.

FAIL

Optional keyword: introduces an element failure model, represented by a failure criterion and a by failure limit value. The available failure criteria are: **VMIS** for a criterion based upon Von Mises stress (isotropic criterion), **PEPS** for a criterion based upon the principal strain (see caveat below), **PRES** for a criterion based upon the hydrostatic stress, **PEPR** for a criterion based upon the principal strain if the hydrostatic stress is positive (traction): if the hydrostatic stress is negative (compression) there is no failure.

limit

Optional parameter, indicates the failure limit for the chosen criterion.

"TRAC"

This key-word announces the yield curve.

npts

Number of points (except the origin) defining the yield curve.

sig

Stress.

eps

Total strain (elastic + plastic).

LECTURE

List of the elements concerned.

Comments:

1/ - The young parameter defines Young's modulus during an elastic phase.

2/ - The points (sig,eps) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.

3/ - The slope of the yield curve may not increase from one segment to the following one.

4/ - When using a failure criterion based upon the principal strains (PEPS or PEPR) be aware that the criterion is based upon the *cumulated* strains. These are usually a good approximation of the total strains for elements using a convected reference frame for the stresses and strains (such as e.g. plate, shell or bar elements). The approximation is likely to be very bad, instead, for continuum-like elements, at least when there are large rotations.

Outputs:

The components of the ECR table are as follows:

ECR(1): current hydrostatic pressure

ECR(2): current equivalent stress (Von Mises)

ECR(3): current equivalent plastic strain

ECR(4): total x-strain

ECR(5): total y-strain

ECR(6): total xy-strain

ECR(7): current yield stress

ECR(8): total z-strain

ECR(9): total yz-strain (only in 3D)

ECR(10): total xz-strain (only in 3D)

ECR(11): sound speed

ECR(12): failure flag (0=virgin Gauss Point, 1=failed Gauss Point)

9.6.32 “VM1D” MATERIAL

Object:

This is the material to be used for the interface elements of type ED1D (see INT.80).

Syntax:

```
"VM1D"  "PT1D"  pt1d  /LECTURE/
```

pt1d

Associated node index in the 1-D model.

Comments:

Note that when several ED1D elements are present in a coupled 1-D/multi-D calculation, then each ED1D element must have a separate VM1D material, because the material is used to carry the information of the associated 1-D node to each one ED1D element (pt1d).

Outputs:

The components of the ECR table are as follows :

ECR(1) : unused

ECR(2) : unused

ECR(3) : unused

ECR(4) : unused

ECR(5) : unused

ECR(6) : unused

9.6.33 “DONE” MATERIAL

Object:

This is a viscoplastic material model mostly used to describe the sensitivity of commonly used stainless steels (e.g. AISI 304 and 316) to the rate of loading. It uses the theory of viscoplasticity based on total strain and overstress. To date, it is limited to small strains.

Syntax:

```
"DONE"  "RO" rho  "YOUN" young  "NU" nu  "ELAS" sige ...  
...  "VIS1" vis1  "VIS2" vis2  "VIS3" vis3  ...  
...  "VIS4" vis1  "VIS5" vis2  "VIS6" vis3  ...  
...  "TRAC" npts*(sig eps) /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

sige

Elastic limit.

vis1,...,vis6

Viscous coefficients.

"TRAC"

This key-word announces the yield curve (static).

npts

Number of points (except the origin) defining the static yield curve.

sig

Stress.

eps

Total strain (elastic + plastic).

LECTURE

List of the elements concerned.

Comments:

- 1/ - The young parameter defines Young's modulus during an elastic phase.
- 2/ - The points (sig,eps) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.
- 3/ - The slope of the static yield curve may not increase from one segment to the following one.

Outputs:

The components of the ECR table are as follows:

- ECR(1): current hydrostatic pressure
- ECR(2): current equivalent stress (Von Mises)
- ECR(3): current equivalent plastic strain
- ECR(4): total x-strain
- ECR(5): total y-strain
- ECR(6): total xy-strain
- ECR(7): current yield stress
- ECR(8): total z-strain
- ECR(9): x-overstress
- ECR(10): y-overstress
- ECR(11): xy-overstress
- ECR(12): z-overstress
- ECR(13): previous time
- ECR(14): total yz-strain (only 3D)
- ECR(15): total xz-strain (only 3D)
- ECR(16): yz-overstress
- ECR(17): xz-overstress

Let P represent the point of intersection (in the equivalent stress - equivalent strain space) between the unloading path and the equilibrium stress-strain curve.

- ECR(18): ECR(4) at point P
- ECR(19): ECR(5) at point P
- ECR(20): ECR(8) at point P
- ECR(21): ECR(6) at point P
- ECR(22): ECR(14) at point P
- ECR(23): ECR(15) at point P
- ECR(24): equivalent total strain at point P
- ECR(25): old equivalent total strain
- ECR(26): EPSC (equivalent strain parameter)

EPSC is defined by the cyclic hardening law. It corresponds to the distance between point P and the new origin in the strain direction.

- ECR(27): current cumulative value of number of crossings of the unloading path with the equilibrium stress-strain diagram
- ECR(28): new x-stress at point P
- ECR(29): new y-stress at point P
- ECR(30): new xy-stress at point P
- ECR(31): new z-stress at point P
- ECR(32): new yz-stress at point P
- ECR(33): new xz-stress at point P
- ECR(34): new equivalent stress at point P
- ECR(35): old equivalent stress
- ECR(36): old equilibrium equivalent stress
- ECR(37): old (Young's modulus * total strain)
- ECR(38): sound speed

9.6.34 VON MISES WITH VISCOPLASTIC REGULARIZATION

Object:

This directive enables to choose an elastoplastic constitutive theory with Von Mises yield surface, associative flow rule, and isotropic hardening or softening, including a viscoplastic regularization. Elasto-plasticity is implemented via a radial return algorithm.

For more information about the theory, please refer to: J.C. Simo, J.G. Kennedy and S. Govindjee, "Non-Smooth Multisurface Plasticity and Viscoplasticity. Loading/Unloading Conditions and Numerical Algorithms", Int. J. Num. Meth. Eng., Vol 26, pp. 2161-2185 (1988).

Syntax:

```
"VMSF" "RO" rho "YOUN" young "NU" nu "ELAS" sige "ETA" eta  
... "TRAC" npts*(sig eps) /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

sige

Elastic limit.

eta

Viscoplastic parameter (relaxation time).

"TRAC"

This key-word announces the yield curve.

npts

Number of points (except the origin) defining the yield curve.

sig

Stress.

eps

Total strain (elastic + plastic).

LECTURE

List of the elements concerned.

Comments:

1/ - The young parameter defines Young's modulus during an elastic phase.

2/ - The points (sig,eps) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.

3/ - The slope of the yield curve may become negative in the softening part of the curve.

Outputs:

The components of the ECR table are as follows:

ECR(1): current hydrostatic pressure

ECR(2): current equivalent stress (Von Mises)

ECR(3): current equivalent plastic strain

ECR(4): total x-strain

ECR(5): total y-strain

ECR(6): total xy-strain

ECR(7): current yield stress

ECR(8): total z-strain

ECR(9): total yz-strain (only in 3D)

ECR(10): total xz-strain (only in 3D)

ECR(11): x-stress before viscoplastic correction

ECR(12): y-stress before viscoplastic correction

ECR(13): xy-stress before viscoplastic correction

ECR(14): z-stress before viscoplastic correction

ECR(15): yz-stress before viscoplastic correction

ECR(16): xz-stress before viscoplastic correction

ECR(17): current time

ECR(18): sound speed

9.6.35 DRUCKER PRAGER WITH VISCOPLASTIC REGULARIZATION

Object

This directive enables to choose an elastoplastic constitutive theory with Drucker Prager yield surface, associative or non-associative flow rule, including hardening or softening, and a viscoplastic regularization.

This material is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC.

The regularization technique is the same as the one implemented in the VMSF material, see: J.C. Simo, J.G. Kennedy and S. Govindjee, "Non-Smooth Multisurface Plasticity and Viscoplasticity. Loading/Unloading Conditions and Numerical Algorithms", Int. J. Num. Meth. Eng., Vol 26, pp. 2161-2185 (1988).

The model uses two parameters, α and c , related to the angle and the cohesion parameters of the classical Drucker Prager model. These two parameters are not constant in general, but depend on the plastic strain. Hardening and/or softening are thus possible.

References

More information on the formulation of this material model may be found in reference [\[120\]](#).

Syntax

```
"DPSF"  "RO" rho  "YOUN" young  "NU" nu
        "ALF1" alf1 "C1" c1 "BETA" beta "ETA" eta
        <"FAIL" $[ PEPS ; PEPR ]$ "LIMI" limit >
        "TRAA" npta*(alfa epsp)
        "TRAC" npts*(c      epsp) /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

alf1

First value ("yield limit") of the TRAA curve for the alfa parameter, see below.

c1

First value ("yield limit") of the TRAC curve for the c parameter, see below.

beta

Parameter indicating whether the model is associative or non associative. If the alfa parameter (given by the "TRAA" directive below) does not depend upon the plastic strain, and beta=alf1, then an associative rule is taken. Otherwise, the law is non associative. E.g., beta=0 corresponds to return along a cylinder.

eta

Viscoplastic parameter (relaxation time).

FAIL

Optional keyword: introduces an element failure model, represented by a failure criterion and a by failure limit value. The available failure criteria are: **PEPS** for a criterion based upon the principal strain (see caveat below), **PEPR** for a criterion based upon the principal strain if the hydrostatic stress is positive (traction): if the hydrostatic stress is negative (compression) there is no failure.

limit

Optional parameter, indicates the failure limit for the chosen criterion.

"TRAA"

This key-word announces the curve defining the variation of the alfa parameter with the plastic strain.

npta

Number of points defining the curve.

alfa

Value of the alfa parameter.

epsp

Correponding value of the plastic strain.

"TRAC"

This key-word announces the curve curve defining the variation of the c parameter with the plastic strain.

npts

Number of points defining the curve.

c

Value of the c parameter.

epsp

Corresponding value of the plastic strain.

LECTURE

List of the elements concerned.

Comments:

4/ - When using a failure criterion based upon the principal strains (PEPS or PEPR) be aware that the criterion is based upon the *cumulated* strains. These are usually a good approximation of the total strains for elements using a convected reference frame for the stresses and strains (such as e.g. plate, shell or bar elements). The approximation is likely to be very bad, instead, for continuum-like elements, at least when there are large rotations.

Outputs:

The components of the ECR table are as follows:

- ECR(1): current hydrostatic pressure
- ECR(2): current equivalent stress (Von Mises)
- ECR(3): current equivalent plastic strain
- ECR(4): total x-strain
- ECR(5): total y-strain
- ECR(6): total xy-strain
- ECR(7): cohesion
- ECR(8): total z-strain
- ECR(9): total yz-strain (only in 3D)
- ECR(10): total xz-strain (only in 3D)
- ECR(11-16): stresses before viscoplastic correction
- ECR(17): current time
- ECR(18): alfa

ECR(19): zone (sigma - tau plane)

ECR(20): yield ($f = \alpha \cdot \sigma + \tau - \text{cohe}$), >0 if plast

ECR(21): failure flag (0=virgin Gauss Point, 1=failed Gauss Point).

ECR(22): sound speed

9.6.36 COMPOSITE MATERIAL (LINEAR ORTHOTROPIC) ISPRA IMPLEMENTATION

Object:

The option is used to enter materials with a linear orthotropic behaviour into a coordinate system defined by the user. The model is suitable to represent e.g. composite materials.

Syntax:

```
"COMM"  "R0"  rho    "YG1"  yg1    "YG2"  yg2  "YG3"  yg3
          "NU12" nu12  "NU13" nu13  "NU23" nu23
          "G12"  g12   "G13"  g13   "G23"  g23
          /LECTURE/
```

rho

Density of the material.

yg1

Young's modulus along direction 1.

yg2

Young's modulus along direction 2.

yg3

young's modulus along direction 3.

nu12

Poisson's ration between direction 1 and 2.

nu 13

Poisson's ration between direction 1 and 3.

nu23

Poisson's ratio between direction 2 and 3.

g12

Shear modulus between direction 1 and 2.

g13

Shear modulus between direction 1 and 3.

g23

Shear modulus between direction 2 and 3.

LECTURE

List of the elements concerned.

Comments:

This option may be repeated as many times as necessary.

The associated orthotropy directions are to be specified via the `COMP ORTS` directive (see page C.97).

Outputs:

The different components of the ECR table are as follows:

ECR(1) : CURRENT HYDROSTATIC PRESSURE ($1/3(SX+SY+ST)$)

ECR(2) : CURRENT EQUIVALENT STRESS (VON MISES)

ECR(3) : CURRENT EQUIVALENT PLASTIC STRAIN

ECR(4) : cumulated X-STRAIN

ECR(5) : cumulated Y-STRAIN

ECR(6) : cumulated XY-STRAIN

ECR(7) : CURRENT YIELD STRESS

ECR(8) : cumulated Z-STRAIN

ECR(9) : cumulated YZ-STRAIN (ONLY 3-D)

ECR(10): cumulated XZ-STRAIN (ONLY 3-D)

ECR(11): sound speed (added 26.2.92)

ecr(12): angle alpha between lamina coordinate 1 and orthotropy direction 1

ecr(13): 10.

ecr(14): 10.

9.6.37 MODIFIED CAM-CLAY MATERIAL

Object

The directive is used to enter materials with a modified Cam-clay behaviour. The model is suitable to represent e.g. cohesive soil materials.

Although the model includes some treatment of the water possibly present in soils, the use of this feature is strongly discouraged because the modeling appears somewhat inconsistent in that case: for example, water motion within the soil is not treated, water pressure is not taken into account to compute internal forces, and finally the calculation of masses seems inconsistent. To model a dry soil, just leave out the keyword ROW: then the code assumes $\rho_w = 0$, the value given for ρ is the density of the (dry) soil alone, and the value given for z_f , if any, is irrelevant.

References

More information on the formulation of this material model may be found in reference [\[147\]](#).

Syntax

```
CAMC  RO ro |[ NU nu ; G g ]|
      M m      LAM lam  K k      E e      <ROW row>
      KO kO     OCR ocr
      |[ ZF zf SLEV slev GRAV grav ; PRES pres ]|
      /LECT/
```

ro

Initial density ρ of the soil (including the water, if any: but see the comments above and below).

nu

Poisson's coefficient ν . If this value is given, then the shear modulus G may not be given and the calculation is done with constant Poisson's coefficient (G will vary accordingly).

g

Shear modulus G . If this value is given, then ν may not be given and the calculation is done with constant shear modulus (ν will vary accordingly).

m

Critical state parameter M . Corresponds to the CLAY model's M parameter. For the physical meaning, see the Remarks below.

lam

Isotropic consolidation modulus (λ).

k

Unloading-reloading modulus (κ).

e

Initial void ratio e , defined as: $e = V_{\text{Voids}}/V_{\text{Solid}}$.

row

Water density ρ_w . Use 0.0 for dry soils, or just leave out this keyword since the default value is 0.0. Note that in this case the value for ρ given above indicates the density of the (dry) soil alone.

k0

Coefficient of earth pressure at rest (K_0).

ocr

Overconsolidation ratio O_{cr} : ($O_{cr} = 1$ for normal-consolidated soil).

zf

Upper level of water, i.e. water surface “vertical” coordinate (y in 2D, z in 3D). Used to compute the in-situ (initial) stress and hardening state. This quantity is unused, and thus any value may be given, e.g. $z_f = 0$, if the user has specified $\rho_w = 0$ (dry soil).

slev

Upper level of soil, i.e. soil surface “vertical” coordinate (y in 2D, z in 3D). Used to compute the in-situ (initial) stress and hardening state.

grav

Acceleration of gravity along the “vertical” coordinate (y in 2D, z in 3D). Used to compute the in-situ (initial) stress and hardening state.

pres

Initial hydrostatic (uniform) pressure state. Note that here (but not for stresses SIG etc.) a positive value should be used to indicate an initial compression (negative stress).

LECTURE

List of the elements concerned.

Comments

This option may be repeated as many times as necessary.

This material model seems unable to start from initial stress-free conditions, so that in-situ (initial) stresses should always be specified.

The initial in-situ conditions (stresses and some of the ECR components) for elements using this material are computed by using the parameters (**zf**, **slev**, **grav**) or **pres**. One and only one of these two sets must be given. In the following discussion, the term “vertical” refers to the y -coordinate in 2D, to the z -coordinate in 3D calculations.

A) If **pres** (p) is specified, then the initial state is uniform hydrostatic stress ($-p$) all over the current CAMC material. This is typical, e.g., of simple one-element tests to check the behaviour of the constitutive law, or of simple laboratory experiments.

In this case, the code simply sets:

$$\sigma_1 = -p \quad , \quad \sigma_2 = -p \quad , \quad \sigma_3 = -p.$$

B) If (**zf**, **slev**, **grav**) are specified, then the initial conditions are computed as follows. The model assumes a horizontally stratified (homogeneous) soil, the lower part of which may contain water. The quantities z_f and s_{lev} are the vertical coordinates of the upper water and soil levels, respectively. Normally it should be $s_{lev} > z_f$ so that the soil layer between z_f and s_{lev} is dry (no water) while the soil below that level is saturated by water.

For each element with the current CAMC material, the code computes the vertical coordinate of its centroid z_c . Then the vertical stress due to the soil weight (effective stress) is:

$$\sigma_v = -g(\rho - \rho_w)(s_{lev} - z_c),$$

where ρ is the density of the wet soil (soil plus water), ρ_w is the density of the water. Thus, the difference between the two is the density of the (dry) soil. The vertical stress may not be positive:

$$\sigma_v = \text{MIN}(\sigma_v, 0).$$

The horizontal stress is given by:

$$\sigma_h = K_0 \sigma_v,$$

where K_0 is the **k0** parameter specified above. Then, the code sets:

$$\sigma_1 = \sigma_h \quad , \quad \sigma_2 = \sigma_h \quad , \quad \sigma_3 = \sigma_v.$$

In addition to soil (effective) stresses, the water pressure (hydrostatic) is also evaluated:

$$p_w = -g\rho_w(z_f - z_c).$$

The water pressure may not be positive:

$$p_w = \text{MIN}(p_w, 0).$$

This quantity is stored in ECR(13). Note, however, that the water pressure does *not* contribute to internal forces in the CAMC model: only the effective (soil) stresses are used.

Note also that if (**zf**, **slev**, **grav**) are specified one should also probably specify a “global” gravity term (equal to the value of g given above) by means e.g. of the **CHAR CONS GRAV** directive, in order to have (at least approximate) equilibrium in the initial configuration. In addition, suitable boundary conditions must also be prescribed along the envelope of the CAMC soil region.

Outputs

The different components of the ECR table are as follows:

ECR(1) : current hydrostatic pressure $\frac{1}{3}\text{tr}(\sigma)$

ECR(2) : square root of the second invariant of the deviatoric stress tensor J'_2 (i.e. square root of Von Mises equivalent stress)

ECR(3) : current void ratio

ECR(4) : total x -strain

ECR(5) : total y -strain

ECR(6) : total xy -strain

ECR(7) : hardening parameter P_c (isotropic consolidation pressure)

ECR(8) : total z -strain

ECR(9) : total yz -strain

ECR(10): total zx -strain

ECR(11): sound speed

ECR(12): water overpressure (u)

ECR(13): initial water pressure (p_0), $p = p_0 + u$ (p is the total water pressure)

ECR(14): volumetric strain ($\epsilon_V = \epsilon_x + \epsilon_y + \epsilon_z$)

ECR(15): deviatoric strain

$$\epsilon_d = \sqrt{\frac{2}{3}[(\epsilon_x - \epsilon_y)^2 + (\epsilon_y - \epsilon_z)^2 + (\epsilon_z - \epsilon_x)^2] + (\gamma_{xy}^2 + \gamma_{yz}^2 + \gamma_{xz}^2)}$$

The components of the stress tensor are as follows:

SIG(1): σ_x

SIG(2): σ_y

SIG(3): σ_z

SIG(4): τ_{xy}

SIG(5): τ_{yz} (only 3D)

SIG(6): τ_{xz} (only 3D)

Remarks

Let M_1 be the ratio between the second invariant of the stress tensor J_2 and the first invariant of the stress tensor J_1 at critical state (i.e. for stress points which lie on the failure surface). This is the quantity which is usually available from tests.

Let M_2 be the ratio between the second invariant of the *deviatoric* stress tensor J'_2 and the first invariant of the stress tensor J_1 at critical state.

The M parameter defined above in the input syntax corresponds to M_1 . However, note that in the model description of the CAMC material the quantity $g(\theta)$ corresponds rather to M_2 .

The following relation holds between the two quantities: $M_2 = M_1/\sqrt{3}$.

9.6.38 MODIFIED CAM-CLAY MATERIAL WITH VISCOPLASTIC REGULARIZATION

Object

The option is used to enter materials with a modified Cam-clay behaviour. The model is suitable to represent e.g. (dry) soil materials. The main differences with respect to the CAMC material are that:

- the CLAY model uses a fully implicit backward algorithm for the trial stress point to return onto the yield surface (similar to the radial return algorithm);
- the model includes an optional viscoplastic regularization;
- no attempt is made to take into account the possible presence of water (this feature is dubious in the CAMC material anyway).

Like for the CAMC material, the user may choose between a calculation with constant shear modulus and one with constant Poisson's coefficient.

References

More information on the formulation of this material model may be found in reference [\[123\]](#).

Syntax

```
CLAY RO ro |[ NU nu ; G g ]|
      M m      LAM lam K k PO p0
      KO k0
      < BETA beta > < NUM num >
      |[ SLEV slev GRAV grav ; PRES pres ]|
      /LECT/
```

ro

Initial density ρ of the (dry) soil. Water content is not taken into account by this model.

nu

Poisson's coefficient ν . If this value is given, then G may not be given and the calculation is done with constant Poisson's coefficient (G will vary accordingly).

g

Shear modulus G . If this value is given, then ν may not be given and the calculation is done with constant shear modulus (ν will vary accordingly).

m

Critical state parameter M . Corresponds to the CAMC model's M parameter. For the physical meaning, see the Remarks below.

lam

First loading slope (λ). This is the slope of the normal consolidation line, divided by the reference volume V_λ . The normal consolidation line is defined in the plane $[V, \ln(P)]$, where V is the so-called specific volume ($V = 1 + e$, e being the void ratio i.e. the volume of the voids divided by volume of the solid). V_λ is the specific volume at unit pressure. P is the pressure.

k

Unloading-reloading slope κ . This is the slope of the unloading-reloading line, divided by the reference volume V_λ . The unloading-reloading line is defined in the plane $[V, \ln(P)]$, where V is the so-called specific volume ($V = 1 + e$, e being the void ratio i.e. the volume of the voids divided by volume of the solid). V_λ is the specific volume at unit pressure. P is the pressure.

p0

Initial value of the hardening parameter p_0 .

k0

Coefficient of earth pressure at rest (K_0).

beta

Relaxation modulus β for the viscoplastic regularization. If $\beta = 0$, then no regularization is performed. By default, the code assumes $\beta = 0$.

num

Index (integer) n used by the initialization routine INICLA in order to set some initial properties of the soil (initial stresses and initial hardening parameters). By default, the program assumes $n = 0$.

slev

Upper level of soil, i.e. soil surface “vertical” coordinate (y in 2D, z in 3D). Used to compute the in-situ (initial) stress and hardening state.

grav

Acceleration of gravity along the “vertical” coordinate (y in 2D, z in 3D). Used to compute the in-situ (initial) stress and hardening state.

pres

Initial hydrostatic (uniform) pressure state. Note that here (but not for stresses SIG etc.) a positive value should be used to indicate an initial compression (negative stress).

LECTURE

List of the elements concerned.

Comments

This option may be repeated as many times as necessary.

The initial in-situ conditions (stresses and some of the ECR components) for elements using this material are computed by using the parameters (**slev**, **grav**) or **pres**. One and only one of these two sets must be given. In the following discussion, the term “vertical” refers to the y -coordinate in 2D, to the z -coordinate in 3D calculations.

A) If **pres** (p) is specified, then the initial state is uniform hydrostatic stress ($-p$) all over the current **CLAY** material. This is typical, e.g., of simple one-element tests to check the behaviour of the constitutive law, or of simple laboratory experiments.

In this case, the code simply sets:

$$\sigma_1 = -p \quad , \quad \sigma_2 = -p \quad , \quad \sigma_3 = -p.$$

B) If (**slev**, **grav**) are specified, then the initial conditions are computed as follows. The model assumes a horizontally stratified (homogeneous) soil in dry conditions, i.e. containing no water. The quantity s_{lev} is the vertical coordinate of the upper soil level.

For each element with the current **CLAY** material, the code computes the vertical coordinate of its centroid z_c . Then the vertical stress due to the soil weight (effective stress) is:

$$\sigma_v = -g\rho(s_{lev} - z_c),$$

where ρ is the density of the (dry) soil. The vertical stress may not be positive:

$$\sigma_v = \text{MIN}(\sigma_v, 0).$$

The horizontal stress is given by:

$$\sigma_h = K_0\sigma_v,$$

where K_0 is the k_0 parameter specified above. Then, the code sets:

$$\sigma_1 = \sigma_h \quad , \quad \sigma_2 = \sigma_h \quad , \quad \sigma_3 = \sigma_v.$$

Note that if (**slev**, **grav**) are specified one should also probably specify a “global” gravity term (equal to the value of g given above) by means e.g. of the **CHAR CONS GRAV** directive, in order to have (at least approximate) equilibrium in the initial configuration. In addition, suitable boundary conditions must also be prescribed along the envelope of the **CLAY** soil region.

Outputs

The different components of the ECR table are as follows:

- ECR(1) : current hydrostatic pressure $\frac{1}{3}(\sigma_x + \sigma_y + \sigma_z)$
- ECR(2) : current bulk modulus
- ECR(3) : second invariant of the deviatoric cumulated strain
- ECR(4) : cumulated x -strain
- ECR(5) : cumulated y -strain
- ECR(6) : cumulated xy -strain
- ECR(7) : hardening parameter p_0
- ECR(8) : cumulated z -strain
- ECR(9) : cumulated yz -strain (only 3D)
- ECR(10): cumulated xz -strain (only 3d)
- ECR(11): sound speed
- ECR(12): current value of the shear modulus G
- ECR(13): current value of the Poisson's coefficient ν

The components of the stress tensor are as follows:

- SIG(1): σ_x
- SIG(2): σ_y
- SIG(3): σ_z
- SIG(4): τ_{xy}
- SIG(5): τ_{yz} (only 3D)
- SIG(6): τ_{xz} (only 3D)

Remarks

Let M_1 be the ratio between the second invariant of the stress tensor J_2 and the first invariant of the stress tensor J_1 at critical state (i.e. for stress points which lie on the failure surface). This is the quantity which is usually available from tests.

Let M_2 be the ratio between the second invariant of the *deviatoric* stress tensor J_2' and the first invariant of the stress tensor J_1 at critical state.

The M parameter defined above in the input syntax corresponds to M_1 . However, note that in the model description of the CLAY material (*An Implementation of the Cam-Clay Elasto-Plastic Model Using a Backward Interpolation and Visco-Plastic Regularization*, Technical Note I.96.239) the quantity M corresponds rather to M_2 .

The following relation holds between the two quantities: $M_2 = M_1/\sqrt{3}$.

9.6.39 FUNE (SPECIALIZED CABLE MATERIAL)**Object:**

This model represents an elastoplastic cable, with no resistance in compression, and should be used in conjunction with special cable elements FUN2 (in 2D) and FUN3 (in 3D). The material is elasto-plastic in traction.

Syntax:

```
"FUNE"  "RO" rho  "YOUN" young  "NU" nu  "ELAS" sige "ERUP" erup ...  
...  "TRAC" npts*(sig eps) /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

sige

Elastic limit.

erup

Rupture strain.

"TRAC"

This key-word announces the yield curve (in traction).

npts

Number of points (except the origin) defining the yield curve.

sig

Stress.

eps

Total strain (elastic + plastic).

LECTURE

List of the elements concerned.

Comments:

- 1/ - The young parameter defines Young's modulus during an elastic phase.
- 2/ - The points (sig,eps) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.
- 3/ - The slope of the yield curve may not increase from one segment to the following one.

Outputs:

The components of the ECR table are as follows:

- ECR(1): total longitudinal strain of the cable element
- ECR(2): total lateral strain of the cable element
- ECR(3): plastic longitudinal strain of the cable element
- ECR(4): current yield stress in traction (0 if broken)
- ECR(5): sound speed

9.6.40 THE JOHNSON-COOK MODEL

Object:

This directive enables to choose the Johnson-Cook model with the implementation developed at Ispra. Elasto-plasticity is implemented via a radial return algorithm. Only isotropic hardening is activated to date and strain-rate dependency is included in the model. However, no temperature effects are included in the present implementation of the model.

References

The implementation of this material model is described in reference [167].

The Johnson-Cook constitutive relation is given by:

$$\sigma_{eq} = \left[A_1 + A_2 \left(\varepsilon_{eq}^p \right)^{\lambda_2} \right] \left[1 + \lambda_1 \ln \left(\frac{\dot{\varepsilon}_{eq}^p}{\dot{\varepsilon}_{eq,ref}^p} \right) \right] (1 - \theta^m) \quad (1)$$

where:

- A_1 is COA1 (1st constant) in Europlexus
- A_2 is COA2 (2nd constant) in Europlexus
- λ_1 is CLB1 (3rd constant) in Europlexus
- λ_2 is CLB2 (hardening parameter) in Europlexus
- $\dot{\varepsilon}_{eq,ref}^p$ is SRRF (reference strain rate) in Europlexus
- θ is the homologous temperature $\frac{T-T_{room}}{T_{melting}-T_{room}}$ (currently not implemented in Europlexus)
- m is the homologous temperature exponent (currently not implemented in Europlexus)

The Johnson-Cook model is a simple empirical generalization of Ludwik's constitutive law (see VMLU on page C.253), represented by the first term of the above equation, trying to account for dynamic (strain rate) effects (included in the second term of the equation) and for temperature effects (third and last term). The “reference” strain rate is the minimum plastic strain rate for which calibration of the model has been made.

In Johnson-Cook's model the Ludwik's law (first term) is multiplied by a function of the equivalent plastic strain rate. The form of this function is related to the often made experimental observation that the increase in flow stress is a logarithmic function of the strain rate.

The reference (or minimum) equivalent plastic strain rate $\dot{\varepsilon}_{eq,ref}^p$ is the value of equivalent plastic strain rate under which the material behaves in a “static” (i.e., strain-rate independent)

way. In practice, in the code, when the equivalent plastic strain rate is below this value, only the static part of the model is considered.

Syntax:

```
"VMJC"  "R0" rho  "YOUN" young  "NU" nu  "COA1" coa1
        "COA2" coa2 "CLB1" clb1 "CLB2" clb2 "SRRF" srrf
        <"FAIL" $[ "VMIS" "LIMI" limit ;
                  "DPLS" "LIMI" limit ;
                  "JOCO" "COD1" cod1 "COD2" cod2 "COD3" cod3 "COD4" cod4
                  ]$ >
... /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

coa1

1st constant (A_1) in the Johnson-Cook model.

coa2

2nd constant (A_2) in the Johnson-Cook model.

clb1

3rd constant (λ_1) in the Johnson-Cook model.

clb2

Hardening coefficient (λ_2) of the Johnson-Cook model.

srrf

Reference strain rate ($\dot{\varepsilon}_{\text{eq,ref}}^{\text{p}}$) of the Johnson-Cook model.

FAIL

Optional keyword: introduces an element failure model. The available failure criteria are: **VMIS** for a criterion based upon the equivalent Von-Mises stress, **DPLS** for a criterion based upon the equivalent plastic strain, **JOCO** for the so-called Johnson-Cook criterion based upon an equivalent plastic strain, depending on the strain rate and the triaxiality ratio. See comments below.

limit

Optional parameter, indicates the failure limit for the VMIS or PLAS criterion.

cod1

Optional parameter, 1st constant (D_1) in the Johnson-Cook criterion.

cod2

Optional parameter, 2nd constant (D_2) in the Johnson-Cook criterion.

cod3

Optional parameter, 3rd constant (D_3) in the Johnson-Cook criterion.

cod4

Optional parameter, 4th constant (D_4) in the Johnson-Cook criterion.

LECTURE

List of the elements concerned.

Comments:

1/ - The **young** parameter defines Young's modulus during an elastic phase.

2/ - The points (σ, ε) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.

3/ - The slope of the yield curve may not increase from one segment to the following one.

4/ - The Johnson-Cook failure criterion is given by:

$$\varepsilon_p^f = [D_1 + D_2 \exp(D_3 \sigma^*)] \left[1 + D_4 \ln \left(\frac{\dot{\varepsilon}_{eq}^p}{\dot{\varepsilon}_{eq,ref}^p} \right) \right] \quad (2)$$

where:

- D_1 is COD1 (1st constant) in Europlexus
- D_2 is COD2 (2nd constant) in Europlexus
- D_3 is COD3 (3rd constant) in Europlexus
- D_4 is COD4 (4th constant) in Europlexus
- $\dot{\varepsilon}_{eq,ref}^p$ is SRRF (reference strain rate) in Europlexus

- $\sigma^* = \frac{p}{q}$ is the triaxiality ratio, where p is the hydrostatic pressure and q is the Von Mises equivalent stress.

A damage parameter triggers failure when it reaches 1. It is computed as:

$$D = \sum \frac{\Delta \varepsilon_p}{\varepsilon_p^f} \quad (3)$$

Outputs:

The components of the ECR table are as follows:

- ECR(1): current hydrostatic pressure
- ECR(2): current equivalent stress (Von Mises)
- ECR(3): current equivalent plastic strain
- ECR(4): total x-strain
- ECR(5): total y-strain
- ECR(6): total xy-strain
- ECR(7): current yield stress
- ECR(8): total z-strain
- ECR(9): total yz-strain (only in 3D)
- ECR(10): total xz-strain (only in 3D)
- ECR(11): sound speed
- ECR(12): equivalent strain rate (Von Mises)
- ECR(13): failure flag (0=virgin Gauss Point, 1=failed Gauss Point)
- ECR(14): damage parameter for the Johnson-Cook criterion

9.6.41 THE LUDWIG-PRANDTL MODEL

Object:

This directive enables to choose the Ludwig-Prandtl model, a purely elasto-plastic model implemented at Ispra. Elasto- plasticity is implemented via a radial return algorithm. Only isotropic hardening is activated to date. There is no dependency on temperature but strain rate effects are included.

References

The implementation of this material model is described in reference [\[167\]](#).

Syntax:

```
"VMLP"  "R0" rho  "YOUN" young  "NU" nu  "COA1" coa1
        "COA2" coa2 "CLB1" clb1  "CLB2" clb2 "CLB3" clb3
        "CLB4" clb4
... /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

coa1

1st constant in the Ludwig-Prandtl model.

coa2

2nd constant in the Ludwig-Prandtl model.

clb1

3rd constant in the Ludwig-Prandtl model.

clb2

4th constant in the Ludwig-Prandtl model.

c1b3

5th constant in the Ludwig-Prandtl model.

c1b4

6th constant in the Ludwig-Prandtl model.

LECTURE

List of the elements concerned.

Comments:

1/ - The young parameter defines Young's modulus during an elastic phase.

2/ - The points (sig,eps) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.

3/ - The slope of the yield curve may not increase from one segment to the following one.

Outputs:

The components of the ECR table are as follows:

ECR(1): current hydrostatic pressure

ECR(2): current equivalent stress (Von Mises)

ECR(3): current equivalent plastic strain

ECR(4): total x-strain

ECR(5): total y-strain

ECR(6): total xy-strain

ECR(7): current yield stress

ECR(8): total z-strain

ECR(9): total yz-strain (only in 3D)

ECR(10): total xz-strain (only in 3D)

ECR(11): sound speed

ECR(12): equivalent strain rate (Von Mises)

9.6.42 THE LUDWIK MODEL

Object:

This directive enables to choose the Ludwik model, a purely elasto-plastic model implemented at Ispra. Elasto-plasticity is implemented via a radial return algorithm. Only isotropic hardening is activated to date. There is no dependency on temperature nor on strain rate.

References

The implementation of this material model is described in reference [\[167\]](#).

Syntax:

```
"VMLU"  "RO" rho  "YOUN" young  "NU" nu  "ELAS" sige ...  
        "COA2" coa2 "COEN" coen  
... /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

sige

Elastic limit.

coa2

Plastic threshold value.

coen

Hardening coefficient.

LECTURE

List of the elements concerned.

Comments:

- 1/ - The young parameter defines Young's modulus during an elastic phase.
- 2/ - The points (sig,eps) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.
- 3/ - The slope of the yield curve may not increase from one segment to the following one.

Outputs:

The components of the ECR table are as follows:

- ECR(1): current hydrostatic pressure
- ECR(2): current equivalent stress (Von Mises)
- ECR(3): current equivalent plastic strain
- ECR(4): total x-strain
- ECR(5): total y-strain
- ECR(6): total xy-strain
- ECR(7): current yield stress
- ECR(8): total z-strain
- ECR(9): total yz-strain (only in 3D)
- ECR(10): total xz-strain (only in 3D)
- ECR(11): sound speed

9.6.43 THE ZERILLI-ARMSTRONG MODEL

Object:

This directive enables to choose the Zerilli-Armstrong model with the implementation developed at Ispra. Elasto-plasticity is implemented via a radial return algorithm. Only isotropic hardening is activated to date and strain-rate dependency is included. However, no dependency on temperature exist in the present version of the model.

References

The implementation of this material model is described in reference [\[167\]](#).

Syntax:

```
"VMZA"  "RO" rho  "YOUN" young  "NU" nu  "COA1" coa1 ...  
        "COA2" coa2 "COA3" coa3 "COA4" coa4 "CLB1" clb1  
        "CLB2" clb2 "CLB3" clb3  
... /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

coa1

1st coefficient of the Zerilli-Armstrong model.

coa2

2nd coefficient of the Zerilli-Armstrong model.

coa3

3rd coefficient of the Zerilli-Armstrong model.

coa4

4th coefficient of the Zerilli-Armstrong model.

c1b1

1st hardening coefficient of the Zerilli-Armstrong model.

c1b2

2nd hardening coefficient of the Zerilli-Armstrong model.

c1b3

3rd hardening coefficient of the Zerilli-Armstrong model.

LECTURE

List of the elements concerned.

Comments:

1/ - The young parameter defines Young's modulus during an elastic phase.

2/ - The points (sig,eps) may have any position; however, concerning the first point, there must be a compatibility between the coordinates, Young's modulus and the elastic limit.

3/ - The slope of the yield curve may not increase from one segment to the following one.

Outputs:

The components of the ECR table are as follows:

ECR(1): current hydrostatic pressure

ECR(2): current equivalent stress (Von Mises)

ECR(3): current equivalent plastic strain

ECR(4): total x-strain

ECR(5): total y-strain

ECR(6): total xy-strain

ECR(7): current yield stress

ECR(8): total z-strain

ECR(9): total yz-strain (only in 3D)

ECR(10): total xz-strain (only in 3D)

ECR(11): sound speed

ECR(12): equivalent strain rate (Von Mises)

9.6.44 DRUCKER-PRAGER (MOHR-COULOMB) (JRC)**Object:**

This directive enables to specify a Drucker-Prager (Mohr-Coulomb) material. The material behaves in a linear elastic way until failure is reached, and thereafter it behaves like a fluid (i.e. it resists only to compression). Failure occurs when the stress point in the J_1 - $\sqrt{J'_2}$ space reaches the failure line (a straight line) of equation:

$$\sqrt{J'_2} = K - \alpha J_1$$

where $J_1 = \sigma_x + \sigma_y + \sigma_z$ is the first invariant of the stress tensor and J'_2 is the second invariant of the deviatoric stress tensor:

$$J'_2 = \frac{1}{3}(\sigma_x^2 + \sigma_y^2 + \sigma_z^2 - \sigma_x\sigma_y - \sigma_x\sigma_z - \sigma_y\sigma_z) + \tau_{xy}^2 + \tau_{xz}^2 + \tau_{yz}^2$$

The constant K is the intersection of the failure line with the vertical axis and represents the failure stress of the material in pure shear (e.g. in torsion): it is also called cohesion.

The constant α is the slope of the failure line (tangent of the angle) and is also called the internal friction angle.

After failure is reached, the material behaves like a liquid: all tangential stresses are set to zero and the normal stresses are set to equal (hydrostatic) values if the material is under compression (negative volumetric strain), or to zero if the material is under traction (positive volumetric strain).

References:

The material model is described in reference [\[13\]](#) .

Syntax:

```
"DRPR"  "RO" ro "YOUN" youn "NU" nu "COHE" cohe "FRIC" fric
        /LECTURE/
```

ro

Density.

youn

Young's modulus.

nu

Poisson's ratio.

cohe

Failure stress K in pure shear, e.g. in torsion (cohesion).

fric

Slope of the failure line in the J_1 - $\sqrt{J_2}$ diagram (internal friction angle).

/LECTURE/

Numbers of the elements concerned.

Outputs:

The different components of the ECR table are as follows:

ECR(1): current J_1 invariant ($\sigma_1 + \sigma_2 + \sigma_3$).

ECR(2): current $\sqrt{J_2}$ invariant.

ECR(3): failure flag (0=not failed, 1=failed).

ECR(4): cumulated x-strain.

ECR(5): cumulated y-strain.

ECR(6): cumulated xy-strain.

ECR(7): cumulated z-strain.

ECR(8): cumulated yz-strain.

ECR(9): cumulated xz-strain.

ECR(10): sound speed

9.6.45 ALUMINIUM FOAM

Object:

This option enables to specify an aluminium foam material and follows the Deshpande-Fleck model as implemented at NTNU, Trondheim (N).

References:

More information on the formulation of this material model may be found in the following references:

1. V.S. Deshpande and N.A. Fleck, Isotropic models for metallic foams, J. Mech. Phys. Solids 48 (2000), pp. 1253–1283.
2. A. Reyes, O. S. Hopperstad, T. Berstad, A. G. Hansen, M. Langseth, Constitutive modeling of aluminum foam including fracture and statistical variation of density, European Journal of Mechanics – A/Solids, Vol 22, pp 815–835, 2003.

Syntax:

```
FOAM  RO_F ro_f  YOUN youn  NU   nu    SIGP sigp  RO_0 ro_0
      ALFA alfa  GAMM gamm  ALF2 alf2  BETA beta <DERF derf>
      /LECTURE/
```

ro_f

Initial density of the foam material, i.e. considering the voids.

youn

Young's modulus (initial).

nu

Poisson's coefficient (initial).

sigp

Material constant.

ro_0

Initial density of the material, not considering the voids (metal).

alfa

Material constant.

gamm

Material constant.

alf2

Material constant.

beta

Material constant.

derf

Switch to choose the derivation of Fi in the model: 0 means numerical derivation, while 1 means normal derivation. The default is 1.

/LECT/

List of the concerned elements.

Outputs:

The components of the ECR table are as follows:

- ECR(1): Equivalent plastic strain (ϵ_{eq})
- ECR(2): Von Mises effective plastic strain (ϵ_e)
- ECR(3): Volumetric strain (ϵ_m)
- ECR(4): Equivalent stress (σ_{eq})
- ECR(5): Von Mises effective stress (σ_e)
- ECR(6): Mean stress (σ_m)
- ECR(7): Isotropic hardening variable (R)
- ECR(8): Iteration counter
- ECR(9): (Y)
- ECR(10): sound speed

9.6.46 GLRC: REINFORCED CONCRETE FOR SHELLS

Object:

This material is designed to model reinforced concrete shells, possibly with prestressing and steel liner. It consists in a resultant variables constitutive law, using both plasticity (double JOHANSEN's criterion with a kinematic softening) and damage (to take into account concrete cracking). For information about this model see references [742], [743].

New syntax for non-linear GLRC material (elastoplastic with or without damage):

```
"GLRC" < "DAMA" > < "SHEA" >
  "RO"    rho
  "H"     thickness
  "EB"    yconcrete "NUB" pconcrete
  "NLIT"  nlayer * ( |[ "NAPP" ("EA" ysteel < "FY" tsteel >
                        "OMX" ax      "OMY" ay
                        "RX"  rx      "RY"  ry      ) ;
                    "PREC" ( "EA" ysteel < "FY" tsteel >
                        "OMX" ax      "OMY" ay
                        "RX"  rx      "RY"  ry      ) ;
                    "LINR" ( "EA" ysteel < "FY" tsteel >
                        "OMLR" epliner "NULR" nuliner
                        "RLR"  rliner      ) ]| )
  < "OMT" atrast      "EAT" ytrast >
  < "BT1" shear1      "BT2" shear2 >

  < "BTD1" sheard1      "BTD2" sheard2 >
  < "TSD"  tsheard >

  < "FT"  tconcrete < "GAMM" gamma >
  "QP1"  qslope1    "QP2"  qslope2 >

  "C1N1" pragemb1x "C1N2" pragemb1y "C1N3" pragemb1xy
  "C2N1" pragemb2x "C2N2" pragemb2y "C2N3" pragemb2xy
  "C1M1" pragbend1x "C1M2" pragbend1y "C1M3" pragbend1xy
  "C2M1" pragbend2x "C2M2" pragbend2y "C2M3" pragbend2xy

  $[  "FC"    cconcrete      ;
    (  "MP1X" < "FONC" > plaslim1x
      "MP1Y" < "FONC" > plaslim1y
      "MP2X" < "FONC" > plaslim2x
      "MP2Y" < "FONC" > plaslim2y
```

```

< "D1X"      "FONC"      dplaslim1x >
< "D1Y"      "FONC"      dplaslim1y >
< "D2X"      "FONC"      dplaslim2x >
< "D2Y"      "FONC"      dplaslim2y >
< "DD1X"     "FONC"      ddplaslim1x >
< "DD1Y"     "FONC"      ddplaslim1y >
< "DD2X"     "FONC"      ddplaslim2x >
< "DD2Y"     "FONC"      ddplaslim2y > ) ]$

```

```

< "PREX" nprecx  "PREY" nprecy >

```

```

< "KRAY" kray    "MRAY" mray    >

```

```

/LECTURE/

```

rho

Density of the plate material (concrete and steel).

thickness

Thickness of the concrete (thickness of the plate).

yconcrete

Young's modulus of the concrete material.

ypoisson

Poisson's ratio of the concrete material.

NAPP

Keyword for the description of bending steel reinforcement (steel grid).

PREC

Keyword for the description of prestressing.

LINR

Keyword for the description of the steel liner.

ysteel

Young's modulus of the steel material.

tsteel

Yield stress of the steel. Used to calculate automatically the generalized Johansen criterion (when the **plaslim** functions are not specified).

ax, ay

Areas (per meter of plate) of the reinforcement layer in the x and y directions (m^2/m).

rx, ry

Nondimensional position of the layer in the x and y directions ($-1 \leq r \leq 1$).

epliner

Thickness of the liner.

nuliner

Poisson's ratio of the liner steel.

rliner

Nondimensional position of the liner ($-1 \leq r_{liner} \leq 1$).

shear

Coefficients of the elastic shear matrix (for elements that take into account the tranverse shear like Q4GR or Q4GS):

$$\begin{bmatrix} T_x \\ T_y \end{bmatrix} = \begin{bmatrix} \text{shear1} & 0 \\ 0 & \text{shear2} \end{bmatrix} \begin{bmatrix} \gamma_x \\ \gamma_y \end{bmatrix}$$

When the shear coefficients are not specified, they are calculated, for Q4GR and Q4GS elements, using the following expression:

$$T = k \frac{h}{2} \left(\frac{E_b}{1 + \nu_b} + E_{aT} \omega_T \right) \gamma$$

with:

- h : shell thickness
- k : shear correcting coefficient usually set as 5/6 (Reissner theory)
- E_b : Young's modulus of concrete (**yconcrete**)
- ν_b : Poisson's ratio of concrete (**ypoisson**)
- E_{aT} : Young's modulus of transverse steel (**ytrast**)
- ω_T : Area of transverse steel (**atrast**)

If the keyword SHEA is specified then a nonlinear evolution of the shear force is taken into account ([756]). This nonlinear evolution can be compared to an elastoplastic constitutive law. Beyond a shear force defined by TSD, the shear force evolves according to a linear slope whose stiffness is defined through the keywords BTD1 and BTD2 and plate elements are then subjected to irreversible deformations.

When the values of the damaged shear coefficients **sheard1** and **sheard2** are not specified, they are calculated using the following expression:

$$\text{sheard1} = \text{sheard2} = k \frac{h}{2} (E_{aT} \omega_T)$$

atrast

Area (per square meter of plate) of the transverse reinforcement (m^2/m^2). Used to calculate the elastic shear coefficients when **shear** are not specified.

ytrast

Young's modulus of the steel material for transverse reinforcement. Used for the computation of the elastic shear coefficients when **shear** are not specified. Default value is the standard **ysteel** value.

tconcrete

Tensile strength of concrete (tensile stress). Must be positive. Used to calculate the bending cracking moment. Used only for damage.

qslope1 qslope2

Slopes quotient for positive and negative bending. The quotient is supposed to be the slope of the (curvature,moment) graph after cracking over the slope before cracking. Used only for damage.

$$Q_p = \frac{p_{eac}}{p_{ebc}}$$

with:

- Q_p : slope quotient **qslope** ($0 < Q_p \leq 1$)
- p_{ebc} : slope before cracking (elastic concrete)
- p_{eac} : slope after cracking (cracked concrete)

gamma

Damage computation parameter which characterizes the slope of the (curvature,moment) graph during cracking. **gamma** can be considered as the slope during cracking over the slope before cracking. If **gamma** > 0, the slope increases. If **gamma** < 0, the slope decreases and the stability is not warranted. In any case, we must have **gamma** < **qslope1** and **gamma** < **qslope2**. Default value is zero. Used only for damage.

$$\gamma = \frac{p_{edc}}{p_{ebc}}$$

with:

- γ : **gamma**
- p_{ebc} : slope before cracking (elastic concrete)
- p_{edc} : slope during cracking

pragmemb, pragmbend

Prager coefficients corresponding to the matrices linking the plastic strain and curvature to the back membrane force and the backmoment.

$$n = CN_1\epsilon_1^p + CN_2\epsilon_2^p$$

$$m = CM_1\kappa_1^p + CM_2\kappa_2^p$$

with:

$$\bullet \quad CN_1 = \begin{bmatrix} \text{pragmemb1x} & 0 & 0 \\ 0 & \text{pragmemb1y} & 0 \\ 0 & 0 & \text{pragmemb1xy} \end{bmatrix}$$

$$\bullet \quad CN_2 = \begin{bmatrix} \text{pragmemb2x} & 0 & 0 \\ 0 & \text{pragmemb2y} & 0 \\ 0 & 0 & \text{pragmemb2xy} \end{bmatrix}$$

$$\bullet \quad CM_1 = \begin{bmatrix} \text{pragbend1x} & 0 & 0 \\ 0 & \text{pragbend1y} & 0 \\ 0 & 0 & \text{pragbend1xy} \end{bmatrix}$$

$$\bullet \quad CM_2 = \begin{bmatrix} \text{pragbend2x} & 0 & 0 \\ 0 & \text{pragbend2y} & 0 \\ 0 & 0 & \text{pragbend2xy} \end{bmatrix}$$

- ϵ_1^p and κ_1^p : plastic strain and curvature linked to the first criterion (`plaslim1`)
- ϵ_2^p and κ_2^p : plastic strain and curvature linked to the second criterion (`plaslim2`)

$$C = \frac{p_e p_p}{p_e - p_p}$$

with:

- C : Prager coefficient
- p_e : elastic slope (or slope after cracking, in case of bending)
- p_p : plastic slope

`cconcrete`

Compressive strength of concrete. Used to calculate automatically the generalized Johansen criterion (when the `plaslim` functions are not specified).

`plaslim`

Functions used in the generalized Johansen criterion. They describe the "beam" plastic limit moment depending on the membrane force. When they are not specified, they are automatically calculated and interpolated.

`plaslim1x plaslim2x`

Positive and negative plastic limit moments for a perfect bending in the x-direction (referring to the orthotropic axes of the shell element). If the directive "FONC" is used, `plaslim1x` or `plaslim2x` are integers referring to a function number (function of N_x). We should have usually `plaslim1x` > `plaslim2x`.

`plaslim1y plaslim2y`

Positive and negative plastic limit moments for a perfect bending in the y-direction (referring to the orthotropic axes of the shell element). If the directive "FONC" is used, `plaslim1y` or `plaslim2y` are integers referring to a function number (function of N_y). We should have usually `plaslim1y` > `plaslim2y`.

`dplaslim1x dplaslim2x dplaslim1y dplaslim2y`

Function number of the first derivative of `plaslim1x`, `plaslim2x`, `plaslim1y` and `plaslim2y` plastic limit functions. They are used when the membrane plasticity is taken into account and when they cannot be computed directly from the `plaslim1x`, `plaslim2x`, `plaslim1y` and `plaslim2y` functions.

`ddplaslim1x ddplaslim2x ddplaslim1y ddplaslim2y`

Function number of the second derivative of `plaslim1x`, `plaslim2x`, `plaslim1y` and `plaslim2y` plastic limit functions. They are used when the membrane plasticity is taken into account and when they cannot be computed directly from the `plaslim1x`, `plaslim2x`, `plaslim1y`, `plaslim2y` or `dplaslim1x`, `dplaslim2x`, `dplaslim1y`, `dplaslim2y` functions.

`nprecx, nprecy`

Prestressing force in the x and y directions (should be negative since it is normally a compression force).

`kray, mray`

Rayleigh's stiffness and mass proportional damping coefficients, used only by finite elements of the following types: DKT3, T3GS, Q4GS. Default values: `kray`=0, `mrays`=0. For information about Rayleigh's damping see reference [738].

LECTURE

List of the elements concerned.

Syntax for perforation analysis (always used with the new syntax) :

```
"GLRC" < "DAMA" > "PERF" < "SHEA" >
  "RO"   rho
  "H"    thickness
  "EB"   yconcrete  "NUB"  pconcrete
  "NLIT" nlayer * ( |["NAPP" ( "EA" ysteel < "FY" tsteel >
                      "FS"  tsteelp
```



```

                                "OMX" ax      "OMY" ay
                                "RX"  rx      "RY"  ry      ) ;
"PREC" ( "EA" ysteel < "FY" tsteel >
        "FS" tsteelp
        "OMX" ax      "OMY" ay
        "RX"  rx      "RY"  ry      ) ;
"LINR" ( "EA" ysteel < "FY" tsteel >
        "FS" tsteelp
        "OMLR" epliner "NULR" nuliner
        "RLR"  rliner      ) ] ] )

"OMT" atrast < "EAT" ytrast > "FST" tsteelp_t
< "BT1" shear1 "BT2" shear2 >

< "BTD1" sheard1 "BTD2" sheard2 >
< "TSD" tsheard >

< "FT" tconcrete < "GAMM" gamma >
  "QP1" qslope1 "QP2" qslope2 >

"FC" cconcrete "PHI" friction < "NUFC" eff_factor >
< "NPER" nper >

"C1N1" pragemb1x "C1N2" pragemb1y "C1N3" pragemb1xy
"C2N1" pragemb2x "C2N2" pragemb2y "C2N3" pragemb2xy
"C1M1" pragbend1x "C1M2" pragbend1y "C1M3" pragbend1xy
"C2M1" pragbend2x "C2M2" pragbend2y "C2M3" pragbend2xy

< "MP1X" < "FONC" > plaslim1x
  "MP1Y" < "FONC" > plaslim1y
  "MP2X" < "FONC" > plaslim2x
  "MP2Y" < "FONC" > plaslim2y
< "D1X" "FONC" dplaslim1x >
< "D1Y" "FONC" dplaslim1y >
< "D2X" "FONC" dplaslim2x >
< "D2Y" "FONC" dplaslim2y >
< "DD1X" "FONC" ddplaslim1x >
< "DD1Y" "FONC" ddplaslim1y >
< "DD2X" "FONC" ddplaslim2x >
< "DD2Y" "FONC" ddplaslim2y > >

< "PREX" nprecx "PREY" nprecy >

< "KRAY" kray "MRAY" mray >

/LECTURE/

```

cconcrete

Compressive strength of concrete. Used to calculate automatically the generalized Johansen criterion (when the **plaslim** functions are not specified). Mandatory for perforation analysis.

friction

Friction angle of concrete (degrees). Mandatory for perforation analysis.

tsteelp, tsteelp_t

Limit stress of steel (for each layer and for transverse reinforcement). Mandatory for perforation analysis.

eff_factor

Effectiveness factor for concrete. When not specified, a default value is taken.

nper

Frequency of verification of the perforation criterion. Default value is 1 (every time step). For information about the perforation criterion see references [740], [742].

Old syntax for the standard material (without damage):

```
"GLRC" "OLD"
  "RO" rho "BN11" memb11      "BN12" memb12
        "BN22" memb22      "BN33" memb33
        "BM11" bend11      "BM12" bend12
        "BM22" bend22      "BM33" bend22
        < "BC11" coup11 > < "BC12" coup12 >
        < "BC22" coup22 > < "BC33" coup22 >
        < "BT1"  shear1 > < "BT2"  shear2 >
        < "C1N1" prgmemb1x "C1N2" prgmemb1y "C1N3" prgmemb1xy >
        < "C2N1" prgmemb2x "C2N2" prgmemb2y "C2N3" prgmemb2xy >
        "C1M1" prgbend1x "C1M2" prgbend1y "C1M3" prgbend1xy
        "C2M1" prgbend2x "C2M2" prgbend2y "C2M3" prgbend2xy
        "MP1X" < "FONC" > plaslim1x
        "MP1Y" < "FONC" > plaslim1y
        "MP2X" < "FONC" > plaslim2x
        "MP2Y" < "FONC" > plaslim2y
        < "D1X"      "FONC"      dplaslim1x >
        < "D1Y"      "FONC"      dplaslim1y >
        < "D2X"      "FONC"      dplaslim2x >
        < "D2Y"      "FONC"      dplaslim2y >
        < "DD1X"     "FONC"      ddplaslim1x >
```

```

< "DD1Y"      "FONC"      ddplaslim1y >
< "DD2X"      "FONC"      ddplaslim2x >
< "DD2Y"      "FONC"      ddplaslim2y >
/LECTURE/

```

rho

Density of the material.

memb, bend, coup

Coefficients of the elastic matrix:

$$\begin{bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \\ M_{xx} \\ M_{yy} \\ M_{xy} \end{bmatrix} = \begin{bmatrix} \text{memb11} & \text{memb12} & 0 & \text{coup11} & \text{coup12} & 0 \\ \text{memb12} & \text{memb22} & 0 & \text{coup12} & \text{coup22} & 0 \\ 0 & 0 & \text{memb33} & 0 & 0 & \text{coup33} \\ \text{coup11} & \text{coup12} & 0 & \text{bend11} & \text{bend12} & 0 \\ \text{coup12} & \text{coup22} & 0 & \text{bend12} & \text{bend22} & 0 \\ 0 & 0 & \text{coup33} & 0 & 0 & \text{bend33} \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ 2\epsilon_{xy} \\ \kappa_{xx} - \kappa_{xx}^p \\ \kappa_{yy} - \kappa_{yy}^p \\ 2(\kappa_{xy} - \kappa_{xy}^p) \end{bmatrix}$$

When the coupling coefficients are not specified, they take the zero value.

shear

Coefficients of the elastic shear matrix (for elements that take into account the tranverse shear like Q4GR or Q4GS):

$$\begin{bmatrix} T_x \\ T_y \end{bmatrix} = \begin{bmatrix} \text{shear1} & 0 \\ 0 & \text{shear2} \end{bmatrix} \begin{bmatrix} \gamma_x \\ \gamma_y \end{bmatrix}$$

When the shear coefficients are not specified, they take the zero value. Classical assumptions in elasticity give the following expression:

$$T = h \frac{kE}{2(1 + \nu)} \gamma$$

with:

- h : shell thickness
- k : shear correcting coefficient usually set as 5/6 (Reissner theory)
- E : Young's modulus
- ν : Poisson's ratio

pragmemb, pragmbend

Prager coefficients corresponding to the matrices linking the plastic strain and curvature to the back membrane force and the backmoment.

$$n = CN_1 \epsilon_1^p + CN_2 \epsilon_2^p$$

$$m = CM_1 \kappa_1^p + CM_2 \kappa_2^p$$

with:

$$\begin{aligned}
\bullet \quad CN_1 &= \begin{bmatrix} \text{pragmemb1x} & 0 & 0 \\ 0 & \text{pragmemb1y} & 0 \\ 0 & 0 & \text{pragmemb1xy} \end{bmatrix} \\
\bullet \quad CN_2 &= \begin{bmatrix} \text{pragmemb2x} & 0 & 0 \\ 0 & \text{pragmemb2y} & 0 \\ 0 & 0 & \text{pragmemb2xy} \end{bmatrix} \\
\bullet \quad CM_1 &= \begin{bmatrix} \text{pragbend1x} & 0 & 0 \\ 0 & \text{pragbend1y} & 0 \\ 0 & 0 & \text{pragbend1xy} \end{bmatrix} \\
\bullet \quad CM_2 &= \begin{bmatrix} \text{pragbend2x} & 0 & 0 \\ 0 & \text{pragbend2y} & 0 \\ 0 & 0 & \text{pragbend2xy} \end{bmatrix} \\
\bullet \quad \epsilon_1^p \text{ and } \kappa_1^p &: \text{ plastic strain and curvature linked to the first criterion (plaslim1)} \\
\bullet \quad \epsilon_2^p \text{ and } \kappa_2^p &: \text{ plastic strain and curvature linked to the second criterion (plaslim2)}
\end{aligned}$$

The membrane Prager coefficients are not mandatory. If they are not specified by the user, the model takes into account only bending plasticity. Thus it has a non-normal plasticity flow if the plastic limits vary with the membrane force. This could lead to convergence problems. But if the membrane Prager coefficients are given, both membrane and bending plasticity are taken into account. The model is in fact regularized compared to the preceding one.

plaslim1x plaslim2x

Positive and negative plastic limit moments for a perfect bending in the x-direction (referring to the orthotropic axes of the shell element). If the directive "FONC" is used, **plaslim1x** or **plaslim2x** are integers referring to a function number (function of N_x). We should have usually **plaslim1x** > **plaslim2x**.

plaslim1y plaslim2y

Positive and negative plastic limit moments for a perfect bending in the y-direction (referring to the orthotropic axes of the shell element). If the directive "FONC" is used, **plaslim1y** or **plaslim2y** are integers referring to a function number (function of N_y). We should have usually **plaslim1y** > **plaslim2y**.

dplaslim1x dplaslim2x dplaslim1y dplaslim2y

Function number of the first derivative of **plaslim1x**, **plaslim2x**, **plaslim1y** and **plaslim2y** plastic limit functions. They are used when the membrane plasticity is taken into account and when they cannot be computed directly from the **plaslim1x**, **plaslim2x**, **plaslim1y** and **plaslim2y** functions.

ddplaslim1x ddplaslim2x ddplaslim1y ddplaslim2y

Function number of the second derivative of **plaslim1x**, **plaslim2x**, **plaslim1y** and **plaslim2y** plastic limit functions. They are used when the membrane plasticity is taken

into account and when they cannot be computed directly from the `plaslim1x`, `plaslim2x`, `plaslim1y`, `plaslim2y` or `dplaslim1x`, `dplaslim2x`, `dplaslim1y`, `dplaslim2y` functions.

LECTURE

List of the elements concerned.

Old syntax for the material with damage (for cracking):

```
"GLRC" "OLD" "DAMA"
      "RO" rho "BN11" memb11      "BN12" memb12
            "BN22" memb22      "BN33" memb33
            "E"  young      "NU"  poisson
            "MF1" cracklim1 "MF2" cracklim2
            "QP1" qslope1   "QP2" qslope2
            "GAMM" gamma
      < "BT1" shear1 > < "BT2" shear2 >
      < "C1N1" pragemb1x "C1N2" pragemb1y "C1N3" pragemb1xy >
      < "C2N1" pragemb2x "C2N2" pragemb2y "C2N3" pragemb2xy >
      "C1M1" pragbend1x "C1M2" pragbend1y "C1M3" pragbend1xy
      "C2M1" pragbend2x "C2M2" pragbend2y "C2M3" pragbend2xy
      "MP1X" < "FONC" > plaslim1x
      "MP1Y" < "FONC" > plaslim1y
      "MP2X" < "FONC" > plaslim2x
      "MP2Y" < "FONC" > plaslim2y
      < "D1X"      "FONC"      dplaslim1x >
      < "D1Y"      "FONC"      dplaslim1y >
      < "D2X"      "FONC"      dplaslim2x >
      < "D2Y"      "FONC"      dplaslim2y >
      < "DD1X"     "FONC"      ddplaslim1x >
      < "DD1Y"     "FONC"      ddplaslim1y >
      < "DD2X"     "FONC"      ddplaslim2x >
      < "DD2Y"     "FONC"      ddplaslim2y >
      /LECTURE/
```

DAMA

Enable the option which allows to take in account the concrete cracking by damage.

`rho`, `shear`, `pragemb`, `pragbend`, `plaslim`, `dplaslim`, `ddplaslim`

Same parameters as those described for the standard GLRC material.

`memb`

Coefficients of the elastic matrix:

$$\begin{bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \end{bmatrix} = \begin{bmatrix} \text{memb11} & \text{memb12} & 0 \\ \text{memb12} & \text{memb22} & 0 \\ 0 & 0 & \text{memb33} \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ 2\epsilon_{xy} \end{bmatrix}$$

There is no elastic coupling between the bending and the membrane behavior.

`young, poisson`

Homogenized elastic characteristics (Young's modulus and Poisson's ratio) for bending.

`cracklim1 cracklim2`

Positive and negative cracking limit moments.

`qslope1 qslope2`

Slopes quotient for positive and negative bending. The quotient is supposed to be the slope of the (curvature,moment) graph after cracking over the slope before cracking.

Comments:

All the limit plastic moments must be defined carefully. When they are declared as functions (using "FONC"), the domain defined as `plaslim1-plaslim2 > 0` must be a close convex domain: note particularly that the program tries to find two intersections of `plaslim1` and `plaslim2`.

When the limit plastic functions are not defined as polynomial (e.g. when "LSQU" is not used), the program requires prolongation of the functions: it is necessary to compute the elastic predictor which can be located outside the close convex elastic domain.

The first and second derivative of the limit plastic functions can be surely computed from the original limit plastic functions (i.e. without using the functions associated with the "D1", "D2", "DD1" and "DD2" directives) when these limit plastic functions are polynomials (see "LSQU" 11.1 to use table functions as polynomials).

After (and never before) the definition of the material characteristics ("MATE" directive), the orthotropy characteristics of the elements are mandatory. The syntax is:

```
"COMP"  "ORTS"  vx vy vz  /LECTURE/
```

See the "ORTS" directive for more details.

Outputs:

The components of the ECR table are as follows:

Plastic strain and curvature (ϵ^p and κ^p) in the orthotropic axes:

ECR(1): ϵ_x^p

ECR(2): ϵ_y^p

ECR(3): $2 \times \epsilon_{xy}^p$

ECR(4): κ_x^p

ECR(5): κ_y^p

ECR(6): $2 \times \kappa_{xy}^p$

Energy dissipated during plasticity:

ECR(7): plastic dissipation per Gauss point. The sum of ECR(7) on all Gauss points of the element gives the plastic dissipation in the element.

Damage parameters:

ECR(8): D_1/D_{1max} for positive bending

ECR(9): D_2/D_{2max} for negative bending

Energy dissipated during damage:

ECR(10): damage dissipation per Gauss point. The sum of ECR(10) on all Gauss points of the element gives the damage dissipation in the element.

Orthotropy characteristics:

- Components, in the global reference frame, of the vector whose projection on the local coordinate system of the 3D shell element indicates the orthotropy direction (data following the "ORTS" directive).

ECR(11): v_x

ECR(12): v_y

ECR(13): v_z

- After the first time step, the orthotropy characteristics are:

ECR(11): angle defining the orthotropic axes referring to the local axes in the shell element plane.

ECR(12): 10.

ECR(13): 10.

Membrane force and moment minus back force and backmoment ($N - n$ and $M - m$) in the orthotropic axes:

ECR(14): $N_x - n_x$

ECR(15): $N_y - n_y$

ECR(16): $N_{xy} - n_{xy}$

ECR(17): $M_x - m_x$

ECR(18): $M_y - m_y$

ECR(19): $M_{xy} - m_{xy}$

Post-treatment parameters for the perforation criterion:

ECR(20): = 0 if the criterion is not reached

= 1 if the criterion is reached in bending mode

= 2 if the criterion is reached in shear mode

ECR(21): normalized value of the perforation criterion (> 0 if the criterion is reached)

ECR(22): nx (components of the vector which is

ECR(23): ny normal to the failure plan,

ECR(24): nz in the global reference frame)

9.6.47 HYPERELASTIC MATERIAL

Object:

This sub-directive enables materials with an hyperelastic behaviour to be used. Only two types of shell (Q4GS et DST3) and several solid elements (CUBE, TETR, etc.) can be used with this material. Three kinds of hyperelastic materials can be selected :

- "TYPE" 1: Mooney Rivlin Material;
- "TYPE" 2: Hart Smith Material;
- "TYPE" 3: Ogden Material

For type 1, the expression of the strain energy density corresponds to :

$$\begin{aligned}
 W = & c1*(I1-3) + c2*(I2-3) + c3*(I1-3)**2 + \\
 & c4*(I1-3)*(I2-3) + c5*(I2-3)**2 + \\
 & c6*(I1-3)**3 + c7*(I2-3)*(I1-3)**2 + \\
 & c8*(I1-3)*(I2-3)**2 + c9*(I2-3)**3 + \\
 & c10*(I1-3)**4 + c11*(I2-3)**2*(I1-3)**3 + \\
 & c12*(I1-3)**2*(I2-3) + c13*(I1-3)*(I2-3) + \\
 & K*(Log(I3))**2
 \end{aligned}$$

For type 2, the expression of the strain energy density corresponds to :

$$W = A*\text{Integrale}(C*(I1-3)**2)dI1 + 3*B*\text{Log}I2 + K*\text{Log}(I3)**2$$

Type 3, the ogden material can be expressed with the following equation

$$W = \sum_{p=1}^N \frac{\mu_p}{\alpha_p} (\lambda_1^{\alpha_p} + \lambda_2^{\alpha_p} + \lambda_3^{\alpha_p} - 3) \quad (4)$$

with $\mu_p = 2 \cdot m u_i / \alpha_i$.

Syntax:

Case 1 : TYPE = 1

```

"HYPE"
"TYPE"      1
"RO"        rho
"CO1"       c1

```

```
      .      .  
      .      .  
    "C014"    c14  
    "BULK"    K  
  /LECTURE/
```

rho

Density.

C01

First coefficient of the potential

C014

14st coefficient of the potential

K

Compressibility coefficient

LECTURE

List of the concerned elements.

Case 2 : TYPE = 2

```
    "HYPE"  
    "TYPE"      2  
    "RO"        rho  
    "C01"        c1  
    "C02"        c2  
    "C03"        c3  
    "BULK"      K  
  /LECTURE/
```

rho

Density.

C01

First coefficient of the potential (=A)

C02

Second coefficient of the potential (=B)

C03

Third coefficient of the potential (=C)

K

Compressibility coefficient

LECTURE

List of the concerned elements.

Case 3 : TYPE = 3

```
"HYPE"  
"TYPE"      3  
"R0"        rho  
"C01"       c1  
"C02"       c2  
"C03"       c3  
"C04"       c4  
"C05"       c5  
"C06"       c6  
"C07"       c7  
"C08"       c8  
"C09"       c9  
"C010"      c10  
"C011"      c11  
"C012"      c12  
"BULK"      K  
/LECTURE/
```

rho

Density.

C01,C02,C03,C04

Alpha coefficients of the potential (α_i)

C05,C06,C07,C08

Mu coefficients of the potential (μ_i)

C09,C010,C011,C012

(1/D) coefficient of the potential

LECTURE

List of the concerned elements.

Outputs:

The components of the ECR table are as follows:

ECR(1): Pressure

ECR(2): Von Mises Stress

ECR(3): Normal transverse strain

ECR(4): Updated thickness

ECR(5): Initial thickness

ECR(6): Energy potential

ECR(7): Time step for the element

9.6.48 MATERIAL FOR INTERFACE ELEMENT

Object:

This directive allows to choose the material for interface element typically used for cohesive zone models. This material can only be used with interface elements (type 124, 125, 126). Only TYPE=2 material is implemented.

Three damage laws could be chosen with material TYPE 2: exponential, linear or Cachan meso-model.

- To select the exponential law, parameters CO1 to CO7 are required.
- To select the linear law, parameters CO1 to CO8 are required.
- To select the Cachan meso-model law, parameters CO1 to CO9 (CO8 is optional) are required.
- Parameters CO10 to CO12 are optional in any case.

References:

- For the Cachan damage meso-model:
D. Lévêque, Analyse de la tenue au délaminage des composites stratifiés : identification d'un modèle d'interface interlaminaire. Thèse de doctorat. LMT-Cachan, 1998.
- For the TYPE=2 material: [\[730\]](#)

Syntax:

```
"MINT" "TYPE" 2
      "CO1" co1 "CO2" co2 "CO3" co3
      "CO4" co4 "CO5" co5 "CO6" co6
      "CO7" co7 "CO8" co8 "CO9" co9
      "CO10" co10 "CO11" co11 "CO12" co12
      /LECTURE/
```

co1

Young's modulus along direction 3.

co2

Shear modulus between direction 1 and 3.

co3

Shear modulus between direction 2 and 3.

co4

Critical release rate in mode 1.

co5

Critical release rate in mode 2.

co6

Critical release rate in mode 3.

co7

Power coefficient to couple the thermodynamic forces of the three modes.

co8

Thermodynamical force threshold for damage. Required for the linear damage law. Optional for the Cachan meso-model.

co9

Exponent n for the Cachan meso-model.

co10

Delay effect : parameter tau (optional).

co11

Delay effect : parameter a (optional).

co12

Maximum damage (optional, default value = 1.0).

LECTURE

List of the concerned elements.

Comments:

When damage reaches the maximum damage value co12, element rigidity becomes null. Erosion algorithm is activated with EROS keyword (see page A.30, Section 6.4).

Outputs:

The components of the ECR table are as follows:

ecr(1): Damage

ecr(2): Equivalent thermodynamic force

ecr(3): Time at which damage reaches 1.0 for failed elements

9.6.49 THE SL-ZA MODEL

Object:

This directive enables to choose the SLZA model which is an extension of both STEINBERG LUND and ZERILLI ARMSTRONG models. This model uses an expression for the internal stress that comes from the ZA model and an expression of the effective stress that comes from SL model.

Syntax:

```
"SLZA" "RO" rho    "YOUN" young  "NU" nu    "SIGE" sige
      "YA" ya     "YMAX" ymax   "YP" yp    "ER" er
      "N"  n      "C1"  c1     "UK" uk    "CP" cp
      "TM" tm     "T0"  t0     "BETA" beta
      /LECTURE/
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

sige

Elastic limit at ambient temperature.

ya

Coefficient of the CEA SL-ZA model.

ymax

Coefficient of the CEA SL-ZA model.

yp

Coefficient of the CEA SL-ZA model.

er

	Coefficient of the CEA SL-ZA model.
n	
	Coefficient of the CEA SL-ZA model.
c1	
	Coefficient of the CEA SL-ZA model.
uk	
	Coefficient of the CEA SL-ZA model.
cp	
	Heat capacity per unit mass of the solid.
tm	
	Melting temperature of the solid.
t0	
	Initial temperature of the solid.
beta	
	Taylor and Quiney coefficient.
LECTURE	
	List of the elements concerned.

Comments:

The expression of the elastic limit is given by:

$$y_d = (y_a + (y_{\max} - y_a)((1 - \exp(-e_p/e_r))^n)) + y_p(1 - \sqrt{kt/2u_k} \log(c_1/\dot{e}))$$

where k is the Boltzmann constant and \dot{e} is the strain rate.

Outputs:

The components of the ECR table are as follows:

ecr(1) : Hydrostatic pressure
ecr(2) : Von mises stress
ecr(3) : Equivalent plastic strain
ecr(4) = Increment of temperature
ecr(5) = Elastic limit
ecr(6) = Total strain at the last timestep
ecr(7) = Time of the last call of the element
ecr(8) = Equivalent strain rate

9.6.50 RTM composite material

Object:

This directive allows to chose a composite material made by a RTM process. The behavior is orthotropic and the 9 independant coefficents can defined by using abaques of 3 or 4 parameters. These parameters are the volumic fraction, the angle between warp and weft directions and the warp and weft ratio. The 4th parameter is the temperature which can be optionnal.

Syntax:

```
"CRTM"
  "RO" rho
  "NTEM" ntem "NVF" nvf "NANG" nang "NRCT" nrct
  "PTM" ptem "PVF" pvf "PANG" pang "PRCT" prct
  "E11" ne11
    PAR1 val-par1-1 PAR2 val-par2-1 PAR3 val-par3-1
    TABLE nval-par4
      nval-par4 *(E11 PAR4)
  PAR1 val-par1-1 PAR2 val-par2-1 PAR3 val-par3-2
  TABL nval-par4
    nval-par4*(E11 , PAR4)
... then loop on PARA3, then PAR2 and PARA1.

"E22" ne22
  -idem-

"E33" ne33
  -idem-

"G12" ng12
  -idem-

"G13" ng13
  -idem-

"G23" ng23
  -idem-

"NU12" nnu12
  -idem-
```

"NU13" nnu13
-idem-

"NU23" nnu23
-idem-

/LECTURE/

rho

Density of the material.

ntem

Number of values of temperature

nvf

Number of values of volumic fraction

nang

Number of values of angle between warp and weft

nrct

Number of values of ratio between warp and weft

ptem

Number of the temperature parameter

pvf

Number of the volumic fraction parameter

pang

Number of the angle parameter

prct

Number of the ratio between warp and weft parameter

ne11

Number of the abaque for E11

val-par1-1

First value of the parameter 1

val-par2-1

First value of the parameter 2

val-par3-1

First value of the parameter 3

val-par3-2

Second value of the parameter 3

nval-par4

Number of values of parameter 4

LECTURE

List of the elements concerned.

Comments:

1/ - It is possible to suppress the temperature dependant. In this case, one can use 3 parameters (from 1 to 3).

2/ - By defining ptem, prct, pang and pvf, it is possible to declare that temperature is parameter 1, volumic fraction is parameter 2 and any combination the user likes. It permits to use as general as possible an abaque of 4 parameters.

3/ - The values of angle, volumic fraction and ration between warp and weft have to be define by using the directive RTMANG, RTMVF and RTMRCT (page C63). The temperature is defined as initial values (command INIT TETA page E80).

Outputs:

The components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : Von mises criterion

ECR(3) : modulus E11

ECR(4) : modulus E22

ECR(5) : modulus E33

ECR(6) : modulus G12

ECR(7) : modulus G13

ECR(8) : modulus G23

ECR(9) : Poisson coefficient NU12

ECR(10) : Poisson coefficient NU13

ECR(11) : Poisson coefficient NU23

9.6.51 TVMC (LOI ELASTOPLASTIQUE POUR COMPOSITES)**Object:**

Ce materiau permet de modeliser le comportement elastoplastique endommageable de composites a fibres courtes.

C'est le cas par exemple des composites injectes de type thermoplastique charge de fibres (verre, carbone, ...) comme ULTEM 2100, ou encore des composites SMC-R de type polyester charge de fibres (verre, carbone).

Cette loi est utilisable pour les elements volumiques. Elle se decompose en trois etapes :

- homogeneisation micro-mecanique,
- endommagement,
- plasticite couplee a l'endommagement.

Syntax:

```
"TVMC"  "ROF"  rhof  "ROM"  rhom  "TAUX"  taux  "EM"   em   "NUM"  num ...
...  "EF"   ef   "NUF"  nuf   "R"    rap   "TVF"  tvf  "TE"   te   ...
...  "PH"   ph   "NF"   nf    "Y1C"  y1c   "Y2C"  y2c  "CRIT" choix ...
...  "NFD1" n1   "NFD2" n2   "NFR"  n3    /LECTURE/
```

rhof

Masse volumique de la fibre.

rhom

Masse volumique de la resine (chargee ou non chargee).

taux

Taux de porosite.

em

Module d'Young de la matrice.

num

Coefficient de Poisson de la matrice.

ef

Module d'Young de la fibre.

nuf

Coefficient de Poisson de la fibre.

rap

Rapport de forme de la fibre (longueur sur diametre).

tvf

Taux volumique de fibres.

te

Orientation dans le plan de la fibre (inutilise ici).

ph

Orientation hors plan de la fibre (inutilise ici).

nf

Nombre d'orientations de fibres dans le plan.

y1c

Taux de restitution limite de la matrice en traction.

y2c

Taux de restitution limite de la matrice en cisaillement.

choix

Numero du critere definissant la forme de la surface de charge.

n1

Numero de la fonction definissant l'endommagement en traction-compression en X et Y
.

n2

Numero de la fonction definissant l'endommagement en cisaillement.

n3

Numero de la fonction definissant la courbe de plasticite a ecrouissage isotrope.

LECTURE

List of the elements concerned.

Comments:

Le parametre "CRIT" peut prendre l'une des 4 valeurs suivantes :

- 1 = VON MISES,
- 2 = TRESCA,
- 3 = TSAI-HILL (en σ_1 et σ_4),
- 4 = TSAI-HILL (en σ_1 , σ_2 et σ_4),

Outputs:

The components of the ECR table are as follows:

- ECR(1): pression hydrostatique,
- ECR(2): Y1 = taux de restitution d'energie en traction,
- ECR(3): Y2 = taux de restitution d'energie en cisaillement,
- ECR(4): D1 = endommagement en traction,
- ECR(5): D2 = endommagement en cisaillement,
- ECR(6): deformation plastique cumulee,
- ECR(7) : limite elastique courante,
- ECR(8:10): inusites,
- ECR(11): vitesse du son locale (pour la stabilite).

9.6.52 THE HILL MATERIAL MODEL**Object:**

This directive enables to choose the HILL model which is a model with isotropic plasticity associated with a HILL criterion. The elastic behavior of the material can be orthotropic.

Syntax:

```
"HILL" "R0" rho    "YG1" yg1    "YG2" yg2    "YG3" yg3
          "G12" g12    "G13" g13    "G23" g23
          "NU12" nu12  "NU13" nu13  "NU23" nu23
          "XT1" xt1    "XT2" xt2    "XT3" xt3
          "RST1" rst1  "RST2" rst2  "RST3" rst3
"TRAC" npts*( sig  eps ) /LECTURE/
```

/LECTURE/

rho

Density of the material.

yg1

Young's modulus - direction 1

yg2

Young's modulus - direction 2

yg3

Young's modulus - direction 3

g12

shear modulus - plane 12

g23

shear modulus - plane 23

g13

shear modulus - plane 13

nu12

shear modulus - plane 12

nu23

shear modulus - plane 23

nu13

shear modulus - plane 13

xt1

yield stress - direction 1

xt2

yield stress - direction 2

xt3

yield stress - direction 3

rst1

yield stress - plane 12

rst2

yield stress - plane 23

rst3

yield stress - plane 13

"TRAC"

This key-word introduces the yield curve.

npts

Number of points (except the origin) defining the yield curve.

sig

normalised stress.

eps

Equivalent plastic strain. Note that the first point must be always (1., 0.)

LECTURE

List of the elements concerned.

Outputs:

The components of the ECR table are as follows:

ecr(1) : Hydrostatic pressure

ecr(2) : Von mises stress

ecr(3) : Equivalent plastic strain

ecr(7) = New elastic limit

9.6.53 GLASS MATERIAL

Object:

This option enables to choose a material that considers the strain rate effect of glass. A linear elastic material is used up to the failure. The failure limit **PSAR** uses the area under the stress-time curve (equivalent constant stress).

Syntax:

```
"GLAS"  "RO" rho  "YOUN" young  "NU" nu  "CORR" corr
        "FAIL" $[ VMIS ; PEPS ; PRES ; PEPR; PSAR ]$ "LIMI" limit
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

corr

Stress corrosion fraction. Default value is 16.

FAIL

Introduces an element failure model, represented by a failure criterion and a failure limit value. The available failure criteria are: **VMIS** for a criterion based upon Von Mises stress (isotropic criterion), **PEPS** for a criterion based upon the principal strain (see caveat below), **PRES** for a criterion based upon the hydrostatic stress, **PEPR** for a criterion based upon the principal strain if the hydrostatic stress is positive (traction): if the hydrostatic stress is negative (compression) there is no failure. **PSAR** for a criterion based upon equivalent constant stress of the duration of 60 s.

limit

Indicates the failure limit for the chosen criterion.

Comments:

When using a failure criterion based upon the principal strains (PEPS or PEPR) be aware that the criterion is based upon the *cumulated* strains. These are usually a good approximation of the total strains for elements using a convected reference frame for the stresses and strains (such as e.g. plate, shell or bar elements). The approximation is likely to be very bad, instead, for continuum-like elements, at least when there are large rotations.

Outputs:

The components of the ECR table are as follows:

ECR(1): current hydrostatic pressure

ECR(2): current equivalent stress (Von Mises)

ECR(3) Area under the (principal stress to the power of CORR)-time curve, the stress is divided by 1.E6 to avoid too big numbers.

ECR(4): total x-strain

ECR(5): total y-strain

ECR(6): total xy-strain

ECR(7): total z-strain

ECR(8): total yz-strain (only in 3D)

ECR(9): total xz-strain (only in 3D)

ECR(10): equivalent constant stress of the duration 60 s.

ECR(11): sound speed

ECR(12): failure flag (0=virgin Gauss Point, 1=failed Gauss Point)

9.6.54 BL3S: REINFORCED CONCRETE LAW FOR DEM**Object:**

This material law prescribes properties of the reinforced concrete for structures modeled with the discrete element method (DEM) via ELDI elements. Usually, both steel and concrete phases are present. Nevertheless, they may be used separately, i.e. it is possible to use only one material phase, either concrete or steel.

This model was developed in J.Rousseau's PhD thesis with EDF financial support and collaboration with 3S-R Laboratory (Grenoble). For theoretical description of the laws see [749].

Syntax:

```
"BL3S" | [ "BETON"  "RO"    rho    "YOUN" youn  "NU"    nu
              "T"      tens   "CO"    cohe  "PHII"  phii
              "PHIC"  phic   "ADOU"  adou
              < "ALPH"  alpha  "BETA"  beta  "GAMM"  gamma >
              < "BIMA"  "YOUN" youn  "NU"    nu    "TN"    tn
              "CN"    cn    "TE"    te    "TMAX"  tmax
              "UMAX"  dmax  "PHII"  phii  "PHIC"  phic >
              /LECTURE/ ;

              "ACIER"  "RO"    rho    "YOUN" youn  "NU"    nu
              "T"      tens   "ECRO"  sigmr  "AMAX"  amax
              < "BIMA"  "YOUN" youn  "NU"    nu    "TN"    tn
              "CN"    cn    "TE"    te    "TMAX"  tmax
              "UMAX"  dmax  "PHII"  phii  "PHIC"  phic >
              /LECTURE/
    ] |
```

Parameters for concrete (BETON):

rho

Density of the material

youn

Young's modulus

nu

Poisson's ratio

tens

Maximum tensile strength ($T > 0$).

cohe

Cohesion

phii

Internal friction angle

phic

Contact friction angle

adou

Softening coefficient (ratio between elastic and softening slopes >0)

alpha

1st parameter for micro-macro relations $K=f(E, \nu, \alpha, \beta, \gamma)$. The default value is 3.9 (see [755]).

beta

2nd parameter for micro-macro relations $K=f(E, \nu, \alpha, \beta, \gamma)$. The default value is 3.03125 (see [755]).

gamma

3rd parameter for micro-macro relations $K=f(E, \nu, \alpha, \beta, \gamma)$. The default value is 4.8115 (see [755]).

LECTURE

List of the elements concerned.

Parameters for steel (ACIER):

rho

Density of the material

youn

Young's modulus

nu

Poisson's ratio

tens

Maximum elastic stress ($T > 0$).

sigmr

Maximum stress for steel

amax

Maximum allongation (%)

LECTURE

List of the elements concerned.

Parameters for steel-concrete interface (BIMA):

youn

Young's modulus

nu

Poisson's ratio

tn

Maximum normal tensile strength (perpendicular to the steel bar)

cn

Maximum normal compression strength (perpendicular to the steel bar)

te

Elastic limit in the tangential direction

tmax

Maximum strenght in the tangential direction

dmax

Coefficient to define maximum tangential sliding ($u_{max}=d_{max}*u_{glis}$)

phii

Internal friction angle

phic

Contact friction angle

LECTURE

List of the elements concerned.

Comments:

Concrete and reinforcement properties being defined separately (keywords BETON and ACIER respectively), one can define also a specific behavior for the steel-concrete interface. This can be done by using a sequence of parameters introduced by the BIMA option. This option should be used only once, either with BETON or ACIER definition. If the sequence BIMA is not specified, the steel-concrete interface behaves as a concrete without taking into account the main direction of the reinforcement.

Don't forget to use directive ARMA in CELDI to declare the steel discrete elements. ARMA calculates the main direction of the reinforcement needed to define normal and tangential forces for the BIMA links.

Outputs:

In the discrete element calculation BL3S material is used for the links. However, for post-processing purpose the number of active links and the degree of damage are reported onto the discrete elements.

The components of the ECR table are as follows:

ECR(1): number of COHE-type links per element at $t = t_0$

ECR(2): number of BIMA-type links per element at $t = t_0$

ECR(3): number of COHE-type links per element at $t \geq t_0$

ECR(4): number of BIMA-type links per element at $t \geq t_0$

ECR(5): degree of damage of COHE-type links per element

ECR(6): degree of damage of BIMA-type links per element

ECR(7): diameter of the discrete element.

9.6.55 LAMINATED SECURITY GLASS MATERIAL

Object:

This option enables to choose a material that considers laminated security glass. A linear elastic material is used up to the failure. After the failure, the material can react to compression but not more to tension. This material is recommended with a sandwich structure, where the interlayer can be built up with a elastoplastic material.

Syntax:

```
"LSGL"  "RO" rho  "YOUN" young  "NU" nu  <"CORR" corr>
        <"FAIL" $[ VMIS ; PEPS ; PRES ; PEPR; PSAR ]$ "LIMI" limit>
        <"CR2D"> <"NEIG"> <"REDU" redu>
```

rho

Density of the material.

young

Young's modulus.

nu

Poisson's ratio.

corr

Stress corrosion fraction. Default value is 16. This value is only used by the failure criterion **PASR**. See following reference: Beason, W. Lynn, Morgan, James R.: Glass failure prediction model. Journal of Structural Engineering, 110 (2), pp. 197-212, 1984.

FAIL

Introduces an element failure model, represented by a failure criterion and a by failure limit value. The available failure criteria are: **VMIS** for a criterion based upon Von Mises stress (isotropic criterion), **PEPS** for a criterion based upon the principal strain (see caveat below), **PRES** for a criterion based upon the hydrostatic stress, **PEPR** for a criterion based upon the principal strain if the hydrostatic stress is positive (traction): if the hydrostatic stress is negative (compression) there is no failure. **PSAR** for a criterion based upon equivalent constant stress of the duration of 60 s.

limit

Indicates the failure limit for the chosen criterion.

CR2D

Introduces two-dimensional cracks, which means that the direction of the principle stress or strain is used to introduce a first crack. This crack is implemented in such a way that only the stresses normal to the crack direction are set to 0 (in the case of tension). If the failure criterion is reached for the direction parallel to the crack, then the integration point fails in both directions.

NEIG

If this material is used for 3D calculations, the glass part of the model should mainly eroded after the erosion of the interlayer. By using the keyword **NEIG** erosion of an element of the **LSGL** material is only taken into account, if a neighbour element (e.g. interlayer of another **LSGL** element) is already eroded.

redu

In case of hydrostatic tension, the stresses are set to 0. Using keyword **REDU** the decreasing of the stresses can be smoothed. The tension stresses are multiplied with the value **redu**, which should be less than 1.0. Default value for **redu** is 0.0.

Comments:

When using a failure criterion based upon the principal strains (**PEPS** or **PEPR**) be aware that the criterion is based upon the *cumulated* strains. These are usually a good approximation of the total strains for elements using a convected reference frame for the stresses and strains (such as e.g. plate, shell or bar elements). The approximation is likely to be very bad, instead, for continuum-like elements, at least when there are large rotations.

The material should only be used with shell elements. The third component of the stresses and strains is neglected in the calculation of the failure criterion.

Outputs:

The components of the ECR table are as follows:

ECR(1): current hydrostatic pressure

ECR(2): current equivalent stress (Von Mises)

ECR(3) Area under the (principal stress to the power of **CORR**)-time curve, the stress is dived by 1.E6 to avoid too big numbers.

ECR(4): total x-strain

ECR(5): total y-strain

ECR(6): total xy-strain

ECR(7): total z-strain

ECR(8): total yz-strain (only in 3D)

ECR(9): total xz-strain (only in 3D)

ECR(10): equivalent constant stress of the duration 60 s.

ECR(11): sound speed

ECR(12): failure flag (0=virgin Gauss Point, 1=failed Gauss Point)

ECR(13): angle of failure

ECR(14): status of the spalling: 0 no failure of the g.p.; -1 g.p. under compression;
+1 g.p. under tension.

9.6.56 SMAZ: MAZARS-LINEAR ELASTIC LAW WITH DAMAGE FOR SPHC ELEMENTS**Object:**

Isotropic linear elastic with Mazars damage for SPHC elements.

References:

1- Jacky MAZARS, "Application de la mécanique de l'endommagement au comportement non linéaire et la rupture du béton de structure", Thèse de doctorat, Université Pierre et Marie Curie - Paris 6, 1984.

Syntax:

```
"SMAZ" "RO"    rho  "YOUN" young  "NU"    nu    "EPSD" epsd  
      "DCRI" dcri "A"     a      "B"     b  
      "TAUC" tauc "CSTA" csta                               /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

epsd

Initial strain threshold.

dcri

Critical value of damage (=1 per default).

a

Parameter A of the tension law (asymptote of the curve stress-strain)

b

Parameter B of the tension law (shape of the curve stress-strain)

tauc

Characteristic time for delay-damage

csta

Parameter of the delay-damage (=1 per default)

Outputs:

The components of the ECR table are as follows:

ECR(1) : Pressure

ECR(2) : Von Mises criterion

ECR(3) : Equivalent strain

ECR(4) : Failure state (0: no failure, 1: failed)

9.6.57 SLIN: LINEAR ELASTIC LAW WITH DAMAGE FOR SPHC ELEMENTS**Object:**

Isotropic linear elastic with damage for SPHC elements.

Syntax:

```
"SLIN" "RO" rho "YOUN" young "NU" nu "EPSD" epsd  
      "DCRI" dcri "EPSR" epsr  
      "TAUC" tauc "CSTA" csta /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's ratio.

epsd

Initial strain threshold.

dcri

Critical value of damage (=1 per default).

epsr

Maximum strain before failure.

tauc

Characteristic time for delay-damage

csta

Parameter of the delay-damage (=1 per default)

Outputs:

The components of the ECR table are as follows:

ECR(1) : Pressure

ECR(2) : Von Mises criterion

ECR(3) : Equivalent strain

ECR(4) : Failure state (0: no failure, 1: failed)

9.6.58 JCLM**Object :**

This directive allows to describe the behaviour of an elasto-plastic material that may undergo some damage, according to the Lemaitre model. There is coupling between damage and plasticity, represented by the Johnson-Cook model. The damage evolution rate is a function of the triaxiality ratio of stresses and of the equivalent plastic strain rate. A failure criterion is implicitly contained within the model: rupture occurs when the damage exceeds a critical value. Two optional parameters allow to introduce a limitation of the damage rate (thanks to the delayed damage model) in order to avoid the mesh dependency.

Syntax:

```
"JCLM"  "RO" rho "YOUN" young "NU" nu
        "EPSD" epsd "S0" s0 "DC" dc
        <"CSTA" csta "TAUC" tauc "NOCO" noco>
        "COA1" coa1 "COA2" coa2
        "CLB1" clb1 "CLB2" clb2 "SRRF" srrf /LECTURE/
```

rho

Density.

young

Young's modulus.

nu

Poisson's coefficient.

epsd

Damage threshold (i.e. equivalent plastic strain, weighted by a function of stress triaxiality, within which damage vanishes).

s0

Parameter driving the damage evolution rate.

dc

Critical damage defining the rupture criterion.

csta

Parameter of the delayed damage model

tauc

Characteristic time of the delayed damage model. $(1/\text{tauc})$ represents the maximum damage rate.

noco

Optional parameter indicating what to do when no convergence is reached in the material routine. The value 0 is the default and means that an error message is issued and the calculation is stopped. The value 1 indicates that the element (or more precisely, the element's current Gauss point) is made to fail (eroded).

coa1

1st constant in the Johnson-Cook model.

coa2

2nd constant in the Johnson-Cook model.

clb1

3rd constant in the Johnson-Cook model.

clb2

Hardening coefficient of the Johnson-Cook model.

srrf

Reference strain rate of the Johnson-Cook model.

LECTURE

List of concerned elements.

Comments:

A detailed description of the damage model can be found in the report DMT/98-026A, available on request.

The implementation of the Johnson-Cook model is described in reference [\[167\]](#).

This material is currently restricted to SPHC elements.

Outputs:

The components of the ECR table are as follows for **Continuum elements**:

ECR(1) : pressure

ECR(2) : Von Mises criterion

ECR(3) : equivalent plastic strain

ECR(4) : plasticity multiplier

ECR(5) : damage

ECR(7) : new elastic limit

When the “erosion” algorithm is activated (see page A.30, Section 6.4, keyword **FAIL**), an element is considered as failed if **damage** \geq **dc**.

9.7 FLUID MATERIALS

Object:

The following directives describe fluid materials for continuum elements.

Here are the different material types:

- "FLUI" : isothermal fluid ($c = \text{const.}$)
- "GAZP" : perfect gas
- "SOUR" : time-dependent internal pressure
- "NAH2" : sodium-water reaction (1 liquid and 1 gas)
- "RSEA" : sodium-water reaction (1 liquid and 2 gases)
- "EAU" : water in two-phase homogeneous mixture
- "ECOU" : 1D flow
- "PARO" : 1D wall
- "MULT" : multiple material
- "LIQU" : incompressible liquid 1D
- "FLFA" : pipe bundle (homogeneous acoustic model)
- "MHOM" : pipe bundle (homogeneous asymptotic model)
- "PUFF" : state equation of type PUFF
- "GZPV" : Van Leer perfect gas
- "ADCR" : homogeneous mixture with 3 components (1 liquid and 2 gases)
- "EXVL" : hydrogen explosion Van Leer
- "JWL" : state equation of type Jones-Wilkins-Lee
- "CHOC" : state equation of type Rankine-Hugoniot
- "GPDI" : Van Leer diffusive perfect gas
- "ADCJ" : same as "ADCR" but with a "JWL" gas
- "BILL" : specialised equation of state for the particle elements
- "GVDW" : Van der Waals gas
- "JWLS" : equation of state of type Jones-Wilkins-Lee (solids)
- "BUBB" : bubble model: bubble with compressed air instead of explosive

Comments:

These materials are detailed in the following pages.

All pressures given as parameters are absolute pressures that must account for the external pressure. If one wants to avoid an unwanted transient expansion, it is necessary to specify the reference pressure "PREF", which must be the same for all fluid materials in a calculation.

For example, for a reservoir filled with gas at the relative pressure of 10 MPa ($P_{int} - P_{ext} = 10 \text{ MPa}$), it is necessary to specify an internal pressure of 10.1 MPa if the atmospheric pressure is 0.1 MPa. Then, two cases are possible:

1) The reservoir is initially in equilibrium:

The calculation aims at simulating the response of the reservoir to an overpressure which appears later on (shock, explosion, imposed velocity ...). The reference pressure must then be: $p_{ref} = 10.1 \text{ MPa}$, so that the reservoir remains initially in equilibrium.

2) The reservoir is not initially in equilibrium:

The calculation aims at simulating the response of the reservoir to an internal pressure which appears abruptly. The reference pressure must then be: $p_{ref} = 0.1 \text{ MPa}$, so that the final status be correct.

9.7.1 FLUID

Object:

This option enables a fluid (liquid-like) behaviour for continuum elements to be input. The fluid (isothermal) can be perfect (no viscosity) or viscous.

The expression used to compute the absolute pressure p in the fluid is:

$$p = p_{\text{ini}} + (\rho - \rho_{\text{ini}})c^2$$

where p_{ini} is the fluid pressure in the initial state, ρ is the current density, ρ_{ini} is the initial density and c is the sound speed, which is considered constant.

By default the fluid is considered “free” (i.e. fluid alone, keyword **LIBR**). However, it is also possible to take into account the volume occupied by some fixed internal structures (which are not meshed) by specifying the optional keyword **POREUX**. Such a “porosity” may be specified either in 2D or in 3D, but only for the elements of type **CAR1**, **CUBE** and **PRIS**.

Syntax:

For a "free" fluid (no internal structures) :

```
"FLUID"  < "LIBR" >  "RO" rho  "C" c  <"PINI" pini>  ...
... <"PREF" pref >  <"PMIN" pmin >  <"VISC" mu >  ...
... /LECTURE/
```

For a "porous" fluid (with internal structures) :

```
"FLUID"  "PORE"  "RO" rho  "C" c  <"PINI" pini>  ...
... <"PREF" pref >  <"PMIN" pmin >  <"VISC" mu >  ...
... "PORO" alpha  < "SMOU" sur >  < "BETA" beta >  ...
... $ "KPER" kp ; "KPX" kpx  "KPY" kpy  < "KPZ" kpz > $ ...
... /LECTURE/
```

"LIBR"

The fluid is “free”, i.e. without internal structures. This is the default option.

"PORE"

The fluid is “porous”, i.e. it occupies just one part of the meshed volume, the rest being occupied by some fixed internal structures.

rho

Initial density ρ_{ini} of the fluid.

c

Sound speed c in the fluid, considered constant.

pini

Absolute initial pressure p_{ini} in the fluid. By default, $p_{\text{ini}} = 0$.

pref

Absolute reference pressure p_{ref} in the fluid. By default, $p_{\text{ref}} = p_{\text{ini}}$ (even when $p_{\text{ini}} = 0$).

pmin

Absolute minimum pressure p_{min} in the fluid. By default, $p_{\text{min}} = 0$. Obviously, it must be $p_{\text{min}} \leq p_{\text{ini}}$. The minimum density ρ_{min} results then from the expression:

$$\rho_{\text{min}} = \rho_{\text{ini}} + \frac{p_{\text{min}} - p_{\text{ini}}}{c^2}$$

mu

Dynamic viscosity coefficient μ (2D or 3D).

alpha

Value of the porosity: ratio of the volume occupied by the fluid with respect to the total volume.

sur

Relative wet surface (value 1 by default).

kp

Head loss coefficient by unit length, assumed isotropic.

kpx, kpy, kpz

Head loss coefficient by unit length in the direction Ox (respectively Oy, Oz).

beta

Reduced damping coefficient for high frequencies. It is zero by default, and should always be very small.

/LECTURE/

List of the elements concerned.

Comments:

The parameters **R0** and **C** are compulsory.

Role of PREF:

When the reference pressure is different from the initial one, the fluid is not in equilibrium at the beginning. This is the case e.g. when a membrane is breaking at $t = 0$, releasing a compressed fluid. For further detail, see page C.300.

In various problems, studies relate to acoustic effects; since it is supposed that a fluid in equilibrium evolves under the effects of loading (motion of a piston, shock,...), in this case it must be: $p_{\text{ref}} = p_{\text{ini}}$.

If **PREF** is omitted, EUROPLEXUS considers that the fluid is in equilibrium and $p_{\text{ref}} = p_{\text{ini}}$ (even when $p_{\text{ini}} = 0$).

For a given minimum pressure p_{min} , the fluid pressure is always greater than or equal to that value, even if the density is decreasing. This is a very simple way to model cavitation. The default value of p_{min} is $p_{\text{min}} = 0$.

Viscosity:

In the presence of viscosity, the tensor of stresses in the fluid has the following form:

$$\sigma(i, j) = -P \delta(i, j) + 2\mu \dot{\epsilon}(i, j)$$

with:

P : pressure

$\delta(i, j)$: Kronecker's symbol

$\dot{\epsilon}(i, j)$: strain rate (derived from $\epsilon(i, j)$)

For water at 20 degrees Celsius: $\mu = 0.001$ SI units (Kg/(m*s)).

Porous fluid:

If the fluid is porous, the parameter **PORO** is mandatory. In this case an equivalent fluid is used by the code for the calculations, which occupies the entire volume of the element. However, the used variables (pressure, velocity, etc.) are those of the **REAL** fluid, so as to obtain directly the physical state of the fluid in the presence of internal structures.

If these internal structures generate a head loss, the parameter **kp** allows to model it in case this loss is isotropic. Otherwise, the parameters **kpx**, **kpy**, **kpz** allow to distinguish between the three directions in the global reference.

The former coefficients are given per unit length. For example, if the head loss is $\Delta P = 0.25$ bar over a length of $L = 2$ m, for a fluid of density $\rho = 1000 \text{ kgm}^{-3}$ with a velocity $V = 5 \text{ ms}^{-1}$, the coefficient will be $K_p = 1$ according to the formula:

$$\Delta P = \frac{1}{2} K_p L \rho V^2$$

The parameter **SMOU** (relative wet surface) is obsolete and may be omitted. It is only kept for compatibility with old input files.

Correlation between bulk modulus and sound speed:

This material model can be compared to that of a fluid with constant bulk modulus (e.g. the **FLUT** material with **NUM 9**) as follows. For the latter, the absolute pressure is given by:

$$p = p_{\text{ini}} + B\eta$$

where B is the bulk modulus (assumed constant and usually expressed in Pa) and η is the relative volume variation:

$$\eta = -\epsilon_V = \frac{V - V_{\text{ini}}}{V_{\text{ini}}} = 1 - \frac{\rho_{\text{ini}}}{\rho}$$

From these expressions one obtains:

$$p = p_{\text{ini}} + (\rho - \rho_{\text{ini}}) \frac{B}{\rho}$$

By comparing this with the pressure expression of the **FLUI** material, one sees that:

$$c = \sqrt{\frac{B}{\rho}}$$

Therefore, strictly speaking the two models are different because in one the sound speed is assumed constant (so that the bulk modulus varies with the density) while in the other the bulk modulus is assumed constant (so that the sound speed varies with the density). However, by assuming that the density varies only slightly from the initial value ρ_{ini} , one obtains the following relation between c and B :

$$c \approx \sqrt{\frac{B}{\rho_{\text{ini}}}}$$

Outputs:

The components of the **ECR** table are as follows:

ECR(1): absolute pressure

ECR(2): density

Reference:

CEA report to appear.

9.7.2 PERFECT GAS

Object:

Euler : perfect gas ($P = \rho(\gamma - 1)E_{\text{internal}}$);

Lagrange: adiabatic perfect gas ($P = k\rho^\gamma$).

In a 1-D case, the frictions against the walls can be taken into account, since the dissipated energy will heat up the gas (modification of the internal energy). To this end the user has to add a PARO material, which must be associated with GAZP by means of the MULT material (see pages C.370 and C.380).

Syntax:

```
"GAZP"  "RO"  rho  "GAMMA"  gamma  "PINI"  pini  <"VISC"  mu  >  ...
      ... < "CV"  cv  >  < "PREF"  pref  >  /LECTURE/
```

rho

Initial density.

gamma

Ratio c_P/c_V (supposed constant).

pini

Initial pressure.

mu

Dynamic viscosity of the gas (for 2-D and 3-D).

cv

Specific heat at constant volume c_V (used to compute the temperature).

pref

Reference pressure. Note that, by default, it is assumed **pref** = **pini**.

/LECTURE/

List of the elements concerned.

Comments:

The reference pressure `pref` enables the initial state to be defined. If `pref = pini`, the gas is in equilibrium just before the computation starts; it will be perturbed by an external action, by the motion of a piston, for instance. If `pref = 0`, the problem consist in a computation with initial stresses determined by `pini`. This is the case when a membrane which was seperating two gases at different states disappears at the initial instant.

If `cv` is omitted, the temperature is not computed. If it is present, the temperature is expressed in degrees Celsius.

Outputs:

The different components of the ECR table are as follows:

- ECR(1): pressure
- ECR(2): density
- ECR(3): velocity of sound
- ECR(4): maximum pressure ever experienced
- ECR(5): minimum pressure ever experienced
- ECR(6): dynamic pressure: ($P_{\text{dyn}} = \frac{1}{2}\rho v^2$)
- ECR(7): temperature (if c_V is not zero)
- ECR(8): total specific energy ($E = h + \frac{1}{2}v^2$)

9.7.3 SOURCE

Object :

This instruction enables a time dependent pressure to be imposed, inside an element.

For fluids modelled in ALE, this material allows to create a source of mass flow, if it is used in conjunction with an imposed velocity directive. However, this source is limited to the case of a liquid (case "FLUI"), of a perfect gas ("GAZP"), of a two-phase mixture of water ("EAU") or of a liquid-gas mixture ("ADCR").

Syntax:

```
"SOUR"    $ "FLUI" ... ; "GAZP" ... ; "EAU" ... ; "ADCR" ... $
... < "FONC" nufo < "FACT" coef > > /LECTURE/
```

FLUI, GAZP, EAU, ADCR

This keyword indicates the source material data which are strictly identical to those of the material with the same name.

For "FLUI" see 9.7.1 page C.305, for "GAZP" see 9.7.2 page C.310, for "EAU" see 9.7.6 page C.350, for "ADCR" see 9.7.14 page C.430,

nufo

Number of the 'FONCTION' allowing to define the pressure as a function of time

coef

Multiplying factor for the pressures given by the preceding function. By default coef=1.

LECTURE

List of the concerned elements.

Comments:

The "FONCTION" directive is described in 11.1 (page E.15).

The initial pressure, must correspond to the origin of the curve.

The element deforms only under the action of forces due to the imposed pressure (no stiffness). The temperature is assumed constant. In the case of water, for example, if the pressure increases one can pass from a liquid phase to a vapor phase, but always at the same temperature.

Outputs:

The different components of the ECR table are the same as the components for the material with the same name.

9.7.4 SODIUM-WATER REACTION ("NAH2")

Object:

Explosion caused by water injection into liquid sodium.

In 2D or 3D, one element or more (contiguous) elements may be affected by the chemical reaction. In 1D, just one "TUBE" or "TUYA" element must be used with this material.

In 1D or 3D there are two options for the water mass flow rate:

- 1) imposed curve as a function of time (keyword "DEBIT")
- 2) calculation of a water-filled pipeline meshed by elements of type TUBE or TUYA and coupling with the sodium mesh (keyword "DCOUP").

Syntax:

```
"NAH2"  "R0" rho  "C" c  "PINI" pinit  "PV" pv  "FACT" facteur
... < "PREF" pref > < "PMIN" pmin > < "CMIN" cmin > < "CH2" ch2 >
...$[ "DEBIT"  npt*(temps , debit) ;
      "DCOUP"  1      tini      qini ]$/LECTURE/
```

rho

Initial density (pure sodium).

c

Sound speed in the sodium.

pinit

Initial pressure.

pref

Reference pressure (see page C.300).

pv

Value of the product $P \cdot V$ for the unit mass of hydrogen at the initial temperature.

facteur

Number of gas moles formed starting from one mole of injected water.

ch2

Initial mass fraction for the hydrogen. Bu default, ch2 = 0 and as a maximum ch2 = 1.

cmin

Minimum mass fraction of the hydrogen in the pure sodium (1.E-8 by default).

pmin

Minimum pressure (zero by default).

"DEBIT"

Keyword that announces the introduction of the curve of total mass flow rate of water injected in the sodium for the whole set of elements concerned.

npt

Number of points defining the mnass flow rate curve for the water.

(temps, dbit)

Coordinates of the points (in axisymmetric, divide the mass flow rate by 2π , as the calculation refers to one radian).

"DCOUP"

The water mass flow rate at the outlet of a pipe is computed (1D) by directive "IMPE" "NAH2" (page C.610).

tini

Initial time.

qini

Initial mass flow rate.

/LECTURE/

List of the elements affected by the injection..

Comments:

For the dimensioning, an injection curve requires a space similar to that of a traction curve. The user will therefore have to specify "TRAC" n1 n2, with n1 the maximum number of curves to be entered (traction and injection), and n2 the maximum number of points.

In the elements affected by the reaction, it is assumed that the reaction is instantaneous, that the mixture of reacting components is always homogeneous and that the reaction is isothermal.

The "facteur" parameter allows to account for the vaporised 'soude'. If there is none, then $\text{facteur} = 0.5$.

If the volume fraction of hydrogen (printed as ECR(4)) is above 1, the program treats the mixture like pure hydrogen.

In a Lagrangian calculation (default option) the "NAH2" material is attached only to the elements where the reaction takes place. The "FLUI" material is used for the other elements.

In an Eulerian calculation (option "EULER" page A.30), the material "NAH2" is affected to ALL fluid elements. The mass flow rate curve for the water is only affected to the elements where the reaction occurs, for the others one must write:

```
"DEBIT" 0 /LECTURE/
```

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density

ECR(3) : hydrogen concentration

ECR(4) : hydrogen volume occupation ratio

ECR(5) : total water mass flow rate for the set of elements

ECR(6) : water mass injected in each element

ECR(7) : sound speed in the Na+H₂O mixture

ECR(8) : hydrogen mass per unit volume (in Eulerian)

ECR(10): phase indicator (1= saturated in H₂; else 0)

9.7.5 SODIUM-WATER REACTION (“RSEA”)

Object:

Explosion caused by water injection into liquid sodium.

It is possible to repeat this material as many times as needed, provided the injections occur in different elements.

There are two possibilities concerning the water mass flow rate:

- 1) a time function is invoked (keyword "NUFO")
- 2) compute a water-filled pipeline meshed with TUBE or TUYA elements and coupled with the sodium mesh (keyword DCOU).

Syntax:

```
"RSEA"  "PTOT" ptot  "PNA" pna  "RONA" rhona  "CSNA" csona
        "PBU"  pbu   "ROBU" rhobu  "NBU"  nbu   "GBU"  gbu
        < "PARG" parg  "ROAR" rhoar  "GAR"  gar >
        < "PSAT" psat  "ROSA" rhovap >
        < "XBU"  xbu  > < "XAR"  xar  > < "PREF" pref >
        < "CMIN" cmin > < "BETA" beta >
        < "VINA" vina > < "VIBU" vibu > < "VIAR" viar >
        < $[ "NUFO" nufo  "COEF" coef  ; "DCOU" 1  ]$ >
        < "FACT" facteur >
        /LECTURE/
```

ptot

Total pressure of the mixture.

pna

Zero pressure of sodium, defining the equation of state of the liquid, by means of the following rhona and csona parameters.

rhona

Zero density of sodium.

csona

Sound speed in the sodium.

pbu

Zero pressure of the hydrogen, defining the equation of state of the gas, by means of the following rhobu and nbu parameters.

rhobu

Zero density of the hydrogen.

nbu

Polytropic coefficient of the transformation followed by the hydrogen. For an isothermal: $nbu = 1$.

gbu

Ratio C_p/C_v for the hydrogen.

parg

Zero pressure of the argon, defining the equation of state of this gas, by means of the following rhoar and gar parameters.

rhoar

Zero density for the argon.

gar

Ratio C_p/C_v for the argon.

psat

Saturation pressure of the sodium vapor. A priori this is very low, and allows to treat correctly the possible cavitation phenomena.

rhovap

Density of the sodium vapor.

xbu

Initial mass fraction of the hydrogen. It allows to account for the presence of gas in the considered domain.

xar

Initial mass fraction of the argon. This refers to the subdomains that are filled with argon initially, such as the cover gas region or the zones located behind membranes.

pref

Reference pressure (see page C.300).

cmin

Maximum mass fraction of gas in the sodium in order to consider it as pure (1.E-8 by default).

vina

Dynamic viscosity of sodium.

vibu

Dynamic viscosity of the gas in the bubble.

viar

Dynamic viscosity of argon.

beta

Reduced damping coefficient for high frequencies. It is zero by default, and should always be very small (< 0.05).

nufo

Number of the adimensional function defined via the "FONCTION" directive of EUROPLEXUS. This function allows to describe the variation of injected water mass flow rate in time.

coef

Multiplicative factor por the precediing function. This allows to account for the units and the number of ruptured pipes.

DCOU

This keyword specifies that a coupled water flow calculation is desired. It is always followed by an integer value.

facteur

Number of gas moles formed by one mole of injected water.

/LECTURE/

List of the concerned elements.

Comments:

In the elements affected by the reaction, it is assumed that the reaction is instantaneous, and that the mixture of reacting materials is always homogeneous.

The "facteur" parameter allows to account for the vapor 'soude'. If there is none, then `facteur` = 0.5.

The mesh will be subdivided in as many zones as necessary, and for each of these an RSEA material will be defined, by possibly varying the initial concentrations and the total pressures, but the other parameters must be identical, so as to have exactly the same constitutive laws for the different components of each zone. Then, starting from the given concentration and the total pressure `ptot`, EUROPLEXUS will compute the density of the mixture. EUROPLEXUS will also recompute the gas concentrations in order to account for the sodium vapor, if `psat` is not zero.

The elements where the reaction occurs will be distinguished by one of two possible options: imposed injection or injection coupled with the calculation of water mass flow rate (DCOU).

If `psat` and `rhovap` are absent, or if one only of these values is given, it is the default value which is used: `psat` is taken equal to one thousandth of `ptot`. The value of `rhovap` is then proportional to `psat`, and corresponds to a monoatomic vapor at a temperature close to 300°C.

Outputs:

The components of the ECR table are as follows:

- ECR(1) : absolute pressure,
- ECR(2) : density of the two-phase mixture,
- ECR(3) : sound speed in the mixture,
- ECR(4) : void fraction,
- ECR(5) : argon mass fraction,
- ECR(6) : hydrogen mass fraction,
- ECR(13) : water mass flow rate (dm/dt)
- ECR(14) : mass of water injected since the beginning.

9.7.6 WATER

Object :

This directive allows to treat water and its vapour as an homogeneous mixture. It is also possible to treat a water vapor explosion when energy is released within liquid water.

Syntax :

```
"EAU"  $[ "EQUI"                ;
          "META"  "NBUL" nbul  "ALFN" alfn ]$  ...

...  "PINI" pini  |[ "TINI" tini ; "TITR" x ]| ...
...  < "PREF" pref > < "BETA" beta > ...
...  < "VISL" mul  "VISV" muv  >  ...
```

For a direct injection:

```
$ < "ENMA" enma  "FONC" numf  ...
... < "XCOR" xcor > < "MODE" mode ; "COEF" coef > > $
... < "DPROP" dpropag  "ORIGINE"  /LECTURE/  > $
```

For an injection of corium particles:

```
...$ < "DIAM" diam  "CECH" hh  ...
...  "TCOR" tcor  "VCOR" vcor  > $
```

One ends this directive by:

```
.../LECTURE/
```

EQUI

The mixture will be in equilibrium (same pressure and same temperature for the liquid and vapour phases).

META

The mixture will be metastable (same pressure but different temperatures for the liquid and vapour phases).

nbul

Number of vapour bubbles by unit volume (of the order of 1.E9).

alfn

Minimal void fraction for the nucleation of vapor bubbles (of the order of 1.E-4).

pini

Initial pressure of the mixture.

tini

Initial temperature of the equilibrium mixture (in degrees Celsius).

x

Initial mass title of the vapor, between 0 and 1. (eau=0, vapeur=1).

pref

Reference pressure (for its meaning see page C.300). By default, it is equal to the initial pressure. All reference pressures must be equal.

beta

Reduced damping coefficient for high frequencies. It is zero by default, and should always be very small (< 0.05).

mul

Dynamic viscosity of the water. Recall: below 1 bar, at 25°C, mul = 9.E-4 Poiseuille or Pascal * second. This value drops rapidly as the temperature increases. By default mul = 0.

muv

Dynamic viscosity of the water vapor. Recall: below 1 bar, at 100°C, muv = 1.3E-5 Poiseuille. This value increases with the temperature. By default, muv = 0.

For a direct injection of energy in the water:

enma

Specific power injected in the water. Multiplicative coefficient (dimensional) of the following function.

numf

Number of the non-dimensional function defined by the "FONCTION" directive of EUROPLEXUS. This function allows to vary the injected power in time.

xcor

Ratio between the corium mass and the water mass contained within the element. The specific power (enma) applies only to the present corium. By default, xcor = 1.

mode

Choice of the injection mode: (0, 1, 2 or 3). The meaning is explained in the comments below. By default, mode = 0.

coef

Ratio between the limit density and the initial density. This allows to limit the injection. By default, coef = 0. Is only relevant for mode = 0 or 1.

dpropag

Propagation velocity of the energy injection signal for the elements of the considered domain. By default, the injection occurs simultaneously in all such elements.

ORIGINE

This keyword announces the reading of the element in which the injection is initiated. The injection then propagates with the speed dpropag in all directions.

For an injection by means of corium particles:**diam**

Diameter of the particles.

hh

Heat exchange coefficient between a corium particle and the liquid water.

tcor

Initial temperature of the corium particles.

vcor

Volume fraction occupied by the corium.

LECTURE

List of the concerned elements.

Comments :

Do not forget to create the tables of physical properties of the water by means of directive "TEAU" or "TH2O" (page C.74).

The "TH2O" table is only working with equilibrated water.

If the mixture is single-phase, one should give the pressure and the temperature, but the title is irrelevant. If the mixture is two-phase, one should give the pressure and the title: EUROPLEXUS then computes the temperature.

The damping coefficient beta allows to damp out high-frequency oscillations caused by the discretisation. By default beta is zero. However, it is advised to use beta between 0.1% et 5%. One should be aware of the inevitable attenuations of lower frequencies, especially if the mesh is coarse. In fact, the eigenfrequencies of the structure are not very different from the frequencies associated with the finite elements.

If the viscosity has to be considered, the two parameters mul and muv must be related and given together.

Energy injection:

The "MODE" parameter allows to account for the evolution of fluid close to the injection zone, in a less brutal fashion compared with directive "COEF". Its meaning is as follows:

- mode = 0 : the injected energy is independent from the fluid mass and nature,
- mode = 1 : the injected energy is proportional to the mass of water, but independent from its nature of liquid or vapor,
- mode = 2 : the injected energy is proportional to the mass of liquid water,
- mode = 3 : the injected energy is proportional to the volume of liquid water.

Modes 2 and 3 should not be used if the pressure exceeds the critical value ($P_{crit} = 221$ bar).

The "COEF" directive allows to limit the quantity of injected energy: if the density during the computation becomes lower than the limit density, then the injection is stopped. This directive is brutal and not advisable. It is preferable to use the "MODE" directive.

One may obtain the energy quantity released in a certain region of the mesh by means of keyword "WINJ" in directive "REGION" (page G.100).

By assigning a propagation velocity associated with an origin element allows to avoid a brutal and instantaneous injection over an extensive domain, which is unrealistic: this option is recommended in case of a steam explosion calculation.

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density of the mixture

ECR(3) : sound speed

ECR(4) : mass title of the vapor (vapor mass/total mass)

ECR(5) : temperature of the mixture for equilibrated water, liquid temperature for meta-stable

ECR(6) : enthalpy of the mixture

ECR(7) : temperature of the mixture for equilibrated water, vapor temperature for meta-stable

ECR(9) : void ratio or volume ratio of the vapor (vapor volume/total volume)

For meta-stable water:

ECR(21) : vapor relative density (vapor mass/total volume)

ECR(23) : index : 0 = equilibrium ; 1 = meta-stable

ECR(24) : specific enthalpy of the liquid water

In case of direct energy injection:

ECR(8) : power injected in the element

ECR(14) : corium mass within the element

In case of energy injection by particles:

ECR(19) : initial volume of corium within the element

ECR(20) : mean temperature of a corium particle

For a "BREC" element:

ECR(25) : Pipeline rupture area

ECR(26) : Mass flow

ECR(27) : Total ejected mass

9.7.7 HELIUM

Object :

This directive allows to treat helium and its liquid as an homogeneous mixture.

Syntax :

```
"HELI"  "PINI"  pini  |[ "TINI" tini ; "TITR" x ]| ...  
      ... < "PREF" pref > < "BETA" beta >      ...  
      ... < "VISL" mul  "VISV" muv  >      ...  
      ... /LECTURE/
```

pini

Initial pressure of the mixture.

tini

Initial temperature of the equilibrium mixture (in degrees Kelvin).

x

Initial mass title of the vapour, between 0 and 1. (liquid=0, vapour=1).

pref

Reference pressure (for its meaning see page C.300). By default, it is equal to the initial pressure. All reference pressures must be equal.

beta

Reduced damping coefficient for high frequencies. It is zero by default, and should always be very small (< 0.05).

mul

Dynamic viscosity of the liquid. By default $mul = 0$.

muv

Dynamic viscosity of the vapour. By default, $muv = 0$.

Comments :

Do not forget to create the tables of physical properties of helium by means of directive "THEL" (page C.75).

If the mixture is single-phase, one should give the pressure and the temperature, but the title is irrelevant. If the mixture is two-phase, one should give the pressure and the title: EUROPLEXUS then computes the temperature.

The damping coefficient beta allows to damp out high-frequency oscillations caused by the discretisation. By default beta is zero. However, it is advised to use beta between 0.1% et 5%. One should be aware of the inevitable attenuations of lower frequencies, especially if the mesh is coarse. In fact, the eigenfrequencies of the structure are not very different from the frequencies associated with the finite elements.

If the viscosity has to be considered, the two parameters mul and muv must be related and given together.

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density of the mixture

ECR(3) : sound speed

ECR(4) : mass title of the vapor (vapor mass/total mass)

ECR(5) : temperature of the mixture

ECR(6) : specific enthalpy of the mixture

9.7.8 1D WALL

Object:

This directive allows, in association with the **MULT** material, to account for the effects of pipe walls and cavity walls, for elements of type **TUBE**, **TUYA** or **CAVI**.

Syntax:

```
"PARO"  $[  "RUGO"  rug      "VISC" mu                ;
            "RUGO"  rug      "VISL" mul      "VISV" muv    ;
            "TPAR"  teta     "COND" cof    < "SURF" su >    ;
            "PSIL"  kl
            "FONC"  numf < "SURF" su    >                ]$

            < "COEF" nbr >  /LECTURE/
```

rug

Absolute rugosity of the pipe (warning, this parameter has the dimension of a length).

mu

Dynamic viscosity (single-phase fluid).

mul

Liquid dynamic viscosity (two-phase fluid).

muv

Vapor dynamic viscosity (two-phase fluid).

teta

Wall temperature.

cof

Conductance.

su

Heat exchange surface.

kl

Head loss coefficient per unit length (1/m).

nbr

Multiplicative coefficient in the case of an assembly of identical pipes ($\text{nbr} = 1$ by default).

numf

Number of the **FUNCTION** allowing to define the conductance as a function of the flow velocity of the fluid.

LECTURE

List of the elements concerned.

Comments:

The **MULT** material is used (page C.380) to associate the wall defined by **PARO** with the corresponding internal fluid.

If the internal fluid is of type **NAH2**, **EAU** or **RSEA**, it is mandatory to specify the viscosities of both phases, **mul** for the liquid and **muv** for the vapor. In all other cases, the fluid is supposed to be single-phase, and only one viscosity **mu** will be required.

In the case of heat exchange with the wall, the keyword **SURF** is mandatory for the **CAVI** elements. For the elements of type **TUBE** and **TUYA** it may be omitted, and in this case **EUROPLEXUS** will compute the exchange surface starting from the geometrical characteristics of the elements.

The **k1** coefficient (keyword **PSIL**) allows to compute the head loss in the following way:

$$DP = k1 * long * 0.5 * rho * V ** 2$$

with:

DP : head loss,
long : length of the pipe,
rho : fluid density,
V : mean velocity,

Hence: $k1 = \text{psi} / Dh$, where **Dh** is the hydraulic diameter.

Outputs:

The parameters associated with the wall material not included in ECR are placed after the ECR for the fluid (see the fluid documentation).

The components of the ECR table are as follows:

ECR(1) : wall temperature

ECR(2) : conductance

ECR(4) : Reynolds number

ECR(5) : psi (of the formula $K = \text{psi} * L / Dh$)

9.7.9 MULTIPLE MATERIALS

Object:

This directive allows to assign several materials to the same element. For example, it is the case of a pipeline element, where one has to specify both the material for the internal fluid and the material of the wall plus, if necessary, a material describing the friction.

Syntax:

```
"MULT"  n1  n2  < n3 > /LECTURE/
```

n1

Number assigned to the first material.

n2

Number assigned to the second material.

n3

Number assigned to the third material.

LECTURE

List of the elements concerned.

Comments:

The materials concerned must be defined previously, and are referenced by their law index (see LOI, page C.100). This is either the number explicitly given by the LOI keyword, or the material definition order in the input file.

In the case of "TUBE" elements, n1 will be the index of the fluid material and n2 the index of the friction material.

In the case of "TUYA" elements, the fluid must be referenced first, the wall second and, if necessary, the friction as third. For example, if one has listed the wall material first (1), then the fluid material (2) and finally the friction material (3), the "MULT" directive will be coded as follows:

"MULT" 2 1 3 /LECTURE/

Outputs:

The stresses and the hardening parameters will be those of the component materials. For example, for an element of pipeline, the printed stresses will be those of the associated beam (no printout of the stresses for the internal fluid), and the ECR(i) will give first the quantities related to the fluid material, then those related to the wall material.

9.7.10 LIQUID**Object:**

This option enables the processing of an incompressible or quasi-incompressible fluid. The implicit algorithm of "LIAISONS" is used.

Syntax:

```
"LIQU"  "RO"  rho < "C"  c > < "PINI"  pini >  ...  
... < "PREF" pref > < "VISC" visc > < "RUGO" rugo >  
/LECTURE/
```

rho

Density.

c

Velocity of sound (only for a compressible fluid).

pini

Initial pressure.

pref

Reference pressure (see page C.300).

visc

Twice the dynamic viscosity (2μ).

rugos

Absolute rugosity.

LECTURE

List of the elements concerned.

Comments:

This material only makes sense if the option "NAVIER" has been required for the definition of the problem.

If the material is incompressible, `c` is useless. On the contrary, `c` is necessary if the fluid is compressible, even at a low level. In this case `c` is read and then the value of $(1/c^2)$ is stored in the material property.

Warning:

It is essential to invert the connection matrix at each step. Do not forget to add the option "FREQ" 1 when using the instruction "LIAISON" (see page D.20).

Outputs:

The components of the ECR table are as follows:

ECR(1): absolute pressure of the element due to the viscous terms

ECR(2): density

ECR(3): additive term to the diagonal of B_L , the connections matrix.

ECR(4): additive term to the right-hand side of the connections system

ECR(5): multiplicative term of the pressure

ECR(6): friction coefficient (see M1FROT)

ECR(7): Reynolds number

ECR(8:10): unused

9.7.11 TUBE BUNDLES

Object:

Replaces a heterogeneous medium composed by a bundle of tubes submerged in a fluid, by an equivalent homogeneous isotropic medium in the acoustic sense. The densities and sound speeds will be different along the three directions in space.

In the case of helicoidal coaxial bundles (2D axisymmetric or 3D), the axis of the bundle must be along the Oz direction, the helices have all the same axial step and are regularly spaced.

In the case of a straight bundle, (2D or 3D), the bundle axis is Oz, and a side of the base 'motif' must be parallel to Ox or Oy.

There are two options for the definition of the three densities and of the three sound speeds:

- a) The values are computed by EUROPLEXUS as a function of the geometrical data (plane waves propagation).
- b) The values have another origin, and are prescribed.

The possible combinations between options and geometries of the bundle are given in the following table:

				Value of	"TYPE"
Geometry	2D		3D		
Option	DPLA	AXIS	TRID		
Triangular step	yes	no	yes	2*	
Rectangular step	yes	yes	yes	1*	
Imposed anisometry:					
Frame Ox,Oy,Oz	no	no	yes	2	
Frame Or,Ot,Oz	no	yes	yes	1 in 3D, 2 in AXIS	

* These values are automatically affected to "TYPE" by EUROPLEXUS.

Syntax:

```

"FLFA"  "RO" rho  "C" c    < "PINI" pini > < "PREF" pref >
                                < "PMIN" pmin > ...

$[ "DIAM" d  $[ "PRAD" pr  "PAXI" pa ]$ < "VISC" mu > < "COEF" coef>
                                $[ "PTRI" pt  "BASE" ba ]$
                                ;

"ROX" rox  "ROY" roy  "ROZ" roz  "CX" cx    ...
    ...  "CY" cy  "CZ" cz  "TYPE" type "TAUX" taux    ;

"ROR" ror  "ROT" rot  "ROZ" roz  "CR" cr    ...
    ...  "CT" ct  "CZ" cz  "TYPE" type "TAUX" taux    ]$

... /LECTURE/

```

rho

Density.

c

Sound speed.

pini

Initial pressure.

pref

Reference pressure (see meaning on page C3.300).

pmin

Minimum pressure (see meaning on page C3.305).

mu

Fluid dynamic viscosity.

coef

Multiplicative factor for the friction (= 1 by default).

d

External diameter of the tubes.

pr

Radial step of the tubes.

pa

Axial step of the tubes.

pt

Equilateral triangular step.

ba

Direction of the triangle base (ba=1 or ba=2).

rox, roy, roz

Densities along directions Ox, Oy and Oz.

cx, cy, cz

Sound speeds along directions Ox, Oy and Oz.

ror, rot, roz

Densities along directions Or, Ot and Oz.

cr, ct, cz

Sound speeds along directions Or, Ot and Oz.

type

Type of bundle: 1 = helicoidal, 2 = straight.

taux

Volume fraction of the fluid ($0 < \text{taux} < 1$).

LECTURE

List of the concerned elements.

Comments:

The calculation may be done in Lagrangian or in Eulerian.

To conserv the fluid mass, an apparent fluid mass is used (printed in ECR(2)), corresponding to that of a fictitious liquid that would occupy the whole volume.

The modelisation chosen for the bundle implies anisotropy effects on inertia and compressibility. For each principal direction i of the bundle, the two parameters ρ_{oi} and $\rho_{oi} \cdot c_i^{**2}$ must be defined. Hence the nodal masses are quite different from those computed from the apparent density.

However, for an Eulerian calculation, the fluxes involve the mass effectively transferred from an element to the other, i.e. the code uses the density of the free fluid and the total cross section of the passage. Consequently, the computed (and printed) velocity is that of a fictitious free fluid placed at the bundle entry (entry speed). In order to estimate the mean velocity of the fluid within the tubes, it is necessary to multiply the computed velocity by the ratio between the cross-sections.

If one has to impose an absorbing boundary condition at the bundle border using elements CL2D, CL3D ou CL3T, he must care that the product $\rho \cdot c$ be the one corresponding to the considered direction; since the CLxD take as ρ the value of the neighbouring element, i.e. the apparent density of the bundle, the sound of speed must be accordingly corrected within directive "IMPE ABSO".

The presence of keyword "VISC" followed by the value of the fluid viscosity triggers the calculation of the head losses in the bundle. The formulation given by I.E.Idel'cik is adopted (Mémento des pertes de charge, Eyrolles, Paris, 1978) for the two principal directions of the bundle in the plane orthogonal to the tubes. The Blasius formula is used in the direction parallel to the tubes (the Reynolds number is then computed by means of the hydraulic diameter).

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : apparent density

Then if VISC is present:

ECR(3) : half-coefficient of friction along direction Ox or Or

ECR(4) : half-coefficient of friction along direction Oy or Ot

ECR(5) : half-coefficient of friction along direction Oz

9.7.12 HOMOGENEISATION OF TUBE BUNDLES**Object:**

Replaces a heterogeneous medium composed by a tube bundle surrounded by fluid, by a homogenised medium (asymptotic homogenisation method).

Syntax :

```
"MHOM"  "RO"  rho  "C"  c  "EPSILON"  epsilon  "YSTAR"  ystar ...  
...     "COEF" coef  "MTUBE" mtube  "KTUBE" ktube...  
...     "NBTUBE" nbtube*("BXX" bxx  "BXY" bxy  "BYX" byx  "BYX" byx ...  
...           "CXX" cxx  "CXY" cxy  "CYX" cyx)  /LECTURE/
```

rho

Fluid density.

c

Sound celerity in the fluid.

epsilon

Size of the elementary cell.

ystar

Fluid surface in an increased elementary cell.

coef

Ratio y/y_{star} (y : total surface of the increased elementary cell).

mtube

Mass of a tube.

ktube

Stiffness of a tube.

nbtube

Number of tubes per elementary cell.

bx

Value of beta-xx obtained by a homogeneisation pre-treatment.

by

Value of beta-xy obtained by a homogeneisation pre-treatment.

yy

Value of beta-yy obtained by a homogeneisation pre-treatment.

cx

Value of eta-xx obtained by a homogeneisation pre-treatment.

cy

Value of eta-xy obtained by a homogeneisation pre-treatment.

yy

Value of eta-yy obtained by a homogeneisation pre-treatment.

LECTURE

List of the elements concerned.

9.7.13 PUFF**Object:**

Equation of state of type PUFF, allowing to treat very fast mechanical phenomena for which the material behaviour is hydrodynamic.

Syntax:

```
"PUFF"  "RO"  rho  "MK"  mk                      ...
... < "PINI"  pini  "PREF"  pref >                ...
...  "GAMMA"  gamma  "D"  d  "S"  s                ...
...  "GAMZ"  gamz  "CV"  cv                        ...
...  $[ "ES"  es                                "SIMPLE"  ;
      "GAM1"  gam1  "N"  n  "EV"  ev  "COMPLEXE"  ]$  ...
...                                              /LECTURE/
```

rho

Initial density.

mk

Compressibility modulus.

pini

Value of the initial pressure.

pref

Value of the reference pressure.

gamma

Constant of perfect gases.

d

Parameter of the PUFF material.

s

Parameter of the PUFF material.

gamz

Gruneisen coefficient for the initial density.

cv

Specific heat at constant volume.

gam1

Parameter of the PUFF material (equals zero for SIMPLEs materials).

n

Parameter of the PUFF material. Example: $n=0.5$ for materials "SIMPLE"; $n=1.67$ for ceramic oxydes.

es

Sublimation enthalpy.

ev

energy at the beginning of vaporisation.

SIMPLE

SIMPLE material (in this case: $\text{gam1} = 0$, $n = 0.5$).

COMPLEXE

PUFF material of type COMPLEXE.

/LECTURE/

List of the concerned elements.

Comments:

The energy will be under form of a field of initial temperatures (see directive INITIAL, page E.50).

Since the phenomenon is very rapid, it is supposed to be adiabatic.

The meaning of PINI and PREF is the same as for the FLUIDE material (page C3.305).

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density

ECR(3) : temperature

9.7.14 ADCR

Object :

Modelling of the behaviour of a liquid-gas mixture. The mixture, which has three components, is assumed to be homogeneous and includes two phases.

By this model it is possible to analyse the consequences of an explosion in a liquid contained within a tank in the presence of a cover gas.

The calculation is A.L.E. only, and for elements CAR1 and TRIA (2D), or CUBE, PRIS and TETR (3D), or TUBE, TUYA, CAVI and BIFU (1D).

Syntax :

```
"ADCR"  "ROLI" roli  <"PLIQ"  pliq  > "CSON"  cson
...  <"PSAT" psat    "ROSA" rosat > <"BETA"  beta  >
...  "ROBU" robu     "PBU"  pbu     "GBU"  gbu   "NBU"  nbu
...  "ROGA" roga     "PGAZ" pgaz    "GAMG" gamg
...  "PTOT" ptot     "CBU"  cbu     "CGAZ" cgaz
...  <"PREF" pref > <"VIBU" mubu > <"VILI" muli  >
...  <"CMIN" cmin > /LECTURE/
```

roli, pliq

Initial density and pressure of the liquid assumed alone, defining the state equation for the liquid. If "PLIQ" is absent, the pressure considered is that of the cover gas.

cson

Sound speed in the liquid.

rosat, psat

Density and pressure of the saturated liquid vapor at the temperature of the liquid, defining the equation of state for the vapor. These parameters are coupled: if one is present, the other must also appear. If they are omitted, they are supposed to be zero.

beta

Reduced damping coefficient for high frequencies. It is zero by default, and should always be very small (< 0.05).

robu, pbu

Initial density and pressure of the bubble gas, supposed alone, defining the equation of state for the gas.

gbu

Ratio C_p/C_v for the bubble gas.

nbu

Polytropic coefficient of the bubble gas.

roga, pgaz

Initial density and pressure of the cover gas, assumed alone, defining the equation of state of the gas.

gamg

Ratio C_p/C_v of gas.

pref

Reference pressure. By default, one takes that of liquid.

cmin

Maximum mass concentration of gas within the liquid, so that it is considered to be pure (1.E-8 by default).

mul i

Dynamic viscosity of liquid.

mubu

Dynamic viscosity of the gas in the bubble.

ptot

Total pressure (see also the comments below).

cbu

Mass concentration of the gas in the bubble (0 or 1).

cgaz

Initial mass concentration of the cover gas (0 or 1).

/LECTURE/

List of the elements concerned.

Remarks :

This material has been initially developed to treat the behaviour of fast reactors in the case of a containment accident (ADC in French), or of a hypothetical core disruptive accident (HCDA in English), but the adopted method lends itself to a more general use.

For this reason, some aliases are introduced. For the liquid, ROLI, PLIQ, CSON, VILI, are identical to RONA, PNA, CNA, VISO (sodium). For the cover gas, ROGA, PGAZ, GAMG, CAR are identical to ROAR, PARG, GAR and CAR (argon). For the bubble generated by the explosion, no aliases are available.

Comments :

The mesh will be subdivided into three zones (bubble, liquid, cover gas) and the ADCR material will be listed three times by varying each time the initial concentrations of the bubble and of the cover gas (0 or 1 depending on the case), but the other parameters must be identical, in order to have the same constitutive laws for the components of each zone. Then, starting from these concentrations and from the total pressure p_{tot} , EUROPLEXUS will compute the density of the mixture. EUROPLEXUS will also recompute the concentrations in the gases in order to account for the liquid vapor, if p_{sat} is not zero.

It is possible, however, to use more than 3 zones, if for example the bubble or the liquid are subdivided into concentric zones. In such cases the total pressure p_{tot} , varying from zone to zone, will define the initial state. EUROPLEXUS will automatically compute the mixture density using the constitutive law of each component. Of course, also in this case the other parameters such as $robu$, pbu , $roli$, $roga$, ... will have to be identical.

The gas generated by the explosion is assumed perfect and the state variables are linked by the following equation:

$$P_B = P_{B0} \left(\frac{\rho_B}{\rho_{B0}} \right)^{n_B}$$

Here P_{B0} and ρ_{B0} are the initial values and n_B is the exponent. This exponent may be an arbitrary real number. The value $n_B = 1$ corresponds to an isothermal transformation while $n_B = \gamma_B$ to an adiabatic transformation.

The keyword **PBU**, **ROBU**, **NBU** and **GBU** allow to specify the parameters P_{B0} , ρ_{B0} , n_B and γ_B in the material data.

The behaviour of the cover gas is of the adiabatic perfect gas type and follows the equation:

$$P_A = P_{A0} \left(\frac{\rho_A}{\rho_{A0}} \right)^{\gamma_A}$$

Here again, P_{A0} and ρ_{A0} are the initial values of the gas pressure and mass density, and γ_A the ratio of specific heats.

The keywords **PGAZ**, **ROGA** and **GAMG** allow to specify the parameters P_{A0} , ρ_{A0} and γ_A in the material data.

The liquid is assumed compressible and isothermal. The state equation is the same as for the FLUI material (page C.305). It is of the form:

$$P_L = P_{L0} + (\rho_L - \rho_{L0})c_L^2$$

Here P_{L0} and ρ_{L0} are the initial values, and c_L the sound speed, assumed constant.

The liquid pressure is not allowed to decrease below a minimal value corresponding to the saturation pressure of the vapor P_{sat} . The vapor (if it exists) then has a density ρ_{sat} .

The parameters P_{L0} , ρ_{L0} , c_L , P_{sat} and ρ_{sat} are defined by means of the keywords **ROLI**, **PLIQ**, **CSO**, **PSAT** and **ROSA**, respectively.

Within an element, the mixture is assumed homogeneous and each component is followed by the evolution of its mass concentration.

Initially, the composition is determined by parameters **CBU** and **CGAZ**, which impose the initial concentrations for the bubble and the cover gas, x_B and x_A . The concentration of the liquid may be computed as the difference, since:

$$x_A + x_B + x_L = 1$$

During the transient calculation, these concentrations evolve as a function of the mass flow across the elements, which determines the the mass density of the mixture and the proportions of each component.

It is then necessary to compute the corresponding pressure of the mixture by respecting the state equations. This is done by successive iterations.

It is also possible to assign dynamic viscosities to the liquid and to the gas (parameters **VILI** and **VIBU**).

Finally, the parameter **CMIN** allows to define the threshold of concentration below which one (or two) component(s) is (are) considered to be absent.

One may obtain the work of pressure forces in a region of the mesh by means of the "PDV" keyword of the "REGION" directive (page G.100).

Outputs :

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density of the two-phase mixture

ECR(3) : sound speed in the mixture

- ECR(4) : mass concentration of the cover gas
- ECR(5) : mass concentration of the bubble gas
- ECR(6) : cover gas mass per unit volume
- ECR(7) : bubble gas mass per unit volume
- ECR(8) : cover gas mass increment
- ECR(9) : bubble gas mass increment
- ECR(10) : volume fraction occupied by the gases

9.7.15 EXVL—VAN LEER HYDROGEN DETONATION**Object:**

Modeling of hydrogen detonation by Van Leer formulation.

Syntax:

```

"EXVL"  <"ROIN" roin>   "TINI" tini    "PINI"  pini
...     <"PREF" pref>   "TSEU" tseu
...     "ALPH" alph     <"GAMM" gamm >
...     <"CVME" cvme>   "CONS" cons
...     "MOO2" moo2     "GAO2" gao2     "COO2" coo2
...     "MON2" mon2     "GAN2" gan2     "CON2" con2
...     "MOH2" moh2     "GAH2" gah2     "COH2" coh2
...     "MOH"  moh      "GAH"  gah      "COH"  coh
...     "MOOH" mooh     "GAOH" gaoh     "COOH" cooh
...     "MEAU" meau     "GEAU" geau     "CEAU" ceau
...     "DEBX" debx     "DEBY" deby     <"DEBZ" debz >
...     "CPVA"          /LECTURE/

```

roin

Initial density of the mixture.

tini

Initial temperature of the mixture.

pini

Initial pressure of the mixture.

pref

Reference pressure.

tseu

Temperature at which the reaction is started.

alph

Parameter for the delay time of the reaction.

gamm

Ratio C_p/C_v of the mixture.

cvme

Mean C_v of the mixture.

cons

Constant R of perfect gases.

moo2

Oxygen molar mass.

gao2

Ratio C_p/C_v of the oxygen.

coo2

Volume concentration of the oxygen.

mon2

Molar mass of nitrogen.

gan2

Ratio C_p/C_v of nitrogen.

con2

Volume concentration of nitrogen.

moh2

Molar mass of hydrogen.

gah2

Ratio C_p/C_v of hydrogen.

coh2

Volume concentration of hydrogen.

moh

Molar mass of H.

gah

Ratio C_p/C_v of H.

coh

Volume concentration of H.

mooh

Molar mass of OH.

gaoh

Ratio C_p/C_v of OH.

cooh

Volume concentration of OH.

meau

Molar mass of water.

geau

Ratio C_p/C_v of water.

ceau

Volume concentration of water.

debx

Horizontal mass flow rate in x (density * velocity = mass flow rate)

deby

Vertical mass flow rate in y (density * velocity = mass flow rate)

debz

Mass flow rate in z (density * velocity = mass flow rate)

cpva

Keyword indicating that the C_p and C_v of the different components vary with temperature.

/LECTURE/

List of the concerned elements.

Comments:

This material may be used in EULERIAN with the option DPLA only.

When one uses the keyword "CPVA" the C_p and C_v of the various components are functions of the temperature (interpolation of the JANAF tables by a fourth-degree function). Generally during a detonation it is necessary to reduce the stability step at a value of the order of 0.1 microseconds.

Outputs:

The different components of the ECR table are:

- ECR(1) : pressure
- ECR(2) : density
- ECR(3) : sound speed
- ECR(4) : mass flow rate along x
- ECR(5) : mass flow rate along y
- ECR(6) : mass flow rate along z (in 3D)
- ECR(18) : Mach number (u/c)
- ECR(19) : temperature
- ECR(20) : entropy
- ECR(35) : concentration in O2
- ECR(36) : concentration in N2
- ECR(37) : concentration in H2
- ECR(38) : concentration in H
- ECR(39) : concentration in OH
- ECR(40) : concentration in H2O

9.7.16 JWL—JONES-WILKINS-LEE LAW**Object:**

State equation of JWL (Jones-Wilkins-Lee) type, allowing to treat explosive phenomena.

Syntax:

```
"JWL"  "R0"  rho  "EINT"  eint  < "ROS"  ros  >          ...
      ... "A"   a    "B"   b    "R1"  r1    "R2"  r2          ...
      ... "OMEG" omeg <"BETA" beta  >  /LECTURE/
```

rho

Initial density.

eint

Initial specific internal energy.

a, b, r1, r2, omeg

Coefficients of the state equation of JWL.

ros

Density of the explosive (solid phase).

beta

Reduced damping coefficient for high frequencies. It is zero by default, and should always be very small (< 0.05).

/LECTURE/

List of the elements concerned.

Comments:

The JWL equation of state gives the value of pressure according to the expression:

$$P = A\left(1 - \frac{\omega}{R_1 V}\right) e^{-R_1 V} + B\left(1 - \frac{\omega}{R_2 V}\right) e^{-R_2 V} + \omega \rho e_{int}$$

with :

ρ : current density,

e_{int} : current internal energy per unit mass,

V : the ratio $\frac{\rho_{sol}}{\rho}$ where ρ_{sol} is the density of the explosive.

The coefficients A , B , R_1 , R_2 and ω are the model parameters, together with ρ_{sol} which represents the initial density of the explosive. R_1 , R_2 and ω are dimensionless parameters, but A and B have the dimensions of a pressure.

Once exhausted the solid explosive, and after the expansion of the resulting combustion gases, the above formula tends asymptotically towards the perfect gas law:

$$P = \omega \rho e_{int}$$

Therefore the parameter ω is related to the ratio γ between the specific heats of the gas, by the equation: $\omega = \gamma - 1$.

Often in the literature one finds the internal energy per unit volume E_o of the explosive, which also has the dimension of a pressure, like A and B . In order to find e_{int} it is sufficient to divide E_o by ρ_{sol} .

As an example, here are the values for TNT extracted from a publication of the Lawrence Livermore Laboratory by E. Lee, M. Finger, W. Collins (1973) :

$$\begin{array}{lll} A = 3.738 & B = 0.03747 & E_o = 0.0600 \\ R_1 = 4.15 & R_2 = 0.90 & \omega = 0.35 \end{array}$$

Pressures are in megabars, and the density of the solid TNT is $\rho_{sol} = 1630 \text{ kgm}^{-3}$.

If ROS is missing, it is assumed that the explosion starts from the solid material and $\rho_{sol} = \rho$ initial.

When doing an ALE calculation, it is necessary that all the fluids contained in the same domain (which may therefor mix up with one another) have the same constitutive law. However, the initial status may be different at different locations. For example, if the explosion takes place in air, one may use the JWLS model both for the explosive and for the air itself. For the latter, use the limiting perfect gas behaviour of the preceding equation by specifying the corresponding density and internal energy. In this case, one assumes that the γ values are the same for the air and for the combustion gases.

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density

ECR(3) : sound speed

9.7.17 CHOC—RANKINE-HUGONIOT SHOCK**Object**

Equation of state derived from the Rankine-Hugoniot equations, allowing to treat phenomena of shock wave propagations. It is based upon a linear relationship that links the wave propagation velocity to the velocity of fluid particles (empirical relation).

Syntax:

```
"CHOC"  "R0"  rho    "A"  a    "B"  b
...    < "PMIN" pmin >                /LECTURE/
```

rho

Initial density.

a,

Coefficients of the law.

pmin

Cavitation pressure (by default, pmin = 0).

List of the elements concerned.

Comments

Let c be the wave propagation speed and vp the velocity of the particles. These two velocities are related by the following linear relationship:

$$c = a + b * vp$$

The a quantity is the wave propagation speed at rest.

For details on this material, see the corresponding theoretical report.

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density

ECR(3) : sound speed

9.7.18 GPDI—DIFFUSIVE VAN LEER PERFECT GAS**Object:**

Diffusive Van Leer perfect gas.

Syntax:

```
"GPDI"    "ROIN" roin    "TINI" tini    "PINI" pini
...  <"PREF" pref>    "GAMA" gama    "CLAM" clam
...    "PLAM" plam    "CMU"  cmu      "PMU"  pmu
...    "CKAP" ckap    "PKAP" pkap    <"IGRA" igra>
...  <"TMUR" tmur>    <"QMUR" qmur>    <"ICLI" icli>
...  <"ECLI" eccli>    "DEBX" debx    <"DEBY" deby>
...                                     /LECTURE/
```

roin

Initial density of the mixture.

tini

Initial temperature of the mixture.

pini

Initial pressure of the mixture.

pref

Reference pressure.

gama

Ratio C_p/C_v of the gas.

clam

Constant λ (Lamé coefficient).

plam

Slope of λ .

cmu

Constant μ (Lamé coefficient).

pmu

Slope of μ .

ckap

Constant κ (conductivity)

pkap

Slope of κ .

igra

Choice of the gradient.

tmur

Wall temperature.

qmur

Wall flux.

icli

Boundary layer choice (0 or 1).

ecli

Thickness coefficient of the boundary layer.

debx

Horizontal mass flow rate in x (density * velocity = mass flow rate)

deby

Vertical mass flow rate in y (density * velocity = mass flow rate)

/LECTURE/

List of the concerned elements.

Comments:

This material may be used in EULERIAN only with the DPLA option.

If icli is equal 1, the boundary layer is taken into account.

If igrad is equal 0, the gradient is computed at the beginning of step n, if igrad is 1 the gradient estimation is done at the step $n + 1/2$. When igrad is 2, the gradient is computed at the end of the step.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : mass flow rate in x

ECR(5) : mass flow rate in y

ECR(18) : Mach number (u/c)

ECR(19) : temperature

ECR(20) : entropy

9.7.19 VAN LEER PERFECT GAS**Object:**

Euler: perfect gas ($P = \rho * (\gamma - 1) * E_{\text{interne}}$).

Syntax:

```
"GZPV"  "R0"  rho  "GAMMA"  gamma  "PINI"  pini  <"PREF" pref > ...  
        "DEBX" debx "DEBY" deby  <"DEBZ" debz>    /LECTURE/
```

rho

Initial density.

gamma

Ratio C_p/C_v (supposed constant).

pini

Initial pressure.

pref

Reference pressure.

debx

Horizontal mass flow rate in x (density * velocity = mass flow rate).

deby

Vertical mass flow rate in y (density * velocity = mass flow rate).

debz

Mass flow rate in z (density * velocity = mass flow rate).

/LECTURE/

List of the concerned elements.

Comments:

Material usable in EULERIAN or ALE, with the option DPLA or TRIDI.

The reference pressure $pref$ allow to define the initial state. if $pref = pini$, the gas is initially in equilibrium and will be perturbed by an external action, e.g. the motion of a piston. If $pref = 0$, the problem reduces to an initial stress problem whose value corresponds to $pini$. It is the case of a membrane separating two gases in different states, which disappears at the initial calculation time.

Outputs:

The different components of the ECR table are as follows:

- ECR(1) : pressure
- ECR(2) : density
- ECR(3) : sound speed
- ECR(4) : mass flow rate in x
- ECR(5) : mass flow rate in y
- ECR(6) : mass flow rate in z (in 3D)
- ECR(18) : Mach number (u/c)
- ECR(19) : temperature
- ECR(20) : entropy

9.7.20 ADCJ**Object :**

Modeling of the behaviour of a liquid-gas mixture. The mixture, which has 3 components, is assumed homogeneous and has 2 phases.

By this model it is possible to analyse the consequences of an explosion in a liquid contained within a tank in the presence of a cover gas.

This material is similar to material ADCR (page C.430) but the behaviour of the gas generated by the explosion is of type JWL instead of being a perfect gas (see page C.440).

The calculation is A.L.E. for elements CAR1 and TRIA (2D), or CUBE, PRIS and TETR (3D).

Syntax :

```
"ADCJ"  "ROLI" roli  <"PLIQ" pliq >  "CSON" cson
...  <"PSAT" psat   "ROSA" rosat >
...  "ROBU" robu    "A"    a          "B"    b
...  "R1"  r1       "R2"  r2          "OMEG" omega
...  "EIBU" eibu    <"PARA" param > <"ROBZ" robz >
...  "ROGA" roga    "PGAZ" pgaz       "GAMG" gamg
...  "PTOT" ptot    "CBU"  cbu        "CGAZ" cgaz
...  <"PREF" pref > <"VILI" muli  >
...  <"CMIN" cmin >  /LECTURE/
```

roli, pli

Initial density and pressure of liquid, assumed alone, defining the equation of state for the liquid. If "PLIQ" is absent, the assumed pressure is that of the cover gas.

cson

Sound speed in the liquid.

rosat, psat

Density and pressure of saturated liquid vapor, at the temperature of the liquid, defining the equation of state for the vapor. These parameters are coupled: if one is present, the other must also be given. If they are missing, they are assumed to be zero.

robu

Initial density of the bubble gas, assumed alone.

a, b ,r1 ,r2, omeg

Coefficients of the JWL state equation for the bubble (see C.440).

eibu

Initial specific internal energy of the bubble gas.

param

Number of integrations to solve the adiabaticity condition for a JWL gas. By default, param = 100, suggested value for a good precision level.

robz

Density of the bubble gas, defining the equation of state for the gas. By default, robz is equal to robu.

roga, pgaz

Initial density and pressure of the cover gas, assumed alone, defining the state equation of the gas.

gamg

Ratio C_p/C_v of the cover gas.

pref

Reference pressure. By default, one takes that of liquid.

cmin

Maximum mass concentration of gas in the liquid so that it is considered to be pure (1.E-8 by default).

mul

Dynamic viscosity of liquid.

ptot

Total pressure (see also the comments). In the case of the JWL gas, the pressure is re-computed strating from the energy.

cbu

Initial mass concentration of the bubble gas (0 or 1).

cgaz

Initial mass concentration of the cover gas (0 or 1).

/LECTURE/

List of the concerned elements.

Remarks :

This material has been initially developed to treat the behaviour of fast reactors in the case of a containment accident (ADC in French), or of a hypothetical core disruptive accident (HCDA in English), but the adopted method lends itself to a more general use.

For this reason, some aliases are introduced. For the liquid, ROLI, PLIQ, CSON, VILI, are identical to RONA, PNA, CNA, VISO (sodium). For the cover gas, ROGA, PGAZ, GAMG, CAR are identical to ROAR, PARG, GAR and CAR (argon).

Comments :

The mesh will be subdivided into three zones (bubble, liquid, cover gas) and the ADCJ material will be listed three times by varying each time the initial concentrations of the bubble and of the cover gas (0 or 1 depending on the case), but the other parameters must be identical, in order to have the same constitutive laws for the components of each zone. Then, starting from these concentrations and from the total pressure p_{tot} , EUROPLEXUS will compute the density of the mixture. EUROPLEXUS will also recompute the concentrations in the gases in order to account for the liquid vapor, if p_{sat} is not zero.

It is possible, however, to use more than 3 zones, if for example the bubble or the liquid are subdivided into concentric zones. In such cases the total pressure p_{tot} , varying from zone to zone, will define the initial state. EUROPLEXUS will automatically compute the mixture density using the constitutive law of each component. Of course, also in this case the other parameters such as $robu$, pbu , $rona$, $roar$, ... will have to be identical.

The gas generated by the explosion is assumed to follow a JWL behaviour. On this subject, see page C.440. The state variables are linked by the following equation:

$$P_B = A(1 - \frac{\Omega\eta}{R_1})e^{-\frac{R_1}{\eta}} + B(1 - \frac{\Omega\eta}{R_2})e^{-\frac{R_2}{\eta}} + \Omega\rho_B E_B$$

The variable η is the reduced mass density ($\eta = \frac{\rho_B}{\rho_{B0}}$), and E_B the specific internal energy.

The other parameters (A , B , R_1 , R_2 , Ω) are characteristic constants of the gas under consideration. It is remarked that if $A = B = 0$ and $\Omega = \gamma_B - 1$, the equation of state of a perfect gas is recovered:

$$P_B = (\gamma_B - 1)\rho_B E_B$$

The keywords **ROBU**, **EIBU**, **A**, **B**, **R1**, **R2** and **OMEG** allow to specify, respectively, the initial density ρ_{B0} , the initial internal energy E_{B0} , and the 5 constants of the JWL law.

The state variables P_B , ρ_B and E_B evolve during the calculation but remain linked by the state equation. In addition, it is assumed that the transformation followed by the bubble gas

is adiabatic, which allows to eliminate one of these state variables, e.g. E_B . One then recovers an equation of the form:

$$P_B = f(\rho_B)$$

The other parameters are identical to those of the ADCR material (see page C.430).

One may obtain the work of pressure forces in a region of the mesh by means of the "PDV" keyword of the "REGION" directive (page G.100).

For more informations see reference [\[488\]](#).

Outputs :

The components of the ECR table are as follows:

- ECR(1) : absolute pressure
- ECR(2) : density of the two-phase mixture
- ECR(3) : sound speed in the mixture
- ECR(4) : mass concentration of the cover gas
- ECR(5) : mass concentration of the bubble gas
- ECR(6) : cover gas mass per unit volume
- ECR(7) : bubble gas mass per unit volume
- ECR(8) : mass increment of the cover gas
- ECR(9) : mass increment of the bubble gas
- ECR(10) : volume occupation rate of the gases
- ECR(11) : partial pressure of the cover gas
- ECR(12) : partial pressure of the bubble gas if $\text{ecr}(5) > \text{cmin}$, else initial pressure
- ECR(13) : density of the cover gas
- ECR(14) : density of the bubble gas if $\text{ecr}(5) > \text{cmin}$, else initial density

9.7.21 FLUID PARTICLE**Object:**

This directive allows to specify a fluid behaviour for the particle elements (BILLE). The fluid is isothermal and perfect.

Syntax:

```
"BILLE" "FLUIDE" "R0" rho "C" c <"PINI" pini>
      <"PREF" pref > <"PMIN" pmin >
      <"VISC" mu > /LECTURE/
```

rho

Density.

c

Sound speed (constant).

pini

Value of the initial pressure.

pref

Value of the reference pressure.

pmin

Value of the minimum pressure (by default, $pmin = 0$).

mu

Dynamic viscosity.

/LECTURE/

List of the concerned elements.

Comments:

The first two parameters are mandatory. If the initial pressure is given, EUROPLEXUS will take it into account for the calculation of the absolute pressure (ECR(1)).

Rôle of PREF:

When the reference pressure is different from the initial pressure, the fluid is initially not in equilibrium. It is the case of a membrane which breaks at $t = 0$, thus releasing a compressed fluid.

In numerous problems the focus is on acoustic effects, and one assumes that a fluid initially in equilibrium evolves under the effects of a loading; in such cases one will take $\text{pref} = \text{pini}$.

If "PREF" is omitted, EUROPLEXUS considers that the fluid is in equilibrium and that $\text{pref} = \text{pini}$ (even in the case that $\text{pini} = 0$).

The pressure in the fluid will always remain greater than or equal to the minimum pressure pmin , even though the density diminishes. This is a very simplified way of treating cavitation. The default value of pmin is $\text{pmin} = 0$.

If "PINI" is omitted, the initial pressure is null.

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density

9.7.22 PRGL—POROUS JELLY**Object :**

This directive allows to specify a behaviour simulating porous jelly (used in the bird impact studies onto turbine blades) for particle elements and SPH. For the description of this material, please consult the thesis of Antoine Letellier.

Syntax:

```

"PRGL"  "R01" rho1    "R02" rho2    "GAMM" gamma      ...
...     "CSN1" csn1   "CSN2" csn2   "CVT1" cvt1      "CVT2" cvt2      ...
...     "PROP" prop   "PINI" pini   "PREF" pref      "PMIN" pmin      ...
...     <"VISC" mu >                                     ...

...                                           /LECTURE/

```

rho1

Density of the fluid.

rho2

Density of the gas.

gamma

Perfect gas constant for the gas.

csn1

Sound speed for the fluid.

csn2

Sound speed for the gas.

cvt1

Coefficient expressing a linear relationship between the shock speed and the impact velocity for the fluid (see comments below).

cvt2

Idem for the gas.

prop

Fluid proportion in the mixture (in volume).

pini

Value of the initial pressure.

pref

Value of the reference pressure.

pmin

Value of the cavitation pressure.

mu

Dynamic viscosity.

/LECTURE/

List of the concerned elements

Comments:

The coefficients cvt1 and cvt2 are used in the following equation:

$$V_{\text{choc}} = V_{\text{son}} + \text{cvt} * V_{\text{impact}}$$

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density

9.7.23 VAN DER WAALS GAS

Object :

This directive allows to specify a real gas material with a Van Der Waals behaviour.

In the 1-D case, the friction onto the walls may be accounted for, and the dissipated energy heats up the gas (modification of the internal energy). To this end, one must add a "PARO" material and associate it with "GVDW" by means of a "MULT" material (pages C.370 and C.380).

Syntax :

```
"GVDW"  "RO"  rho  "PINI"  pini  < "PREF"  pref >  ...
...     "RM"  rm   "CV"    Cv     ...
...     $[ "A"  a   "B"    b   ; "PCRI" pc  "TCRI" tc ]$ ...
...           /LECTURE/
```

rho

Initial density.

pini

Initial pressure.

pref

Reference pressure.

rm

Constant of perfect gases R divided by the molecular mass M (assumed constant).

Cv

Specific heat at constant volume.

a

Constant a of the gas (assumed constant).

b

Constant b of the gas (assumed constant).

pc

Pressure at the critical point.

tc

Temperature at the critical point.

/LECTURE/

List of the concerned elements.

Comments :

The state equation of a Van der Waals gas has the form:

$$P = \frac{r_m T}{\left(\frac{1}{\rho} - b\right)} - a\rho^2$$

or, equivalently:

$$P = \frac{r_m(e + a\rho)}{C_v\left(\frac{1}{\rho} - b\right)} - a\rho^2$$

The first form is obtained by the state variables P , ρ and T (respectively pressure, density and temperature), while the second uses P , ρ and e , i.e. the temperature is replaced by the specific internal energy.

Some theoretical complements are given in the report [517], which lists also the values of parameters a and b for some fluids.

These coefficients a and b are related to the critical pressure P_c and to the critical temperature T_c by the following expressions :

$$P_c = \frac{a}{27b^2} \quad T_c = \frac{8a}{27br_m}$$

The parameter r_m is the ratio between the perfect gas constant ($R=8.31441 \text{ JK}^{-1}\text{mol}^{-1}$) and the molar mass M of the gas. Pay attention to units: for example, for air it is $M = 0.029 \text{ Kg}$.

The reference pressure p_{ref} allows to define the initial state. If $p_{ref} = p_{ini}$, the gas is initially in equilibrium and will be perturbed by an external action, e.g. motion of a piston. If $p_{ref} = 0$, the problem reduces to a calculation with initial stresses, defined by p_{ini} . It is the case of a membrane, that initially separates two gases in different conditions, which suddenly breaks at the initial time.

Outputs :

The various components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : temperature

ECR(5) : dynamic pressure: $(\frac{1}{2}\rho V^2)$

ECR(6) : total energy $(e + \frac{1}{2}V^2)$.

9.7.24 JWLS**Object:**

State equation of type JWL (Jones-Wilkins-Lee), allowing to treat explosion phenomena by accounting for the detonation speed in the medium. The user may specify the initiation point of the detonation process.

Syntax:

```
"JWLS"  "RO"  rho  <"ROS"  ros>  "PINI"  pini  <"PREF"  pref>  ...
...  "A"   a      "B"   b      "R1"   r1      "R2"   r2      ...
...  "OMEG" omeg      "EINT" eint  <"BETA"  beta  >      ...
...  < "D"   d      "XDET" xdet  "YDET" ydet  <"ZDET" zdet> >  ...
...      /LECTURE/
```

rho

Initial density.

pini

Initial pressure.

pref

Reference pressure.

a,b,r1,r2,omeg

Coefficients of the JWL equation of state.

d

Detonation velocity.

eint

Initial specific internal energy.

beta

Reduced damping coefficient for high frequencies. It is zero by default, and should always be very small (< 0.05).

ros

Density of the explosive (solid form).

xdet

Coordinate along x of the detonation point.

ydet

Coordinate along y of the detonation point.

zdet

Coordinate along z of the detonation point.

/LECTURE/

List of the concerned elements.

Comments:

The JWL state equation gives the value of the pressure according to the following formula (see also page C.440) :

$$P = A\left(1 - \frac{\omega}{R_1 V}\right) e^{-R_1 V} + B\left(1 - \frac{\omega}{R_2 V}\right) e^{-R_2 V} + \omega \rho e_{int}$$

with :

ρ : current density,

e_{int} : current internal energy per unit mass,

V : the ratio $\frac{\rho_{sol}}{\rho}$ where ρ_{sol} is the density of the explosive.

If **ROS** is omitted, it is assumed that the explosion starts from the explosive before ignition. In this case **ROS** = **RO** and the initial density is that of the solid. In this case it is mandatory to specify the detonation speed **D** and the ignition point, in order to start the reaction.

The coordinates **xdet**, **ydet** (and **zdet** in 3D) define the ignition point where the reaction begins. From this point the reaction progresses with the velocity **D**, and when the shock wave reaches the center of each finite element the reaction is activated.

Like for material **JWL** it is possible to use this constitutive law to approximate the behaviour of a perfect gas. It is sufficient to give **ROS** and the initial values **RO** and **EINT** for the gas to find the corresponding pressure by the state equation. In this case **PINI** is not used by the model. In this case, it is useless to specify **D** and the coordinates of the ignition point.

Like for the other fluid models, the reference pressure **PREF** allows to account for an external pressure.

Outputs:

The components of the ECR table are as follows:

ECR(1) : absolute pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : fraction burned (between 0 and 1).

9.7.25 USER-DEFINED FLUID (FLUT)

Object:

This material is used with specialised elements of type "FLU1", "FLU3", "FL23", "FL24", "FL34", "FL35", "FL36" and "FL38". The subroutine describing the fluid's equation of state has to be written by the user (an example is given below).

Syntax:

```
"FLUT"  "RO" ro  "EINT" eint
        $[ "GAMM" gamm ; R r C0 c0 C1 c1 C2 c2 ]$
        < "CL" cl  "CQ" cq  "PB" pb  "PMIN" pmin
          "AHGF" ahgf  "ITER" iter  "ALF0" alf0
          "BET0" bet0  "KINT" kint  "NUM" num
          "VXFF" vxff  "VYFF" vyff  "VZFF" vzff
          "CONV" conv  "PREF" pref  "RREF" rref
          "RMIN" rmin  "RMAX" rmax
          "IMIN" imin  "IMAX" imax  "RANG" rang
          "GENE" gene  "GENM" genm                >
        < additional_data_for_JWLS_material      >
        /LECTURE/
```

ro

Initial mass density.

eint

Specific internal energy.

gamm

Constant value of the ($\gamma = c_p/c_v$) ratio; alternatively (but only for a perfect gas, i.e. NUM=1), the next four parameters may be given which are used to describe the dependence of c_p , and hence c_v , γ and the internal energy of the perfect gas upon temperature. To this end, we use the following relations: $c_p(T) = c_0 + c_1T + c_2T^2$ where T is the temperature in K, c_p and c_0 are typically expressed in J/kgK, c_1 in J/kg(K2) and c_2 in J/kg(K3). Then, the specific heat at constant volume c_v is given in J/kgK by the expression $c_v(T) = c_p(T) - R$, where R is the specific perfect gas constant of the gas considered, also expressed in J/kgK. Note that $R = R'/w$ where R' is the universal constant of perfect gases expressed in J/kmolK (which has the standard value of 8314.3) and w is the

molar weight of the gas expressed in kmol/kg. Alternativley, one could express c_p and c_0 in J/kmolK, c_1 in J/kmol(K2) and c_2 in J/kmol(K3). In this case, c_v would result in J/kmolK from the expression $c_v(T) = c_p(T) - R'$, where R' is the universal constant of perfect gases defined above.

cl

Coefficient of linear artificial viscosity. Range is 0.0 to 0.8, default is 0.0.

cq

Coefficient of quadratic artificial viscosity. Range is 0.0 to 4.0, default is 0.0.

pb

Constant used in sound speed evaluation. For material 9 this is the reference pressure. For material 10 this is the index of the FONC used to describe the imposed pressure as a function of time.

pmin

Cut-off pressure. Default is 0.0.

ahgf

Anti-hourglass coefficient. Suggested value is 0.01, default is 0.0.

iter

There are two possibilities. If a number greater than or equal to 1 is given, then this quantity represents the (fixed) number of iterations for the calculation of tilde pressure and tilde specific energy. If, instead, one gives a (real) value smaller than 1 (for example **ITER 1.E-6**), then this quantity is interpreted as the tolerance for convergence of the tilde pressure and tilde specific energy, and the maximum number of iterations for convergence is in that case limited to 100. Finally, note that if one does not specify this quantity (or specifies 0), then by default just one iteration is performed.

alf0

Parameter used to compute the donor element weighting factor in connection with mass and energy transport across element boundaries. When **alf0=0** mass and energy fluxes are centered; when **alf0=1** the fluxes are full donor.

bet0

Parameter used to compute the donor element weighting factor in connection with momentum transport across element boundaries. When **bet0=0** momentum flux is centered; when **bet0=1** the flux is full donor.

kint

In the 2D case: represents the integration type for the forces due to momentum transport (FL24 element only) (-1: no momentum transport at all, to be used only for debugging purposes; 0: use centroid values, is the default; 1: use values at integration points, i.e. a 2×2 Gauss rule; 2: use a 3×3 Gauss rule, which is exact in axisymmetric cases). Note that the default value (kint = 0) is known to destroy symmetry even in 2D plane cases, so use at least kint = 1 if a perfectly symmetric solution is desired (e.g. in academic validation tests). In the 3D case: represents the integration type for nodal mass distribution (0: element mass/number of nodes (default); 1: Gauss integration).

num

Number of the material in the user's subroutine FLUTIL (see below). By default, num=1.

vxff

X-Component of the far-field velocity (this and the following two components are only used in the case that the FLUT material is used to specify a far-field status via a CLxx element); by default it is 0.0.

vyff

Y-Component of the far-field velocity; by default it is 0.0.

vzff

Z-Component of the far-field velocity; by default it is 0.0.

conv

Units conversion factor for the pressure; it is used only in empirical or semi-empirical user's laws; by default it is 1.0.

pref

Reference pressure (see Note below), by default it is 0.0.

rref

Reference mass density to be used in the material law; by default, it is assumed rref=ro, i.e. the reference density is assumed equal to the initial density.

rmin

The minimum value of the density that can be accepted; by default it is 0.

rmax

The maximum value of the density that can be accepted; by default it is set equal to the 'grand' quantity (usually of the order of 1.E12, but the actual number depends on the computer).

imin

The minimum value of the specific internal energy that can be accepted; by default it is 0.

imax

The maximum value of the specific internal energy that can be accepted; by default it is set equal to the 'grand' quantity (usually of the order of 1.E12, but the actual number depends on the computer).

rang

An integer defining the type of check that should be performed. When 0 is used no range check is performed (this is the default); when 1 is specified, a warning message is issued each time the validity ranges are violated but, in order to reduce the number of messages, the current value of the range is updated each time a message is issued. The last messages will therefore show the absolute minimum or maximum out-of-range values that have been used in the calculation; when 2 is specified, a warning message is issued each time the validity ranges are violated, so there may be a lot of messages in practical cases; finally, when 3 is used an error message is given the first time ranges are violated, and the calculation is stopped immediately.

gene

Associate energy generation to this material: gene is the number of the function (see "FONC" directive) used to describe the variation in time of the specific power (i.e., per unit mass) that will be generated in this FLUT material.

genm

Associate mass generation to this material: genm is the number of the function (see "FONC" directive) used to describe the variation in time of the specific mass (i.e., per unit mass) that will be generated in this FLUT material.

Additional data for JWLS material

For the special case of a Jones-Wilkins-Lee (JWLS) material (NUM = 11), the user may specify a set of additional parameters that are detailed below. This material is suitable for the modelling of explosives and is similar to CEA's JWL/JWLS materials, but adapted for use with JRC's specialized fluid elements.

The JWL state equation gives the value of the pressure according to the following formula (see also page C.440) :

$$P = A\left(1 - \frac{\omega}{R_1 V}\right) e^{-R_1 V} + B\left(1 - \frac{\omega}{R_2 V}\right) e^{-R_2 V} + \omega \rho e_{int}$$

with :

ρ : current density,

e_{int} : current internal energy per unit mass,

V : the ratio $\frac{\rho_{sol}}{\rho}$ where ρ_{sol} is the density of the explosive (specific volume).

The release of the chemical energy can be controlled by the burn mass fraction. The burn mass fraction smears the detonation front over a certain number of time steps.

$$P = P_{EOS} \min(1, F_1)$$

with :

$$F_1 = \begin{cases} (t - t_1)d/(B_s \cdot l_e) & \text{for } t > t_1 \\ 0 & \text{for } t \leq t_1 \end{cases}$$

with :

t_1 : ignition time of the current element (calculated with the detonation velocity d),

B_s : controls the width of the burn wave

l_e : average element length

Syntax:

```
< <"ROS" ros>  "A" a  "B" b
    "R1" r1 "R2" r2
    <"D" d  "XDET" xdet "YDET" ydet <"ZDET" zdet>>
    <"TDET" tdet> "PINI" pini <"BMF" bmf> >
```

ros

Density of the explosive in solid state (before detonation). If omitted, it is assumed **ROS** = **R0** i.e. the explosion starts from the solid state, whose density coincides with the initial density of the material (**R0** parameter given in the standard **FLUT** properties). In this case, it is mandatory to specify also the detonation speed **D** and the ignition point **XDET**, **YDET** and **ZDET** from which the detonation starts.

a

Coefficient A of the JWL equation of state. It has the dimensions of a pressure.

b

Coefficient B of the JWL equation of state. It has the dimensions of a pressure.

r1

Coefficient R_1 of the JWL equation of state. It has no dimensions.

r2

Coefficient R_2 of the JWL equation of state. It has no dimensions.

d

Detonation speed. The detonation is supposed to start at point **XDET**, **YDET** and **ZDET** and to propagate through the (solid) charge according to a spherical wave traveling with constant speed given by **d**.

xdet, ydet, <zdet>

Ignition point. If **zdet** is omitted, it is assumed equal to 0.

tdet

Starting time for the detonation. By default it is 0.

pini

Initial pressure (absolute) of the undetonated material.

bmf

Parameter for the burn mass fraction B_S . If the parameter equals 0 or is not defined, burn mass fraction is not activated. The suggested value is 2.5.

The parameter ω appearing in the JWL equation of state is not given explicitly since $\omega = \gamma - 1$ where $\gamma = C_p/C_v$ is the ratio of specific heats that is provided by means of the **gamm** parameter.

Like for CEA's materials JWL and JWLS it is possible to use this constitutive law to approximate the behaviour of a perfect gas. It is sufficient to give **ROS** and the initial values **RO** and **EINT** for the gas to find the corresponding pressure by the state equation. In this case **PINI** is not used by the model. In this case, it is useless to specify **D**, the coordinates of the ignition point and the starting time of the detonation.

Comments

The meaning of the reference pressure **pref** is as follows. The pressure value **p** resulting from the material equation of state is considered as the absolute pressure. Its value is stored in **ECR(1)**, see below. However, in order to compute the internal forces, the "effective" pressure value ($p' = p - \text{pref}$) is used. This is useful when e.g. modeling a pressure vessel filled in by fluids and surrounded by the atmosphere in the outside part. In this case, if the material law for the

inside fluid is given in absolute terms, we may proceed in two ways in order to take into account the outside pressure. The first possibility is to ignore `pref` (thus, `pref=0.0`) and explicitly impose an outside pressure of 1 atmosphere on the external surface of the model, either by the directive "CHAR" or by using specialized CLxx elements. However, this is likely to be too expensive for just representing a constant external pressure. The second possibility is to define `pref=1` atmosphere for the internal fluids, so that only the pressure exceeding this value will be used in order to compute the forces.

If a material of this type is assigned to far-field boundary condition elements only, then all the above optional parameters are ignored if specified, except NUM, VXFF, VYFF and VZFF. On the other hand, VXFF, VYFF and VZFF are only used when the material is assigned to a far-field (CLxx) element.

The optional directives GENE and GENM allow to associate energy or mass generation, respectively, to a FLUT material. During the transient calculation, the generation "follows" this material, rather than being tied to a spatial zone (elements) like in the INJE injection model (see INJE directive). Another difference is that in the GENE model the given time function (FONC) represents the specific generated energy per unit time, and not the total energy per unit time like in the INJE model.

It should be noted that the behaviour of the GENE/GENM generation model will be different, depending on whether the associated FLUT material is a single-component, or part of a multi-phase multi-component (FLMP) material model.

In the first case, the generation will actually remain confined to the elements that are assigned the present material in the initial configuration, which is probably NOT what the user would expect (unless of course the generation zone boundary is Lagrangian, but in that case the generation model would coincide with the INJE model). This 'wrong' behaviour is due to the way the program deals with fluid transport across boundaries in the single-component model: even when fluid is transported each element retains its (initial) material index, so the generation information may not be transported to neighbours.

On the other hand, when the FLMP multiphase multicomponent model is used, then full tracking of each single component is performed, and therefore the generation model works as expected (generation information is transported to neighbour elements).

Mass generation requires some conventions as to what are the conditions of the injected mass, since this mass will also introduce corresponding energy, in the form of internal energy and possibly of momentum. Here it is assumed for simplicity that the mass generation occurs in the initial conditions of the corresponding material, and that the injection process does not perturbate the velocity field (i.e., the mass enters the element already at the current velocity).

Outputs:

The components of the ECR table are as follows:

ECR(1): current element fluid absolute pressure p

ECR(2): current element mass density

ECR(3): current element sound speed

ECR(4): current element specific internal energy

ECR(5): current element bulk modulus. Attention! Until recently this quantity was uncorrectly indicated as bulk modulus in the code. In reality, this is the derivative of pressure with respect to density, dp/dr , which is related to the true bulk modulus B by the relation: $B = (dp/dr) * r$. Note that the quantity is only used (and updated at each time step) if the user selects the anti-hourglass model, i.e. specifies a non-zero value for the AHGF coefficient.

ECR(6): current element pseudoviscous pressure

ECR(7): current element minimum pressure flag (0 if $p > p_{min}$, 1 if $p = p_{min}$)

ECR(8): maximum pressure ever experienced by this element

ECR(9): minimum pressure ever experienced by this element

ECR(10): fraction of detonated material (for JWLS only) (0 or 1)

The components of the SIG table are as follows:

SIG(1): current element fluid relative pressure ($p - p_{ref}$)

The following global results can be accessed via TPLLOT when materials having either energy or mass generation are present in the calculation:

GENE: Total energy generated in materials with generation

GENM: Total mass generated in materials with generation

Comparing FE and FV solutions.

Frequently it is desirable to compare solutions obtained with Finite Element (FE) models to equivalent ones obtained with Finite Volumes (FV), typically for the case of perfect gases.

In the case of MCGP material used for Finite Volumes (see page C.550), the equivalence of initial conditions is not completely straightforward.

A procedure for converting between the two formulations is detailed on Page C.550.

Example of user-defined material subroutine.

The name of the subroutine is FLUTIL. The variables received in input and returned in output are explained in the comments.

```

C FLUTIL      SOUPLEX   ISPRA      89/04/19    21:02:26
      SUBROUTINE FLUTIL(NUM,rcur,rref,GAM,UP,PB,PMIN,conver,
        >                pres,SOUND,XKP)
C-----
C
C                                     F.CASADEI,J.P.HALLEUX 11-87
C USER'S EQUATION(S) OF STATE FOR FLUID MATERIAL(S) OF TYPE "FLUT"
C UPDATES FLUID PRESSURE, SOUND SPEED AND MINIMUM PRESSURE FLAG
C-----
c Input:
c =====
C   NUM      = NUMBER OF CURRENT "FLUT" MATERIAL
C   rcur     = CURRENT ELEMENT MASS DENSITY
C   rref     = reference MATERIAL MASS DENSITY
C   GAM      = num=1-8: CP/CV, RATIO OF MATERIAL SPECIFIC HEATS
c             num=9 : Bulk modulus (same units as pressure)
C   UP       = CURRENT ELEMENT SPECIFIC INTERNAL ENERGY
C   PB       = num=1-8: CONSTANT USED IN SOUND SPEED EVALUATION
c             num=9 : Reference pressure
c             num=10: Index of FONC defining the pressure p(t)
C   PMIN     = CUT-OFF PRESSURE
c   conver   = units conversion factor for the pressure
c Output:
c =====
C   pres     = NEW      ELEMENT FLUID PRESSURE: pres=pres(rcur,UP)
C   SOUND    = NEW      ELEMENT SOUND SPEED
C   XKP      = NEW      ELEMENT MINIMUM PRESSURE FLAG (0 IF P>PMIN,
C                                     1 IF P=PMIN)
C
C       include 'R8AHOZ.INC'
c
C       include 'ALLO.INC'
C       include 'CONSTA.INC'
C       include 'TEMPX.INC'
C
C=====
C EQUATIONS OF STATE
C
C       GOTO(1,2,3,4,5,6,7,8,9,10),NUM
C       CALL ERRMSS('FLUTIL',' FLUT MATERIAL NUMBER OUT OF RANGE')
C       STOP
C
C 1/ PERFECT GAS -----
C
C       1 pres=(GAM-1.D0)*rcur*UP
C       GO TO 100

```

```
C
C 2/ IT8 LOW DENSITY EXPLOSIVE CHARGE -----
C   (pinc=[DYNES/CM2])
C
  2 v=rref/rcur
    T1=1.7039D+11*(1.0D0-1.0D0/(90.0D0*V))*EXP(-9.0D0*V)
    T2=1.1595D+10*(1.0D0-1.0D0/(24.0D0*V))*EXP(-2.4D0*V)
    t3=-1.02884d9/v
    t4=0.1D0*rcur*up
    pinc=t1+t2+t3
    pres=conver*pinc+t4
    GO TO 100
C
C 3/ APRICOT-4 AND IT8 LIQUID WATER -----
C   (pinc=[DYNES/CM2])
C
  3 v=rref/rcur
    if(rcur.eq.rref) then
      t1=0.0d0
    ELSE
      am=1.0D0-V
      a2=am*am
      a3=2.086D0*am
      a3m=a3-1.0D0
      a4=a3m*a3m
      a5=0.8293D0*a2
      a6=2.796D0*am
      a7=sqrt(a4+a5)
      om=(a3-1.0D0+a7)/a6
      a8=(0.1483D0+2.086D0*om-1.398D0*om*om)*om
      a9=1.0D0-0.14D0*am/v
      t1=1.0D12*a9*a8
    END IF
    t2=0.28d0*rcur*up/v
    pinc=t1
    pres=conver*pinc+t2
    GO TO 100
C
C 4/ CONT PROBLEM EXPLOSIVE BUBBLE -----
C   (pinc=[Pa])
C
  4 VV=rcur/rref
    IF(VV.GT.0.0D0) THEN
      pinc=1.0D7*VV**GAM
    ELSE
      pinc=0.0D0
```

```
ENDIF
pres=conver*pinc
GO TO 100

C
C 5/ CONT PROBLEM LIQUID SODIUM AT 773 K AND ABOUT 10 MPA -----
C   (pinc=[Pa])
C
  5 VV=rcur/rref
    AMU=VV-1.0D0
    T1=4.440D3*AMU
    T2=4.328D9*AMU*ABS(AMU)
    T3=1.218D0*rcur*UP*(1.D0+AMU)
    pinc=t1+t2
    pres=conver*pinc+t3
    GO TO 100

c
C 6/ APRICOT-4 EXPLOSIVE GAS PRODUCTS -----
C   (pinc=[DYNES/CM2])
c
  6 v=rref/rcur
    t1=6.70695d12*(1.0d0-0.25d0/(4.660599d0*v))*exp(-4.660599d0*v)
    t2=9.26460d10*(1.0d0-0.25d0/(0.991617d0*v))*exp(-0.991617d0*v)
    t3=0.25d0*rcur*up/v
    pinc=t1+t2
    pres=conver*pinc+t3
    go to 100

c
C 7/ WTO LOW-DENSITY EXPLOSIVE CHARGE -----
C   (pinc=[DYNES/CM2])
c
  7 v=rref/rcur
    t1=1.7039D11*(1.d0-1.d0/(90.0d0*v))*exp(-9.0d0*v)
    t2=1.1595D10*(1.d0-1.d0/(24.0d0*v))*exp(-2.4d0*v)
    t3=0.1d0*rcur*up
    pinc=t1+t2
    pres=conver*pinc+t3
    GO TO 100

c
C 8/ WTO LIQUID WATER -----
C   (pinc=[DYNES/CM2])
c
  8 vv=rcur/rref
    if(vv.ge.1.d0) then
      t1=1.2222D11*vv*vv+5.1562D10-(1.2222D11+5.1562D10)*vv**1.28D0
    else
      t1=-1.2222D11*vv*vv-8.5937D10+(1.2222D11+8.5937D10)*vv**1.28D0
```

```
        endif
        pinc=t1
        pres=conver*pinc
        GO TO 100
c
C 9/ Liquid with bulk response -----
c
    9 v=rref/rcur
      eta=1.0d0-v
      p0=pb
      bulk=gam
      pres=p0+bulk*eta
      go to 100
c
C 10/ Imposed pressure -----
c
    10 ifonc=pb
      tcur=t
      call ffonct(ifonc,tcur,valfon,a(n91),a(n92))
      pres=valfon
      go to 100
C
c=====
C PRESSURE CUT-OFF TEST
C
    100 IF(pres.LE.PMIN) THEN
      pres=PMIN
      XKP=1.D0
    ELSE
      XKP=0.D0
    ENDIF
C
c=====
C SOUND SPEED
C
      goto(101,101,101,101,101,101,101,101,102,103),num
c
c materials 1-8
c
    101 SOUND=sqrt(GAM*(pres+PB)/rcur)
      go to 999
c
c material 9
c
    102 sound=sqrt(bulk/rcur)
      go to 999
```

```
c
c material 10
c
  103 sound=zero
    go to 999
C
  999 RETURN
    END
```


9.7.26 MATERIAL FOR MINERAL OIL PYROLISIS

Object

This material is used to simulate mineral oil pyrolysis phenomena subsequent to electrical arcs in oil-filled electrical apparatuses (e.g. transformers).

The material is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC.

Rather than a new material, it is an extension of the user-defined FLUT type material (see Page C3.520). The extension is introduced by the keyword "PYRO", as explained in the syntax below.

References

More information on the formulation of this material model may be found in references [106, 160].

Syntax

```
"FLUT"  FLUT_material_data (see P. C.520)
  < "PYRO" "NC" nc "EACT" eact "RGAS" rgas "TREF" tref
      "ROIL" roil "KOIL" koil "TOIL" toil "PINI" pini
      "TINI" tini "QTAB" qtab "DHR0" dhro
      "NGAS" ngas
      ngas * ( "GAS" 'nomgas' "STEC" stec "PMOL" pmol
                "DHR" dhr "CP" cp )
      "CARB" 'nomcar' "STEC" stec "PMOL" pmol
                "DHR" dhr "CP" cp
  >
/LECTURE/
```

nc

Number of carbon atoms in oil molecule.

eact

Activation energy.

rgas

	Gas constant.
tref	
	Reference temperature.
roil	
	Liquid oil density.
koil	
	Liquid oil thermal conductivity.
toil	
	Liquid oil temperature.
pini	
	Initial pressure of the bubble.
tini	
	Initial temperature of the bubble.
qtab	
	Index of "FONCTION" for electrical power. This function will describe the power as a function of time.
dhro	
	Liquid oil enthalpy of formation.
ngas	
	Number of gas products.
nomgas	
	Name of gas product (8 chars max.).
stec	
	Stechiometric coefficient for this product in the pyrolysis reaction.
pmol	
	Molar weight of the product.
dhr	
	Molar enthalpy of formation of the product.
cp	

Heat capacity at constant pressure of the product.

nomcar

Name of the carbon product.

Comments:

This model is still under development and testing and should therefore be used with great care.

Do not forget to dimension adequately, see keyword "PYRO", page A.70. Currently there may be up to 4 distinct pyrolysis bubbles in a calculation.

9.7.27 ADVECTION-DIFFUSION FLUID (ADFM)**Object:**

This material is used with specialised elements of type "ADC8" or "ADQ4", for advection-diffusion calculations.

Syntax:

```
"ADFM"  "RO" rho  "EXPA" expa  "TREF" tref  
        < "COND" cond  "CAPA" capa  "VISC" visc  
          "LAPI" lapi                                     >  
        /LECTURE/
```

rho

Initial mass density.

expa

Volumetric expansion coefficient.

tref

Reference temperature at which volumetric expansion is null.

cond

Thermal conductivity. By default, is f(T).

capa

Thermal capacity. By default is f(T).

visc

Dynamic viscosity. By default is f(T).

lapi

Lapudus viscosity. Default is 0.0.

Comments:

If omitted, thermal conductivity, capacity and viscosity are assumed to be temperature dependent. The user must insuch cases supply routines that return temperature dependent values.

9.7.28 MULTICOMPONENT FLUID MATERIAL (MCGP)**Object:**

This material is used with specialised elements of type MCxx for multicomponent fluid flows. A mixture of calorically perfect gases is assumed, i.e. the internal energy is a function of the temperature only. This function may be different for each component and has a generic polynomial form.

The contribution of the CESI team (formerly at ENEL, Milano) to the development of this material model in collaboration with JRC is gratefully acknowledged.

Syntax:

```
"MCGP"  "NCOM" ncom  "R" r
         ncom * ( "COMP" 'nomcomp'
                  "PM" pm
                  "CV1" cv1 "CV2" cv2 "CV3" cv3 )
/LECTURE/
```

ncom

Number of components in this material.

r

Universal constant of gases (R), must be expressed in J/(kmol K).

nomcomp

Name of the component (max 8 characters) in quotes.

pm

Molecular weight w of the component, in kg/kmol.

cv1/2/3

Coefficients of the expression of the specific heat at constant volume (c_v), which must be expressed in J/kmolK, as a function of absolute temperature T : $c_v(T) = c_{v1} + c_{v2}T + c_{v3}T^2$. The specific internal energy i results from $i(T) = c_v T = c_{v1}T + c_{v2}T^2 + c_{v3}T^3$. Note that i must be expressed in J/kmol. T is the temperature in K. Consequently, c_{v1} must be in J/kmolK, c_{v2} must be in J/kmol(K2) and c_{v3} must be in J/kmol(K3).

Comments:

Only one material of type multicomponent fluid is allowed in a model. This is not a restriction, since the number of components is arbitrary and component mass fractions can be locally zero.

The polytropic exponent $\gamma = c_p/c_v = (R + c_v)/c_v$, is determined by the code. Therefore, note that the units of the various c_v coefficients in the above expression for i should be consistent with the units used for R . For example, c_{v1} should be expressed in J/(kmol K).

Also the density is determined by the code.

Comparing FE and FV solutions.

Frequently it is desirable to compare solutions obtained with Finite Element (FE) models to equivalent ones obtained with Finite Volumes (FV), typically for the case of perfect gases. In the case of FLUT material used for Finite Elements (see page C.520), the equivalence of initial conditions is not completely straightforward. A procedure for converting between the two formulations is detailed hereafter.

The two fluid solvers have completely different ways to carry out the numerical discretization of the same governing equations (Euler equations); each of them has its specific formulation, its own set of variables and its own parameters, whose value has to be assigned in the input data.

More in detail, in the FE model the perfect gas state equation used to close the system of Euler equations has the form:

$$p = (\gamma - 1)\rho i,$$

where p is the pressure (Pa), ρ is the density (Kg/m³), i is the internal energy per unit mass (J/Kg) and γ (-) is the ratio between the constant pressure and constant volume specific heats c_p (J/kmolK) and c_v (J/kmolK). The user must in this case provide in input the values of γ , ρ and i (see MATE FLUT on Page C.520).

In the FV model the same state equation takes the form:

$$p = RT \frac{\rho}{w},$$

where R is the universal constant of gases (J/kmolK), which has the standard value of 8314.3, T is the absolute temperature (K) and w is the molar weight (kg/kmol). Note that the state equation in the FV model could be more complex, taking into account a more general mixture

of Joule gases. We consider here a single-component perfect gas for simplicity. The user must in this case provide in input the values of R , c_v and w for the material (see MATE MCGP above), plus the initial values of p and T at each node (and thus at each finite volume), via the directive INIT MCOM (see Page E.150).

Switching from FV to FE, an equivalent input can be obtained readily from the identities:

$$\gamma = \frac{R}{c_v} + 1 \qquad i = \frac{c_v}{w} T \qquad \rho = \frac{wp}{RT}$$

The inverse path is not so straightforward. The switch from FE to FV is not univocally determined unless the molar weight w is known. Indeed the physics of the problem only depends on the internal energy i (see above), which is proportional to the ratio T/w by means of the value:

$$c_v = \frac{R}{\gamma - 1}$$

Then it is possible to choose any couple T and w so as to have the appropriate i , but values of temperature would in general *not* be correct during a calculation.

As an example, consider the following set of initial conditions, which have been chosen without actual physical relevance and have been rounded in order to easily check the equivalence of the several parameters values in the FE and FV representation.

Assume we want to simulate two perfect gases, one initially at high pressure and the other initially at low pressure. Let $\gamma = 1.5$ and $\rho = 2 \text{ kg/m}^3$ for both gases. If the HP-gas has an initial pressure of $p_H = 5.E5 \text{ Pa}$, then we get from the equation of state in FE form: $i_H = p_H/(\gamma - 1)\rho = 5.E5 \text{ J/kg}$. Similarly, for the LP-gas at, say, $p_L = 1.E5 \text{ Pa}$ we obtain $i_L = p_L/(\gamma - 1)\rho = 1.E5 \text{ J/kg}$. These values completely define the FE material data.

To get an equivalent FV description, we must provide the constant of perfect gases, which in standard units is about $R = 1.E4 \text{ J/kmolK}$, and we must choose a molar weight, say $w = 20 \text{ kg/kmol}$ for both gases. Then we obtain the specific heat at constant volume (same for both gases) from the relation $c_v = R/(\gamma - 1) = 2.E4 \text{ J/kmolK}$.

Assuming for both gases the same initial density $\rho = 2 \text{ kg/m}^3$ as in the FE case, we may compute the temperature from the relation $T = wp/\rho R$. For the H-P gas ($p_H = 5.E5 \text{ Pa}$) this gives $T = 500 \text{ K}$, while for the L-P gas ($p_L = 1.E5 \text{ Pa}$) this gives $T = 100 \text{ K}$.

9.7.29 MULTICOMPONENT FAR-FIELD FLUID MATERIAL (MCFF)**Object:**

This material is used with specialised elements of type CL22, CL3I or CL3Q to specify far-field conditions of multicomponent flows. It specifies the constant physical state: $\rho(1)$, ... $\rho(\text{ncom})$, ρu , ρv , ρw , ρE , assumed outside the discretized fluid domain.

Unlike material MCGP, which must be unique in a single calculation, an arbitrary number of MCFF materials is allowed. However, note that the declaration of the MCGP material **MUST** precede the declaration of the MCFF(s) in the input data set.

The contribution of the CESI team (formerly at ENEL, Milano) to the development of this material model in collaboration with JRC is gratefully acknowledged.

Syntax:

```
"MCFF" "BDF0" bdfo "TEMP" temp "PRES" pres "VEL1" vel1
      "VEL2" vel2 "VEL3" vel3
      ncom * ( "COMP" 'nomcomp' "MFRA" mfra )
      /LECTURE/
```

bdfo

Option for boundary flux: 1 = Roe, 2 = Van Leer, 3 = Steger-Warming. Recall that the flux type in the bulk fluid is chosen (independently from the boundary flux type) by directive OPTI MC NUFL.

temp

Temperature of the far-field state.

pres

Pressure of the far-field state.

vel1

X-velocity of the far-field state.

vel2

Y-velocity of the far-field state.

vel3

Z-velocity of the far-field state.

ncom

Number of components (must be the same as for the MCGP material).

nomcomp

Name of the component (max 8 characters) in quotes, must be spelled exactly as in the definition of the MCGP material.

mfra

Mass fraction of the component.

Comments:

The key-words **TEMP ... VEL3** must precede the declaration of the mass fractions **COMP ... MFRA** in the input data set.

9.7.30 MULTIPHASE MULTICOMPONENT FLUID MATERIAL (FLMP)**Object:**

This material is used with FLxx elements to describe multi-phase multicomponent fluid flows. In this formulation, more than one fluid material (liquids, perfect gases) may be present at the same time inside a generic finite element. The material is treated as a homogeneous mixture of the component fluids.

The velocity field is unique (i.e., all components have the same velocity. The pressure is defined at the element level as follows: if one or more liquids are present, they are subjected to the same pressure. If more than one gas is present, the gases in the mixture follow Dalton's law: the sum of the partial pressures of the gas components equals the pressure of the mixture.

This 'material' (type FLMP) is composed of several FLUT materials (see page C3.520).

Note: this material is still under development and testing. It has to be used with great care.

Syntax:

```
"FLMP" "NLIQ" nliq "NGAS" ngas
      nliq * ( "FLUT" liquid_material_description )
      ngas * ( "FLUT" gas_material_description      )
```

nliq

Total number of liquid materials in the mixture.

ngas

Total number of gas materials in the mixture.

Comments:

The liquid materials must precede the gas materials.

The program considers the first nliq FLUT-material descriptions encountered in the input file after the "FLMP" directive as descriptions of the liquid materials, and the successive ngas FLUT-material descriptions as descriptions of the gas materials.

The elements to which each FLUT material is associated are specified as usual via a /LECTURE/ directive at the end of each FLUT-material description. As a consequence, each element containing a FLMP mixture will effectively contain only one component (with a 100 per cent mass fraction) at the initial time. Because of this, it is not possible to effectively have more than one material in any element in the initial conditions. However, during the transient analysis the materials will mix up because of transport between adjacent elements, thus forming the mixture.

Outputs:

The components of the ECR table are as follows:

Positions 1-9 are equivalent to those of the FLUT material:

ecr(1): current element pressure of the fluid mixture

ecr(2): current AVERAGE density of the fluid mixture

ecr(3): current sound speed of the fluid mixture

ecr(4): current AVERAGE specific internal energy of the fluid mixture

ecr(5): current AVERAGE bulk modulus of the fluid mixture

ecr(6): current pseudoviscous pressure of the fluid mixture

ecr(7): current minimum pressure flag of the fluid mixture

ecr(8): maximum pressure ever experienced by the element

ecr(9): minimum pressure ever experienced by the element

ecr(10): number of effective components in the element

Then, for each component icom of the mixture:

`iad=nfixmp+(icom-1)*necrmp (see FLUTMP.INC)`

ecr(iad+1): current relative mass fraction of the component

ecr(iad+2): current density of the component

ecr(iad+3): current specific internal energy of the component

ecr(iad+4): current partial pressure of the component

ecr(iad+5): mass fraction of the component at the end of Lagrangian phase

ecr(iad+6): specific internal energy of the component at the end of Lag. ph.

There may be at most 4 different components in a FLMP material, at present. Thus, there is place for up to $(10 + 6 \times 4) = 34$ components of ECR at each Gauss point of an element with a FLMP material.

9.7.31 BUBBLE MODEL

Object :

This directive simulates an explosion in the air. It allows to load the structures without having to model the explosive. Instead of the explosive a compressed bubble is used. The overpressure of this bubble is calculated depending on the charge and the size of the bubble. The model can be used with **GAZP** and **FLUT** materials.

Syntax:

```
"BUBB" "MASS" m /LECTURE/
```

m

Mass of the explosive in Kilograms.

```
/LECT/
```

Elements concerned (bubble volume).

Comments :

The mass of the explosive is always the mass in the bubble defined with the concerned elements. This is to consider by simulations with symmetry conditions or in a case of a hemispherical condition. At a border with symmetry, also the charge has to be cut. At a border of a hemispherical model, the charge must not be cut.

The material uses the material parameters of the neighbour elements. Therefore these materials have to be defined before.

Outputs:

The different components of the ECR table are the components of the **GAZP** or **FLUT** material.

9.7.32 CDEM—Discrete Equation Method for Combustion

Object:

Extension of the Reactive Discrete Equation method (Le Métayer *et al.* 2005) to the combustion case [719, 720] (two phases: the phase of the unburnt gas, the phase of the burnt gas). The combustion is governed by the irreversible exothermic chemical reaction

$$\sum_{i=1, \text{nlhs}} c_i A_i \rightarrow \sum_{i=\text{nlhs}+1, \text{nesp}} c'_i A_i$$

or, in compact form,

$$\sum_{i=1, \text{nesp}} c_i A_i \rightarrow 0$$

with c_i negative if $i > \text{nlhs}$.

The governing equations are the Euler Equations in conservative form (the main conserved variables for each phase are the mass densities of the components, the momentum, the total energy (sensible + chemical + kinetic) per unit volume) and a transport equation for the volume fraction.

Syntax:

```
"CDEM"  "TINI" tini    "PINI" pini  <"PREF" pref>
...     "KSI0" ksi0    "K0" k0
...     "TMAX" tmax    "R" rgas
...     "NESP" nesp    "ORDP" ordp    "NLHS" nlsh
...
...     "COMP1"
...     "MMOL" mmol    "H0" h0      "CREA" crea
...     "CV0" cv0     "CV1" cv1 ... "CVordp" cvordp
...     "YMAS" ymas
...
...     "COMPnesp"
...     "MMOL" mmol    "H0" h0      "CREA" crea
...     "CV0" cv0     "CV1" cv1 ... "CVordp" cvordp
...     "YMAS" ymas
...
...     <"KOF" kof>
...     <"UCDS" ucds>
...     <"DIRE" dire>
...     <"T0"  " temp0>
...     <"H"   " hcoef>
...     <"GX"  " gx>
```

```
... <"GY  " gy>  
... <"GZ  " gz>
```

tini

Initial temperature of the mixture.

pini

Initial pressure of the mixture.

pref

Reference pressure.

ksi0

Initial volume fraction of the burnt mixture.

k0

Part of the fundamental flame speed transported with the flow.

tmax

Maximum value of the temperature for the computation of the specific heats.

rgaz

Gas constant (8.3144621 in SI unit).

nesp

Number of species involved in the mixture.

nlhs

Number of species in the left hand side of the chemical reaction.

ordp

Temperature polynomial degree for the constant volume specific heat computation.

"COMP1", ..., "COMPnesp"

Keywords which state that we will describe the properties of the species 1,...,nesp.

mmol

Molar mass.

h0

Formation enthalpy at $T = 0$ K.

crea

Coefficient in the chemical reaction (positive if the species is a reactant).

cv0, ..., cvordp

Coefficient of $T^0, \dots, T^{\text{ordp}}$ for the computation of the constant volume specific heat.

ymas

Initial mass fraction before the combustion occurs.

k0f

Part of the fundamental flame speed fixed in space (default value 0).

ucds

High order reconstruction for the volume fraction.

- 0 The same limited reconstruction as in [720] (default value).
- 1 Limited reconstruction combined with the Upwind Downwind Controlled Splitting (see [723]).
- 2 Anti-diffusive reconstruction combined with the Upwind Downwind Controlled Splitting (see [723]).

dire

Value of the fundamental flame speed in multi-dimensional computations.

- 0 Fundamental flame speed equal to $((k0 + k0f) (\vec{n} \cdot \vec{n}_f))$, where \vec{n} and \vec{n}_f are the normal to the flame surface and the normal to the interface of the finite volume cell (default value).
- 1 Fundamental flame speed equal to $(k0 + k0f)$.

temp0, hcoef

Loss coefficients for the total energy with the environment (default value is zero for both).

- $\alpha_1 \text{hcoef}(T_1 - \text{temp0})$ (loss of energy per unit time and volume in the unburnt gas)
- $\alpha_2 \text{hcoef}(T_2 - \text{temp0})$ (loss of energy per unit time and volume in the burnt gas)

gx, gy, gz

Gravity components ($\vec{g} = (\text{gx}, \text{gy}, \text{gz})$) (default values are zero). The gravity force is taken into account according to the following approximate formulae.

- $\alpha_1 \vec{g}(\rho_1 - \rho_1) = 0$ (gravity force per unit volume acting on the unburnt gas)
- $\alpha_2 \vec{g}(\rho_2 - \rho_1)$ (gravity force per unit volume acting on the burnt gas)

Comments:

For the first species ("COMP1"), the coefficient of the chemical reaction crea should be equal to 1.

The sum of all initial mass fractions should be equal to 1.

Outputs:

The different components of the ECR table are:

ECR(1) : pressure

ECR(2) : density

ECR(3) : maximum between the sound speed in burnt gas and the sound speed in unburnt gas

ECR(4) : volume fraction of the unburnt gas

ECR(5) : density of the unburnt gas

ECR(6) : velocity along x of the unburnt gas

ECR(7) : velocity along y of the unburnt gas

ECR(8) : velocity along z of the unburnt gas (3D)

ECR(6 + ndim) : pressure of the unburnt gas

ECR(7 + ndim) : volume fraction of the burnt gas

ECR(8 + ndim) : density of the burnt gas

ECR(9 + ndim) : velocity along x of the burnt gas

ECR(10 + ndim) : velocity along y of the burnt gas

ECR(11 + ndim) : velocity along z of the burnt gas (3D)

ECR(9 + (2 ndim)) : pressure of the burnt gas

...

iespmax = min(nesp, 20 - (9 + (2 ndim)))

ECR(9 + iesp + (2 ndim)) : mass fraction of the iesp-species after the combustion occurs (iesp = 1, ..., nlhs; iesp < iespmax)

ECR(9 + iesp + (2 ndim)) : mass fraction of the iesp-species before the combustion occurs (iesp = nlhs + 1, ..., nesp - 1; iesp < iespmax)

...

nn = min(20, (9 + (2 ndim) + nesp)) = 9 + (2 ndim) + ispmax

ECR(nn) : difference between the mass fraction of the first species in the unburnt gas and in the burnt gas

ECR(21) : burnt surface per unit volume

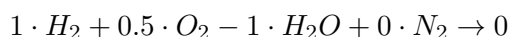
ECR(22) : fundamental flame speed (including "K0F")

Example:

We consider the combustion of a stoichiometric mixture of hydrogen-air:



or



Here COMP1, ..., COMP4 represent H₂, O₂, H₂O and N₂ (namely, nesp=4, nlhs=2). The constant volume specific heat coefficients are obtained via a forth order regression of JANAF tables (ordp=4).

CDEM

PINI 1279.9 PREF 1e5 TINI 251.0

KSI0 0.999

K0 45.2

TMAX 6000. R 8.31441

NESP 4

NLHS 2

ORDP 4

COMP1

MMOL 2.01594E-3 H0 -4.195E6 CREA 1.

CV0 9834.91866 CV1 0.54273926 CV2 0.000862203836

CV3 -2.37281455E-07 CV4 1.84701105E-11

YMAS 0.1

COMP2

MMOL 31.9988E-3 H0 -2.634E5 CREA 0.5

CV0 575.012333 CV1 0.350522002 CV2 -0.000128294865

CV3 2.33636971E-08 CV4 -1.53304905E-12

YMAS .2

COMP3

MMOL 18.01534E-3 H0 -1.395D7 CREA -1.0

CV0 1155.95625 CV1 0.768331151 CV2 -5.73129958E-05

CV3 -1.82753232E-08 CV4 2.44485692E-12

YMAS 0.3

COMP4

MMOL 28.0134E-3 H0 -2.953D5 CREA 0.0

CV0 652.940766 CV1 0.288239099 CV2 -7.80442298E-05

CV3 8.78233606E-09 CV4 -3.05514485E-13

YMAS 0.4

LECT ... TERM

If we suppose that we are in 2D ($\text{ndim} = 2$), for the different components of the ECR table we have:

$$\text{iespmax} = \min(\text{nesp}, 20 - (9 + (2 \text{ ndim}))) = \min(4, 20 - 13) = 4$$

$$\text{nn} = \min(20, (9 + (2 \text{ ndim}) + \text{nesp})) = 9 + (2 \text{ ndim}) + \text{iespmax} = 17$$

$\text{ECR}(9 + 1 + (2 \text{ ndim})) = \text{ECR}(14)$: mass fraction of the H_2 after the combustion occurs

$\text{ECR}(9 + 2 + (2 \text{ ndim})) = \text{ECR}(15)$: mass fraction of the O_2 after the combustion occurs

$\text{ECR}(9 + 3 + (2 \text{ ndim})) = \text{ECR}(16)$: mass fraction of the H_2O before the combustion occurs

$\text{ECR}(\text{nn}) = \text{ECR}(17)$: difference between the mass fraction of the H_2 in the unburnt gas and in the burnt gas

$\text{ECR}(21)$: burnt surface per unit volume

If we suppose that we are in 3D ($\text{ndim} = 3$), for the different components of the ECR table we have:

$$\text{iespmax} = \min(\text{nesp}, 20 - (9 + (2 \text{ ndim}))) = \min(4, 20 - 15) = 4$$

$$\text{nn} = \min(20, (9 + (2 \text{ ndim}) + \text{nesp})) = 9 + (2 \text{ ndim}) + \text{iespmax} = 19$$

$\text{ECR}(9 + 1 + (2 \text{ ndim})) = \text{ECR}(16)$: mass fraction of the H_2 after the combustion occurs

$\text{ECR}(9 + 2 + (2 \text{ ndim})) = \text{ECR}(17)$: mass fraction of the O_2 after the combustion occurs

$\text{ECR}(9 + 3 + (2 \text{ ndim})) = \text{ECR}(18)$: mass fraction of the H_2O before the combustion occurs

$\text{ECR}(\text{nn}) = \text{ECR}(19)$: difference between the mass fraction of the H_2 in the unburnt gas and in the burnt gas

$\text{ECR}(21)$: burnt surface per unit volume

$\text{ECR}(22)$: fundamental flame speed (including "K0F")

Note that one can easily determine the mass fraction in both the burnt or unburnt gas, as explained in [719]. Indeed, via the knowledge of $(Y_{\text{H}_2,\text{u}} - Y_{\text{H}_2,\text{b}})$, one can determine the other variations by using the molar mass (MMOL) and the reaction coefficient (CREA):

$$\frac{Y_{\text{H}_2,\text{u}} - Y_{\text{H}_2,\text{b}}}{2.01594 \cdot 10^{-3} \cdot 1} = \frac{Y_{\text{O}_2,\text{u}} - Y_{\text{O}_2,\text{b}}}{31.9988 \cdot 10^{-3} \cdot 0.5} = \frac{Y_{\text{H}_2\text{O},\text{u}} - Y_{\text{H}_2\text{O},\text{b}}}{18.01534 \cdot 10^{-3} \cdot (-1)}$$

9.7.33 DEMS—Discrete Equation Method for Two Phase Stiffened Gases

Object:

Modeling of two phase flows involving stiffened gases via the Discrete Equation method of Abgrall and Saurel 2003 (see also [722, 723]). The governing equations are the Euler Equations in conservative form (the main conserved variables for each phase are the mass densities of the components, the momentum, the total energy (sensible + chemical + kinetic) per unit volume) and a transport equation for the volume fraction.

Syntax:

```
"DEMS"  "PINI"  pini  <"PREF" pref>
...     "ALP1"  alp1  "ROI1" roi1 "ROI2" roi2
...     "NESP"
...
...     "COMP1"
...     "GAMM"  gamm  "CP"  cp  "PI"  pi
...     "YMA1"  yma1
...     "YMA2"  yma2
...     "MMOL"  mmol   "H0"  h0

...
...     "COMPnesp"
...     "GAMM"  gamm  "CP"  cp  "PI"  pi
...     "YMA1"  yma1
...     "YMA2"  yma2
...     "MMOL"  mmol   "H0"  h0
...

...     <"RELA" rela>
...     <"UCDS" ucds>
```

pini

Initial pressure of the mixture.

pref

Reference pressure.

alp1

Initial volume fraction of the phase 1

roi1

Mass density of the phase 1

roi2

Mass density of the phase 2

nesp

Number of species involved in each phase

"COMP1", ..., "COMPnesp"

Keywords which state that we will describe the properties of the species 1,...,nesp.

gamm

Specific heat ratio

cp

Constant pressure specific heat

pi

Molecular attraction effect parameter

yma1

Mass fraction in the phase 1

yma2

Mass fraction in the phase 2

mmol

Molar mass.

h0

Formation enthalpy at $T = 0$ K.

ucds

High order reconstruction for the volume fraction.

0 The same limited reconstruction as in [720] (default value).

1 Limited reconstruction combined with the Upwind Downwind Controlled Splitting (see [723]).

2 Anti-diffusive reconstruction combined with the Upwind Downwind Controlled Splitting (see [723]).

rela 1 Relaxation of the pressure and of the velocity (the pressure and the velocity are the same on the two phases).

0 No relaxation of the pressure and of the velocity (default value).

Comments:

The sum of all initial mass fractions should be equal to 1 in each phase.

Outputs:

We define $\text{nesp}_{\text{ecr}} = \min(\text{nesp}, 7 - \text{ndim})$, which represents the number of species that are represented in the ECR table. Then the different components of the ECR table are:

ECR(1) : pressure

ECR(2) : density

ECR(3) : maximum of the sound speed in the two phases

ECR(4) : velocity along x

ECR(5) : velocity along y

ECR(6) : velocity along z (if existing)

ECR(4 + ndim) : volume fraction of the phase 1

ECR(5 + ndim) : density of the phase 1

ECR(6 + ndim) : velocity along x of the phase 1

ECR(7 + ndim) : velocity along y of the phase 1

ECR(8 + ndim) : velocity along z of the phase 1 (if existing)

ECR(6 + (2 * ndim)) : pressure of the phase 1

ECR(7 + (2 * ndim)) : mass fraction of the first component of the phase 1

ECR(6 + (2 * ndim) + nesp_{ecr}) : mass fraction of the nesp_{ecr} -th component of the phase 1

ECR(7 + (2 * ndim) + nesp_{ecr}) : volume fraction of the phase 2

ECR(8 + (2 * ndim) + nesp_{ecr}) : density of the phase 2
 ECR(9 + (2 * ndim) + nesp_{ecr}) : velocity along x of the phase 2
 ECR(10 + (2 * ndim) + nesp_{ecr}) : velocity along y of the phase 2
 ECR(11 + (2 * ndim) + nesp_{ecr}) : velocity along z of the phase 1 (if existing)
 ECR(9 + (3 * ndim) + nesp_{ecr}) : pressure of the phase 2
 ECR(10 + (3 * ndim) + nesp_{ecr}) : mass fraction of the first component of the phase 2
 ECR(9 + (2 * ndim) + (2 * nesp_{ecr})) : mass fraction of the nesp_{ecr}-th component of the phase 2

Example:

We consider a mixture of two stiffened gases in a shock tube (two zones, Z_HP and Z_BP). The zone Z_HP is almost totally occupied by the phase 1 (volume fraction of the phase 1 equal to 0.9999) and it only contains the first stiffened gas. The zone Z_BP is almost totally occupied by the phase 2 (volume fraction of the phase 1 equal to 0.0001) and it only contains the second stiffened gas.

```

*                               ZONE 1
*                               -----
*
DEMS
*
*   PINI 9.12E3 PREF 1e0
*   ALP1 0.9999 ROI1 1.271 ROI2 0.99
*
*   NESP 2
*
*   COMP ! AIR. H0 is false, but not used for the moment
*   GAMM 1.4 CP 1010. PI 0.0
*   YMA1 1.0 ! YMA1 = mass fraction in the phase 1
*   YMA2 0.0 ! YMA2 = mass fraction in the phase 2
*   MMOL 28.8E-3 H0 0.0
*
*
*   COMP ! STIFFENED GAS.
*   GAMM 7.0 CP 7990 PI 3.0E3
*   YMA1 0.00
*   YMA2 1.00
*   MMOL 18E-3 H0 0.0
*
*
*                               LECT Z_HP TERM
*
*                               ZONE 2
*                               -----

```

```
DEMS
*
  PINI 1.0E0 PREF 1e0
  ALP1 0.0001 ROI1 1.271 ROI2 0.99
*
*
  NESP 2
*
  COMP ! AIR. H0 is false, but not used for the moment
    GAMM 1.4 CP 1010. PI 0.0
    YMA1 1.0
    YMA2 0.0
    MMOL 28.8E-3 H0 0.0
* *
  COMP ! STIFFENED GAS.
    GAMM 7.0 CP 7990 PI 3.0E3
    YMA1 0.00
    YMA2 1.00
    MMOL 18E-3 H0 0.0
*
*
```

LECT Z_BP TERM

9.8 IMPEDANCES

Object :

This directive enables impedances for elements with CLxx boundary conditions to be input.

There are three forms of the directive, depending upon whether the impedance concerns:

- Finite Elements (IMPE), or
- Elements using the Van Leer formulation (IMPV), or
- Cell-Centred Finite Volumes (VFCC) (CLVF)

The available options for IMPE are the following:

ABSO : absorbing boundary

PCHA : head loss

PIMP : imposed pressure

DIAP : diaphragm with imposed pressure

GRIL : grid model

MEMB : safety membrane

GRFS : grid model with fluid/structure coupling

NAH2 : coupling of the water mass flow rate in sodium-water reaction

RSEA : coupling of the water mass flow rate in sodium-water reaction (new model)

VANN : closure of a valve

SWVA : swing check valve with fluid-structure coupling (EDF implementation)

DCRI : critical mass flow rate

FOND : closed bottom

FSUI : following force

POMP : pump model

PPLT : perforated plate (JRC implementation)

RUDI : rupture disk (JRC implementation)

STAC : Stacey's 1st-order absorbing boundary (JRC implementation)

RDMC : MC rupture disk (JRC implementation)

SVAL : safety valve (JRC implementation)

RDK2 : rupture disk (JRC implementation 2)

ABSI : absorbing boundary (JRC implementation)

ABSZ : absorbing boundary (Zienkiewicz for geotechnical materials)

AIRB : air blast wave

FPLT : fragile plate

Not all these options are available for all boundary condition elements. See the following table :

Option	CL1D	CL2D	CL3D	CL3T	CLTU	CL22	CL3Q	CL3I
ABSO	x	x	x	x	x			
PCHA	x	x	x	x	x			
PIMP	x	x	x	x	x	x	x	x
DIAP	x	x			x			
GRIL	x	x	x	x	x			
MEMB	x	x			x			
GRFS		x	x	x				
NAH2	x				x			
RSEA	x				x			
VANN	x				x			
SWVA	x				x			
DCRI	x		x		x			
FOND					x			
FSUI					x			
POMP	x				x			
PPLT						x	x	x
RUDI						x	x	x
STAC						x	x	
RDMC				x			x	
SVAL						x	x	x
RDK2				x		x	x	x
ABSI						x	x	x
ABSZ								
AIRB		x	x	x		x	x	x
FPLT						x	x	x

The available options for IMPV are the following:

ABSO : absorbing boundary

INFI : conditions at infinity for a fluid

PIMP : imposed pressure

DEGP : imposed mass flow rate for a perfect gas

MUR : total reflexion (rigid obstacle)

Not all these options are available for all boundary condition elements. See the following table:

Option	CL1D	CL2D	CL3D	CL3T	CLTU	CL22	CL3Q	CL3I
ABSO	?	?	?	?	?			
INFI		x						
PIMP	?	?	?	?	?			
DEGP	?	?	?	?	?			
MUR	?	?	?	?	?			

The available options for CLVF are the following:

ABSO : Absorbing boundary

INFI : Conditions at infinity for a fluid

PIMP : Imposed pressure

DEBI : Imposed mass flow rate

ESUB : Imposed mass flow rate, sub-sonic inflow

LOD1 : Lodi, quasi-1D condition

FOUR : Fourier modes in 2D

RIEM : Riemann in 3D

LODG : Lodi 3D with generalized coordinates

CARM : Lodi 3D with multi-dimensional characteristics

ASYM : Asymptotic 3D

LIBR : Free impedance (can be programmed by the user)

FOND : Closed bottom

Not all these options are available for all boundary condition elements. See the following table:

Option	CL1D	CL2D	CL3D	CL3T	CLTU	CL22	CL3Q	CL3I
ABSO								
INFI								
PIMP								
DEBI								
ESUB								
LOD1								
FOUR								
RIEM								
LODG								
CARM								
ASYM								
LIBR								
FOND								

Syntax :

```

"IMPE"      $  "ABSO" . . . $
              $  "PCHA" . . . $
              $  "PIMP" . . . $
              $  "DIAP" . . . $
              $  "GRIL" . . . $
              $  "MEMB" . . . $
              $  "GRFS" . . . $
              $  "NAH2" . . . $
              $  "RSEA" . . . $
              $  "VANN" . . . $
              $  "SWVA" . . . $
              $  "DCRI" . . . $
              $  "FOND" . . . $
              $  "FSUI" . . . $
              $  "POMP" . . . $
              $  "PPLT" . . . $
              $  "RUDI" . . . $
              $  "STAC" . . . $
              $  "RDMC" . . . $
              $  "SVAL" . . . $
              $  "RDK2" . . . $

```

	\$	"ABSI"	. . .	\$
	\$	"AIRB"	. . .	\$
	\$	"FPLT"	. . .	\$
"IMPV"	\$	"ABSO"	. . .	\$
	\$	"INFI"	. . .	\$
	\$	"PIMP"	. . .	\$
	\$	"DEGP"	. . .	\$
	\$	"MUR"	. . .	\$
"CLVF"	\$	"ABSO"	. . .	\$
	\$	"INFI"	. . .	\$
	\$	"PIMP"	. . .	\$
	\$	"DEBI"	. . .	\$
	\$	"ESUB"	. . .	\$
	\$	"LOD1"	. . .	\$
	\$	"FOUR"	. . .	\$
	\$	"RIEM"	. . .	\$
	\$	"LODG"	. . .	\$
	\$	"CARM"	. . .	\$
	\$	"ASYM"	. . .	\$
	\$	"LIBR"	. . .	\$
	\$	"FOND"	. . .	\$

Comments :

These key-words can be repeated as many times as necessary with different options each time (if need be). The keyword IMPE, IMPV or CLVF should not be separated from the options : ABSO , PCHA , PIMP , etc.

Beware that the CLVF models are still experimental and under development. Only the ABSO, LOD1, FOUR and RIEM models have been tested somewhat (the ABSO being by far the most used one), but are for the moment available only in 3D, for perfect gas material and for first-order in space and in time VFCC formulations. Furthermore, they may function with only some of the flux solvers available. For an overview of the state of the art of these developments see references [751] and [752].

9.8.1 ABSORBING MATERIAL

Object :

This option enables to specify absorbing or partially absorbing boundary conditions for 1-D, 2-D or 3-D elements.

This model is appropriate for CLxx elements developed at CEA, namely CL1D, CL2D, CL3D or CL3T. There exists a similar, but not identical, absorbing boundary model developed at JRC which is appropriate for their CLxx elements (CL22, CL3I or CL3Q), see page C.880.

Only pressure waves normal to the boundary are absorbed. The model consists simply in applying a fictitious external pressure $p = -\rho c v_n$, where ρ is the density of the material at the boundary, c its sound speed and v_n the normal component of the particle velocity at the boundary, in Lagrangian calculations, or of the relative (particle minus mesh) velocity in Eulerian or ALE calculations. The “internal” forces due to the absorbing boundary are finally computed by spatial integration of a modified pressure $\pi = (p + p_{\text{old}})/2$, where p_{old} is the value of π at the previous time integration step.

Syntax:

```
"IMPE"  "ABSO"  <"R0" rho>  <"C" c>  /LECTURE/
```

rho

Fixed, user-imposed value of the density. If omitted, the code will try to determine the density automatically.

c

Fixed, user-imposed value of the sound speed. If omitted, the code will try to determine the sound speed automatically. Since for a structural material the code is sometimes unable to determine the sound speed automatically, the value becomes useful in this case (and is of course constant). However, since the physical sound speed is fairly constant in such a case, the behaviour of the model should be quite good. A notable exception are materials **CAMC** and **CLAY**, for which the sound speed varies considerably: for these materials the code is indeed able to retrieve the current sound speed automatically, so that specifying **c** in these cases is unnecessary.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL1D, CL2D, CL3D or CL3T)

Comments :

If the acoustic waves are to be absorbed, the `rho` and `c` parameters must be the same on both sides of the boundary. The effect will then be that of an infinite medium. In the opposite case, there will be partial reflections.

If the user has omitted `c` and the code is unable to determine it automatically, an error message is issued and the calculation is stopped.

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(3) : density times normal velocity in the local reference frame

9.8.2 HEAD LOSS

Object :

This instruction enables localized head losses to be input.

Syntax :

```
"IMPE"  "PCHA"  "R0" rho  "K"  k  /LECTURE/
```

rho

Density.

k

Coefficient of a localized head loss.

LECTURE

Reading procedure of the element numbers (CL1D, CL2D or CL3D).

Comments :

The head loss (DP) is deduced from the density (rho) and the velocity (V) up-stream of the singularity:

$$DP = 0.5 * k * rho * V*V$$

The result is a resisting force which is always opposed to the velocity.

It is evident that this model may not be applied to the extremities of pipelines.

Outputs:

The different components of the ECR are as follows:

ECR(1) : pressure

ECR(2) : density

9.8.3 GRID

Object:

This instruction enables to model the influences of grids or rigid perforated plates on a fluid.

See also on page C4.650 directive GRFS.

Syntax:

```
"IMPE"  "GRIL"  "R0" rho  "C" c  "ALP" alpha  ...  
        ...  "TAU" tau  /LECTURE/
```

rho

Density.

c

Velocity of sound in the fluid.

alpha

Dissipative impedance.

tau

Time constant.

LECTURE

Reading procedure of the element numbers (CL2D or CL3D).

Comments:

The model is based on the hypothesis of an acoustic propagation of plane waves.

The meaning of the parameters alpha and tau is as follows:

Let:

L : equivalent length of the grid holes

ST : total cross section

s : cross section open to flow

M : Mach number of the permanent flow up-stream of the singularity

xi : head loss coefficient

Then:

$$\alpha = \xi * M \quad \tau = \frac{ST}{s} * \frac{L}{2*c}$$

Remarks:

The ratio s / ST represents the perforation ratio of the plate.

In the steady-state regime the pressure drop across the plate assumes the form:

$$DP = \alpha * \rho * c * V$$

Recall that for an absorbing boundary $\alpha = i$, since the pressure and mass flow rate fluctuations are in quadrature.

The equivalent length L is not equal to the plate thickness. To take into account three-dimensional effects, add a length equivalent to the diameter of the holes to that thickness.

Warning:

This directive allows to transmit the stresses from the plate to the fluid. It allows to modify the flow by taking into account the pressure drops introduced by a perforated plate. The directives "IMPE" "GRFS" should be used in order to correctly transfer the fluid stresses onto an equivalent shell structure representing the plate.

For an A.L.E. computation with a fluid-structure coupling of the perforated plate, the user has to mention twice the "FS2D" elements, which link together the plate and the fluid : first, in the instruction "GRILLE" "ALE" ... "FS" ..., in order to make the fluid nodes follow the motion of the grid; and then in the instruction "LIAISON" "FS" ..., in order to transmit correctly the forces of the fluid towards the plate. The fluid has to be meshed continuously through the plate. In this case, the elements "FS2D" are located inside the fluid and not in its surrounding area. All the other "FS2D" elements, normally located at the boundaries of the fluid, are mentioned

only once in the instruction "GRILLE" "ALE" ... "FS" ..., for in A.L.E. the fluid-structure coupling is done automatically on the boundaries.

In a lagrangian computation, for the same mesh this problem does not exist, because only the following instruction exists : "LIAS" "FS"...

Outputs :

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

9.8.4 IMPOSED PRESSURE

Object:

This instruction enables a pressure at the boundary of different elements to be imposed by the means of a "CLxx" element.

Syntax:

```
"IMPE"  "PIMP"  "RO" rho  "PRES" pres  ...
        ... < "PREF" pref >  < "ENTH" enth >  ...
        ... < $[ "TABP" npt*( t , p ) ; "FONC" nufo ]$ > ...
        ... /LECTURE/
```

rho

Density.

pres

Constant imposed pressure, or multiplying factor for the ordinates of the following table or function if it exists.

pref

Reference pressure. Note that if a zero reference pressure is desired, it is mandatory to specify 'PREF 0' in the input file. This is because, if no value for PREF is specified, the code assumes $PREF = PRES(t=0)$ so the initial imposed pressure has no effect, since $p = PRES - PREF = 0$!

enth

Input enthalpy.

npt

Number of points defining the pressure curve.

t , p

Coordinates of a point on the curve (time , pressure).

nufo

Number of the function to be used to describe the pressure versus time.

LECTURE

Reading procedure of the number of the "CLxx" element defining the boundary.

Comments:

For the meaning of pref, see page C.300

If the input enthalpy is zero, the code uses the enthalpy value of the neighbour element. Otherwise, the user's imposed value is taken into account. This possibility works only in Van Leer.

ATTENTION ! The keyword "TABP", that introduces the time function of the pressure, is obsolete. Use preferably the directive "FONC". Keyword "TABP" is maintained just to ensure compatibility with old input data sets.

The keywords "TABP" and "FONC" are mutually exclusive.

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(9) : enthalpy per unit volume (Van Leer only)

9.8.5 DIAPHRAGM

Object:

This directive introduces, at the end of a pipeline, a diaphragm that causes a localised pressure drop, and an external imposed pressure.

Syntax:

```
"IMPE"  "DIAP"  "RO" rho  "PFIN" pfin  <"PREF" pref> ...  
        ...  "PINI" pini  "TAU" tau  "K" k  /LECTURE/
```

rho

Density.

pfin

External imposed pressure in steady-state flow.

pref

Reference pressure.

pini

Initial pressure at the diaphragm level.

tau

Time constant of the exponential function that transforms pinit into pfin.

k

Diaphragm head loss coefficient (>1 or $= 1$).

LECTURE

Reading procedure of the CL1D element forming the boundary.

Comments:

The meaning of pref is given on page C.300.

The imposed pressure passes from pini to pfin following an exponential function whose time constant is tau. It is possible to take $\tau = 0$ to represent an abrupt change (depressurisation). However, by giving $\tau \neq 0$ it is possible to simulate a finite opening time of the diaphragm.

The K coefficient allows to account for the diaphragm cross section, that may be smaller than the tube diameter.

$$K = \sup(K_o , k_{si} * R * R)$$

 |_|
 S - s
-----|_|

R is the ratio of cross-sections: $R = S / s$

$$k_{si} = 1 + 0.5 * (1 - 1/R)$$

$$+ \eta * \text{sqr} (1 - 1/R)$$

$$+ \lambda * \text{long} / D_h$$

K_o is the coefficient corresponding to $S = s$: $1.06 < K_o < 1.10$

The preceding formula is taken from IDEL'CIK for the openings with a thick diaphragm, and for large Reynolds numbers.

The η coefficient varies along with the ratio long / D_h . It passes from 1.35 to 0 when this ratio goes from 0 to 2.4.

The long parameter is the length of the diaphragm (thickness of the bottom), and D_h the hydraulic diameter ($h = 4 * \text{area} / \text{perimeter}$).

The λ parameter allows to define the head loss along the small tube equivalent to the diaphragm.

If the distribution of velocities at the outlet would be uniform, one would have $K_o = 1$.

Most often, it may suffice to use the IDEL'CIK formula for a thin diaphragm with sharp border:

$$k_{si} = (1 + 0.707 * \text{sqr}(1 - 1/R)) ** 2$$

By full opening, take $K = K_o$, with $K_o = 1.06$.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : density

ECR(3) : mass velocity ($\rho * v$)

9.8.6 MEMBRANE

Object:

Introduces a safety membrane to the extremity of a pipeline (1D), or on the axis of an axisymmetric reservoir (2D). The membrane rupture occurs: either when the pressure in the neighbouring element exceeds the rupture pressure, or when the time exceeds a prescribed value.

Syntax:

```
"IMPE"  "MEMB"  "RO" rho  "PINI" pini  "PFIN" pfin  ...  
... <"PREF" pref>  "TAU" tau  "K" k  $[ "PRUP" prup ;  
                                     "TRUP" trup ]$ /LECTURE/
```

rho

Density.

pini

Initial pressure.

pfin

Imposed external pressure in steady-state flow.

pref

Reference pressure.

tau

Time constant of the exponential function that leads pini to pfin.

k

Head loss coefficient (>1 or $= 1$).

prup

Rupture pressure of the membrane.

trup

Rupture instant.

LECTURE

List of the concerned elements.

Comments:

In the "GEOM" directive, the elements with a material "MEMB" must be listed after the adjacent fluid elements.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : density

ECR(3) : mass velocity

ECR(5) : rupture instant of the membrane

ECR(6) : rupture indicator (0=intact, 1=broken)

9.8.7 CRITICAL MASS FLOW RATE

Object:

Computes the critical mass flow rate for a perfect gas or for a water-steam mixture at the extremity of a pipeline (1D).

Syntax:

```
"IMPE"  "DCRI"  "PINI" pini  "PFIN" pfin < "PREF" pref >  ...  
        "RAP"  rapport  < "KSI" ksiz >  < "MOD" mode >    ...  
        < "TAU" tau >  < "TRUP" trupt >  < "FONC" nufo >    ...  
        /LECTURE/
```

pini

Initial pressure.

pfin

Imposed external pressure in steady-state conditions.

pref

Reference pressure.

rapport

Ratio of the cross-sections : $S(\text{exit})/S(\text{upstream})$.

ksiz

Head loss coefficient at the outlet for the fluid.

mode

Choice of the critical mass flow rate model (for the two-phase water only : see below)

tau

Time constant of the exponential function leading from pini to pfin.

trupt

Rupture instant of the membrane. By default, $\text{trupt} = 0$.

nufo

function number for a progressive opening.

LECTURE

List of the concerned elements.

Comments:

The user may choose among four modes for the equilibrated two-phase water, which differ in the phase-to-phase sliding:

mode = 1 : homogeneous model

mode = 2 : MOODY model

mode = 3 : FAUSKE model

mode = 4 : DENT model (developped by M. Lepareux)

For the three first modes, the results do not depend upon the size of the opening. On the contrary, the DENT mode developped by Michel Lepareux [581] accounts for this effect provided the following value is not exceeded (rapport=ratio):

$$\text{rapport} < \text{or} = 0.75$$

For the liquids, the **ksiz** coefficient allows to account for the form of the outlet. By default, the value suggested by IDEL'CIK is retained for **ksiz**, corresponding to a straight pipe outlet and a thin diaphragm with sharp sides, i.e.:

$$ksiz = (1 + 0.707\sqrt{1 - rapport})^2$$

Of course, with the data of the upstream medium, the formula becomes:

$$ksi = \frac{ksiz}{rapport^2}$$

The old data input files included a parameter K, equal to 'ksi' with **ksiz** = 1. In order to better reproduce the phenomena, it is suggested to no longer use K, and to declare the two parameters 'ksiz' and 'rapport' instead.

Outputs:

The different components of the ECR table are as follows:

ECR(1) : pressure at the opening

ECR(2) : density of the donor element

ECR(3) : mass velocity ($Q = \rho \cdot V_n$) of the donor

ECR(4) : outlet pressure (P_{ext} or P_{crit})

ECR(6) : indicator (0 if liquid; 1 if gas; mass title if two-phase mixture)

ECR(7) : Mach number at the outlet (only for perfect gases and two-phase water)

ECR(9) : maximum mass velocity (Q_{max}), if $ECR(4) = P_{crit}$

ECR(10) : indicator (0 = virgin membrane, 1 = ruptured membrane)

9.8.8 CLOSED BOTTOM

Object:

This option allows to impose a closed bottom condition at the extremity of a pipeline, by means of a CLTU element.

Syntax:

```
"IMPE"  "FOND"  /LECTURE/
```

LECTURE

List of the elements concerned.

Comments:

This directive automatically ensures the fluid-structure coupling between the pipe and the internal fluid.

Warning

The couplings between degrees of freedom are done directly in the CLTU element, therefore it is necessary that the number of the CLTU element concerned be greater than all other elements that arrive in the same point (TUBE, TUYA, POUT). This in order to have, at the moment of computing the coupling, the resultant of the applied forces. A very simple way of proceeding is to declare the CLTU elements as the last ones in the GEOM directive.

9.8.9 PUMP

Object:

This option allows to impose on a pipeline, by means of a CL1D or CLTU element, a pump model represented by its characteristic curve.

Syntax:

```
"IMPE"  "POMP"  "R0" rho  "COEF" coef  "NUF0" nf  /LECTURE/
```

rho

Density.

coef

Multiplicative coefficient allowing to define the functioning direction of the pump, and to convert the units.

nf

Number of the function to be used to describe the characteristic curve of the pump as a function of the volume flow rate.

LECTURE

List of the concerned elements.

Comments:

The pressure difference is of the form:

$$\Delta P = C f(Q_v)$$

where C is coef, Q_v is the volume flow rate and f the function that models the characteristic.

Warning

The characteristic curve of the pump must be given by means of the keyword **FONCTION** in the following way: in abscissa the volume flow rate and in ordinate the height of the pump. Be careful that the chosen units are coherent! For example, pump height in Pascal and flow rate in m^3s^{-1} . In the case of a normal functioning of the pump, all these values (pressure and flow rate) are positive.

The coefficient **COEF** allows to use different units for the pressures, and its sign allows to orient the pump as a function of the positive orientation associated with the node where the pump is located.

Thus, if the numbers of the elements located at either part of the node where the pump is located are in growing order along the direction of the flow, then the **COEF** coefficient is positive.

If the flow is in the opposite direction than the normal (the fluid flows back through the pump), it is assumed that the head loss is zero. Furthermore, in accident calculations, it is prudent to foresee a characteristic (even zero) for very large flow rates.

Outputs:

The different components of the ECR table are as follows:

ECR(1) : pressure difference

ECR(2) : density

ECR(3) : mass velocity

9.8.10 FOLLOWING FORCE

Object:

This directive allows to impose, at the extremity of a pipeline, by means of a CLTU element, a “following force” applied along the direction of the last element of the pipeline.

Syntax:

```
"IMPE"  "FSUI"  "R0" rho  "FORC" fs  "NUF0" nf  /LECTURE/
```

rho

Density.

fs

Multiplicative coefficient for the force.

nf

Number of the function used to describe the force as a function of time.

LECTURE

List of the concerned elements.

Comments:

A positive force will be directed towards the interior of the pipe.

The function to be used will be defined by means of the "FONC" directive described on page E.10.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : density

ECR(3) : following force

9.8.11 CRITICAL MASS FLOW RATE COUPLING (NAH2)

Object:

Computation of the critical mass flow rate of a water-steam mixture at the extremity of a pipeline (1D) accounting for the pressure produced by the hydrogen bubble generated by a sodium-water reaction. This material uses a model with 2 components (sodium and hydrogen) and two phases. The starting instant of the tube opening may be prescribed in advance (case of several pipes whose rupture times are different)

Syntax:

```
"IMPE"  "NAH2"  "RO" rho  "PINI" pini  <"PREF" pref>
...  "TAU" tau  "K" k  "TUBE" ntube  "MOD" mode
...    <"TRUP" trup> <"PFIN" pfin> /LECTURE/
```

rho

Density.

pini

Initial pressure.

pref

Reference pressure.

tau

Time constant of the exponential (see DIAP page C.550).

k

Head loss coefficient (single-phase flow).

ntube

Total number of ruptured tubes.

mode

Choice of the critical flow rate model.

trup

Rupture instant (zero by default).

`pfin`

Imposed final pressure.

LECTURE

List of the concerned elements.

Comments:

The user may choose among four modes:

mode = 1 : homogeneous model

mode = 2 : MOODY model

mode = 3 : FAUSKE model

mode = 4 : DMT model (developped by M. Lepareux)

This directive is similar to "DCRI" and must be used in conjunction with the "NAH2" material within a "TUBE" or "TUYA" element.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : density

ECR(3) : $Q = \rho \cdot V_n$ at the rupture (mass velocity).

ECR(4) : pressure at the outlet (P_{ext} or P_{crit})

ECR(6) : indicator (0= single-phase ; 1= two-phase)

ECR(7) : status of the tube (0 = intact, 1 = ruptured)

ECR(9) : $Q_{max} = (\rho \cdot V_n)$ maximu if one is in the critical regime. This parameter only makes sense for water or for gases.

9.8.12 CRITICAL MASS FLOW RATE COUPLING (RSEA)**Object:**

Computation of the critical mass flow rate of a water-steam mixture at the extremity of a pipeline (1D) accounting for the pressure produced by the hydrogen bubble generated by a sodium-water reaction. This material is similar to the previous one ("NAH2" page C4.710), but is coupled to the "RSEA" material, which uses three components (sodium, hydrogen and argon) and 2 phases.

Syntax:

```
"IMPE"  "RSEA"  "RO" rho  "PINI" pini  <"PREF" pref>
...  "TAU" tau  "K" k  "TUBE" ntube  "MOD" mode
...    <"TRUP" trup> <"PFIN" pfin> /LECTURE/
```

rho

Density.

pini

Initial pressure.

pref

Reference pressure.

tau

Time constant of the exponential (see DIAP page C.550).

k

Head loss coefficient (single-phase flow).

ntube

Total number of ruptured tubes.

mode

Choice of the critical flow rate model.

trup

Rupture instant (zero by default).

`pfin`

Final prescribed pressure.

LECTURE

List of the elements concerned.

Comments:

The user may choose among four modes:

mode = 1 : homogeneous model

mode = 2 : MOODY model

mode = 3 : FAUSKE model

mode = 4 : DMT model (developped by M. Lepareux)

This directive is similar to the "DCRI" directive and must be used in conjunction with the "RSEA" material within a "TUBE" or "TUYA" element.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : masse volumique

ECR(3) : $Q = \rho \cdot V_n$ at the rupture (mass velocity).

ECR(4) : outlet pressure (P_{ext} or P_{crit})

ECR(6) : indicator (0 = single-phase ; 1 = two-phase)

ECR(7) : tube status (0 = intact, 1 = ruptured)

ECR(9) : $Q_{max} = (\rho \cdot V_n)$ maximum if one is in the critical regime. This parameter makes sense only for water or for a gas.

9.8.13 “VANNE” - SAFETY AND REGULATING VALVES

Object:

This directive allows the user to model the behaviour of a safety and a regulating valve (“vanne”) placed within a pipeline or at its extremity. It introduces a localised pressure drop with a variable pressure loss coefficient depending on the opening cross section of the valve. This impedance, available for the elements of type CL1D and CLTU, is useful to model water hammer effects (“coup de belier”) in the pipeline systems.

For a safety valve, the beginning and the duration of the closure must be specified. For a regulating valve, the user has to specify a tabulated function allowing to govern the valve opening during the calculation (partial or complete opening or closure).

Syntax:

for a safety valve:

```
"IMPE"  "VANN"  "TFER" tferm  "TAU" tau  "SMIN" smin  "PREF" pref
          /LECTURE/
```

for a regulating valve:

```
"IMPE"  "VANN"  "SMIN" smini  "PREF" pref  "NFSL" numfo
          /LECTURE/
```

tferm

Initial time of the valve closure.

tau

Duration of the closure.

smin

Minimum cross section, below which the valve is considered as closed.

pref

Reference pressure.

numfo

Number of the function prescribing the time variation of the ratio $\frac{S_{free}}{S_{upstream}}$.

LECTURE

List of the elements concerned.

Comments:

The parameters **SMIN** and **PREF** are compulsory.

Safety valve:

Initially the valve is considered to be open. It starts to close at time **tferm**, and the closure has a duration of **tau**. It is assumed that the opening cross-section varies linearly between time **tferm** and **tferm + tau**, at which the valve is completely closed.

During the closure, the pressure drop varies as a function of the opening cross-section:

$$K(t) = \left(\frac{S_{upstream}}{S_{free}} \right)^2 - 1$$

Its value therefore varies from zero (fully open valve) to K_{max} , which is determined by assuming that the closure is complete when $S_{free} < \mathbf{SMIN}$. In most cases the value 0.01 may be assumed for **SMIN**.

Once the valve closed, a blockage condition (CL1D) or closed end condition (CLTU) is applied to the node to which the element is attached. In order to know whether the valve is open or closed, EUROPLEXUS checks the value of **ECR(6)**.

Regulating valve:

For the regulating valve a tabulated function given by the user prescribes the time variation of the valve cross section ($\frac{S_{free}}{S_{upstream}}$ ratio) allowing thus to govern the valve opening during the calculation.

Outputs:

The components of the **ECR** table are as follows:

ECR(1) : pressure drop

ECR(2) : density of the upstream element

ECR(4) : ratio of cross-sections ($\frac{S_{free}}{S_{upstream}}$)

ECR(6) : if =0 the valve is open, if =1 the valve is closed

ECR(7) : time

ECR(8) : pressure drop coefficient $K(t)$

9.8.14 SWING CHECK VALVE WITH FLUID-STRUCTURE COUPLING

Object:

This directive allows the user to enter along a pipeline an anti-backflow valve which can close rapidly when the flow is inversed. The valve disc dynamics is governed by its angular inertia as well as by different moments due to the disc weight and hydrodynamic forces. The disc motion causes a localised pressure drop taken into account via a variable head loss coefficient depending on the aperture of the valve disc. The model is attached to CL1D elements.

Syntax:

```
"IMPE"  "SWVA"  "MASS" mass <"RO"  rho>  "ITOT" itot  <"PREF" pref>
          "STUB" stub  "DIST" dist  <"AINI" aini>  <"AMAX" amax>
          <"POPE" pope> "FNUM" nume  /LECTURE/
```

mass

Mass of the valve disc.

rho

Fluid density.

itot

Total moment of inertia accounting for the disc and added fluid inertia.

pref

Reference pressure.

stub

Flow-tube cross section.

dist

Length of the valve disc moment arm.

aini

Initial opening angle.

amax

Maximum opening angle.

pope

Opening pressure.

nume

Number of the singular head loss $K(\alpha)$ curve.

LECTURE

Reading procedure of the CL1D element forming the boundary.

Comments:

The meaning of pref is given on page C.300.

The head loss (DP) is deduced from the density (ρ) and the velocity (V) up-stream of the singularity and it is function of variable head loss coefficient depending on the aperture angle of the disc:

$$DP = 0.5 * k(\alpha) * \rho * V * V$$

The result is a resisting force which is always opposed to the velocity.

The integration of the disc motion equation is abandoned after closure of the valve and EUROPLEXUS replaces the boundary condition by a zero velocity condition (case of CL1D). The valve can reopen when DP exceeds the opening pressure specified by the user.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : pressure drop

ECR(2) : fluid density

ECR(3) : mass velocity ($\rho * v$)

ECR(4) : reference pressure

ECR(5) : current angle of disc (degrees)

ECR(6) : current angular velocity of disc (rad/s)

ECR(7) : current time then $t+dt$

ECR(8) : current head loss coefficient

9.8.15 FLUIDE-STRUCTURE GRID**Object:**

This directive allows to model for a fluid the influence of grids or perforated plates and to apply the resulting pressure drop to the structure (grid).

See also on page C.530 the GRILLE directive.

Syntax:

```
"IMPE"  "GRFS"  "R0" rho  "C" c  "ALP" alpha  ...  
        ...  "TAU" tau  /LECTURE/
```

rho

Density.

c

Sound speed in the fluid.

alpha

Dissipative impedance.

tau

Time constant.

LECTURE

Lecture procedure of the elements concerned.

Comments:

The model assumes that plane acoustic waves are propagated.

The meaning of the alpha and tau parameters is as follows:

Let:

L : equivalent length of the grid holes

ST : total cross-section

s : flow cross-section

M : mach number of the permanent upstream flow

k : head loss coefficient

Then:

$$\alpha = 0.5 * k * M \quad \tau = \frac{ST}{s} * \frac{L}{2*c}$$

Remarks:

The ratio s / ST represents the perforation ratio of the plate.

The head loss coefficient k takes the form:

$$DP = 0.5 * k * \rho * V^2$$

Recall that for an absorbing boundary $\alpha = i$, since the pressure and flow rate fluctuations are in quadrature.

The equivalent length L is not equal to the plate thickness. To account for three-dimensional effects, to this thickness a further length must be added, of the order of the orifice diameter.

Warning:

EUROPLEXUS automatically searches for the structure nodes that ‘touch’ the boundary condition elements. It is then mandatory to mesh the structure in the same way as the boundary condition elements (same mesh density and same topology of the faces in contact).

Outputs:

The various components of the ECR table are as follows:

ECR(1) : pressure

ECR(2) : density

9.8.16 PERFORATED PLATE (JRC)**Object:**

This instruction enables the modelling of a perforated structure (e.g. a plate) embedded in a fluid.

The pressure drop across the plate is computed according to the expression:

$$\text{Deltap} = \text{zeta} * \text{rho} * \text{v} * \text{v} / 2.0$$

Here zeta is the resistance coefficient, rho the fluid density and v the velocity normal to the plate in the undisturbed upstream region of the fluid.

Syntax :

```
"IMPE"  "PPLT"  "ZETA" zeta  
/LECTURE/
```

zeta

Resistance coefficient (assumed as constant in the present model).

/LECT/

Concerned elements.

Outputs:

The different components of the ECR table are as follows :

ECR(1) : current pressure drop across the plate

ECR(2) : density upstream the plate

ECR(3) : resistance coefficient

ECR(4) : structural node 1

ECR(5) : structural node 2

ECR(6) : structural node 3

ECR(7) : structural node 4

ECR(8) : unused

ECR(9) : unused

Note that the positions 4 to 7 contain up to 4 indexes of the structural nodes corresponding to the CLxx element's (fluid) nodes. These quantities are determined only once at the beginning of the calculation and never change. Furthermore, the resistance coefficient is also assumed as constant in the present implementation.

Comments:

Normally, the determination of the above mentioned node correspondence is performed automatically. However, in case of problems the user may either change the tolerance for node matching (see OPTI TOLC on page H.40) or force the node correspondence by using the directive COMP CNOD, see page C.92.

9.8.17 RUPTURE DISK (JRC)

Object:

This instruction enables the modelling of a rupture disk structure (e.g. a plate) embedded in a fluid.

The disk reacts to the pressure drop caused by a fluid on the two sides of the disk, until a certain *deltap* is reached, which causes the rupture of the disk.

Usually, the disk is modelled by a series of structural finite elements (plate, shell) and the fluid on both sides is discretized by continuum finite elements.

Special boundary condition elements (CLxx) are attached to the fluid nodes in order to connect the disk with the fluid. These elements are assigned the present IMPE RUDI material.

The disk rupture is caused by a fixed (nominal) *deltap*, which can occur at any point of the disk, in case this is discretized by several finite elements. When one element reaches this condition, failure is initiated also in all other elements at the same time.

A certain rupture time interval can be prescribed from failure initiation to failure completion in order to simulate the fact that in reality the disk is a mechanical system with inertia and can not break instantaneously.

Unlike in the models of safety valves, note that once the disk rupture is initiated at a certain point, it continues until full failure even if the *deltap* is reduced.

Syntax :

```
"IMPE"  "RUDI"  "DPRU" dpru      < "TRUP" trup >  
/LECTURE/
```

dpru

Pressure difference (*deltap*) between the two sides that causes rupture of the disk. The *deltap* is measured between the fluid elements directly attached to the disk on each side. The *deltap* includes only the fluid pressure and not the pseudo-viscous pressure. However, the pseudo-viscous pressure is taken into account in the computation of the resistance forces developed by the disk.

trup

Rupture time interval of the disk. By default it is $\text{trup} = 0$ (instantaneous rupture). This can be specified > 0 in order to simulate the inertia of the disk and the gradual opening of the orifice.

/LECT/

Concerned elements.

Outputs:

The different components of the ECR table are as follows :

ECR(1) : current pressure drop across the disk (at this element), doesn't include the pseudoviscous pressure

ECR(2) : 0 for virgin disk, 1 for broken disk

ECR(3) : time at which failure is initiated

ECR(4) : structural node 1

ECR(5) : structural node 2

ECR(6) : structural node 3

ECR(7) : structural node 4

ECR(8) : unused

ECR(9) : unused

Note that the positions 4 to 7 contain up to 4 indexes of the structural nodes corresponding to the CLxx element's (fluid) nodes. These quantities are determined only once at the beginning of the calculation and never change.

Comments:

Normally, the determination of the above mentioned node correspondence is performed automatically. However, in case of problems the user may either change the tolerance for node matching (see OPTI TOLC on page H.40) or force the node correspondence by using the directive COMP CNOD, see page C.92.

9.8.18 STACEY'S 1ST ORDER ABSORBING BOUNDARY (JRC)**Object:**

This instruction enables the modelling of an absorbing boundary according to Stacey's 1st-order law, see R. Stacey, "Improved transparent boundary formulations for the elastic-wave equation", Bull. Seis. Soc. Am., Vol. 78, pp. 2089-2097, December 1988.

The model is only available in conjunction with spectral elements and can be applied only to CL22 elements, in 2D, and CL3Q elements, in 3D.

Syntax :

```
"IMPE"  "STAC"  /LECTURE/
```

```
/LECT/
```

Concerned elements.

Outputs:

The components of the ECR table are as follows. If the material belongs to a CL22 element, then:

```
ECR(1) : unused  
ECR(2) : unused  
ECR(3) : unused  
ECR(4) : unused  
ECR(5) : unused  
ECR(6) : unused  
ECR(7) : unused  
ECR(8) : unused  
ECR(9) : unused
```

If the material belongs to a CL3Q element, then:

ECR(1) : x-component of 1st tangent vector
ECR(2) : y-component of 1st tangent vector
ECR(3) : z-component of 1st tangent vector
ECR(4) : x-component of 2nd tangent vector
ECR(5) : y-component of 2nd tangent vector
ECR(6) : z-component of 2nd tangent vector
ECR(7) : x-component of normal vector (outwards the associated MS38)
ECR(8) : y-component of normal vector (outwards the associated MS38)
ECR(9) : z-component of normal vector (outwards the associated MS38)

Remarks:

It has been noted that the use of this type of absorbing boundary conditions can in some cases reduce the stability step. If needed, the safety coefficient of the calculation may be reduced (from the default value of 0.5) by using the directive "OPTI CSTA", see Group H (Options).

Usually it has been found that OPTI CSTA 0.25 is sufficient to prevent instabilities (especially in 2D), but in some 3D cases it has been necessary to use OPTI CSTA 0.125.

9.8.19 RUPTURE DISK FOR MC FORMULATION (JRC)

Object:

This instruction enables the modelling of a rupture disk structure (e.g. a plate) embedded in a fluid modeled with the MC finite volume formulation.

The disk reacts to the pressure drop caused by a fluid on the two sides of the disk, until a certain Δp is reached, which causes the rupture of the disk.

The mechanical behaviour (deformability) of the disk is not taken into account, so there is no need to introduce structural elements.

The fluid mesh is not continuous, i.e. the rupture disk separates two distinct zones of the fluid domain, and the elements facing each other have coincident (same coordinates) fluid nodes on the discontinuity.

Special boundary condition elements (CLxx) are attached to the fluid element at one of the two sides of the aforementioned discontinuity, no matter which one of them. These elements are assigned the present IMPE RDMC material.

The disk rupture is caused by a fixed (nominal) Δp , which can occur at any point of the disk, in case this is discretized by several CLxx elements. When one element reaches this condition, failure is initiated also in all other elements at the same time.

Unlike in the models of safety valves, note that once the disk rupture is initiated at a certain point, it continues until full failure even if the Δp is reduced.

Syntax :

```
"IMPE"  "RDMC"  "DPRU" dpru
```

```
/LECTURE/
```

```
dpru
```

Pressure difference (Δp) between the two sides that causes rupture of the disk. The Δp is measured between the fluid elements directly attached to the disk on each side.

```
/LECT/
```

Concerned elements.

Outputs :

The different components of the ECR table are as follows :

- ECR(1) : current pressure drop across the disk (at this element)
- ECR(2) : 0 for virgin disk, 1 for broken disk
- ECR(3) : time at which failure is initiated
- ECR(4) : fluid node opposite to CLxx element node 1
- ECR(5) : fluid node opposite to CLxx element node 2
- ECR(6) : fluid node opposite to CLxx element node 3
- ECR(7) : fluid node opposite to CLxx element node 4
- ECR(8) : index of fluid element attached to this CLxx
- ECR(9) : index of fluid element opposed to this CLxx

Note that the positions 4 to 7 contain up to 4 indexes of the fluid nodes opposite to the CLxx element's (fluid) nodes. These quantities are determined only once at the beginning of the calculation and never change.

Comments:

Normally, the determination of the above mentioned node correspondence is performed automatically. However, in case of problems the user may either change the tolerance for node matching (see OPTI TOLC on page H.40) or force the node correspondence by using the directive COMP CNOD, see page C.92.

9.8.20 ABSORBING MATERIAL VAN LEER**Object :**

This directive enables an absorbing boundary conditions for Van Leer elements to be input.

Syntax:

```
"IMPV"  "ABSO"  "RO" rho /LECTURE/
```

rho

Density.

LECTURE

Reading procedure of the numbers of the elements concerned.

9.8.21 CONDITION AT INFINITY VAN LEER**Object:**

This directive allows to impose a rest condition at infinity for a fluid by means of a CL2D element.

Syntax:

```
"IMPV" "INFI" "R0" ro "PRES" press "PREF" pref "GAMA" ga /LECTURE/
```

R0

Density at infinity.

PRES

Pressure at infinity.

PREF

Reference pressure.

GAMA

Value of the ratio of specific values for perfect gases.

LECTURE

List of the elements concerned.

Comments:

The coupling with the GZPV material is automatically ensured.

9.8.22 IMPOSED PRESSURE VAN LEER**Object:**

This instruction enables a pressure at the boundary of different Van Leer elements to be imposed by the means of a "CLxx" element.

Syntax:

```
"IMPV"  "PIMP"  "R0" rho "PRES" pres < "PREF" pref >  
... "GAMA" gamma < "IMPO" impo > /LECTURE/
```

rho

Density.

pres

Constant imposed pressure.

pref

Reference pressure.

gama

Ratio of specific heats.

impo

The value impo=0 corresponds to free pressure, density and velocity (absorbing), impo=1 to constant entropy and mass flow rate, and impo=2 to constant entropy and velocity.

Comments:

For the meaning of pref, see page C.300.

9.8.23 IMPOSED PERFECT GAS MASS FLOW RATE VAN LEER**Object:**

This directive allows to impose a mass flow rate of perfect gas in case of Van Leer elements.

Syntax:

```
"IMPV" "DEGP" "RO" ro "PRES" press "PREF" pref "GAMA" ga  
      "DEBX" dx "DEBY" dy "DEBZ" dz /LECTURE/
```

ro

Initial density.

press

Imposed initial pressure.

pref

Reference pressure.

ga

Value of the ratio of specific values for perfect gases.

dx dy dz

Components of mass flow rate.

LECTURE

List of the elements concerned.

9.8.24 RIGID OBSTACLE VAN LEER**Object:**

This directive allows to impose a total reflection (rigid obstacle) condition in case of Van Leer elements.

Syntax:

"IMPV" "MUR" /LECTURE/

LECTURE

List of the elements concerned.

9.8.25 SAFETY VALVE (JRC)**Object**

This instruction enables the modelling of a safety valve. The valve is characterized by a cross section A (at full opening) a resistance coefficient ζ , the external pressure, a differential pressure that determines the starting of the opening, a differential pressure that determines the full opening, opening and closing time intervals, and a maximum mass flow rate. Only the first three parameters are necessary.

This model is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC.

The nominal opening of the valve a' is a linear function of the differential pressure between the fluid (p) and the external medium (p_{ext}):

$$Dp = p - p_{ext}$$

```

for Dp .le. dpmi      --->  a' = 0
for dpmi < Dp < dpma  --->  a' = [(Dp-dpmi)/(dpma-dpmi)] A
for Dp .ge. dpma      --->  a' = A

```

The actual opening a at a given time depends on the opening or closure times, and on the valve opening reached at the previous time, a_{old} . Let v_a and v_c represent the velocity of aperture and closure of the valve:

$$v_a = A / t_{ope}$$

$$v_c = -A / t_{clo}$$

Then:

```

for a' > aold and tope > 0 --->  a = min[(aold + Dt va), a']
for a' < aold and tclo > 0 --->  a = max[(aold + Dt vc), a']
in all other cases,      --->  a = a'

if a > A --->  a = A
if a < 0 --->  a = 0

```


Then, the current mass flow rate is given by:

$$q = (atub / A) \cdot a \cdot \sqrt{2 \rho (p - p_{ext}) / \zeta}$$

```
if q > qmax ---> q = qmax
if q < 0     ---> q = 0    (no back-flow)
```

where ρ represents the density in the fluid upstream the valve and $atub$ is the area of the flow tube of the safety valve.

References

More information on the formulation of this model may be found in reference [\[94\]](#).

Syntax

```
"IMPE"  "SVAL"  "ATUB" atub "DPMI" dpmi
      < "DPMA" dpma "TOPE" tope "TCLO" tclo
      "QMAX" qmax "PEXT" pext "AMAX" amax
      "CC"   cc >
/LECTURE/
```

atub

Cross-section of the safety valve flow tube.

dpmi

Pressure difference at which the valve starts to open.

dpma

Pressure difference at which the valve is fully open; by default, it is equal to **dpmi**.

tope

Opening time interval; by default, it is 0.

tclo

Closure time interval; by default, it is 0.

qmax

Maximum mass flow rate; by default it is infinite (no limit).

pext

External pressure; by default it is 0.

amax

Maximum opening area of the valve. Must be $\text{amax} \leq \text{atub}$. By default, it is $\text{amax} = \text{atub}$.

cc

Flow contraction coefficient ($0 < \text{cc} \leq 1$). By default it is $\text{cc} = 1$.

LECTURE

Reading procedure of the number of the "CLxx" element defining the boundary.

Outputs:

The different components of the ECR table are as follows :

- ECR(1) : current internal fluid pressure
- ECR(2) : current nominal opening area
- ECR(3) : current actual opening area
- ECR(4) : previous time this element was treated
- ECR(5) : previous actual opening area
- ECR(6) : total ejected mass
- ECR(7) : total ejected energy (by mass transport)
- ECR(8) : current flow tube velocity
- ECR(9) : current internal fluid density

9.8.26 RUPTURE DISK (JRC NEW)

Object

This instruction enables the modelling of a rigid rupture disk structure (e.g. a plate) embedded in a fluid.

This model is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC.

The disk reacts to the pressure drop caused by a fluid on the two sides of the disk, until a certain Δp is reached, which causes the rupture of the disk.

In this particular model the disk structure is NOT represented, while the fluid on both sides is discretized by continuum finite elements.

Special boundary condition elements (CLxx) are attached to the fluid nodes in order to represent the action of the disk on the fluid. These elements are assigned the present IMPE RDK2 material.

The disk rupture is caused by a fixed (nominal) Δp , which can occur at any point of the disk, in case this is discretized by several CLxx elements. When one CLxx element reaches this condition, failure is initiated also in all other elements at the same time.

As long as the disk is not ruptured, it acts as a rigid boundary with infinite friction. In other words, all fluid nodes attached to the disk are completely blocked.

The blocking forces are instantly removed as the disk breaks.

References

More information on the formulation of this material model may be found in reference [\[131\]](#).

Syntax

```
"IMPE"  "RDK2"  "DPRU"  dpru  /LECTURE/
```

dpru

Pressure difference (Δp) between the two sides that causes rupture of the disk. The Δp is measured between the fluid elements directly attached to the disk on each side. The Δp includes only the fluid pressure and not the pseudo-viscous pressure.

/LECT/

Concerned elements.

Outputs :

The different components of the ECR table are as follows :

ECR(1) : current pressure drop across the disk (at this element), doesn't include the pseudoviscous pressure

ECR(2) : 0 for virgin disk, 1 for broken disk

ECR(3) : time at which failure occurred

ECR(4) : unused

ECR(5) : unused

ECR(6) : unused

ECR(7) : unused

ECR(8) : unused

ECR(9) : unused

9.8.27 ABSORBING MATERIAL (Jrc implementation)

Object :

This option enables to specify absorbing or partially absorbing boundary conditions for 2-D or 3-D elements developed at JRC (CL22, CL3I or CL3Q).

There exists a similar, but not identical, absorbing boundary model developed at CEA which is appropriate for their CLxx elements (CL1D, CL2D, CL3D or CL3T), see page C.610.

Only pressure waves normal to the boundary are absorbed. The model consists simply in applying a fictitious external pressure $p = -\rho c v_n$, where ρ is the density of the material at the boundary, c its sound speed and v_n the normal component of the particle velocity at the boundary, in Lagrangian calculations, or of the relative (particle minus mesh) velocity in Eulerian or ALE calculations. The “internal” forces due to the absorbing boundary are finally computed by spatial integration of the pressure p (and not, like in the IMPE ABSO material, of a modified pressure $\pi = (p + p_{\text{old}})/2$, where p_{old} is the value of π at the previous time integration step).

Syntax:

```
"IMPE"  "ABSI"  <"R0" rho>  <"C" c>  /LECTURE/
```

rho

Fixed, user-imposed value of the density. If omitted, the code will try to determine the density automatically.

c

Fixed, user-imposed value of the sound speed. If omitted, the code will try to determine the sound speed automatically. Since for a structural material the code is sometimes unable to determine the sound speed automatically, the value becomes useful in this case (and is of course constant). However, since the physical sound speed is fairly constant in such a case, the behaviour of the model should be quite good. A notable exception are materials CAMC and CLAY, for which the sound speed varies considerably: for these materials the code is indeed able to retrieve the current sound speed automatically, so that specifying c in these cases is unnecessary.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL22, CL3I or CL3Q).

Comments :

If the acoustic waves are to be absorbed, the `rho` and `c` parameters must be the same on both sides of the boundary. The effect will then be that of an infinite medium. In the opposite case, there will be partial reflections.

If the user has omitted `c` and the code is unable to determine it automatically, an error message is issued and the calculation is stopped.

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : normal velocity in the local reference frame

9.8.28 ABSORBING MATERIAL (Zienkiewicz for geotechnical materials)

Object :

This option enables to specify absorbing or partially absorbing boundary conditions for 2-D or 3-D elements developed at JRC (CL22, CL3I or CL3Q).

The formulation of this model is due to Zienkiewicz (Computational Geomechanics) and is related to geotechnical materials only.

Both the pressure waves normal to the boundary and those tangential to the boundary are absorbed. The model consists simply in applying a fictitious external pressure $p = -\rho cv$, where ρ is the density of the material at the boundary, c its sound speed and v the appropriate component (normal or tangential) of the particle velocity at the boundary (in Lagrangian calculations). The “internal” forces due to the absorbing boundary are finally computed by spatial integration of the pressure p (and not, like in the IMPE ABSO material, of a modified pressure $\pi = (p + p_{\text{old}})/2$, where p_{old} is the value of π at the previous time integration step).

For the normal component one has:

$$p_n = \rho c_n v_n$$

where v_n is the particle velocity normal to the boundary and the normal sound speed c_n is computed by:

$$c_n = \sqrt{k/\rho}$$

where k is given by:

$$k = \frac{E(1-\nu)}{(1+\nu)(1-2\nu)}$$

.

For the tangential component (2 components in 3D) one has:

$$p_t = \rho c_t v_t$$

where v_t is the particle velocity tangential to the boundary and the tangential sound speed c_t is computed by:

$$c_t = \sqrt{G/\rho}$$

where G (the shear modulus) is given by:

$$G = \frac{E}{2(1+\nu)}$$

.

Syntax:

```
IMPE ABSZ <R0 rho> <CN cn> <CT ct> /LECTURE/
```

rho

Fixed, user-imposed value of the density. If omitted, the code will try to determine the density automatically.

cn

Fixed, user-imposed value of the normal sound speed. If omitted, the code will try to determine the sound speed automatically. Since for a structural material the code is sometimes unable to determine the sound speed automatically, the value becomes useful in this case (and is of course constant). However, since the physical sound speed is fairly constant in such a case, the behaviour of the model should be quite good. A notable exception are materials **CAMC** and **CLAY**, for which the sound speed varies considerably: for these materials the code is indeed able to retrieve the current sound speed automatically, so that specifying **cn** in these cases is unnecessary.

ct

Fixed, user-imposed value of the tangential sound speed. In 3D, this is the tangential speed in both tangential directions. If omitted, the code will try to determine the sound speed automatically. Since for a structural material the code is sometimes unable to determine the sound speed automatically, the value becomes useful in this case (and is of course constant). However, since the physical sound speed is fairly constant in such a case, the behaviour of the model should be quite good. A notable exception are materials **CAMC** and **CLAY**, for which the sound speed varies considerably: for these materials the code is indeed able to retrieve the current sound speed automatically, so that specifying **cn** in these cases is unnecessary.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL22, CL3I or CL3Q).

Comments :

If the acoustic waves are to be absorbed, the **rho**, **cn** and **ct** parameters must be the same on both sides of the boundary. The effect will then be that of an infinite medium. In the opposite case, there will be partial reflections.

If the user has omitted **cn** or **ct** and the code is unable to determine them automatically, an error message is issued and the calculation is stopped.

Outputs:

The different components of the ECR table are as follows :

ECR(1) : normal pressure

ECR(2) : density

ECR(3) : normal sound speed

ECR(4) : normal velocity

ECR(5) : tangential sound speed

ECR(6) : tangential velocity

ECR(7) : tangential pressure

9.8.29 AIR BLAST WAVE

Object :

This directive simulates an explosion in the air (see References below). It allows to load the structures without having to model the fluid domain. It does not take into account **multiple** wave reflections on structural walls, but optionally allows to take into account in a very simplified way the first wave reflection at a wall (see below).

The position of the charge may be specified either by giving its coordinates (x , y and z) or the node at which the charge is placed.

Syntax:

```
"IMPE"  "AIRB"  |[ "X" x "Y" y <"Z" z> ; "NODE" /LEC1/ ]|
           "MASS" m $[ "TINT" t ; "TAUT" ]$ <"OPOS">
           <"PMA"> <"TD"> <"B">
           <"CONF" c> /LECTURE/
```

x

X-coordinate of the explosive source.

y

Y-coordinate of the explosive source.

z

Z-coordinate of the explosive source. This is 0 by default.

NODE /LEC1/

Introduces the node where the explosive charge is located. Typically, a PMAT element may be located at the charge position, so as to be able to visualize it.

m

Mass of the explosive in Kilograms.

t

Starting time of the explosion. By default it is equal to the initial time of the calculation.

TAUT

Indicates that the starting time is calculated automatically by the code, in such a way that the air blast wave reaches the first CLxx element shortly after the starting of the calculation. This is to avoid an “idle” calculation at the beginning of the transient.

OP05

Indicates that only the part with the positive pressure (overpressure) is regarded. After the time of duration of the positive phase the pressure is set to 0.

c

Choice between different available explosion models, see the References below. By default it is 1 (unconfined, reflected, Kingery). The term “unconfined” below means that the explosion takes place in an unconfined space, as opposed to “half-confined” where the charge is placed close to a rigid ground and so the wave propagation occurs in a half-space (experimentally, the measured pressure is somewhat lower in this case because some of the energy is absorbed by the ground). The term “reflected” hereafter means that the model accounts for the pressure increase due to (first) wave reflection at a rigid wall as it is typically measured in experiments. The pressure value in this case may be between 2 and 8 times the incident pressure in the “non-reflected” case, i.e. without taking into account this first reflection.

1. unconfined (full space), reflected (Kingery)
2. unconfined (full space), not reflected (Kingery)
3. unconfined (full space), not reflected (Kinney)
4. half-confined (half space), reflected (Kingery)
5. half-confined (half space), not reflected (Kingery)
6. Blast parameters given

CONF 6 indicates that the blast parameters P_{MAX}, TD and B are given and should not be calculated. All other parameters are not allowed. Only the positive pressure (overpressure) is considered. The pressure-time function is identical on each element.

/LECT/

Elements concerned. These must be of type CLxx.

Comments :

This model requires that the user adopts the standard Unit system, i.e. metres, Kilograms, seconds.

Care must be taken in the orientation of the CLxx elements, in such a way that the pressure load resulting from the AIRB model acts in the right sense.

The force generated by a positive AIRB overpressure pushes the CLxx element in the sense of the normal to the element itself. This normal is determined by the numbering of the element in EUROPLEXUS.

Therefore, typically the CLxx elements must be oriented in such a way that their normal points **away from** the AIRB charge, or away from the direction from which the AIRB overpressure is expected to arrive.

This convention has been assumed because of the possibility of using AIRB model to load not directly a structure, but a fluid boundary. In this case, the CLxx elements must be attached to the fluid boundary (i.e. to continuum-like fluid elements) and the code always orients them (irrespective of the orientation chosen by the user) in such a way that the normal to the CLxx elements points **inside** the fluid.

A final *caveat*: since the orientation of the AIRB load is related to the orientation of the CLxx element, a problem may arise in case of extremely large rotations of the structure (and thus of the attached CLxx element), such as for example in the case of a rupturing structure which breaks up into large flying debris. In fact, if the element rotates by more than, say, 90 degrees, then the AIRB load may appear to act in the wrong direction. This is an inherent limitation of the model which may not be avoided, and the user should be aware of it.

The equations of Kingery are only usable up to a scaled distance of $Z=40$. Above this distance, diagrams of Baker are used (linearised in the double logarithmic scale).

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

References:

For more information on the physical models, consult the following references:

- Kingery, Charles N., Bulmash, Gerald: *Airblast Parameters from TNT Spherical Air Burst and Hemispherical Surface Burst*, Defense Technical Information Center, Ballistic Research Laboratory, Aberdeen Proving Ground, Maryland, 1984.
- Baker, Wilfrid E.: *Explosions in the Air*. University of Texas Pr., Austin, 1973.
- Kinney, G.F., Graham, K.J.: *Explosive Shocks in Air*. Springer, Berlin, 1985.

9.8.30 FRAGILE PLATE**Object:**

This instruction enables the modelling of a fragile structure (e.g. a plate) embedded in a fluid.

The pressure drop across the plate is computed according to the expression:

$$\text{Deltap} = p_1 - p_2$$

Here p_1 and p_2 are the pressures on the two sides of the plate.

As long as the plate element holds, it prevents any fluid from passing through it. As soon as the structural element fails, the structure is replaced by a cloud of debris particles and the fluid may start to flow through.

Syntax :

```
"IMPE"  "FPLT"  
        /LECTURE/
```

```
/LECT/
```

Concerned elements.

Outputs:

The different components of the ECR table are as follows :

ECR(1) : current pressure drop across the plate

ECR(2) : unused

ECR(3) : unused

ECR(4) : structural node 1

ECR(5) : structural node 2

ECR(6) : structural node 3

ECR(7) : structural node 4

ECR(8) : unused

ECR(9) : unused

Note that the positions 4 to 7 contain up to 4 indexes of the structural nodes corresponding to the CLxx element's (fluid) nodes. These quantities are determined only once at the beginning of the calculation and never change.

Comments:

Normally, the determination of the above mentioned node correspondence is performed automatically. However, in case of problems the user may either change the tolerance for node matching (see OPTI TOLC on page H.40) or force the node correspondence by using the directive COMP CNOD, see page C.92.

9.8.31 CLVF ABSORBING MATERIAL

Object :

This option enables to specify absorbing or partially absorbing boundary conditions for 1-D, 2-D or 3-D Cell-Centred Finite Volumes (VFCC).

Beware that the CLVF models are still experimental and under development. Only the **ABSO**, **LOD1**, **FOUR** and **RIEM** models have been tested somewhat (the **ABSO** being by far the most used one), but are for the moment available only in 3D, for perfect gas material and for first-order in space and in time VFCC formulations. Furthermore, they may function with only some of the flux solvers available. For an overview of the state of the art of these developments see references [\[751\]](#) and [\[752\]](#).

Syntax:

```
"CLVF"  "ABSO"  "R0" rho /LECTURE/
```

rho

Fixed, user-imposed value of the density.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL1D, CL2D, CL3D or CL3T).

Comments :

If the acoustic waves are to be absorbed, the **rho** parameter must be the same on both sides of the boundary. The effect will then be that of an infinite medium. In the opposite case, there will be partial reflections.

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : normal velocity

ECR(5) : tangential velocity

ECR(6) : tangential velocity (2nd component in 3d)

9.8.32 CLVF CONDITIONS AT INFINITY

Object :

This option enables to specify conditions at infinity for a fluid modelled by Cell-Centred Finite Volumes (VFCC).

Beware that the CLVF models are still experimental and under development. Only the ABSO, LOD1, FOUR and RIEM models have been tested somewhat (the ABSO being by far the most used one), but are for the moment available only in 3D, for perfect gas material and for first-order in space and in time VFCC formulations. Furthermore, they may function with only some of the flux solvers available. For an overview of the state of the art of these developments see references [751] and [752].

Syntax:

```
"CLVF" "INFI" "R0" rho "PRES" pres <"PREF" pref> "GAMA" gama
/LECTURE/
```

rho

Density at infinity.

pres

Pressure at infinity.

pref

Reference pressure at infinity.

gama

Gamma (ratio of specific heats) at infinity.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL1D, CL2D, CL3D or CL3T).

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : normal velocity

ECR(5) : tangential velocity

ECR(6) : tangential velocity (2nd component in 3d)

9.8.33 CLVF IMPOSED PRESSURE

Object :

This option enables to specify an imposed outside pressure on a fluid modelled by Cell-Centred Finite Volumes (VFCC).

Beware that the CLVF models are still experimental and under development. Only the ABSO, LOD1, FOUR and RIEM models have been tested somewhat (the ABSO being by far the most used one), but are for the moment available only in 3D, for perfect gas material and for first-order in space and in time VFCC formulations. Furthermore, they may function with only some of the flux solvers available. For an overview of the state of the art of these developments see references [\[751\]](#) and [\[752\]](#).

Syntax:

```
"CLVF"  "INFI" "R0" rho "PRES" pres <"PREF" pref> "GAMA" gama  
        <"IMPO" impo>  
        /LECTURE/
```

rho

Density.

pres

Pressure.

pref

Reference pressure.

gama

Gamma (ratio of specific heats).

impo

By default this is 2.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL1D, CL2D, CL3D or CL3T).

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : normal velocity

ECR(5) : tangential velocity

ECR(6) : tangential velocity (2nd component in 3d)

9.8.34 CLVF LODI QUASI 1-D CONDITION**Object :**

This option enables to specify a Lodi quasi 1-D condition on a fluid modelled by Cell-Centred Finite Volumes (VFCC).

Beware that the CLVF models are still experimental and under development. Only the ABSO, LOD1, FOUR and RIEM models have been tested somewhat (the ABSO being by far the most used one), but are for the moment available only in 3D, for perfect gas material and for first-order in space and in time VFCC formulations. Furthermore, they may function with only some of the flux solvers available. For an overview of the state of the art of these developments see references [751] and [752].

Syntax:

```
"CLVF"  "LOD1" "R0"  rho    "PRES" pres  "GAMA" gama
          "VNOR" vnor  <"VTG1" vtg1> <"VTG2" vtg2>
          <"COXA" coxa> <"COYA" coya> <"COZA" coza>
          <"COXB" coxb> <"COYB" coyb> <"COZB" cozb>
          "TYPE" type  "CELP" celp  "VITP" vitp
          "LONP" lonp  <"ROP"  rop>  <"TABT" tabt>
          <"TABO" tabo>
/LECTURE/
```

rho

Density.

pres

Pressure.

gama

Gamma (ratio of specific heats).

vnor

Normal velocity (give a negative value for an undefined velocity at an exit).

vtg1

Tangential velocity (first component).

vtg2

Tangential velocity (second component).

coxa, coya, coza

XA, YA, ZA coefficients.

coxb, coyb, cozb

XB, YB, ZB coefficients.

type

Type of boundary condition.

celp

Reference sound speed.

vitp

Reference normal speed.

lonp

Reference normal length.

rop

Reference density.

tabt

Unknown.

tabo

Unknown.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL1D, CL2D, CL3D or CL3T).

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : normal velocity

ECR(5) : tangential velocity

ECR(6) : tangential velocity (2nd component in 3d)

9.8.35 CLVF FOURIER MODES IN 2D

Object :

This option enables to specify a Fourier modes in 2D condition on a fluid modelled by Cell-Centred Finite Volumes (VFCC).

Beware that the CLVF models are still experimental and under development. Only the ABSO, LOD1, FOUR and RIEM models have been tested somewhat (the ABSO being by far the most used one), but are for the moment available only in 3D, for perfect gas material and for first-order in space and in time VFCC formulations. Furthermore, they may function with only some of the flux solvers available. For an overview of the state of the art of these developments see references [751] and [752].

Syntax:

```
"CLVF"  "FOUR" "RO"   rho    "PRES" pres  "GAMA" gama
          "VNOR" vnor  <"VTG1" vtg1> <"VTG2" vtg2>
          <"COXA" coxa> <"COYA" coya> <"COZA" coza>
          <"COXB" coxb> <"COYB" coyb> <"COZB" cozb>
          "TYPE" type  "CELP" celp  "VITP" vitp
          "LONP" lonp  <"ROP"  rop>  <"TABT" tabt>
          <"TABO" tabo>
/LECTURE/
```

rho

Density.

pres

Pressure.

gama

Gamma (ratio of specific heats).

vnor

Normal velocity (give a negative value for an undefined velocity at an exit).

vtg1

Tangential velocity (first component).

vtg2

Tangential velocity (second component).

coxa, coya, coza

XA, YA, ZA coefficients.

coxb, coyb, cozb

XB, YB, ZB coefficients.

type

Type of boundary condition.

celp

Reference sound speed.

vitp

Reference normal speed.

lonp

Reference normal length.

rop

Reference density.

tabt

Unknown.

tabo

Unknown.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL1D, CL2D, CL3D or CL3T).

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : normal velocity

ECR(5) : tangential velocity

ECR(6) : tangential velocity (2nd component in 3d)

9.8.36 CLVF RIEMANN 3-D CONDITION

Object :

This option enables to specify a Riemann 3-D condition on a fluid modelled by Cell-Centred Finite Volumes (VFCC).

Beware that the CLVF models are still experimental and under development. Only the ABSO, LOD1, FOUR and RIEM models have been tested somewhat (the ABSO being by far the most used one), but are for the moment available only in 3D, for perfect gas material and for first-order in space and in time VFCC formulations. Furthermore, they may function with only some of the flux solvers available. For an overview of the state of the art of these developments see references [751] and [752].

Syntax:

```
"CLVF"  "RIEM" "R0"   rho    "PRES" pres  "GAMA" gama
          "VNOR" vnor  <"VTG1" vtg1> <"VTG2" vtg2>
          <"COXA" coxa> <"COYA" coya> <"COZA" coza>
          <"COXB" coxb> <"COYB" coyb> <"COZB" cozb>
          "TYPE" type  <"TABT" tabt>
/LECTURE/
```

rho

Density.

pres

Pressure.

gama

Gamma (ratio of specific heats).

vnor

Normal velocity (give a negative value for an undefined velocity at an exit).

vtg1

Tangential velocity (first component).

vtg2

Tangential velocity (second component).

`coxa`, `coya`, `coza`

XA, YA, ZA coefficients.

`coxb`, `coyb`, `cozb`

XB, YB, ZB coefficients.

`type`

Type of boundary condition.

`tabt`

Unknown.

LECTURE

Reading procedure of the numbers of the elements composing the boundary (CL1D, CL2D, CL3D or CL3T).

Outputs:

The different components of the ECR table are as follows :

ECR(1) : pressure

ECR(2) : density

ECR(3) : sound speed

ECR(4) : normal velocity

ECR(5) : tangential velocity

ECR(6) : tangential velocity (2nd component in 3d)

9.9 MECHANISMS

Object:

This option allows to assign behaviour laws to the elements of a mechanical joint "MECA" that link together two sub-structures.

There are currently 5 directives, with several sub-directives.

FORC : Imposed force as a function of time over a slider ('glissière').

COUP : Imposed couple as a function of time over a pivot.

FOCO : Force and couple imposed on a sliding pivot.

MOCC : Electric motor with continuous current, on a pivot.

RESS : Linear spring.

Options	Sub-directives	
	mandatory	optional
FORC	FONC	INER
COUP	FONC	INER ASSE
FOCO	FONC	INER
MOCC	FONC ELEC	INER REDU TACH ASSE
RESS	/	/

Syntax:

```
$ "FORC" | <"INER" ...> | /LECTURE/ $
$          | "FONC" ... | $
```

	\$			\$
	\$	"COUP"	<"INNER" ...> /LECTURE/	\$
	\$		"FONC" ...	\$
	\$		<"ASSE" ...>	\$
	\$			\$
"MECA"	\$	"FOCO"	<"INNER" ...> /LECTURE/	\$
	\$		"FONC" ...	\$
	\$			\$
	\$	"MOCC"	<"INNER" ...> /LECTURE/	\$
	\$		"ELEC" ...	\$
	\$		"FONC" ...	\$
	\$		<"REDU" ...>	\$
	\$		<"TACH" ...>	\$
	\$		<"ASSE" ...>	\$
	\$			\$
	\$	"RESS"	/LECTURE/	\$

LECTURE

Index of the associated mechanism element.

Comments:

It is not necessary to systematically give a material law to elements of type "MECA". If this is missing, one will have a simple kinematic joint.

9.9.1 IMPOSED FORCE

Object:

This directive allows to introduce a driving force on a junction of type ‘glissière’ (slider).

Syntax:

"FORC"		"FONC" ...		/LECTURE/
		<"INER" ...>		

Comments:

The behaviour law is defined by an imposed force $F(t)$ un the axis of the slider.

The force is computed by the function: $A : F(t) = \text{Function } A$.

The B function is redundant.

Outputs:

ECR(5) : relative displacement of the slider since the start of the calculation

ECR(6) : relative velocity

ECR(7) : applied force

9.9.2 MOTOR COUPLE

Object:

This directive allows to define a motor couple on a connection of type 'pivot' (simple pin joint).

Syntax:

```
"COUP"      |      "FONC" ...      |      /LECTURE/  
            |      <"INNER" ...>  |  
            |      <"ASSE" ...>   |
```

Comments:

The law of behaviour is define by a motor couple $C(t)$ on the axis of the pivot.

If the motor is not a servomotor: $C(t) = \text{Function A}$. In this case, the B function is redundant.

If the motor is a servomotor: $C(t)$ is computed by the control function.

Outputs:

ECR(1) : angular displacement of the motor from the calculation origin (in radians)

ECR(2) : angular velocity (rad / s)

ECR(3) : couple on the arm

9.9.3 IMPOSED FORCE AND COUPLE

Object:

This directive is a combination of the 2 preceding ones ("FORC" and "COUP"), and is applied to sliding pivots ("PIGL").

Syntax:

"FOCO"		"FONC" ...		/LECTURE/
		<"INER" ...>		

Comments:

The behaviour law is defined by an imposed force $F(t)$ on the slider axis, and by a motor couple $C(t)$ around this axis.

The force is computed from function A: $F(t) = \text{Function A}$.

The couple is computed from function B: $C(t) = \text{Function B}$.

Outputs:

ECR(1) : angular displacement of the motor from the beginning of the calculation (in radians)

ECR(2) : angular velocity (rad / s)

ECR(3) : couple on the arm

ECR(5) : relative displacement of the slider from the beginning of the calculation

ECR(6) : relative velocity

ECR(7) : applied force

9.9.4 DIRECT CURRENT MOTOR

Object:

This directive allows to specify a motor couple on a junction of type 'pivot' (simple pin joint). The behaviour and the material characteristics are those of a direct current motor.

Syntax:

"MOCC"		"INER" ...		/LECTURE/
		"ELEC" ...		
		"FONC" ...		
		<"REDU" ...>		
		<"TACH" ...>		
		<"ASSE" ...>		

Comments:

* $U(t)$ - Tension at the motor poles

* $C(t)$ - Motor couple at the outlet shaft

* $tetap$ - Angular velocity of the motor shaft

The other variables are defined in the electric parameters.

The behaviour law is of the following type.

If the motor is not a servo-motor: $U(t) = \text{Function A}$. The B function is then redundant.

If the motor is a servo-motor: $U(t)$ is computed from the control mechanism:

$$C(t) = (K_c * N * U(t) / R) - (N * N * tetap (F_f + K_c * K_c / R)) - (N * F_s)$$

(for the meaning of parameters K_c , N , R , etc., consult pages C.760 and C.770).

Outputs:

ECR(1) : angular displacement of the motor since the beginning of the calculation
(in radians)

ECR(2) : angular velocity (rad / s)

ECR(3) : couple on the arm

9.9.5 SPRING

Object:

This directive allows to introduce stiffnesses on the free d.o.f.s of mechanisms in order to model linear or non-linear springs.

Syntax:

```
"RESS"      ("K"   "  k | "KFON"  kfon)
              ("C"   "  c | "CFON"  cfon)
              "STAK"  stak      "STAC"  stac      /LECTURE/
```

k

Longitudinal stiffness for the linear spring.

kfon

Function number describing the longitudinal non-linear behavior. The function gives the force depending on the displacement.

c

Rotational stiffness for the linear spring.

cfon

Function number describing the rotational non-linear behavior. The function gives the moment depending on the angular displacement.

stak

Estimation of the maximal longitudinal oscillation period.

stac

Estimation of the maximal rotational oscillation period.

Comments:

This directive may be used in conjunction with a mechanism defined by a 2-nodes "MECA" element and an "ARTI" connection (except a ROTULE) on this very element.

The stiffnesses introduced in the articulated systems are associated with the free d.o.f.s: for example, for a sliding pivot, the stiffness K corresponds to the translational d.o.f. ('glissière'), while the stiffness C corresponds to the free rotational d.o.f. (pivot).

The direction of the displacement for the non-linear spring is defined according to the local axis of the "MECA" element or, in case of merging points, thanks to the axis defined for the connection.

Outputs:

ECR(1) : angular displacement since the beginning of the calculation (in radians)

ECR(2) : angular velocity (rad / s)

ECR(3) : applied couple

ECR(5) : relative displacement of the slider since the beginning of the calculation

ECR(6) : relative velocity

ECR(7) : applied force

ECR(8) : initial length of the MECA element

9.9.6 FUNCTIONS RELATED TO MECHANISMS

Object:

This directive allows to specify the functions used for the articulated systems (mechanisms).

Syntax:

```
"FONC"          "COEA" coea    "NUFA" nufa    ...  
...             <"COEB" coeb>  <"NUFB" nufb>  ...
```

coea

Multiplying coefficient of function A.

coeb

Multiplying coefficient of function B.

nufa

Index of function A.

nufb

Index of function B.

Comments:

This directive is mandatory for articulated systems.

9.9.7 MECHANISM INERTIA

Object:

This directive allows to enter the values of mass and inertia for the articulated systems (mechanisms).

Syntax:

"INER"		"MMT1" m1	"IMT1" i1	...
	...	"MMT2" m2	"IMT2" i2	...

m1

Added mass on the first node of the mechanism.

i1

Added rotational inertia on the first node of the mechanism.

m2

Added mass on the second node of the mechanism.

i2

Added rotational inertia on the second node of the mechanism.

Comments:

The needed values are the additional inertias which have not been taken into account in the structures connected by the articulated system.

Rotational inertias are added along the axis of the mechanism.

Warning:

In the case of the direct current motor "MOCC", it is assumed that the rotor is placed on the first node, and the stator on the second node of the mechanism. Verify that the mesh conforms to this convention.

9.9.8 SERVOMECHANISM

Object:

This directive allows to specify a servo-mechanism.

The servo-control law is of the type P.I.D. (Proportional - Integrated - Derived):

$$F(t) = K_0 - K_i * (I - I_c) - K_p * (A - A_c) - K_v * (A' - A_c')$$

with:

I = integral of the angular position from 0 to t ,

I_c = integral of the command for the position.

Syntax:

"ASSE" <"KP" kp> <"KV" kv> <"KI" ki> <"K0" k0>

k0

Off-set tension constant.

kp

Parameter for the control in position.

kv

Parameter for the control in velocity.

ki

Parameter for the integral control.

Comments:

If this directive is missing, the mechanism is considered non-controlled.

9.9.9 ELECTRIC PARAMETRES

Object:

This directive allows to specify the electric parameters of a direct current motor.

Syntax:

```
"ELEC"  "R" r  <"KC" kc>   <"TNSN" tnsn> ...  
        ... <"INTS" is> <"FFMT" ffmt>  <"FSMT" fsmt>
```

r

Electric resistance of the motor.

kc

Torque constant per Ampère.

tnsn

Saturation tension in Volt.

is

Saturation intensity in Ampère.

ffmt

Viscous friction coefficient.

fsmt

Torque constant for dry friction.

Comments:

The friction values are those of the motor alone.

9.9.10 REDUCER

Object:

This directive allows to add the characteristics of a reducer to a mechanism of type motor.

Syntax:

```
"REDU"    <"MRD1" m1>    <"MRD2" m2>    <"IRD1" i1> <"IRD2" i2>    ...  
          ...    <"FFRD" ff> <"FSRD" fs> <"N" n>    ...
```

m1

Added mass on the first node of the mechanism.

i1

Added rotational inertia on the first node of the mechanism.

m2

Added mass on the second node of the mechanism.

i2

Added rotational inertia on the second node of the mechanism.

n

Reduction ratio.

ff

Viscous friction coefficient.

fs

Torque constant for dry friction.

Comments:

The rotational inertias are applied along the axis of the mechanism.

Warning:

In the case of the direct current motor "MOCC", it is assumed that the rotor is placed on the first node, and the stator on the second node of the mechanism. Verify that the mesh conforms to this convention.

9.9.11 TACHYMETRIC GENERATOR

Object:

This directive allows to add the characteristics of a tachymetric generator to a mechanism of type motor.

Syntax:

```
"TACH"    <"MGT1" m1>    <"MGT2" m2>    <"IGT1" i1>    <"IGT2" i2>  
          ... <"JASS" jbras>
```

m1

Added mass on the first node of the mechanism.

i1

Added rotational inertia on the first node of the mechanism.

m2

Added mass on the second node of the mechanism.

i2

Added rotational inertia on the second node of the mechanism.

jbras

Inertia seen by the mechanism.

Comments:

The "JASS" parameter is only used for the stability step calculation. An estimation, even in first approximation, is sufficient.

The rotational inertias are applied along the axis of the mechanism.

Warning:

In the case of the direct current motor "MOCC", it is assumed that the rotor is placed on the first node, and the stator on the second node of the mechanism. Verify that the mesh conforms to this convention.

9.10 ASSIGNING MATERIALS TO MULTILAYER SHELL ELEMENTS

Object:

When using sandwiches composed of multiple layers in certain shell elements it is necessary to assign a (possibly different) material to each layer.

Sandwiches and layers are defined in the Geometry Complements section via the **SAND** directive, see page C.45.

In order to assign a given material to certain layers, first declare the material with all its properties as usual and list via the **/LECT/** directive all the elements that have that material, including sandwich elements that possess this material in at least one layer. Then, after the **/LECT/** directive, list all the layers which have the material (each layer is identified by a progressive index, as explained on page C.45).

Syntax :

```
"MATE" "Material_Definition"  
  
( /LECT/    < "LAYE" /LECT_LAY/ > )
```

Material_Definition

Definition of the material and its properties (see preceding sections), except the elements to which it is assigned.

/LECT/

Elements possessing this material.

/LECT_LAY/

Layers of the **/LECT/** elements that possess the given material. Each layer is identified by an index, as explained on page C.45.

Comments:

Note that the /LECT/ directive (with its optional LAYE subdirective) may be repeated more than once for the same material. This allows e.g. to assign the same material to a few unlayered elements, then to a group of layered elements (i.e. a sandwich) in layers 1 and 3, then to another group of layered elements (i.e. another sandwich) in layers 2 and 4, and so on. For example:

```
"MATE" "Material_Definition"  
  LECT 1 2 3 TERM  
  LECT 4 6 TERM  LAYE LECT 1 3 TERM  
  LECT 5 8 TERM  LAYE LECT 2 4 TERM  
  . . .
```

The only element types that accept layers are ED01, COQI, CQD3, CQD4, CQD6 and CQD9. Since version 2005, Q4MC and T3MC are also available.

In order to be accepted in a layer, the material type must be available for the unlayered element type as well.

All layers of multilayer elements (sandwiches) must be explicitly assigned a material.

9.11 JOINT PROPERTIES

9.11.1 BUSHING ELEMENT

Object:

All the characteristics of the bushing element have to be given of the material type called "JOINT PROPERTIES". This sub-directive allow the definition of a stiffness or a damping in providing both the amplitude and the number of the function, which describes the force-displacement (force-velocity) or torque-rotation (torque-angular velocity) law. Nevertheless a simplified syntax is allowed for the case of linear laws, which can be expressed only by a constant stiffness/damping coefficient.

Note that two models have been implemented. The first one is "BSHT" with only translation degrees of freedom and the second one is "BSHR" with rotational degrees of freedom too. In the latter case the stiffness and/or damping can be applied also to the rotationnal degrees of freedom.

A suitable criteria to properly introduce a model of rupture in the riveted joints can be used. This criteria is based on the rigid behavior of the joints; the rupture occurs when the following limit curve is reached :

$$(N/N_u)^{**a} + (T/T_u)^{**b} \geq 1$$

Four different laws of behaviour can be used :

- 1) Bushing with elastic behaviour;
- 2) Bushing with plastic behaviour;
- 3) Bushing with elastic behaviour and elliptic rupture criteria;
- 4) Bushing with plastic behaviour and elliptic rupture criteria;

Syntax:

1) Generic data

Each behaviour is declared by a different sub-directive.

- 1) Case 1

"JPRP" "BUSH"

- 2) Case 2

"JPRP" "BPLA"

3) Case 3

"JPRP" "BELC"

4) Case 4

"JPRP" "BPEC"

- 2) Data concerning the law of behaviour

<KTXC ktxc <KTXN ktxn>> <KTYC ktyc <KTYN ktyn>> <KTZC ktzc <KTZN ktzn>>
<KRXC krxc <KRXN krnx>> <KRYC kryc <KRYN kryn>> <KRZC krzc <KRZN krzn>>

<DTXC dtxc <DTXN dtxn>> <DTYC dtyc <DTYN dtyn>> <DTZC dtzc <DTZN dtzn>>
<DRXC drxc <DRXN drxn>> <DRYC dryc <DRYN dryn>> <DRZC drzc <DRZN drzn>>

<V1X v1x V1Y v1y V1Z v1z >
<V2X v2x V2Y v2y V2Z v2z >
<EX ex > <EY ey > <EZ ez >
<LOCA loca>
<DISP disp>
<RADB radb>

ktxr

Amplitude multiplying the function describing X-translational stiffness or constant X-translational stiffness if the function KTXN is not given (REAL).

ktxn

Number of the function describing X-translational stiffness (INTEGER).

ktyr

Amplitude multiplying the function describing Y-translational stiffness or constant Y-translational stiffness if the function KTYN is not given (REAL).

ktyn

Number of the function describing Y-translational stiffness (INTEGER).

ktzr

Amplitude multiplying the function describing Z-translational stiffness or constant Z-translational stiffness if the function KTZN is not given (REAL).

ktzn

Number of the function describing Z-translational stiffness (INTEGER).

krxr

Amplitude multiplying the function describing X-rotational stiffness or constant X-rotational stiffness if the function KTXN is not given (REAL).

krxn

Number of the function describing X-rotational stiffness (INTEGER).

kryr

Amplitude multiplying the function describing Y-rotational stiffness or constant Y-rotational stiffness if the function KTYN is not given (REAL).

kryn

Number of the function describing Y-rotational stiffness (INTEGER).

krzr

Amplitude multiplying the function describing Z-rotational stiffness or constant Z-rotational stiffness if the function KTZN is not given (REAL).

krzn

Number of the function describing Z-rotational stiffness (INTEGER).

dtxr

Amplitude multiplying the function describing X-translational damping or constant X-translational damping if the function KTXN is not given (REAL).

dtxn

Number of the function describing X-translational damping (INTEGER).

dtyr

Amplitude multiplying the function describing Y-translational damping or constant Y-translational damping if the function KTYN is not given (REAL).

dtyn

Number of the function describing Y-translational damping (INTEGER).

dktzr

Amplitude multiplying the function describing Z-translational damping or constant Z-translational damping if the function KTZN is not given (REAL).

dtzn

Number of the function describing Z-translational damping (INTEGER).

drxr

Amplitude multiplying the function describing X-rotational damping or constant X-rotational damping if the function KTXN is not given (REAL).

drxn

Number of the function describing X-rotational damping (INTEGER).

dryr

Amplitude multiplying the function describing Y-rotational damping or constant Y-rotational damping if the function KTYN is not given (REAL).

dryn

Number of the function describing Y-rotational damping (INTEGER).

drzr

Amplitude multiplying the function describing Z-rotational damping or constant Z-rotational damping if the function KTZN is not given (REAL).

drzn

Number of the function describing Z-rotational damping (INTEGER).

v1x v1y v1z

cartesian components for vector V1 of the user defined frame.

v2x v2y v2z

cartesian components for vector V2 of the user defined frame.

ex ey ez

eccentricity of the user defined frame from node 1

loca

location of the user defined frame origin between node 1 and node 2

disp

flag for taking into account initial position (default 0).

radb

flag for the use of the radial bushing element formulation.

- 3) Data concerning the elliptic criteria:

```
<NULT  nult TULT  tult  >
<COEA  coea COEB  coeb  >
```

NULT

ultimate axial effort

TULT

ultimate radial effort

COEA

coefficient a of the elliptic criteria

COEB

coefficient b of the elliptic criteria

Comments:

As default the force-position and force-velocity laws are defined in the global reference frame: however a user-defined properties frame (x_1, x_2, x_3) is accepted via the directives V1X, V1Y, V1Z, V2X, V2Y, V2Z. After the two vectors V1 and V2 have been defined, the code applies a normalization to obtain the unit versors v_1 and v_3 , and computes the v_3 versor as:

$$\{v_3\} = \{v_1\} \wedge \{v_2\}$$

The laws are assumed to be diagonal in the properties frame (x_1, x_2, x_3), that is the force F_i along x_i depends only upon the relative position u_i and relative velocity v_i along the x_i local direction.

The user can also define a point P in the properties frame where the force and the torque should be applied: this must be done using the parameter LOCA and, if needed the eccentricity parameters EX, EY, EZ. The effect of the parameter LOCA, which must be a real number in the interval $[0,1]$, is to move the application point on the segment AB: it is actually the percentage of the position of P along AB. The default is LOCA 0.0. If offsets ex, ey, ez are also defined, their effect is to translate along the directions (x, y, z) of the global frame, the point P, from its position P defined by LOCA parameter.

Outputs:

The components of the ECR table are as follows:

- ECR(1): Translation force F_{x1} between the two nodes along $x1$ local axis
ECR(2): Translation force F_{y1} between the two nodes along $y1$ local axis
ECR(3): Translation force F_{z1} between the two nodes along $z1$ local axis
ECR(4): Torque T_{x1} between the two nodes around $x1$ local axis
ECR(5): Torque T_{y1} between the two nodes around $y1$ local axis
ECR(6): Torque T_{z1} between the two nodes around $z1$ local axis
ECR(7): Relative position between the two nodes along $x1$ local axis
ECR(8): Relative position between the two nodes along $y1$ local axis
ECR(9): Relative position between the two nodes along $z1$ local axis
ECR(10): Relative rotation between the two nodes around $x1$ local axis
ECR(11): Relative rotation between the two nodes around $y1$ local axis
ECR(12): Relative rotation between the two nodes around $z1$ local axis
ECR(13): Relative translation velocity between the two nodes along $x1$ local axis
ECR(14): Relative translation velocity between the two nodes along $y1$ local axis
ECR(15): Relative translation velocity between the two nodes along $z1$ local axis
ECR(16): Relative angular velocity between the two nodes around $x1$ local axis
ECR(17): Relative angular velocity between the two nodes around $y1$ local axis
ECR(18): Relative angular velocity between the two nodes around $z1$ local axis
ECR(25): Value of the rupture elliptic criteria

10 GROUP D—LINKS

Object:

To introduce links between degrees of freedom. Links may be subdivided into three broad categories:

- **Coupled** links, which are treated (implicitly) by a method of Lagrange multipliers. This ensures proper coupling between all the imposed links, provided of course the specified conditions are compatible.
- **Decoupled** links, which are imposed via ad-hoc direct methods. In this case the conditions should be independent from one another, and the user is responsible for ensuring this property.
- **"Liaisons"**, which are treated using the former LIAISON directive for compatibility purposes. These links can be either coupled or uncoupled (see below).

Following the above subdivision, this directive admits three forms, characterized by the respective sub-directives: LINK COUP, LINK DECO or LINK LIAI. The complete syntax is summarized below. For each variation of the main directive (COUP, DECO or LIAI) the available link types are listed in the relevant column, so as to provide a compact overview.

Syntax:

\$ LINK COUP	\$ LINK DECO	\$ LINK LIAI	\$
\$ <SOLV . . .>	\$	\$ <SOLV . . .>	\$
\$ <RENU ; NORE>	\$	\$ <RENU ; NORE>	\$
\$ <VERI> <ARRA>	\$	\$ <FREQ ifreq>	\$
\$ <SPLT \$ DOF ;	\$	\$ <VERI>	\$
\$ NODE ;	\$	\$	\$
\$ DOMA ;	\$	\$	\$
\$ PART ;	\$	\$	\$
\$ NONE \$>	\$	\$	\$
\$ <SOL2>	\$	\$	\$
\$ <UPDT /CTIME/>	\$	\$	\$
BLOQ . . . ,	BLOQ . . . ,	BLOQ . . .	
CONT . . . ,		CONT . . .	
RAD1 . . . ,		RAD1 . . .	
REL1 . . . ,		REL1 . . .	
ARMA . . . ,	ARMA . . . ,	ARMA . . .	
CROS . . . ,	CROS . . . ,		
	ACBE . . . ,		

	DEPL	. . .	,		,	DEPL	. . .	
	VITE	. . .	,		,	VITE	. . .	
	ACCE	. . .	,		,	ACCE	. . .	
	COQM	. . .	,		,	COQM	. . .	
	INTE	. . .	,		,			
	FLST	. . .	,		,			
	FLSR	. . .	,	FLSR . . .	,			
	FS	. . .	,		,	FS	. . .	
			,		,	UNIL	. . .	
	IMPA	. . .	,	IMPA . . .	,	IMPA	. . .	
			,		,	JEUX	. . .	
	GLIS	. . .	,	GLIS . . .	,	GLIS	. . .	
			,		,	MPEF	. . .	
			,		,	SPHY	. . .	
	TYPL	. . .	,		,			
	EDEF	. . .	,		,			
	BIFU	. . .	,		,	BIFU	. . .	
	ADHE	. . .	,		,			
	TUBM	. . .	,		,	TUBM	. . .	
	TUYM	. . .	,		,	TUYM	. . .	
	TUYA	. . .	,		,	TUYA	. . .	
	SOLI	. . .	,		,	SOLI	. . .	
	COMP	. . .	,		,	COMP	. . .	
	ARTI	. . .	,		,	ARTI	. . .	
	ROTA	. . .	,		,	ROTA	. . .	
	MENS	. . .	,		,	MENS	. . .	
	DIST	. . .	,		,	DIST	. . .	
	BARY	. . .	,		,	BARY	. . .	
	RIGI	. . .	,		,	RIGI	. . .	
			,		,	SPLI	. . .	
			,		,	COLL	. . .	
	FSA	. . .	,		,	FSA	. . .	
	FSR	. . .	,		,	FSR	. . .	
	PINB	. . .	,	PINB . . .	,	PINB	. . .	
			,	FSS . . .	,			
	SH3D	. . .	,		,	SH3D	. . .	
			,	FLSW . . .	,			
	MAP2	. . .	,		,	MAP2	. . .	
	MAP3	. . .	,		,	MAP3	. . .	
	MAP4	. . .	,		,	MAP4	. . .	
	MAP5	. . .	,		,	MAP5	. . .	
	MAP6	. . .	,		,	MAP6	. . .	
	MAP7	. . .	,		,	MAP7	. . .	
	FESE	. . .	,		,			
	NAVI	. . .	,		,			
	BREC	. . .	,		,			

| , PELM . . . , |

COUP

Introduces the set of coupled links. All coupled links must be declared within this set (the keyword **COUP** may not be repeated).

DECO

Introduces the set of decoupled links. All decoupled links must be declared within this set (the keyword **DECO** may not be repeated).

LIAI

Introduces the set of "liaison" links. All "liaison" links must be declared within this set (the keyword **LIAI** may not be repeated).

Comments:

The **COUP** and **LIAI** subdirectives are mutually exclusive. However, they may be combined with the **DECO** links within the same calculation, with the following syntax:

```
LINK $[COUP ; LIAI]$
      (declare all coupled/"liaison" links here ...)
LINK DECO
      (declare all uncoupled links here ...)
```

The subdirective **LIAI** is not compatible with the other types of links. Only the coupled links declared with **LINK COUP** may be used also in conjunction with *domain decomposition* (see the **STRU** directive), while this is not the case for the **LINK LIAI** directive.

Beware that, for the moment, only the **BLOQ**, **DEPL**, **VITE** and **ACCE** models are accepted in calculations with sub-domains.

Note that the availability of each link formulation introduced above with one or more of the link types is given in their specific manual page below.

Warning:

The **LINK COUP** directive allows to build a coupling matrix between the different degrees of freedom appearing in the connections. This matrix must be invertible, and a problem occurs in this sense if the different connections are not independent from each other.

In principle, EUROPLEXUS is able to eliminate the redundant relations in order to be able to invert the connections matrix. But since this elimination is a trial-and-error process, it is preferable when possible to avoid this situation. If this is not possible, it is recommended to start by giving the most complex connections (solids or articulations), and to finish by giving the simplest ones (relations or blockages).

Comments:

Be sure to check the various options available in relation to connections: see Page H.160.

10.1 LINK CATEGORY

Object:

Choose between the available categories for links (see page D.10).

Syntax:

`$[COUP ; DECO ; LIAI]$`

COUP

Introduces the set of coupled links. All coupled links must be declared within this set (the keyword `COUP` may not be repeated).

DECO

Introduces the set of decoupled links. All decoupled links must be declared within this set (the keyword `DECO` may not be repeated).

LIAI

Introduces the set of "liaison" links. All "liaison" links must be declared within this set (the keyword `LIAI` may not be repeated).

10.2 OPTIONS FOR COUPLED LINKS

Object:

Syntax:

```

LINK COUP
    <SOLV  |[ CHOL                                ;
        SPLI < TYPE  imet > < PCON ipre >
          < PITE  prec > < IPA1 ip1 >
          < IPA2  ip2 > < RPAR  rp  >
          < INIS  inig >                                ]| >
    $[ RENU ; NORE ]$ <VERI>
    <ARRA>
    <SPLT |[ DOF  ; NODE ; DOMA ; PART ; NONE ]| >
    <SOL2>
    <UPDT /CTIME/>

```

Comments:

The **SOLV** directive is fully described in [10.3](#) (page D.20).

Concerning imposed motions (directives **DEPL**, **VITE** and **ACCE**), note that no dimensions are needed relative to the motions themselves. However, dimensioning relative to the time tables describing such motions is still necessary (see keywords **FNOM**, **FTAB**).

The **ARMATURE** directive is described in [10.9](#) (page D.125).

The optional keyword **RENU** makes it possible to renumber the links in order to minimize the size of the matrix.

By default, links are renumbered (option **RENU**).

If the optional keyword **NORE** is specified, the links are taken in the order of their definition, and the matrix can be very large and ill-conditioned. If **RENU** (or nothing) is specified, then the connections are renumbered in an attempt to minimize the size of the matrix.

The optional keyword **VERI** can be used to verify a posteriori that the imposed links are effectively satisfied. This option produces heavy output and CPU overhead and should therefore be used only for debugging purposes.

The optional keyword **ARRA** can be used to choose storage of the links in a dynamic array rather than in a doubly linked list of doubly linked lists. This may increase efficiency (but is still under development).

The optional keyword **SPLT** can be used to choose the desired strategy for splitting the constraints into groups. The following possibilities are currently available:

- **DOF** requests dof-based splitting. This is the default and normally needs not being specified explicitly.
- **NODE** requests node-based splitting.
- **DOMA** requests subdomain-based splitting. Of course, this is only effective in calculations with domain decomposition.
- **PART** requests splitting based upon intrinsic node level factor. Of course, this is only effective in calculations with space partitioning.
- **NONE** requests no splitting. All constraints end up in a single, big group.

The optional keyword **SOL2** can be used to choose closed-form solution for groups of links containing just two links (in addition to groups containing just one link). By default, all groups of links containing more than one links are solved by the general numerical method (Choleski's method). In some cases, closed-form solution may be more efficient.

The optional keyword **UPDT** can be used to introduce an update frequency for time-varying links, in order to save CPU time. This is especially useful for fluid-structure interaction, when links are classically updated at each time-step, with a frequency obtained from the CFL condition, whereas their update should follow the physical structural velocity, often much smaller than sound speed in the different media.

As far as **GLIS** models are concerned, only 3D models (sliding surface) are currently available. For 2D models (sliding lines), please use decoupled or "liaison" links.

The **EDEF** directive is described in [10.25](#) (page D.189).

10.3 OPTIONS FOR "LIAISON" LINKS

Object:

The option "SOLV" makes it possible to change the resolution method in order to reduce the time spent to solve large matrix systems. For iterative solvers ("SPLI" keyword) the data structure uses the CSR format (Compressed Sparse Rows), well suited for iterative solution and used in the SPLIB library.

The option "RENUM" makes it possible to renumber the connections in order to minimize the size of the matrix.

The optional keyword "FREQ" can be used to avoid the inversion of the connection matrix at each computation step, in the cases where this is possible (see below).

The optional keyword "VERI" can be used to verify a posteriori that the imposed liaisons are effectively satisfied. This option produces heavy output and CPU overhead and should therefore be used only for debugging purposes.

Syntax:

```
< "SOLV"  |[ "CHOL"                                ;
              "SPLI" < "TYPE"  imet > < "PCON" ipre > ...
              ...   < "PITE"  prec > < "IPA1" ip1  > ...
              ...   < "IPA2"  ip2  > < "RPAR" rp   > ...
              ...   < "INIS"  inig >                                ]| >

< $[ "RENU" ; "NORE" ]$ >  < "FREQ"  ifreq > <"VERI">
```

imet

iterative solver number i, default 8

ipre

preconditioner number k, default 3

prec

tolerance on residual, default 1.D-6

ip1

first integer parameter, default 20

ip2

second integer parameter, default 10

rp

real parameter, default 1.D-4

inig

initial guess, default 0.0

ifreq

The matrix will be inverted each ifreq computation step; by default, ifreq is 1, i.e. the matrix is inverted at each step. This has only effect in Lagrangian computations. In ALE or Eulerian cases, the matrix is inverted anyway at each time step (like if ifreq=1) because the nodal masses are continuously changing due to transport.

Comments:

If the option "CHOL" is specified, the standard direct solver is used. This is the default option.

When the keyword "SPLI" is specified, the iterative solver is used. SPLIB is a library of iterative solvers with preconditioners for symmetric and nonsymmetric systems. It has been adapted for large matrix systems in the EUROPLEXUS code and uses the Compressed Sparse Row format (CSR).

<http://www.netlib.org/utk/papers/iterative-survey/node61.html>

The integer parameter **imet** specifies which SPLIB solver to use :

imet = 1 : Bi-Conjugate Gradients

imet = 2 : Conjugate Gradients with $AA'y = b$, $x = A'y$

imet = 3 : Conjugate Gradients with $A A' x = A'b$

imet = 4 : Conjugate Gradients Squared (CGS)

imet = 5 : Conjugate Gradients Stabilized

imet = 6 : GMRES(ip2)

imet = 7 : Transpose free QMR

imet = 8 : Template version of Conjugate Gradients Stabilized

imet = 9 : Template version of GMRES(ip2)

The integer parameter `ip2` used in GMRES solvers defines the Krylov subspace size (default value 10). When an `imet` of zero or less is specified, CG-Stabilization is used.

The integer parameter `ipre` specifies which preconditioner to use :

`ipre = 0` : no preconditioner

`ipre = 1` : ILU(`ip1`)

`ipre = 2` : MILU(`ip1`,`rp`)

`ipre = 3` : ILUT(`ip1`,`rp`)

`ipre = 4` : SSOR(`rp`)

`ipre = 5` : TRID(`ip1`)

`ipre = 6` : ILU0

`ipre = 7` : ECIMGS(`rp`)

The integer parameter `ip1` indicates the levels of fill-in to allow for ILU and MILU and the block size to use in the tridiagonal preconditioner TRID. For ILUT, `ip1` is the maximum additional entries allowed per row in the preconditioner compared to the original matrix. The real parameter `rp` is the relaxation parameter, the amount of multiply discarded fill-in entries before adding them to the diagonal. For SSOR it is the relaxation parameter. For ILUT, it is the drop tolerance.

For ECIMGS, `rp` specifies the sparsity pattern of the preconditioner :

- `rp = 0` : use the non zero pattern of the matrix
- $0 < rp < 1.$: use a sparser pattern than that of the matrix
- `rp = 1.` : use a diagonal pattern
- $rp > 1.$: use a denser pattern with `int(rp)` levels of fill-in

It is greatly recommended to use default values by entering only the following key words : "LIAIS" "SOLV" "SPLIB".

More informations to use SPLIB options can be found in this paper : "SPLIB : A library of iterative methods for sparse linear systems" by R. Bramley and X Wang, department of computer science - Indiana University, 1995.

For information about methods implemented, see for example the following reference by Y. Saad : "Iterative Methods for Sparse Linear Systems". This book can be found at <http://www-users.cs.umn.edu/~saad>

By default, connections are renumbered (option "RENU").

If the option "NORE" is specified, the connections are taken in the order of their definition, and the matrix can be very large and ill-conditioned. If "RENU" (or nothing) is specified, then the connections are renumbered in an attempt to minimize the size of the matrix.

If the connections are simple fixed displacements, a new numeration is useless because the matrix is diagonal.

The option `FREQ` is not compulsory. If it is not specified, a new computation is done at every time step.

When the coefficients of the relations between the degrees of freedom depend on the updated geometry (see `COQM` and `FS`), it is necessary to perform new computations and to invert the matrix at each time step during a `EUROPLEXUS` run. This operation is very costly if there are many coupled degrees of freedom. The keyword "FREQ" requests a new computation and an inversion only every `ifreq` computation steps.

In the case of an incompressible fluid or an A.L.E or Eulerian computation it is necessary to invert the matrix at each time step because the nodal masses are continuously changing due to transport. Therefore, the code ignores the user-supplied value for `ifreq` in these cases.

The same holds for an incompressible calculation, or for a calculation involving non-deformable sub-structures (keywords "NAVIER" and "SOLIDE").

10.4 AUXILIARY FILE

Object:

This directive allows to read the connections data from an auxiliary file.

Syntax:

```
< "FICHIER"      'nom.fic'  >
```

In certain cases the data may be bulky. It is then recommended to store them on an auxiliary file to shorten the main input data file. The auxiliary file is activated by means of the keyword "FICHIER" that precedes the file name (complete under Unix). In the main data file then only the keywords "LIAISON" "FICHIER" remain.

The auxiliary file (in free format) contains the whole set of connections data, except the keyword "LIAISON". To return to the main input data, the auxiliary file must be terminated by the keyword "RETOUR".

10.5 BLOCKAGES

Object:

To prescribe a zero displacement to (i.e., to block) a degree of freedom, that is to say to ensure the relation $U(i) = 0$.

Compatibility: COUP, DECO, LIAI

Syntax:

"BLOQ" (/LECDDL/ /LECTURE/)

/LECDDL/

Reading procedure of the degrees of freedom concerned.

/LECTURE/

Reading procedure of the numbers of the blocked nodes.

Comments:

Do not forget to dimension sufficiently: ("BLOQ" n) , see page A160.

It is possible to define the same blockage several times. Indeed, when a boundary is described, it is often simpler to use the implicit definition of the procedure /LECTURE/; in this case the points which are located at the ends are written twice. The EUROPLEXUS program eliminates these double definitions before it builds up the matrix.

Note, however, that the program is unable to eliminate the repeated points if e.g. several "BLOQ" keywords are used.

10.6 GEOMETRIC BILATERAL RESTRAINTS (“CONTACTS”)

Object:

The following instructions are used to automatically write relations imposed by boundary conditions of geometrical origin. For instance, the user wants certain nodes of an element to stay on a given structure, or to impose symmetry conditions for one part of the boundary.

Compatibility: COUP, LIAI

Syntax:

```
"CONT"
  | "PLAN"  ...      |
  | "SPHE"  ...      |
  | "CYLI"  ...      |
  | "CONE"  ...      |
  | "TORE"  ...      |
  | "SPLA"  ...      |
```

Comments:

Do not forget to dimension (see "RELA" n1 n2, page A80).

Here n1 represents the maximum number of nodes in contact and n2 is equal to 2 for 2-D computations and 3 for 3-D computations.

It is very important to note that the behaviour of these directives (except PLAN and SPLA) is different, according to the fact that the constraints coefficients are considered to be constant, or allowed to vary in time (the desired behaviour may be chosen via the OPTI CONT option, described in Section H). By default the constraint coefficients are determined on the initial configuration and are kept constant in time. This treatment is always adequate for the PLAN and SPLA types of constraint (since the normal to the plane does not vary in time anyway). However, for the other directives it is only adequate if the nodes do not move, i.e. for Eulerian nodes. In this case, the directives represent a handy shortcut for specifying constraints with coefficients different from point to point (but constant in time), without having to write such conditions explicitly in the input file.

But when the nodes move in time, i.e. for Lagrangian or ALE nodes, the use of constant coefficients in time is no longer adequate. The coefficients should be recomputed at each time step, which may be a costly operation. The user may require this updating of the coefficients by specifying the OPTI CONT VARI option, see Section H. Using variable coefficients has as effect that the nodes move by remaining on the imposed surface with first-order accuracy.

The instructions are described in detail on the following pages.

10.6.1 PLANE/LINEAR RESTRAINT (“CONTACT PLAN”)**Object:**

The specified nodes lay (and remain) on a plane normal to a given vector. In 2D, the plane reduces to a straight line. Only translational degrees of freedom are blocked.

Compatibility: COUP, LIAI

Syntax:

```
"PLAN"  | [  "NX" x  "NY" y  < "NZ" z > ;
            "NR" r  "NZ" z          ;
            "POIN" /LECT1/          ;
            "AUTO"                  ] | /LECT2/
```

x y

Components of the normal vector (2-D).

x y z

Components of the normal vector (3-D).

r z

Components of the normal vector (Axisymmetric).

POIN /LECT1/

Must specify a node belonging to the mesh, either via its index or via its CASTEM 2000 name. The coordinates of this node are taken as the components of the normal.

AUTO

The components of the normal are determined automatically from the position of the nodes listed in /LECT2/. Therefore, in this case, the nodes contained in the following /LECT2/ list must lie on the same line (in 2D) or plane (in 3D). In the 3D case, of course, the listed nodes must define a plane (not be along the same line).

/LECT2/

Numbers of the nodes concerned.

Comments:

It is not necessary that the normal vector be unitary, since it is automatically normalised by the program. Furthermore, it is not necessary that the nodes initially belong to the same line or plane (except in the AUTO case).

The difference between this directive and the CONT SPLA directive (see below) is that CONT PLAN blocks only translational degrees of freedom, while CONT SPLA blocks both translational and rotational degrees. Therefore, the two directives are identical for nodes of continuum elements, which do not possess rotational degrees of freedom. However, for structural nodes (with rotations), CONT PLAN represents a hinge while CONT SPLA represents a symmetry line or plane (the relevant rotations are automatically blocked in that case).

10.6.2 SPHERICAL/CIRCULAR RESTRAINT**Object:**

The specified nodes lay on (a) sphere(s) of given center. In 2D, the sphere reduces to a circle.

Compatibility: COUP, LIAI

Syntax:

```
"SPHE"  |[  "CX" x  "CY" y  < "CZ" z >  ;
           "CR" r  "CZ" z                ;
           "CENT" /LECT1/                ]|  /LECT2/
```

x y

Coordinates of the center of the sphere (2-D).

x y z

Coordinates of the center of the sphere (3-D).

r z

Coordinates of the center of the sphere (Axisymmetric).

"CENT" /LECT1/

Node at the center of the sphere. Points should be sufficiently far from the sphere center so as to define the radial direction with sufficient accuracy.

/LECT2/

Nodes located at the surface of the sphere.

Comments:

This constraint only ensures that, at each time step, the displacement increment of the specified nodes be tangent to the (current) sphere. For finite displacement increments, therefore, the nodes will only approximately remain on the initial spherical surface. It is not necessary that the nodes initially belong to the same sphere.

This directive blocks only translational degrees of freedom.

In case variable coefficients are specified (via the OPTI CONT VARI option), remember to dimension adequately by the DIME VCON directive). Each sphere/circle requires 3 coefficients.

10.6.3 CYLINDRICAL RESTRAINT

Object:

The specified nodes lay on (a) circular cylinder(s) of given axis. At each step the displacement increment along the axial direction is free, while that in the plane orthogonal to the axis is tangent to a circle.

The instruction only applies to a 3-D analysis. In 2D, the SPHE directive described in the previous Section may be used to obtain a circular restraint.

Compatibility: COUP, LIAI

Syntax:

```
"CYLI"  |[ "P1X" x1  "P1Y" y1  "P1Z" z1  ; "POI1" /LECT1/ ;  
           "P2X" x2  "P2Y" y2  "P2Z" z2  ; "POI2" /LECT2/ ]| /LECT3/
```

x1 y1 z1

Coordinates of a point of the cylinder axis.

"POI1" /LECT1/

Node at the first point of the cylinder axis.

x2 y2 z2

Coordinates of another point of the axis.

"POI2" /LECT2/

Node at the other point of the cylinder axis.

/LECT3/

Nodes concerned. Points should be sufficiently far from the cylinder axis so as to define the radial direction with sufficient accuracy.

Comments:

This constraint only ensures that, at each time step, the displacement increment of the specified nodes be tangent to the cylinder (current or initial, depending on OPTI CONT option).

For finite displacement increments, therefore, the nodes will only approximately remain on the initial cylindrical surface. It is not necessary that the nodes initially belong to the same cylinder.

This directive blocks only translational degrees of freedom.

In case variable coefficients are specified (via the OPTI CONT VARI option), remember to dimension adequately by the DIME VCON directive). Each cylinder requires 6 coefficients.

10.6.4 CONICAL RESTRAINT

Object:

The specified nodes lay on (a) cone(s) of given axis.

The instruction only applies to a 3-D analysis.

Compatibility: COUP, LIAI

Syntax:

```
"CONE"  $[ "SX" x1  "SY" y1  "SZ" z1 ; "APEX" /LECT1/ ]$
          $[ "PX" x2  "PY" y2  "PZ" z2 ; "POIN" /LECT2/ ]$ /LECT3/
```

x1 y1 z1

Coordinates of the apex of the cone.

"APEX" /LECT1/

Node at the cone apex.

x2 y2 z2

Coordinates of a point on the cone axis different from the apex.

"POIN" /LECT2/

Node along the cone axis different from the apex.

/LECT3/

Nodes concerned. Points should be sufficiently far from the cone axis so as to define the radial direction with sufficient accuracy.

Comments:

This constraint only ensures that, at each time step, the displacement increment of the specified nodes be tangent to the cone (current or initial, depending on OPTI CONT option). For finite displacement increments, therefore, the nodes will only approximately remain on the initial conical surface. It is not necessary that the nodes initially belong to the same cone.

This directive blocks only translational degrees of freedom.

In case variable coefficients are specified (via the OPTI CONT VARI option), remember to dimension adequately by the DIME VCON directive). Each cone requires 6 coefficients.

10.6.5 TOROIDAL RESTRAINT

Object:

The specified nodes lay on (a) torus(es) of given axis and center.

This instruction only applies to a 3-D analysis.

Compatibility: COUP, LIAI

Syntax:

```
"TORE"  | [ "P1X" x1  "P1Y" y1  "P1Z" z1 ; "POI1" /LECT1/ ;
            "P2X" x2  "P2Y" y2  "P2Z" z2 ; "POI2" /LECT2/ ;
            "P3X" x3  "P3Y" y3  "P3Z" z3 ; "CENT" /LECT3/ ] | /LECT4/
```

x1 y1 z1

Coordinates of a point on the torus (circular) axis.

"POI1" /LECT1/

First node on the torus (circular) axis.

x2 y2 z2

Coordinates of another point on the circular axis.

"POI2" /LECT2/

Second node on the torus (circular) axis.

x3 y3 z3

Coordinates of the center of the torus.

"CENT" /LECT3/

Node at the torus center.

/LECT4/

Nodes concerned.

Comments:

This constraint only ensures that, at each time step, the displacement increment of the specified nodes be tangent to the torus (current or initial, depending on OPTI CONT option). For finite displacement increments, therefore, the nodes will only approximately remain on the initial torical surface. It is not necessary that the nodes initially belong to the same torus.

This directive blocks only translational degrees of freedom.

In case variable coefficients are specified (via the OPTI CONT VARI option), remember to dimension adequately by the DIME VCON directive). Each torus requires 9 coefficients.

10.6.6 PLANE/LINE OF SYMMETRY RESTRAINT

Object:

The specified nodes lay (and remain) on (a) plane(s) of given normal vector, that defines the symmetry. In 2D, the plane reduces to a straight line.

Compatibility: COUP, LIAI

Syntax:

```
"SPLA"  |[  "NX" x  "NY" y  < "NZ" z > ;  
           "NR" r  "NZ" z           ;  
           "POIN" /LECT1/           ;  
           "AUTO"                   ]| /LECT2/
```

x y

Components of the normal vector (2-D).

x y z

Components of the normal vector (3-D).

r z

Components of the normal vector (Axisymmetric).

POIN /LECT1/

Must specify a node belonging to the mesh, either via its index or via its CASTEM 2000 name. The coordinates of this node are taken as the components of the normal.

AUTO

The components of the normal are determined automatically from the position of the nodes listed in /LECT2/. Therefore, in this case, the nodes contained in the following /LECT2/ list must lie on the same line (in 2D) or plane (in 3D). In the 3D case, of course, the listed nodes must define a plane (not be along the same line).

/LECT2/

Numbers of the nodes concerned.

Comments:

It is not necessary that the nodes initially belong to the same plane (except in the AUTO case).

The difference between this directive and the CONT PLAN directive (see above) is that CONT PLAN blocks only translational degrees of freedom, while CONT SPLA blocks both translational and rotational degrees. Therefore, the two directives are identical for nodes of continuum elements, which do not possess rotational degrees of freedom. However, for structural nodes (with rotations), CONT PLAN represents a hinge while CONT SPLA represents a symmetry line or plane (the relevant rotations are automatically blocked in that case).

Remember to dimension adequately with 'SYME' (see page A.80).

When AUTO is used, the search for enough non-coincident nodes, among those contained in LECT2, so as to define a line in 2D or a plane in 3D is affected by a tolerance. In case of necessity, this tolerance may be set by OPTI TOLC, see page H.40.

10.7 IMPOSED CIRCULAR SHAPE

Object:

The displacements of the specified nodes are constrained to be in the radial direction with respect to a point (center) and to have the same modulus. If the nodes lie initially on the same circle, they remain on a circle, whose radius may vary with time.

The instruction is available only for a 2-D analysis.

For an Eulerian computation (no mesh displacements), the fluid velocities are radial and of the same modulus.

Compatibility: COUP, LIAI

Syntax:

```
"RADI" "SPHE" "CENT" /LECTURE/  
...    "CONT" /LECTURE/
```

```
"CENT" /LECTURE/
```

Number of the node at the center of the circle.

```
"CONT" /LECTURE/
```

Numbers of the nodes concerned.

Comments:

The instruction is used to avoid instabilities e.g. when a gas bubble collapses after an initial expansion.

For n points ($n > \text{or} = 2$), EUROPLEXUS writes $2n-1$ relations.

Do not forget to dimension "RELA" $n1$ $n2$ (page A80) when using "LIAI" keyword. Here, $n1=2n-1$ and $n2=2$.

10.8 RELATIONS

Object:

Several displacement (or velocity) components are linked by constant coefficients during the whole computation.

Compatibility: COUP, LIAI

Syntax:

```
"RELA"  ngroup*(  
    ... nrel  nterm*( coef  icomp $[ nuneu ipas ;  
                                /LECTURE/ <SHIF s> ]$ )  
    ... "EGAL" /LECDDL/ /LECTURE/ )
```

ngroup

Number of relation sets.

nrel

Number of relations to be generated in a set.

nterm

Number of terms in a relation of the set.

coef

Coefficient of a term.

icomp

Displacement component of the node "nuneu" involved in the relation.

nuneu

Number of the node concerned.

ipas

Increment on the number of the node "nuneu" in order to get the next relation of the set.

LECTURE

List of concerned nodes.

SHIFT **s**

Force circular permutation of the list (see example below). The increment in traversing the list circularly is indicated by the **s** quantity (normally 1).

/LECDDL/

Reading procedure of the numbers of the blocked nodes.

EGAL

Indicates the equality along the component /LECDDL/ of the motion of the nodes defined by the following /LECT/.

Comments:

Each displacement will be specified by the number of the node (**nuneu**) and its component (**icom**). Formula of the relation:

$$0 = \text{coef}(1)*U(1) + \text{coef}(2)*U(2) + \dots + \text{coef}(k)*U(k)$$

There are two ways to define a set of relations. The first is to give the node number **nuneu** and the step **ipas**. The second is to use the procedure /LECTURE/, which allows to use object names created by GIBI. In this latter case, one passes from one relation to the next one in a set by taking the next node in the procedure /LECTURE/ associated with each term. In this case, there must be exactly **nrel** nodes in each one of the lists specified via the /LECT/ procedures (assuming that the optional SHIF keyword has not been specified).

The optional SHIF keyword can be used to force a circular permutation of the list. In this case, the number of nodes in the lists need not be the same. For example, assume that one wants to impose the same displacement along *z* (i.e. global direction 3) to all nodes of an object named "face1". Then the command would be:

```
RELA 1 0 2
      1. 3 LECT face1 TERM
      -1 3 LECT face1 TERM SHIFT 1
```

Note that in this case the number of relations **nrel** can be set to 0 because the code computes it automatically.

Example:

RELA	5	2	2	1.	1	288	1	-1.	1	6	1	
		1	3	3.4	2	287	0	-1.	2	5	0	1. 1 5 0
		3	2	0.5	3	115	7	-1.	3	9	5	
		5	2	1.	3	LECT	toto	TERM				
				-1.	3	LECT	tata	TERM				
EGAL					13	LECT	228 321 842	TERM				

There are five groups. The first group has 2 relations of 2 terms, the second 1 relation of 3 terms, the third 3 relations of 2 terms, the fourth 5 relations of 2 terms and the last one 2 relations of 2 terms.

In the first group, d.o.f 1 of node 288 has been linked to d.o.f 1 of node 6 (first relation), then d.o.f 1 of node 289 to d.o.f 1 of node 7 (second relation). In fact $ipas=1$ for the two terms.

In the second group, d.o.f. 2 of node 287 has been linked to d.o.f. 2 of node 5 and to d.o.f. 1 of the same node 5. There is just one relation since $ipas = 0$ for the three terms.

On the contrary, in the third group, $ipas=7$ for the first term and 5 for the second. Therefore, d.o.f 3 of node 115 has to be linked to d.o.f 3 of node 9 (first relation of the group), then d.o.f 3 of node 122 to d.o.f 3 of node 14 (2nd relation), and finally d.o.f 3 of node 129 to d.o.f 3 to node 19 (3rd relation).

In the fourth group, there are 5 relations between the d.o.f. 3 of the nodes belonging to objects 'toto' and 'tata' taken in the order in which they appear.

In the fifth group, there are 4 equalities between the d.o.f. 1 and 3 of nodes 228, 321 and 842.

$$\begin{aligned}
 U_x(228) &= U_x(321) & ; & & U_z(228) &= U_z(321) \\
 U_x(321) &= U_x(842) & ; & & U_z(321) &= U_z(842)
 \end{aligned}$$

10.9 ARMATURES

Object:

In calculations of structures made of reinforced concrete, this directive allows to link the displacements of the nodes belonging to continuum-like elements made of concrete, with those of bar-like elements made of steel.

Decoupled treatment of this link consists in introducing a penalty spring between the reference position of a steel node in the corresponding concrete element and its actual position.

The default spring's stiffness is obtained from concrete element through the formula:

$$k = GL$$

with:

G : bulk modulus of concrete element's material,

L : radius of a sphere whose volume equals concrete element's volume.

Compatibility: COUP, DECO, LIAI

Syntax:

```
"ARMA"          < "TSTA" ista> < "CSTI" cstif >
                  "BETO" /LECTURE/
                  "FERR" /LECTURE/
```

ista

DECO only: flag for stability control over the penalty spring's stability (default: 1, see comment below)..

cstif

DECO only: coefficient multiplying default stiffness of the penalty spring (default: 1.0).

BETO

Introduces the list of the concrete (continuum-like) elements.

FERR

Introduces the list of the steel or reinforcement (bar-like) elements.

Comments:

The concrete elements must be of continuum-like type 2D or 3D. The steel or reinforcement elements must be of bar-like type (e.g. BR3D in 3D or BARR in 2D).

If `ista` equals 2, stiffness of the penalty spring is limited so that the associated stability time-step is not smaller than the reference concrete element's one.

10.10 CROSSING

Object:

This directive allows the user to automatically interconnect crossing longitudinal and transverse reinforcement bars (rebars) to constitute rebar cages (carcasses) frequently used to reinforce the concrete structures. In the real life, the rebars in the cages are usually connected either by welding, tying steel wire, or with mechanical connections.

Links are created between the elements of longitudinal reinforcement and the nodes of transverse reinforcing steel (stirrups).

Both coupled and decoupled links are implemented: the coupled links are treated using Lagrange multipliers method whereas the decoupled ones are solved by the penalty method.

Only the translation degrees of freedom are concerned by this link.

Nodes of the transverse rebars may eventually coincide with the nodes of the longitudinal rebars but should remain distinct.

Compatibility: COUP, DECO

Syntax:

```
"CROS" < "TSTA" ista> < "CSTI" cstif >  
      "LONG" /LECTURE/  
      "TRAN" /LECTURE/
```

ista

DECO only: flag for stability control over the penalty spring's stability (default: 1, see comment below)..

cstif

DECO only: coefficient multiplying default stiffness of the penalty spring (default: 1.0).

LONG

Introduces the list of longitudinal rebar elements.

TRAN

Introduces the list of transverse rebar elements.

Comments:

Both the longitudinal and transverse rebars must be modelled as POUT elements.

If **ista** equals 2, stiffness of the penalty spring is limited so that the associated stability time-step is not smaller than the reference longitudinal steel element's one.

10.11 ACBE: REBAR(FEM)-CONCRETE(DEM) LINK

Object:

This directive allows creating nonlinear links between a steel reinforcement bar (rebar) modelled as FEM beam and plain concrete modelled by the discrete element method (DEM). Only one link between a given discrete element and a finite element beam may be created. Each link contains a normal and a tangential component.

A decoupled (DECO) link model is implemented only.

Syntax:

```
"ACBE" < "TSTA" ista> < "CSTI" cstif >
      "BETO" "COEF" c1 /LECTURE/
      "ARMA" "COEF" c2 /LECTURE/
      "YOUN"  youn  "TN"  tn  "CN"  cn  "ADUN" adun
      "FTAN"  "NUMF" nf
```

ista

Flag for stability control over the penalty link's stability (default: 1, see comment below).

cstif

Coefficient multiplying default stiffness of the link (default: 1.).

BETO

Introduces the list of discrete elements concerned.

c1

Coefficient defining the interaction range for the discrete elements.

ARMA

Introduces the list of rebar (POUT type) elements concerned.

c2

Coefficient defining the interaction range for the beam elements.

youn

Young's modulus used to calculate the normal stiffness of the link ($kn=youn*S/L$).

tn

Maximum normal tensile strength (perpendicular to the rebar direction)

cn

Maximum normal compression strength (perpendicular to the rebar direction)

adun

Softening coefficient (ratio between elastic and softening slopes >0)

nf

Number of the function describing the tangential behaviour of the link.

Comments:

If **ista** equals 2, stiffness of the penalty spring is limited so that the associated stability time-step is not smaller than the reference concrete element's one.

10.12 IMPOSED MOTIONS

Object:

These instructions define imposed motions (displacements, velocities or accelerations) depending on time, for different degrees of freedom.

Compatibility: COUP, LIAI

Syntax:

```
"DEPL" ( /LECDDL/ coef "FONC" ifonc /LECTURE/ )  
"VITE" ( /LECDDL/ coef "FONC" ifonc /LECTURE/ )  
"ACCE" ( /LECDDL/ coef "FONC" ifonc /LECTURE/ )
```

/LECDDL/

Reading procedure of the different d.o.fs concerned.

coef

Multiplying factor of the values of the function.

ifonc

Number of the function to be used.

/LECTURE/

Reading procedure of the numbers of the nodes concerned.

Comments:

The function to be used will be defined by means of the principal instruction "FONC" which enables the user to choose a tabulated function (linear interpolation between the points), or a function programmed by the user by means of a subroutine.

At a time t , the imposed motion is : $\text{coef} \cdot F(t)$. In this case, only one function is to be defined, if the motions vary only in amplitude.

If the same d.o.f is submitted to several motions, EUROPLEXUS only takes into account the motion which has been defined first.

10.13 CONNECTIONS BETWEEN SHELLS AND SOLID ELEMENTS

Object:

The purpose is to link together the degrees of freedom at the boundary of two parts of the structure. One part is meshed with shells, the other with solid elements. This link is available only for two-dimensional computations (plane or axisymmetric).

Compatibility: COUP, LIAI

Syntax:

```
"COQM"  ngroup*(  nco  nma  /LECTURE/  )
```

ngroup

Number of groups of shell and solid element connections.

nco

Number of the shell node linked to a solid element.

nma

Number of the solid node linked to a shell.

/LECTURE/

Reading procedure to input the other solid element nodes which are connected (no compulsory order).

Comments:

```

      M1  x-----
            I
(shell)   I
-----x  x  M0 (nma)      R(i) : distance M0-Mi
      /   I              D(i) : normal displacement
      nco  x  Mi              of Mi
            I      (solid element)
      Mp   x-----
```

A "shell-solid element" relation is represented by the following $p+2$ equations.

2 equations for the displacements:

$$U(1,nco) = U(1,nma)$$

$$U(2,nco) = U(2,nma)$$

p equations for the rotations of p other solid element nodes:

$$U(3,nco) = R(i) * D(i)$$

10.14 INTERFACES

Object:

This directive allows to define an interface between two lines or two surfaces. Link relations are created so that the velocity field is continuous through the interface. In the case of non-matching meshes on both sides of the interface, continuity is imposed in a weak sense.

This directive is very similar to the INTERFACE directive used in the sub-domain calculation framework.

Compatibility: COUP

Syntax:

```
"INTERFACE"    |[ "COMP"      ;
                  "MORTAR"    ;
                  "OPTIMAL" ]| <"TOLE" tole> ...

... |[ "SIDE"      ;
      "SCOARSE"   ;
      "SFINE"     ]| /LECTURE/ ...

... |[ "SIDE"      ;
      "SCOARSE"   ;
      "SFINE"     ]| /LECTURE/
```

"COMP"

Keyword declaring an interface with matching meshes.

"MORTAR"

Keyword declaring an interface with non-matching meshes, treated by the mortar method (see comment below).

"OPTIMAL"

Keyword declaring an interface with non-matching meshes, treated by the optimal method.

tole

Tolerance given to find matching nodes (default=1.E-3).

"SIDE"

Keyword introducing the support of both sides of the interface for **COMP** and **OPTI** cases (see note below).

"SCOARSE"

Keyword introducing the support of the side of the interface with coarse mesh in the **MORTAR** case (see note below).

"SFINE"

Keyword introducing the support of the side of the interface with fine mesh in the **MORTAR** case (see note below).

Comments:

There **MUST NOT** be any coincident nodes between the two sides of an interface.

When using the mortar method, the side of the interface whose mesh is used to discretize Lagrange multipliers has to be specified. It is the mesh introduced by the **SFINE** keyword, the other mesh being introduced by the **SCOARSE** keyword.

When using interfaces with non-matching meshes, so-called **CLxx** elements (see pages INT.90 and INT.100) have to be affected to meshes of both sides of the interface. These elements **must** be given the “phantom” material (**MATE FANT**) with density equal to zero.

The treatment of non-matching meshes with 3D solid elements is restricted to hierarchical meshes. In this case, the mortar method and the optimal method are identical, and a mortar interface has to be declared.

The **mortar** method may be used with any element types in 2D, but only with shell element types in 3D. When using the **mortar** method with linear interfaces (2-noded element sides), there must be at least one geometrical point that has the same coordinates, within the tolerance **tole** defined above, in the two facing meshes. This is necessary because the interface model uses the point’s coordinates internally in order to define a reference frame on the interface.

10.15 FLUID-STRUCTURE COUPLING

Object:

This directive allows to specify the coupling between a fluid and a structure modelled by topologically independent meshes.

Compatibility: COUP

Syntax:

```
FLST <SLID> STRU /LECTS/ FLUI /LECTF/  
      <DGRI> $[ HGRI hgri ; NMAX nmax ; DELE dele ]$
```

/LECTS/

List of structural **nodes** concerned. They must be declared as Lagrangian.

/LECTF/

List of fluid **elements** concerned.

DGRI

Dump out the initial grid of cells used for fast searching on the listing (only at step 0).

HGRI

Specifies the size of the grid cell. Each cell has the same size in all spatial directions and is aligned with the global axes.

NMAX

Specifies the maximum number of cells along one of the global axes.

DELE

Specifies the size of the grid cell as a multiple of the diameter of the largest coupled fluid element. Element “diameters” are computed only along each global spatial direction and the maximum is taken. For example, by setting **DELE 4** the size of the cell is four times the diameter of the largest coupled fluid element. By default, i.e. if neither **HGRI**, nor **NMAX**, nor **DELE** are specified, the code takes **DELE 3**.

10.16 FLUID-STRUCTURE COUPLING 2 (FLSR)

Object:

This directive allows to specify the coupling between a fluid and a structure modelled by topologically independent meshes.

The fluid mesh may be either fully general (unstructured) or regular (structured), as specified by the **STFL** directive described on page C.68. In the latter case, the search operations are faster.

Compatibility: **COUP**, **DECO**.

Syntax:

```
FLSR      STRU /LECTS/
          |[ FLUI /LECTF/ ; STFL ]|
          $[ R r          ; GAMM gamm ; PHIS phis ]$
          $[ HGRI hgri ; NMAX nmax ; DELE dele ]$
          <DGRI>
          <BFLU bflu> <FSCP fscp>
          <ADAP LMAX lmax>
```

STRU

Introduces the structure mesh coupled with the fluid. The concerned elements are specified next.

/LECTS/

List of structural **elements** concerned. All their nodes must be declared as Lagrangian.

FLUI

The fluid mesh coupled with the structure is fully general (unstructured). The concerned elements are specified next.

/LECTF/

List of fluid **elements** concerned. The fluid mesh is unstructured.

STFL

The fluid mesh coupled with the structure is regular (structured). The concerned **elements** need not be specified. In fact, they are simply the elements generated by the **STFL** directive described on page C.68, which must in this case have been specified previously in the input file.

R

Prescribed (fixed) radius R of influence spheres at each coupled structural node. In the special, but frequent, case of a uniform structured fluid mesh (uniform square or cube elements) it is suggested to take R slightly larger than the semi-diagonal of a fluid element. This means that, for a 2D uniform square fluid mesh of side L_Φ one should take $R = 0.71L_\Phi$ while for a 3D uniform cube fluid mesh of side L_Φ one should take $R = 0.87L_\Phi$.

GAMM

Coefficient γ for the automatic determination of influence spheres at each coupled structural node, based on the size of the enclosing fluid element. The sphere radius is $R = \gamma R_F = \gamma \delta L_\Phi$ where L_Φ is the local length (size) of the fluid mesh, δ is a coefficient related to the space dimension d of the problem ($\delta = \frac{\sqrt{d}}{2}$, i.e. about 0.71 in 2D and about 0.87 in 3D calculations). The quantity indicated as R_F above is the “natural” size of the sphere radius, i.e. the radius of a sphere (circle in 2D) which exactly encompasses all nodes of a regular element (regular cube in 3D or regular quadrilateral in 2D). By default it is $\gamma = 1.01$. This value should ensure “tightness” of the structure, at least for a regular mesh. By increasing the value, tightness is safer but the amount of fluid “attached” to the structure also increases. By decreasing the value, some local spurious passage of fluid across a solid structure might occur.

PHIS

Coefficient ϕ_s for the automatic determination of influence spheres at each coupled structural node. The sphere radius is equal to ϕ_s times the minimum structural element length at the concerned node. By default it is $\phi_s = 0.3$. This option should be rarely used. It is advisable to use **GAMM** instead.

HGRI

Specifies the size of the grid cell. Each cell has the same size in all spatial directions and is aligned with the global axes. Note that the size of this grid is related to the size of **structural** elements, not of fluid elements.

NMAX

Specifies the maximum number of cells along one of the global axes.

DELE

Specifies the size of the fast search grid cell as a multiple of the length of the largest coupled **structural** element. Element “diameters” are computed only along each global spatial direction and the maximum is taken. For example, by setting **DELE** 2 the size of the cell is two times the length of the largest coupled structural element. By default, i.e. if neither **HGRI**, nor **NMAX**, nor **DELE** are specified, the code takes **DELE** 3 (this value is probably too large, a value of 1.1 or so should be more appropriate in most cases).

DGRI

Dump out the initial grid of cells used for fast searching on the listing (only at step 0).

BFLU

Type of treatment of numerical fluxes (density and energy, but **not** momentum) in fluid models, when used in conjunction with the present **FLSR** directive. The value 0 (default) indicates that fluxes are freely computed. The value 1 indicates that fluxes are blocked between two fluid nodes (or points) which are both within the influence domain of the structure. The value 2 indicates that fluxes are blocked between two fluid nodes (or points) of which at least one lies within the influence domain of the structure. Note that, for the moment, this directive only applies to fluids modelled by multicomponent node-centered Finite Volumes (NCFV, i.e. MCxx ‘elements’). Thus, fluxes are computed (or blocked) between fluid nodes.

FSCP

Type of coupling by Lagrange multipliers between fluid nodes and corresponding structural points, when used in conjunction with the present **FLSR** directive. The value 0 (default) indicates that coupling occurs only in the direction normal to the structure. The value 1 indicates that coupling occurs along all spatial directions.

ADAP

Activates mesh adaptivity for automatic refinement and un-refinement of the fluid mesh specified by **/LECTF/** in the vicinity of the structure specified by **/LECTS/**. Note that this type of mesh adaptivity is at the moment incompatible with other types of adaptivity such as those activated by the **WAVE** or **INDI** directives.

lmax

Maximum adaptive refinement level L^{\max} of the fluid mesh. If the prescribed sphere radius is r so that the sphere diameter is constant and equal to $d = 2r$, and the fluid mesh is regular with the (base) element diagonal equal to D_1 , then it is $L^{\max} = 1 + \lceil \log_2 \frac{D_1}{d} \rceil$, where $\lceil a \rceil$ is the ceiling operator, indicating the smallest integer number greater than or equal to the argument a .

Remarks:

If neither **R** nor **GAMM** nor **PHIS** are specified, the code performs an automatic determination of influence spheres at each coupled structural node by using the default value of **GAMM** ($\gamma = 1.01$).

In case of automatic determination of influence spheres based on the **GAMM** keyword in conjunction with an **unstructured** fluid grid, a fast search over the coupled fluid elements is needed in addition to the normal fast search over the coupled structural elements. Scope of this search is to determine, for each structural node, which is the fluid element currently containing the node. For this purpose, the code uses a fast search algorithm by means of the same parameters (**DGRI**, **HGRI**, **NMAX**, **DELE**) specified above for the search over structural elements. Note, however, that as concerns this second search if **DELE** is specified it refers to the size of the fluid element rather than to the size of the structural element. However, if a **structured** fluid grid is specified, then no additional search is needed.

References

The FLSR model was first described in report [250]. A short description of the model is also given in reference [244].

10.17 FLUID-STRUCTURE INTERACTIONS

Object:

This is aimed at linking together the degrees of freedom at the boundary of 2 parts of the structure:

- one part meshed with shells or solid elements;
- one solid element part meshed with a fluid material.

This possibility exists for two- and three-dimensional computations.

This connection may be expressed in two ways:

- By using specific fluid-structure elements FS2D or FS3D: directive "FS";

Without using fluid-structure elements: directive "FSA".

Compatibility: COUP, LIAI

Comments:

The first directive is available for a Lagrangian or an ALE calculation. Instead, the second is only valid for ALE problems.

The elements FS2D and FS3D behave like incompressible fluids. In order to avoid spurious effects (related to the flow along the boundary), the thickness of this boundary zone must be as small as possible, and possibly 0.

The "FS" directive is described in the next page, while for the "FSA" directive please consult page D.260.

10.17.1 FLUID-STRUCTURE CONNECTION (FS)**Object:**

The contact between the fluid and the structure is modelled by elements FS2D and FS3D. This directive is available for Lagrangian and ALE calculations.

Compatibility: COUP, LIAI

Syntax:

```
"FS"  /LECTURE/
```

```
/LECTURE/
```

Numbers of the FS2D, FS3D or FS3T elements composing the boundary.

Comments:

The FS2D, FS3D and FS3T elements are in fact incompressible fluids. In order to avoid any parasitic effects (due to a potential flow along the boundary), the thickness of that boundary zone has to be as small as possible and even equal to zero.

It is strongly advised to use the new directives "FSA" and "FSR".

10.18 UNILATERAL RESTRAINT [OBSOLETE]

Foreword:

This directive is now obsolete, use the IMPACT directive described below (page D.170) which allows to compute at the same time the shock parameters (impulse, reaction, ...).

10.19 IMPACT

Object:

As for unilateral restraints, certain nodes of the structure must remain in the same half-space. However, the boundary is linked to the position of a material point and can be mobile. Impacts are possible in 2D or 3D.

The method of Lagrange multipliers may be activated by adding the keyword "LAGC" in the problem type (see page A.30). This method allows to couple the calculation of contact forces with the permanent connections (relations, boundary conditions, ...). It also allows to take into account the form of a projectile 'nose' in the case of a non-deformable projectile.

Compatibility: COUP, DECO, LIAI

Syntax:

```
"IMPACT" "DDL" iddl "COTE" alpha      ...

      ... < "NEZ" |[ "HEMI" "RAYO" rayon1      ;
              "PLAT" "LARG" larg1 < "LONG" long1 > ;
              "CONE" "LARG" larg2 < "ANGL" beta > ;
              "CYLI" "RAYO" rayon2 ]| >

      ... <"FROT" "MUST" must "MUDY" muddy "GAMM" gamm >

      ... "PROJ" /LECTURE/ "CIBL" /LECTURE/
```

iddl

Component concerned. Indicates the first direction.

alpha

Enables one to choose between the 2 half-spaces separated by the plane of equation $x = x_0$. It must be an integer. Typically, one uses either +1 or -1.

rayon1

Radius of a projectile with hemispherical nose.

larg1

Width of a projectile with rectangular flat nose. It is in the second direction obtained by circular permutation of the Euclidean frame.

long1

Only in 3D, length of the rectangular projectile nose. It is in the third direction (obtained by circular permutation).

larg2

Width of a projectile with a conical nose.

beta

Half-angle of the cone (in degrees).

ray2

Radius of the projectile with cylindrical nose.

FR0T

Introduces the (optional) declaration of friction characteristics.

must

Static friction coefficient μ_s : ($0 < \mu_s < 1$).

mudy

High-velocity (dynamic) friction coefficient μ_d : ($0 < \mu_d < \mu_s < 1$).

gamm

Coefficient γ of the friction law. This law is similar to the one used for sliding lines and sliding surfaces (see page C.98). The friction coefficient μ varies from μ_s to μ_d as the relative tangential velocity V_r of the two bodies increases. The passage is governed by the exponential decay law: $\mu = \mu_d + (\mu_s - \mu_d)e^{(-\gamma V_r)}$.

PROJ

Introduces the number of the material point located on the boundary (if relevant, it is the tip of the projectile “nose”).

CIBL

Introduces the numbers of the nodes which are submitted to the impact.

Comments:

The boundary plane is perpendicular to one of the axes of the general coordinate system. This axis is defined by the component `idd1`, just as for unilateral contacts.

The half-space admissible for a point M (of the target) of coordinate x is such that, if x_0 is the abscissa of the material point:

$$\alpha(x - x_0) \geq 0$$

These impacts are available in 2-D or 3-D.

It is suggested to displace the projectile in such a way that the impact occurs after at least one time step.

Do not forget to dimension the keywords "IMPA" and "PSIM" correctly, see (page A.80).

The "NEZ" directive is available **only** with the LAGC option. When it is present, only those "CIBLE" nodes which are in contact with the geometric boundary thus defined, will be considered.

Without the option "LAGC":

It should be noted that the nodes which undergo shocks may not be connected by other imposed relations (LIAISONS).

The shock between the material point (projectile) and the point(s) of the target is treated elastically. The energy and impulse will therefore be conserved during the impact. This requires a modification of the time step so that the impact instant coincides with the beginning of a time step.

This effect introduces a small error in the work of forces during the impact, of the order: $dW = F * v * dt$. It is therefore advisable to shorten the time step in order to obtain better energy conservation.

These recommendation are irrelevant with the option "LAGC".

10.20 GAPS (“JEUX”)**Object :**

This is an impact between the (uncoupled) nodes along the direction defined by the user. This directive is available in 2D and 3D. In 3D, the gap must be defined also along a direction normal to the first one.

Compatibility: LIAI

Syntax:

In 2D:

```
"JEUX"      "AXE1" a1x a1y      "JEU1"  jeu1
...         "NOE1" /LECTURE/
...         "NOE2" /LECTURE/
```

In 3D:

```
"JEUX"      "AXE1" a1x a1y a1z  "AXE2" a2x a2y a2z
...         "JEU1"  jeu1      "JEU2" jeu2a jeu2b
...         "NOE1" /LECTURE/
...         "NOE2" /LECTURE/
```

a1x,a1y (,a1z)

Components of the first vector of the local reference.

a2x,a2y,a2z

Components of the second vector of the local reference (3D only).

jeu1

Gap along the direction of the first vector.

jeu2a,jeu2b

Gap along the direction of the second vector (3D only).

NOE1

Announces the first group of nodes.

NOE2

Announces the second group of nodes.

Comments:

To each node P1 belonging to the first group, is associated one node P2 of the second group, and reciprocally. In 3D, in the local frame of origin P1, defined by vectors AXE1 and AXE2, the impact occurs when: 1) the abscissa of point P2 is less than jeu1; 2) and the ordinate of point P2 lies between jeu2a and jeu2b.

In 2D, in the local reference of origin P1 of which AXE1 is the first axis, the impact occurs when the abscissa of point P2 is less than jeu1.

The direction defined by vector(s) AXE1 (or AXE2) does not change during the calculation.

Do not forget to dimension, by keyword "NBJEUX" (see page A.80).

10.21 SLIDING LINES AND SLIDING SURFACES

Object:

This directive defines one or more couple(s) of mutually sliding lines (2D) or sliding surfaces (3D). In 3D the “master” and “slave” objects may be composed of continuum elements or shells. In 2D they are composed by an ordered series of nodes.

In 3D, an auto-contact model is available. An auto-contacting surface is both master and slave at the same time.

Compatibility: COUP, DECO, LIAI

LIAI only: the method of Lagrange multipliers may be activated by adding the keyword **LAGC** in the problem type (see page A.30, Section 6.4). This method allows to couple the computation of contact forces with the permanent connections (relations, boundary conditions, ...).

Penalty method is only available in 3D with **DECO** keyword. If not activated, the same uncoupled algorithm is used as with **LIAI** and **LAGC** deactivated (see above and comments below).

Syntax:

```
"GLIS"  nglis * (
    < "PENA" > < "PFSI" rfac > < "PGAP" rgap >
    < "SELF" >
    | [
        | [ "MAIT"                /LECTURE/                ;
          "CMAI" /LECTURE/ | [ "EXTE" /LECTURE/ ;
                                "INTE" /LECTURE/ ] | ] |
        | [ "ESCL" ; "CESC" ; "PESC" ] | /LECTURE/ ;

        "AUTO" FACE iface /LECTURE/
    ] |
    < "COPT" 1 /LECTURE/ > )
```

nglis

Number of couples of lines (or surfaces) (MAIT, ESCL) or AUTO.

PENA

DECO only: toggles the use of penalty method to compute contact forces (see comment below).

rfac

DECO only: scale factor for the automatically computed contact stiffness (see comment below).

rgap

Value of the gap between master surface and slave nodes (3D only). Default value is zero in the case of solid master elements and half of shell's thickness in the case of shell master elements.

SELF

Toggles self-contact treatment for shells (3D only, see comment below).

MAIT /LECTURE/

Numbers of the master nodes (in 2D) or numbers of the continuum elements (in 3D).

MAIT NODE /LECTURE/

In 3D continuum elements: numbers of the master nodes belonging to the sliding surface.

CMAI /LECTURE/

Numbers of the master elements of the structure meshed by shell elements ("COQUES") (in 3D).

EXTE /LECTURE/

Number of the node defining the half-space external to the solid.

INTE /LECTURE/

Number of the node defining the half-space internal to the solid.

ESCL /LECTURE/

Numbers of the slave nodes (in 2D) or numbers of the slave elements (in 3D).

CESC /LECTURE/

Numbers of the slave elements (in 3D) if the structure is meshed by shell elements (COQUES).

PESC /LECTURE/

3D only. Numbers of the slave nodes.

iface

Number (local) of the face of the elements submitted to auto-contact.

COPT 1

If COPT 1 is activated, the thickness of the slave surface is taken into account.

Comments:

Sliding lines (2D):

The order of the numbers of the nodes determines the orientation of the contour and defines in that way the inner side of the two domains after a rotation of +90 degrees.

The *slave* nodes must be located just at or above the boundary of the region defined by the line of the *master* nodes.

Without the LAGC option, it is preferable that the two lines have similar mesh densities. But, if the *master* domain presents a high convexity, it is better to have *master* segments which are a bit longer than the *slave* segments in front of them. This is aimed at minimizing the interpenetration of the two domains. It is suggested to fix a point of the master line (blocked material point) to avoid the interpenetration of the two domains.

With the LAGC option, the recommendations of the preceding paragraph are irrelevant.

When the “erosion” algorithm is activated (See page A.30, Section 6.4, keyword FAIL), the sliding surfaces are updated at each time step by eliminating the failed elements.

Sliding surfaces (3D):

For the sliding surfaces, the master and slave entities are defined by the elements composing them (possibly these are GIBI objects). If continuum elements are used, then it is not necessary to define the “inner” or “outer” sides of such entities. However, when shell elements are used, it is mandatory to define the outer half-space of the shell structure by a point.

SELF keyword is necessary if contact on both sides of a shell is considered with the same set of slaves (typically, the nodes of the shell itself). It prevents contact from being detected if a slave node has penetrated the shell of a value greater than the gap (see PGAP keyword). Without this, each slave node would initially be found in contact with one of the side of the shell.

With the MAIT NODE option, the master entity must be defined by the nodes belonging to the sliding surface.

It is not admitted to define master objects (nor slave objects) formed by continuum and shell elements at the same time.

Remark (2D and 3D):

The sliding nodes may not be linked by other imposed relations (LIAISONS), except in the case where the treatment of sliding lines (or surfaces) is done by the method of Lagrange multipliers (option LAGC).

Auto-contact:

This directive indicates that the surface formed by the set of faces defined by the user may be in contact. This surface is both master and slave at the same time.

The surface may only be formed by faces of continuum elements (CUBE, PRISME, etc.) or by thick shell elements with 8 nodes (SHB8). The shells with 3 or 4 nodes (DKT3, DST3, Q4GS, etc.) are currently not treated by this directive.

When using the penalty method to compute contact forces, contact stiffness is computed automatically from the stiffness of master elements using the following formulae :

$$k = r_{\text{fac}} \frac{GS^2}{V}$$

in the case of solid master elements, with :

G : bulk modulus of master element's material,

S : area of contacting face,

V : volume of master element.

$$k = r_{\text{fac}} \frac{GS}{L}$$

in the case of shell master elements, with :

G : bulk modulus of master element's material,

S : area of master element,

L : maximum length of master element's edges.

10.22 METHOD OF PARTICLES AND FORCES

Object:

This directive usually defines the interaction between a solid (structure, composed either of continuum or of shell elements) and a set of particles representing a soft body. The solid is composed either by continuum elements or by shell elements while the set of particles (“billes”) are related by specific interaction forces.

Alternatively, by omitting the definition of the solid, interaction can occur between particles only.

Compatibility: LIAI

Syntax:

```
"MPEF" nbpef * (
    $[ "STRU" /LECTURE/          ;
      "COQU" /LECTURE/ "EXTE" /LECTURE/ ]$
      "BILL" /LECTURE/          )
```

nbpef

Number of pairs (“STRU”, “BILL”) or (“COQU”, “BILL”) or of single “BILL” groups (in case of no structure).

"STRU" /LECTURE/

Numbers of the “master” elements of the solid structure, meshed by continuum elements.

"COQU" /LECTURE/

Numbers of the “master” elements of the solid structure, meshed by shell elements.

"EXTE" /LECTURE/

Number of a node defining the “external” half-space to the solid shell structure. Here external means the side of the shell onto which the particles are going to impact.

"BILL" /LECTURE/

Numbers of the nodes belonging to the “BILL” elements.

Comments:

If the structure domain presents a large convexity, it is advisable that the faces of the elements of the structure be longer than the diameter of the neighbouring particles. This in order to minimize the interpenetration between the two domains.

The data relative to this method are identical to those of the SPH method, described on page D.187.

10.23 SMOOTHED PARTICLE HYDRODYNAMICS METHOD (S.P.H.)

Object:

This directive usually defines the interaction between a solid (structure, composed either of continuum or of shell elements) and a set of particles representing a soft body. The solid is composed either by continuum elements or by shell elements while the set of particles (“billes”) is governed by the so-called Smoothed-Particle Hydrodynamics (SPH) method.

Alternatively, by omitting the definition of the solid, interaction can occur between particles only.

Compatibility: LIAI

Syntax:

```
"SPHY" nbpef * (
    $[ "STRU" /LECTURE/          ;
      "COQU" /LECTURE/ "EXTE" /LECTURE/ ]$
      "BILL" /LECTURE/          )
```

nbpef

Number of pairs (“STRU”, “BILL”) or (“COQU”, “BILL”) or of single “BILL” groups (in case of no structure).

"STRU" /LECTURE/

Numbers of the “master” elements of the solid structure, meshed by continuum elements.

"COQU" /LECTURE/

Numbers of the “master” elements of the solid structure, meshed by shell elements.

"EXTE" /LECTURE/

Number of a node defining the “external” half-space to the solid shell structure. Here external means the side of the shell onto which the particles are going to impact.

"BILL" /LECTURE/

Numbers of the nodes belonging to the “BILL” elements.

Comments:

The data relative to this method are identical to those of the PEF method, described on page D.185.

10.24 LINKS FOR THE DISCRETE ELEMENT METHOD

Object:

This directive defines different types of links (interactions) between discrete elements (ELDI) within one or several sets of particles. Links may be of two kinds : cohesive links and contacts. The interaction forces between the discrete elements are then computed with respect to the types of material used.

Compatibility: COUP

Syntax:

```
"TYPL"  nbtypl*([ [ "COHE" <"IMPR"> <"COEF" val>    /LECTURE/ ;
                  "BIMA" <"IMPR">
                        "MAT1" <"COEF" val> /LECTURE/
                        "MAT2" <"COEF" val> /LECTURE/ ;
                  "CONT" <"IMPR"> <"COEF" val>
                        "OBJ1"          /LECTURE/
                        "OBJ2"          /LECTURE/ ] | )
```

nbtypl

Number of sequences beginning from one of the following words : "COHE" or "BIMA" or "CONT".

"COHE"

This keyword initializes the search of cohesive interactions within a set of discrete elements.

<"COEF" val>

Interaction range. The default value of the interaction range *val* is 1.

"BIMA"

This keyword initializes the search of cohesive interactions between two sets of discrete elements (permanent contact of two materials).

"MAT1" <"COEF" val>

This keyword allows to define the first set of discrete elements and its interaction range *val*. The default value is 1.

"MAT2" <"COEF" val>

This keyword allows to define the second set of discrete elements and its interaction range *val*. The default value is 1.

"CONT"

This keyword initializes the search of contact interactions between two sets of discrete elements. In this case, the value of the interaction range is 1.

"OBJ1"

This keyword allows to define the first set of discrete elements.

"OBJ2"

This keyword allows to define the second set of discrete elements.

<"IMPR">

This optional keyword allows to print out in the output listing the result of the interactions search.

/LECTURE/

List of the discrete elements concerned.

Comments:

To identify the interacting neighbors, a grid subdivision method is used.

An interaction between elements a and b of radius R^a and R^b respectively, is defined within an interaction range val and does not necessarily imply that two elements are in contact (for cohesive interactions). Then, these elements will interact if,

$$val * (R^a + R^b) > or = D^{a,b}$$

where $D^{a,b}$ is the distance between the centroids of element a and b and where val is the interaction range.

val is mandatory $> or = 1$.

10.25 CONNECTING FINITE AND DISCRETE ELEMENT MODELS

Object:

This directive defines a bridging (recovering) zone allowing to couple a set of discrete elements (ELDI) with a 3D finite element model (meshed with the CUB8 element only) or a shell model (Q4GS elements only).

The coupling equations are solved using Lagrange multipliers. To simplify, a diagonal matrix is used. It's possible to couple discrete elements by using the complete matrix through the LINK procedure.

Compatibility: COUP

Syntax:

```
"EDEF" nbcoup
      nbcoup*( "NCOU"  ncouches
               "ELDI"  /LECTURE/
               "FRON"  /LECTURE/ )
```

nbcoup

Number of combined finite/discrete zones to connect.

"NCOU"

Number of finite element range defining the combined finite/discrete element zone.

"ELDI" /LECTURE/

List of the discrete elements concerned to research in the combined finite/discrete element zone.

"FRON" /LECTURE/

List of nodes forming the border of the finite elements mesh in the bridging finite/discrete element zone.

10.26 BIFURCATION CONNECTION

Object:

Writes the relations that ensure the conservation of mass flow rate for the fluid, and the equality of mechanical d.o.f.s if necessary (case of 1D coupled fluid calculation).

This directive may only be used in 1D.

Compatibility: COUP, LIAI

Syntax:

```
"BIFU"    < LIBR >    /LECTURE/
```

```
/LECTURE/
```

Numbers of the BIFU elements for which the conservation of flow rate must be imposed.

Comments:

This directive may only be used in 1D, coupled or not, and for the junctions between the following elements:

		TUBE		TUYA		POUT	

TUBE		yes		yes		-	
TUYA		yes		yes		no	
POUT		-		no		no	

In the case of a bifurcation linking an element TUBE with an element TUYA, there may be only two nodes connected in the directive /LECTURE/ (no multiple branches).

In the case of bifurcations (even multiple) between TUYA, the 6 mechanical d.o.f.s are connected (continuity of the beam). In order to avoid these connections (for rxample in the case of a 'soufflet'), add the keyword "LIBR". On the contrary, between a TUBE and a TUYA the 6 mechanical d.o.f.s are left free, and the keyword "LIBR" is irrelevant.

Outputs:

The various components of the ECR table are as follows:

ECR(1) : density (all materials)

ECR(6) : internal energy (water)

10.27 ADHESION CONNECTION

Object:

This link can describe adhesion connections between two surfaces. The contact will be opened, when a failure criterion is reached. From this point on, the link can not sustain any tension forces. But can still react to compression forces, when the gap is closed.

Until now, the adhesion connection is only implemented for 2D cases.

Compatibility: COUP

Syntax:

```
"ADHE"  "AUTO"  auto <"CRIT" "TENS" tens >  /LECTURE/
```

auto

Maximum distance for the automatic search.

tens

Maximum tensile strength for the definition of the failure.

/LECTURE/

Objects which should be taken into account for the automatic search.

10.28 TUBM CONNECTION (3D-1D JUNCTION)**Object:**

Write the relations ensuring the conservation of mass flow rate for the fluid (Eulerian formulation)

Compatibility: COUP, LIAI

Syntax:

"TUBM" /LECTURE/

/LECTURE/

Numbers of the "TUBM" elements (or names of the GIBI objects), which form the junctions.

10.29 TUYM CONNECTION (3D-1D JUNCTION)**Object:**

Write the relations ensuring the conservation of mass flow rate for the fluid (moving meshes).

Compatibility: COUP, LIAI

Syntax:

"TUYM" /LECTURE/

/LECTURE/

Numbers of the "TUYM" elements (or names of the GIBI objects), which form the junctions.

10.30 TUYA CONNECTION (3D-1D JUNCTION)

Object:

Automatically writes the mechanical relations among d.o.f.s of a pipeline meshed by beams and a pipeline meshed by thin shells.

Compatibility: COUP, LIAI

Syntax:

```
"TUYAU"  "CENTRE" /LECTURE/  
          "LISTE"  /LECTURE/
```

"CENT" /LECTURE/

Number of the node (or name of the object) corresponding to the extremity of the pipeline meshed by beams.

"LIST" /LECTURE/

Number(s) of the node(s) (or name of the object) corresponding to the circle, extremity of the pipeline meshed by thin shells.

Comments:

This directive automatically writes the relations between the displacements of nodes belonging to the shells and the beam. All rotations are supposed to be equal.

All nodes involved by the link (including the CENT node) must have 6 dofs, since the imposed relations involve also the rotations. Therefore, the CENT node cannot be simply represented by a (stand-alone) PMAT, which has only 3 dofs. In such a case, it is sufficient to attach a (dummy) beam or shell element to the CENT.

10.31 RIGID BODY (SOLIDE INDEFORMABLE)

Object:

This directive defines the sub-structures that will be considered as rigid bodies.

It also allows to impose the inertia tensor of the solid, or to leave EUROPLEXUS compute it starting from the mesh, or from a composition of simple homogeneous solids.

The directive may be used in two ways:

- The solid is meshed, i.e. its form is represented by a set of elements
- The solid is not meshed, i.e. one imposes that a small number of points be rigidly connected.

Compatibility: COUP, LIAI

Syntax:

```
"SOLI" nsol*( ... )
```

1st case - Solid meshed by elements:

```

"ELEM" /LECTURE/  "PLIE" /LECTURE/ ...
      $ < "COMP" ncomp*( "INNER" ... )   >  $
      $ < "INNER" ...   >                  $

```

2nd case - Rigidly connected points:

```
"POIN" /LECTURE/
```

nsol

Number of non-deformable solids.

```
"ELEM" /LECTURE/
```

Numbers of the elements composing the solid.

```
"PLIE" /LECTURE/
```

Numbers of the points of the solid to be conserved because they take part in a connection (linked points).

ncomp

Number of homogeneous simple solids whose combination allows to compute the inertia tensor. In the case that $ncomp = 1$, this parameter is optional.

"INER"

This option allows to introduce the parameters of inertia of a solid, that will replace the ones computed starting from the mesh and the initial materials.

"POIN" /LECTURE/

Numbers of the points rigidly linked (case of the non-meshed solid).

Comments:

A sub-structure described like a non-deformable solid will reduce to a system of four material points. The calculation will be done with these points, and the solid will then be reconstructed to be visualized.

The linked points (participatin in a connection) will be conserved in the calculation in order to be able to write down the connection relation.

The other points are not conserved in the calculation. However, they are used for the calculation of the inertia tensor. Care must then be taken that the discretization be sufficient, else the parameters related to the solid will be imprecise, and the computation will be affected by errors.

The "INER" directive is optional. It imposes to the solid inertia values coming from an external calculation. If it is absent, EUROPLEXUS computes inertias from the mesh.

If you impose the inertia tensor via "INER", you may limit the mesh to the minimum indispensable, by directly connecting the linked points (wireframe mesh). In any case, at least ONE free poit per solid is necessary, i.e. two linked points will be connected by at least two beam elements.

In the case of complex solids, it is interesting to mesh them finely from the beginning, and to let EUROPLEXUS compute the inetria tensor. The option VERIF is enough for that. For the real dynamic calculation, a coarser mesh (wireframe) will be sufficient, and one will then impose the previously found inertia tensor, by nmeans of the INER directive.

In the case that the solid is not meshed (directive "POIN"), all points of the list will be considered linked. The inertia tensor data is then useless.

Dimension sufficiently by means of directives "SOL", "PLIE" and "PLIB" (page A.80).

10.31.1 INERTIA**Object:**

This directive allows to specify inertia parameters for a non-deformable solid. It also allows to compute the inertia tensor starting from simple shapes.

Compatibility: COUP, LIAI

Syntax:

```
"COMP"  ncomp*(    "INER"      "MASS"  m  ...
                  ... <"XG"  xg>    <"YG"  yg>    <"ZG"  zg>    ...
                  ... <"IXX"  ixx>   <"IYY"  iyy>   <"IZZ"  izz>   ...
                  ... <"IXY"  ixy>   <"IXZ"  ixz>   <"IYZ"  iyz> )
```

"COMP"

Announces that the inertia tensor is composed by assembly of simple tensors.

ncomp

Number of inertia tensors to be read in order to compute the inertia tensor of the composite solid.

"INER"

Announces the beginning of the data relative to an inertia tensor.

m

Mass of the isolated solid (without taking into account the added masses).

xg,yg,zg

Coordinates of the center of gravity of the solid isolated in the general reference frame (frame of the mesh).

ixx,iyy,izz

Diagonal elements of the inertia tensor of the isolated solid, in the general frame translated to the center of gravity of the solid.

ixy,iyz,ixz

Off-diagonal elements of the inertia tensor of the isolated solid, in the general frame translated to the center of gravity of the solid.

Comments:

If one single inertia tensor is given ($ncomp = 1$), the keywords `< "COMP" ncomp >` are optional. One may start directly by the keyword `"INER"`.

If the `"INER"` directive is absent, the inertia values will be computed from the initial mesh and densities.

The inertia tensor has the followig form:

$$I = \begin{pmatrix} i_{xx} & i_{xy} & i_{xz} \\ i_{xy} & i_{yy} & i_{yz} \\ i_{xz} & i_{yz} & i_{zz} \end{pmatrix}$$

If some parameters are not explicitly given, they are supposed to be zero by default.

In case of complex solids, it is interesting to discretise them finely, and let EUROPLEXUS compute the inertia tensor with high precision. When this operation is terminated, one can take a coarser mesh, by imposing the formerly obtained inertia terms. In this way, the output files will be smaller. But the precision of the calculation will ne the same.

10.32 ARTICULATION

Object:

This directive allows to link two sub-structures by meand of a kinematic relationship.

Compatibility: COUP, LIAI

Syntax:

```
"ARTI"  
  | "VERR"  ...  |  
  | "ROTU"  ...  |  
  | "PIVO"  ...  |  
  | "GLIS"  ...  |  
  | "PIGL"  ...  |  
  | "DRIT"  ...  |
```

Comments:

Articulations may only be defined by means of a mechanism element "MECA". It is therefore necessary that such elements be present in the mesh.

The linked sub-structures may be described as either non-deformable or deformable.

The various types of articulations are described in the following pages.

10.32.1 RIGID ARTICULATION (“VERROU”)**Object:**

This directive allows to join two sub-structures by means of a blocked articulation, i.e. a rigid connection.

Compatibility: COUP, LIAI

Syntax:

```
"VERR" /LECTURE/ ...

... ( "NOEU" /LECTURE/  "VOIS" $[ "ABSENT"      ;
                                "INDEF"  isol  ;
                                /LECTURE/    ]$ )
```

"VERR" /LECTURE/

Number of the "MECA" element to be rigidly connected.

"NOEU" /LECTURE/

Number of the node of the "MECA" element to which the following neighbour will be associated.

"VOIS" "ABSENT"

There is no need to define a neighbour because the nodes of this sub-structure already have 6 d.o.f.s. The node of the mechanism is then sufficient.

"VOIS" "INDEF" isol

The neighbour is part of the non-deformable solid isol. The points resulting from the decomposition will be used. It seems that this directive cannot be used when the solid is defined by "POIN" (not meshed solid).

"VOIS" /LECTURE/

Number of the points forming the neighborhood (the point belonging to the mechanism must be excluded).

Comments:

The two sub-structures are rigidly connected. The six degrees of freedom are coupled on both parts of the mechanism.

The couple "NOEU" "VOIS" must be described twice, i.e. for each of the two points of the mechanism.

10.32.2 PIVOT**Object:**

This option allows to link two sub-structures by a frictionless hinge.

Compatibility: COUP, LIAI

Syntax:

```
"PIVOT" /LECTURE/      ...
      ...  "AXE"  "VX" vx  "VY" vy  "VZ" vz  ...

      ... ( "NOEU" /LECTURE/  "VOIS" $[  "ABSENT"      ;
                                           "INDEF"  isol  ;
                                           /LECTURE/      ]$ )
```

"PIVOT" /LECTURE/

Number of the "MECA" element of the hinge.

vx,vy,vz

Components of the initial direction of the hinge axis.

"NOEU" /LECTURE/

Number of the node of the "MECA" element to which the following neighborhood will be associated.

"VOIS" "ABSENT"

There is no need to define a neighborhood because the nodes of this sub-structure already possess 6 d.o.f.s. The first node of the mechanism is then sufficient.

"VOIS" "INDEF" isol

The neighbourhood is part of the non-deformable solid isol. The points resulting from the decomposition will then be used. It seems that this directive cannot be used when the solid is defined by "POIN" (not meshed solid).

"VOIS" /LECTURE/

Numbers of the points forming the neighbourhood (the point belonging to the mechanism must be excluded).

Comments:

The pivot axis is modified accounting for the motions of the sub-structures.

The pair "NOEU" "VOIS" must be described twice, once for each of the 2 points of the mechanism.

Special care must be taken for the neighborhood. In fact, these parts will be considered as rigid for the calculations of angular relations.

10.32.3 PIN JOINT (“ROTULE”)**Object:**

This option allows to connect two sub-structures by a friction-less pin joint (“rotule”).

Compatibility: COUP, LIAI

Syntax:

```
"ROTU"  /LECTURE/  ...

          ... ( "NOEU" /LECTURE/  "VOIS" $[  "ABSENT"          ;
                                           "INDEF"  isol    ;
                                           /LECTURE/          ]$ )
```

"ROTU" /LECTURE/

Number of the "MECA" element of the pin joint.

"NOEU" /LECTURE/

Number of the node of the "MECA" element to which the following neighborhood will be associated.

"VOIS" "ABSENT"

There is no need to define a neighborhood because the nodes of this sub-structure already possess 6 d.o.f.s. The first node of the mechanism is then sufficient.

"VOIS" "INDEF" isol

The neighbourhood is part of the non-deformable solid isol. The points resulting from the decomposition will then be used. It seems that this directive cannot be used when the solid is defined by "POIN" (not meshed solid).

"VOIS" /LECTURE/

Numbers of the points forming the neighbourhood (the point belonging to the mechanism must be excluded).

Comments:

The two sub-structures are linked in translation but free in rotation.

The pair "NOEU" "VOIS" must be described twice, once for each of the 2 points of the mechanism.

10.32.4 SLIDER (“GLISSIERE”)**Object:**

This option allows to connect two sub-structures by a friction-less slider (“glissière”).

Compatibility: COUP, LIAI

Syntax:

```
"GLIS" /LECTURE/    ...
      ...  "AXE"  "VX" vx  "VY" vy  "VZ" vz  ...

      ... ( "NOEU" /LECTURE/  "VOIS" $[  "ABSENT"      ;
                                           "INDEF"  isol  ;
                                           /LECTURE/    ]$ )
```

"GLIS" /LECTURE/

Number of the "MECA" element of the pin joint.

vx, vy, vz

Components of the initial direction of the axis.

"NOEU" /LECTURE/

Number of the node of the "MECA" element to which the following neighborhood will be associated.

"VOIS" "ABSENT"

There is no need to define a neighborhood because the nodes of this sub-structure already possess 6 d.o.f.s. The first node of the mechanism is then sufficient.

"VOIS" "INDEF" isol

The neighbourhood is part of the non-deformable solid isol. The points resulting from the decomposition will then be used. It seems that this directive cannot be used when the solid is defined by "POIN" (not meshed solid).

"VOIS" /LECTURE/

Numbers of the points forming the neighbourhood (the point belonging to the mechanism must be excluded).

Comments:

The slider axis is modified to account for the motion of the sub-structures.

The pair "NOEU" "VOIS" must be described twice, once for each of the 2 points of the mechanism.

The axis defined by "AXE" is used only in case of a spring ("RESS") on the connection (to compute the forces coming from the spring) or in case of merging points of the MECA element. In general case, the sliding axis is defined by the two points of the MECA element (local axis of the element).

10.32.5 SLIDING PIVOT**Object:**

This option allows to connect two sub-structures by a friction-less sliding pivot.

Compatibility: COUP, LIAI

Syntax:

```
"PIGL" /LECTURE/    ...
      ...  "AXE"  "VX" vx  "VY" vy  "VZ" vz  ...

      ... ( "NOEU" /LECTURE/  "VOIS" $[  "ABSENT"      ;
                                           "INDEF"  isol  ;
                                           /LECTURE/    ]$ )
```

"PIGL" /LECTURE/

Number of the "MECA" element of the sliding pivot.

vx,vy,vz

Components of the initial direction of the axis.

"NOEU" /LECTURE/

Number of the node of the "MECA" element to which the following neighborhood will be associated.

"VOIS" "ABSENT"

There is no need to define a neighborhood because the nodes of this sub-structure already possess 6 d.o.f.s. The first node of the mechanism is then sufficient.

"VOIS" "INDEF" isol

The neighbourhood is part of the non-deformable solid isol. The points resulting from the decomposition will then be used. It seems that this directive cannot be used when the solid is defined by "POIN" (not meshed solid).

"VOIS" /LECTURE/

Numbers of the points forming the neighbourhood (the point belonging to the mechanism must be excluded).

Comments:

The sliding pivot's axis is modified to account for the motion of the sub-structures.

The pair "NOEU" "VOIS" must be described twice, once for each of the 2 points of the mechanism.

The rotational axis is supposed to be identical with the sliding axis. This single axis is defined with the "AXE" keyword. Nevertheless, for the sliding behavior, the axis defined by "AXE" is used only in case of a spring ("RESS") on the connection (to compute the forces coming from the spring) or in case of merging points of the MECA element. In general case, the sliding axis is defined by the two points of the MECA element (local axis of the element).

10.32.6 IMPOSED RELATIVE DISPLACEMENT - D.R.I.T.**Object:**

This D.R.I.T. directive (Déplacement Relatif Imposé en fonction du Temps = Prescribed Time-dependent Relative Displacement) allows to link two sub-structures by an actuator (“vérin”) whose length is a prescribed time function.

Compatibility: LIAI

Syntax:

```
"DRIT"  /LECTURE/      ...
        ...  "AMPLI"  ampli  "FONCTION" ifonc      ...

        ... ( "NOEU" /LECTURE/  "VOIS" $[  "ABSENT"      ;
                                           "INDEF"  isol  ;
                                           /LECTURE/      ]$ )
```

"DRIT" /LECTURE/

Number of the "MECA" element of the "DRIT" mechanism.

ampli

Amplification coefficient.

ifonc

Number of the function defined by the "FONCTION" directive (see page E.10).

"NOEU" /LECTURE/

Number of the node of the "MECA" element to which the following neighborhood will be associated.

"VOIS" "ABSENT"

There is no need to define a neighborhood because the nodes of this sub-structure already possess 6 d.o.f.s. The first node of the mechanism is then sufficient.

"VOIS" "INDEF" isol

The neighbourhood is part of the non-deformable solid isol. The points resulting from the decomposition will then be used.

"VOIS" /LECTURE/

Numbers of the points forming the neighbourhood (the point belonging to the mechanism must be excluded).

Comments:

The relative displacement between the two nodes of the element is equal to the product $\text{ampli} * F(\text{ifonc}, t)$.

The pair "NOEU" "VOIS" must be described twice, once for each of the 2 points of the mechanism.

10.33 ROTATION

Object:

In the case of a rotating structure, this directive allows to define the symmetry condition with respect to a rotating plane, whose axis and rotation velocity are prescribed by the user.

This directive allows, for example, to model just one sector of a rotating disk instead of the whole disk.

Compatibility: COUP, LIAI

Syntax:

```
"ROTATION"      "ORIGINE"    x0 y0  < z0 >
                  < "VECTEUR"  vx vy   vz >
                  "FONCTION"  ifonc
                  ...                               /LECTURE/
```

x0,y0,z0

Coordinates of the origin point (z0 is redundant in 2D).

vx,vy,vz

Components of the vector defining the rotation axis. These data are not necessary in 2D (see comments below).

ifonc

Number of the function defining the rotation velocity (in rad/s) as a function of time.

LECTURE

Numbers of the concerned nodes.

Comments:

This directive may be used at most once in a calculation.

The rotation axis is supposed fixed. The velocity of rotation varies in time according to the user-specified function.

In 2D plane calculations, the rotation axis is normal to the plane xOy.

10.34 GLOBAL MOTION

Object:

In the case of a rotating structure, this directive allows to define a global motion of rotation for certain nodes. The axis and the rotation velocity are prescribed by the user.

Compatibility: COUP, LIAI

Syntax:

```
"MENS"      "POINT"    x0 y0 < z0 >
              < "VECTEUR"  vx vy    vz >
              "FONCTION"  ifonc
              ...                               /LECTURE/
```

x0,y0,z0

Coordinates of the origin point (z0 is redundant in 2D).

vx,vy,vz

Components of the vector defining the rotation axis. These data are not necessary in 2D (see comments below).

ifonc

Number of the function defining the rotation velocity (in rad/s) as a function of time.

LECTURE

Numbers of the concerned nodes.

Comments:

This directive may be used at most once in a calculation.

The rotation axis is supposed fixed. The velocity of rotation varies in time according to the user-specified function.

In 2D plane calculations, the rotation axis is normal to the plane xOy.

10.35 CONSTANT DISTANCE CONNECTION (“DIST”)**Object:**

Automatic prescription of the 3D mechanical relations between the translational degrees of freedom of a point with a set of points.

Compatibility: COUP, LIAI

Syntax:

"DISTANCE" /LECTURE/

/LECTURE/

Numbers of the two nodes (or name of the object) whose distance must be kept constant during the calculation.

10.36 BARYCENTRIC JUNCTION

Object:

Automatic prescription of mechanical relations (links) such that the displacement of a point equals the **mean value** of the displacements of a set of points, i.e. the displacement of the barycenter of the set of points (considered all with the same weight).

Compatibility: COUP, LIAI

Syntax:

```
BARY  CENT /LECT/  
      LIST /LECT/  
      <VECT <VX vx> <VY vy> <VZ vz>>
```

CENT /LECT/

Number of the node (or name of the object) corresponding to the “central” (or reference) node. This node may be located anywhere and does not need to be at (or close to) the true center of the following points set.

LIST /LECT/

Numbers of the nodes (or name of the object) corresponding to the set of nodes, whose *mean* displacement will be identical to that of the reference node.

VECT

Introduces the optional definition of a direction (vector) along which the constraint will act. By default, the constraint acts along all space directions.

VX, VY, VZ

Introduce the optional components of the vector **vx**, **vy**, **vz**. By default, they are zero. At least one non-zero component must be specified. The **vz** component may only be specified in 3D. Note that only the direction, not the norm, of the vector counts. The vector is always normalized to unit length internally.

Comments:

By default (no **VECT** specified) this directive imposes the following (vectorial) condition on nodal velocities \underline{v} :

$$\underline{v}_C - (\underline{v}_1 + \underline{v}_2 + \cdots + \underline{v}_N)/N = \underline{0}$$

which corresponds to the following 2 or 3 scalar *independent* links:

$$v_{Cx} - (v_{1x} + v_{2x} + \cdots + v_{Nx})/N = 0$$

$$v_{Cy} - (v_{1y} + v_{2y} + \cdots + v_{Ny})/N = 0$$

$$v_{Cz} - (v_{1z} + v_{2z} + \cdots + v_{Nz})/N = 0 \quad (3D \text{ only})$$

where C is the “central” node defined by **CENT** and $1, 2, \dots, N$ are the N nodes defined by **LIST**.

When a vector \underline{V} is specified by **VECT**, then the following single condition on nodal velocities is imposed:

$$\underline{v}_C \cdot \underline{V} - \frac{1}{N}(\underline{v}_1 - \underline{v}_2 - \cdots - \underline{v}_N) \cdot \underline{V} = 0$$

which corresponds to the following scalar link (assuming a 2D case):

$$v_{Cx}V_x + v_{Cy}V_y - \frac{1}{N}(v_{1x}V_x + v_{1y}V_y + \cdots + v_{Nx}V_x + v_{Ny}V_y) = 0$$

Note that the above conditions, both without and with the definition of a vector **VECT**, do *not* strictly ensure that the displacements of all nodes in the set will be all equal among them. To obtain this effect, use the **RIGI** link, see page D.326.

10.37 RIGID JUNCTION

Object:

Automatic prescription of mechanical relations (links) such that the displacement of each point in a certain set of points equals the displacement of a reference point, like if all these points were all rigidly connected among them.

Compatibility: COUP, LIAI

Syntax:

```
RIGI  CENT /LECT/  
      LIST /LECT/  
      <VECT <VX vx> <VY vy> <VZ vz>>
```

CENT /LECT/

Number of the node (or name of the object) corresponding to the “central” or reference node. This node may be located anywhere and does not need to be at (or close to) the true center of the following points set.

LIST /LECT/

Numbers of the nodes (or name of the object) corresponding to the set of nodes, whose displacement will be identical to that of the reference node.

VECT

Introduces the optional definition of a direction (vector) along which the constraint will act. By default, the constraint acts along all space directions.

VX, VY, VZ

Introduce the optional components of the vector **vx**, **vy**, **vz**. By default, they are zero. At least one non-zero component must be specified. The **vz** component may only be specified in 3D. Note that only the direction, not the norm, of the vector counts. The vector is always normalized to unit length internally.

Comments:

By default (no **VECT** specified) this directive imposes the following set of N (vectorial) conditions on nodal velocities \underline{v} :

$$\underline{v}_C - \underline{v}_1 = \underline{0}$$

$$\underline{v}_C - \underline{v}_2 = \underline{0}$$

...

$$\underline{v}_C - \underline{v}_N = \underline{0}$$

which corresponds to the following $2N$ or $3N$ scalar *independent* links:

$$v_{Cx} - v_{1x} = 0$$

$$v_{Cy} - v_{1y} = 0$$

$$v_{Cz} - v_{1z} = 0 \quad (3D \text{ only})$$

...

$$v_{Cx} - v_{Nx} = 0$$

$$v_{Cy} - v_{Ny} = 0$$

$$v_{Cz} - v_{Nz} = 0 \quad (3D \text{ only})$$

where C is the “central” node defined by **CENT** and $1, 2, \dots, N$ are the N nodes defined by **LIST**.

When a vector \underline{V} is specified by **VECT**, then the following N conditions on nodal velocities are imposed:

$$\underline{v}_C \cdot \underline{V} - \underline{v}_1 \cdot \underline{V} = 0$$

$$\underline{v}_C \cdot \underline{V} - \underline{v}_2 \cdot \underline{V} = 0$$

...

$$\underline{v}_C \cdot \underline{V} - \underline{v}_N \cdot \underline{V} = 0$$

which correspond to the following N scalar links (assuming a 2D case):

$$v_{Cx}V_x + v_{Cy}V_y - v_{1x}V_x - v_{1y}V_y = 0$$

$$v_{Cx}V_x + v_{Cy}V_y - v_{2x}V_x - v_{2y}V_y = 0$$

...

$$v_{Cx}V_x + v_{Cy}V_y - v_{Nx}V_x - v_{Ny}V_y = 0$$

10.38 CONTACTS DEFINED BY SPLINE FUNCTIONS

Object:

In the case of a rotating structure, this directive allows to define the possible contacts between the rotating parts (blades) with the fixed wall (carter). The geometrical forms of these parts are defined by means of spline functions starting from the positions of mesh nodes. This interpolation allows thus to approximate the real geometry of such structures.

Compatibility: LIAI

Syntax:

```
"SPLINE"
nspline * (      "SURFACE"  /LECTURE/      ...
                ... "COURBE"  ncourbe * ( "LIGNE" /LECTURE/ )      ...
                ... "METC"   metc   "METS"   mets   "NPTT"   nptt   ...
                ... "DEGC"   degc   "DGST"   dgst   "DGSZ"   dgsz   ...
                ... "EPAIS"  epais  "FREQ"   freq                                     )
```

nspline

Number of splines.

SURFACE

Defines the nodes forming the surface. This surface **MUST** be a cylinder,

ncourbe

Number of curves that may get in contact with the surface.

LIGNE

Introduces the nodes that compose a curve. **The user must enter these nodes in the order of their position along the curve.**

metc

Method for the modelisation of the curve (see comments below).

mets

Method for the modelisation of the surface (see comments below).

nptt

Number of nodes of the surface lying on the same circumference.

degc

Degree for the modelisation of the curve.

dgst

Degree for the modelisation of the surface, circumferential direction.

dgst

Degree for the modelisation of the surface, axial direction Oz.

epais

Thickness of the shell elements composing the surface.

freq

Frequency of the updationg of surface nodes.

Comments:

The methods for the modelisation (of the curve and of the surface along the circumferential and axial directions) may assume the values: 1 (direct), 2 (interpolation) or 3 (smoothing by least squares).

The surface MUST be a cylinder of axis Oz. Furthermore, the nodes composing it must be regularly spaced.

10.39 COLLISIONS

Object :

This directive allows to simulate the contact and/or shock without friction between the envelopes of 3D rigid bodies.

Compatibility: LIAI

Syntax :

```
"COLL"      "REST" crest      "SGEO"  tolgeo      "SVIT"  tolvit
      ( "CHAI"
        ( "SOLI"  nusoli
          "SURF"  /LECTURE/
          "EPAI"  epais
          "ORIE"  xp  yp  zp      )      )
      "CONT"
      ( "CHA1"  nucha1      "CHA2"  nucha2      )
      "FCON"
```

COLL

This keyword announces the data relative to collisions.

crest

Energy restitution coefficient.

tolgeo

Geometric tolerance of the contact.

tolvit

Kinematic tolerance of the contact.

CHAI

This keyword announces the data relative to a chain.

SOLI

This keyword announces the data relative to one of the solids that define the chain.

nusoli

Number of the solid associated to the chain. This number corresponds to the order under which the solid has been listed under the sub-directive "SOLIDE" of the directive "LIAISON".

/LECTURE/

Reading procedure of the triangular elements defining the envelope of the chain, i.e. the contact surfaces.

epais

Thickness of the contact surfaces.

xp, yp, zp

Coordinates of a point interior to the envelope, used to define the orientation of the triangular elements.

CONT

This keyword allows to introduce the list of pairs of chains for which contact may take place.

nucha1

Number of the first chain of the pair.

nucha2

Number of the second chain of the pair.

FCON

This keyword announces the end of the collisions data.

Warning :

It is mandatory:

- to mesh the surfaces by triangular elements;
- to declare these elements as "phantoms" by directive "MATE",
- to define the data block "SOLIDE" before the block "COLLISIONS",
- to specify in "DIME" the dimensioning parameter:

"CSCO" nbpcon

With:

nbpcon

Maximum number of contact points.

Comments :

The coefficient of energy restitution is between 0 and 1. For $\text{crest} = 0$, one obtains a perfectly soft shock, while for $\text{crest} = 1$ one gets a perfectly elastic shock (the energy is conserved).

The thickness of surfaces must be of the order of the size of elements at most. If this value is too small, it is possible that the interpenetration of the two surfaces will not be detected.

The contact geometric tolerance determines the distance starting from which one considers that there is contact.

The kinematic tolerance must be of the order of the time step. The larger this tolerance, the more the discontinuity at the velocity level due to a shock is ignored.

If a contact surface is fixed (instead of being defined via a rigid body), it is sufficient to declare `nusoli = 0`. In this case it is redundant to block the concerned nodes, since it is done automatically by the code.

References :

For further information, please consult the reference [\[532\]](#).

10.40 FLUID-STRUCTURE SLIDING OF THE ALE TYPE (FSA)

Object:

To define fluid-structure sliding of the ALE type according to the FSA model developed at JRC Ispra.

The program writes for each node subjected to this type of sliding a 'liaison' that forces the fluid (slave) velocity to be equal to the structure (master) velocity along the normal to the FS interface. In the tangent direction (tangent plane in 3D), the fluid velocity is free.

In the case of a curved interface, the normal direction is determined at each step by taking into account all the element faces that lie along the fluid-structure boundary and include the node under consideration (influence domain) and by imposing that the net flux of mass out of some faces be balanced by the flux entering the other faces.

Since the geometry varies in time, the coefficients of the liaison have to be recalculated and the matrix inverted at each step.

The nodes declared in this directive should be fluid nodes and be declared as Eulerian in the GRIL directive. The program then automatically searches for each slave node a corresponding master node: this is defined as the Lagrangian node having the same coordinates as the slave node (within a small tolerance) and if it exists (nodally **conforming** FS interface), it must be unique. Usually this will be a structural node, but it could be also a fluid (Lagrangian) node, in case the sliding takes place along a fluid-fluid interface.

If no such node exists, then the FS interface is nodally **non-conforming** and the program searches a Lagrangian master **face** on which the slave fluid node lies. The motion of the fluid node is automatically set so as to follow the motion of the master face.

Note that the treatment of non-conforming FS interfaces requires a special optional keyword (NCFS) to be explicitly chosen by the user. If this keyword is not specified and a non-conforming node is found, then an error message is issued and the calculation is stopped. This is to make sure that the user intentionally wanted to specify a non-conforming interface and there was not just an error in mesh specification.

Compatibility: COUP, LIAI

Syntax:

```
"FSA" <"STRU" /LECT_STRU/> <"NCFS"> /LECTURE/
```

STRU /LECT_STRU/

Optional sub-directive used to tell the code in which object (/LECT_STRU/) it should search to determine the “structural” (i.e. the Lagrangian) nodes corresponding to the FSA fluid nodes that will be specified in the final /LECTURE/. By default, the search is extended to the whole mesh.

NCFS

The FS interface may contain non-conforming fluid nodes.

/LECTURE/

List of fluid (slave) nodes subjected to FSA sliding.

Comments:

The fluid nodes subjected to FSA sliding should preferably be declared Eulerian in the grid movement directive (**GRILL**). The program will automatically consider these nodes as manually rezoned when it encounters the **LIAI FSA** directive. The user might also declare these nodes as automatically rezoned in **GRILL** (e.g., as a consequence of an **AUTO AUTR** directive), with no effect on the results, but in this case the dimensioning for automatically rezoned nodes (**DIME NBLE**) should include these nodes, although this is not necessary for the actual computation.

Beware that the behaviour of the FSA algorithm may be modified by setting appropriate options, see page H.120. In particular, the **FSCR** option activates the correction of normals based on equilibrium considerations (**FSCR** algorithm).

Occasionally, the automatic search for the master node corresponding to a slave node might fail. The code then reports the concerned node number by an appropriate error message. This may happen because either the code finds zero nodes, or it finds more than one Lagrangian nodes matching the slave node.

In the first case, the tolerance for node matching determination might be too small, e.g. due to the fact that mesh coordinates are generated by an external, and not too precise, mesh generator. The user may adjust this tolerance, see **OPTI TOLC** on page H.40.

The second case may occur for example when there are superposed structures (coincident nodes) in the initial mesh. In such cases, there are two possibilities. Either the user specifies the required nodes correspondence by the **COMP CNOD** directive, see page C.92, but this is only practical if there are just a few of these nodes. Or, the user specifies the **STRU /LECT_STRU/** optional sub-directive, so that the search for matching structural (more precisely, Lagrangian) nodes is confined to the specified object /LECT_STRU/ rather than to the whole mesh. This is the method of choice e.g. in case a large shell structure is subjected to FSA on one side, and to Lagrangian sliding (say, by **GLIS**) on the other side, so that the number of “superposed” structural nodes is potentially large.

10.41 RIGID-BOUNDARY/FLUID SLIDING OF THE ALE TYPE

Object:

To simplify the description of fluid sliding along inviscid, rigid boundaries. The simplification lies in the fact that the program automatically computes the correct sliding conditions, in particular the normal (or possibly the 2 normals, in 3D cases) to the rigid boundary and automatically prescribes the relevant "connections" (liaisons).

For complex geometric shapes this is very convenient with respect to the "manual" prescription of all such connections.

This condition is similar to the "FSA" condition, but with the following differences:

- Since the boundary is rigid, there is no need to represent it by a structure. The sliding condition therefore involves only a fluid node.

- The geometry of the boundary does not vary in time, therefore the coefficients of the liaison are constant and do not need to be recalculated during the transient.

- The program does not search for a Lagrangian node having the same coordinates as the fluid node.

The nodes declared in this directive (/LECT/) should all be fluid nodes and be declared as Eulerian in the GRIL directive.

Compatibility: COUP, LIAI

Syntax:

"FSR" /LECTURE/

/LECTURE/

List of fluid nodes subjected to FSR sliding.

Comments:

The fluid nodes subjected to FSR sliding should preferably be declared Eulerian in the grid movement directive (GRILLE). The program will automatically consider these nodes as

Eulerian when it encounters the LIAI FSR directive. The user might also declare these nodes as automatically rezoned in GRILLE (e.g., as a consequence of an AUTO AUTR directive), with no effect on the results, but in this case the dimensioning for automatically rezoned nodes (DIME NBLE) should include these nodes, although this is not necessary for the actual computation.

10.42 IMPACT/CONTACT BY THE PINBALL MODEL

Warning:

The present directive is currently still under implementation and validation. It may not be used yet for production runs.

Object:

The purpose is to define impact and contact conditions between Lagrangian subdomains (typically two or more solid bodies) by means of the “pinball” model. The model is inspired to a formulation proposed by Belytschko and co-workers in the papers: *(i)* Ted Belytschko and Mark O. Neal, “Contact-Impact by the Pinball Algorithm with Penalty and Lagrangian Methods”, Int. J. Num. Meths. Eng., Vol. 31, pp. 547-572 (1991), and *(ii)* T. Belytschko and I.S. Yeh, “The splitting pinball method for contact-impact problems”, CMAME, 105, pp. 375-393, (1993).

The user defines the elements that may enter in contact with one another and a pinball (a sphere or circle) is associated to these elements. Interpenetration is detected by comparing the distance of the centers of two pinballs with the sum of their radii. If this condition is satisfied, equal normal velocity is enforced by the method of Lagrange multipliers and the corresponding contact forces are computed.

Optionally, contact may be verified on a hierarchy of “descendent” pinballs derived from the “parent” pinballs described above by recursively halving the pinball dimensions. This allows finer spatial resolution of the contact conditions.

The uncoupled version of the pinball algorithm (DECO keyword) uses a penalty method instead of (coupled) Lagrange multipliers.

Compatibility: COUP, DECO, LIAI

Syntax:

```
"PINB" $[ "PENA" <"SFAC" sfac> ]$
    ( $[ "BODY" ; "SELF" ]$
    < $[ "DMIN" dmin ; "MLEV" mlev ]$ >
    < "DIAM" diam >
    < "HARD" hard >
    /LECT/ )
```

PENA

DECO only: mandatory keyword (ignored with COUP or LIAI), must immediately follow the PINB keyword and indicates that a penalty method is used.

sfac

DECO only: optional coefficient ϕ for the automatic determination of the contact stiffness (see comments below). By default it is 1.0.

BODY

Introduces the declaration of a set of pinballs that form one of the bodies that may come in contact with other bodies. There may not be contact between pinballs belonging to the same body.

SELF

Introduces the declaration of a set of pinballs that form one of the bodies that may come in contact with other bodies. In this case, there may be contact between different pinballs belonging to this body (this model is called self-contact or auto-contact).

dmin

Minimum diameter of descendent pinballs that will be generated from the set being declared. By default, this value is 0 for continuum elements (the size is then governed by **mlev**, see below), or it is the element thickness for beam or shell elements.

mlev

Maximum hierarchy level for descendent pinballs that will be generated from the set being declared. The value 0 means that no descendents are generated (contact forces are computed based on interpenetration between parent or 0-level pinballs). The pinball radius is roughly divided by two at each new level produced. If specified, **mlev** must be greater or equal to 0. If not specified, **mlev** has to be computed. If **dmin** is given, then max level is computed such that min pinball diameter is of the order of **dmin**. If **dmin** is not given, for beam/shell elements **mlev** is computed such that min pinball diameter is of the order of element thickness. For continuum element and for other element types, the **mlev** value is 0.

diam

Fixed pinball diameter to be associated with elements of the material-point type (PMAT). These elements have just one node and thus their pinball radius may not be computed by the code but must be provided by the user. This value is ignored for the associated elements in the /LECT/ that are not of the PMAT type (i.e., elements having more than one node). When **diam** is specified, **dmin** may not be specified and **mlev** must be 0 (i.e., either unspecified, or specified to be 0).

hard

Optional “hardness” value to be associated with the body. This information is only used in conjunction with options OPTI PINS MASL or OPTI PINS MAS2, see page H.160, in order to eliminate constraints in multiple flat contact situations. Values of hardness are arbitrary. The only important thing is the relative value of hardness of two bodies that come into flat contact. The body with lower hardness behaves like a “slave”, and the other one as a “master”. It is advised to use simple integer values, e.g. 1, 2, 3 etc.

/LECT/

List of the elements that will be associated with a (parent or 0-level) pinball of the set being described. For continuum-like bodies, these should typically contain only those elements along the body surface which are likely to come in contact with other objects.

Comments:

Be sure to consult also the options related to the pinball model in Section H, see Page H.160, and the interactive commands for the visualization of pinballs and of contacts, see Page A.25.

When using penalty method to compute contact forces, contact stiffness is computed automatically from the stiffness of master elements using the following formulae :

$$k = \phi \frac{GS^2}{V}$$

in the case of solid master elements, with :

G : bulk modulus of master element’s material,

S : area of contacting face,

V : volume of master element.

$$k = \phi \frac{GS}{L}$$

in the case of shell master elements, with :

G : bulk modulus of master element’s material,

S : area of master element,

L : maximum length of master element’s edges.

The bulk modulus G of the material is:

$$G = \frac{E}{3(1 + \nu)}$$

where:

E : Young's modulus of master element's material,
 ν : Poisson's coefficient of master element's material.

References

Examples of application of the contact model by the pinball method are presented in the following papers: [\[268\]](#).

10.43 FLUID-STRUCTURE SLIDING BY “FSS” (JRC)

Object:

The purpose is to define fluid-structure sliding lines of the ALE, Lagrangian or fixed type according to the models developed at JRC Ispra.

These directives are obsolete and are maintained only for compatibility with old input files. Use the ”LINK COUP FSA” or ”LINK COUP FSR” directives instead.

Compatibility: DECO

Syntax:

```
"FSS" | "ALE" . . . |  
      | "LAGR" . . . |  
      | "FIXE" . . . |
```

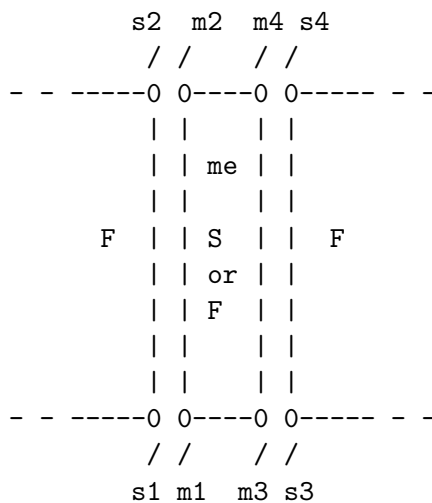
Comments:

These directives use a rather primitive input syntax that obliges the user to use node indexes and often leads to complex and lengthy input data. A simplification of the input structure to allow the use of GIBI objects is foreseen, but not yet available.

FLUID-STRUCTURE SLIDING OF THE ALE TYPE

Object:

Defines fluid-structure sliding lines of the ALE type according to the model developed at JRC Ispra. In this type of sliding, the couples of nodes remain permanently aligned. Thus, there is sliding of the fluid along the structure or with respect to another (master) fluid, but the mesh does not slide. This type of sliding is useful for permanently submerged parts of a structure.



F = fluid element
S = structural element

Note:

Nodes (s1, m1) (s2, m2) (s3, m3) (s4, m4) are coincident in the real geometry.

Master (structural or fluid) nodes are Lagrangian, while slave nodes are treated by the ALE formulation and are constrained to follow the corresponding master nodes.

Compatibility: COUP

Syntax:

```
"ALE"      "NCOT" nasle * ( /LECTURE/ )
            "NPOI" nasln
```

where:

/LECTURE/ = LECT me m1 m2 s1 s2 m3 m4 s3 s4 c1 c2 TERM

nasle

Number of ALE sliding element side couples of the type shown in the above sketch, to be described by the following /LECTURE/.

me

Master element index.

m1, m2

Nodes defining the first master element side.

s1, s2

Nodes defining the first slave element edge.

m3, m4

Nodes defining the second master element side (default is 0 0, i.e. sliding occurs along one side only of the master element).

s3, s4

Nodes defining the first slave element edge (default is 0 0, i.e. sliding occurs along one side only of the master element).

c1, c2

Key to define the type of connection for nodes (s1, m1, m3, s3) and (s2, m2, m4, s4), respectively. Normally these values are both 1, that means ALE sliding. A value of 0 means connection without sliding: this allows to rapidly eliminate a sliding condition, i.e. as if the nodes were rigidly connected, without modifying too much the input. Note that, when a sliding condition is eliminated by posing c1 or c2 equal 0, the corresponding slave node must be declared Lagrangian in the GRILLE directive. Finally a third possibility, indicated by the value -1, is used to model a so-called U-bend ALE sliding. This is useful for situations where a thin structure is permanently submerged in a fluid, in order to model the U-shaped flow around a tip in the structure (represented by the shell element thickness). In this case, the two structural nodes on the tip have different normals (while for 'inner' nodes the normal is unique), so a special treatment is needed.

nasln

Total number of nodes defining each of the (slave or master) ale sliding lines.

Comments:

If a negative value is given for m_1 , m_2 , s_1 , s_2 , m_3 , m_4 , s_3 or s_4 , then the corresponding node is not considered in the ALE sliding process. This feature is useful when modeling e.g. a continuous fluid-structure interface of which one part has a sliding condition of the ALE type, while the rest has a condition of the Lagrangian type. In this case, the element couple at the transition between the two conditions will have one couple of ALE sliding nodes, and the other one Lagrangian. This Lagrangian couple of nodes, say m_2 and s_2 , should have negative indexes.

Finally, note that in this type of sliding the number of nodes in the fluid and in the structure must coincide (the nodes themselves must coincide two by two), so the mesh size is necessarily the same on both sides and it is not possible to use a finer mesh on one of the sides with respect to the other.

FLUID-STRUCTURE SLIDING OF THE LAGRANGIAN TYPE**Object:**

Defines fluid-structure sliding lines of the Lagrangian type according to the model developed at JRC Ispra. In this type of sliding, the couples of nodes do not remain permanently aligned. Thus, there is sliding of the fluid mesh along the structure. This type of sliding is useful when the interface nodes cannot be kept permanently aligned, e.g. near free surfaces. The first side of the sliding line consists of fluid nodes only; the second side may consist either of structural or of (master) fluid nodes.

```

first side  second side
          f2 s2
          /  /
- - ----0 0----0
          | |   |
        fe | | se |
          | |   |
          F | | S |
          | | or |
          | | F |
          | |   |
          | |   |
- - ----0 0----0
          /  /
          f1 s1

F = fluid element
S = structural element

```

Compatibility: COUP

Syntax:

```

"LAGR"    "NCT1" lsle1 * ( /LECTURE1/ )
          "NPOI" lsln1
          "NCT2" lsle2 * ( /LECTURE2/ )
          "NPOI" lsln2

```

where:

```

/LECTURE1/ = LECT fe f1 f2 TERM
/LECTURE2/ = LECT se s1 s2 TERM

```

lsle1

Number of element sides on the first side (slave side) of the Lagrangian sliding line.

fe

Index of the fluid (slave) element.

f1, f2

Indexes of the nodes of the slave edge (first side).

lsln1

Total number of nodes defining the slave edges.

lsle2

Number of element edges on the second side (master side) of the Lagrangian sliding line.

se

Index of the structural (or master fluid) element.

s1, s2

Indexes of the nodes of the master edge (second side).

lsln2

Total number of nodes defining the master edges.

Comments:

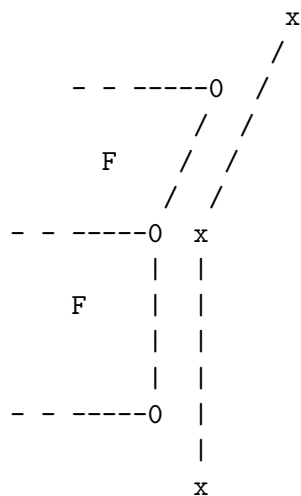
If a negative value is given for f1, f2, s1 or s2, then the corresponding node is not considered in the Lagrangian sliding process. This feature is useful when modeling e.g. a continuous fluid-structure interface of which one part has a sliding condition of the ALE type, while the rest has a condition of the Lagrangian type. In this case, the element couple at the transition between the two conditions will have one couple of ALE sliding nodes, and the other one Lagrangian. The ALE couple of nodes, say m2 and s2, should have negative indexes.

In this type of sliding, the number of nodes on the fluid side may be different from that on the structural side, since the nodes don't have to be aligned in the initial configuration, as it is the case for ALE sliding. It is therefore possible to use meshes of different size for the fluid with respect to the structure.

FLUID-STRUCTURE SLIDING OF THE FIXED TYPE

Object:

Defines fluid-structure sliding lines of the fixed type according to the model developed at JRC Ispra. This type of sliding is sometimes useful to model rigid inviscid boundaries. Nodes belonging to a fixed sliding line are treated as Lagrangian. The fixed boundary is defined via a series of points identified by their coordinates.



F = fluid element
 O = fluid node
 x = fixed point defining fixed sliding line

Compatibility: COUP

Syntax:

```
"FIXE"  "NPOI" n1fsl /LECTURE/
        "NFI"  n2fsl * ( xcoor ycoor )
```

n1fsl

Number of nodes on the fixed sliding line.

n2fsl

Number pf fixed points used to define the fixed boundary.

`xcoor`, `ycoor`

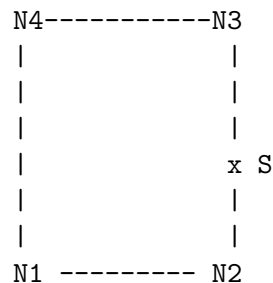
Coordinates of the fixed point.

10.44 NODE TO SHELL CONNECTOR

Object:

This element is used in order to connect node to a master edge of shell. Note that the "SH3D" directive is a subdirective of the "LIAI" directive but it is needed to define an element which defines the nodes (master and slave) of the liaisons. It is listed in this Section because it consists in the definition of kinematic constraints between the dof of one slave node and 2 master nodes.

Compatibility: COUP, LIAI



N1,N2,N3,N4 = nodes shell
 S = Slave node

Syntax:

```
"SH3D OPT 2"
  ( /LECTURE/)
```

/LECTURE/

Reading procedure of the elements

Comments:

Note that in the "GEOM" directive the definition of the element must be in this order : N1 N2 S ie the slave node is defined after the two master nodes.

10.45 WEAK FLUID-STRUCTURE COUPLING 2 (FLSW)

Object:

ATTENTION: this directive is still under development!

This directive allows to specify a “weak” coupling between a fluid and a structure modelled by topologically independent meshes. It is similar to **FLSR** (see page D2.143) but uses a weak approach (direct application of the fluid pressure onto the structure) rather than a strong approach (constraint on velocity imposed by Lagrange multipliers). It is (primarily) intended for use with cell-centered Finite Volumes (CCFV) modeling of the fluid. However, it is also compatible with Finite Element, using then master/slave approach instead of (coupled) Lagrange Multipliers.

The fluid mesh may be either fully general (unstructured) or regular (structured), as specified by the **STFL** directive described on page C.68. In the latter case, the search operations are faster.

Using **STFL** directive with **FLSW** produces by default a cell-centered Finite Volumes regular mesh. To create a regular Finite Element mesh instead and use master/slave approach, add extra **FELM** keyword.

Compatibility: DECO

Syntax:

```
FLSW      STRU /LECTS/
          |[ FLUI /LECTF/ ; STFL <FELM> ]|
          $[ R r          ; GAMM gamm ; PHIS phis ]$
          $[ HGRI hgri ; NMAX nmax ; DELE dele ]$
          <DGRI>
          <VOLU ; FACE>
          <BFLU bflu> <FSCP fscp>
```

/LECTS/

List of structural **elements** concerned. All their nodes must be declared as Lagrangian.

FLUI

The fluid mesh coupled with the structure is fully general (unstructured). The concerned elements are specified next.

/LECTF/

List of fluid **elements** concerned. The fluid mesh is unstructured.

STFL

The fluid mesh coupled with the structure is regular (structured). The concerned **elements** need not be specified. In fact, they are simply the elements generated by the STFL directive described on page C.68, which must in this case have been specified previously in the input file. By default, cell-centered Finite Volumes are created.

FELM

Regular Finite Element mesh is created instead of cell-centered Finite Volume mesh.

R

Prescribed (fixed) radius R of influence spheres at each coupled structural node. In the special, but frequent, case of a uniform structured fluid mesh (uniform square or cube elements) it is suggested to take R slightly larger than the semi-diagonal of a fluid element. This means that, for a 2D uniform square fluid mesh of side L_Φ one should take $R = 0.71L_\Phi$ while for a 3D uniform cube fluid mesh of side L_Φ one should take $R = 0.87L_\Phi$.

GAMM

Coefficient γ for the automatic determination of influence spheres at each coupled structural node, based on the size of the enclosing fluid element. The sphere radius is $R = \gamma R_F = \gamma \delta L_\Phi$ where L_Φ is the local length (size) of the fluid mesh, δ is a coefficient related to the space dimension d of the problem ($\delta = \frac{\sqrt{d}}{2}$, i.e. about 0.71 in 2D and about 0.87 in 3D calculations). The quantity indicated as R_F above is the “natural” size of the sphere radius, i.e. the radius of a sphere (circle in 2D) which exactly encompasses all nodes of a regular element (regular cube in 3D or regular quadrilateral in 2D). By default it is $\gamma = 1.01$. This value should ensure “tightness” of the structure, at least for a regular mesh. By increasing the value, tightness is safer but the amount of fluid “attached” to the structure also increases. By decreasing the value, some local spurious passage of fluid across a solid structure might occur.

PHIS

Coefficient ϕ_s for the automatic determination of influence spheres at each coupled structural node. The sphere radius is equal to ϕ_s times the minimum structural element length at the concerned node. By default it is $\phi_s = 0.3$. This option should be rarely used. It is advisable to use GAMM instead.

HGRI

Specifies the size of the grid cell. Each cell has the same size in all spatial directions and is aligned with the global axes. Note that the size of this grid is related to the size of **structural** elements, not of fluid elements.

NMAX

Specifies the maximum number of cells along one of the global axes.

DELE

Specifies the size of the fast search grid cell as a multiple of the length of the largest coupled **structural** element. Element “diameters” are computed only along each global spatial direction and the maximum is taken. For example, by setting **DELE 2** the size of the cell is two times the length of the largest coupled structural element. By default, i.e. if neither **HGRI**, nor **NMAX**, nor **DELE** are specified, the code takes **DELE 3** (this value is probably too large, a value of 1.1 or so should be more appropriate in most cases).

DGRI

Dump out the initial grid of cells used for fast searching on the listing (only at step 0).

VOLU

For use with fluid Cell Centered Finite Volumes only. The search for fluid elements “contained” in the influence domain of the structure is based upon the element volume, more precisely on the centroid of the element. This is the default.

FACE

For use with fluid Cell Centered Finite Volumes only. The search for fluid elements “contained” in the influence domain of the structure is based directly upon the element “faces” (interfaces between neighboring cells). The centroid of the face is considered rather than the centroid of the finite volume. In this case, there is no difference between using **BFLU 1** or **BFLU 2**, see below. However, please note that by omitting **BFLU** or by specifying **BFLU 0** (the default value for **BFLU**) no numerical fluxes are blocked. So, if **FACE** is used and fluxes must be blocked (as is normally the case), one must specify either **BFLU 1** or **BFLU 2** (with no difference in the results).

BFLU

For use with cell-centered Finite Volumes only. Type of treatment of numerical fluxes (density and energy, but **not** momentum) in fluid models, when used in conjunction with the present **FLSW** directive. The value 0 (default) indicates that fluxes are freely computed. The value 1 indicates that fluxes are blocked between two fluid nodes (or points) which are both within the influence domain of the structure. The value 2 indicates that fluxes are blocked between two fluid nodes (or points) of which at least one lies within the influence domain of the structure. If the **FACE** keyword has been specified (see above), there is no difference between using **BFLU 1** or **BFLU 2**. However, please note that by omitting **BFLU** or by specifying **BFLU 0** (the default value for **BFLU**) no numerical fluxes are blocked. So, if **FACE** is used and fluxes must be blocked (as is normally the case), one must specify either **BFLU 1** or **BFLU 2** (with no difference in the results).

FSCP

For use with cell-centered Finite Volumes only. Type of weak coupling between fluid points and corresponding structural points, when used in conjunction with the present **FLSW** directive. The value 0 (default) indicates that coupling occurs only in the direction normal to the structure. The value 1 indicates that coupling occurs along all spatial directions.

Remarks:

If neither **R** nor **GAMM** nor **PHIS** are specified, the code performs an automatic determination of influence spheres at each coupled structural node by using the default value of **GAMM** ($\gamma = 1.01$).

In case of automatic determination of influence spheres based on the **GAMM** keyword in conjunction with an **unstructured** fluid grid, a fast search over the coupled fluid elements is needed in addition to the normal fast search over the coupled structural elements. Scope of this search is to determine, for each structural node, which is the fluid element currently containing the node. For this purpose, the code uses a fast search algorithm by means of the same parameters (**DGRI**, **HGRI**, **NMAX**, **DELE**) specified above for the search over structural elements. Note, however, that as concerns this second search if **DELE** is specified it refers to the size of the fluid element rather than to the size of the structural element. However, if a **structured** fluid grid is specified, then no additional search is needed.

References

The **FLSR** model (similar to **FLSW** in many aspects) was first described in report [250]. A short description of the model is also given in reference [244].

10.46 NODE ON FACET ELEMENT

Object:

Note that the "MAPi" directive ($i = 2,..7$) is a subdirective of the "LIAI" directive but it is needed to define an element which defines the nodes (master and slave) of the liaisons. It is listed in this Section because it consists in the definition of kinematic constraints between the dof of one slave node and master nodes.

The purpose is to glue one slave node to a master face. It can be used in 2-D (the face is a line) or in 3-D.

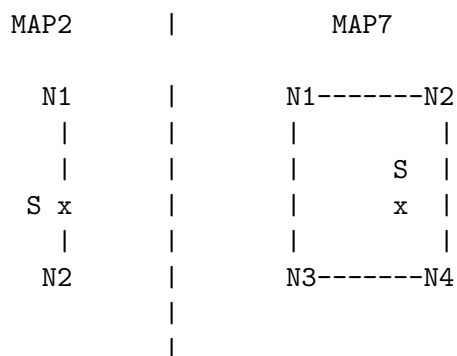
Compatibility: COUP, LIAI

Different cases can be used and are listed below.

Name	Dimension	Npt	Dof	Nb. of liaisons	Remarks
MAP2	2	3	2	2	point on solid line
MAP3	3	4	3	3	point on triangular solid facet
MAP4	3	5	3	3	point on quadrangular solid facet
MAP5	2	3	3	3	point on 2D shell line
MAP6	3	4	6	6	point on triangular shell facet
MAP7	3	4	6	6	point on quadrangular shell facet

Note:

In 3-D the slave node S should be on the face.



N1,N2,N3,N4 = Master nodes
S = Slave node

Syntax:

```
"MAPi "  
  ( /LECTURE/ )
```

```
/LECTURE/
```

Reading procedure of the elements.

Comments:

Note that in the "GEOM" directive the declaration of the element must be in this order : S N1 N2 (N3 N4), ie the slave node is the first node of the list.

10.47 FINITE-ELEMENT/SPECTRAL-ELEMENT INTERFACE

Object:

This directive allows to specify the interface between a Finite Element domain and a Spectral Element domain in a coupled analysis.

It replaces the former principal directive FESE, which is no longer accepted. The difference is that FE/SE interfacing is now coupled with any other (coupled) links specified in the calculation (LINK COUP), while formerly the FE/SE interface conditions were treated as a separate set of conditions.

Compatibility: COUP

Syntax:

```
"FESE" "FNOD" /LECT1/  
      "SNOD" /LECT2/
```

/LECT1/

List of Finite Element nodes along the FE/SE interface.

/LECT2/

List of (micro) Spectral Element nodes along the FE/SE interface.

Remarks:

The model is quite general and accepts the node lists in any order. It is even possible to define interfaces formed by several disjoint lines (or surfaces, in 3D).

The only restriction is that FE and (micro) SE nodes must lie with sufficient precision on the interface, which is defined geometrically by the macro Spectral Element faces.

Furthermore, note that to every macro Spectral Element node on the interface, there must exist one and only one FE node in LECT1 that has the same coordinates. This is necessary in order to ensure that to every FE face on the interface there correspond one and only one opposite macro Spectral Element face (the reverse is not true, in general).

10.48 NAVIER-STOKES (INCOMPRESSIBILITY)

Object:

This directive allows to specify an incompressible or quasi-incompressible behaviour for selected fluid elements. These elements must possess the LIQU material (see page C.390).

It replaces the former NAVI problem type directive (see page A.30) which automatically generated liaison conditions for all elements containing a LIQU material.

Compatibility: COUP

Syntax:

"NAVI" /LECT/

/LECT/

List of Finite Elements concerned. These must possess the LIQU material.

Remarks:

With this directive, it is not allowed to specify the NAVI keyword in the problem type. Use either the old (NAVIER problem type) directive or the present one, but not together in the same run.

For the moment, only elements of type CAR1, TUBE and TUYA are accepted.

Be aware the verification of a link of type NAVI, as activated by the optional keyword VERI of the LINK directive (see page D2.10), makes sense only when the corresponding LIQU material is perfectly incompressible. In fact, when the material is (even slightly) compressible, as indicated by a finite sound speed *C* specified in the material parameters, an extra term is added to the diagonal of the assembled links matrix during the solution process. Therefore, it is normal that the original link specification does not hold any more.

10.49 PIPELINE RUPTURE CONNECTION

Object:

This FSI model allows modelling a break of a pipeline discretized with TUYA elements. Prior to the pipeline rupture instant, the conservation of the internal fluid mass flow rate and the continuity of the mechanical degrees of freedom are ensured.

Compatibility: COUP

Syntax:

```
"BREC"    < "TRUP" trup >    /LECTURE/
```

trup

Rupture instant (no breaking by default).

/LECTURE/

Number or the name of the BREC element.

Comments:

This directive may only be used to connect two TUYA elements.

Outputs:

The components of the ECR table are as follows:

ECR(25): pipeline rupture area (water)

ECR(26): mass flow (water)

ECR(27): total ejected mass (water)

10.50 SURFACE PRESSURE MEASURED IN AN ELEMENT (PELM)

Object:

This directive allows to apply a pressure on structural facets (referred to as slave facets), which is measured in a given fluid element of the model (referred to as master element).

It is typically useful when a cavity is modelled by an equivalent pipe network instead of a full 3D mesh, but the pressure on its structural envelop must still be taken into account. Reference element would then be one of the TUBE or TUYA elements used for the cavity and the facets the structural envelop.

Compatibility: DECO

Syntax:

```
PELM    ( MAIT /LECTURE/  
          ESCL /LECTURE/  
          < |[ INTE ; EXTE ]| /LECTURE/ >  
          < PREF pref > )
```

pref

Reference pressure.

Comments:

Only one master element must be provided for each set of slave facets.

If slave elements are 3D continuum elements, pressure is applied on any of their free facets, along the inward normal direction.

If slave elements are 3D shell elements, keywords INTE or EXTE are used to enter a node defining the internal or external side of the structure respectively and again, pressure is applied along the inward normal direction.

No retroaction occurs from the structure onto the fluid element, which is licit only in the case of a large cavity which imposes its pressure and for limited structural displacements.

11 GROUP E—FUNCTIONS AND INITIAL CONDITIONS

Object :

The directives described in this Section allow to introduce functions in a variety of forms, necessary for the description of materials, loads, or initial conditions of a calculation.

We distinguish functions of the form $y = f(x)$ (directive **FONC**, page E.15), from the functions depending from a parameter p , which are of the form $y = f(x, p)$ (directive **ABAQ**, page E.30).

The abscissa x will most often be the time t in case of a load function, but it is possible to use arbitrary variables for x , y or p .

A special energy injection model (developed at JRC) is also described on page E.38.

11.1 FUNCTIONS

Object :

This directive defines functions in the form $y = f(x)$. These may be used e.g. for imposed motions or other conditions which depend upon time, and also to define material properties.

Syntax :

```
"FONC" ( < "NUM" > ifonc |[ "TABL" npts*(xi,yi)          ;
                             "ROUT" <"PARA" n p1 p2 ... pn>   ;
                             < "LSQU" deg > "TABL" npts*(xi,yi) ;
                             "HARM"  nhar*(C c <TYPE type>
                                           $[ OMEG omeg ; FREQ freq ]$
                                           $[ PHIR phir ; PHID phid ]$
                                           <TMIN tmin> <TMAX tmax>)]| )
```

"NUM"

Optional keyword introducing the number of the function.

ifonc

Number of the function necessary to identify it when it will be used.

"TABL"

The function is defined by a table (sequence of pairs). See below page E.20.

"ROUT"

The function is computed by means of the subroutine TABANA written by the user. See page E.25. Optionally, a list of parameters may be passed to the routine.

"LSQU"

The function is defined by a table (sequence of pairs). See below page E.20. But a least square polynomial fitting is performed in order to store the function as a polynomial. The fitting quality can be controled in the listing by the mean of standard deviation. If the least square polynomial fitting fails, the function is stored as a table and the computation goes on.

deg

Maximum degree ($\text{deg} > 0$) of the polynomial which fits the table function. Be careful not to use too high degrees: it does not give good results because of too many oscillations.

"HARM"

The function is defined as a combination of harmonic functions.

Comments :

The key-word **FONC** may appear at most once, at the beginning of the sequence relative to the functions.

Warning :

If there are imposed displacements, dimension also **FCOE**, see page A.80.

11.1.1 TABLE FUNCTION**Object :**

To define a function $y = f(x)$ by the means of couples of points.

Syntax :

```
"TABL"  npts*( xi , yi )
```

npts

Number of couples (xi, yi) defining the table.

xi , yi

Abscissa (time for example) and value (ordinate) of the function at point i.

Comments :

The value of the function at time t (or at the abscissa x) is determined by a linear interpolation.

11.1.2 SUBROUTINE *TABANA*

Object :

To define a function by the means of a subroutine written by the user. Optionally, a list of parameters may be passed to the routine.

Syntax :

"ROUT" <"PARA" n p1 p2 ... pn>

"PARA"

Optional keyword introducing a list of parameters to be passed to the routine. These parameters will be made available in subroutine *TABANA* by means of the `COMMON /CTABANA/`.

n

Number of parameters to be passed. The maximum is 10.

p1, p2, ... pn

The n parameters to be passed.

Comments :

The user has to write a subroutine which computes the function at every time.

```

SUBROUTINE TABANA(IFONC,T,COEF,DERIV)
*
*   cette routine permet d'entrer une table sous forme analytique
*
*   ifonc      : numero de la fonction
*   t          : temps du calcul en sec. ou + generalement abscisse
*   coef       : valeur de la fonction ifonc au temps ou abscisse t
*   deriv      : derivee de la fonction ifonc en t (seulement pour courbe
*               de traction du materiau lem2)
*
*   attention ! la fonction 2 est utilisee par le benchmark :
*               bm_rob_smr
*
*   USE M_FONCTIONS
*
*   IMPLICIT NONE
*
*---  variables globales :
INTEGER, INTENT(IN) :: IFONC
REAL(8), INTENT(IN) :: T
REAL(8), INTENT(OUT) :: COEF, DERIV
*
*---  variables locales :
REAL(8) :: TO,T1,T2,X,XV,ALPHA,ALPHAV,XF,ALPHAF,TF,Y2
REAL(8) :: F_0, T_BAR, TAU, TT, VV, FAC, V_SUR_F
REAL(8), PARAMETER :: PIGR = 3.14159265359D0
*
REAL(8), EXTERNAL :: FOLC01
*
DERIV=0.D0
*
SELECT CASE (IFONC)

```

```

*
CASE(1) ! 0 AVANT T0, RAMPE DE T0 A T1, 1 APRES T1
T0 = 0D0
T1 = 5D-3
IF(T < T0) THEN
  COEF = 0
ELSEIF(T >= T0 .AND. T <= T1) THEN
  COEF = T/T1
ELSE
  COEF = 1
ENDIF

*
CASE(2) ! CAS DU BENCH
T0 = 0D0
T1 = 0.5D0
T2 = 1.5D0
IF(T <= T0) THEN
  COEF = 0D0
ELSEIF(T > T0 .AND. T <= T1) THEN
  COEF = 673D0*T - 508D0
ELSEIF(T > T1 .AND. T <= T2) THEN
  COEF = 148D0*EXP(-5.5D0*(T-T1)) + 8D0
ELSEIF(T > T2) THEN
  COEF = 240D0
ENDIF

*
CASE (3) ! FC FOR PARTITIONS PAPER: applied sinusoidal force
*
* p_tabana(1) = f_0      max. value of applied sinusoidal force
* p_tabana(2) = t_bar    period of applied sinusoidal force
*
  IF (N_TABANA < 2) THEN
    CALL ERRMSS ('TABANA', 'TOO FEW PARAMETERS ENTERED')
    STOP 'TABANA : N_TABANA < 2'
  ENDIF
  F_0 = P_TABANA(1)
  T_BAR = P_TABANA(2)
  COEF = FOLCO1 (F_0, PIGR, T, T_BAR)

*
CASE (4) ! FC FOR PARTITIONS PAPER: velocity for sinusoidal force
*
* p_tabana(1) = f_0      max. value of applied sinusoidal force
* p_tabana(2) = t_bar    period of applied sinusoidal force
* p_tabana(3) = tau      traversal time of bar length
* p_tabana(4) = v_sur_f  ratio between v and F (=c/SE)
*                        (v = velocity, F = applied force,
*                        c = sound speed, S = bar cross-section,
*                        E = Young's modulus)
*
  IF (N_TABANA < 4) THEN
    CALL ERRMSS ('TABANA', 'TOO FEW PARAMETERS ENTERED')
    STOP 'TABANA : N_TABANA < 4'
  ENDIF
  F_0 = P_TABANA(1)
  T_BAR = P_TABANA(2)
  TAU = P_TABANA(3)
  V_SUR_F = P_TABANA(4)
  COEF = FOLCO1 (F_0, PIGR, T, T_BAR)
  FAC = -2.D0
  TT = T - (TAU+TAU)
  DO WHILE (TT >= 0.D0)
    VV = FOLCO1 (F_0, PIGR, TT, T_BAR)
    COEF = COEF + FAC*VV
    FAC = -FAC
    TT = TT - (TAU+TAU)
  END DO
  COEF = V_SUR_F*COEF

*
CASE (5) ! FC FOR PARTITIONS PAPER: velocity for constant force
*
* p_tabana(1) = f_0      max. value of applied sinusoidal force
* p_tabana(2) = t_bar    period of applied sinus. force (unused here)
* p_tabana(3) = tau      traversal time of bar length
* p_tabana(4) = v_sur_f  ratio between v and F (=c/SE)
*                        (v = velocity, F = applied force,
*                        c = sound speed, S = bar cross-section,
*                        E = Young's modulus)
*
  IF (N_TABANA < 4) THEN
    CALL ERRMSS ('TABANA', 'TOO FEW PARAMETERS ENTERED')
    STOP 'TABANA : N_TABANA < 4'
  ENDIF
  F_0 = P_TABANA(1)
  TAU = P_TABANA(3)
  V_SUR_F = P_TABANA(4)
  COEF = F_0
  FAC = -2.D0
  TT = T - (TAU+TAU)

```

```

      DO WHILE (TT >= 0.D0)
        VV = F_0
        COEF = COEF + FAC*VV
        FAC = -FAC
        TT = TT - (TAU+TAU)
      END DO
      COEF = V_SUR_F*COEF
*
      CASE DEFAULT
        CALL ERRMSS('TABANA',
&      'VOUS AVEZ APPELE LE SS-PROGRAMME TABANA SANS LE CREER ')
        STOP 'TABANA NON ECRIT !'
      END SELECT
*
      END SUBROUTINE TABANA
=====
      REAL(8) FUNCTION FOLC01 (F_0, PIGR, T, T_BAR)
*
      IMPLICIT NONE
*
      REAL(8), INTENT(IN) :: F_0, PIGR, T, T_BAR
*
      FOLC01 = 0.5D0*F_0*(1.D0+SIN(PIGR*((2.D0*T/T_BAR)-0.5D0)))
*
      END FUNCTION FOLC01

```

The arguments have the following meaning:

IFONC : number of the function (input);

T : computing time (input);

COEF : value of the function at time (abscissa) T (output);

DERIV : value of the function derivative at the abscissa T (output). This is mandatory for some materials.

Warning :

It is strongly advised to foresee adequate error messages, like in the above example on the function number.

11.1.3 HARMONIC FUNCTION

Object :

To define a harmonic function of the form (sum of n terms): $y = C_1 \cos(\omega_1 t + \phi_1) + C_2 \cos(\omega_2 t + \phi_2) + \dots$, where for the i -th term C_i is a coefficient, ω_i is the pulsation in rad/s and ϕ_i is the phase in rad. Note, however, that if the user prefers, the frequency f (in Hz) can be specified, in place of ω . Also, the phase can be specified in degrees if so preferred.

Syntax :

```
HARM nhar*( VHAR
              C c <TYPE type>
              $OMEG omeg ; FREQ freq$
              <$PHIR phir ; PHID phid$> )
              <TMIN tmin> <TMAX tmax>
```

nhar

Number of terms in the sum of harmonic functions.

VHAR

Mandatory keyword that introduces the reading of the set of values for the i -th term.

c

Coefficient of the i -th harmonic term.

type

Type of the i -th harmonic term: 1 means sine, 2 means cosine. By default it is 1 (sine).

omeg

Pulsation (angular frequency) ω of the i -th harmonic term in rad/s. Recall that it is $\omega = 2\pi f$ where f is the frequency in Hz.

freq

Frequency f of the i -th harmonic term in Hz. Recall that the pulsation (angular frequency) is then $\omega = 2\pi f$ in rad/s.

phir

Phase ϕ of the i -th harmonic term *in radians*.

phid

Phase ϕ of the i -th harmonic term *in degrees*.

tmin

Time t_{\min} at which the harmonic function (all terms) starts. The function is 0 for $t < t_{\min}$.
By default, the function acts over the entire time scale.

tmax

Time t_{\max} at which the harmonic function (all terms) ends. The function is 0 for $t > t_{\max}$.
By default, the function acts over the entire time scale.

Comments :

The value of the function at time t (or at the abscissa x) is determined by computing the above analytical expression.

If both PHIR and PHID are omitted, a phase of 0 is assumed for the concerned harmonic component.

11.1.4 ABAQUE : PARAMETRISED TABLE FUNCTION

Object :

To define a set of parametrised functions in the form $y = f(x, p)$ where p is a parameter.

Syntax :

```
"ABAQ" ( "SET" ifonc "PARA" flot "TABL" npts*( ti , fi ) )
```

"SET"

Mandatory keyword to describe a parametrised function.

ifonc

Number of the function necessary to identify it when it will be used.

"PARA"

Announces the value of the parameter.

flot

Value of the parameter.

npts

Number of couples (ti, fi) defining the table relative to the parameter flot.

ti,fi

Abscissa and ordinate of the function at point i.

Comments :

The value of the function for an abscissa t is obtained by linear interpolation.

The "PARA" sequence must appear at least twice.

Warning :

Do not forget to dimension sufficiently (directives "FNOM" and "FTAB", page A.90).

11.2 ENERGY INJECTION HISTORY (JRC)

Object

This instruction defines the energy injected in: i) fluid elements of types FLxx (JRC model) or CUBE, PRIS, TETR (CEA model), or ii) multicomponent fluid finite volumes (MCxx elements). This directive is extended to fluid finite volumes (MCxx elements) using the MCVO keyword.

References

More information on the formulation of this model may be found in reference [\[131\]](#).

Syntax

```
"INJE"  "QTAB" ifon  |[ "MASS" ; "VOLU" ;  
                        "MCMA" ; "MCVO" ]|  
/LECT/
```

ifon

Number of the function (see FONC) used to describe the variation in time of the total injected power.

MASS

This keyword applies only in the case of fluid elements, see i) above. The injected power will be distributed among the different elements that form the energy injection zone proportionally to the mass of each element. This is probably the best choice for an energy injection zone bounded by a Lagrangian surface, because in this case the mass of the zone stays constant in time.

VOLU

This keyword applies only in the case of fluid elements, see i) above. The injected power will be distributed among the different elements that form the energy injection zone proportionally to the volume of each element. This is probably the best choice for an energy injection zone bounded by an Eulerian surface, because in this case the volume of the zone stays constant in time.

MCMA

This keyword applies only in the case of fluid finite volumes, see ii) above. Unlike the **MASS** model, in this case the energy injection is associated to one (or more) particular component(s) of the gas mixture. The presence of such component(s) identifies, at each instant, the current injection zone. The injected power will be distributed among the different control volumes (nodes) that form the current energy injection zone, proportionally to the mass of the chosen component(s) at each node.

MCVO

This keyword applies only in the case of fluid finite volumes, see ii) on page E.38. The injected power will be distributed among the different control volumes that form the energy injection zone, proportionally to the volume of each node. This model is similar to **VOLU**.

/LECT/

In the **MASS** or **VOLU** cases, these are the elements defining the injection zone. In the **MCMA** case, these are the selected gas components. Finally, in the **MCVO** case, these are the control volumes (nodes) defining the injection zone.

Comments:

The **INJE** directive can be repeated as many times as necessary. An additional injection zone is defined each time. At present, up to 5 different zones can be defined.

One may define zones associated with finite elements, and other zones associated with finite volumes, in the same run.

Note that the **FONC** of index **ifon** describes the **total** injected power for the zone, as a function of time. This power is distributed at each instant among the various elements that form the injection zone according to either the mass or the volume of the element in relation to the total mass or volume of the zone.

The following global results can be accessed via **TPLOT**, for each injection zone that has been defined:

```
INJVxxxx : Volume of injection zone number xxxx
INJMxxxx : Mass of injection zone number xxxx
INJQxxxx : Power injected in injection zone number xxxx
INJIxxxx : Energy injected in injection zone number xxxx
```

Note that energy injection may also be applied to finite elements (only of **FLxx** types) having the **FLMP** multiphase multicomponent material. In that case, the injected energy is distributed among all fluid components currently present in the element proportionally to their respective relative mass fractions.

EUROPLEXUS offers another (completely distinct) mechanism for prescribing energy (or even mass) generation, namely via the **GENE** and **GENM** parameters of the user's fluid (**FLUT** or **FLMP**) material. The main difference is that in that case, the generation applies to the material rather than to a spatial zone (elements or nodes). Another difference is that in that case the given time function (**FONC**) represents the specific generated energy per unit time, and not the total energy per unit time.

11.3 INITIAL CONDITIONS

Object:

The directives in this section enable the input of initial conditions relative to displacements, velocities, stresses, temperatures etc.. The temperature can be entered directly or under the form of injected energy.

Syntax:

```
"INIT"  
  < "FICH" 'nom_fich' >  
  < "DEPL" . . . >  
  < "VITE" . . . >  
  < "VITC" . . . >  
  < "VFCC" . . . >  
  < "CONT" . . . >  
  < "TETA" . . . >  
  < "TNOD" . . . >  
  < "ENER" . . . >  
  < "GRAD" . . . >  
  < "GRAV" . . . >  
  < "DEBI" . . . >  
  < "ROTA" . . . >  
  < "ALIC" . . . >  
  < "MCOM" . . . >  
  < "CQST" . . . >  
  < "CQDF" . . . >  
  < "MEDL" . . . >  
  < "STAT" . . . >  
  < "DMAS" . . . >  
  < "EQUI" . . . >  
  < "CRAK" . . . >  
  < "ADAP" . . . >
```

Comments:

The key-word "INIT" may appear at most once, at the beginning of the sequence relative to the initial conditions.

11.3.1 AUXILIARY FILE

Object:

This directive allows to read the initial conditions data from an auxiliary file.

Syntax:

```
< "FICHIER"      'nom.fic'  >
```

In certain cases the data may be bulky. It is then recommended to store them on an auxiliary file to shorten the main input data file. The auxiliary file is activated by means of the keyword "FICHIER" that precedes the file name (complete under Unix). In the main data file then only the keywords "INITIAL" "FICHIER" remain.

The auxiliary file (in free format) contains the whole set of initial conditions data, except the keyword "INITIAL". To return to the main input data, the auxiliary file must be terminated by the keyword "RETOUR".

11.3.2 INITIAL DISPLACEMENTS

Object:

The initial conditions concern the nodes of the mesh.

The coordinates of the original mesh are modified.

Syntax:

```
"DEPL" (  icomp  xm  /LECTURE/ )
```

icomp

Number of the component to which the initial condition applies.

xm

Value of the displacement.

LECTURE

List of the numbers of the nodes concerned.

Comments:

An initial displacement enables the modification of the geometry for a few points without using the meshing program again.

If several values must be entered, it is not necessary to repeat the number of the component.

Example :

```
"INIT"  "DEPL"  1  3.1  LECTURE  1 2  TERM
          2  1.5  LECTURE  3   TERM
          "VITE" 1  0.2  LECTURE  5 PAS 1 10 TERM
```

11.3.3 INITIAL VELOCITIES

Object:

These initial conditions are relative to the nodes of the mesh.

- VITE : prescribed initial particle (material) velocity
- VITG : prescribed initial mesh velocity (only available in an ALE calculation)

Syntax:

```
|[ "VITE" ; "VITG" ]|
  ( icomp  vi  /LECTURE/
  ( "RADI"  vr  "CENT" /LECTURE/  "SURF" /LECTURE/ )
  $[ "FILE" ifich ; "LIST" ... ]$
  ( "NOEU" n vx vy <vz> )
```

icomp

Number of the component where the initial condition is located.

vi

Value of the prescribed velocity according to the component icomp.

RADI CENT SURF

Prescribe a radial velocity (see comments below).

vr

Modulus of the prescribed radial velocity (see comments below).

FILE

Velocities are read from a file, as defined next.

ifich

Data-set (logical unit number) or file name (in quotes) of the file from which initial nodal velocities are to be read. The data have to be written on the file according with the following (fixed) format. For 3D calculations there are 4 data per line, i.e. (NODE, VX, VY, VZ), and the format is (I6,3E12.5). For 2D calculations there are 6 data per line, i.e. (NODE, VX, VY, NODE, VX, VY) and the format is 2(I6,2E12.5). Note that **all nodes** of the mesh must be specified in this case, even those having zero initial velocities, because the code reads on from the file until the total number of nodes in the model has been reached.

LIST

Velocities are read from a list, which directly follows the **LIST** keyword in the input file (starting on a new line). The format of this list is exactly the same as for the **FILE** option described above. This syntax allows to embed in the input file data originally contained in a separate file, without any format changes, but it should be avoided for newly written inputs.

n vx vy <vz>

Velocity components of node **n** are **vx vy** (and **vz** in 3D).

LECTURE

List of the numbers of the nodes concerned.

Comments:

If several values must be entered, it is possible to repeat the number of the component or the word "RADI".

If the prescribed initial velocity is a radial one the user just has to put the word "RADI" followed by the velocity modulus, and thereafter the word "CENT", to give the number of the central node and to enter the series of nodes having the same radial velocity, after the word "SURF". EUROPLEXUS then automatically computes the initial components of the velocity in the global coordinate system.

Warning: the positive direction goes from the center to the exterior.

Example :

```
"INIT"  "VITE"   1   -0.2  LECTURE  5 PAS 1 10 TERM
          2   -0.3  LECTURE  5 PAS 1 10 TERM
          "RADI"   1.5  "CENT"  LECTURE  pcen  TERM
                          "SURF"  LECTURE  psur  TERM
```

11.3.4 INITIAL VELOCITIES FOR CELL-CENTRED FINITE VOLUMES

Object:

These initial conditions are relative to the cell centers of a mesh composed of Cell-Centred Finite Volumes.

Syntax:

```
("VITC" "VITX" vx "VITY" vy <"VITZ" vz> /LECTURE/)
```

vx

Initial velocity along x-axis.

vy

Initial velocity along y-axis.

vz

Initial velocity along z-axis (3D only).

LECTURE

List of the numbers of the elements concerned.

11.3.5 INITIAL CONDITIONS FOR CELL-CENTRED FINITE VOLUMES

Object:

These initial conditions are relative to the cell centers of a mesh composed of Cell-Centred Finite Volumes. For the moment, we can only impose these initial conditions in GAZP and CDEM materials. These new imposed initial conditions override the ones imposed when applying the material to the concerned elements.

Syntax for perfect gas (GAZP) material:

```
("VFCC" "VITX" vx "VITY" vy <"VITZ" vz> "PINI" pini "RHO" rho /LECTURE/)
```

vx

Initial velocity along x-axis.

vy

Initial velocity along y-axis.

vz

Initial velocity along z-axis (3D only).

pini

Initial pressure.

rho

Initial density.

LECTURE

List of the numbers of the elements concerned.

Syntax for the CDEM material:

```
("VFCC" "VITX" vx "VITY" vy <"VITZ" vz> "PINI" pini "TINI" tini  
"KSI0" ksi0 "K0" k0 "Y1" y1 "Y2" y2 "Y3" y3 ... /LECTURE/)
```

See the CDEM material for the meaning of each term.

11.3.6 INITIAL STRESSES

Object:

To prescribe initial stresses to different elements.

Syntax:

```
"CONT" (  comp  sig /LECTURE/ )
```

comp

Number of the component to which the initial condition applies.

sig

Value of the initial stress.

LECTURE

List of the numbers of the elements concerned.

Comments:

The components of the stress tensor are stored in a one-dimensional array. The number of the components depends on the element. See the description of the elements available page INT.80 .

The initial stresses are provided at the integration points of the element.

If there are several integration points in one element, an initial value is given to the component comp of all the points concerned.

If the initial stresses are the result of a static load, it may be more adequate to use EUROPLEXUS for the necessary computations, by using a quasi static damping (see the instruction "OPTION"). A restart after that first computation, by changing the load, will provide the desired result.

Be careful and respect the writing conventions for the stress tensor of each element (see page G.20).

11.3.7 INITIAL TEMPERATURES

Object:

To prescribe different temperatures on certain elements. The values are either directly read from the input file, or as the results of a DELFINE file.

Syntax:

Direct reading:

```
"TETA"    (  ti  /LECTURE/  )
```

Reading from a DELFINE file:

```
"TETA"  "DELFINE" ndelfine  "TDELFINE"  tdelfine
```

ti

Initial temperature.

LECTURE

List of the numbers of the elements concerned.

ndelfine

Logical unit number of the DELFINE file.

tdelfine

Time written on file DELFINE. The field of temperatures at that instant is considered as the initial temperature field for the EUROPLEXUS computation.

Comments:

The definition of a temperature is compulsory for a isotropic Von Mises material dependant on temperature (VMIS TETA) as well as for the material PUFF. By default it is supposed to be equal to zero.

If the user wants to impose to the material VMIS TETA a field of initial stresses due to heating, he has to use the instruction "ENERGIE" (see page E.90).

11.3.8 NODAL TEMPERATURES FOR ADVECTION-DIFFUSION (JRC)

Object:

To prescribe initial nodal temperatures for advection-diffusion problems. These can optionally be read from a separate data-set.

Syntax:

```
"TNOD"  |[ ( tnod  /LECTURE/) ;  
          "FILE"  ifich      ;  
          $ "LIST"  ...       $
```

tnod

Initial temperature.

/LECTURE/

List of concerned nodes.

FILE

Temperatures are read from a file, as defined next.

ifich

Unit number of data-set containing nodal temperatures in the form (node index, T). Values are read using format (4(i6,e12.5)).

LIST

Temperatures are read from a list, which directly follows the **LIST** keyword in the input file (starting on a new line). The format of this list is exactly the same as for the **FILE** option described above. This syntax allows to embed in the input file data originally contained in a separate file, without any format changes, but it should be avoided for newly written inputs.

11.3.9 INITIAL ENERGY SUPPLY

Object:

The directive is used to input the temperature field and the field of the corresponding initial stresses generated by a supply of external energy.

The instruction can be used only in connection with the material VMIS TETA (Von Mises isotrope dependant on temperature).

Syntax:

```
"ENERGIE" ( "CV" cv "ALPHA" alpha "DEPOSE" wd /LECTURE/ )
```

cv

Specific heat.

alpha

Coefficient of linear expansion.

wd

Supplied energy.

LECTURE

List of the numbers of the elements concerned.

Comments:

The definition of a temperature is compulsory for an isotropic Von Mises material depending on temperature. By default it is supposed to be equal to zero.

For this material, the phenomenon is considered isothermal and the temperature remains constant during the computation.

A supplied energy result in an initial stress sig0:

$$\text{sig0} = E * \alpha * (wd / cv)$$

Here E represents Young's modulus. Obviously, the temperature increment relative to the supplied energy is taken into account:

$$DT = wd / cv.$$

11.3.10 INITIAL BENDING STRESSES DUE TO TEMPERATURE GRADIENT

Object:

The instruction inputs a field of initial bending stresses, generated by a temperature gradient through the thickness of shell elements.

Syntax:

```
"GRADIENT"  gradt      ...
... (  $[  "ALPH"  alpha      ;
        "ALP1"  alp1      "ALP2"  alp2  ]$ /LECTURE/  )
```

gradt

Temperature gradient (along the normal to the element).

alpha

Coefficient of linear expansion (isotropic materials).

alp1,alp2

Coefficients of linear expansion (orthotropic material) in the orthotropy coordinte system relative to the element CMC3.

LECTURE

List of the numbers of the elements concerned.

Comments:

The orientation the normal depends on the numbering of the nodes of the element (see Maxwell's cork-screw rule).

This option concerns only the following shell elements: COQUE, COQ3, COQ4, CMC3, and the following materials: LINEAIRE, VMIS ISOTROPE, VMIS TETA, ORTHOTROPE.

For isotropic materials, the initial stress sigf0 is :

$$\text{sigf0} = - \alpha * \text{gradt} * \text{epaisseur} * \text{young} / (1-\nu)$$

11.3.11 GRAVITY LOADING

Object:

This directive allows to introduce an initial stress field due to gravity (or to hydrostatic pressure, for the fluids) that equilibrates the body weight at the initial time. In the case of pipelines (elements "TUBE" or "TUYA"), it is preferable to use directive "INIT" "DEBI" (page E.120).

Syntax:

```
"GRAVITE"      "PTSL" xcoor ycoor < zcoor >      "PSL" psl
                "G"      gx      gy      < gz >    < "STRUC" >
                "COUCH"  ncouch*( "RO" rho    "H" h )
                ...      /LECTURE/
```

xcoor, ycoor, zcoor

Coordinates of the point defining the free surface supposed normal to the vector defining the body weight.

psl

Initial pressure at the free surface.

gx gy gz

Components of the weight.

STRUC

Indicates that the concerned elements are associated to a structural material (the stress field is computed in the global reference frame).

ncouch

Number of layers composed by different materials.

rho

Density of the layer.

h

Height of the layer.

LECTURE

Numbers of the elements subjected to gravity.

psf					
	v	v	v	v	v
	-----			 free surface
		layer no. 1 (rho1)			h
G					1
		-----		V
v		layer no. 2 (rho2)			h
		-----		V 2
		layer no. 3 (rho3)			h
					3
		-----		V
.	.			.	.
.	.			.	.

Comments:

The component $\langle gz \rangle$ of the acceleration is only used in a 3D calculation.

The pressure is associated to the elements' Gauss points.

The gravity may only be used with continuum elements; the elements of type shell, beam, bar are excluded.

In the case of fluid materials gravity may only be taken into account with the following material laws:

number	name	fluid material law
-----	-----	-----
7	FLUI	isothermal fluid (c = cte)
8	CAVI	liquid with cavitation
21	ECOU	isothermal fluid flow
10	NAH2	sodium-water reaction
34	ADCR	confinement accident (fast neutrons)
53	ADCJ	confinement accident with JWL gas
22	EAU	two-phase water (liquid + vapour)
-----	-----	-----

Warning:

In order to obtain a state of equilibrium at $t = 0$, it is necessary to use the directive CHARGE-MENT CONSTANT GRAVITE (see page F.30). The stresses computed will then be equilibrated by the forces applied to the nodes of the structure defined by the directive "CHARG CONST GRAV".

For the fluid materials for which the density is re-computed, it is necessary to do a first calculation of just one step (for example) so as to obtain the correct values of the density RHO.

11.3.12 PRESCRIBED CONSTANT FLOW RATE

Object:

This directive allows to specify a field of initial pressures in the branch of a pipeline, which equilibrates at the initial time instant the forces due to head losses, so as to obtain a constant mass flow rate. The hydrostatic pressure is also accounted for, in the case that a gravity load is also prescribed.

This directive may be used only in 1D: elements "TUBE" or "TUYA".

Syntax:

```
"DEBIT"
      "LIGN" "PENT" pent    "DEBI" dmas    < "GRAV"  gx gy gz >
      "ORIG"                /LECTURE/
      "LIST"                /LECTURE/
```

LIGN

Keyword introducing the reading of the data relative to a pipeline branch.

pent

Inlet pressure of the pipeline branch.

dmas

Imposed mass flow rate in the pipeline branch.

gx,gy,gz

Gravity components.

ORIG

Keyword announcing the reading of the node number at the inlet of the pipeline branch.

LIST

Keyword announcing the reading of the element numbers of the pipeline branch.

Comments:

In the case of a complex pipeline, it is mandatory to decompose it in several elementary branches, for each of which the directive "LIGN" "PENT" ... will be repeated.

The pressure is applied at the element centers.

The velocities in each node of the branch will be computed by EUROPLEXUS, as a function of the mass flow rate (dmas) and of the local diameters.

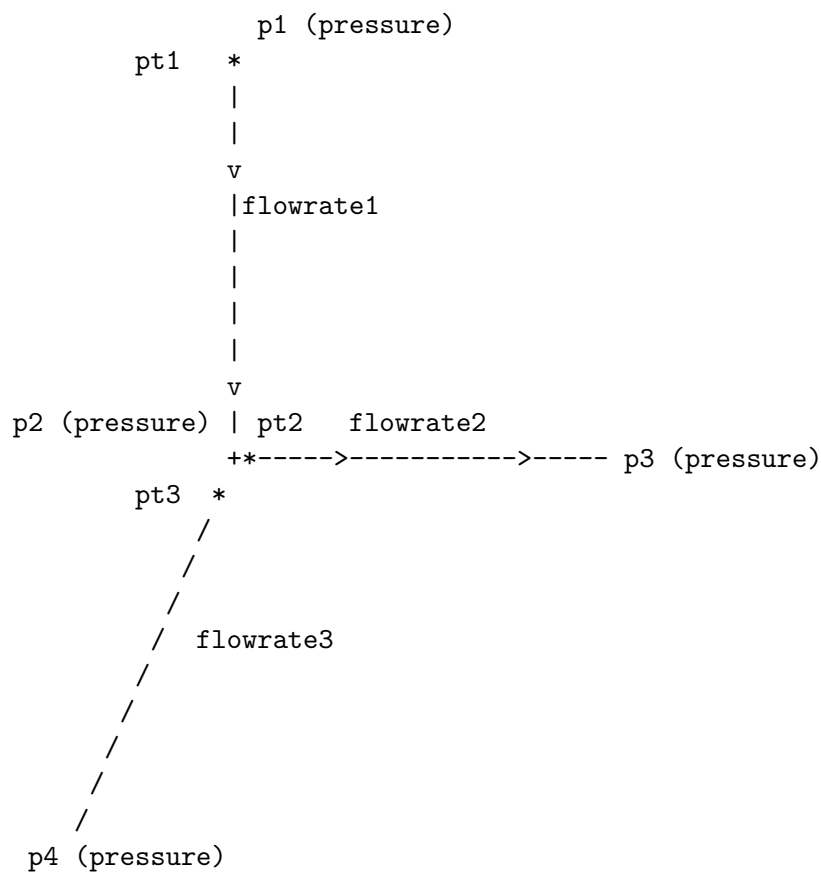
If there is a gravity loading, do not forget to prescribe also "CHARGE" "CONST" "GRAVITE" (page F.30) for the rest of the calculation. In fact, the directive "INIT" "DEBI" computes only the initial status.

There is just one d.o.f. for the elements of type "TUBE", therefore only the weight component along the axis is taken into account for the hydrostatic pressure.

This directive may be used only for the pipelines, i.e. with elements "TUBE", "TUYA", "CAVI", "BIFU", "CL1D" and "CLTU".

It is currently possible to account for the initial mass flow rate only for the following materials:

number	name	fluid material law
7	FLUI	isothermal fluid (c = cte)
8	CAVI	liquid with cavitation
21	ECOU	isothermal fluid flow
10	NAH2	sodium-water reaction
22	EAU	two-phase water (liquid + vapour)

Example:

In this example the directive will become:

"INITIAL"

"DEBIT"

"LIGN"	"PENT"	p1	"DEBI"	debit1	
	"ORIG"		"LECT"	pt1	"TERM"
	"LIST"		"LECT"	ligne1	"TERM"
"LIGN"	"PENT"	p2	"DEBI"	debit2	
	"ORIG"		"LECT"	pt2	"TERM"
	"LIST"		"LECT"	ligne2	"TERM"
"LIGN"	"PENT"	p3	"DEBI"	debit3	
	"ORIG"		"LECT"	pt3	"TERM"
	"LIST"		"LECT"	ligne3	"TERM"

Remarks:

In the preceding example, if one just knows the inlet pressure and the mass flow rate in the branch 'ligne1', the following procedure may be followed to obtain the equilibrium status:

- A first EUROPLEXUS calculation is performed for just 1 time step, by taking as initial pressure the inlet pressure p_1 for all the pipeline branches with the desired mass flow rates (flowrate1, flowrate2 and flowrate3).

- On the EUROPLEXUS listing, the head losses are printed for each branch of the pipeline. By knowing the head losses in the branch 'ligne1', the outlet pressure for this branch may be deduced: $p = p_1 - p_{cha1}$ which corresponds to the inlet pressures for the branches 'ligne2' and 'ligne3' of our example.

- For the real calculation, the initial conditions to be assumed are then an initial pressure p_1 for the branch 'ligne1' and an initial pressure $p_2 = p_1 - p_{cha1}$ for the branches 'ligne2' and 'ligne3'. The output pressures of branches 'ligne2' and 'ligne3' are respectively equal to $p_3 = p_2 - p_{cha2} = p_1 - p_{cha1} - p_{cha2}$ and $p_4 = p_2 - p_{cha3} = p_1 - p_{cha1} - p_{cha3}$; where p_{cha2} and p_{cha3} indicate the head losses in the branches 'ligne2' and 'ligne3'.

- In order to obtain an even better precision, one must take care that the inlet pressure of the branch and the initial pressure of the fluid elements in the branch be coherent. This implies using a "MATERIAU" directive per branch, with the same elements as for the initial mass flow rate.

11.3.13 INITIAL CONDITIONS FOR ROTATING STRUCTURES

Object:

This directive allows to initialise a velocity field (and a field of initial rotations, if adequate) for a rotating structure.

Syntax:

```
"ROTATION"
    "ORIGINE"  xo yo < zo >          < "VECTEUR"  v1 v2 v3 >
    "OMEGA"    omega
    < "VAXE"    vaxe >

    ...                               /LECTURE/
```

xo,yo,zo

Coordinates of the origin point (z0 is redundant in 3D).

v1,v2,v3

Components of the vector defining the rotation axis. These data are not necessary in 2D (see comments below).

omega

Rotation velocity in rad/s.

vaxe

Axial velocity. If this keyword is omitted, the axial velocity will be zero. This keyword is only available in 3D.

LECTURE

Numbers of the nodes belonging to the rotating structure.

Comments:

This directive initializes, for each node defined by the preceding /LECTURE/, a velocity field equal to:

$$V_0 = \omega \wedge OM + v_{axe} * v_{axe}$$

where:

v_{axe} : unit vector collinear with the rotation axis.

When the concerned node also has rotation degrees of freedom, the initial rotation velocities are initialised as well.

In 2D, it is supposed that the rotation axis is normal to the plane.

It is also necessary to define the initial field of the stresses generated by the centrifuge forces, else the structure will not satisfy the equations of motion at the initial time.

11.3.14 INITIALISATIONS FROM A PREVIOUS ALICE FILE

Object:

This directive allows to initialise the geometry, the stress field and the field of hardening parameters starting from the ALICE file of a previous EUROPLEXUS calculation.

Syntax:

```
"ALICE"  ndfic
          < "MFRO"      omega   >
          < "ECRO"      >
          < "TEMPS"     temps   >
          < "NPAS"      npas    >
          < "POINT"     noe1 noe2 >
          < "ELEMENT"   iel1 iel2 >
```

ndfic

Logical unit number of the preliminary ALICE file.

omega

Activate “mise a froid” for the structure according to the **omega** parameter.

ECRO

Initialize the ECR array. By default, ECR is not initialized: one makes a preliminary calculation avec a very large elastic limit, which yields an elastic solution. In the following calculation, the true material curve is used (one starts with zero plastic deformation).

temps, npas

Time or step number of the ALICE file starting from which one reads the fields of displacement, stress and hardening parameters necessary for the initialisation of the present calculation.

noe1, noe2

The initialisation will only occur for the nodes between **noe1** and **noe2**.

iel1, iel2

The initialisation will only occur for the elements between **iel1** and **iel2**.

Comments:

The meshes of the current calculation and of the preliminary one may be different. But it is **MANDATORY** that all nodes and elements of the preliminary calculation **MAINTAIN THE SAME NUMBERS** in the present one. The present mesh must therefore be a superset of the previous one.

When the directives "TEMPS" or "NPAS" are not given, EUROPLEXUS initialises starting from the last step of the preliminary calculation.

When the directive "POINT" (respectively "ELEM") is not present, EUROPLEXUS initialises for all points (respectively all elements).

11.3.15 INITIAL STATUS OF MULTICOMPONENT FLOW (JRC)**Object:**

The instruction defines initial conditions for multicomponent fluid flows.

Syntax:

```

"MCOM"  "TEMP"          $ ( val /LECT/ )      $
                        $ "CHAM" /LCHP/ /LECT/  $

        "PRES"          $ ( val /LECT/ )      $
                        $ "CHAM" /LCHP/ /LECT/  $

        "VEL1"          $ ( val /LECT/ )      $
                        $ "CHAM" /LCHP/ /LECT/  $

        "VEL2"          $ ( val /LECT/ )      $
                        $ "CHAM" /LCHP/ /LECT/  $

        "VEL3"          $ ( val /LECT/ )      $
                        $ "CHAM" /LCHP/ /LECT/  $

                        (      "COMP" 'namecomp'
                          "MFRA"  $ ( val /LECT/ )      $
                                $ "CHAM" /LCHP/ /LECT/  $ )

```

TEMP

Introduces the value(s) of temperature.

PRES

Introduces the value(s) of pressure.

VEL1

Introduces the value(s) of x -velocity.

VEL2

Introduces the value(s) of y -velocity.

VEL3

Introduces the value(s) of z -velocity.

COMP

Introduces the component (identified by its name **namecomp**). The name must be spelled exactly as in the declaration of the MCGP material, up to 8 characters.

MFRA

Introduces the value(s) of mass fraction.

val

Value.

LCHP

Values in the form of a CASTEM2000 ‘champoint’.

LECTURE

List of the numbers of the nodes concerned.

Remarks:

Normally, the **COMP** . . . **MFRA** directive should be repeated at least **ncom** times, where **ncom** is the total number of components of **THE** multicomponent material of type **MCGP** declared in the **MATE** directive. Note that only one material of type multicomponent (**MCGP**) is allowed within a model.

If values are not specified for a certain zone, they will be set to 0.

11.3.16 INITIAL TENSOR OF STRESS**Object:**

To prescribe initial tensor of stress to different elements.

Syntax:

```
"CQST" ( igauss sig /LECTURE/ )
```

igauss

Number of the gauss point.

sig

Values of the initial components of stress tensor.

LECTURE

List of the numbers of the elements concerned.

Comments:

The components of the stress tensor are stored in a one-dimensional array. The number of the components depends on the element. See the description of the elements available page INT.80 .

The initial stresses are provided at the integration points of the element. Bending stresses can then be taken into account.

Be careful and respect the writing conventions for the stress tensor of each element (see page G.20).

The table gives the position of the integration points for the Q4GS shell element.

N. Point Gauss	XSI		ETA	ZETA

1	-1		-1	-0.9..

	2		1		-1		-0.9..	
	3		1		1		-0.9..	
	4		-1		1		-0.9..	
	5		-1		-1		-0.5..	
	6		1		-1		-0.5..	
	7		1		1		-0.5..	
	8		-1		1		-0.5..	
	9		-1		-1		0	
	10		1		-1		0	
	11		1		1		0	
	12		-1		1		0	
	13		-1		-1		0.5..	
	14		1		-1		0.5..	
	15		1		1		0.5..	
	16		-1		1		0.5..	
	17		-1		-1		0.9..	
	18		1		-1		0.9..	
	19		1		1		0.9..	
	20		-1		1		0.9..	

For a shell element and a non linear material (such as von Mises material or Hyperelastic material), the stress tensor has 8 components : 3 components of membrane-bending (xx, yy, xy), 3 components that are zero and 2 components of shear (yz, xz).

11.3.17 INITIAL TENSOR OF STRAIN**Object:**

To prescribe initial tensor of strain to different elements.

Syntax:

```
"CQDF" ( igauss strain /LECTURE/ )
```

igauss

Number of the gauss point.

strain

Values of the initial components of strain tensor.

LECTURE

List of the numbers of the elements concerned.

Comments:

The components of the strain tensor are stored in a one-dimensional array. The number of the components depends on the element. See the description of the elements available page INT.80 .

The initial strains are provided at the integration points of the element. Flexural strains can then be taken into account.

Be careful and respect the writing conventions for the strain tensor of each element (see page G.20).

The table gives the position of the integration points for the Q4GS shell element.

	N.	Point	Gauss		XSI	

		1			-1	
		2			1	

	3		1		1		-0.9..	
	4		-1		1		-0.9..	
	5		-1		-1		-0.5..	
	6		1		-1		-0.5..	
	7		1		1		-0.5..	
	8		-1		1		-0.5..	
	9		-1		-1		0	
	10		1		-1		0	
	11		1		1		0	
	12		-1		1		0	
	13		-1		-1		0.5..	
	14		1		-1		0.5..	
	15		1		1		0.5..	
	16		-1		1		0.5..	
	17		-1		-1		0.9..	
	18		1		-1		0.9..	
	19		1		1		0.9..	
	20		-1		1		0.9..	

For a shell element and a non linear material (such as von Mises material or Hyperelastic material), the strain tensor has 8 components : 3 membrane components (xx, yy, xy), 3 bending components (xx, yy, xy) and 2 components of shear (yz, xz).

11.3.18 INITIALISATIONS FROM A MED FILE

Object:

This directive allows to read displacements, stresses and internal variables from a MED file. The coordinates of the original mesh are modified. If reading of stresses is not specified, EUROPLEXUS calculates the initial stress field and the initial internal forces from the displacements.

Syntax:

```
"MEDL"  
    < $[ "CONT" ;  
        "CONT" "ECRO" ]$ >  
    < "EQUI" >  
    < "NITE"      niter    >
```

niter

Number of iterations to calculate the initial conditions. Default value : NITE = 1

Comments:

The name of the displacement fields stored in the MED file must begin by 'DEPL'. Field names containing 'DEPL' from the 9th to 13th character are also accepted (output names from EDF software Code_Aster).

The directive "CONT" allows reading both displacements and corresponding stresses stored in the MED file, which is produced by the Code_Aster quasi-static analysis. Only BR3D, T3GS, Q4GS, TETR and CUB8 elements are accepted. When using "CONT" option, no iterations are needed because stresses are not calculated from displacements.

When "ECRO" is added to "CONT", internal variables are also read from the MED file.

When the directive "EQUI" is given, the initial state is in equilibrium (the initial accelerations are equal to zero), and EUROPLEXUS computes the initial external forces that balance the initial displacements.

Be careful when using this option. When the finite element formulations between EUROPLEXUS and the code from which the displacements come are different, some very important initial external forces can be introduced to balance the displacements.

The directive "NITE" allows to initialize a non-linear initial status. At least one material has a non-linear behaviour or the initial displacements, rotations are large.

11.3.19 INITIALISATIONS FROM A STATIC ANALYSIS

Object:

This directive allows to read results from a former static analysis performed to compute the initial state of the current model. Results are entered by means of displacement fields. Several fields can be given if the static analysis is non-linear, so that the non-linear loading path can be correctly followed. For each field, the corresponding displacement can be applied within a certain number of iterations.

The coordinates of the original mesh are modified.

From displacements EUROPLEXUS calculates the initial stress field and the initial internal forces.

Syntax:

```
"STAT"  ifich
```

`ifich`

Number of the logical unit of the formatted file or file name in quotes.

Comments:

In order for the initial state to be in equilibrium, keyword (INIT) EQUI must also be used (e.g. INIT STAT ifich EQUI).

The formatted file is structured as follows:

On the first line, with the format ('NSTP ',I10,' NBNO ',I10), are given the number of displacement fields *nstep* and the number of nodes *nbnodes* for all fields.

The file then contains *nstep* blocks, each block corresponding to one displacement field.

On the first line of each block, with the format ('NCYC ',I10) is given the number of iterations used to apply the corresponding displacement field.

Then, *nbnodes* lines follow, each containing field values for one node of the mesh.

Node line structure is the following:

Format: (I10,1X,I2,*ncomp*(1X,A2,1X,E12.5))

1st integer: number of the node in the EUROPLEXUS model

2nd integer: *ncomp*=number of dofs to be read

For each dof:

Character: Name of the dof:

 'UX','UY','RZ' for 2D problems

 'UR','UZ','RT' for axisymmetrical problems

 'UX','UY','UZ','RX','RY','RZ' for 3D problems

Real: Value of the displacement field for this dof

11.3.20 PRESCRIBED MASS FLOW RATE**Object:**

This directive allows to specify a prescribed mass flow rate in a pipeline.

This directive may be used only in 1D.

Syntax:

```
"DMAS" dmas /LECTURE/
```

dmas

Value of the imposed mass flow rate.

/LECT/

List of node numbers.

11.3.21 PRESCRIBED INITIAL EQUILIBRIUM

Object:

This directive allows to impose initial equilibrium conditions over the whole mesh or in a region specified by the user.

The code computes external applied forces which exactly equilibrate the **internal** forces (e.g. resulting from an initial stress state) in the initial configuration and adds these (constant) force values to the other prescribed external forces at every time step during the transient computation.

In other words, at step 0 the code computes the **equilibrating** forces:

$$F_{\text{eq}} = F_{\text{int}}$$

Then, at every time step (including step 0) these equilibrating forces are added to the other imposed **external** forces:

$$F_{\text{ext}} = F_{\text{ext}} + F_{\text{eq}}$$

Optionally, by specifying the keyword **FTOT**, the above equilibration procedure is applied to the **total** forces (i.e., to the difference between internal and external forces) rather than to the internal forces alone. In this case, at step 0:

$$F_{\text{eq}} = F_{\text{int}} - F_{\text{ext}}$$

and then:

$$F_{\text{ext}} = F_{\text{ext}} + F_{\text{eq}}$$

Syntax:

```
"EQUI" $[ "FINT" ; "FTOT" ]$ </LECTURE/>
```

```
/FINT/
```

Equilibrium takes into account only initial internal forces: this is the default behaviour.

```
/FTOT/
```

Equilibrium takes into account both initial internal forces and initial external forces.

```
/LECT/
```

List of node numbers for which initial equilibrium is prescribed. By default (no /LECT/ specified) all nodes in the mesh are taken.

Comments:

The use of /LECT/ to specify the zone subjected to initial equilibrium may be useful in some special cases. For example, assume a stratified (layered) soil subjected to an internal explosion (gas bubble). The soil has initial (hydrostatic-like) stresses due to soil weight, and must be initially equilibrated. However, the fluid (explosive bubble) should not be equilibrated. In this case the /LECT/ would specify just the soil sub-domain (or just its envelope, in case e.g. gravity load is applied to the whole problem).

The use of the FTOT optional keyword may be useful in case e.g. there are some “external” forces that should be equilibrated as well in the initial configuration. For example, fluid-structure interaction forces.

As indicated by the syntax, several zones of imposed equilibrium can be declared, by choosing in each zone the appropriate type of equilibrium (internal or total). For example:

```
EQUI FTOT LECT zone1 TERM
      FINT LECT zone2 TERM
```

Note that the initialization of the equilibrating forces is done at the initial step of a calculation containing the present EQUI directive. In practice, in most cases, this means at step 0 of the calculation.

However, this feature may be exploited, in conjunction with the use of restart, in order to start the equilibration of forces at a time different from the initial time.

For example: perform a first run with saving for restart and without the EQUI directive. In this way, no equilibration takes place at the initial time. Then, restart the calculation by specifying the EQUI directive: the equilibrating forces are computed at the restart time and are applied thereafter (even in case of successive restarts).

1) first run (without EQUI but with QUAS STAT and some blockages):

```
. . .
LINK COUP BLOQ 12 LECT absr TERM
ECRI ... FICH SAUV LAST
OPTI QUAS STAT 1. 105. UPTO 0.E0
CALC TINI -50.E-3 TEND 0.D0
. . .
```

2) restart run (without QUAS STAT but with EQUI, blockages are removed):

```
. . .
REPR 'xxxx.sau' POSI 1
```

```
LINK COUP ! empty directive to remove the previous links
EQUI FTOT LECT zone1 TERM
      FINT LECT zone2 TERM
. . .
```

11.3.22 PRESCRIBED INITIAL CRACK IN SPHC MODEL**Object:**

This directive allows to define one or several initial crack(s) in a SPHC model (SPH approach for shells - see Page C.93n Section [8.1.27](#)).

Each crack is defined by a set of segments between given stress-points. Every stress-point located on the path of a crack is supposed to have failed at initial time.

Syntax:

```
"CRAK" ncrk * ( /LECTURE/ )
```

`ncrk`

Number of initial cracks.

`/LECTURE/`

Ordered list of stress-points defining the segments of a crack.

11.3.23 INITIAL CONDITIONS FOR ADAPTIVITY

Object:

This directive allows to impose initial conditions for adaptive calculations, namely mesh refinement operations to be performed in the initialization phase of a calculation (see sub-directives **SPLI** and **ROUT**). These operations are performed starting from the *base* mesh, i.e. the mesh provided in input by means of the **GEOM** directive.

Furthermore, it is possible to fine-tune the initial state (i.e. at step 0) of the material in an adapted mesh by means of the sub-directive **IMAT**.

Syntax:

```
ADAP ( SPLI LEVE leve /LECTURE/ )  
      ROUT CASE ncas  
      IMAT nimat*(MATE mat OBJE /LECT/ $ INSI ; OUTS $  
                  $ SURF ; VOLU $ /LECT/)
```

SPLI

Split the specified object to the specified level of refinement. Recall that the base mesh has level 1. The first splitting (level 2) halves the elements size, the second splitting (level 3) further halves the mesh size (1/4) and so on. Note that splitting can be applied both to base elements and to descendent elements. The splitting operations can be defined in any order, but they are applied in the natural order, i.e. by growing element index.

LEVE

Specifies the level **leve** of the split operation.

/LECT/

List of element to be split to the specified level.

ROUT

Initializations are performed by a user-written routine `INIT_ADAP.ROUT` *after* any split and unsplit operations specified by this same directive.

CASE

Introduces the case number **ncas**. Different cases correspond to different initializations which are programmed in the `INIT_ADAP.ROUT` user subroutine.

IMAT

Initialize (at step 0) the material properties (i.e. the initial physical state) of the material in an adapted mesh. Initializations are performed immediately *after* the split operations done, at step 0, by the **WAVE** directive.

nimat

Number of different **IMAT** zones, to be defined next.

MATE mat

Index of the material (as declared in the **MATE** directive) to be assigned to the elements defined in the following.

OBJE

The elements in the following **/LECT/** are the candidates for the assignement of the initial material properties (subjected to the following conditions).

INSI

The material **mat** has to be assigned to the elements inside the following defined surface or volume.

OUTS

The material **mat** has to be assigned to the elements outside the following defined surface or volume.

SURF

The (surfacic) elements in the following **/LECT/** define a closed, simply connected surface. All elements belonging to the surface must be consistently oriented.

VOLU

The (volumetric) elements in the following **/LECT/** define a volume.

Comments:

As indicated by the brackets, the **SPLI** sub-directive can be repeated as many times as necessary to specify all the parts of the mesh to be initially split.

If smooth refinement of the mesh is desired, in addition to the splitting specified, use can be made of the **OPTI ADAP RCON** optional directive.

Note the difference between prescribing, say, initial velocities by means of the **INIT VITE** directive or by means of the **INIT ADAP ROUT** directive. The **INIT VITE** directive can only

prescribe velocity values for the *base* mesh nodes. If the mesh is then split by means of the `INIT ADAP SPLI` directive, the velocity values in the descendent nodes are *linearly* interpolated starting from those at the base nodes. On the contrary, the `INIT ADAP ROUT` directive applies the desired initialization to the adapted mesh, i.e. to both the base and the descendent nodes in the same way. No interpolation is done in this case, and therefore the prescription of, say, initial velocities is more precise than in the other case if the initial velocity field is non-linear.

An example of use of the `IMAT` sub-directive is as follows. Assume one wants to simulate the explosion of a solid charge. The bomb domain needs to be discretized very finely in the first steps of the calculation, until the solid charge has detonated and the pressure waves start to expand. Thereafter, the mesh should be un-refined to recover CPU efficiency (large time increment). The initial mesh refinement of the bomb region can be specified by the `WAVE` directive, until a certain time `T1` (see page B.200). The region is discretized coarsely (base mesh) and the refinement is done at step 0 by the `WAVE` directive. In the `MATE` directive, air material (via e.g. `JWLS`) is assigned to the whole fluid coarse mesh, including the bomb. Then when `WAVE` refines the mesh, the air material is automatically propagated to all descendent elements. By using the `IMAT` sub-directive it is then possible to (re-)assign an initial solid explosive material to the effective (fine-meshed) bomb region, while (fine) elements outside the bomb remain with the air material inherited from the base mesh. It is not possible to do this fine-tuning by the `MATE` directive because, when `MATE` is read, mesh refinement has *not* taken place yet.

12 GROUP F—LOADS

Object:

These instructions determine the loads. The directive "LOAD" is an alias of the "CHAR" directive.

Syntax:

```
"CHARGE"  
  < "CONSTANTE" . . . >  
  < ndcha $[ "FACTORISEE" . . . ;  
            "PROGRAMMEE" . . . ]$ >  
  < "ADDF" . . . >  
  < "SPEC" . . . >  
  < "FCTE" . . . >  
  < "FIMP" . . . >  
  < "FDYN" . . . >
```

ndcha

Number of the temporary disc (logical unit number) where the loads are stored (disk of time-dependent loads). **This data is optional:** by default ndcha=1.

The various sub-directives, detailed in the following pages, are summarized hereafter:

1/ Constant loads :

```

"CONSTANTE"  $ "GRAVITE"      gx  gy  < gz >      $
              $ "ROTATION"    omega                /LECTURE/ $

```

2/ Factorized loads :

```

"FACTORISEE" <ndfact> ( | ( "DEPLA" . . . ) |
                        | ( "FORCE"  . . . ) |
                        | ( "PRESS"  . . . ) |
                        "TABLE" . . . )

```

3/ Programmed loads :

```

"PROGRAMMEE" ndprog $ "FORCE"      . . . $
                  $ "PRESSION"    . . . $
                  "ROUTINE"      . . .
                  < "ROUTINE"    . . . >

```

4/ Generalized loads for advection-diffusion calculations (JRC)

```

"ADDF"  $ "TIMP" . . . $
        $ "FLUX" . . . $
        $ "QGEN" . . . $
        $ "CONV" . . . $
        $ "RADI" . . . $
        $ "PRES" . . . $
        $ "VELO" . . . $
        $ "BLOQ" . . . $
        $ "VPLA" . . . $
        $ "VLIN" . . . $

```

5/ Seismic-like loads for use with spectral elements (JRC)

```

"SPEC" | "POIN" . . . |
       | "PLAN" . . . |
       | "SISM" . . . |

```

6/ New Constant loads:

```

FCTE  NODE /LECT/ $ FORC f ; MOME m $ VECT x y z

```

7/ Imposed time-dependent loads:

```

FIMP  NODE /LECT/ $ FORC f ; MOME m $ VECT x y z NUFO nf

```

8/ Dynalpy loads:

```

FDYN  NODE /LECT/ PZER p0 COEF c VECT x y z ELEM e

```

Comments:

1/- CONSTANTES

The constant loads act all along the calculation. It is the case of body weight or of a fixed acceleration.

2/- FACTORISEES

The defined loads are multiplied by a coefficient which varies in time and is interpolated from a table:

$$Q(t) = A * C(t)$$

3/- PROGRAMMEES

The loads will be read and computed for some elementary times which are fixed a priori by the user in a subroutine given by him (FPROG or PPROG). EUROPLEXUS will then perform a linear interpolation to determine the loads at the precise instants of the calculation.

1/ Constant loads:

FCTE NODE /LECT/ \$ FORC f ; MOME m \$ VECT x y z

2/ Imposed time-dependent loads:

FIMP NODE /LECT/ \$ FORC f ; MOME m \$ VECT x y z NUFO nf

3/ Dynalpy loads:

FDYN NODE /LECT/ PZER p0 COEF c VECT x y z ELEM e

4/- FCTE

The constant loads act all along the calculation. It is the case of body weight or of a fixed load.

5/- FIMP

The defined loads are multiplied by a coefficient which varies in time and is interpolated from a table:

$$Q(t) = A * C(t)$$

6/- FDYN

These loads are only related to 1-D elements of type TUBE or TUYA.

These instructions are described in detail on the following pages.

12.1 AUXILIARY FILE

Object:

This directive allows to read the initial conditions data from an auxiliary file.

Syntax:

```
"CHARGE"    ndcha  < "FICHIER"    'nom.fic'  >
```

In certain cases the data may be bulky. It is then recommended to store them on an auxiliary file to shorten the main input data file. The auxiliary file is activated by means of the keyword "FICHIER" that precedes the file name (complete under Unix). In the main data file then only the keywords "CHARGE" "FICHIER" remain.

The auxiliary file (in free format) contains the whole set of load data, except the keyword "CHARGE". To return to the main input data, the auxiliary file must be terminated by the keyword "RETOUR".

12.2 CONSTANT LOADS

Object:

This directive allows to introduce constant accelerations (most often gravity) during the whole computation. It also gives the possibility to compute a structure in a rotating frame (with a constant rotation speed). Definition of sinusoidal acceleration is available (PERIODE and PHASE). It is also possible that the acceleration is linear and then keeps a constant value (RAMPE).

Syntax:

```
"CONSTANTE"  | [ "GRAVITE"      gx  gy  < gz > /LECTURE/  ;
                  "ROTATION"    omega      /LECTURE/  ] |
                  <"PERIODE"> Tx   Ty   Tz
                  <"PHASE">  Phix Phiy Phiz
                  <"RAMPE">   tx   ty   tz
```

gx gy gz

Components of the acceleration or of the gravity.

omega

Rotation speed (rad/s).

Tx Ty Tz

Period of the sinusoidal acceleration

Phix Phiy Phiz

Phase of the sinusoidal acceleration

tx ty tz

Time after which acceleration is constant

LECTURE

Numbers of the concerned nodes.

Comments:

The component $\langle g_z \rangle$ of the acceleration only makes sense for a three-dimensional computation.

Forces due to gravity or the acceleration of a moving frame are applied to the nodes of the structure defined in the directive `/LECTURE/`.

In the case of a calculation in a rotating frame, it is assumed that the rotation axis is Oz. The forces applied to the nodes are:

$$F_x = M\omega^2 x$$

$$F_y = M\omega^2 y$$

$$F_z = 0$$

This force applies to the nodes specified by the following `/LECTURE/` directive.

If all nodes are concerned, it is sufficient to put the word `TOUS` in place of the directive `/LECTURE/`.

In the case of pipelines, it is important to couple this directive with `"INIT"` `"DEBIT"` (page E.120) so as to avoid the transient due to sudden application of gravity.

For the tubes and the pipelines, the rotation constant charge does not make sense. Note that, although there is just one d.o.f. for the elements of type `TUBE`, the gravity vector may have an arbitrary orientation.

For a sinusoidal acceleration, the amplitude is defined by the component of the acceleration and is multiply by a sinus function. The user has to define the period of the function and the phase. If a component is not exited, the period and the phase for this component have to be zero.

The ramp acceleration is defined by a linear part that starts from zero at the initial time and then grow linearly to pass by the point `tx=gx` (for example). Then after `tx`, the acceleration is kept constant at a `gx` level. If a component is not exited, the time for this component has to be zero.

12.3 FACTORIZED LOADS

Object:

This directive allows to input loads varying in time, of the following type: $Q(t) = A * C(t)$, with:

- A as a base value (displacement, force or pressure);
- C(t) as a coefficient whose values, depending on time, are supplied by a table.

Syntax:

```
"FACT"  ndfact
      (    ( "DEPLA"  . . . )
          ( "FORCE"   . . . )
          ( "PRESS"   . . . )
          ( "ACCE"    . . . )

          "TABLE" . . .
      )
```

ndfact

Number of the disc (logical unit number) on which the data of the factorized loads are stored. **This data is optional:** by default ndfact=2.

Comments:

The instruction "FACT" cannot be used more than once.

On the contrary, the sequence terminating with the "TABLE" directive may be repeated as many times as necessary.

Warning:

The instruction "TABL" is mandatory. For the case of charges constant in time, just give a two-entry table with the same value in both entries, i.e. of the form "TABL 2 t1 v t2 v", where v is the (constant) value, t1 is less or equal to the initial time of the computation and t2 is greater or equal to the final time of the computation.

12.3.1 DISPLACEMENT

Object:

This option enables displacements depending on time to be prescribed.

Syntax:

```
"DEPLA"  /LECDDL/  d0  /LECTURE/
```

LECDDL

Reading procedure of the numbers of the degrees of freedom concerned.

d0

Base value of the imposed displacement. The instruction "TABL" must follow to determine the variation in time, **even for constant displacements** (see note below).

LECTURE

Numbers of the nodes concerned.

Comments:

The displacements of the nodes defined by the procedure LECTURE and along the directions determined by LECDDL, are of the following type:

$$D(t) = D0 * C(t)$$

$C(t)$ is provided by the first array met after that option (see "TABL").

That option can be used as many times as necessary .

If the imposed displacement is a blockage ($d0 = 0$), it is better to use the option "BLOQUE" of the instruction "LIAISON" (page D.30).

Warning:

The instruction "TABL" is mandatory. For the case of charges constant in time, just give a two-entry table with the same value in both entries, i.e. of the form "TABL 2 t1 v t2 v", where v is the (constant) value, t1 is less or equal to the initial time of the computation and t2 is greater or equal to the final time of the computation.

12.3.2 FORCE**Object:**

This option enables nodal forces, varying in time, to be imposed.

Syntax:

```
"FORCE" /LECDDL/ f0 /LECTURE/
```

LECDDL

Reading procedure of the numbers of the degrees of freedom concerned.

f0

Base value of the imposed force. The instruction "TABL" must follow to determine the variation in time, **even for constant forces** (see note below)

LECTURE

Numbers of the nodes concerned.

Comments:

The forces applied to the nodes defined by the procedure LECTURE and according to the directions determined by LECDDL, have an intensity of:

$$F(t) = F0 * C(t)$$

C(t) is provided by the first array (TABLE) met after that option (see "TABL", page F.150).

That option may be used as often as necessary.

For an textbfaxisymmetric computation, the force must be divided by 2π :

$$F0 = \frac{\text{real force}}{2\pi}$$

Warning:

The instruction "TABL" is mandatory. For the case of charges constant in time, just give a two-entry table with the same value in both entries, i.e. of the form "TABL 2 t1 v t2 v", where v is the (constant) value, t1 is less or equal to the initial time of the computation and t2 is greater or equal to the final time of the computation.

12.3.3 PRESSURE

Object:

This option introduces a pressure which is exerted on a segment set (2-dimensional case) or on a surface composed of shell elements (2-dimensional or 3-dimensional case), or on the faces of solid elements (3-dimensional case).

Syntax:

```
"PRESSION"  
| ( "SEGMENT" . . . ) |  
| ( "COQUE" . . . ) |  
| ( "FACE" . . . ) |  
| ( "NODE" . . . ) |
```

Comments:

The word "PRES" is the first keyword of the option.

There is no need to repeat "PRES" to re-define a new line or surface corresponding to the same table (the one which immediately follows the word "PRES").

On the contrary, it is compulsory to define again the word "PRES" if it is necessary to create lines or surfaces relative to another table.

SEGMENT PRESSURE**Object:**

This directive is mainly used to apply a pressure to 2D continuum elements (in 3D cases, use PRES FACE). It allows to enter a pressure which is exerted on a certain number of adjacent segments or lines, in the case of 2-dimensional computation. The pressure may vary in time (factorized loads) or be hydrostatic.

Syntax:

```
"SEGMENT"  |[  p0      ;  
              "HYDRO" rho  g  z0  ]|  /LECTURE/
```

p0

Base value p_0 of the pressure. The instruction TABL must follow to determine the variation in time, even for constant pressures (see note below).

HYDRO

Keyword that announces a hydrostatic pressure. The pressure field is generated by the presence of a “vertical” acceleration of gravity, acting along the y -axis in 2D or the z -axis in 3D.

rho

Density ρ of the liquid which generates the pressure.

g

Acceleration of gravity g applied to the fluid in the “vertical” direction, i.e. along the y -axis in 2D or the z -axis in 3D. The value should normally be negative (see also comments below).

z0

Vertical level z_0 of the free surface of the fluid, supposed horizontal.

LECTURE

Numbers of the nodes composing the pressure line. Each couple of subsequent nodes forms a segment. For example, if the list contains the three indexes 25, 4, 39, then two segments are considered: (25, 4) and (4, 39). Special care must be taken if one uses the name of GIBI object(s) to define the line(s) subjected to pressure, see comments below.

Comments:

The user has the choice between a pressure p_0 which varies in time and a hydrostatic pressure. It is not allowed to define both at the same time without re-using the word **SEGM**.

For a defined basic value p_0 , the pressure intensity is:

$$p(t) = p_0 C(t)$$

$C(t)$ is provided by the first array (**TABL**) met after the option **PRES** (see also **TABL**).

If a hydrostatic pressure is defined (keyword **HYDR**), the pressure is only applied to the segments of the line at levels z , such that:

$$g(z - z_0) \geq 0$$

From the previous expression it appears that, since the hydrostatic pressure should be exerted on segments at $z < z_0$, then the value of g specified should normally be negative.

Moreover, the intensity of the pressure is:

$$p = \rho g(z - z_0)$$

The values ρ , g , z_0 may be negative.

Each new option **SEGM** defines a different line.

If the line of segments is defined by giving the name of a **GIBI** object, make sure that the line is oriented, i.e. that the nodes are listed from one extremity to the other in the correct order (not randomly, as it may sometimes occur in some mesh generators).

If there is more than one line, and the lines are disjoint from one another (i.e. the final node of one line is not the initial node of the next one), then the **PRES** directive must be repeated (one directive for each line). Otherwise, a “spurious” segment joining the final node of a line to the initial node of the next one in the list would also be considered as subjected to pressure. This is particularly dangerous if more than one **GIBI** object is listed in the same **LECT**. In case of doubt, it is always safer to use a separate **PRES** directive for each line.

Sign of the pressure:

The order in which the points of the line are read by means of the procedure **LECT**, defines the orientation of the contour of that line. The normal vector (\vec{n}) results from a rotation of $\pi/2$ of the contour itself.

Positive values will create forces in the orientation of the normal (\vec{n}) thus obtained.

Warning:

The keyword **TABL** is mandatory. For the case of charges constant in time, just give a two-entry table with the same value in both entries, i.e. of the form **TABL 2 t1 v t2 v**, where **v** is the (constant) value, **t1** is less or equal to the initial time of the computation and **t2** is greater or equal to the final time of the computation.

SHELL PRESSURE**Object:**

This directive is mainly used to apply a pressure to 2D or 3D shell elements. For two- or three-dimensional computations, this option enables a pressure to be entered, which is exerted on a surface composed of shell elements. The pressure may vary in time (factorized loads) or be hydrostatic.

Syntax:

```
"COQUE"  |[      p0      ;  
           "HYDRO" rho  g  z0      ;  
           "HYDRO" rho  gx gy gz  x0 y0 z0 ]| /LECTURE/
```

p0

Base value of the pressure (non hydrostatic case). The instruction "TABL" must follow to determine the variation in time, even for constant pressures (see note below).

HYDRO

Hydrostatic pressure.

rho

Density of the fluid which generates the pressure.

For two-dimensional computations:**g**

Acceleration applied to the fluid.

z0

Level of the free surface, which is supposed horizontal.

LECTURE

Numbers of the shell elements submitted to the pressure.

For three-dimensional computations:

g_x, g_y, g_z

Components of the acceleration vector (G) which is applied to the fluid.

x_0, y_0, z_0

Coordinates of a point located on the surface supposed horizontal.

LECTURE

Numbers of the shell elements submitted to the pressure.

Comments:

The user has the choice between a pressure p_0 which varies in time and a hydrostatic pressure. It is impossible to define both at the same time, without re-using the word "COQUE".

If a basic value p_0 has been defined, the pressure intensity is:

$$P(t) = P_0 * C(t)$$

$C(t)$ is provided by the first array met after the option "PRESS" (see TABLE).

If a hydrostatic pressure is defined (keyword HYDR), the pressure is applied to the points on the surface of coordinates X,Y,Z such that :

$$g_x * (x-x_0) + g_y * (y-y_0) + g_z * (z-z_0) > 0 \text{ (or } = 0 \text{)}$$

The pressure intensity will be for these points:

$$P = \rho * (g_x * (x-x_0) + g_y * (y-y_0) + g_z * (z-z_0))$$

In a two-dimensional case, the definition of the hydrostatic pressure is the same as for the sub-directive "SEGMENT".

For a three-dimensional computation, see the definition of hydrostatic pressure with the sub-directive "FACE".

Each new option "COQUE" defines a new pressure surface.

Sign of the pressure:

The normal vector on the surface is oriented according to the numeration of the shell nodes. Positive pressures will create forces in the orientation of that normal.

The orientation of the normal is given by the following rule (Maxwell's cork-screw rule). An observer placed at the centre of the shell element which is crossed by the normal from the bottom to the top, must be able to notice that the shell element is numbered in increasing order, by rotating in a trigonometric sense (anticlockwise).

Warning:

If the mesh developed by means of "COCO" is used, the elements are not necessarily oriented in the same way (this has to be explicitly requested).

If the mesh is entered by the user, all elements have to be numerated so that their orientation is coherent. A shell surface composed of elements which are oriented in a different way, may produce errors and confusion concerning the direction of the pressures from the data point of view as well as from the results.

The instruction "TABL" is mandatory. For the case of charges constant in time, just give a two-entry table with the same value in both entries, i.e. of the form "TABL 2 t1 v t2 v", where v is the (constant) value, t1 is less or equal to the initial time of the computation and t2 is greater or equal to the final time of the computation.

FACE PRESSURE**Object:**

This directive is mainly used to apply a pressure to 3D continuum elements. For a three-dimensional computation, this option enables a pressure to be entered, which is exerted on a surface composed of the sides of solid elements. The pressure may vary in time (factorized loads) or be hydrostatic.

Syntax:

```
"FACE" iface  | [      p0
                  "HYDRO" rho  gx gy gz  x0 y0 z0 ] | /LECTURE/
```

iface

Number of the side (face) of the elements read by the procedure. LECTURE

p0

Base value of the pressure (non hydrostatic). The instruction "TABL" must follow to determine the variation in time, even for constant pressures. (see note below).

HYDRO

Hydrostatic pressure.

rho

Density of the fluid which generates the pressure.

gx, gy, gz

Components of the acceleration vector (G) applied to the fluid.

x0, y0, z0

Coordinates of a point of the free surface which is supposed horizontal.

LECTURE

Numbers of the elements submitted to the pressure.

Comments:

The user has the choice between a pressure P0 which varies in time and a hydrostatic pressure. It is impossible to define both at the same time without re-using the word "FACE".

For a defined basic value P0, the intensity of the pressure is:

$$P(t) = P0 * C(t)$$

C(t) is provided by the first array met after the option "PRESS" (see TABLE).

Hydrostatic pressure:

If a hydrostatic pressure is defined (keyword HYDR), it is applied to the points of the pressure surface of coordinates X,Y,Z such that:

$$gx * (x-x0) + gy * (y-y0) + gz * (z-z0) > 0 \text{ (or } = 0 \text{)}$$

For these points, the intensity is:

$$P = \rho * (gx * (x-x0) + gy * (y-y0) + gz * (z-z0))$$

The free surface of the liquid is composed of the plane which passes through the point M0 of coordinates (x0, y0, z0) and perpendicular to the vector (G) whose components are : (gx,gy,gz).

If the vector (G) is drawn with the point M0 as origin, the pressure will be applied to the points M of the pressure surface which are located in the half-space containing (G).

The pressure intensity at point M is:

$$P = \rho * g * h$$

Here h represents the distance between M and the free surface; g represents the gravity.

The vector (G) with its 3 components enables the surface of a liquid (horizontal) to be entered, when the vertical axis of the mesh is distinct from the physical vertical line. In this case, the surface is an inclined plane in the coordinate system of the mesh.

Each new definition of the option "PRES" "SEGMENT" generates a new pressure surface.

Sign of the pressure:

The normal to the face of an element is the outward normal of that element. A positive pressure creates a force in the orientation of that normal.

Warning:

The instruction "TABL" is mandatory. For the case of charges constant in time, just give a two-entry table with the same value in both entries, i.e. of the form "TABL 2 t1 v t2 v", where v is the (constant) value, t1 is less or equal to the initial time of the computation and t2 is greater or equal to the final time of the computation.

NODE PRESSURE**Object:**

This directive is mainly used to apply a pressure to continuum elements (2D or 3D). This option enables a pressure to be entered, which is exerted on a surface composed of the nodes belonging to the edge of structure. The pressure may vary in time (factorized loads) or be hydrostatic.

Syntax:

```
"NODE"      | [      p0
               "HYDRO" rho  gx gy gz  x0 y0 z0 ] | /LECTURE/
```

p0

Base value of the pressure (non hydrostatic). The instruction "TABL" must follow to determine the variation in time, even for constant pressures. (see note below).

HYDRO

Hydrostatic pressure.

rho

Density of the fluid which generates the pressure.

gx, gy, gz

Components of the acceleration vector (G) applied to the fluid.

x0, y0, z0

Coordinates of a point of the free surface which is supposed horizontal.

LECTURE

Numbers of the nodes belonging to the edge of structure, submitted to the pressure.

Comments:

The user has the choice between a pressure P0 which varies in time and a hydrostatic pressure. It is impossible to define both at the same time without re-using the word "NODE".

For a defined basic value P0, the intensity of the pressure is:

$$P(t) = P0 * C(t)$$

C(t) is provided by the first array met after the option "PRESS" (see TABLE).

Hydrostatic pressure:

This directive is described in [12.3.3](#) (page F.130).

Sign of the pressure:

The normal to the face of an element is the outward normal of that element. A positive pressure creates a force in the orientation of that normal.

Warning:

The instruction "TABL" is mandatory. For the case of charges constant in time, just give a two-entry table with the same value in both entries, i.e. of the form "TABL 2 t1 v t2 v", where v is the (constant) value, t1 is less or equal to the initial time of the computation and t2 is greater or equal to the final time of the computation.

12.3.4 ADDITIONAL ACCELERATION

Object:

This option enables additional acceleration depending on time to be prescribed.

Syntax:

```
"ACCE"  /LECDDL/  g0  /LECTURE/
```

LECDDL

Reading procedure of the numbers of the degrees of freedom concerned.

g0

Base value of the acceleration to be added. The instruction "TABL" must follow to determine the variation in time, **even for constant displacements** (see note below).

LECTURE

Numbers of the nodes concerned.

Comments:

The external forces applied on the nodes defined by the procedure LECTURE and along the directions determined by LECDDL, are of the following type:

$$F(t) = g0 * M * C(t)$$

M is the nodal mass. C(t) is provided by the first array met after that option (see "TABL").

That option can be used to enter gravity like forces depending on time (for example, deceleration forces inside a tank during a crash).

Warning:

The instruction "TABL" is mandatory. For the case of charges constant in time, just give a two-entry table with the same value in both entries, i.e. of the form "TABL 2 t1 v t2 v", where v is the (constant) value, t1 is less or equal to the initial time of the computation and t2 is greater or equal to the final time of the computation.

12.3.5 TABLE**Object:**

The tables provide the different values of the coefficients $C(t)$ which appears in the options of the instruction "CHARGE FACTORISEE" (factorised load).

The $C(t)$ functions are continuous, linear by parts and therefore defined by points. They can linearly interpolate more complex functions.

Syntax:

```
"TABLE"  npts*( tk  ck )
```

npts

Number of couples (tk, ck) defining the table.

tk

Elementary time.

ck

Multiplying factor at the time tk: $ck=C(tk)$.

Comments:

Each array refers to the options "DEPL", "FORC", "PRES" which are defined in the data set before the table and follow the preceeding table, if it exists. The first table refers to all options defined after the key-word "FACT". It is not allowed to have two consecutive tables in the data set.

If the time $t_1 = 0$ is not specified in the table, the point of origin (0, 0) is assumed to belong to the curve.

The last time used in the array must be greater than the final time of the computation.

12.4 PROGRAMMED LOADS

Object:

This option enables the user to enter forces or pressures applied to certain nodes or certain elements of the structure, for time instants defined by the user himself.

In this case, the values are defined at each time by linear interpolation.

Syntax:

```
"PROG"  ndprog    $  "FORCE"    . . .    $
                  $  "PRESSION" . . .    $
                  "ROUTINE"   . . .
                  <  "ROUTINE" . . .    >
```

ndprog

Number of the disc (logical unit number) where the programmed loads are stored. This disc is temporary (the data is destroyed after the creation of a disc of loads depending on time ndcha).

Comments:

The word "PROG" is the first key-word of the option. It may be used only once in the EUROPLEXUS data set.

There are 2 subdirectives "PRES" and "FORC" which respectively enable pressures or forces to be input.

For each of them, the user has to initially provide a list, which contains according to the circumstances:

- The numbers of the nodes defining the pressure line or the numbers of the elements to which pressures are applied.
- The numbers of the degrees of freedom and the numbers of the nodes to which forces are applied.

The list is entered by the means of the pocedures LECTURE and LECDDL.

There are three ways to enter the values of the forces and pressures at the different time increments.

1/- The user directly inserts, into the data set, the cards which each correspond to one time increment. Each card successively contains :

- The value of the time increment concerned.
- The values of the pressures and forces determined according to the preceeding list.

2/- The user has at his disposal or creates a data file with an imposed standard writing format. The file must successively store several sets of values, each corresponding to one time increment. Each set sequentially defines :

- The value of the time increment concerned
- The values of the pressures and forces determimed in the same way as the cards.

3/- Regarless of the program, the user provides a subroutine which computes the values of the forces or pressures for each time increment before the program runs. These values are stored in array F or P according to the list.

The data can be read on a file the format of which is not standard, or the values can be directly entered by the means of an analytic formula chosen by the user.

The keywords "ROUTINE" introduce respectively the data associated with each of the subroutines FPROG and PPROG, written by the user.

ATTENTION: the data relative to FPROG are read first.

If there are only forces (or only pressures), a single keyword "ROUTINE" is sufficient to introduce the corresponding data.

12.4.1 FORCE**Object:**

For each time increment, this option enables nodal forces to be applied to certain nodes of the structure and according to certain directions (degrees of freedom).

Syntax:

```

$ "FORCE" "DDL" /LECDDL/ /LECTURE/ ... $
$ "MXTF" ntmax | "CART" ( "INST" ti f1 ... fn ) | $
$ | "BAND" nb | $
$ | "ROUTINE" | $

```

LECDDL

Reading procedure of the degrees of freedom to which the forces are applied.

LECTURE

Reading procedure of the numbers of the nodes to which the forces are applied.

ntmax

Maximum number of time instants for which the loads are defined.

CART

Keyword which enables the input of cards.

BAND

Keyword which enables the data on a file to be read.

nband

File number.

ROUT

Keyword which enables the user to provide a computation subroutine. After the word ROUT, and on a new card, the user can, if he wants, provide the data which is read by the subroutine in the order of the writing.

Comments:

The directive "FORCE" may be used at most once.

For an axisymmetric calculation, the forces must be divided by 2π , since the calculation refers to ONE radians.

The elements of the list defined by LECDDL and LECTURE are stored according to their nodes and each node according to its degree of freedom, in the order of their definitions in the procedures LECTURE and LECDDL.

For example :

"FORC" "DDL" 123 "LECT" 7 8 10 "TERM"

will define the list $n(1,7)$ $n(2,7)$ $n(3,7)$ $n(1,8)$ $n(2,8)$ $n(3,8)$ $n(1,10)$ $n(2,10)$ $n(3,10)$ where $n(i,j)$ represents the i th degree of freedom of the node j . The 7 th element of this list is the first degree of freedom of the 10 th node.

The parameter $ntmax$ represents the maximum number of time increments for which the loads are defined. At the most $ntmax$ cards or $ntmax$ data sets can be read on the file. If a subroutine is entered, it is used $ntmax$ times.

The user must choose between 3 input modes for the data :

CARD ("CART")

FILE ("BAND")

SUBROUTINE ("ROUTINE")

Only one can be used.

File:

The parameter $nband$ represents the number of the file from which the data is read. The file has been first defined at the level of the control cards.

The different sets of values are written on that file. Each set contains:

- the value of the time increment to which the data of the set is associated;
- the values of the forces $F(j,t)$, as for the cards.

The number of the values $F(j,t)$ must corresponds to the number of the elements defined in the LECDDL-LECTURE list, that is to say s values (s : number of nodes ; x : number of

degrees of freedom). The j th value $F(j,t)$ represents the value of the force applied to the node at time t , according to the direction defined by the j th element of the list.

The values are written on the file unformatted.

If the user reaches the end of the file before he has read all nt value sets, EUROPLEXUS considers that there are no more values to be read. In this case, the loading is finished and the program goes to the next option or instruction.

If there are more than nt data sets, the loading is considered as finished after the reading of the nt th set.

In both cases, the last time increment must be greater than the final time of computation defined in the instruction "CALCUL".

The reading of the $nband$ number leads EUROPLEXUS to read the file automatically.

Subroutine:

The key-word "ROUT" must be the last data of the card. It automatically calls the user's subroutine. At the most, the latter will be used nt times (but if he wants, the user can stop the subroutine before the nt th . After each call, the subroutine provides the EUROPLEXUS program with an array (F or P) which contains as many values as elements defined in the LECDDL-LECTURE list. The j th value of the table is the value of the node to which a force is applied according to the direction determined by the j th element of the list. If he wants, the user can provide, after the word "ROUTINE", the data which are written on new cards. This data is read sequentially and in the order required by the subroutine. For more explanations, see the chapter "PROGRAMMED LOADS-SUBROUTINE".

12.4.2 CARDS

Object :

This suboption enables loads (forces or pressures) to be read on cards.

Syntax :

```
"CARTES" ( "INSTANT" ti f1 ... fs ) "TERMINE"
```

INSTANT

First keyword of each card.

ti

Time instant to which the data of the card is associated.

f1 ... fs

Value of the force applied to the node and degree of freedom defined at the j th place of the list entered by LECDDL-LECTURE, at time t_i . There are s f_j values per card (s representing the product of the number of degrees of freedom and the number of nodes).

TERMINE

This key-word denotes that there is no more data to be read. The loading is finished and the program takes the next option.

Comments:

At the maximum, the word "INSTANT" is written nt times.

Each card must include, after the value of the time increment t_i , as many f_j values as elements defined in the LECDDL-LECTURE list, that is to say s values. The j th f_j value is the value of the force which is applied, at time t_i , to the node and according to the direction defined by the j th element of the list. All the numerical values are read in free format.

The word "TERM" indicates that there is no more data to be read. The program considers the loading as finished and takes the next option or instruction.

The last time increment to be read must be greater than to the final time of computation defined in the instruction "CALCUL".

The word "CARTE" leads EUROPLEXUS to read the cards automatically.

12.4.3 PRESSURE

Object:

This option enables the following to be entered for each time instant:

- a pressure which is exerted on a set of adjacent segments or "pressure line", for two-dimensional computations.
- a pressure which is exerted on shell elements defining a surface, for two- and three-dimensional computations.
- a pressure which is exerted on the faces of solid elements defining a "pressure surface".

Syntax:

```

$ "PRESS"          | "SEGM" /LECTURE/          | ...    $
$                  | "COQU" /LECTURE/          |        $
$                  | "FACE" iface /LECTURE/ |        $
$ "MXTP" ntmax     | "CART" ( "INST" ti f1 ... fn ) |        $
$                  | "BAND"  nb              |        $
$                  | "ROUTINE"              |        $

```

SEGM

This concerns two-dimensional computations; the user enters a pressure line. Then, the procedure LECTURE is used for the input of the numbers of the nodes composing the line. On the whole, there are $s + 1$ nodes defining s segments.

COQU

This concerns two- or three-dimensional computations; the user enters a surface composed of shell elements. Then, the numbers of the shell elements are entered by the procedure LECTURE.

FACE

This concerns three-dimensional computations; the user enters a surface composed of the faces of solid elements.

iface

Number of the solid element face belonging to the surface. In this case, the procedure LECTURE is used for the input of the solid elements concerned.

ntmax

Maximum number of time instants where the pressures are defined.

Comments:

The input syntax of the data is the same as for the "PROGRAMMED FORCES".

The words "CART" "BAND" "ROUTINE" and the parameters ntmax and nb have the same signification as for the "PROGRAMMED FORCES".

As for the latter, the j-th value defined after the time increment, on the cards or in the data sets written on the file, gives the value of the pressure which is exerted, at that time increment, on the j-th element of the list provided by the procedure LECTURE.

The j th element of the array provided by the subroutine has the same meaning (see the chapter "PROGRAMMED FORCES").

For a two-dimensional computation, the options "SEGM" and "COQU" can be used one after the other.

For example :

```
"PRES"  "SEGM"  "LECT" 1 2 3 4 "TERM"  
        "COQU"  "LECT" 9 13 18 "TERM"
```

The list given by the two procedures LECTURE is respectively composed by the 3 segments 1-2 2-3 3-4, and by the 3 shells 9 13 18. The 6-th element of the list represents the shell number 18.

For three-dimensional computation, the options "COQU" and "FACE" can be used one after the other.

For example :

```
"PRES"  "COQU"  "LECT" 9 10 25 "TERM"  
        "FACE" 3 "LECT" 16 2 12 "TERM"
```

The list given by the two procedures LECTURE is respectively composed of the shell elements 9 10 25, and the solid elements 16 2 12. The 4-th element of the list represents the solid element number 16.

If there are two procedures LECTURE, the list which must take into account the values of the cards, data sets or tables may be obtained by putting the 2 lists defined by the procedures end to end, in the order of their definitions (the second list follows the first).

If he wants, the user can define several times the words "SEGM" "COQU" or "COQU" "FACE" with the corresponding lists, and in any order. The principle used to obtain the final list on which the loads are defined is the same one used if there are only two LECTURE procedures.

Examples:

```
1/ 2-D      "PRES"  "SEGM"  /LECTURE/  
            "COQU"  /LECTURE/  
            "SEGM"  /LECTURE/  
  
2/ 3-D      "PRES"  "FACE"  1 /LECTURE/  
            "COQU"  /LECTURE/  
            "FACE"  4  /LECTURE/
```

For the conventions relative to the pressure signs, see the chapter on FACTORIZED LOADS and the corresponding options.

12.4.4 ROUTINE

Object:

This is a FORTRAN subroutine provided by the user and compiled independently from the program.

There is one subroutine for the forces (FPROG) and one for the pressures (PPROG). For each time increment, these subroutines create an array (F for forces and P for pressures), containing the values of the forces and the pressures according to the list supplied by the LECDDL - LECTURE procedures of the options "PROG" "FORC" or "PROG" "PRES".

Syntax:

1. Programmed forces :

```
SUBROUTINE FPROG(KAR,IMP,IPT,T,N,NUF,FEP,TAB)
IMPLICIT DOUBLE PRECISION(A-H,O-Z)
DIMENSION FEP(*),TAB(*),NUF(2,*)
. . .
  FORTRAN data set to compute the forces
. . .
RETURN
END
```

2. Programmed pressures :

```
SUBROUTINE PPROG(KAR,IMP,IPT,T,N,IPOS,NUP,PRP,PRF,TAB)
IMPLICIT DOUBLE PRECISION(A-H,O-Z)
DIMENSION IPOS(N),NUP(*),PRP(*),PRF(N),TAB(*)
. . .
  FORTRAN data set to compute the pressures
. . .
RETURN
END
```

KAR

Input variable. At the origin, it is a number of the file the value of which is 0 by default (card reader). It can be read or defined again inside the subroutine. If KAR is negative at the exit of the subroutine EUROPLEXUS considers that the loading is finished and stops the operation.

IMP

Input variable, number of the output (listing) file. It may be used to write messages to the listing. The value of IMP cannot be defined again inside the subroutine.

IPT

Input variable. It is a pointer which is increased by 1 by PLEXIS-3C after each use of the subroutine. IPT is equal to 1 at the first call. The value of IPT cannot be defined again inside the subroutine.

T

Output variable. At the exit of the subroutine T must be equal to the value of the elementary time linked to the values of the forces and the pressures (array FEP or PRP).

FEP or PRP

Output arrays. At the exit of the subroutine, they must respectively mention the values of the forces and the pressures, according to the list provided by the procedures LECDDL and LECTURE of the options "PROG" "FORC" and "PROG" "PRES".

N

Input variable equal to the number of the elements on the list. The arrays FEP and PRP are automatically dimensioned at N. The value of N must not be defined again inside the subroutine.

NUF

Indexes of nodes and ddls where a force is applied. $NUF(1,K)$ = number of k-th node in the list; $NUF(2,K)$ = associated ddl.

IPOS

Pointer on NUP and PRP: $KDEB = IPOS(I)$: address of beginning of face I; $KFIN = IPOS(I+1) - 1$: address of the end of face I; $NBNF = IPOS(I+1) - IPOS(I)$: number of nodes of face I.

NUP(KDEB:KFIN)

Indexes of nodes of face I.

PRP(KDEB:KFIN)

Pressures on nodes of face I.

PRF

Work array (dimension N). For example, pressures applied on the N faces at time T.

TAB

Work array. It enables (if necessary) to store values during the run of the subroutine (see example).

Comments:

The user's subroutine is called automatically nt times (nt is the maximum number of elementary times). It has been defined in the options "PROG" "FORC" and "PROG" "PRES".

However the user can stop the calling sequences before having achieved nt calls, by giving a negative value to NB at the exit of the subroutine (the call relative to the negative definition of NB is not taken into account).

If the user wants to define loads for n elementary times, with n inferior to nt, the value of NB must be negative at the (n+1)-th call. In any case, the last elementary time which has been computed must be superior to the final time of computation defined in the instruction "CALCUL".

SAMPLE ROUTINE FPROG

Here is the sample routine FPROG contained in the program:

```
C FPROG      SOUPLEX  LPRE      91/06/07    19:19:31
      SUBROUTINE FPROG(KAR,IMP,IPT,T,N,NUF,FEP,TAB)
C
C  OBJET : LIRE LES FORCES AUX NOEUDS POUR UN INSTANT CONSIDERE
C  -----
C
C  ==> EN RETOUR : CHARGER T ET REMPLIR  FEP CORRECTEMENT
C
C      KAR  : NUMERO DU FICHIER DE LECTURE ET INDICATEUR D'ARRET
C      IMP  : NUMERO DU FICHIER DE SORTIE (IMPRIMANTE)
C      IPT  : NUMERO DE L'INSTANT CONSIDERE
C      T    : VALEUR DU TEMPS CONSIDERE
C      N    : NOMBRE DE NOEUDS OU UNE FORCE EST APPLIQUEE
C      NUF(1,K) : NUMERO DU KIEME NOEUD DE LA LISTE
C      NUF(2,K) : DDL CONCERNE
C      FEP(K)  : FORCE CORRESPONDANTE
C      TAB   : TABLEAU DE TRAVAIL (DIM AJUSTABLE AVEC "TRAV" )
C
C  ==> IMPORTANT : NE PAS MODIFIER  IPT, N, NUF.
C
C      SI KAR EST MIS < 0 , ON ARRETE LA LECTURE AVANT D'AVOIR EPUISE
C      TOUS LES INSTANTS PREVUS
C
C      IMPLICIT DOUBLE PRECISION (A-H,O-Z)
C
C      CHARACTER*(10) MOT
C
C      PARAMETER (NFORC=20)
C
C      DIMENSION NUF(2,*),FEP(*),TAB(NFORC,2)
C      DIMENSION X(3)
C
C      IF(KAR.LE.0) RETURN
C
C      READ (KAR,*) MOT
C      IF(MOT.EQ.'TERMINE') GOTO 91
C      IF(MOT.NE.'INSTANT') GOTO 90
C
C---  LECTURE D'UN BLOC  (A T DONNE : NP COUPLES R,F )
      READ (KAR,*) T,NP
```



```
      IF(NP.GT.NFORC) GOTO 88
      READ (KAR,*) (TAB(K,1),TAB(K,2),K=1,NP)
C
      WRITE (IMP,*) ' ==> INSTANT : ',T
      CALL DPRINT(IMP,NP,TAB(1,1),'RAYON')
      CALL DPRINT(IMP,NP,TAB(1,2),'VALEUR')
C
C---   COORDONNEES DU CENTRE :
      XC=0
      YC=0
C
C---   CALCUL DES FORCES PAR INTERPOLATION LINEAIRE
      DO 50 K=1,N
      NUPO=NUF(1,K)
      CALL QUIDNE(0,NUPO,LON,X)
      IF(LON.NE.3) GOTO 89
      DX=X(1)-XC
      DY=X(2)-YC
      R=SQRT(DX*DX+DY*DY)
      CALL DITPL1(NP,TAB(1,1),TAB(1,2),R,F,DFSDR,NX,IER)
      IF(IER.NE.0) GOTO 87
      FEP(K)=F
50    CONTINUE
      RETURN
C
C---   ERREURS ET SORTIE
87    CALL ERRMSS('FPROG','INTERPOLATION INCORRECTE')
      STOP
88    CALL ERRMSS('FPROG','IL Y A TROP DE VALEURS')
      STOP
89    CALL ERRMSS('FPROG','IL N Y A PAS 3 COORDONNEES')
      STOP
90    CALL ERRMSS('FPROG','SYNTAXE INCORRECTE')
      STOP
91    KAR=-KAR
      RETURN
C
      END
```

SAMPLE ROUTINE PPROG

Here is the sample routine PPROG contained in the program:

```
C PPROG      SOUPLEX  LPRE      91/06/07      19:19:12
      SUBROUTINE PPROG(KAR,IMP,IPT,T,N,IPOS,NUP,PRP,PRF,TAB)
C
C  OBJET : LIRE LES PRESSIONS AUX NOEUDS POUR UN INSTANT CONSIDERE
C  -----
C
C ==> EN RETOUR : CHARGER T ET REMPLIR  PRP CORRECTEMENT
C
C      KAR  : NUMERO DU FICHIER DE LECTURE ET INDICATEUR D'ARRET
C      IMP  : NUMERO DU FICHIER DE SORTIE (IMPRIMANTE)
C      IPT  : NUMERO DE L'INSTANT CONSIDERE
C      T    : VALEUR DU TEMPS CONSIDERE
C      N    : NOMBRE DE FACES SOUS PRESSION
C      IPOS : POINTEUR SUR NUP ET PRP :
C              KDEB = IPOS(I) : ADRESSE DU DEBUT DE LA FACE I
C              KFIN = IPOS(I+1)-1 : ADRESSE DE LA FIN DE LA FACE I
C              NBNF = IPOS(I+1)-IPOS(I) : NBR DE NOEUDS DE LA FACE I
C      NUP(KDEB:KFIN) : NUMEROS DES NOEUDS DE LA FACE I
C      PRP(KDEB:KFIN) : PRESSION AUX NOEUDS DE LA FACE I
C      PRF  : TABLEAU DE TRAVAIL (DIM N) : PAR EXEMPLE :
C              PRESSIONS APPLIQUEES SUR LES N FACES AU TEMPS T
C      TAB  : TABLEAU DE TRAVAIL (DIM AJUSTABLE AVEC "TRAV" )
C
C ==> IMPORTANT : NE PAS MODIFIER  IPT, N, IPOS, NUP.
C
C      SI KAR EST MIS < 0 , ON ARRETE LA LECTURE AVANT D'AVOIR EPUISE
C      TOUS LES INSTANTS PREVUS
C
C      SI NUP(K) EST > 10000 : IL S'AGIT D'UN NOEUD APPARTENANT A UN
C      ELEMENT DE COQUE (SINON EL. MASSIF), LE NUMERO EST ALORS
C          NUCO=MOD(NUP(K),10000)
C
C
C      IMPLICIT DOUBLE PRECISION (A-H,O-Z)
C
C      CHARACTER*(10) MOT
C
C      PARAMETER (NPRES=20)
C
C      DIMENSION IPOS(N),NUP(*),PRP(*),PRF(N),TAB(NPRES,2)
C      DIMENSION X(3)
C
```

```
C
      IF(KAR.LE.0) RETURN
C
      READ (KAR,*) MOT
      IF(MOT.EQ.'TERMINE') GOTO 91
      IF(MOT.NE.'INSTANT') GOTO 90
C
C---  LECTURE D'UN BLOC  (A T DONNE : NP COUPLES R,P )
      READ (KAR,*) T,NP
      IF(NP.GT.NPRES) GOTO 88
      READ (KAR,*) (TAB(K,1),TAB(K,2),K=1,NP)
C
      WRITE (IMP,*) ' ==> INSTANT : ',T
      CALL DPRINT(IMP,NP,TAB(1,1),'RAYON')
      CALL DPRINT(IMP,NP,TAB(1,2),'VALEUR')
C
C---  COORDONNEES DU CENTRE :
      XC=0
      YC=0
C
C---  CALCUL DES PRESSIONS PAR INTERPOLATION LINEAIRE
      DO 50 IFA=1,N
      KDEB=IPOS(IFA)
      KFIN=IPOS(IFA+1)-1
      NBNF=IPOS(IFA+1)-KDEB
      PRF(IFA)=0
      DO 40 K=KDEB,KFIN
      NUCO=MOD(NUP(K),10000)
      CALL QUIDNE(0,NUCO,LON,X)
      IF(LON.NE.3) GOTO 89
      DX=X(1)-XC
      DY=X(2)-YC
      R=SQRT(DX*DX+DY*DY)
      CALL DITPL1(NP,TAB(1,1),TAB(1,2),R,P,DPSDR,NX,IER)
      IF(IER.NE.0) GOTO 87
      PRP(K)=P
      PRF(IFA)=PRF(IFA)+P
40    CONTINUE
      PRF(IFA)=PRF(IFA)/NBNF
50    CONTINUE
      RETURN
C
C---  ERREURS ET SORTIE
87    CALL ERRMSS('PPROG','INTERPOLATION INCORRECTE')
      STOP
88    CALL ERRMSS('PPROG','IL Y A TROP DE VALEURS')
```

```
      STOP
89  CALL ERRMSS('PPROG','IL N Y A PAS 3 COORDONNEES')
      STOP
90  CALL ERRMSS('PPROG','SYNTAXE INCORRECTE')
      STOP
91  KAR=-KAR
      RETURN
C
      END
```

EXAMPLE 1**Problem:**

The user wants to enter FORCES into the directions x and y (degrees of freedom 1 and 2). These forces are applied to the nodes 1, 3, 5, 7, 9, 11.

Their values are the same for all the nodes, for both directions; these values are defined by analytical formulas:

$$F = 2.5 * \sin(\pi * t) \text{ For } t \leq 20 \text{ milliseconds}$$

$$F = 2.9 * \sin(2 * \pi * t) \text{ For } t > 20 \text{ milliseconds}$$

The programming concerns 51 elementary times, a value is calculated for each millisecond from 0 to 50.

At each elementary time, the user must provide $2*6=12$ values stored in array F (2 degrees of freedom and 6 nodes).

The user's subroutine:

```
      SUBROUTINE FPROG(KAR,IMP,IPT,T,N,NUF,F,TAB)
C
      IMPLICIT DOUBLE PRECISION(A-H,O-Z)
C
      DIMENSION F(*),TAB(*)
      DATA PI/3.1416/
C
      T=1E-3*(IPT-1)
      IF(IPT.GT.21) GOTO 20
      DO 10 I=1,N
10    F(I) = 2.5 * DSIN( PI * T )
      RETURN
20    IF(IPT.GT.51) GOTO 40
      DO 30 I=1,N
30    F(I) = 2.9 * DSIN( 2*PI * T )
      RETURN
40    KAR=-KAR
      RETURN
      END
```

Data of the instruction "CHARGE":

```
"CHARGE" 1 "PROG" 2 "FORCE" "DDL" 12 "LECT" 1 PAS 2 11 "TERM"  
          "MXTF" 60 "ROUTINE"
```

Comments:

The user can directly define these coefficients inside the subroutine. There are no data cards of the user behind the word "ROUTINE".

The value of ntmax must be superior or equal to 51, since the subroutine has performed a test in order to stop the loading at this value.

EXAMPLE 2**Problem:**

The user has at his disposal a file containing data sets (elementary times and loads). It is supposed that there are a hundred values of pressure per elementary time and each data set is written under the form : (F10.5,100F15.7).

The user wants to enter only one elementary time out of ten (numbers 10, 20, 30, ...) from the initial file. For each elementary time, the user selects the pressures occupying the ranks 1, 3, 7, 9, 11, 12 and 15 among the initial data sets. Therefore, 7 values must be taken into account. The file has to be read completely and contains no more than 100 value sets. The file has been defined by number 9 at the level of the control cards.

The user's subroutine:

```
      SUBROUTINE PPROG(KAR,IMP,IPT,T,N,IPOS,NUP,P,PRF,TAB)
C
      IMPLICIT DOUBLE PRECISION(A-H,O-Z)
C
      DIMENSION P(*),TAB(*),IPOS(*),NUP(*),PRF(*)
      DIMENSION NV(7)
      DATA NV/1,3,7,9,11,12,15/
C
      IF(IPT.GT.1) GOTO 10
C-----  THE PRESSURES ARE EQUAL TO ZERO AT T=0
      T=0
      DO 5 I=1,N
5      P(I)=0
C-----  READING OF THE NUMBER OF THE FILE AND THE LENGTH OF THE SETS
      READ(KAR,*) NB,NMAX
      RETURN
10  DO 20 I=1,10
      READ(KAR,1000,END=50) T,(TAB(I),I=1,NMAX)
1000 FORMAT(F10.5,100F15.7)
20  CONTINUE
C-----  AT THE END OF THE LOOP ON RECORDS HAVE BEEN SKIPPED
      DO 30 I=1,7
30  P(I)=TAB(NV(I))
      RETURN
50  KAR=-KAR
      RETURN
      END
```

Data of the instruction "CHARGE":

```
"CHARGE" 1 "PROG" 2 "PRESS" "COQUE" "LECT" 1 2 3 5 "TERM"  
          "FACE" 3 "LECT" 7 9 11 "TERM"  
          "MXTP" 101 "ROUTINE"  
          9 100
```

Comments:

The subroutine gives an example of use of the work array TAB.

The data are read at the first call and the value of the pressures at $t = 0$ is fixed on zero. The user selects 7 values ($N = 7$).

At the following calls, 9 records on the file are skipped in order to get positionned on the sets 10, 20, 30, ... 100, at the end of the reading loop. Then, the right values supplied by the work array TAB must be stored in another array (P).

At the end of the file, the value of NB is -9 to inform the EUROPLEXUS program that the loading is finished.

12.5 ADVECTION-DIFFUSION "LOADS" (JRC)

Object:

This option enables the user to enter generalized loads for advection-diffusion calculations (see also keyword "ADDF" in Section A). These 'loads' include:

- imposed time-dependent temperatures
- imposed time-dependent heat fluxes
- imposed time-dependent heat generation
- imposed time-dependent heat convection
- imposed time-dependent heat radiation
- imposed time-dependent external pressure
- imposed time-dependent velocity
- imposed zero velocity
- imposed velocity parallel to a plane
- imposed velocity parallel to a line

Syntax:

```
"ADDF"  
$ "TIMP" . . . $  
$ "FLUX" . . . $  
$ "QGEN" . . . $  
$ "CONV" . . . $  
$ "RADI" . . . $  
$ "PRES" . . . $  
$ "VELO" . . . $  
$ "BLOQ" . . . $  
$ "VPLA" . . . $  
$ "VLIN" . . . $
```

The single sub-options are described below.

Comments:

The word "ADDF" is the first key-word of the option . It should be used only once in the EUROPLEXUS data set.

12.5.1 PRESCRIBED TEMPERATURE

Object:

To prescribe time-dependent nodal temperatures in an advection-diffusion calculation.

Syntax:

```
"TIMP"  nti*( "NOEU" /LECTURE/  
              "TPOI" ntp*(T,t))
```

nti

Number of groups of nodes using the same time function to describe a prescribed temperature.

/LECTURE/

List of nodal point indexes.

ntp

Number of (Temperature, time) couples used for the time evolution.

T

Temperature.

t

Associated time.

Comments:

Remember to dimension sufficiently for the number of groups (nti) and of time points (ntp), see Section A.

12.5.2 PRESCRIBED HEAT FLUX

Object:

To prescribe a time-dependent normal heat flux on element faces in an advection-diffusion calculation.

Syntax:

```
"FLUX"  ntf*( "NELE" nel /LECTURE/  
            "TPOI" ntp*(flux,t))
```

ntf

Number of groups of nodes using the same time function to describe a prescribed edge heat flux.

nel

Number of elements in the group.

/LECTURE/

The 4*nel nodes defining element faces, in the form nel * (n1, n2, n3, n4).

ntp

Number of (flux, time) couples used for the time evolution.

flux

Heat flux.

t

Associated time.

Comments:

Remember to dimension sufficiently for the number of groups (ntf) and of time points (ntp), see Section A.

12.5.3 PRESCRIBED HEAT GENERATION

Object:

To prescribe a time-dependent volumetric heat generation in an advection-diffusion calculation.

Syntax:

```
"QGEN"  ntq*( "FIRS" n1 "LAST" n2  
              "TPOI" ntp*(Q,t))
```

ntq

Number of groups of elements using the same time function to describe a prescribed volumetric heat generation. Each group must contain elements with consecutive indexes.

n1

Index number of the first element in the group.

n2

Index number of the last element in the group.

ntp

Number of (heat gen., time) couples used for the time evolution.

Q

Volumetric heat generation.

t

Associated time.

Comments:

Remember to dimension sufficiently for the number of groups (ntq) and of time points (ntp), see Section A.

12.5.4 PRESCRIBED CONVECTIVE HEAT TRANSFER

Object:

To prescribe a convective heat transfer condition in an advection-diffusion calculation.

Syntax:

```
"CONV"  ntb*( "NELE" nel /LECTURE/  
              "TPOI" ntp*(H,T,t))
```

ntb

Number of groups of elements using the same time function to describe a prescribed time-dependent convective heat transfer on one edge.

nel

Number of elements in the group.

/LECTURE/

The 4*nel nodes defining element faces, in the form nel * (n1, n2, n3, n4).

ntp

Number of (heat transf. coef., flux, time) triples used for the time evolution.

H

Heat transfer coefficient.

T

Temperature.

t

Associated time.

Comments:

Remember to dimension sufficiently for the number of groups (ntb) and of time points (ntp), see Section A.

12.5.5 PRESCRIBED RADIATION HEAT TRANSFER

Object:

To prescribe a radiation heat transfer condition in an advection-diffusion calculation.

Syntax:

```
"RADI"  nrad*( "NELE" nel /LECTURE/  
              "TPOI" ntp*(Hr,T,t))
```

nrad

Number of groups of elements using the same time function to describe a prescribed time-dependent radiation heat transfer on one face.

nel

Number of elements in the group.

/LECTURE/

The 4 * nel nodes defining element faces, in the form nel * (n1, n2, n3, n4).

ntp

Number of (radiation heat transf. coef., flux, time) triples used for the time evolution.

Hr

Radiation heat transfer coefficient.

T

Temperature.

t

Associated time.

Comments:

Remember to dimension sufficiently for the number of groups (nrad) and of time points (ntp), see Section A.

12.5.6 PRESCRIBED TIME-DEPENDENT PRESSURE

Object:

To prescribe a time-dependent external pressure in an advection-diffusion calculation.

Syntax:

```
"PRES"  npres*( "NELE" nel /LECTURE/  
                "TPOI" ntp*(p,t))
```

npres

Number of groups of elements using the same time function to describe a prescribed time-dependent external pressure on one face.

nel

Number of elements in the group.

/LECTURE/

The 4*nel nodes defining element faces, in the form nel * (n1, n2, n3, n4).

ntp

Number of (pressure, time) couples used for the time evolution.

t

Associated time.

Comments:

Remember to dimension sufficiently for the number of groups (npres) and of time points (ntp), see Section A.

12.5.7 PRESCRIBED VELOCITIES

Object:

To prescribe time-dependent nodal fluid velocities in an advection-diffusion calculation.

Syntax:

```
"VTIM"  nvel*( "NOEU" /LECTURE/  
              "TPOI" ntp*(v,t)  
              $[ "PERP"                      ;  
                "PARA" "ANG1" a1 "ANG2" a2 ]$ )
```

nvel

Number of groups of nodes using the same time function to describe a prescribed time-dependent velocity.

/LECTURE/

Indexes of the nodes in the group.

ntp

Number of (velocity, time) couples used for the time evolution.

v

Velocity.

t

Associated time.

"PERP"

The nodal velocity should be perpendicular to the boundary.

"PARA"

The velocity should be parallel to a line defined by the following angles.

a1,a2

Angles of the velocity direction in spherical coordinates: a1 is the angle between the global x axis and the projection of the velocity vector on the xy plane, a2 is the angle between the z axis and the velocity vector.

Comments:

Remember to dimension sufficiently for the number of groups (nvel) and of time points (ntp), see Section A.

12.5.8 PRESCRIBED PARALLEL VELOCITIES

Object:

To prescribe null nodal fluid velocities or velocities parallel to a plane or to a line in an advection-diffusion calculation.

Syntax:

```
"VELO" $[ "BLOQ" /LECTURE/                ;  
          "VPLA" /LECTURE/                  ;  
          "VLIN" "NOEU" /LECTURE/           ;  
          $[ "PERP"                          ;  
            "PARA" "ANG1" a1 "ANG2" a2 ]$ ]$
```

"BLOQ"

The following nodes will have null velocity at all times.

"VPLA"

The following nodes will have a velocity parallel to a plane tangent to the local boundary.

"VLIN" "NOEU"

The following nodes will have a velocity parallel to a line defined as follows.

"PERP"

The nodal velocity should be perpendicular to the boundary.

"PARA"

The velocity should be parallel to a line defined by the following angles.

a1,a2

Angles of the velocity direction in spherical coordinates: a1 is the angle between the global x axis and the projection.

Comments:

Sub-directives "BLOQ", "VPLA" and "VLIN", if present, should appear in this order.

12.6 SEISMIC-LIKE LOADS FOR USE WITH SPECTRAL ELEMENTS

Object:

To impose seismic-like loads in a domain discretized by spectral elements, for the simulation of earthquakes and wave-propagation problems.

There are two main classes of loadings:

- punctual sources, introduced by the keyword "POIN";
- plane wave sources, introduced by the keyword "PLAN";
- seismic moment sources, introduced by the keyword "SISM".

12.6.1 PUNCTUAL SEISMIC LOAD SOURCES

Object:

This directive specifies punctual sources of loadings for spectral elements. The load is computed as the product of a time function and a function of space:

$$f(x,y,z,t) = h(t) * g(x,y,z)$$

Syntax:

```
"POIN" <"NODP" /LECT/> |[ "BET" ; "COS" ] |
|[ "DELT" ; "SPRE" ; "PRES" ; "SHEA" ] |
"SOUR" "AMP" amp "T0" t0
"BETA" beta "ALFA" alfa
<"X" x0 "Y" y0 <"Z" z0>>
<"NX" nx "NY" ny <"NZ" nz> >
```

"NODP"

The load will be applied to the node specified in the following /LECT/. Only one nodal point must be specified. This directive may be used in alternative to the load point coordinates X, Y, Z.

"BET"

The time function $h(t)$ has the form:

$$h(t) = \text{amp} * (1 - 2 * \text{beta} * (t - t_0)^2) * \exp(-\text{beta} * ((t - t_0)^2))$$

"COS"

The time function $h(t)$ has the form:

$$h(t) = \text{amp} * \cos(\text{Pi} * \text{beta} * (t - t_0)) * \exp(-0.5 * \text{beta}^2 * ((t - t_0)^2))$$

"DELT"

The spatial distribution is punctual (delta function), i.e. the force only acts in the point (node) specified below by (x0, y0, z0). Note that this force is an absolute one, i.e. it does not depend on the nodal volume.

"SPRE"

The spatial distribution is a radial function:

$$g(x,y,z) = \exp(- \text{alfa}^2 * R^2)$$

.

where R^2 is the distance from point (x_0, y_0, z_0) . Note that this force is a volume force, i.e. it is scaled for each node by the nodal volume. All points at the same distance R from (x_0, y_0, z_0) have the same value of $g(x,y,z)$, but note that the direction of the force is NOT radial, but is defined by (n_x, n_y, n_z) .

"PRES"

Similar to SPREAD but the direction is radial, thus there is no need to define (n_x, n_y, n_z) . Like SPRE, this force is volumetric, i.e. scaled by the nodal volume:

$$f_x = h(t) * 2 * \text{alfa}^2 * (x - x_0) * \exp(- \text{alfa}^2 * R^2)$$

$$f_y = h(t) * 2 * \text{alfa}^2 * (y - y_0) * \exp(- \text{alfa}^2 * R^2)$$

$$f_z = h(t) * 2 * \text{alfa}^2 * (z - z_0) * \exp(- \text{alfa}^2 * R^2)$$

"SHEA"

Similar to PRES but the force is directed along the tangential direction (and not along the radial one). Also here (n_x, n_y, n_z) are redundant. Furthermore, this kind of loading is only available in 2D. Like SPRE, this force is volumetric, i.e. scaled by the nodal volume. The components are:

$$f_x = - h(t) * 2 * \text{alfa}^2 * (y - y_0) * \exp(- \text{alfa}^2 * R^2)$$

$$f_y = h(t) * 2 * \text{alfa}^2 * (x - x_0) * \exp(- \text{alfa}^2 * R^2)$$

"SOUR"

Introduces a source, that will be characterized by its parameters AMP, T0 etc.

amp

Amplitude of the time function.

t0

The value appearing in the expression of $h(t)$. This time corresponds to the maximum of the time function: $h(t_0) = \text{amp}$.

beta

The beta value appearing in the expression $h(t)$. It governs the exponential decay of the time function. Note that it is possible to avoid the exponential decay in time of the load by specifying a negative value for beta. In this case, the exponential function is eliminated from the expression of $h(t)$, and the absolute value of beta is used. In the BET case:

$$h(t) = \text{amp} * (1 - 2*\text{abs}(\text{beta})*(t - t_0)^2)$$

.

In the COS case:

$$h(t) = \text{amp} * \cos(\text{Pi}*\text{abs}(\text{beta})*(t - t_0))$$

alfa

The alfa value appearing in the expression $g(x,y,z)$. It governs the exponential decay of the space function.

x, y, z

Coordinates of the source. The load will be applied to the node closest to point x,y,z. If these are given, use of NODP is not permitted.

nx, ny, nz

Define the normal direction for DELTA or SPREAD.

Comments:

These loads are all volumetric (specific) forces, except in the case of DELT. This is in order to avoid mesh dependency of the total energy introduced in the system.

Therefore, amp represents the total force (per unit volume) acting on all affected nodes at time t_0 , and this quantity varies in time according to $h(t)$.

In the special case of DELT, the user is responsible of calibrating the (non-volumetric) force with respect to the nodal mass of the application point.

12.6.2 PLANE WAVE SEISMIC LOAD SOURCES

Object:

This directive specifies loading sources in the form of distributed loadings for spectral elements. The load is applied to all nodal points (typically) along a plane (or a line in 2D) and varies in time according to the same time function in all loaded points (except in the case "DLEN", see below).

Syntax:

```
"PLAN" |[ "BET" ; "COS" ] |
        "SOUR" "AMP" amp "T0" t0
          "BETA" beta <"DLEN" dlen>
          "NX" nx "NY" ny <"NZ" nz>
          "NOEU" /LECTURE/
```

"BET"

The time function $h(t)$ has the form:

$$h(t) = \text{amp} * (1 - 2*\text{beta}*(t - t0)^2) * \exp(-\text{beta} * ((t - t0)^2))$$

"COS"

The time function $h(t)$ has the form:

$$h(t) = \text{amp} * \cos(\text{Pi}*\text{beta}*(t - t0)) * \exp(-0.5*\text{beta}^2 * ((t - t0)^2))$$

"SOUR"

Introduces a source, that will be characterized by its parameters AMP, T0 etc.

amp

Amplitude of the time function at $t = t0$.

t0

The value appearing in the expression of $h(t)$. This time corresponds to the maximum of the time function: $h(t0) = \text{amp}$.

beta

The beta value appearing in the expression $h(t)$. It governs the exponential decay of the time function.

dlen

If this value is given, then the loading model of ELSE is strictly used. In this model the load applied to nodal points is proportional to the time derivative of the function $h(t)$, in order to obtain $h(t)$ as an imposed displacement. The nodal force is then:

$$F = 2 \cdot \rho \cdot c \cdot (dh/dt) \cdot vol \cdot fac$$

.

where ρ is the density, c is the wave propagation speed, vol is the nodal volume and fac is a scaling factor, defined as:

$$fac = 2 / (dlen * W)$$

.

where W is the Gauss-Lobatto weight. The physical meaning of $dlen$ is the maximum height of the spectral elements strip in which the load is prescribed. Note, however, that in this case the following restriction applies (coming from the ELSE implementation): the line of nodal points affected by the load must coincide with the one given in ELSE (typically, an horizontal row of nodes, in a region where the mesh must be structured). Therefore, this option makes only sense for strict comparisons between ELSE and Ahnse.

nx, ny, nz

Define the direction for the loading source.

/LECT/

Nodal points in which the loading acts. The code does NOT require that the given nodes actually lie on a line or a plane (3D). The user may therefore specify a non-linear or non-planar source if so desired.

Comments:

These loads are volumetric (specific) forces. This is in order to avoid mesh dependency of the total energy introduced in the system.

Therefore, amp represents the total force (per unit volume) acting on all affected nodes at time t_0 , and this quantity varies in time according to $h(t)$.

12.6.3 SEISMIC MOMENT LOADS

Object:

This directive specifies loading sources in the form of seismic moment loadings for spectral elements. The load is applied to a 'fault' defined by a series of nodes, introduced by the keyword NOEU. The program verifies the alignment of the given nodes along a line in 2D and on a plane in 3D. As a special case, a single fault node can be specified (pointwise loading).

A slipping vector must be defined by keywords SX, SY, SZ. The user can also specify the normal to the fault by the keywords NX, NY, NZ. If the normal is missing, the program assumes it coincident with the slipping vector.

The load applied to each nodal point varies in time according to the distance from the hypocenter, defined by its coordinates X, Y, Z. The time delay in each point depends upon the above mentioned distance and on the speed of transversal wave propagation. This speed is automatically evaluated by the code, or can alternatively be imposed by the keyword VRUP.

Syntax:

```
"SISM" |[ "BET" ; "COS" ]|
        "SOUR" "AMP" amp "T0" t0
            "BETA" beta
            "SX" sx "SY" sy <"SZ" sz>
            <"NX" nx "NY" ny <"NZ" nz>>
            "X" x "Y" y <"Z" z>
            <"VRUP" vrup>
            "NOEU" /LECTURE/
```

"BET"

The time function $h(t)$ has the form:

$$h(t) = \text{amp} * (1 - 2*\text{beta}*(t - t_0)^2) * \exp(-\text{beta} * ((t - t_0)^2))$$

"COS"

The time function $h(t)$ has the form:

$$h(t) = \text{amp} * \cos(\text{Pi}*\text{beta}*(t - t_0)) * \exp(-0.5*\text{beta}^2 * ((t - t_0)^2))$$

"SOUR"

Introduces a source, that will be characterized by its parameters AMP, T0 etc.

amp

Amplitude of the time function at $t = t_0$.

t0

The value appearing in the expression of $h(t)$. This time corresponds to the maximum of the time function: $h(t_0) = \text{amp}$.

beta

The beta value appearing in the expression $h(t)$. It governs the exponential decay of the time function.

sx, sy, sz

Components of the slipping vector.

nx, ny, nz

Components of the normal vector. If omitted, the normal direction is assumed coincident with the slipping vector.

x, y, z

Coordinates of the ipocenter.

vrup

Velocity of propagation of loading (by default, this is the speed of transverse wave propagation).

NOEU /LECT/

Nodal points in which the loading acts. Unlike in the PLAN case, the code requires that the given nodes actually lie on a line or a plane (3D).

Comments:

The present model has been borrowed from the ELSE code of CRS4 and is mesh-dependent in that the applied loadings depend on mesh size. A slightly modified, mesh-independedent implementation is also available in EUROPLEXUS, and is activated by choosing a negative value for VRUP.

12.7 NEW CONSTANT LOADS

Object:

This directive allows to introduce constant forces or moments during the whole computation.

Syntax:

```
FCTE  NODE /LECT/ $[ FORC f ; MOME m ]$ VECT x y z
```

/LECT/

Concerned nodes.

f

Force modulus.

m

Moment modulus.

x y z

Components of the vector or moment-vector.

Comments:

Three vector components must be specified, even in 2-D calculations. The vector is used to determine the direction of the applied force or moment. It does not need to be unitary, since it is automatically normalized immediately after reading.

12.8 IMPOSED TIME-DEPENDENT LOADS

Object:

This directive allows to introduce time-dependent forces or moments during the transient computation. The time-dependency of the load is specified by a function (see directive **FONC** on page E.15 and following ones).

Syntax:

```
FIMP  NODE /LECT/  $[ FORC f ; MOMO m ]$ VECT x y z NUFO nf
```

/LECT/

Concerned nodes.

f

Force modulus (base value, to be multiplied by the time function).

m

Moment modulus (base value, to be multiplied by the time function).

x y z

Components of the vector or moment-vector.

nf

Number of the function defining the time-dependency of the imposed load.

Comments:

Three vector components must be specified, even in 2-D calculations. The vector is used to determine the direction of the applied force or moment. It does not need to be unitary, since it is automatically normalized immediately after reading.

The defined load A is multiplied by a coefficient $C(t)$ which varies in time and is interpolated from the specified table:

$$Q(t) = A * C(t)$$

12.9 DYNALPY LOADS

Object:

This directive allows to introduce time-dependent dynalpy loads in 1-D elements of type TUBE or TUYA.

Syntax:

```
FDYN  NODE /LECT/ PZER p0 COEF c VECT x y z ELEM e
```

/LECT/

Concerned node. Only one single node is possible to define.

p0

Reference pressure.

c

Coefficient.

x y z

Components of the vector or moment-vector.

e

Number of the concerned element.

Comments:

Three vector components must be specified, even in 2-D calculations. The vector is used to determine the direction of the applied load. It does not need to be unitary, since it is automatically normalized immediately after reading.

These loads are only related to 1-D elements of type TUBE or TUYA and are defined as:

$$\underline{F} = c(p - p_0)\underline{v}$$

where p is the current pressure in the specified element.

13 GROUP G—PRINTOUT AND STORAGE OF RESULTS

Object:

The **ECRITURE** directive enables the user to specify the requested printouts and data storages during a computation (including data saving for successive restart).

The **REGION** directive enables to define certain “regions” of the mesh, on which the printout of results will then be performed.

The **MEASURE** directive enables to request the printout of various “measurements” on the current geometrical mesh.

Syntax:

```
< ECRITURE . . . /CTIM/
    < NOPO ; POIN /LECTURE/ >
    < NOEL ; ELEM /LECTURE/ >
    < FICH . . . > >

< REGION ( 'nom region' . . . ) >

< MEASURE ... >
```

Comments:

The keyword **ECRI** must only appear once in an input sequence. It allows to specify the values to be printed on the listing file, and the nodes and elements for which these values must be printed.

Furthermore, the directive allows to define which results files should be produced, in view of a subsequent post-processing.

If the keyword **ECRI** is absent, a printout is performed for all the time steps, all the nodes and all the elements of the mesh.

In the following subsections, first a general description of the **ECRI** directive is given. Then, all the optional keywords (**NOPO** ...) and optional sub-directives (**FICH** ...) are described below. Output regions created by the directive **REGI** are described on page G.100. Finally, the **MEAS** directive is described on page G.105.

13.1 SELECTIVE PRINTOUTS ("ECRITURE")

Object:

The "ECRITURE" directive can be used to select specific quantities to be printed out in the output listing at user-chosen times. It allows also to choose the nodes and elements for which the quantities will be printed.

The various quantities are associated to nodes or elements as described on page G.30 and following ones.

Syntax:

"ECRITURE"

```
< "COOR" > < "DEPL" > < "VITE" >  
< "ACCE" > < "FINT" > < "FEXT" > < "FLIA" >  
< "FDEC" > < "CONT" > < "EPST" > < "ECRO" >  
< "ENER" > < "MCVA" > < "MCVC" > < "MCVS" >  
< "FAIL" > < "VFCC" >
```

COOR

Printout of nodal coordinates.

DEPL

Printout of nodal displacements.

VITE

Printout of nodal velocities. In ALE cases, both fluid and grid velocities are printed out.

ACCE

Printout of nodal accelerations.

FINT

Printout of nodal internal forces.

FEXT

Printout of (total) nodal external forces.

FLIA

Printout of nodal forces due to liaisons (coupled links: LINK COUP).

FDEC

Printout of nodal forces due to decoupled links: LINK DECO.

CONT

Printout of element stresses.

EPST

Printout of element total strains.

ECRO

Printout of element material parameters (hardening for solids, pressure and density for fluids, etc.).

ENER

Printout of energies.

MCVA

Printout of nodal quantities related to multicomponent fluids: pressure, density, temperature, sound speed and mass fractions. Note that this type of printout is incompatible with MCVC and MCVS.

MCVC

Printout of conserved variables (nodal quantities) related to multicomponent fluids: partial densities (ρ_i) of the various components i , momentum ($\rho \underline{u}$) (each spatial component separately), energy (ρE). Note that this type of printout is incompatible with MCVA.

MCVS

Printout of secondary variables (nodal quantities) related to multicomponent fluids: total density (ρ), total pressure p , sound speed c , pressure derivative ($\frac{\partial p}{\partial(\rho e)}$), absolute temperature (T), pressure derivative ($\frac{\partial p}{\partial(\rho_i)}$) for each component, mass fraction (μ_i) for each component. Note that this type of printout is incompatible with MCVA.

MCFL

Printout of MC numerical fluxes: partial densities (first two components), momentum (each spatial component separately) and total energy. In case of FLSR fluid-structure interaction, print out also the list of blocked MC fluxes (in the form of node couples).

MCEF

Printout of MC numerical "external" fluxes: partial densities (first two components), momentum (each spatial component separately) and total energy.

MCMU

Printout of MC "MUSCL" conserved variables (2nd order in space and time): partial densities (first two components), momentum (each spatial component separately) and total energy.

MCVM

Printout of MC volumes (at n , $n+1/2$ and $n+1$) and masses (at $n+1/2$, $n+1$ and $n+3/2$).

FAIL

Printout of reached failure level of the elements. This is only available if the element erosion model has been activated by means of the **FAIL** keyword in the problem type declaration, see page A.30. A value of 0 indicates a virgin element, a value of 100 indicates a completely failed element, an intermediate value indicates the ratio (in per cent) between the number of failed Gauss points and the total number of Gauss points in the element.

VFCC

Printout at each selected output time of "element" quantities related to cell-centred Finite Volumes: various volume-related quantities and conserved variables.

Comments

The keyword "ECRITURE" should only appear once in an input data sequence. Keywords "COOR", "DEPL", etc. should immediately follow the "ECRI" keyword.

Warning:

If none of the preceding keywords is specified, nothing will be printed.

In a standard calculation (not a restart), EUROPLEXUS always prints the last computed time step.

Take care when choosing output frequencies, because the size of listing files may grow very fast.

Note that the results file of type "ALICE" allows to re-construct a listing. You may therefore choose to print out the bare minimum. Later on, if additional results need to be printed, you may do so by re-reading the "ALICE" file (which must have been specified, of course).

13.2 PRINTABLE QUANTITIES

13.2.1 NODE-RELATED QUANTITIES

For each chosen node, one may ask to print:

- current coordinates;
- displacements;
- velocities;
- accelerations;
- internal forces;
- total external forces;
- external reaction (coupled link) forces for the nodes subjected to “coupled” conditions (see LINK COUP);
- external reaction (decoupled link) forces for the nodes subjected to “decoupled” conditions (see LINK DECO);
- multi-component flow-related data (finite volumes).

Coordinates For each chosen node, the code prints X,Y or R,Z o X,Y,Z.

Displacements, velocities, accelerations and forces The chosen nodes are grouped by increasing number of degrees of freedom (d.o.f.). First all the nodes with 1 d.o.f. are printed, then those with 2 d.o.f.s, etc.

13.2.2 ELEMENT-RELATED QUANTITIES

For each chosen element and each integration point one may ask to print:

- the stress components (SIG);
- the deformation (EPST);
- the hardening parameters (ECR).

Stresses and material parameters The stress tensor and the total deformation tensor are related to the element, and independent from the material.

The material parameters, contained in the ECR table, sre independent of the element, and are only function of the material.

The choices done in EUROPLEXUS for these two quantities are detailed in the following pages.

13.3 STRESSES AND DEFORMATIONS

For a given element type, the stress components have always the same meaning, for whatever material is assigned to the element. On the contrary, the hardening values (ECR) are strictly related to the chosen material, and do not depend upon the element type.

The stress tensor is stored and printed in vector form, and is printed for each integration point of the element.

Remark 1:

In 2D, it will be necessary to distinguish the axisymmetric case (AXIS) from the plane strain (DPLA) and plane stress (CPLA) cases.

Remark 2:

For continuum-like elements, the stresses are written in the global reference frame, while for the other types of elements (shells, beams, bars) they are expressed in a local frame attached to the element.

Remark 3:

Instead of computing a bending moment, one computes a “bending stress” (sigf), that may be directly compared with the membrane stresses. This bending stress is related to the bending moment as follows:

$$\text{Moment} = E * I * K_{hi} \quad K_{hi} : \text{curvature}$$

$$\text{sigf} = E * (h/2) * K_{hi} \quad h : \text{thickness}$$

$$\text{Hence:} \quad \text{Moment} = 2 * (I / h) * \text{sigf}$$

$$\text{For a shell:} \quad \text{Moment} = (h * h / 6) * \text{sigf}$$

2D ELEMENTS

BARR and PONC

These elements may only work in traction and compression. There is just one stress component (scalar).

COQU

This element works in membrane and bending. There are 4 stress components per element, expressed in a local reference frame.

The first direction of the local frame is along the element from node 1 to node 2. The second, located in the mesh plane, is normal to the first. The third direction is such that the reference (u, v, w) be right-handed.

sig(1) : membrane (u)	sig(3) : bending (v)
sig(2) : membrane (w)	sig(4) : bending (w)

COQC

Also this element works in membrane and bending. But besides the 4 preceding stress components, there is a fifth one for the shear, which is treated elastically.

sig(1) : membrane (u)	sig(3) : bending (v)
sig(2) : membrane (w)	sig(4) : bending (w)
sig(5) : shear	

CONTINUUM ELEMENTS

The stress components are expressed in the global frame. For a calculation in plane stress, there are three stresses expressed in the (x, y) frame. For an axisymmetric calculation or a plane strain calculation there are 4 stress components, expressed in the frame (x, y, z). The z direction is the normal to the mesh plane, and such that (x, y, z) be right-handed.

1) CPLA:

$$(\text{sig}) = \begin{pmatrix} \text{SIG}(1) & \text{SIG}(3) \\ & \\ \text{SIG}(3) & \text{SIG}(2) \end{pmatrix}$$

2) AXIS or DPLA:

$$\begin{pmatrix} \text{SIG}(1) & \text{SIG}(3) & 0 \end{pmatrix}$$

$$(\text{sig}) = \begin{pmatrix} & & & \\ \text{SIG}(3) & \text{SIG}(2) & 0 & \\ & & & \\ 0 & 0 & \text{SIG}(4) & \end{pmatrix}$$

3D ELEMENTS

BR3D

Like for the BARR and PONC elements in 2D, these elements may only work in traction and compression. The stress tensor has just one component.

POUT

This element works in traction, torsion and bending. There are always 4 stresses per Gauss point, expressed in the local frame.

The first direction of the local frame (u) is along the element, from node 1 to node 2. The second (v) is in the plane defined by u and the local vector V, on the same side as V. The third one (w) is deduced from the others.

Here, due to beam assumptions, the bending stresses are expressed in the frames (u, v) and (u, w):

$$\begin{array}{ll} \text{sig}(1) : \text{traction (u)} & \text{sig}(3) : \text{bending (u,v)} \\ \text{sig}(2) : \text{torsion (u)} & \text{sig}(4) : \text{bending (u,w)} \end{array}$$

Important:

In order to determine the local state of the beam, only the moments and the deformation have a sense! It is therefore mandatory to estimate the moments starting from the stresses, by the following relation:

$$M = \sigma \frac{I}{h}$$

The value of σ is read on the listing, I and h are specified in the input data set (see Chapter C1). The deformations are read directly from the listing.

For an elastic calculation ONLY, it is then possible to compute the stresses in any point of the cross-section.

COQ3 and COQ4

These elements work in membrane and bending. There are always 6 stress components per Gauss point, expressed in a local frame.

For the triangular elements COQ3, the first direction of the local frame (u) is along the first side of the element, from node 1 to node 2. The second (v) lies on the element plane, such that node 3 is on the positive side.

Because of shell hypotheses, the stresses are expressed in the (u, v) frame.

The quadrangular elements COQ4 are composed by 4 triangles:

1-2-3 3-4-1 1-2-4 3-4-2

Each of these triangles has a local reference frame as defined above. If the quadrangle is a parallelogram, the 4 local frames are identical.

The 4 Gauss points of element COQ4 are at the centers of the triangles mentioned above. If the element has an irregular shape, the stresses at the various Gauss points will not be directly comparable.

sig(1) : membrane (u)	sig(4) : bending (u)
sig(2) : membrane (v)	sig(5) : bending (v)
sig(3) : membrane (uv)	sig(6) : bending (uv)

Continuum elements

The stresses are expressed in the global frame (x, y, z).

$$(\text{sig}) = \begin{pmatrix} \text{SIG}(1) & \text{SIG}(4) & \text{SIG}(6) \\ \text{SIG}(4) & \text{SIG}(2) & \text{SIG}(5) \\ \text{SIG}(6) & \text{SIG}(5) & \text{SIG}(3) \end{pmatrix}$$

13.3.1 TOTAL DEFORMATIONS

The tensor of total deformations is the dual of the stress tensor. Its structure is therefore the same as that of the stresses (see the previous Section).

13.4 MATERIAL PARAMETERS (“ECROU”)

All internal variables pertaining to the different materials are stored in the ECR table. Initially reserved only for the hardening parameters, this table has been considerably enlarged, as more complex materials have been implemented in EUROPLEXUS.

Only the simplest materials use just the first hardening quantities. For the others, the meaning of the ECR components are described within each material law description (see page C.100 and following ones).

ECR	shells	continua (solids)	continua (fluids)
ECR(1)	V.M. membrane	Pressure	Pressure
ECR(2)	V.M. memb. + bend.	Von Mises	Density
ECR(3)	plast. deform.	plast. deform.	-

Remarks:

The equivalent plastic deformation (ECR(3)) is only printed for elasto-plastic calculations.

The Von Mises criterion for the shells is expressed as:

$$\text{sig}(\ast) = \text{SQRT}(\text{sig}(\text{m}) \ast \text{sig}(\text{m}) + (\text{alpha} \ast \ast 2) \ast \text{sig}(\text{f}) \ast \text{sig}(\text{f}))$$

with $\text{sig}(\text{m})$ and $\text{sig}(\text{f})$ Von Mises stresses in membrane and bending and $\text{alpha} = 2/3$ by default.

The Von Mises criterion for the beams is expressed as:

$$\text{sig}(\ast) = \text{SQRT}(\text{ap} \ast \text{press} \ast \text{press} + \text{am} \ast \text{sig}(1) \ast \text{sig}(1) + \text{at} \ast \text{sig}(2) \ast \text{sig}(2) + \text{af} \ast (\text{sig}(3) \ast \text{sig}(3) + \text{sig}(4) \ast \text{sig}(4)))$$

The $\text{sig}(\text{i})$ are defined above, and press is the internal pressure, if the beam is a pipe. The coefficients ap (pressure), am (membrane), at (torsion) and af (bending) are computed by EUROPLEXUS according to the type of beam, of the existence or not of a curvature, etc.

13.5 TIME CHOICE (PROCEDURE /CTIME/) FOR THE PRINTOUTS

Object:

The /CTIME/ procedure, described in the introduction (see page INT.57) is used to specify when printouts should take place during a computation.

Comments:

If nothing is specified, the printouts are performed for all time steps.

If the keyword "NUPA" is used, do not forget to dimension adequately by means of the word "MNTT" as described on page A.100.

If the keyword "TIME" is used, do not forget to dimension adequately by means of the word "MTT" as described on page A.100.

Be aware that printout times specified via "TFRE" or "TIME" are rounded to the closest time unit, that can be chosen via the "OPTI TION" directive.

Warning:

Be careful in the choice of your printouts if you do not want to produce unnecessarily large listings. In general, it is advisable to use graphics post-processing in order to analyse the results, instead of reading values on a listing.

It is useful to know that the output file of the results ("FICH" "ALICE"), enables to print selected results on a listing after completion of a calculation. Therefore it is advisable to print only the bare minimum, although it will be necessary to read the results file again, in order to output interesting things after a calculation has been completed.

13.6 NODES OR ELEMENTS TO BE PRINTED

Object:

The user can choose the nodes and/or elements where he wants to print the results.

Syntax:

```
$["NOPOINT" ; "POINT" /LECTURE/ ]$  
$["NOELEM" ; "ELEM" /LECTURE/ ]$
```

"NOPOINT"

No point is printed.

"POINT"

Selection of the points which have to be printed.

"NOELEM"

No element is printed.

"ELEMENT"

Selection of the elements which have to be printed.

/LECTURE/

List of the points or elements for which the results are printed.

Comments:

The two options "POINT" and "NOPOINT" are mutually exclusive, and the same is true for the two options "ELEMENT" and "NOELEM".

If none of the two options is specified, the results are printed for all nodes and for all elements.

13.7 RESULT FILES

Object:

This directive is aimed at creating files for the post-processing of the computation results or a saving file to restart the calculation. The following file types are available:

- SAUVER file, (saving file for successive restart)
- ALICE file, (postprocessor: EUROPLEXUS)
- ALIT (ALICE TEMP) file, (postprocessor: EUROPLEXUS)
- SPTAB file, (postpr.: SUPERTAB (old I-DEAS interface))
- TPLLOT file, (postprocessor: TPLLOT)
- XPLOT file, (postprocessor: TPLLOT)
- K2000 file, (postprocessor: CAST3M)
- AVS file, (postprocessor: AVS and old ParaView)
- PLOT-MTV file (postprocessor: PLOT-MTV)
- UNIV file (postprocessor: I-DEAS)
- MED file (compatible with many softwares)
- TABL file (a simple formatted table)
- PVTk file (postprocessor: PARAVIEW VTK format)
- POCH file (for Pochhammer-Chree post-treatment by EPX)

The SAUVER file is used to restart the calculation. It is described in detail on page G.110.

CAST3M is the product of CEA (Cf. <http://www-cast3m.cea.fr>), SUPERTAB is a commercial software by SDRC, TPLLOT is a software by JRC, AVS is a commercial product by Advanced Visual Systems, PLOT-MTV is a public utility (2D plotting only), I-DEAS is a commercial software by SDRC. The MED format is a format co-developed by the CEA and EDF which is compatible with many softwares. PARAVIEW is an open-source multi-platform application designed to visualize data sets of size varying from small to very large (see www.paraview.org).

Note that, besides the I-DEAS interface (UNIV keyword) described hereafter, another version is available, that had been independently developed by the CESI (formerly ENEL) group, and which is described on Page G.72.

Syntax:

```
< FICHER      |[ "SAUV" <ndsauv> <"PROT" 'maclef'> <"LAST"> </CTIME/> ;

               "ALICE" <"FORMAT"> <"SPLIT"> <ndgrap>    /CTIME/ ;

               "ALIT"   <"FORMAT">           <ndalic>    /CTIME/ ...
                               ... < "POINT" /LECTURE/ > ...
                               ... < "ELEM"  /LECTURE/ >    ;
```

```

"SPTAB" <ndspta> /CTIME/ ;

"TPLOT" <"FORMAT"> <ndtplo> /CTIME/ ...
... "DESC" 'dddddd' ...
... < "POINT" /LECTURE/ > ...
... < "ELEM" /LECTURE/ > ;

"XPLOT" <"FORMAT"> <ndxplo> /CTIME/ ...
... "DESC" 'dddddd' ...
... < "POINT" /LECTURE/ > ;

"K2000" < $[ "FORM" ; "XDR" ; "BINA" ]$ > <"SPLIT"> ...
... <ndcast> /CTIME/ ...
... < "POINT" /LECTURE/ > ...
... < "ELEM" /LECTURE/ > ...
... <"SHEL" vx vy vz /LECTURE/ >
<"CHAMELEM">
<"VARI" < "DEPL"> <"VITE"> <"FEXT"> <"ACCE"> <"MCXX">
<"SIGN"> <"ECRN">
<"CONT"> <"EPST"> <"ECRO">
<"ECRC" /LECT/>
> ;

"AVS" "FORMAT" <"PRVW"> <ndavs> /CTIME/
<"VARI" <"DEPL"> <"VITE"> <"FEXT"> <"ACCE"> <"MCXX">
<"CONT"> <"EPST"> <"ECRO"> <"XLVL">
<"ECRC"> > ;

"PMTV" "FORMAT" <npmtv> /CTIME/
<"VARI" <"DEPL"> <"VITE"> <"SIGN"> <"ECRN"> > ;

"UNIV" <FORMAT> $["CURR" ; "OBSO"]$ <nuniv>
/CTIME/ ;

"MED" /CTIME/
< "POINT" /LECTURE/ >
< "ELEM" /LECTURE/ > ;

"TABL" <ndtabl> /CTIME/
"VARI" nv * (
| [ "COOR" "COMP" ic "NOEU" /LECT/ ;
"DEPL" "COMP" ic "NOEU" /LECT/ ;
"VITE" "COMP" ic "NOEU" /LECT/ ;
"ACCE" "COMP" ic "NOEU" /LECT/ ;
"FINT" "COMP" ic "NOEU" /LECT/ ;

```

```

"FEXT" "COMP" ic "NOEU" /LECT/ ;
"CONT" "COMP" ic "GAUS" gp "ELEM" /LECT/ ;
"ECRO" "COMP" ic "GAUS" gp "ELEM" /LECT/ ;
"EPST" "COMP" ic "GAUS" gp "ELEM" /LECT/ ;
"FONC" ifon ]| ) ;

"PVTK" < $["FORM" ; "FOLD"]$ > <ndpara> /CTIME/
<"PINB"> <"MPI">
<"GROU" |[ "AUTO" ; nobj*("OBJE" <'groupname'>
$[ "GAUS" ngaus ; "GAUZ" ngauz ]$ /LECT/) ]| >
<"VARI" <"DEPL"> <"VITE"> <"FEXT"> <"ACCE"> <"MCXX">
<"SIGN"> <"ECRN"> <"RISK"> <"FAIL"> <"VCVI">
<"CONT"> <"EPST"> <"ECRO"> <"XLVL">
<"FLIA"> > ;

"POCH" <"FORMAT"> <"SPLIT"> <ndpoch> /CTIME/ ...
... "NLIN" nl * ( /LECT/ ) ...
... "VARI" $[ "DEPL" ; "VITE" ; "ACCE" ]$ ;

]| >

```

FORMAT

If this keyword is present, the file will be formatted, otherwise it will be unformatted ("BINA") (but only where both possibilities exist).

XDR

Only for K2000 file if this keyword is present, the K2000 file will be independant of hardware and operating systems. This is the default option for K2000 file.

ALICE

A file of results is written in the standard ALICE format. This file can be read again by the EUROPLEXUS or ALICE programs.

SPLI

Split the ALICE or K2000 (formatted file only) or Pochhammer-Chree results into several files, one for each time instant, instead of producing just a single, big file. Useful e.g. to produce animations of results, which require typically many tens or even a few hundred time instants and to overcome the file size limitations that hold under some operating systems (e.g., under 32 bit MS-Windows maximum file size is 2 GB).

ALIT

A file of results is written in the ALICE format as a function of time. This file will only contain the results at the nodal points and elements defined with the keywords POIN and ELEM. This file can be read again by the EUROPLEXUS or ALICE programs.

SPTAB

An output results file in the SUPERTAB universal file format is produced (old I-DEAS version).

TPLOT

A file of results is written in the standard TPLOT format. This file can be further processed by the TPLOT program.

XPLOT

A file of results is written in the XPLOT format. This file can be further processed by the TPLOT program.

K2000

A file of results is written in the CAST3M format. The default mode of K2000 output is **XDR**. It can be read by CAST3M by using the keyword **RESTITUER**. The file format is the standard format of a CAST3M file obtained with the **SAUVER** directive. It is mandatory in this case to indicate the list of the nodes for which the results have to be stored (possibly **TOUS**). The directive **CHAMELEM** is optional. The values defined on the elements (stresses, hardening quantities, strains) are only stored if this keyword is specified. Note that these element values are averaged on the element GPs (or on the GPs of a specific fibre of the element: see **K2FB** option) and are affected either to the nodes or to the barycentre of the element (see **K2CH** option).

CHAM

This keyword introduces the the definition of “chamelems” in the K2000 results file. If it is omitted, the latter will only contain the selected “champoints”, defined on the nodes chosen by directive **POIN** above.

VARI

Alternatively to **CHAM**, one can use the directive **VARI** which allows finer-grain control over the quantities effectively stored. Each nodal quantity (**DEPL**, **VITE**, **FEXT**, **ACCE**, **MCXX** in case of multicomponent gases, **SIGN** + **ECRN** in case of spectral elements, **VCVI** in case of finite volumes) and each element quantity (**CONT**, **ECRO**, **EPST**) may be specified separately. Furthermore, one may specify exactly which components of the ECR vector are to be stored, via the **ECRC** /**LECT**/ directive. This mechanism allows to greatly reduce the size of output files in large complex 3D calculations.

DEPL

Nodal displacements.

VITE

Nodal velocities.

FEXT

Nodal external forces (including reactions).

FINT

Nodal internal forces.

ACCE

Nodal accelerations.

MCXX

Nodal variables for multicomponent fluids (pressure, density, temperature, sound speed and component mass fractions).

SIGN

Nodal stress components (only for spectral elements).

ECRN

Nodal hardening variables (only for spectral elements).

VCVI

Velocity in the centre of finite volumes (CEA formulation, only available for PVTk).

RISK

Risk limit for eardrum failure and death (only available for PVTk).

FAIL

Failure level. Not active debris get a failure level of -1.0. (only available for PVTk).

XLVL

Level set of XFEM (only available for PVTk).

FLIA

Nodal forces due to liaisons (links); (only available for PVTk).

SHEL

Option which enables to print in the K2000 output file or PVTk output file the stress (CONT) and strain (EPST) of shells according to specific axes rather than local axes (default). vx vy vz are the coordinates of the vector which is projected onto the shells read in /LECTURE/ in order to define the first direction of the posttreatment axes. The third posttreatment direction is identical with the third one in local axes. This definition of the posttreatment axes is made in the initial configuration and is not updated ; that means this option is only relevant for small strains. This command may be repeated as many times as needed. This option is available only for DST3, DKT3, T3GS and Q4GS shells.

AVS

A set of result files (see note below) are written in the AVS format. For the moment, only a formatted output is available for AVS, so the **FORM** option is actually mandatory in this case.

PRVW

The AVS files are modified to be imported into PARAVIEW software (see note below). This is only compatible with older versions of ParaView (less than 2.9 or so). For newer versions, use the PVTk type of output, which produces files in the VTK format.

UNIV

A results file in “universal” (I-DEAS) format is to be written. This file may be of two types: “current” or “obsolete”. By default it is of “current” type.

CURR

The universal results file will be of type “current”. This is the default.

OBSO

The universal results file will be of type “obsolete”.

PVTk

A set of result files (see note below) are written in the PARAVIEW format (**.pvd** and **.vtu** files). This is the VTK format, compatible with the newer versions of ParaView. By default, use is made of the library **LIB_VTK_IO**, written by Stefano Zaghi (see http://stefano.zaghi.googlepages.com/lib_vtk_io), which allows to produce either ASCII or binary output formats. However, if the (obsolete) keyword **FOLD** is specified in place of **FORM**, then formatted output is produced without making use of the **LIB_VTK_IO** library (note that in this case only ASCII format, not binary format, is possible).

PINB

The geometry of the pinballs are written in a separate **vtu**-file.

MPI

MPI calculations only. One set of **vtu**-files is written for each subdomain, to avoid centralizing data from all subdomains on one thread before writing.

GROU

Groups can be defined automatically (using keyword **AUTO**) or manually by entering **nobj** objects defined with the **OBJE** keyword (which must be repeated exactly **nobj** times). A name can be optionally given to each manually defined group. Each separate group is written as output files (see comments below for syntax). For manually defined groups, the number of the Gauss point **ngaus** or the number of the layer **ngauz**, which is used for output, can be defined for each group with **GAUS** or **GAUZ** respectively. The logic of the

VTX format is not the same as the one of CAST3M and EUROPLEXUS. If an element is contained in several groups, the element will be written to the output files several times. This element is also repeated in the output of ParaView. If present, only the elements (and nodes) defined by the **GROU** keyword are written to the output.

ndgrap

Number of the logical unit of the ALICE file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is **.ALI**. For the special case of split ALICE files, see comments below.

ndalic

Number of the logical unit of the ALICE TEMPS file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is **.ALT**.

ndspta

Number of the logical unit of the SPTAB file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is **.SPT**.

ndtplo

Number of the logical unit of the TPLOTT file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is **.TPL**.

ndxplo

Number of the logical unit of the XPLOTT file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is **.XPL**.

ndcast

Number of the logical unit of the CAST3M file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is **.K20**. For the special case of split CAST3M files, see comments below.

ndavs

Number of the logical unit of the AVS file(s) or file name in quotes. If omitted, the program chooses a (base) file name by default (see page A.27). The default extension is **.AVS**. Split files are generated for this type of output. See comments below.

ndpara

Number of the logical unit of the PARAVIEW file(s) or file name in quotes. If omitted, the program chooses a (base) file name by default (see page A.27). The default extension is **.pvd**. This file contains links to files with the data (vtu-format, extension **.vtu**). See comments below.

npmtv

Number of the logical unit of the PLOT-MTV file or file name in quotes. If omitted, the program chooses a (base) file name by default (see page A.27). Split files are generated for this type of output. The default extension is .MTV. See comments below.

nuniv

Number of the logical unit of the universal file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is .UNV.

ndtabl

Number of the logical unit of the table file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is .TAB.

/CTIM/

This procedure is described in the introduction (page INT.57). If the keywords **NUPA** or **TIME** are used, do not forget to dimension adequately by means of keywords **MNTI**, **MTTI** (see also page A.100).

DESC 'dddddd'

Six-character descriptor to identify the run for the TPLOT (or XPLOT) data-base. Note that this item is in text format, therefore it must be enclosed in quotes. When loading data on the TPLOT database, a prefix **PL** is automatically placed in front of this descriptor. Thus, the full descriptor on the database will be **PLdddddd**. For XPLOT, the prefix is **XL**, so the full descriptor will be **XLdddddd**.

POIN /LECTURE/

List of nodes for which the results have to be stored for successive treatment by TPLOT or XPLOT.

ELEM /LECTURE/

List of elements for which the results have to be stored for successive treatment by TPLOT.

MED

A file of results is written in the MED format. This file will only contain the results based on the nodes and the elements selected with the keywords **POIN** and **ELEM**. For the selected nodes, nodal displacements, nodal velocities, nodal accelerations, nodal external forces (including reactions), nodal internal forces are stored. Only elements with one integration point and the **POUT** elements are treated. For these elements, stresses and strains fields are stored. Do not forget to create the MED file before with the keyword **MEDE** (see page A.30).

TABL

A file of results is written in a simple text file in tabular form. This type of output is meant to monitor just a few variables, say a nodal displacement and an element stress component, but with great precision. Results are always written formatted, and with full double precision (16 significative digits). The table contains one line per storage station (typically, each time step). The first two columns of the table always contain the current time step (**npas**) and the current time (**t**). The following columns contain the chosen variables. Obviously, no more than just a few variables can be specified, in practice, else the length of the table line would become excessive. The keyword **VARI** introduces the chosen variables. Their number is given by **nv** and thereafter exactly **nv** variable specifications must be given, chosen among the following possibilities: **COOR** for nodal coordinates, **DEPL** for nodal displacements, **VITE** for nodal velocities, **ACCE** for nodal accelerations, **FINT** for nodal internal forces, **FEXT** for nodal external forces, **CONT** for elemental stresses, **ECRO** for elemental hardening quantities, **EPST** for elemental total strains, **FONC** for the value of a specified function. The keyword **COMP** introduces the component, **GAUS** the Gauss point, **NOEU** the node and **ELEM** the element. Note that the **/LECT/** directives must specify one single node or element. The value **ifon** after **FONC** specifies the function identifier (see the **FUNCTION** directive).

ndpoch

Number of the logical unit of the Pochhammer-Chree file(s) or file name in quotes. If omitted, the program chooses a (base) file name by default (see page A.27). The default extension is **.poc**.

NLIN

Introduces the number of “lines” along which the results will be sampled for the successive Pochhammer-Chree post-treatment. Each line is formed by an ordered sequence of nodes and is defined by the following **/LECT/**.

Comments:

The keyword **FICHIER** is not compulsory. If it is used, the last step is systematically saved.

Do not forget to define the logical unit(s) of the file(s), on the control cards. As an alternative, EUROPLEXUS accepts the name of the file enclosed in quotes.

If one does not pay attention, result files may become very bulky, because the total number of computed time steps is often very large. It is then advisable to estimate the total number of steps from the stability step computed by the program, and then choose a reasonable number of storages on the **ALICE** file. It is also possible to obtain a smaller results file by using the **ALICE TEMPS (ALIT)** directive: in this case only the variables relative to the nodes and elements given in directives **POIN** and **ELEM** will be stored.

It is rare that one needs more than a dozen of time stations to plot the deformed shapes of the mesh — in this case the **ALICE** directive will be used. It is also infrequent that one needs

more than a few hundred points to plot curves as a function of time — it is suggested to use the directive **ALIT** which allows to obtain a file reduced to just the selected points. Therefore it is possible to specify a much larger number of saving stations on an **ALIT** file than on an **ALIC** one.

For **ALICE**, the **SPLI** option allows to split the results into many small files, one for each stored time instant, rather than producing just one big file. This option is useful for very large computations and/or for producing animations, which typically require many saved instants. In this case, **ndgrap** is just the base name of the output files. The single file names are automatically given progressive numbers (**_0001**, **_0002**, etc.) appended to the name, which identify the storage index. A file with suffix **_0000** is also produced, which contains the initial mesh topology.

In order to post-treat these splitted results with **EUROPLEXUS**, proceed exactly as if the results would be in a single file, but remember to specify the **SPLIT** keyword in the **RESU ALIC** directive, see page ED.20.

XPLOT storage is intended to perform pseudo-1D visualization of data in a 2D or 3D run. A curvilinear abscissa is built up passing through the nodes defined in **POIN /LECT/**. Then, nodal and element quantities are stored as a function of this abscissa. By using the **TPLOT** program, the relevant quantities can then be plotted along the curvilinear abscissa (either initial or current). Note that nodal quantities (displacements, velocities, etc.) are stored without modification at the specified nodes. Element quantities, however, (such as stresses and hardening parameters), that are usually defined only at points internal to the elements, are first extrapolated to the nodes, then stored. Currently, the extrapolation consists of simply: 1/ averaging each quantity over each element (by using the values at the different Gauss points), 2/ averaging all neighbour element contributions to obtain nodal values. Neighbour elements to a node are those elements that contain that node.

Note also that, in the extrapolation process, only certain types of elements are considered. For example, shell or beam elements are rejected, because the mean value of the stress components on all the Gauss points is likely to be meaningless for such elements. Only the following element types are considered: **TRIA**(2), **CAR1**(8), **CAR4**(9), **CUBE**(11), **CUB6**(13), **PR6**(20), **TETR**(21), **PRIS**(27), **FLU1**(52), **FLU3**(53), **FL23**(64), **FL24**(65), **FL34**(66), **FL35**(67), **FL36**(68), **FL38**(69), **Q41** (71), **Q42** (72), **Q41N**(73), **Q42N**(74).

In order to read with **CAST3M** a file written by **EUROPLEXUS**, use the following **CAST3M** commands:

1/ Formatted file:

```
OPTI REST FORM 'file';  
REST FORM;  
. . . (post-treatment commands)
```

2/ Unformatted file:

```
OPTI REST BINA 'file';  
REST BINA;  
. . . (post-treatment commands)
```

3/ XDR file:

```
OPTI REST 'file';  
REST ;  
. . . (post-treatment commands)
```

For K2000, note that two syntaxes are possible. With the ‘old’ syntax (the keyword **VARI** does not appear), all nodal quantities are always stored. Furthermore, all element quantities are stored if **CHAM** appears. The components of the ECR table which are stored depends in this case from the material: they are the same components of ECR that are printed on the listing.

With the new syntax (the keyword **VARI** appears) only the specified nodal quantities, element quantities and ECR table components are actually stored.

Note that, strictly speaking, it is only possible to produce an output file for K2000 when the input mesh has also been produced (and read into EUROPLEXUS) in this format. However, if this is not the case but you still desire to postprocess your EUROPLEXUS calculation with K2000, consider transforming your mesh in K2000 by the option **K2MS** (see Section H, output options). Beware, however, that this may require some manual intervention and in any case the obtained mesh will be less flexible to use than a “real” K2000 mesh (e.g., no sub-objects etc.).

In the case of standard AVS storage, a set of files is written, one for each stored variable. The files basename is given by **ndavs**. If **ndavs** as given by the user contains the extension **.avs** or **.AVS**, this extension is removed by the program. An extension of the form **.VARI.N.inp** is then automatically provided by EUROPLEXUS. Here **VARI** is the variable type (**DEPL**, **VITE**, ...) and **N** an integer counter which is automatically incremented by 1 at each successive storage in time.

Recall that AVS storage can also be requested interactively, i.e. during an interactive execution of EUROPLEXUS (See Group A, Interactive (Foreground) Execution).

In the case of AVS storage modified for usage with PARAVIEW, elements are split into groups with same element topology and same material law. One file is stored for each group of elements at each successive storage in time. This file contains geometry and both nodal fields and elemental fields required by the user. It is named from the name given by **ndavs**, with its

extension removed. An extension of the form `_N1_N2.inp` is then added to the name. `N1` is the number of the element group and `N2` is an integer counter as for standard AVS files. If `ndavs` is not defined, the base of the EUROPLEXUS input file name is used to build AVS-PARAVIEW file names.

If a directory name is provided for AVS-PARAVIEW files with the `OPNF` directive, files are written in this directory, excepted if `ndavs` represents a name with full path. In this latter case, the above given directory is ignored.

The AVS-PARAVIEW format is only readable with older versions of PARAVIEW (less than 2.9). For newer versions of PARAVIEW, the PARAVIEW output (see `PVTK`) with a `.pvd` and `.vtu` files is recommended.

Automatic group definition for PARAVIEW output (with keyword `GROU AUTO`) consists in the same splitting of elements as described above for the AVS-PARAVIEW output.

The PLOT-MTV output is only available in 2D and for spectral elements. the program will only include spectral elements and nodes in these files. A separate file with the extension `.mtv` is produced for each nodal quantity and at each selected storage time.

The `SIGN` keyword produces storage of 4 nodal stress (6 in 3D) components (`SGXX`, `SGYY`, `SGZZ`, `SGXY`, `SGYZ`, `SGZX`). The `ECRN` keyword produces storage of 2 nodal hardening quantities: the hydrostatic stress (`HYDR`) and the Von Mises stress (`VMIS`).

When using `SPTAB` output format, a file named `sptab.param` must exist in the current directory, containing the following key-words useful to declare the variables to print out: `DISP`, `VELO`, `ACCE`, `INTF`, `EXTF`, `MCVAR`, `MCVEL`, `MCMFR`, `FLVAR`, `SCUB8`, `ECOQI`.

For `CAST3M`, the `SPLI` option allows to split the results into many small files, one for each stored time instant, rather than producing just one big file. In this case, the chosen output type `MUST` be formatted (`FORM`). This option is useful for very large computations and/or for producing animations, which typically require many saved instants. In this case, `ndcast` is just the base name of the output files. The single file names are automatically given progressive numbers (`_0001`, `_0002`, etc.) appended to the name, which identify the storage index. A file with suffix `_0000` is also produced, which contains the initial mesh topology.

In order to post-treat one of these splitted results with `CAST3M`, proceed as follows: choose the instant to be treated, say number 3 i.e. the third storage performed; then, produce a file by concatenating the 'zero' file (`_0000`) and the file for the chosen instant; finally, read and post-process the resulting file with `CAST3M`. In this example:

```
cat mytest_0000.k20 mytest_0003.k20 >out.k20
```

```
K2000
```

```
opti rest form 'out.k20'; rest form; ...
```

Warning :

Be careful: output files may become very large, because the total number of the time steps computed is often large. Therefore it is better to estimate that number from the stability step computed by the program. Then, the user can choose a reasonable number of writings on the output files.

The user seldom needs more than a dozen storage stations ('cases') to draw deformed structures and no more than fifty points to draw time functions.

13.8 POST-PROCESSING BY I-DEAS MASTER SERIES

Object:

This is aimed at creating files for the post-processing of computation results by I-DEAS master series.

This model is part of the models developed by the CESI team (formerly at ENEL, Milano) in collaboration with JRC.

I-DEAS is a commercial software by SDRC.

Syntax:

```
< "FICHIER" <FORMAT> "IDEA" ndidea  /CTIME/
                                ... < "POINT" /LECTURE/ > ...
                                ... < "ELEM"  /LECTURE/ > ...
    < "VARI" < "DEPL" "VITE" "FEXT" "ACCE" "MCXX" >
              < "CONT" "MESH" "MCVA" "MCVE" "MCMF" >
              < "FLVA" "ECOQ" >                >                >
```

"FORMAT"

If present, the file is formatted, otherwise it is unformatted.

"IDEA"

An output results file in the I-DEAS universal file format is produced (new version).

ndidea

File name in quotes of the universal output file for I-DEAS.

"POIN" /LECT/

Same meaning as for the other result files, see page G.70.

"ELEM" /LECT/

Same meaning as for the other result files, see page G.70.

"CTIM" .. "CONT"

Same meaning as for the other result files, see page G.70.

"VARI"

Introduces the list of variables to be stored.

"DEPL"

The I-DEAS universal file in output will contain the displacements.

"VITE"

The I-DEAS universal file in output will contain the velocities.

"FEXT"

The I-DEAS universal file in output will contain the external forces.

"ACCE"

The I-DEAS universal file in output will contain the accelerations.

"MCXX"

The I-DEAS universal file in output will contain the MC variables.

"CONT"

The I-DEAS universal file in output will contain the stresses.

"MESH"

The I-DEAS universal file in output will contain the mesh data (only node and element datasets).

"MCVA"

The I-DEAS universal file in output will contain the MC conserved variables: pressure, density, internal energy, maximum pressure, minimum pressure, temperature.

"MCVE"

The I-DEAS universal file in output will contain the MC velocities: x, y, z velocity components, velocity modulus, sound speed and Mach number.

"MCMF"

The I-DEAS universal file in output will contain the MC mass fractions.

"FLVA"

The I-DEAS universal file in output will contain the fluid finite elements variables (FLxx family with FLUT material): pressure, density, internal energy, max pressure, min pressure, sound speed.

"ECOQ"

The I-DEAS universal file in output will contain the COQI element ECR variables: hydrostatic pressure, Von Mises, plasticity flag, current yield stress.

Comments:

The general comments of page G.70 apply to the I-DEAS results file as well.

When using "IDEA" output format, the default options for the selection of the results are: whole geometry (i.e. all nodes/elements are treated if "POIN", "ELEM" are omitted), no variable (i.e. only the variables specified in the "VARI" option are stored).

Element output in I-deas universal file format is only available for FLxx, MCxx, CUB8, COQI, CQD3, CQD4 elements at the moment.

13.9 OUTPUT REGIONS

Object:

This directive enables the printing of physical values within a given region.

A region is defined by the list of the elements which compose it. The region could correspond to a GIBI object.

Syntax:

```
"REGION" ( 'nom region'
    $[ "RMAS" ; "VOLU" ; "BARY" ; "DIMX" ; "DIMN" ;
        "VEMX" ; "VEMN" ; "VMOY" ; "IMPU" ; "ECIN" ;
        "WINT" ; "WEXT" ; "PDV" ; "WINJ" ; "RESU" ;
        "IRES" ; "ECRG" ; "ECRM" ; "EMAS" ; "FLIR" ;
        "TOUT" ]$

    |[      /LECTURE/      ;
        "POIN" /LECTURE/  ]| )
```

nom region

Name of the region given by the user (in apostrophees).

RMAS

Mass (components, computed via XMEL).

VOLU

Volume.

BARY

Center of gravity (barycentre) of the region.

DIMX

Maximum displacement (absolute) of the region (vector), only components 1 to 3.

DIMN

Minimum displacement (absolute) of the region (vector), only components 1 to 3.

VEMX

Maximum velocity (absolute) of the region (vector), only components 1 to 3.

VEMN

Minimum velocity (absolute) of the region (vector), only components 1 to 3.

VMOY

Average velocity (components).

IMPU

Impulse (components).

ECIN

Kinetic energy (norm and components).

WINT

Internal energy.

WEXT

Work of external forces applied to the nodes of the region.

PDV

Work of pressure forces in ALE for a stand-alone domain.

WINJ

Injected energy (only for material EAU).

RESU

Resultant of the external forces applied at the nodes.

IRES

Impulse corresponding to the above resultant.

ECRG

For each component of ECR (in fact, for the first 10 components), sum of the values on the Gauss points of the region.

ECRM

Average of the ECR over the region.

EMAS

Mass (scalar, computed via the element's mass XM0).

FLIR

Resultant of the force due to LINK/LIAI applied at the nodes.

TOUT

All possible physical values are required.

LECTURE

List of the elements composing the region.

POIN /LECTURE/

List of the nodes composing the region.

Comments:

The computation takes place within the elements.

It is possible to have elements which belong to several regions.

If the region is only known by its nodes (it has not been defined in the directive "GEOM"), the only possible balances are WEXT, RESU and IRES. In this case, it is mandatory to use the keyword "POIN" before the /LECTURE/ procedure, to avoid confusion between nodes and elements.

If the region is nothing else but the whole structure, the values of WINT and ECIN are the same as those printed in the energy balance.

The physical values of the regions are computed during the printing.

Important: restart calculation (see page ED.10)

There is no problem if the computed quantity does not depend on masses (WINT). If the physical values are dependant on the masses (ECIN BARY VMOY IMPU RMAS VOL PDV), the computation will be correct only if the masses are constant during the EUROPLEXUS computation. In fact, in order to lighten the file of the results (FICHIER ALICE), only the initial masses are copied out. There is no problem concerning a Lagrangian computation. For an Eulerian or A.L.E computation, the masses change. Therefore, the physical values will not be correct.

All this happens during a restart; if the physical values are computed during a normal (non restart) EUROPLEXUS run, all the results are correct.

13.10 MEASUREMENTS

Object:

This directive enables the printing of various types of “measurements” on the current geometrical mesh.

Normally, it is called from the input file and the requested measurements are then printed on the listing. However, the same directive (same syntax) is available also from the command line during an interactive execution (see page A.25). For this reason, the present directive **must** be terminated by the keyword **TERM**, as shown below in the syntax. In case of interactive use, the requested measurements are printed on the console window, not on the listing.

Syntax:

```
MEASURE  $[ ELEM e           ;
          NODE n             ;
          OBJE /LECT/        ;
          EMIN /LECT/        ;
          EMAX /LECT/        ;
          DIST /LEC1/ /LEC2/ ]$ TERM
```

ELEM e

Returns information about the chosen element **e**: its type, the associated nodes (with their coordinates), its size, its mass and the list of objects or groups to which it belongs.

NODE n

Returns information about the chosen node **n**: its coordinates, its mass, the list of elements to which it belongs and the list of objects or groups to which it belongs.

OBJE /LECT/

Returns information about the chosen object **/LECT/**: the associated elements, the associated nodes, its size and its mass.

EMIN /LECT/

Returns information (see **ELEM** above) about the “smallest” element among those belonging to the object specified in the following **/LECT/**. Use **LECT tous TERM** to get the smallest element in the whole mesh. As smallest element, we consider the one having the shortest (non-zero) intra-nodal distance among its nodes (and thus most likely the shortest element characteristic length as far as stability is concerned).

EMAX /LECT/

Returns information (see **ELEM** above) about the “largest” element among those belonging to the object specified in the following /LECT/. Use **LECT tous TERM** to get the largest element in the whole mesh. As largest element, we consider the one having the largest intra-nodal distance among its nodes (and thus most likely the longest element characteristic length as far as stability is concerned).

DIST /LEC1/ /LEC2/

Returns the minimum (intra-nodal) distance between the two objects defined in /LEC1/ and /LEC2/.

TERM

Indicates the termination of measurements. This keyword is necessary so that the present directive may be used also in interactive mode.

Comments

An example of this directive is as follows:

```
MEASURE
  ELEM 123
  NODE 74
  ELEM 1
  OBJE LECT toto TERM
  EMIN LECT tous TERM
  EMAX LECT 1 2 4 TERM
  DIST LECT toto TERM LECT tata TERM
TERM
```

When used interactively, the code pauses for input after each sub-command, waiting for the next sub-command. To exit from the **MEAS** directive, give the final **TERM** command.

13.11 SAVING FILE FOR SUCCESSIVE RESTART

Object:

This keyword creates a saving file and, in conjunction with the keyword REPR (to be used in a subsequent run), allows splitting a computation in two or more parts.

This directive replaces the old (deprecated and obsolescent) directive SAUV (group A, see page SR.20).

The results are saved on a file (saving file) at times specified by the user. Each saving corresponds to a number or position on the file (1, 2, 3 etc.), from which a restart of the computation can be carried out in a successive run (see directive REPR on page SR.30).

Syntax:

```
FICH  SAUV <ndsauv>  <PROT 'maclef'>  <LAST>  </CTIM/>
```

nbansav

Number of the saving file or name of the file in quotes. If completely omitted, the code will assume the default file name <basename>.sau where <basename> is the root of the input file name (i.e. without extension .epx).

PROT

Keyword entering a protection on the saving file.

'maclef'

Key of up to 8 characters, enclosed in apostrophees. In order to restart the computation from that file, the instruction REPR must contain the keyword PROT with an identical key.

LAST

This keyword indicates that the saving file should contain just one saving station, corresponding to the last saved time station in the present calculation. In other words, each new saving station replaces the former one, if any. This allows to obtain a saving file of the smallest possible size. However, restarting from an intermediate time is obviously not possible in this case: the only possibility to restart the calculation will be REPR ... POSI 1 (see page SR.30).

/CTIM/

The /CTIM/ procedure (see page INT.57) is used to specify the saving times via a step frequency (FREQ), a time frequency (TFRE), a list of steps (NUPA) or a list of times (TIME). If NUPA or TIME are used, do not forget to dimension accordingly using MTTI or MNTI, respectively, see page A.100. Note that the code always saves the last step of the calculation (if the run is terminated normally), irrespective of the frequency chosen. Therefore, if one is only interested in getting the possibility to continue the calculation further on, simply omit the /CTIM/ procedure.

Comments:

The keyword PROT is not compulsory. If it is not used, there is no protection (this is equivalent to a key of 8 blanks).

If a unit number is used for nbansav, the saving file and its number must have been defined before on the control cards.

A first saving station (position number 1) containing some header data is always produced at the initial time (step 0 of the calculation). Of course, it is normally meaningless to restart from this time station, unless the LAST keyword has been specified (see above), because it would be the same as starting the calculation anew from the initial time. On the contrary, if LAST has been specified, the only possibility for restart is to use the first time station which, in this case, will contain the data of the last saving performed (not the first one in general).

Examples:

Assume a calculation performs 4994 time steps to arrive at its final time. The following saving directives are accepted:

- FICH SAUV 'myfile.sau' FREQ 1000 saves data for restart on the indicated file every 1000 steps. The following six saving stations are produced: 1 (step 0), 2 (step 1000), 3 (step 2000), 4 (step 3000), 5 (step 4000) and 6 (step 4994, i.e. the last step).
- FICH SAUV FREQ 1000: same as above but the saving file has the default name <basename>.sau.
- FICH SAUV 'myfile.sau' LAST FREQ 1000 saves data for restart on the indicated file every 1000 steps, by always re-writing the previous saving. Only one saving station is thus present on the saving file (assuming that the run terminates normally): 1 (step 4994, i.e. the last step). If the run would fail, say, at step 2500, the saving file would also contain one station (step 2000).
- FICH SAUV FREQ 10000: the saving file has the default name <basename>.sau. Only one saving station is produced (assuming that the run terminates normally): 1 (step 4994, i.e. the last step). If the run would fail, say, at step 2500, no (useful) saving time station would be available (there is still one saving station, but it is at step 0).

- **FICH SAUV LAST:** the saving file has the default name **<basename>.sau**. Only one saving station is produced (assuming that the run terminates normally): 1 (step 4994, i.e. the last step). If the run would fail, say, at step 2500, no (useful) saving time station would be available (there is still one saving station, but it is at step 0).
- **FICH SAUV:** the saving file has the default name **<basename>.sau**. Two saving stations are produced (assuming that the run terminates normally): 1 (step 0) and 2 (step 4994, i.e. the last step). If the run would fail, say, at step 2500, no (useful) saving time station would be available (there is still one saving station, but it is at step 0).

14 GROUP H—OPTIONS

Object:

These keywords give the additional options of the computation. They can be grouped as:

- options associated with time-steps;
- options associated with dampings;
- options for finite elements;
- output options;
- returning to default options;
- options for an advection-diffusion computation;
- options for ALE computations in structures;
- options for debugging purposes;
- options to declare FANTOME certain elements;
- options related to the treatment of shocks and impacts;
- options for FSA (fluid-structure interactions of the ALE type);
- options for node-centered finite volumes;
- options for multiphase, multicomponent fluid flows;
- options for the automatic rezoning in ALE computations;
- options for cell-centred finite volumes;
- options for “LIAISONS”/LINKS (connections);
- options for graphical rendering;
- options for mesh-adaptive computations;
- options for strain rate filtering.

Syntax:

"OPTION"

OPTION

Announces that one or several options will be specified.

Comments:

The keyword "OPTION" may appear more than once in the EUROPLEXUS data.

The following sub-instructions may appear in any order.

The different options are described on the following pages.

14.1 OPTIONS RELATED TO THE TIME-STEP

Object:

Additional options are given to provide optimum time stepping.

Syntax:

```

< |[ "PAS" |[ "UTILISATEUR" ; "AUTOMATIQUE" ]| ;
    "PARTITION" $["PLIN" ; "PNOL"]$ <PLOG> ]| >
< "DTVAR"   dtvar       >
< |[ "NOTEST" ; "TEST" ]| >
< "STABILITE"          >    < "STEL" >
< "NOCRITIC" /LECTURE/ >
< "CSTAB"      cstab    >
< "PASMINI"    pasmi    >
< "DTFORCE"    dtfor    >
< |[ "STEP" "IO" ; "STEP" "IOT" ; "STEP" "LIBR" ]| >
< "TION"       tionor   >
< "DTML"              >
< "DTBE"            kdtbe >
< "DIVG"            divg  >
< "DTDR"            dtdrop >

```

PAS UTILISATEUR

The time step is prescribed by the user (see also keyword **CALCUL**). Note that this option cannot be chosen in the case of impacts (the time increment may be limited by the program in case of an impact).

PAS AUTOMATIQUE

The time-step is determined by the program (see also keyword **CALCUL**). This is the default, i.e. if neither **PAS UTIL** nor **PAR** are specified the time step is automatically computed by the code.

PARTITION

The computation step is partitioned automatically in space (and the step also varies with time), according to the stability step of each element (see also keyword **CALCUL**).

PLIN

In the space partitioning procedure, dofs subjected to any links are treated according to the lowest level among the ones that are linked together. This works only with the `LINK` directive, while conditions imposed by the `LIAI` directive are not treated. The option has no effect in cases without space partitioning.

PNOL

In the space partitioning procedure, dofs subjected to any liaisons or any links (but only of the permanent type) are put in the lowest partition level. This is the default, so this option should not be used, except for changing back from a previously issued `OPTI PLIN`. The option has no effect in cases without space partitioning.

PLOG

In case of space partitioning, a special log file is written `<basename>.plog`. This file contains an output line for each sub-cycle, in contrast to the normal log file, which contains one line for each macro step. The extra information may be quite long but is sometimes useful for debugging. By default no such log file is written.

dtvar

Maximum growth factor of the time step among two subsequent steps in `PAS AUTO`. Default is 2.0.

NOTEST

The energy check and related information is not printed at each step, but only when general printouts are required (see `ECRI`).

TEST

The energy check and related information is printed at each step. This is the default.

STABILITE

The energy check and related informations is printed only if `EUROPLEXUS` reduces the time step.

STEL

At each step for which a printout is produced, the stability steps for all elements are printed out.

NOCRITIC

The elements defined in the following `/LECTURE/` will not be considered in the calculation of the critical time step by `EUROPLEXUS`. In practice, they will be assigned a very large critical step.

cstab

Safety coefficient assumed over the estimated stability (i.e., critical) time step for each element. Default value is 0.8. It is only effective for PAS AUTO or PART. See also the comments below.

PASMINI

The calculation will stop if the time increment becomes less than $\text{dtmax} \times \text{pasmi}$.

DTFORCE

The stability step of the more stringent elements is forced to assume the value dtfor by increasing their mass. This option is dangerous: see the comments below.

STEP IO

During the computation, the time step will be adjusted to exactly fit chosen times for output events such as printouts (see ECRI), storage of data for post-processing (FICH ALIC but not FICH TPLO nor FICH ALIC TEMP nor FICH TABL!), or storage of data for restart (FICH SAUV). Note that TPLLOT, ALICE TEMPS and TABL data storages are not included (use STEP IOT instead). This choice is justified by the fact that TPLLOT, ALICE TEMPS and TABL storage times are usually much more numerous than (normal) ALICE storages, but include only a limited number of nodes and elements. Note that this option has only effect in PAS AUTO or PART, but obviously it has no effect in PAS UTIL.

STEP IOT

Same as STEP IO, but now output events considered for time step adjustment include also TPLLOT, ALICE TEMPS and TABL storages. Note that this option has only effect in PAS AUTO or PART, but obviously it has no effect in PAS UTIL.

STEP LIBR

During the computation, the step will be varied only according to stability limits. No adjusting to output times for printing, etc., will be performed. In this case, if the user chooses a given printout or storage time, the program will perform the action at the first step in which the time is equal or greater than the specified value. In general, the error on time is small since it is of the order of one time increment. This is the default (as opposed to STEP IO).

tionor

Value of time units used for the normalization of selected times and time frequencies for printing and storage. The default is 1.D-6. It is only relevant to the STEP IO option described above. Be aware that the normalization process may only take place if time values are less than $1.E9 \times \text{tionor}$. An error is produced otherwise. This precision should be largely sufficient in practical cases. In fact, this allows to specify a precision of e.g. 1.E-3 times the typical time step, for a computation with up to 1.E6 steps.

DTML

This option chooses a different rule from the standard one to estimate the critical time step of JRC's FLxx fluid elements. The standard rule for FLxx, originating from EUR-DYN, was quite complex and was documented in the report "Implementation of Compressible Fluid Models in PLEXIS-3C", Technical Note No. I.93.86. This rule was found to be inaccurate in some cases. The new rule activated with the present option uses the minimum intra-nodal distance as the characteristic length and the sound speed plus the maximum nodal $(w - v)$ value (mesh velocity minus fluid velocity) as the characteristic speed. This is in accordance with the rule used in CEA's fluid elements. The DTML option can also be invoked to use the minimum intra-nodal distance for calculation of the stability of C27 elements in 3D (by default these elements use an estimation of the element's stretch and shear to compute the element's characteristic length).

DTBE

This option chooses a different rule from the standard one to estimate the critical time step for the POUT element. Three different values can be chosen: `kdtbe = 0` indicates the default version (CEA's formula); `kdtbe = 1` uses an optimized time step (formula for ED01 elements); `kdtbe = 2` considers only the length of the element and disregards the cross section. The default time step used for the POUT element seems to be very conservative. Larger time steps result using the formula for ED01 elements, which is as follows. If the element length L is larger than $\sqrt{3}h$, where h is the element thickness, then the normal expression is used: $\Delta t = L/c$, where c is the sound speed. Otherwise, the element length is corrected: $L_{corr} = L^2/\sqrt{3}h$ and then $\Delta t = L_{corr}/c$.

divg

This options give the possibility to define a value that the energy balance can not exceed. The default is 0.

dtdrop

Define the coefficient `dtdrop`. A warning message is printed on the listing each time the stability is imposed by a finite element and the ratio $\Delta t_2/\Delta t_1$ is smaller than `dtdrop`. The default is 0.3. Some special materials (such as e.g. the JWLS material) used to represent very violent explosions and wave propagations may abruptly reduce the time step in order to preserve stability. In such cases, it may be useful to re-define `dtdrop` to values smaller than the default (e.g., 0.005) in order to avoid too many warning messages on the listing.

Comments:

Options by default : PAS AUTOMATIQUE TEST STEP LIBR.

The calculation stops if the time step becomes too small. The limit value is proportional to `dtmax` (directive CALCUL). By default, `pasmi=0.001`, i.e. the calculation will stop whenever the time step becomes less than $0.001 \times \text{dtmax}$. This option is only active when the old syntax of the CALCUL directive is used. With the new syntax, `pasmin` is redundant because DTMIN directly gives the minimum step (see I.20).

The energy check deals with the energy balance. The value of the stability time step is also printed.

The option **PARTITION** is especially useful when the mesh contains a few very small elements among a large number of bigger ones. In this case, the small elements are paid more attention, without carrying out useless computations on the big ones. This option could be inefficient if used when all elements have nearly the same size, or if there are only a few large elements in the mesh.

Like all explicit programs, EUROPLEXUS requires a sufficiently small time increment in order to ensure the stability of calculations. By default, EUROPLEXUS uses the CFL time step (Courant-Friedrichs-Lévy condition), multiplied by a safety coefficient **CSTAB** = 0.8. However, for very fast phenomena this condition may be insufficient. It is then possible to ensure stability by assuming for **CSTAB** a value smaller than 0.8.

The option **DTFORCE** affects only Lagrangian elements. In an ALE calculation, only the Lagrangian elements (if any) will be considered, and the others will be ignored. Since the mass of elements is modified, it is necessary to check that such modifications do not affect too much the physics of the problem.

To this end, some indications are available on the listing:

- The mass by zones before and after modification;
- The list of the 20 most constraining elements, with the old and the new time step;
- A message **ATTENTION** also appears if the mass of a zone increases more than 10 %.

14.2 OPTIONS RELATED TO THE DAMPINGS

Object:

To enter the dampings.

Syntax :

```
|[  "QUASI" "STATIQUE"  fsys  beta <"FROM" t1> <"UPTO" t2> ;
    "AMORT" "LINE"      betal                                     ;
    "AMORT" "AXIA"      betal                                     ;
    "AMORT" "QUAD"      a2                                       ;
    $[ "HOURG" "VISC"   hvis ; "NOHOURG" ]$                      ]|
```

QUASI STATIQUE

Quasi static computation. A linear damping with a given cut-off frequency is applied.

fsys

Frequency f of the first system mode to be cut off.

beta

Reduced damping coefficient β .

t1

Initial time t_1 at which the quasi-static option starts to operate. By default, this coincides with the initial time of the calculation. See comments below for the use of t_1 and t_2 to define a closed interval or two open intervals.

t2

Final time t_2 until which the quasi-static option operates. By default, this coincides with the final time of the calculation. See comments below for the use of t_1 and t_2 to define a closed interval or two open intervals.

AMORT LINE

Computation with linear damping of high frequencies.

AMORT AXIA

Computation with linear damping of high frequencies, but only for the elements on the symmetry axis (for 2D axisymmetric problems only).

betal

Reduced damping coefficient β_l for linear damping (of type LINE or AXIA).

AMORT QUAD

Computation with quadratic damping.

a2

Coefficient a_2 for the quadratic damping (shock waves).

HOURLG VISC

Anti-hourglass damping on viscous terms.

hvis

Reduced damping for anti-hourglass h_{vis} ($h_{vis} = 0.5$ is suggested).

NOHOURLG

Allows to eliminate the anti-hourglass damping.

Comments :

In the case of the **QUASI STATIQUE** option, $\beta = 1$ corresponds to the critical damping for the frequency f . In fact, one adds an external force F_i^{QS} proportional to the mass M_i and to the particle velocity v_i for each degree of freedom i :

$$F_i^{QS} = -4\pi\beta f M_i v_i = -2\beta\omega M_i v_i$$

where $\omega = 2\pi f$.

In practice, only the product βf is relevant.

Linear damping of high frequencies is only possible for elements of type CAR1, CAR4, TRIA, TUBE, FUN2 and FUN3. This damping allows to eliminate the high-frequency oscillations related to the finite element discretization. In order to obtain the critical damping of a free-free oscillation for each element, take $\beta_l = 1$.

When t_1 is less than t_2 (be these values specified or not) the quasi-static damping acts in the closed time interval $t_1 \leq t \leq t_2$, i.e. in the central part of the transient calculation. However, it is also possible to specify t_1 greater than t_2 : in this case the critical damping acts in the open time interval $t \leq t_2$ (i.e. at the beginning of the calculation) and in the open time interval $t_1 \leq t$ (i.e. at the end of the calculation). This second form of the directive may be useful when one wants to model a structure initially subjected only to gravity loads (with quasi static option so as to rapidly reach the initial static deformed configuration), followed by a dynamic event such as an explosion (without quasi static option), and finally by a stabilization phase (again with quasi static option) so as to rapidly compute the final static deformation. Thus this form of the directive allows to perform the complete analysis of the three phases in just one run of the code, instead of running three separate calculations (each one starting from the results of the previous one) via e.g. the directive **INIT ALIC** (see page E.140).

For the quadratic damping, it is suggested to take $a_2 = 4$.

Quadratic damping is only possible for elements of types CAR1, CAR4, CUBE, PRIS, TRIA and TUBE.

The present linear and quadratic damping models are distinct from the selective damping model (AMOR) described on page C.106, which applies to selected dofs and nodes of a zone specified by the user.

The anti-hourglass damping is currently available only for the elements CAR1 et CUBE. By default, an anti-hourglass damping with $h_{vis} = 0.5$ is affected to a calculation. If the user wants to do a calculation without anti-hourglass damping, he must use the option NOHOURG.

Warnings :

In case of restart, the QUASI STATIQUE damping remains active; to eliminate it, one must specify $\beta = 0$.

Linear damping should be used with care, since it may considerably perturbate the solution. It is advisable not to exceed the value $\beta_l = 0.05$. In case of axisymmetric linear damping, since the concerned elements are usually a few and with a small mass, one may go up to $\beta_l = 0.5$.

14.3 OPTIONS FOR FINITE ELEMENTS AND GEOMETRIC ISSUES

Object:

To introduce optional parameters related to finite elements and geometric issues.

Syntax:

```

<  "DECENT"  |[          "TOTAL" ; "CALC"          ;
                        "IMPOSE" "DCEN" de "DCMA" dm  ]|  >
<  "ROLIM"   rholim                                     >
<  "JAUMAN"                                     >
<  "CODG" < "REFE" zbar >    <SMAL>                                     >
<  "EDSS"                                     >
<  "LFUN"                                     >
<  "P2X2"                                     >
<  "MOMT"   kmtran                                     >
<  "TOLC"   tolc                                     >
<  "HGQ4"   hgq4ro                                     >
<  "CLMT" < "FARF" farf> < " ABSI" absi>                                     >

```

DECENT CALC

A.L.E. only. Upwinding computed by EUROPLEXUS according to the volume covered in one time step with respect to the total volume.

DECENT TOTAL

A.L.E. only. Total upwinding for the mass.

DECENT IMPOSE

A.L.E. only. Prescribed upwinding.

de

Upwinding concerning transport terms.

dm

Upwinding concerning mass fluxes.

ROLIM rholim

ALE/Eulerian only: if the donor element has a density less than `rholim`, the mass and energy fluxes are not considered for this element.

JAUMAN

Large strain computation with JAUMAN's stress tensor.

CODG

Introduces options for calculations with degenerated shell elements (CQDx).

zbar

Parameter defining the position of the reference surface for degenerated shell elements: -1 indicates the lower element surface, 0 the mean surface, +1 the upper surface. By default, $zbar = 0$.

SMAL

Specifies that a small strain model of membrane deformation has to be used for degenerated shell elements, so the thickness of these elements stays constant. By default, a large membrane deformation is assumed and the element thickness is varied accordingly. This option is only useful to compare a solution with an old run done by JRC's SHELL3D.

EDSS

Specifies that certain elements (ED01, FUN2, FUN3) will adopt a small strain, large displacements, large rotations formulation instead of the large strain formulation that is used by default.

LFUN

Specifies that certain elements (FUN2, FUN3) will adopt a fully linear, small strain model: element cross section stays constant and also length stays constant for the calculation of critical time step (which is therefore constant). This option should only be used for debugging purposes and for the study of time integration algorithms (to compare analytical and simplified numerical solutions).

P2X2

This option activates a spatial integration rule for pressure forces in CEA's fluid elements (CUBE, PRIS, TETR) which is equivalent to a 2x2x2 Gauss rule, and is therefore exact also for distorted geometry (e.g. non-planar faces). The standard rule uses a single-point scheme which is under-integrating the function in the presence of distortions. The resulting inaccuracy of pressure force computation leads to the effect that fluid nodes internal to the fluid domain and completely surrounded by a fluid at uniform pressure are not in perfect equilibrium when the surrounding mesh is irregular. Spurious resultant pressure forces cause spurious velocities in the fluid which are non-physical. Although these velocities were usually found to remain relatively small with respect to physical ones in typical applications (explosions etc.), it is generally preferable to avoid them altogether by using the present option, although it is of course slightly more computationally expensive. The standard rule (single-point) is left as a default for compatibility with old input files and applications.

MOMT

This option allows choosing the degree of precision for the spatial integration rule used in the computation of momentum transport forces in Eulerian or ALE calculations using JRC's FL3x fluid elements. The `kmtran` parameter may assume the values 0 (no momentum transport forces at all), 1 (corresponding to single-point integration), 2 (for 2x2x2 spatial integration) or 3 (3x3x3 spatial integration). For distorted geometries only the 3x3x3 rule is exact. The default rule (as used in EURDYN) is the single-point one which is of course the most economical, but unfortunately may lead to spurious mechanisms (appearance of spurious fluid velocities) in some cases, typically when the geometry of the elements is irregular or distorted (e.g., non-planar faces). The mechanisms may rapidly grow and in some cases they completely destroy the numerical computation. In all practical cases investigated so far it was found that a 2x2x2 rule (MOMT 2) is accurate enough and sufficient to prevent the appearance of mechanisms. The cost of this is of the order of 20% to 30% overhead compared with the default, (MOMT 1) option. The MOMT 3 option is exact, but may cause a 100% overhead in computer time. Finally, note that the MOMT 0 option is only to be used for debugging purposes, since computations without momentum transport forces are of course largely inaccurate.

TOLC

This option allows to change the tolerance `tolc` that is used to automatically search for node correspondence, see page C.92. The default behaviour (no OPTI TOLC) is that two nodes are considered to match if their initial positions differ, along each one of the global coordinate axes, by less than 1.E-4 times the “mean” size of the mesh. This mean size is defined as the sum of the sizes of the mesh along each one of the global axes, divided by the space dimension. If `tolc` is explicitly specified, it is retained as the maximum distance between two coincident nodes along the global axes. In this case therefore, the above mentioned mean mesh size is not computed: `tolc` is used directly. Note that, in order to be effective, this option must be specified *before* the directives that might use it, in particular before the LIAI FSA directive.

hgq4ro

Adjusting coefficient for the anti-hourglass rotation stiffness of the Q4GR shell element. The default value of `hgq4ro` is 0.018.

CLMT

This keyword introduces options for the treatment of momentum transport forces in fluid Finite Elements (JRC formulation, i.e. FLxx family of elements). It applies to the CL22, CL2S, CL3I, CL3Q and CL3S element types, associated with either a FLUT material (for far-field conditions) or an IMPE ABSI material (for absorbing boundary conditions).

FARF `farf`

Use `FARF 1` to activate momentum transport forces in CLxx due to far-field conditions, or `FARF 0` to de-activate them. The default is 0, i.e. no momentum transport forces.

ABSI `absi`

Use `ABSI 1` to activate momentum transport forces in `CLxx` due to absorbing (IMPE `ABSI`) conditions, or `ABSI 0` to de-activate them. The default is 0, i.e. no momentum transport forces.

Comments:

A large strain calculation with the `JAUMAN` tensor is only possible at the moment with elements `"CAR1"`, `"CAR4"` and `"TRIA"`.

The upwinding is only effective for a computation with a non-Lagrangian formulation (keyword `"ALE"` or `"EULER"` in the type of problem to deal with , see page A.30).

By default, `EUROPLEXUS` uses the total upwinding (`dm = 1` and `de = 0`).

14.4 OUTPUT OPTIONS

Object:

These options enable the output format to be chosen.

Syntax:

```

< $[ "NOPR" ;
      "PRIN" < "PMESH" > < "PCAST" > < "PCOMP" >
            < "PGRID" > < "PLOAD" > < "PLINK" >
            < "PRESU" > < "PLAW" > ]$ >
<   "DPMA"                                >
< $[ "NWAL" ; "WALI" ]$ >
< $[ "NWSA" ; "WSAU" ]$ >
< $[ "NWTP" ; "WTPL" ]$ >
< $[ "NWXF" ; "WXPL" ]$ >
< $[ "NWAT" ; "WATP" ]$ >
< $[ "NWK2" ; "WK20" ]$ >
< $[ "NWST" ; "WSTB" ]$ >
< $[ "NOEC" ; "ECHO" ]$ >
<   "LOG" nlog                            >
<   "K2FB" k2fibe                        >
< $[ "K2CH" ; "K2GP" ]$ >
<   "K2MS" |[ "MANU" ; "READ" ]| >
<   "DYMS" nobj*("OBJE" /LECT/) >

```

NOPR/PRIN

This option allows to suppress or re-activate a part of the printouts of the following directives.

If one of the keywords PRIN/NOPR is followed by one or more parameters, only the corresponding parts of the listing are activated (or deactivated)

"PMESH" : mesh (nodal coordinates and elements topology)

"PCAST" : detail of the CASTEM objects

"PCOMP" : geometrical complements

"PGRID" : parameters of the ALE rezoning

"PLOAD" : details of the charges

"PLINK" : details of the liaisons/links

"PRESU" : details of the results files

"PLAW" : details of the material laws

See also the comments below.

"DPMA"

Prints nodal and element masses with each general printout. This can be useful to check masses in problems where the mass varies, such as ALE calculations.

NWAL

No printout on the listing of information about each storage of data for ALICE (see "FICH ALIC").

WALI

A line of information containing the time, step number, etc. will be printed on the output listing at each storage of data on the ALICE file (see "FICH ALIC"). This is the default option.

NWSA

No printout on the listing of information about each storage of data for restart (see "SAUV").

WSAU

A line of information containing the time, step number, etc. will be printed on the output listing at each storage of data on the restart file (see "SAUV"). This is the default option.

NWTP

No printout on the listing of information about each storage of data for TPLOT (see "FICH TPLO"). This is the default option, since usually many storages are requested for TPLOT.

WTPL

A line of information containing the time, step number, etc. will be printed on the output listing at each storage of data on the TPLOT file (see "FICH TPLO").

NWXP

No printout on the listing of information about each storage of data for XPLOT (see "FICH XPLO").

WXPL

A line of information containing the time, step number, etc. will be printed on the output listing at each storage of data on the XPLOT file (see "FICH XPLO"). This is the default option.

NWAT

No printout on the listing of information about each storage of data for ALICE TEMPS (see "FICH ALIC TEMPS"). This is the default option, since usually many storages are requested for ALICE TEMPS.

WATP

A line of information containing the time, step number, etc. will be printed on the output listing at each storage of data on the ALICE TEMPS file (see "FICH ALIC TEMPS").

NWK2

No printout on the listing of information about each storage of data for K2000 (see "FICH K2000").

WK20

A line of information containing the time, step number, etc. will be printed on the output listing at each storage of data on the K2000 file (see "FICH K2000"). This is the default option.

NWST

No printout on the listing of information about each storage of data for SUPERTAB (see "FICH SPTAB").

WSTB

A line of information containing the time, step number, etc. will be printed on the output listing at each storage of data on the SUPERTAB file (see "FICH SPTAB"). This is the default option.

NOEC/ECHO

This option allows to suppress or re-activate input data echo in the EUROPLEXUS window.

LOG

Causes a one-line information to be written to standard error file each 'nlog' time steps. The information includes current step number, time, CPU time, critical step, critical element, energy check and mass check. This is useful e.g. to monitor the execution of very long and CPU-intensive runs. Usually, the standard error information will be redirected to a file, e.g. with the Unix command '2>file'. The columns of the log files (S standard calculation, P calculation using partitioning) are described in the table below.

	Description	S	P
STEP	Time step number (main step for Partitioning)	X	X
TIME	Time	X	X
CPU(S)	CPU time used	X	X
DTCRIT	Critical time step used	X	
ELCR	Element with the smallest time step	X	
DELMIN	Time step of the smallest substep		X
MINS	Minimum level factor		X
DE/E	Energy balance per element	X	X
DM/M(NOD)	Mass balance per node	X	X
DM/M(ELE)	Mass balance per element	X	X
DTMX	Maximum time step	X	
EL	Element of the maximum time step	X	
DELMAX	Time step of the main step		X
MAXS	Maximum level factor		X
VITMAX	Maximum velocity	X	X
NODE	Node of the maximum velocity	X	X
ISUBTO	Total number of substeps		X
MAXSTO	Total number of substeps		X
ELSTEP	Number of callings of element routines	X	X

K2FB

Indicates the index of the Gauss Point, along each fiber, for which variables are stored for subsequent K2000 postprocessing. For example, if there are 5 GPs along fibers in the shell elements used in a calculation, then `k2fibe = 1` indicates the GPs closest to one face of the structure, `k2fibe = 5` indicates the GPs closest to the opposite face of the structure, `k2fibe = 3` indicates the GPs on the midsurface of the structure, and so on. Note that this parameter has only effect for shell elements of types ED01, ED41, COQI and CQDx. The default value is `k2fibe = 1`.

K2CH

With this option, the output chamelems for K2000 will be defined for each element at the element nodes, rather than at the element barycenter (default) or at the Gauss points (K2GP option). Note, however, that the computation of values is crude: an average on all GPs is computed, and this value is affected to all nodes of the element (although the contributions to the same node from different elements may be different). The default (without the K2CH option) is to compute an average on all GPs and affect this value to the barycenter of the element.

K2GP

With this option, the output chamelems for K2000 will be defined for each element at the Gauss points, rather than at the element barycenter (default) or at the element nodes (K2CH option). The exact value is affected at each GPs of the element. In case of multilayer plates (CEA-plates: DKT3, Q4GS...) an average on the GPs in the thickness is computed, and each of these values is affected to the corresponding GP on the surface of the element.

The default (without the K2GP option) is to compute an average on all GPs and affect this value to the barycenter of the element.

K2MS

With this option, the code will produce a file containing a series of GIBIANE instructions that, when processed by CASTEM2000, will produce the current mesh in CASTEM2000 format. This option is only useful when the mesh has been produced by a pre-processor different from CASTEM2000 (see also comments below).

MANU

The CASTEM2000 mesh generation commands will use the CASTEM2000 operator MANU. The name of the generated file is pxtok200.dgibi on the current directory

READ

The data for CASTEM2000 will be written on file pxtok200.inp on the current directory. These data are suitable to be read by CASTEM2000 via the READ operator (see also comments below)

DYMS

With this option, the code will produce an input file for LS-DYNA. For each of the `nobj` objects defined by the `OBJE` keyword (which must be repeated exactly `nobj` times), the nodes and elements are written in this file. No material and load definitions are exported.

Comments:

The presence of `OPTI NOPR` immediately after the dimensioning in the input file minimizes the listing file. On the contrary, `OPTI PRIN` maximizes the listing file. It is possible to activate or deactivate the various printouts selectively. For example:

```
OPTI  NOPR  PMESH  PCAST  PLINK
```

will deactivate the printouts relative to the mesh, the CASTEM objects and the liaisons/links. This allows to avoid repeating the commands `NOPR` and `PRIN` within the input file.

In case of re-reading the results file (file ALICE or ALICE TEMPS) the option `NOPR` is taken by default. To have complete printouts, it is sufficient to add `OPTI PRIN` after the keyword `TERM` of directive `DIME`.

The K2MS option can be very useful in the case that an input file for EUROPLEXUS uses a mesh defined in a format different from CASTEM2000, but the user wants to do the post-processing of the calculation by CASTEM2000. This option will produce a file containing data that can be used by CASTEM2000 to generate the desired mesh.

Typically, in such cases one would perform the following steps:

1. - Run the EUROPLEXUS input file with the non-CASTEM mesh, including option K2MS. The calculation can be stopped at step 0 (use VERI or CONV TEKT and then the stop interactive command). This will produce a file of data for CASTEM2000 in either file pxtok200.dgibi or file pxtok200.inp on the current directory.

2. - Run CASTEM2000 on the above mentioned file, to produce a mesh in CASTEM2000 format. See below for examples and details.

3. - Finally, run again EUROPLEXUS by specifying that the input geometrical data are from CASTEM2000 (CASTEM directive). Now, a CASTEM2000 post-processing file can be produced by EUROPLEXUS, because the input is indeed in CASTEM2000 format.

Note, however, that the CASTEM2000 mesh produced by this method will be somewhat special, in that no meaningful subobjects will be generated. Only the global mesh will be accessible as object "mesh".

When the K2MS MANU option is used, the file produced (pxtok200.dgibi) will contain a line for each node, of the form:

```
Pxxxxxx = xcoor ycoor [zcoor];
```

where xxxxxx is the node number (e.g., 00025 for node 25), xcoor, ycoor (and zcoor in 3D) are its coordinates.

For example:

```
P00332 = 1.0000000000000D+01 1.0000000000000D+01 ;
```

Then, for each element there will be a line of the form:

```
Eyyyyyy = manu elem node1 node2 ... ;
```

where yyyyyy is the element number, elem is the element type according to CASTEM2000 (e.g., QUA4 for 4-node quadrilaterals) and node1, node2 etc. are its nodes. For example:

```
E00002=manu QUA4 P00004 P00006 P00005 P00003;
```

The global object will be called mesh. If you need to define sub-objects, use appropriate GIBIANE instructions.

A typical CASTEM2000 command file using pxtok200.dgibi is as follows:

```
(pxtok200.dgibi as produced by EUROPLEXUS) ...
```

```
mesh3 = mesh ELEM 'TRI3';  
mesh4 = mesh ELEM 'QUA4';  
...  
opti sauv 'file';  
sauv mesh;
```

Unfortunately, it has been noted that CASTEM2000 changes the numbering of elements in a mesh generated in this way. The other method (using the READ option) can be used in cases this could cause trouble (which is typically the case if other input directives in the EUROPLEXUS input file use element numbers). Or, alternatively, try using the SORT operator instead of the SAUV operator to save the mesh, as detailed below.

When the K2MS READ option is used, the file produced (pxtok200.inp) contains a simple list of nodal coordinates and element topology (by zones). These data can be read by CASTEM2000 using the READ operator developed at JRC.

To this end, use a command file of the form:

```
...  
mesh = READ 'pxtok200.inp' MESH ELEM;  
mesh3 = mesh ELEM 'TRI3';  
mesh4 = mesh ELEM 'QUA4';  
...  
opti sauv 'file';  
sauv mesh;
```

From the tests performed, it seems that CASTEM2000 maintained the element numbering in this case, but only up to version 9 of the SAUV operator included. For higher versions of the SAUV operator, numbering is generally changed.

In order to try to avoid renumbering, use the CASTEM operator SORT instead of SAUV to save the mesh. The SORT operator is more limited than SAUV (it may only save meshes, for example), but has the advantage that it apparently does not change mesh numbering, and its implementation is somewhat “frozen” in the code, unlike the SAUV operator which evolves constantly.

Recall that a mesh saved with SORT must be read in EUROPLEXUS by the GIBI directive, not by the CAST directive (see page A.30), and that SORT files are formatted by default.

The command file will be in this case of the form:

```
...
```

```
mesh = READ 'pxtok200.inp' MESH ELEM;  
mesh3 = mesh ELEM 'TRI3';  
mesh4 = mesh ELEM 'QUA4';  
...  
opti sort 'file';  
sort mesh;
```

In EUROPLEXUS, the mesh will be read as follows:

```
...  
GIBI 'file' mesh  
...
```

14.5 RETURNING TO DEFAULT OPTIONS

Object:

To set the options relative to a standart computation back to their default values.

Syntax:

```
< "ZERO" >
```

ZERO

Discards any previous options, returning to default values.

Comments:

All the options which have been defined previously are discarded, and the options by default are assumed again.

14.6 OPTIONS FOR AN ADVECTION-DIFFUSION COMPUTATION

Object:

To provide options for an advection-diffusion computation.

Syntax:

```

<  "ADDF" < "GRAV" gravi > < "PSYS" psyst >
      < "ELEM" ielref > < "SORD" nsord >
      < "NGAU" ngau > < "ITER" nitef >
      < "ITEP" niter > < "TOLER" titer >
      < "ADTI" adtime > < "ERRO" errix >
      < "NIMA" nimax >

```

gravi

Acceleration of gravity (default=0.0).

psyst

System pressure, used to remove the singularity of the pressure field solution matrix (default=0.0).

ielref

Index of element in which the pressure is equal to psyst. (default=1)

nsord

When 2, 3 or 4, a Taylor-Galerkin method is used of order 2, 3 or 4, respectively (default=2). When nsord=5, a Least-square, space-time method is used. When nsord=6, a Least-square, Crank-Nicolson method is used.

ngau

Number of Gauss points in each direction for the integration of advection terms, can be 1 or 2 (default=1).

nitef

Number of iterations in the factorization of the consistent mass matrix during the advection phase, can be 1 to 9. (default=3)

niter

Maximum number of iterations for the solution of the system of equations for the pressure phase. If set to null, a direct solution is performed (default=0).

titer

Convergence tolerance for the iterative solution of pressure phase equations (default=0.01).

adtime

Time step fraction.

errix

Tolerance of implicit resolution. Is only used with Least-square method (see nsord above).

nimax

Maximum number of iterations for implicit resolution. Is only used with Least-square method (see nsord above).

14.7 OPTIONS FOR ALE CALCULATIONS IN STRUCTURES

Object:

To provide options for an ALE calculation in structures.

Syntax:

```
< "ALES" |[ "KINT" kintm ; "UPWM" upwm ; "UPWS" upws ]| >
```

kintm

Integration type for momentum transport: 0 means 1x1 (not available for the moment!), 1 means 2x2 (exact for plane problems), 2 means 3x3 (exact for axisymmetric problems). Default is 1.

upwm

Upwind parameter for momentum transport, can be chosen between 0 and 1 (default is 1.0).

upws

Upwind parameter for stress transport, can be chosen between 0 and 1 (default is 1.0).

14.8 OPTIONS FOR DEBUGGING

Object:

To provide options to help in debugging the program (for developers only).

Syntax:

```
< $[ "DUMP" ; "NODU" ]$           >
<   "DPAS" /LECTURE/               >
<   "DPEL" /LECTURE/               >
<   "DPEM"                         >
<   "VIDA" /LECTURE/               >
<   "DPGR"                         >
<   "OLDS"                         >
<   "DPCA"                         >
<   "DPLE"                         >
<   "DPLM"                         >
<   "DPSD"                         >
<   "DPAR"                         >
<   "DPAX"                         >
```

"DUMP"

Prints dump of variables as long as they are initialised in the various routines before starting time integration. Of course, this option tends to produce extremely large output files and is only useful for very small test cases, for program development.

"NODU"

Turns off dumping option.

"DPAS"

The following list enumerates the integration time steps for which extensive information has to be dumped out. A maximum of 200 step indexes can be specified (this dimension is fixed).

"DPEL"

The following list enumerates the elements for which extensive information has to be dumped out. A maximum of 20 element indexes can be specified (this dimension is fixed).

"DPEM"

Prints (on the log file!) tables of available elements and materials in a format suitable for rapid inclusion in this user's manual.

"VIDA"

The following list indicates the indexes of the variables to be dumped (these can range from 1 to the total number of variables, see include MAPORGA), a value of 0 indicates that the contents of the commons has also to be dumped. Note that the commons are dumped at the moment when the directive 'OPTI VIDA LECT 0 TERM' is encountered in the input file, therefore it is suggested to place this directive just before the 'CALC' directive, which starts the time-marching calculation.

DPGR

Prints a table containing the list of all nodes with their grid motion attributes: L for Lagrangian, E for Eulerian, AA for ALE, manually rezoned, AM for ALE, automatically rezoned, AS for ALE, rezoned by "FSS ALE", AZ for ALE, rezoned by "MEAN". The dump is performed after complete processing of the input, immediately before starting the time loop. This allows to check possible changes applied by the program to conditions imposed by the user through the "GRILLE" directive. This option is only active for Eulerian or ALE calculations.

"OLDS"

Specifies that an old model for the VM23 material has to be used in place of the most recent model. The old model was slightly less accurate in elastoplastic cases and was used in the EURDYN programs. This option should only be used for debugging purposes, if a very precise comparison with an old EURDYN calculation is desired.

"DPCA"

Prints on the listing tables of element and material characteristics. For the elements, the NCEL variables are listed in tabular form, for the materials the MATALE and LGEP variables are listed.

"DPLE"

Prints on the listing a table of element characteristics in L^AT_EX input format. This may then e.g. be edited for inclusion in the present User's Manual.

"DPLM"

Prints on the listing a table of material characteristics in L^AT_EX input format. This may then e.g. be edited for inclusion in the present User's Manual.

"DPSD"

In multi-domain calculations, dumps out extra information on the listing file. Furthermore, for each sub-domain a separate log file is produced that reports, at every time station, a line collecting information relevant to the sub-domain. The name of such files is <base_name>_xxx.log, where xxx is the index of the sub-domain (e.g. 012 for the twelfth

sub-domain), and **base_name** is the base name of the test case (without the extension **.epx**). By examining these log files, one is able to follow precisely the time integration history of each sub-domain. At most 10 such log files are produced, therefore if the number of sub-domains is larger only the first 10 sub-domains will be dumped out.

"DPAR"

In calculations with space partitioning, dumps out extra information on the listing. All cycles, in addition to macro steps, are printed out.

"DPAX"

Dump out on the listing a list of all nodes on the axis of revolution i.e. nodes with $x = 0$. This option has only effect in 2D axisymmetric calculations, and must be issued **before** the **GEOM** directive.

Comments:

Another useful debugging tool is the "ECHO" "VERI" directive (see page A.20) that causes, among other things, the memory allocated to each variable to be printed out.

Concerning the "DPSD" option, note that the per-domain log files are automatically opened under the Windows platform. On non-windows platforms (e.g. Unix), it may be necessary to explicitly open these files by including in the input file appropriate **OPNF** directives (see page A.28). Here is an example:

(on non-Windows platform)

```
OPNF FORMAT 51 '/disk1/fauvin/SD_001.LOG'
OPNF FORMAT 52 '/disk1/fauvin/SD_002.LOG'
. . .
OPTI DPSD
. . .
STRUCTURE 2
  DOMA LECT ZON1 TERM
  DOMA LECT ZON2 TERM
. . .
```

In this example there are 2 sub-domains. Note that the unit numbers to be used are 51, 52, etc. up to 60 (max. 10 sub-domains). The names associated with the files are arbitrary, and the files are formatted. On some platforms, full-path names only are accepted as in the above example.

14.9 PHANTOM OPTION

Object:

Some elements are transformed into phantoms (FANTOME) when the time exceeds a given value.

Syntax:

```
"FANTOME"    t_fant  /LECTURE/
```

`t_fan`

Time starting from which the elements become phantoms.

`/LECTURE/`

List of the concerned elements.

Comments:

This option may appear at most once. However, it is possible to declare as many sequences `t_fant`, `/LECTURE/` as needed.

14.10 SHOCK AND IMPACT OPTIONS

Object:

This option allows to define the energy restitution coefficients for the shocks and the impacts.

Syntax:

```
"CHOC"      coechoc
```

`coechoc`

Energy restitution coefficient for shocks and impacts.

Comments:

The restitution coefficient is between 0 (plastic shock) and 1 (perfectly elastic shock).

The default value (when the present option is not activated) is 0.5.

14.11 OPTIONS FOR FSA/FSR

Object:

To provide options for fluid-structure interactions of the ALE type for an either deformable (FSA) or rigid (FSR) structure.

Syntax:

```
< "FSA" "ALF0" alf0 >  
< $[ "NFSC" ; "FSCR" < "INCL" /LEC1/ > < "EXCL" /LEC2/ > ]$ >  
< "FSR" "MFSR" >
```

alf0

Maximum angle, in degrees, between two element faces for which a unique normal is computed. If the actual angle exceeds this value, then two distinct normals are generated. By default, alf0 = 60 degrees.

NFSC

Do NOT correct geometrically computed normals for the FSA and FSR fluid-structure interaction conditions. This is the default.

FSCR

After computing geometrically the normals for the FSA and FSR fluid-structure interaction conditions, apply a correction based on the direction of fictitious internal forces resulting from a uniform pressure field $p = 1$. This correction can be useful e.g. in 3D cases when the element faces are warped (non-planar), or when the integration of the element's internal forces is done with an integration rule that does not exactly match the estimation of the normal to the surface computed by purely geometrical considerations from the surface data.

INCL /LEC1/

An optional list of nodes to which the FSCR option is applied. By default, the option is applied to all FSA and FSR nodes.

EXCL /LEC2/

An optional list of nodes to which the FSCR option is **not** applied. By default, the option is applied to all FSA and FSR nodes.

MFSR

Allow a manually rezoned (i.e., moving) node to be declared FSR at the same time. These two conditions are normally incompatible and therefore an error message is normally issued and the code stops. However, there are cases when this is not an error (but only the user can judge on this, it cannot be done automatically). An example is a node on a rigid plane which at the same time must be moved by some manual rezoning to avoid mesh entanglement. The node can be at the same time manually rezoned (along the plane) and FSR because the link coefficients stay constant even though the node moves. The option deactivates the error message: only one warning message is issued, for the first node concerned. Note that obviously, if this option is specified, it must be inserted in the input file *before* the `LIAI FSR` or the `LINK FSR` directive.

Remarks

In some special cases it may be useful to exclude some `FSA` or `FSR` nodes from the `FSCR` correction. For example, in the transition zone of a pipeline mesh between a 3D representation and a 1D representation by means of the `TUYM` (deformable structure) or `TUBM` (rigid structure) junction: all fluid nodes in the external circumference of the 3D pipe mesh shall be declared `FSA` or `FSR`, but we want to make sure that no `FSCR` correction is applied to them (while it may be desirable for the other nodes). So we may explicitly exclude them by means of the `EXCL /LEC2/` directive.

14.12 OPTIONS FOR NODE-CENTERED FINITE VOLUMES

Object:

To provide options for node-centered Finite Volumes (multicomponent fluid flows).

Syntax:

```
< MC <ORDR ordr>
    <NUFL $[ ROE ; VANL ; STWA ]$ >
    <WBC>
    <SYNC sync>
>
```

ORDR

Introduces the order **ordr** of the numerical integration scheme. May be 1 (first order) or 2 (second order). By default, it is taken **ordr** = 2.

NUFL

Introduces the type of flux calculation in the bulk fluid; may be **ROE** (Roe flux), **VANL** (Van Leer flux) or **STWA** (Steger-Warming flux). It is only accepted in purely Eulerian calculations. Recall that the far-field flux type is chosen (independently from the bulk flux type) by directive **BDF0** in material **MCFF**. By default, it is taken **NUFL ROE** (Roe flux).

WBC

If specified, the boundary conditions are treated according to a weak formulation. It is only accepted in purely Eulerian calculations. In this case external forces at the boundaries are evaluated by imposing zero momentum flux across the solid boundaries, while in the default case (no **WBC** specified) these forces are evaluated by the method of Lagrange multipliers.

SYNC

Introduces the type of synchronization **sync** for the MC variables: 0 (the default) is the old procedure; 1 is the new procedure.

Remarks

The “new” synchronization algorithm (**SYNC 1**), introduced in April 2010, should be used systematically for new calculations. The old algorithm is left only for compatibility with old input files.

14.13 OPTIONS FOR MULTIPHASE MULTICOMPONENT FLUIDS

Object:

To provide options for multiphase multicomponent fluid flows.

Syntax:

```
<  "FLMP"  < "EPS1" eps1 > < "EPS2" eps2 > < "EPS3" eps3 >
      < "EPS4" eps4 > < "NIMA" nima > < "DUMP" dump >  >

< $[ "DPLG"    ;
      "VOFIRE" < "VSWP" > < "CORR" > < "RFCR" >
      < "NOCR" > < "NORC" > ]$ >
```

eps1

Tolerance for the determination of number of effective components (a component is effectively present if its mass fraction is \geq eps1mp). Default is 1.E-7.

eps2

Tolerance for the convergence of Newton-Raphson iterations. Default is 1.E-6.

eps3

Relative density variation to determine initial conditions in FLMPPR (case LIQ + GAS). Default is 1.E-5.

eps4

Tolerance to find the cut-off density for liquids in FLMPRP. Default is 1.E-12.

nima

Max. number of iterations in the above mentioned procedures

dump

Dump (1) or do not dump (0) informations on N-R iterations.

DPLG

Activates Despres-Lagoutiere anti-dissipative algorithm for multi-component flows on structured mesh (see comment below).

VOFIRE

Activates VOFIRE anti-dissipative algorithm for multi-component flows on unstructured mesh (see comment below).

VSWP

If present, exact advected volume is computed for each element face. If not, volume is approximated through the sweep formula.

CORR

Enables the use of CEA improved version of the VOFIRE algorithm.

RFCR

Enables the use of improved algorithm for mixture's density.

NOCR

Disables the use of CEA improved version of the VOFIRE algorithm.

NORC

Disables the use of improved algorithm for mixture's density.

Comments:

Despres-Lagoutiere anti-dissipative algorithm and its extension to unstructured meshes called VOFIRE are used to prevent numerical spreading of the mixing zone of physically non-miscible components. This is still a development in progress and is only available when multi-component material ADCR is used for the fluid in the model.

Improved algorithms for geometric reconstruction and computation of mixture's density on elements faces are currently disabled by default.

14.14 OPTIONS FOR AUTOMATIC REZONING IN ALE COMPUTATIONS

Object:

To provide options for automatic rezoning algorithms in ALE computations.

Syntax:

```
< REZO < SPLI |[ GIUL ; MODI ; BOTH ]| >
    < MVRE |[ NONE ; MODU <VFAC vfac>; MOPR <GAMO gam0> ]| >
    < MEAN |[ POSI ; DEPL ]| >
    < DIRE RMAX rmaxrz >
    < NSTE rznste > < CSHE cshear > < CSTR cstret >
    < YOUN rezyo NU reznu RHO rezro >
    $[ VFLU ; LIAI ]$ >
```

SPLI

Use the splitting algorithm specified next in order to split up the mesh elements around each node and to form the node's influence domain. The available possibilities are: GIUL for Giuliani's original splitting rule, MODI for the modified rule, or BOTH to use a superposition of both methods. The default value is GIUL. This parameter applies only to Giuliani's (AUTO) rezoning model and to the mean (MEAN) rezoning model. For the former, this parameter applies only to 2D quadrilateral ALE finite elements and finite volumes. For the latter, it applies to all elements (2D and 3D), but with a slightly different meaning: the GIUL option considers as neighbours of the node under consideration only the nodes that are connected to it by a face side; the other two options (MODI or BOTH) are equivalent and consider as neighbours all nodes belonging to neighbour elements.

MVRE

Use the mesh velocity restriction algorithm specified next in order to limit the 'raw' optimal mesh rezoning velocity computed by a rezoning algorithm. As shown in the preceding Sections, since all implemented algorithms are explicit, they are unstable unless some limitation is introduced. The available possibilities are: NONE for no restriction (as said, this is likely to be unstable), MODU for the modulus-based rule, or MOPR to use the standard modulus plus projection rule that was adopted in the original Giuliani algorithm. The default value is MOPR. This parameter applies to all rezoning methods described above.

VFAC

The velocity factor to be used in conjunction with the MVRE MODU option. By default it is 2.0. This parameter applies to all rezoning methods described above.

GAMO

The velocity factor to be used in conjunction with the MVRE MOPR option. By default it is 0.2. The obsolete specification of this parameter in the GRIL directive should be avoided from now on. This parameter applies to all rezoning methods described above.

MEAN

Use the mean algorithm variant specified next. The available possibilities are: POSI for an algorithm based on (current) nodal positions, or DEPL for an algorithm based on (current) nodal displacements. The default value is POSI. This parameter applies to all ALE element types.

RMAX

The maximum aspect ratio to be used in conjunction with the DIRE rezoning algorithm. By default it is 5.0. Note, however, that this parameter applies only to 2D quadrilateral ALE finite elements and finite volumes.

NSTE

The number of steps in which rezoning is applied (repartition parameter). By default it is 1.0. This parameter applies to all rezoning methods described above.

CSHE

The shear weight coefficient. By default it is 1.0. Note, however, that this parameter applies only to: a) any elements rezoned by Giuliani's method (AUTO); b) 2D triangles and quadrilaterals rezoned by the SPEC method; c) 2D quadrilaterals rezoned by the QUAD method; d) 2D quadrilaterals rezoned by the MECA method.

CSTR

The stretch weight coefficient. By default it is 1.0. Note, however, that this parameter applies only to: a) any elements rezoned by Giuliani's method (AUTO); b) 2D triangles and quadrilaterals rezoned by the SPEC method; c) 2D quadrilaterals rezoned by the QUAD method; d) 2D quadrilaterals rezoned by the MECA method.

YOUN

The fictitious material Young's modulus to be used in conjunction with the MECA rezoning algorithm. By default it is 1.0. Note, however, that this parameter applies only to 2D quadrilateral ALE finite elements and finite volumes.

NU

The fictitious material Poisson's coefficient to be used in conjunction with the MECA rezoning algorithm,. By default it is 0.0. Note, however, that this parameter applies only to 2D quadrilateral ALE finite elements and finite volumes.

RHO

The fictitious material density to be used in conjunction with the MECA rezoning algorithm. By default it is 1.0. Note, however, that this parameter applies only to 2D quadrilateral ALE finite elements and finite volumes.

VFLU

Choose the 'old' method of dealing with rezoning of nodes that are subjected to liaisons,. The imposed direction(s) are determined indirectly, from the fluid velocity components. As discussed, in 3D cases this method may be too restrictive and prevent the rezoning algorithm from fulfilling its tasks.

LIAI

Choose the 'new' method of dealing with rezoning of nodes that are subjected to liaisons. The imposed direction(s) are determined directly from inspection of the liaison coefficients.

14.15 OPTIONS FOR CELL-CENTRED FINITE VOLUMES

Object:

To provide options for cell-centred finite volume computations.

Syntax:

```
< VFCC <DUMP>
    <FCON fcon> <VISC visc>
    <ORDR ordr>
    $ <OTPS otps> ; ERK2 $
    <RECO reco>
    <LMAS lmas> <LQDM lqdm> <LENE lene> <LALP lalp>
    <LEVEL lvel> <LPRE lpre> <LLAG llag>
    <KMAS kmas> <KQDM kqdm> <KENE kene> <KBAR kbar>
    <RVIT rvit>
    <CENE>      <NTIL>
    <MO  m0>    <VINf vinf>
    <NCFS /LECT/>
    <FLSW flsw>
>
```

VFCC

Introduces the options for cell-centred finite volume computations.

DUMP

Dumps out on listing the data structures FACE_VFCC and SOLUTION_VFCC (only for debugging).

fcon

Solver for the calculation of numerical fluxes at interfaces between volumes. One of the following solvers can be chosen:

1. Rusanov solver (option FCON 1)
2. Flux centred and viscosity (option FCON 2)
3. HLLE (option FCON 3)
4. Riemann for perfect gas (option FCON 4, not yet available)
5. Zha-Bilgen (Flux Vector Splitting) (option FCON 5)
6. HLLC (option FCON 6)

7. Dominant Wave-Capturing (option FCON 7)
8. AUSM+ (Flux Vector Splitting) (option FCON 8)
9. Zha-Bilgen modified (option FCON 9)
10. LDFSS-2 (Flux Vector Splitting) (option FCON 10)
11. AUSM+ Bas Mach (option FCON 11)
12. AUSM+ -up- Bas Mach (option FCON 12)
13. HLLC Bas Mach (option FCON 13)

visc

Defines the viscosity for use with the Flux-centered solver (FCON 2).

ordr

Order in space. Only first or second order is possible. The default is the first order approach.

otps

Order in time. Only first or second order (Van Leer-Hancock predictor-corrector scheme) is possible. The default is the first order approach.

ERK2

This keyword (as an alternative to **OTPS 2**), chooses a Runge-Kutta explicit second-order time integration scheme.

reco

Activates the so-called reconstruction of the variables at the inter-volume interfaces starting from the values at the centroids and from the (spatial) gradients at the centroids. Since the spatial gradients are only computed when second-order in space is activated (**ORDR 2**), reconstruction only makes sense in this case. The default value is 0 (no reconstruction). Option **RECO 1** stays for Green-Gauss reconstruction of the conservative variables (density, momentum and total energy per unit volume). Option **RECO 2** stays for Green-Gauss reconstruction of the primitive variables (density, velocity, internal energy per unit mass, mass fraction). Option **RECO 3** is only available for the CDEM or DEMS materials and stays for Green-Gauss reconstruction of the primitive variables, which in this case involves the pressure instead of the internal energy per unit mass.

lmas, lqdm, lene, lalp, lvel, lpre, llag

Limitation for the reconstruction (**RECO** > 0) of the various quantities: **lmas** for the density, **lqdm** for the momentum, **lene** for the total energy per unit volume, **lalp** for the volume fraction (only for CDEM or DEMS material), **lvel** for the velocity, **lpre** for the pressure, **llag** for the Lagrangian variables, i.e. the mass fractions prior to chemical reaction (only for CDEM material). A limiter typically is a number between 0.0 and 1.0, which multiplies the value of the gradient in order to ensure that the reconstructed values

at the interfaces do not violate some conditions. The value of the limiter is automatically computed by the code in each Finite Volume (and typically varies from volume to volume, and also in time). The available types of limiter are: 0 indicates no limitation (limiter equal to 1.0), 1 indicates a first-order limitation (this corresponds to limiter equal to 0.0 and in practice vanifies the effects of the reconstruction), 2 indicates the limitation of Barth and Jespersen, and 3 indicates the limitation of Dubois (default).

kmas, kqdm, kene

Parameter for the limitation of Dubois for the density (**LMAS 3**), for the momentum (**LQDM 3**) or for the total energy per unit volume (**LENE 3**). This parameter should be between 0.0 and 1.0. The default value is 0.5.

kbar

Parameter for the limitation of Barth and Jespersen, all variables (e.g. **LMAS 2**). 0 indicates the standard one (default), 1 indicates a modified one, which is more robust for the calculation of shock waves. The value **kbar 1** produces the strongest possible limitation.

rvit

Type of reconstruction of the fluid velocity field at VFCC nodes, starting from the velocity field at the VFCC volume centres. This is used to compute the automatic rezoning (mesh velocity) of ALE VFCC fluid nodes and the motion of Lagrangian VFCC fluid nodes. A value of 0 indicates no reconstruction, 1 (default) indicates the arithmetic mean of the neighboring volumes, 2 is the mean weighted by the element volumes, 3 is the mean weighted by the element masses, 4 is the mean weighted by the inverse of the element volumes, 5 is the mean weighted according to Roe.

CENE

This parameter adds a correction of the gradients in a way that the internal energy is always positive (affects only second-order in space calculations with **RECO 1** or **2**, not **3**).

m0

Cut-off value for the Mach number for use with low-Mach solvers (e.g. **FCON 11**, **12** or **13**). For the other solvers, it is ignored.

vinf

Reference velocity for use with low-Mach solvers (e.g. **FCON 11**, **12** or **13**). For the other solvers, it is ignored.

NTIL

No “tilt” in the calculation. This parameter has no effect on calculations with the CDEM or DEMS materials.

NCFS

Announces that a (nodally) non-conforming fluid-structure interaction takes place between a structure (typically meshed by shell elements) and a fluid meshed by VFCC. The following `/LECT/` lists all fluid nodes (which must belong to the VFCC domain) which are located along the non-conforming F-S interface. The code automatically searches the facing structural element, which must be “superposed” (within a small tolerance) to the fluid volume face (such an element *must* exist, else an error message is issued).

flsw

This option allows to choose the type of FLSW algorithm to be used for fluid-structure interaction modeling in conjunction with cell-centred finite volumes. The value 0 (default) means all numerical fluxes across interfaces near the structure are set to zero, except those related to momentum (pressure forces only). The value 1 means that all numerical fluxes across interfaces near the structure are computed by introducing fictitious “ghost” states corresponding to a rigid wall moving with the same speed as the structure.

14.16 OPTIONS FOR CONNECTIONS (“LIAISONS”/LINKS)

Object:

To provide options for the connections in general, for the LIAISON CONTACT directive and for the pinball impact model, see Section D.

Syntax:

```

< LIAJ >
< CONT |[ CONS ; VARI ]| >
< GLIS < NORM |[ ELEM ; NOEU ]|>
    < GAP |[ ELEM ; NOEU ]|> >
< PINS < DUMP > < STAT > < VIDE > < DTPB < CSPB cspb > > < UPDR >
    < EQVL > < EQVD > < EQVF > < NEQV >
    < $[ FACE ; FACI ]$ < FNOR > >
    < CNOR < $[ MIDP ; NCOL < RCEL < $[ MASL ; MAS2 ]$ > > ]$ > >
    < SNOR > < ASN >
    < $[ REB1 ; REB2 ; NORB ]$ >
    < $[ NOGR ; GRID <DGRI> <SORT>
        $[ HGRI hgri ; NMAX nmax ; DPIN dpin ]$
        < PACK ipac > ]$ > >
< LNKS < STAT > < DIAG > < DUMP > >
< FLS <CUB8 c8> >

```

LIAJ

This option causes all constraints on velocities to be imposed on the velocity at time $n+1$, rather than at time $n + 3/2$, which is the default (note that in this notation the current configuration is indicated by $n + 1$). The first form was used for example in PLEXIS-3C. Therefore, this option is mainly useful in order to perform fine grain comparisons between results of PLEXIS-3C and EUROPLEXUS, for debugging purposes.

CONT

Introduces options related to geometric bilateral restraints (LIAISON CONTACT, see Page D.40 and following ones).

CONS

Constant coefficients will be used in the LIAI CONT directives of type SPHE, CYLI, CONE and TORE.

VARI

Variable coefficients will be used in the LIAI CONT directives of type SPHE, CYLI, CONE and TORE. Remember to dimension adequately by the DIME VCON directive.

GLIS

Introduces options to the LIAI or LINK GLIS (sliding surfaces) model.

NORM

Options to control the face normal computation.

ELEM

Exact face normals are computed.

NOEU

Nodal normals are first computed as weighted mean values from the faces surrounding each node. Face normals are then deduced by averaging the nodal normals at the center of each face. This is the default method (see comment below).

GAP

Options to control the way of considering the gap for contact between shell structures.

ELEM

Gap is considered on the master side, which means that the master facet is translated by the gap value in its normal direction before contact detection.

NOEU

Gap is considered on the slave side, which means that the slave node is translated by the opposite of the gap value in the direction normal to the master facet before contact detection. This is the default method (see comment below).

PINS

Introduces options related to the LIAI or LINK PINB (pinball contact) model (see page D.480).

DUMP

Dumps out extensive pinballs information on the listing. Note that even further pinball-related dumps take place by activating the generic option OPTI DUMP in conjunction with the present pinball-specific option.

STAT

Dumps out statistics relative to pinballs on a special file <basename>.pin. At each time step are printed: the number of “raw” detected pinball contacts, the number of contacts remaining after the CNOR algorithm, the number after the NCOL algorithm, the number after the RCEL algorithm and the (final) number of contacts after the *a priori* rebound algorithm.

VIDE

Visualize all descendent pinballs generated by the hierarchic splitting process. This option should only be used for debugging purposes. When activated, all the descendent pinballs (of the highest level) generated during the splitting process are considered in contact, so that they may be visualized interactively e.g. by the **TRAC PINC** command (see page A.25). This allows to visualize the result of the splitting process. Beware that in complex cases a very large number of such pinballs may be generated. When the option is activated, pinball links are not generated, however, since the retained contacts are unphysical. In addition, the calculation is automatically stopped after time step 0, and the **PINS DUMP** (see above) option is automatically activated.

DTPB

Activate automatic limitation of the time increment Δt to account for contacts modelled by pinballs (irrespective of the specific model used, i.e. liaisons, coupled links, or uncoupled penalty). This option is ignored if the user pilots the time increment e.g. by specifying **PAS UTIL**. By default, i.e. without the present option, pinball contacts have no effect on time increments.

CSPB

Introduces the reading of the “stability” coefficient **cspb** to be used in conjunction with the **DTPB** option for the limitation of the time increment Δt due to pinballs. By default the code assumes **cspb=cstab** i.e. the same value as the stability coefficient used for the elements’ stability (see **OPTI CSTA** on page H.20). This quantity should be less than 1.0, like for **CSTA**.

UPDR

Update the radius of parent (0-level) pinballs at every step. By default, the radius is computed only at the initial time. This option may be useful in problems with very large deformations.

EQVL

The radius of parent pinballs (i.e. at the 0-level) is computed in such a way that the pinball volume equals the initial volume of the associated element. By default, the radius is computed so as to encompass all element nodes in the initial configuration.

EQVD

Same as **EQVL** above, but concerning the descendent pinballs generated in hierarchic methods (and this at every level of the hierarchy). The sub-pinball radius is computed in such a way that its volume equals the initial volume of the associated element portion. By default, the radius is computed so as to encompass all element portion “nodes” in the current configuration.

EQVF

Same as EQVD above, but affects only the proper descendent (i.e. of level $L > 0$) pinballs generated in hierarchic methods **at the last (final) level** of the hierarchy. The parent (0-level) pinballs are not affected. The radius of a **final** proper sub-pinball is computed in such a way that its volume equals the initial volume of the associated element portion. By default, the radius is computed so as to encompass all element portion “nodes” in the current configuration. This option should be preferably used in most cases: the other options (EQVL, EQVD or NEQV) are in fact probably useful only in special cases, or for debugging purposes.

NEQV

No equivalent volume calculations. The radius of parent pinballs is computed so as to encompass all element nodes in the initial configuration. The radius of any proper descendent pinballs is computed so as to encompass all element portion “nodes” in the current configuration. This is currently the default. It may be used to restore the default behaviour after one of the other options (EQVL, EQVD or NEQV) has been specified.

FACE

The velocity constraint for a contact between parent pinballs is written at the centroids of the faces crossed by the line joining the pinball centers (it involves only the face nodes). By default, the velocity constraint is written at the pinball centers (which for 0-level pinballs corresponds with the element centroid) and thus involves all the nodes of the element. This option has no effect on contacts between sub-pinballs.

FACI

The velocity constraint for a contact between parent pinballs is written at the intersections of the faces crossed by the line joining the pinball centers (it involves only the face nodes). By default, the velocity constraint is written at the pinball centers (which for 0-level pinballs corresponds with the element centroid) and thus involves all the nodes of the element. This option has no effect on contacts between sub-pinballs.

FNOR

The velocity constraint for a contact between parent pinballs is written along a “mean” of the two face normals $n = (n_A - n_B)$. Note that this requires that either OPTI PINS FACE or OPTI PINS FACI be specified as well. By default, the velocity constraint is written along the direction of the line that joins the pinball centers.

CNOR

The velocity constraint for a contact between sub-pinballs is written along a “common” normal. One such normal is determined for each couple of contacting element faces. When multiple contacts between sub-pinballs occur (pinballs hierarchy at level > 0) in case of flat (face to face) contact, this common normal is an approximation of the normal to the contacting faces.

MIDP

The velocity constraint for a contact between sub-pinballs is written at “midpoints” along the lines that join the retained contacting sub-pinballs. This option is part of the common normal algorithm and therefore it requires that the **CNOR** option be specified as well (see above). This option is incompatible with the **NCOL** option described below. Note that this option has effect only on constraints between sub-pinballs that are part of a “sequence” of two or more contacts between the same couple of ancestors. Single or “isolated” contacts between two ancestors (to which the concept of common normal does not apply) are not affected, and in such cases the constraint is written at the sub-pinball centers.

NCOL

Collapse onto the nearest node of the parent element the center of those descendent pinballs located at “corners”. In addition, for the remaining (non-corner) descendent pinballs, collapse their center onto the element side or face. Note that the above mentioned collapse is performed only as far as the application point of contact reaction forces is concerned, i.e. when writing down the constraints, and it does not affect the position (center, radius) of the descendent pinball itself. By this option the form of the resulting constraints is simpler because they involve less dofs, and the constraints are more independent from one another. This option has only effect on contacts between sub-pinballs (not for contact between parent pinballs) and is incompatible with the **MIDP** option described above. It requires the **CNOR** option.

RCEL

Eliminate repeated constraints for contacts between sub-pinballs that may result after collapse (see option **NCOL** above). This option may help removing a priori from the system repeated constraints that occur e.g. in flat contact between adjacent elements. It requires that the **NCOL** option (and thus also the **CNOR** option) be specified as well. Normally to obtain the maximum benefits a user would specify the three options **CNOR NCOL RCEL**.

MASL

Apply master/slave rule in order to further simplify constraints in case of multiple flat contact between bodies. Constraints of type NP (node-to-point) whose associated node belongs to the “hardest” one of the two contacting bodies are rejected. Body “hardness” is specified optionally in the **PINB BODY HARD** directive, see Page D.480. This option requires that **HARD** has actually been specified for both contacting bodies, and that the **RCEL** option described above has been specified as well. The result should be similar to the more traditional sliding lines (slave node / master surface) algorithm, and might lead to slight under-constraining (spurious penetration) in some cases (if this happens, try using the **MAS2** option below instead).

MAS2

Apply master/slave rule in order to further simplify constraints in case of multiple flat contact between bodies. Multiple constraints of type NP (node-to-point) whose associated node belongs to the “hardest” one of the two contacting bodies are rejected. Body “hardness” is specified optionally in the **PINB BODY HARD** directive, see Page D.480. This option requires that **HARD** has actually been specified for both contacting bodies, and that

the RCEL option described above has been specified as well. The result should be intermediate between a purely pinballs-based algorithm and the more traditional sliding lines (slave node / master surface) algorithm. It might lead to slight over-constraining (contact locking) in some cases (if this happens, try using the MASL option above instead).

SNOR

When a *single* contact occurs between sub-pinballs belonging to the same couple of element faces, and only one of the two sub-pinballs is a face sub-pinball, then the used normal is the normal to that face. This option may be used alone or combined with the CNOR option (which acts only upon *multiple* contacts).

ASN

The so-called “assembled surface normal” (ASN) algorithm of Belytschko and Law (1985) is used to compute a unique (normalized) normal to each external node of the mesh portion subjected to contact, and a unique (normalized) normal to each pinball (parent or descendent). The penetration direction between contacting pinballs is then computed using the ASNs of the two pinballs according to a set of rules. This ameliorates the treatment of flat contact, especially in conjunction with a penalty formulation to compute the contact forces. This option cannot be used together with (is an alternative to) options FNOR, CNOR (and its sub-options), or SNOR.

REB1

The so-called *a priori* pinball contact rebound detection algorithm is used. This is the default contact rebound detection algorithm and therefore specifying this keyword is usually redundant.

REB2

The so-called *a posteriori* pinball contact rebound detection algorithm is used instead of the default *a priori* contact rebound detection algorithm. This option is only intended for internal code testing and verification, because the default algorithm is normally superior to the other one.

NORB

Do not apply any pinball contact rebound detection algorithm. Rebound between pinballs is not treated. This option is useful for pinball contacts treated by the penalty method (see LINK DECO PINB PENA), while it usually makes no sense with pinball contacts treated by Lagrange multipliers (see LINK COUP PINB).

NOGR

Do not use a grid of cells to speed up search of neighbours for contact detection. This is the default.

GRID

Use a grid of cells (as in bucket sorting) to speed up search of neighbours for contact detection. The grid encompasses all elements containing parent pinballs and is built up either automatically (if no further options are specified) in the way specified below, or according to one of the following criteria.

DGRI

Dump out initial grid on the listing (only at step 0).

SORT

Sort the list of contacts in growing order so they (should) become like in the case without grid. This option is only to be used for debugging, since it facilitates the comparison of results with and without grid.

HGRI

Specifies the size of the grid cell. Each cell has the same size in all spatial directions and is aligned with the global axes.

NMAX

Specifies the maximum number of cells along one of the global axes.

DPIN

Specifies the size of the grid cell as a multiple of the diameter of the largest parent pinball. For example, by setting **DPIN 4** the size of the cell is four times the diameter of the largest parent pinball. By default, i.e. if neither **HGRI**, nor **NMAX**, nor **DPIN** are specified, the code takes **DPIN 1.1**. Normally, the cost of searching decreases as one takes smaller values of **DPIN**. However, the memory used tends to increase because there will be more cells. In large cases, a trade-off must be found but it is difficult to say a priori what is the optimal value for **DPIN**. Note that values of **DPIN** at or below 1.0 are **unsafe**. Some contacts may be overlooked (but this depends on the case). To be sure that **all** contacts are detected, use **DPIN** (slightly) larger than 1.0, say 1.001.

PACK

This optional keyword allows to specify a packing size **ipac** for the fast search grid. The search is then done by partially overlapping cubic (square in 2D) “macro cells” each containing **ipac** search cells along each spatial direction. See below for comments.

LNKS

This keyword introduces options which are specific of the links model. They are ignored by the “liaisons” model.

STAT

Dumps out statistics relative to links on a special file **<basename>.lks**. At each time step are printed: the number of link groups (**N_GPS**), the total number of links (**N_LKS**), the total number of permanent links (**N_PLKS**), the total number of non-permanent links (**N_NPLKS**) and finally the number of links of each type (e.g., **BLOQ**, **RELA** etc.).

DIAG

Dumps out additional diagnostics relative to current links (both permanent and non-permanent) on the listing, together with each normal printout (see directive **ECRI**). The information concerns the size of the links matrix, and its “fullness” (i.e. the relative number of non-zero entries). This information can be useful in view of the choice of the most appropriate solution strategy for the links problem.

DUMP

Dumps out all current links (both permanent and non-permanent) on the listing, together with each normal printout (see directive **ECRI**). The generated output can be huge, therefore this option should be used with great care (and for debugging purposes only).

FLS

This keyword introduces options which are specific of the **FLSR** and **FLSW** fluid-structure interaction models.

CUB8 c8

Sets the error level for inverse mapping in 8-node cube elements. By default it is 0, meaning that any error is treated as a real error. By setting it to 1, a lack of convergence in the inverse mapping procedure is not considered an error, but simply that the point considered lies outside the 8-noded cube. By setting it to 2, both a lack of convergence and a zero determinant are considered not as errors, but as an indication that the point considered lies outside the 8-noded cube. These options should be set only in problematic cases (and until the inverse mapping for the **CUB8** shape is reformulated in a more robust way). Note that this option has effects only on the **FLSR** and **FLSW** models, not on other model which use **CUB8** inverse mapping. Note also that this option has the same effect **also** upon inverse mapping in **3-node triangles in 3D**, but with the following meaning of the **c8** parameter: 1 means that $d_{\max} < \text{tol_vol}$ is not considered as an error, while 2 means the above plus also $\text{abs}(\text{err}) > \text{tol_dis}$ is not an error. If you activate this optional switch, it is probably safer to use the value 2 anyway.

Comments:

Be sure to consult also the interactive commands for the visualization of pinballs and of contacts, see Page A.25.

As far as nodal or elementary methods are concerned to compute the facet normals for contact detection and links generation, both may present advantages and drawbacks in different situations. Nodal approach produces smoother variations of the normal along the master side and may be useful for problems such as rolling bodies. However, in the case of strongly folded structures (for example, self-contact crashed bodies), elementary approach ensures better detection of contact between folds and should be preferred.

Considering the gap on slave side is the original way that was implemented in EUROPLEXUS. It has shown recently to potentially produce instabilities for strongly folded structures. In this case, considering the gap on master side has proved to be much more robust. The former approach remains the default until the latter is fully tested and validated.

When a fast search by cells grid is specified for the macro pinballs in contact (`PINS GRID . . .`) and a large 3D problem is being solved with relatively few (but largely scattered) contacts, then one may easily generate an enormous number of cells and the memory required becomes prohibitive. In such cases, it may be convenient to do the search not as a unique scan but by several scans over contiguous “packs” (i.e. rectangular patches) of cells. Each pack or “macro cell” contains a number `ipac` of cells along each spatial direction. In addition, an extra cell is added along each boundary since the packs must be partially superposed for the algorithm to work. Thus in 2D the size of a pack will be $(ipac+2 * ipac+2)$ and in 3D $(ipac+2 * ipac+2 * ipac+2)$ cells. The search by packs is slightly slower than global search because of the increased number of operations and of the more complex algorithm, but the used memory might be much smaller (the user may reduce it by using a lower `ipac`).

14.17 OPTIONS FOR GRAPHICAL RENDERING

Object:

To provide options for the graphical rendering (OpenGL).

Syntax:

```
< REND < $[ FAST ; SAFE ]$ >
    < $[ NODU ; DUMP ]$ >
    <STAT>
    <FAC4 SPLI n TOLE eps>
    <SHAR <ANGL angl> <ABS>> >
```

REND

This keyword introduces the options related to rendering.

FAST

This option uses the fastest available algorithms for the in-software geometric calculations preliminary to geometric rendering operations (see **TRAC REND**). This is the default.

SAFE

This option uses straightforward (but inefficient) algorithms for the in-software geometric calculations preliminary to geometric rendering operations (see **TRAC REND**). It may be useful when one has doubts on the graphical results obtained with the fast version.

NODU

This option does not dump out data related to the in-software geometric calculations preliminary to geometric rendering operations (see **TRAC REND**). This is the default.

DUMP

This option dumps out on the listing data related to the in-software geometric calculations preliminary to geometric rendering operations (see **TRAC REND**). This may be useful for debugging purposes but it produces a big output file.

STAT

This option produces statistics on the allocations performed by the OpenGL graphics module on a special file **<basename>.ogl**. This is useful only for debugging purposes.

FAC4

This option introduces indications about how to render 4-node faces. By default each 4-node face is split into four triangles by generating an extra point at the face center. In this way the rendering of non-planar (warped) 4-node faces is best and does not depend upon face (or element) numbering. Also the representation of iso-values is best. However, a lot of memory is required. Memory can be saved, at the expense of a somewhat worse representation (and not completely numbering-independent), by splitting planar or almost planar 4-node faces into just 2 triangles, or by treating them as a single quadrilateral.

SPLI n

The number of figures into which an almost-planar 4-node face is split. By default it is 4. It may be set to 1 or 2.

TOLE eps

Tolerance ϵ to decide whether a 4-node face is planar or not. The face is considered planar if the scalar product between the two unit normals to triangles 1-2-3 and 1-3-4 obtained from the face is greater than $(1 - \epsilon)$.

SHAR

Introduces options related to the visualization of sharp corners.

ANGL

Sets the minimum angle α_0 (between two 3D faces with a common side) beyond which the side is considered to be a sharp corner. By default, this angle is 60 degrees. Let n_1 and n_2 be unit normals to the two faces. Then the scalar product $n_1 \cdot n_2 = \cos \alpha$ is equal to the cosine of α , the angle between the normals (which is also the angle between the faces). Thus the corner is sharp if $\cos \alpha < \cos 60^\circ$, i.e. when $\alpha < 60^\circ$.

ABS

Consider the absolute value of the above scalar product instead of the signed value. This has the following effect: when two faces have a common side and opposite (or nearly opposite) normals, the side is *not* considered sharp (while by default it would be). This option may be useful in the presence of complex 3D shell structures, because it is not always easy (and sometimes even impossible) to orient them consistently. With this option many “spurious” sharp corners disappear. Thus with this option the rule becomes: the corner is sharp when $|\alpha| < 60^\circ$.

14.18 OPTIONS FOR MESH-ADAPTIVE COMPUTATIONS

Object:

To provide options for mesh-adaptive computations.

Syntax:

```
< ADAP < $ NODU ; DUMP $ > <STAT> <RCON> <MAXL maxl> <NOPP>  
      <RESE> >
```

ADAP

This keyword introduces the options related to mesh-adaptive computations.

NODU

This option does not dump out data related to the mesh-adaptive computations. This is the default.

DUMP

This option dumps out on the listing data related to the mesh-adaptive computations. This may be useful for debugging purposes but it produces a big output file.

STAT

This option prints out on the listing some additional “statistical” data related to the mesh-adaptive computations. The increment in listing size is very small, but the calculation of these data requires some (small) computational effort, therefore they are not computed by default.

RCON

This option imposes a smooth refinement of the mesh, such that the difference in refinement level between two neighboring (or pseudo-neighboring) elements is at most 1.

MAXL

This option introduces an upper limit `maxl` to the level of refinement of the adaptive mesh.

NOPP

Do not propagate `MAXCURV` and `ERRIND` to descendants upon elements split (only for debugging). By default they are propagated.

RESE

Upon un-splitting of a Q41L or Q42L element with a solid material (VM23 with linear elastic characteristics), recompute `SIG` and `ECR` from parent element nodal positions instead of doing averaging on child elements.

14.19 STRAIN RATE FILTERING OPTION

Object:

The strain rate filtering option allows to damp high frequency vibrations wich are not physical and therefore to obtain more physical strain rate values.

Syntax:

"FVIT" alpha

alpha

filter coefficient, must be of the order of the smallest element size.

Comments:

This option is still under development and testing and should therefore be used with great care. this option is available only for isotropic Von Mises material depending on strain rate (VMIS DYNA).

The default value when the present option is not activated is 1. (no filtering).

15 GROUP I—TRANSIENT CALCULATION DEFINITION

Object:

The following directives define, run, verify (qualify) and stop the transient computation which has been defined with all directives given so far.

Furthermore, by means of a so-called “ED1D input deck” it is possible to perform a coupled 1-D/multi-D calculation, see also pages INT.80 and I.23.

Syntax :

```
< "STRUCTURES" . . . >

< "INTERFACES" . . . >

< "XFEM" ... >

"CALCUL" . . .

< "ED1D" {Eurdyn-1D input deck} "ED1D END" >

< "PLAY" {interactive commands} "ENDPLAY" >

< "QUALIFICATION" . . . >

$ "SUITE" ; "FIN" $
```

These instructions are described in detail on the following pages.

15.1 STRUCTURES

Object :

This directive enables the use of the domain decomposition method that has been recently implemented in EUROPLEXUS. Therefore, only some of the elements are currently available, see the list below.

This directive is optional. However, when used it **MUST APPEAR BEFORE** the “CALCUL” directive.

Syntax :

```
"STRUCTURE" <"DTUN">
  | [ "AUTO" <"PMET"> <"ROB"> <"CINI"> ...
    ... <"WFIL" <ndwfil>> <"DACT" /LECDDL/> <"DPRE" ipre> ;

    nbdo * (
      | [ "DOMA" /LECT/ <"IDEN" ndom> <["$DTMX" dtmx ; "DTFX" dtfx]$> ;
        "MODA" /LECT/ <"IDEN" ndom> ...
          ... "FICHIER_VIBRATIONS" <FORMAT> <ndfich> ...
          ... <"POST_TRAITEMENT" $["TOUS" ; "CHPO"]$ > ...
          ... <"NOFO"> ] |
        ) ] |
```

DTUN

Multiple time scales treatment (one per subdomain) is deactivated. Every subdomain has the same time scale (see comment below).

AUTO

MPI only. Automatic domain decomposition using available number of threads.

PMET

MPI only. ParMetis library is used to perform domain decomposition.

ROB

MPI only. Recursive Orthogonal Bisection algorithm is used to perform domain decomposition (see comment below).

CINI

MPI only. Automatic domain decomposition with ROB after a restart is performed using initial coordinates instead of current coordinates.

WFIL

MPI only. Use of an element weight file for automatic domain decomposition (see comment below).

ndwfil

MPI only. Number of the logical unit of the weight file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is .wgt.

DACT

MPI only. Selection of active directions (from 1 to 2 in 2D, from 1 to 3 in 3D) for automatic domain decomposition using ROB.

ipre

Number of the first cutting direction for automatic domain decomposition using ROB (see comment below).

nbdo

Number of subdomains for fixed domain decomposition.

/LECTURE/

Indexes of the elements forming the current subdomain. Note that these must include also the indexes of the CLxx elements used to represent any *non-matching interfaces* belonging to the current subdomain (see directive **INTERFACE** below).

DOMA

This keyword introduces the definition of a subdomain.

MODA

This keyword introduces the definition of a subdomain represented by a modal basis.

ndom

Number to identify the subdomain when declaring the interfaces. If omitted, this number is the rank of the subdomain in the order of declaration of all the subdomains (including modal ones).

dtmx

Maximum time cycle imposed for the current subdomain (see comment below).

dtfx

Fixed (constant) time cycle imposed for the current subdomain (see comment below). This keyword is incompatible with the `dtmx` described above, and may be used only if the step is also user-driven (`OPTI PAS UTIL`). The given value `dtfx` should be an integer sub-multiple of the user-specified time step (typically `pasf`, see the `CALC` directive).

ndfich

Number of the logical unit of the file containing the modes and reduced matrices, or file name in quotes. If omitted, the program chooses a file name and unit number by default (see page A.27). The default extension is `.MSH` and the default unit number is 9, so that by default the modes are read from the same file that contains the CASTEM 2000 mesh (file `<base_name>.msh`).

FORMAT

If this keyword is present, the file is formatted, otherwise it is unformatted.

TOUS

Keyword meaning that all CHAMELEMS and CHAMPOINTS will be calculated within the modal subdomain.

CHPO

Keyword meaning that only CHAMPOINTS will be calculated within the modal subdomain (see comment below).

NOFO

Keyword meaning that no external forces will be calculated within modal subdomain in order to save computation time.

Comments :

The elements currently available for calculations with domain decomposition are:

in 2D :

COQU, TRIA, BARR, MEMB, CL2D, CAR1, CAR4, COQC, Q92 , Q93 , COQI,
ED01, CL2S, CL22, Q41L, Q42L, FUN2, T3VF, Q4VF

in 3D :

CUBE, COQ4, POUT, CL3D, BR3D, PR6, TETR, PRIS, PMAT, CL3T, CUB8,
APPU, MECA, T3GS, FL38, DKT3, SHB8, FUN3, Q4G4, CL3Q, Q4GR, Q4GS,
ASHB, T3MC, TEVF, PYVF, PRVF, CUVF

The code initially performs the following checks:

1. The union of the defined subdomains must cover the entire domain.
2. The intersection of any two subdomains must be empty.

In a calculation by domain decomposition, the following terminology applies:

time cycle Is the time increment associated with a specific subdomain. It varies in general from subdomain to subdomain.

time step Is a (macroscopic) time increment common to all subdomains (global quantity). At the end of each one of these time steps, all subdomains are “synchronized” by solving the equilibrium equations without any interpolation. For this reason, time steps are also called **sync steps**. Printout and storage of results and in general any interaction with the user is only available at sync steps.

time stations An integer counter that counts the union (not the sum) of time cycles and time steps. It is incremented by 1 each time the code computes at least one subdomain.

It is important to note that users normally have limited control over time cycles, which are managed internally by the code according to the characteristics of each subdomain. All time-related quantities, such as for example those of the **CALCUL** directive (see page I.20) or the chosen instants for data printing and storage (see the **ECRITURE** directive, page G.70), concern time steps (i.e. sync steps) as defined above, and not time cycles.

The user may explicitly **control the sync step** in the following ways:

- By choosing the option **OPTI PAS UTIL** and by specifying a **constant** sync step in the **CALCUL** directive by means of **PASFIX pfix**. In this case, the sync step is constant and equal to **pfix**.
- By choosing the option **OPTI PAS UTIL** and by specifying a list of sync steps in the **CALCUL** directive by means of **HIST /PROG/**. In this case, the sync step evolves according to the specified time sequence given in **/PROG/**.
- By choosing the (default) option **OPTI PAS AUTO** and by optionally specifying a factor (see **SDFA sdfac** keyword in the **CALCUL** directive). The code computes the maximum of the stability times steps of the various subdomains, and multiplies this value by **sdfac** to obtain the sync step (by default, **sdfac** is 1.0). This calculation is performed at the end of each sync step, so the sync step generally varies in time. Note also that in this case the sync steps may be automatically adapted to match the chosen printing and storage times by specifying the option **OPTI STEP IO** (default) or **OPTI STEP IOT**, see page H.20. Furthermore, in this case the maximum sync step may be limited by specifying the **DTMA dtmax** keyword of the **CALCUL** directive.

The user may explicitly **control the cycles** in the following ways:

- By choosing the option `OPTI CSTA` that specifies the safety factor over the stability value. This factor applies to all elements, and therefore it equally affects all subdomains (global quantity).
- By specifying, for each subdomain, a limiting value of the associated time cycle, see the `DTMX dtmx` keyword above.
- By specifying, for each subdomain, a constant time cycle, see the `DTFX dtfx` keyword above.

Note, however, that within each time step the time cycles vary in general from subdomain to subdomain, and vary in time for a given subdomain, even in the case that the user chooses `OPTI PAS UTIL` and a fixed time step, except in the case that a fixed cycle value `dtfx` value is explicitly specified.

When the `DTUN` keyword is used, subdomains are forced to follow one unique time scale. According to options set in the `OPTION` directive and to stability conditions on all subdomains, one single time-step is computed at each cycle and given to each subdomain. Subdomains are thus always synchronized.

Since the behaviour of the present domain decomposition model as regards time stepping depends upon the corresponding user option (i.e. upon `PAS AUTO` or `PAS UTIL`), it is advised to specify the `STRUCTURE` directive **after** any options that set the time stepping mode (but **before** the `CALCUL` directive, as already noted).

When option "POST" "CHPO" is activated, the CHAMELEMS are to be computed out of EUROPLEXUS from the CHAMPOINTS with a linear elastic constitutive law, which is the only valid within a modal subdomain.

Example 1 :

```
OPTION PAS AUTO STEP IO
...

STRUCTURE 3
  DOMA LECT zone1 TERM IDEN 91
  DOMA LECT zone2 TERM IDEN 92 DTMX 5e-6
  MODA LECT zone3 TERM IDEN 93 FICH FORM POST TOUS
      NOFO
CALCUL TINI 0.0   DTMAX 40e-5  NMAX 80000 TFIN 350e-3
```

The computational domain consists of three subdomains. The stability time cycle of the first subdomain is computed automatically by the code. The stability cycle of the second subdomain is the minimum between the computed value and 5E-6. The third subdomain is replaced by a modal basis, with modes and matrices given in file '`<base_name>.msh`' (i.e. the same file

that contains the CASTEM 2000 mesh). Both the CHAMELEMS the CHAMPOINTS but no external forces are computed within this subdomain. The sync step is automatically computed by the code as the minimum value between 40E-5 (DTMAX) and 1.0 times the maximum stability step over all subdomains (recall that by default `sdfac` equals 1.0). Furthermore, the selected printing and storage times will be precisely matched by adapting the sync step (`OPTI STEP IO`). The three subdomains are identified as '91', '92' and '93' as far as interface declarations are concerned (see the `INTERFACES` directive next).

Example 2 :

```
OPTION PAS UTIL
```

```
...
```

```
STRUCTURE 3
```

```
  DOMA LECT zone1 TERM DTFX 1e-6
```

```
  DOMA LECT zone2 TERM DTMX 5e-6
```

```
  DOMA LECT zone3 TERM
```

```
CALCUL TINI 0.0   PASF 40e-5   NMAX 80000 TFIN 350e-3
```

The computational domain consists of three subdomains. The stability cycle of subdomain 1 is fixed to the constant value 1E-6. That of subdomain 2 is the minimum between its stability value and 5E-6. That of subdomain 3 is dictated only by local stability. The sync step is constant and has the value 40E-5. Printout and storage of results will occur at the sync steps whose times are greater than or equal to the chosen values. This is because the `OPTI STEP IO` or `OPTI STEP IOT` options may not be used in conjunction with `OPTI PAS UTIL`, i.e. the sync step (being constant) may not be adapted.

MPI calculations

With fixed domain decomposition, the number of parallel threads **must** be equal to the number of declared subdomains.

For automatic domain decomposition, an external file can be entered to provide elementary weights, in order to optimize load balancing. Each line of the weight file is composed of 2 integers: first the number of the concerned elementary entity (finite element, finite volume, SPH particle...), second the weight associated to it.

Classically, EUROPLEXUS is used to generate the weight file (see page I.20). To generate the file from scratch, elementary numbers can be found in the listing file of a previous run with the same model. Every elementary entity, except CL elements and debris elements, must be given a weight.

If no weight file is used, all weights are set to 1.

Automatic decomposition using **Recursive Orthogonal Bisection** consists in successive

recursive splittings of the domain along available space directions with circular permutations among them. Some directions may be deactivated, either by the user (**DACT** keyword) or automatically by the program if the bounds of the model along these directions are too small.

By default, the first splitting direction is the one along which the spatial extension of the model is maximal. However, one specific starting direction can be forced using **DPRE** keyword.

Classically, each splitting involved in the algorithm consists in creating from 1 part of the model 2 subparts of equal weight, yielding that the number of used threads must be a power of 2. However, the proposed algorithm allows to use any number of threads, by adjusting the number of levels of the recursive decomposition and the number of subparts per part created at the last level, 2 subparts per part being created at every other levels.

For example, with 12 threads, 3 levels will be considered, with 3 subparts per part at the last level ($2 \times 2 \times 3 = 12$), whereas with 10 threads, 2 levels will be considered, with 5 subparts per part at the last level ($2 \times 5 = 10$).

15.2 INTERFACES

Object:

This directive allows to set options for the treatment of connections between subdomains. It also allows the explicit declaration of interfaces between couples of sub-domains. These interfaces *may correspond to matching or non-matching meshes*.

In the case of matching meshes, interface declaration is optional, provided the interface nodes are the same (i.e., have the same index) for the two sub-domains. If only the geometric points are identical (i.e., coordinates are the same), but each subdomain has its own nodes (with different indexes), a compatible interface has to be declared (see the `COMP` keyword below).

In the case of non-matching meshes, this directive is **mandatory**.

The `STRUCTURE` directive must appear **before** this directive.

Syntax:

```
"INTERFACE" $[ "LINK" ; "NOLI" ]$ $[ "MULT" ; "NOMU" ]$ ...

... < nbinterf * (
    |[ "COMP" ; "MORTAR" ; "OPTIMAL" ]| <"TOLE" tole> ...

... "DOMAINE" ndom1 /LECTURE/ "DOMAINE" ndom2 /LECTURE/ ) >
```

LINK

Interface connections are coupled with other kinematic links declared with the `LINK` directive (not the `LIAI` directive). This option is mandatory if some declared links concern more than just one subdomain.

Using this option causes the option `DTUN` of the `STRUCTURE` directive to be activated (see comment below).

NOLI

Interface connections are treated independantly from any other kinematic links, which implies that no kinematic link involving more than one subdomain can be declared. This is the default.

MULT

Every interface connections are treated by means of Lagrange multipliers. This is default.

NOMU

Interface connections with matching meshes and coincident nodes are treated directly with no Lagrange multipliers (faster solution). Non-matching meshes or matching meshes with duplicated nodes are still treated by means of Lagrange multipliers.

Using this option causes the option DTUN of the **STRUCTURE** directive to be activated (see comment below).

nbinterf

Number of interfaces.

COMP

Keyword declaring an interface with matching meshes.

MORTAR

Keyword declaring an interface with non-matching meshes, treated by the mortar method (see comment below).

OPTIMAL

Keyword declaring an interface with non-matching meshes, treated by the optimal method.

tole

Tolerance given to find matching nodes (default=1.E-3).

ndom1

Identification number of the first sub-domain (see **STRUCTURE** directive).

ndom2

Identification number of the second sub-domain.

/LECTURE/

In the case of matching meshes, indexes of the **nodes** forming the sub-domains interfaces.

In the case of non-matching meshes, indexes of the interface **elements** forming the sub-domains interfaces (see comment below).

Comments:

Handling multiple time scales in a multi-domain framework requires a special treatment of the interface connections using Lagrange multipliers. This treatment is not available for generic kinematic links, so that multiple time scales option must be deactivated in order to couple interface connections to other kinematic links, which may thus involve more than one subdomain. This is done by automatically activating the **DTUN** option in the **STRUCTURE** directive. This is also the case when the treatment of connections between matching meshes with coincident nodes is accelerated by not using Lagrange multipliers. In those cases, multiple time scales in the model can still be taken into account by using the **PART** keyword in the **OPTION** directive.

When using the mortar method, the sub-domains whose mesh is used to discretize Lagrange multipliers has to be specified. It is the **second one** (**ndom2**) in the order of declaration of the sub-domains concerned by the interface.

When using interfaces with non-matching meshes, so-called **CLxx** elements (see pages INT.90 and INT.100) have to be affected to interface regions of each sub-domain. These elements **must** be given the “phantom” material (**MATE FANT**) with density equal to zero.

The treatment of non-matching meshes with 3D solid elements is restricted to hierarchical meshes. In this case, the mortar method and the optimal method are identical, and a mortar interface has to be declared, **with the second sub-domain corresponding to the finest mesh**.

The **mortar** method may be used with any element types in 2D, but only with shell element types in 3D. When using the **mortar** method with linear interfaces (2-noded element sides), there must be at least one geometrical point that has the same coordinates, within the tolerance **tole** defined above, in the two facing meshes. The node indexes (node numbers) of this point can be different in the two meshes. This is necessary because the interface model uses the point's coordinates internally in order to define a reference frame on the interface.

Example 1 :

```

STRUCTURE 3
  DOMA LECT zone1 inte12 inte13 TERM IDEN 91
  DOMA LECT zone2 inte21 inte23 TERM IDEN 92
  MODA LECT zone3 inte31 inte32 TERM IDEN 93 FICH FORM 'fich.mrd'
                                     POST TOUS NOFO
...
INTERFACE 3
  COMP TOLE 1.E-2
    DOMA 91 LECT inte12 TERM
    DOMA 92 LECT inte21 TERM
  MORTAR TOLE 1.E-2
    DOMA 92 LECT inte23 TERM
    DOMA 93 LECT inte32 TERM
  OPTIMAL
    DOMA 91 LECT inte13 TERM

```

DOMA 93 LECT inte31 TERM

The computational domain consists of three sub-domains. Three interfaces are declared:

1. The first between sub-domain number 91 and sub-domain number 92 with matching meshes.
2. The second between sub-domain number 92 and sub-domain number 93 with non-matching meshes treated by the mortar method. The nodes of sub-domain number 93 are used to enforce kinematical continuity.
3. The third between sub-domain number 91 and sub-domain number 93 with non-matching meshes treated by the optimal method.

Note that `inte12`, `inte21` are nodes groups, whereas `inte23`, `inte32`, `inte13`, `inte31` are elements groups.

15.3 XFEM

Object :

This directive enables the use of the eXtended Finite Element Method (X-FEM). It uses a specific Level-set mesh to describe the crack in space.

The formulation of X-FEM is given by a standard part and an enriched one as follow:

$$\bar{\mathbf{U}} = \sum_{i \in \mathcal{N}} N_i(\mathbf{x}) \mathbf{U}_i + \sum_{j \in \mathcal{N}^e} N_j(\mathbf{x}) H(\mathbf{x}) \mathbf{U}_j^e \quad (5)$$

This formulation allows to take into account a displacement discontinuity in the mechanical mesh (a crack). And the characterization of propagation law makes the crack propagate through the mesh without remeshing at any time. But enriched element are used in order to describe the discontinuity with additionnal degrees of freedom. The corresponding available elements are XCAR and XCUB for 2D and 3D.

References

The model is described in reference [\[709\]](#), [\[711\]](#).

Syntax :

```
"XFEM"      "NI"      ni      "NJ"      nj      "DX"      dx      "DY"      dy
              "XZER"    xzer    "YZER"    yzer    "FISX"    fisx    "FISY"    fisy
              "FISC"    fisc    "PRBX"    prbx    "PRBY"    prby    "PRBC"    prbc
              "ORDR"    ordr    "KICR"    kicr    "RAYO"    rayo    "CHOI"    choi
              "CR"      cr
<"NK"      nk      "DZ"      dz      "ZZER"    zzer    "FISZ"    fisz
              "PRBZ"    prbz    "RPLA"    rpla    "NCOU"    ncou >
/LECTURE/
```

ni

Number of subgrids in x direction.

nj

Number of subgrids in y direction.

nk

Number of subgrids in z direction (Default 1).

dx

Discretization of subgrids in x direction.

dy

Discretization of subgrids in y direction.

dz

Discretization of subgrids in z direction (Default 0.).

xzer

Position x0 of left bottom point of the mesh of the level-set.

yzer

Position z0 of left bottom point of the mesh of the level-set.

zzer

Position z0 of left bottom point of the mesh of the level-set (Default 0.).

fisx

Equation of the planned crack surface.

fisy

Equation of the planned crack surface.

fisz

Equation of the planned crack surface (Default 0.).

fisc

Equation of the planned crack surface.

prbx

Equation of the planned crack front surface.

prby

Equation of the planned crack front surface.

prbz

Equation of the planned crack front surface (Default 0.).

prbc

Equation of the planned crack front surface.

ordr

Paramater for level set algorithms (reinitialization, orthogonalization, propagation, extension).

kicr

Critical value for crack propagation.

rayo

Length to characterize average or integral "near crack tip".

choi

Parameter to choose: 1 for stress intensity factors (only 2D), and 2 for non-local stress near crack tip (2D and 3D).

cr

Rayleigh wave velocity (maximum crack velocity).

rpla

Length parameter to subcut element (choose around 3 times rayo).

ncou

Number of layer in thickness (1 in 2D, and Default 1).

LECTURE

List of the elements concerned (XCAR in 2D or XCUB in 3D).

The localization of the initial crack is given by 2 plans. The first one defines the surface of the crack by: (level-set ϕ_1)

$$fisx \cdot X + fisy \cdot Y + fisz \cdot Z + fisc = 0 \quad (6)$$

The corresponding level-set is:

$$\phi_1(X, Y, Z) = \frac{fisx \cdot X + fisy \cdot Y + fisz \cdot Z + fisc}{\sqrt{fisx^2 + fisy^2 + fisz^2}} \quad (7)$$

And the second one defines the front of the crack: (level-set ϕ_2)

$$prbx \cdot X + prby \cdot Y + prbz \cdot Z + prbc = 0 \quad (8)$$

The corresponding level-set is:

$$\phi_2(X, Y, Z) = \frac{prbx \cdot X + prby \cdot Y + prbz \cdot Z + prbc}{\sqrt{prbx^2 + prby^2 + prbz^2}} \quad (9)$$

The crack is the isozero ϕ_1 and the negative part of ϕ_2 . Both level-sets are exported in paraview output with the keyword XLVL. So the representation of the crack is possible in paraview by doing the negative part of PHI2 on isozero PHI1.

Example 1:

```

...
ECRITURE
...
FICHIER FORMAT AVS PRVW FREQ 10
VARI DEPL XLVL ECRO ECRC LECT 2 TERM
...
XFEM
NI 300    NJ 200    NK 100
DX 0.0005 DY 0.0005 DZ 0.00025
XZER 0.03 YZER 0.01 ZZER -0.004
FISX -0.2 FISY 1.    FISZ 0.    FISC -0.026
PRBX 5.    PRBY 1.    PRBZ 0.    PRBC -0.416
ORDR 8     KICR 12.E6 RAYO 0.001 CHOI 2
CR 250.    RPLA 0.003 NCOU 7
LECT MAILX
...
CALCUL TINI 0.0    DTMAX 40e-5  NMAX 80000 TFIN 350e-3

```

Bibliography:

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Moës N., Dolbow J., Belytschko T., "A finite element method for crack growth without remeshing", *International Journal for Numerical Methods in Engineering* (1999) **46**:131–150.

Moës N., Gravouil A., Belytschko T., "Non-planar 3D crack growth by the extended finite element and level sets - Part I: Mechanical model", *International Journal for Numerical Methods in Engineering* (2002) **53**:2549–2568.

Gravouil A., Moës N., Belytschko T., "Non-planar 3D crack growth by the extended finite element and level sets - Part II: Level set update", *International Journal for Numerical Methods in Engineering* (2002) **53**:2569–2586.

Menouillard T., Réthoré J., Combescure A. and Bung H., "Efficient explicit time stepping for the eXtended Finite Element Method (X-FEM)", *International Journal for Numerical Methods in Engineering* (2006) **68**:911–939.

Menouillard T., "Dynamique explicite pour la simulation numérique de propagation de fissure par la méthode des éléments finis étendus", Thèse de Doctorat INSA de Lyon 2007.

15.4 “CALCUL” DIRECTIVE

Object:

This directive starts the time solution of a given problem. The keyword **CALCUL** is compulsory and should appear after the data sets A, B, C, D, E, F, G and H.

The user can specify the initial and final times of the computation, the value of or constraints on the time step and the maximum number of computation steps.

Syntax:

```
"CALCUL"    "TINI" tini  "TEND" tend
              < "NMAX" nmax                      >
              < "DTMI" dtmin                      >
              < "DTMA" dtmax                      >
              < "TFAI" tfai                      >
              < $[ "HIST" /PROG/ ; "PASFIX" pfix ]$ >
              < "PAS1" pasone                    >
              < "SDFA" sdfac                    >
              < "LBMS" nfreq                     >
              < "LBNS" nstep                     >
              < "LBMD" mxdev                     >
              < "LBST"                          >
              < "LBPW" ndwfil                    >
              < "LBFT"                          >
```

tini

Initial time of the computation. In case of a restart run, this value is ignored (it may actually be left out), since the actual initial time is set to the value read from the restart file.

tend

Final time of the computation (the keyword **TFIN** is also accepted as a synonym of **TEND**).

nmax

Maximum number of computation steps. Default is 1000000.

dtmin

Minimum value for the time step. Is only considered in a PAS AUTO or PART calculation. Default is 1.D-12 at JRC.

dtmax

Maximum value of the time step. Is only considered in a PAS AUTO or PART computation. Default is 1.D12.

tfai

Elements having a smaller stability time step than this value are eroded (failure). Note that the chosen value applies to *all* elements in the mesh. However, only those elements that possess an “erodable” material (see EROS directive on page A.30 for more information) are actually eroded. At the moment these are the following materials: LEM1 (Lemaitre), ZALM (Zerilli-Armstrong), LMC2 (Lemaitre-Chaboche), VM23 (Von Mises Isotropic), MINT (interface material), GLAS (glass) and LSGL (laminated safety glass). Whenever an element is eroded due to this criterion, the element characteristics at the moment of the erosion are written on the listing. This may allow to check *a posteriori* why the element’s stability dropped below the specified value (e.g., excessive distortion of the element).

HIST

Can only be used in PAS UTIL cases. The following /PROG/ procedure defines the actual ‘time history’ of the computation, i.e. all the times for which the solution will be computed. In this way, the user assumes complete control over the time step. No check on minimum or maximum values are performed. Time steps are computed as the difference between two successive specified times. The initial time of the computation should not be specified in the /PROG/.

pfix

This is a shortcut to assign a user defined time step that remains fixed in time. Can only be used in PAS UTIL.

pasone

This option allows to specify the value of the time increment used during step 0 and is only useful for PAS AUTO cases. In particular, it is mandatory to use it in the case of an advection-diffusion calculation, because in such a case the program does not compute the first time increment automatically. Another useful case is in the presence of energy injection in MC (multicomponent fluid) calculations, in order to use a smaller initial *deltat* than that automatically computed by the code. In order to let the time increment grow slowly, use can be made of the DTVA option (see Options related to the time step).

sdfac

Factor for subdomains computations. This optional keyword is only relevant in multi-domain calculations (see STRUCTURE directive on page I.15) that use automatic calculation of the time step (OPTI PAS AUTO). In this case, the sync step common to all subdomains is automatically chosen as **sdfac** times the **largest** stability step of the various subdomains. By default, **sdfac** equals 1.0.

nfreq

MPI only. Number of time-steps between two load-balancing measuring periods (see comment below).

nstep

MPI only. Number of time-steps within a load-balancing measuring period (see comment below).

mxdev

MPI only. Maximum standard deviation for load-balancing quality estimation (see comment below)..

LBST

MPI only. Stop calculation if previous maximum standard deviation is exceeded for elementary tasks balance (see comment below).

LBPW

MPI only. Print elementary weight file at the end of each load-balancing measuring period (see comment below)..

ndwfil

MPI only. Number of the logical unit of the weight file or file name in quotes. If omitted, the program chooses a file name by default (see page A.27). The default extension is .wgt.

LBFT

MPI only. Enable filtering of measured computational costs before writing weight file (see comment below).

Comments:

The word **CALCUL** is compulsory and should appear only once.

If a user does not have an idea of the stability step for a given problem, he can run the program with the

If a user specifies **PAS UTILISATEUR**, he is responsible for the stability of the computation, because **EUROPLEXUS** does not check the stability in this case. This option may therefore lead to instabilities in the calculation, and should only be used in special cases.

The maximum value of the time step enables the time step to be limited in the case of the option **PAS AUTOMATIQUE** or **PARTITION**.

The computation stops when either the maximum number of steps or the final time is reached. A computation in `PAS AUTO` or `PART` also stops if the stability step becomes lower than the minimum value.

If `HIST` is used, remember to dimension adequately (see `TTHI` on page A.105).

Note that, in multi-domain computations, all time-related quantities in the `CALCUL` directive refer to sync steps and not to subdomains cycles (see directive `STRUCTURE` on page I.15).

Load-balance measuring for MPI calculations

This section is still under strong development.

Load-balance is a key point to achieve parallel performance. Load-balance measuring consists in measuring time taken by each thread to perform computational tasks within a given number of time steps.

The first measuring period starts at the first step of the simulation. After that, the number of time steps between the starting steps of two successive measuring periods is given using `LBMS` keyword. Measuring options are activated as soon as a positive integer is read after `LBMS` keyword.

The number of time steps within a measuring period is given using `LBNS` keyword, which should be smaller than the interval between two measuring periods to produce accurate results. Default value is 100.

Quality of load-balance is estimated through the value of the standard deviation of the quantity of interest. If the quantity is well distributed among the threads, standard deviation should be close to 0. 2 quantities are currently considered: first, the time needed to perform elementary computations, which is controled by the quality of domain decomposition, second, the time needed to perform every computational tasks, including treatment of links. `LBMD` is used to enter a maximum authorized standard deviation for both quantities, generating a warning message if it is overcome at the end of a measuring period. The calculation can be forced to stop in the case of unauthorized standard deviation concerning elementary computations (`LBST` keyword), as it indicates that domain decomposition should be improved.

Using time measures during a period, the program is able to estimate the computational cost of elementary entities (finite elements, finite volumes, SPH particles...), as far as elementary operations only are concerned. A weight file can be written using `LBPW` keyword, to be used to improve automatic domain decomposition (see page I.15).

Measured weights can be filtered prior to file writing, to account for numerical noise in measures due to execution environment. Current filter consists in computing a global cost for groups of elements instead of individuals. Elements are grouped with respect to the couple of parameters (type of element, type of material). Group weight is then equally divided among

concerned elements.

This can produce bad results and should be used with care. Raw weights and filtered weights can be visualized using PVTk output file.

15.5 ED1D INPUT DECK

Object:

This directive allows to specify a so-called “EURDYN-1D input deck”, i.e. a set of input data to be read by the EURDYN-1D module (ED1D) that is now embedded within EUROPLEXUS. In this way it is possible to perform a coupled 1-D/multi-D calculation, as described on page I.80.

The ED1D input deck must be included within the normal EUROPLEXUS input file, immediately after the CALCUL directive and before any additional EUROPLEXUS directives (for example, QUALIFICATION).

The ED1D input deck must be immediately preceded by a line containing ED1D (capitals, starting in column 1, followed only by blanks if any) and be immediately followed by a line containing ED1D END (capitals, starting in column 1, followed only by blanks if any).

Syntax:

```
. . .
"CALC" . . . (see CALCUL directive, page I.20)
*
=====
ED1D
(as many ED1D data as needed to describe the 1D part of
the numerical model)
ED1D END
=====
*
<"PLAY" . . . or "QUAL" . . . or "SUIT" or "FIN">
```

Comments:

The keyword ED1D must appear as such and start in column 1. There must not be any other data on the same line.

The keywords ED1D END must appear as such and start in column 1. There must not be any other data on the same line.

The contents of the ED1D data deck proper (i.e. the lines contained between ED1D START and ED1D END) is described in the EURDYN-1D manual, listed in the References: ([33]).

15.6 PLAY (interactive commands)

Object:

This directive allows to execute a set of “interactive” commands, i.e. any of the commands described above on Page A.25, by reading them from a file (actually, from the regular EUROPLEXUS input file) rather than from the keyboard.

If present, this directive must immediately follow the **CALC** directive (and the optional **ED1D** . . . **ED1D** **END** directive, if present).

Normally, interactive commands are typed by the user at the keyboard. With the present directive, it is possible to store such commands in the regular EUROPLEXUS input file, with the advantage that a given “interactive” calculation may be repeated identically as many times as needed.

This feature is especially useful for the automatic execution of calculations with intermediate visualizations (**TRAC**) and for the automatic generation of animations (**AVI**).

After reading the **PLAY** directive, the code continues to read the following commands from the regular EUROPLEXUS input file, but interprets them as interactive commands (i.e. like if they were typed at the keyboard), until the termination sequence **ENDPLAY** is encountered. Then, normal input file reading (again from the regular EUROPLEXUS input file) is restored.

Syntax:

```
. . .
"CALC" . . . (see CALCUL directive, page I.20)
*
<"ED1D" {Eurdyn-1D input deck} "ED1D END">
*
=====
PLAY
(as many 'interactive' commands (see Page A.25) as needed)
ENDPLAY
=====
*
<"QUAL" . . . or "SUIT" or "FIN">
```

Comments:

The keyword **PLAY** must appear as such and start in column 1. There must not be any other data on the same line.

The keyword **ENDPLAY** must appear as such and start in column 1. There must not be any other data on the same line.

The available interactive commands are listed on page A.25.

15.7 QUALIFICATIONS

Object :

This directive allows to verify (qualify) the results of a calculation by comparing them with given reference values.

The keyword **VALIDATION** is still accepted in place of **QUALIFICATION** for backward compatibility. However, new input files should always use the keyword **QUAL**.

Syntax :

```
"QUAL" <"AUTO"> (  | [  "COOR" ; "DEPL" ; "VITE" ; "ACCE" ; "FEXT" ;
                        "MASN" ; "ADFT" ; "MCPR" ; "MCRO" ; "MCTE" ;
                        "MCVI" ; "MCMF" ; "SIGN" ; "ECRN" ; "FINT" ;
                        "FLIA" ; "FDEC" ;
                        "CONT" ; "EPST" ; "ECRO" ; "ENEL" ; "RHO" ;
                        "MASE" ; "EPAI" ; "VCVI" ;
                        "BILA" ; "WINT" ; "WEXT" ; "WCIN" ; "WECH" ] |

                        $[ "COMP"  icomp      <"GAUS"  igauss>  ]$
                        "REFE"  valref      "TOLE"  valtol

                        /LECTURE/  )
```

AUTO

Automatic qualification, see comments below. This keyword should be used only in very special cases.

COOR ... FDEC

Name of the nodal variable to be checked: **COOR** (coordinate), **DEPL** (displacement), **VITE** (velocity), **ACCE** (acceleration), **FEXT** (external force), **MASN** (nodal mass), **ADFT** (advection-diffusion temperature), **MCPR** (finite volume pressure), **MCRO** (finite volume density), **MCTE** (finite volume temperature), **MCVI** (finite volume velocity), **MCMF** (finite volume mass fraction), **SIGN** (spectral element stress at nodes), **ECRN** (spectral element internal variable at nodes), **FINT** (internal force), **FLIA** (force due to LIAI/LINK COUP), **FDEC** (force due to LINK DECO).

CONT ... VCVI

Name of the element variable to be checked: **CONT** (stress), **EPST** (total deformation), **ECROU** (internal variable), **ENEL** (internal energy), **RHO** (density), **MASE** (element mass), **EPAI** (thickness), **VCVI** (FV centroid velocity).

icomp

Number of the component concerned (for nodes or elements only).

igauss

Number of the gaussian point (integration point) concerned (for elements only). By default **igauss**=1.

BILA ... WECH

Name of the variable associated with the energy balance for the whole calculation, which allows to monitor the stability and the energy transfers: **BILA** energy balance, **WINT** internal energy, **WEXT** work of the external forces, **WCIN** kinetic energy, **WECH** energy exchanged with the external world, injected or lost. For these parameters, the component and **/LECTURE/** are redundant.

valref

Reference value expected.

tolref

Relative tolerance.

LECTURE

Number of the node or element concerned.

Comments :

Only one node or element must be concerned by the validation.

The specification of a Gauss point makes sense only for parameters related to elements, such as: **CONTR**, **EPST** or **ECRO**.

It is possible to check as many values as needed, by repeating the name of the variable. The calculation will be considered correct if **all** checks are correct.

It is sometimes tedious to prepare an input data set for a new test case or benchmark calculation, especially when many quantities must be verified. In some cases, the reference values are not well known a priori from physical considerations, analytical solutions or experimental data. Therefore, sometimes these values are computed by the code itself.

Although this is in some sense a misuse of the **QUAL** directive (because it is no longer a true qualification of results!), it may be useful e.g. to verify non-regression of code results during the development phase.

In such cases, the **AUTO** directive may speed up the process of preparing the input file, in the following way. First, write the qualification directives for the desired quantities, but by setting

arbitrary reference values (e.g. all zero). Specify the **AUTO** directive immediately after **QUAL** and run the code.

In this way, qualification is computed normally, but for each verified quantity the code writes on the listing an extra line, introduced by the sequence **AQ:**, and containing the qualification directive with the 'correct' reference value (i.e. the one found by the code).

By searching all such lines in the listing, cutting them and pasting into the input file, it is possible to obtain a 'correct' input file much more rapidly (and with less errors) than by typing in the correct reference values by hand. Do not forget to remove the **AUTO** keyword once done the job.

Outputs :

The expected and obtained values are printed on the listing, together with the relative error, which is compared with the tolerance.

For each correct (respectively incorrect) check, the phrase: **==> VALIDATION : SUCCES** is printed on the listing (resp. **==> VALIDATION : ECHEC**).

At the end of the calculation, if all is fine the phrase: **==> LE CALCUL EST CORRECT !** is printed, else: **==> LE CALCUL EST FAUX !**.

15.8 "SUITE" OR "FIN"

Object :

The word "SUITE" written immediately after the instruction "CALCUL" enables the next data set to be read, and the corresponding case to be processed, immediately after the first (or current) computation.

Using the word "SUITE" placed after each data set, the user can enter as many data sets as he wants.

The last set must end with the word "FIN".

Syntax :

```
$[ "SUITE" ; "FIN" ]$
```

Comments :

The word "SUITE" is the only word of the directive. It must immediately follow the instruction "CALCUL".

The word "FIN" is compulsory at the end of the data.

16 GROUP ED—POST-TREATMENT BY EUROPLEXUS

Object:

To post-treat a results file containing the EUROPLEXUS results from a previously executed transient calculation.

Syntax :

1/ General syntax :

```

... title ...

<"ECHO">

<"OPNF" . . . >

"RESUL" . . .

<"DIME" . . . "TERM">

"SORT"      $  "ARRET"      . . .  $
              $              $
              $  "FICHIER"   . . .  $
              $              $
              $  "ECRITURE"  . . .  $
              $              $
              $  "GRAPHIQUES" . . .  $
              $              $
              $ ( "VISUALISER" . . . ) $

<"QUAL" . . . >

"FIN"
```

Comments:

These directives are described in detail on the following pages, except for the QUAL directive, which has been already presented on page I.25.

The following page shows a full synopsis of the EUROPLEXUS post-treatment directives.

```

... title ...
<ECHO> <OPNF . . . >
RESU <FORM>
    |<SPLI> ALIC;ALIC TEMP;UNIV <CURR>;UNIV OBSO|
    |nban;'nom_fich'| <GARD> <PSCR>
<DIME <TIMP nimp> TERM>
<FONC ... , see page E.15>
SORT $ ARRE <TEMP time;NUPA npas;NSTO nsto> $
$ FICH <FORM> nfic $FREQ nfre;TFRE tfre $ $
$ $NUPA /LECT/;PASM pasm$ $
$ ECRI <COOR> <DEPL> <VITE> <ACCE> <FINT> <FEXT> $
$ <CONT> <EPST> <ECRO> /CTIM/ $
$ <NOPO;POIN /LECT/> <NOEL;ELEM /LECT/> $
$ <FICH <FORM> K200 ndca /CTIM/ POIN /LECT/ <CHAM>> $
$ GRAP AXTE coef 'nom_axe_0x' $
$ <MINM> <FENE tmin tmax> $
$ <PERF 'nom_fic'> <PERK 'nom_fic'> $
$ (COUR nuco <'nomcourbe'> $
$ $WINT;WEXT;WCIN;BILA;WSUM;DTMI;DTMA;MXSU$ <COMP ico> $
$ $COOR;DEPL;VITE;ACCE$ $
$ $FORC;ADFT;MCP;MCRO$ $
$ $MCTE;MCMF;SIGN;ECRN$ $
$ $LFNO;LFNV;ILNO;DTNO$ $COMP ico;NORM$ NOEU /LECT/ $
$ $CONT;ECRO;EPST;ENEL$ $
$ $WAUX;LFEL;LFEV;DTEL$ COMP ico <GAUS igau> ELEM /LECT/ $
$ $VCVI$ $COMP ico;NORM$ ELEM /LECT/ $
$ $SOMM nbrs*(courbe_i coef_i)$ $
$ $PROD pcoef nbrp*(courbe_k) $ $
$ $INTE courbe_i $ $
$ $DIST /LECT/ $ $
$ $LIBR $ $
$ $MASS;VOLU;BARY;VMOY$ $
$ $IMPU;ECIN;EINT;EEXT$ $
$ $EPDV;EINJ;RESU;IRES$ $
$ $ECRG;DT1 $ <COMP ico;NORM> <REGI nure> $
$ (SCOU nuco <'nomcourbe'> <$T t;NPAS npas;NSTO nsto> $
$ SAXE scoe 'nom_saxe' <INIT> /LECT/ $
$ $COOR;DEPL;VITE;ACCE$ $
$ $FORC;ADFT;MCP;MCRO$ $
$ $MCTE;MCMF;SIGN;ECRN$ $
$ $LFNO;LFNV;ILNO;DTNO$ $COMP ico;NORM$ $
$ $CONT;ECRO;EPST;ENEL$ $
$ $WAUX;LFEL;LFEV;DTEL$ COMP ico <GAUS igau> $
$ (RCOU nuco 'nomcourbe' FICH 'nom_fic' $
$ <RENA 'new_name'> <FACX fx> <FACY fy> $
$ (DCOU nuco <'nomcourbe'> $npt*(x y);FONC ifon$) $
$ ($TRAC;XMGR$ $
$ $K200;LIST$ (nuco) <PS <TEXT>;MIF> AXES coef 'nom_axe_0y' $
$ <XAXE nxax coex 'nom_axe_0x'> $
$ <COLO (co)> <THIC (th)> <DASH (da)> $
$ <XZER> <YZER> <XGRD> <YGRD> <XLOG> <YLOG>) $
$ (VISU $T t;NPAS npas;NSTO nsto$ $
$ <PLAY> $
$ <sequel of interactive commands, see page A.25> $
$ <ENDPLAY>) $
<QUAL ... , see page I.25>

```

FIN

16.1 TITLE AND CHOICE OF RESULTS FILE

Object:

The user gives a title and specifies the file (or files) from which the results to be edited will be read. The file(s) must have been produced during a previous execution of EUROPLEXUS (or during a previous phase of a composite execution, where the various phases are separated by the keyword **SUIT**).

Currently, results may be edited from any of the following file types:

- An ALICE file (either single or split);
- An ALICE TEMPS file;
- A file of type UNIVERSAL CURRENT;
- A file of type UNIVERSAL OBSOLETE.
- A file of type POCHHAMMER.

However, note that a file of type POCHHAMMER can only be read *in addition to* a file of the other types (usually an ALICE file). It cannot be read in by itself.

Syntax :

```

/TITLE/

<"ECHO">

<"OPNF"  < "FORMAT" >  nfic  'nom.fic'>

"RESUL" (<"FORMAT">
        |[ < "SPLI" > "ALIC" ; "ALIC" < "TEMP" > ; "POCH" ;
          "UNIV" <"CURR"> ; "UNIV" "OBSO" ]|
        $[ nban ; 'nom_fich' ]$
        <"GARDE">
        <"PSCR"> )

"ECHO"
```

Like for a normal calculation, this keyword indicates that the EUROPLEXUS input directives will be echoed in the execution window.

"OPNF"

This option may be used to open the chosen results file, like for a normal calculation. Refer to page A.28.

"FORMAT"

This keyword indicates that the chosen results file is a formatted file. By default, this file is unformatted.

"SPLI"

The chosen results file is a set of ALICE split files rather than a single file, produced by the directive `ECRI ... FICH SPLI ALIC ...`, see page G.70.

"ALIC"

The chosen results file is an ALICE file (this is the default).

"ALIC TEMP"

The chosen results file is an ALICE TEMPS file.

"POCH"

The chosen results file is a POCHHAMMER file (which is being read in addition to another results file).

"UNIV CURR"

The chosen results file is a file of type UNIVERSAL CURRENT. The keyword `CURR` may be omitted in this case since this is the default for a file of type UNIVERSAL.

"UNIV OBSO"

The chosen results file is a file of type UNIVERSAL OBSOLETE.

nban

Number of the logical unit on which the results file is stored.

nom_fich

Name of the results file, enclosed in single quotes.

"GARDE"

This keyword allows to keep for the drawings the title read in the results file. Else, it is the title defined above.

"PSCR"

This keyword allows to produce the plots resulting from the `GRAP` directive in PostScript. Since 1995 it is the default, so that this keyword is redundant now.

Comments:

The word RESULT is compulsory.

When it is present, only an edition of results may be done and not a normal calculation.

16.2 DIMENSIONING

Object :

Allocation of memory for the post-treatment of a results file by means of EUROPLEXUS.

If one limits itself to graphical output, EUROPLEXUS automatically allocates the necessary space, so it is no longer necessary to give dimensions. This directive must therefore be omitted in that case.

Syntax:

"DIME"

< "TIMP" nimp >

"TERM"

nimp

Number of time steps for which printing on the listing is requested (see option /CTIM/ of ECRI on page ED.50).

16.3 OUTPUTS

Object :

The following directive enables the types of output to be chosen.

Syntax:

```
"SORT"
$  "ARRET"  <"TEMPS" time ; "NUPAS" npas ; "NSTO" nsto> $
$
$  "FICHIER"      . . . $
$
$  "ECRITURE"     . . . $
$
$  "GRAPHIQUES"  . . . $
$
$ ( "VISUALISER" . . . ) $
```

ARRET

This directive allows to stop reading the results file at the time instant, at the time step or at the time station corresponding to values **time**, **npas** or **nsto**, respectively, specified in the directive, rather than reading the whole file. Note that a storage station is always produces at step 0 (beginning of the transient calculation): this storage station has the index **nsto=0**. This directive is only useful for the qualification of a calculation at intermediate times (and not at the final time as per default), since it may **not** be combined with the other directives **FICH**, **ECRI**, **GRAP** and **VISU**, as indicated in the syntax.

FICHIER

To extract from the chosen results file a certain number of computation steps, and to store them in a new results file which will typically contain less information (less storage stations).

ECRI

To print out results on the EUROPLEXUS listing.

GRAP

To produce graphic outputs. The curves of certain variables are drawn with respect to time or are printed on file(s) in a variety of possible formats.

VISU

To produce (a subset of) the visualizations that are possible during direct execution of the code (see Page A.25). These include graphical rendering either interactively in a window or in batch mode on a file and production of animations. Not all visualization types and features are available, though (see below for details).

Comments:

The keyword **SORT** should appear only once in an input data sequence. Note, however, that the **VISU** sub-directive may be repeated as many times as needed inside the **SORT** directive.

The directives **ARRE**, **ECRI**, **GRAP**, **FICH** and **VISU** are mutually exclusive.

In the case that a graphical output is requested (**GRAP** ... **TRAC**), the produced file is in the PostScript format (a product of Adobe Co.).

16.3.1 CREATING A REDUCED RESULTS FILE

Object:

To extract a certain number of computation steps from the chosen results file, in order to create a new results file which has the same structure as if it was created directly, but typically contains less information (less storage stations).

Syntax:

```
"FICHIER" < "FORMAT" >  nfc  | [ "FREQ"      nfreq      ;  
                                "TFREQ"      tfreq      ;  
                                "NUPAS"      /LECTURE/    ;  
                                "PASMAY"      pasmax      ] |
```

FORMAT

This keyword indicates that the new file created will be formatted. By default, it is unformatted.

nfc

Logical number of the new file.

nfreq

All the results whose step number is a multiple of **nfreq** are extracted from the results file.

tfreq

Time interval between two extracted results.

/LECTURE/

List of the step numbers to be taken.

pasmax

Maximum number of the time step to be copied.

Comments:

The options `FREQ`, `TFREQ`, `NUPAS` and `PASMAX` can be combined.

If the step number required is not stored in file `nfic`, EUROPLEXUS takes the step just above it.

The option `PASMAX` allows e.g. to “clean up” a results file that has become unusable due to a computation error. In fact, one may then create a new file containing the results of the steps from the beginning to a step `pasmax` prior to the encountered error.

Warning:

The logical unit number of the new file (`nfic`) must be different from that of the old one, `nban`, defined in the instruction `RESULT` (see page ED.20).

16.3.2 PRINTOUTS ON THE LISTING

Object :

To print data extracted from a chosen results file onto the EUROPLEXUS listing file or to produce a CASTEM 2000 file for further post-processing by CASTEM 2000.

Syntax :

```
"ECRITURE"
      < "COOR" > < "DEPL" > < "VITE" > < "ACCE" >
      < "FINT" > < "FEXT" > < "FLIA" >
      < "CONT" > < "EPST" > < "ECRO" >
      < "ENER" > < "MCVA" > < "MCVC" >
      < "MCVS" > < "FAIL" > < "VFCC" >
      /CTIM/

$[ "NOPOINT" ; "POINT" /LECTURE/ ]$
$[ "NOELEM" ; "ELEM" /LECTURE/ ]$

< "FICHIER" < FORMAT > "K2000" ndcast /CTIM/
                        "POINT" /LECTURE/
                        < "CHAMELEM" > >
```

"COOR"

Coordinates are printed on the EUROPLEXUS listing.

"DEPL"

Displacements are printed on the EUROPLEXUS listing.

"VITE"

Velocities are printed on the EUROPLEXUS listing.

"ACCE"

Accelerations are printed on the EUROPLEXUS listing.

"FINT"

Internal forces are printed on the EUROPLEXUS listing.

"FEXT"

Total external forces are printed on the EUROPLEXUS listing.

"CONT"

Stresses are printed on the EUROPLEXUS listing.

"EPST"

Total strains are printed on the EUROPLEXUS listing.

"ECRO"

Hardening parameters are printed on the EUROPLEXUS listing.

"ENER"

Energies are printed on the EUROPLEXUS listing.

"MCVA"

Printout of nodal quantities related to multicomponent fluids: pressure, density, temperature, sound speed and mass fractions. Note that this type of printout is incompatible with MCVC and MCVS.

MCVC

Printout of conserved variables (nodal quantities) related to multicomponent fluids: partial densities (ρ_i) of the various components i , momentum ($\rho \underline{u}$) (each spatial component separately), energy (ρE). Note that this type of printout is incompatible with MCVA.

MCVS

Printout of secondary variables (nodal quantities) related to multicomponent fluids: total density (ρ), total pressure p , sound speed c , pressure derivative ($\frac{\partial p}{\partial(\rho e)}$), absolute temperature (T), pressure derivative ($\frac{\partial p}{\partial(\rho_i)}$) for each component, mass fraction (μ_i) for each component. Note that this type of printout is incompatible with MCVA.

"FAIL"

Failure values are printed on the EUROPLEXUS listing.

VFCC

Printout at each selected output time of "element" quantities related to cell-centred Finite Volumes: various volume-related quantities and conserved variables.

"NOPoint"

Do not print any nodal variables. By default the chosen nodal variables are printed for all nodes stored in the results file.

"POINT /LECTURE/"

Print the chosen nodal variables only for the nodes defined in the /LECT/ (provided they are stored in the results file).

"NOELEM"

Do not print any element variables. By default the chosen element variables are printed for all elements stored in the results file.

"ELEM /LECTURE/"

Print the chosen element variables only for the elements defined in the /LECT/ (provided they are stored in the results file).

"FICH"

Produce a CASTEM 2000 results file from the EUROPLEXUS results file.

"FORMAT"

If this keyword is present, the CASTEM 2000 results file is formatted; else, it is unformatted (binary).

ndcast

Logical unit number of the CASTEM 2000 file; the results file is written with the standard SAUVER format of CASTEM 2000. It may be read by CASTEM 2000 by using the command RESTITUER. It is mandatory to specify the list of points for which results have to be included in the file, and if necessary also the word CHAMELEM.

/CTIM/

Reading procedure of the chosen time instants at which the results have to be stored. See page INT.57.

"POIN" /LECTURE/

List of the nodes for which the results are stored for a subsequent post-processing by CASTEM 2000. This directive is mandatory for a file of type "K2000".

"CHAMELEM"

This keyword causes the CHAMELEMS to be included in the CASTEM 2000 file. If it is omitted, the latter will only contain the selected CHAMPOINTS, on the nodes identified by the previous directive POINT.

Comments :

The syntax is the same as for directive ECRI. For more details see page G.10 and following ones.

16.3.3 GRAPHIC OUTPUTS

Object :

To produce drawings or lists on files (in a variety of formats) of different quantities in the form of curves with respect to time, or with respect to a curvilinear abscissa, or combined plots (e.g. sigma/epsilon graphs).

Syntax:

```
"GRAP"  "AXTEMP"  coef  'nom_axe_0x'

< "MINMAX" >

< "PERFO"      'nom_fic'  >

< "PERK"       'nom_fic'  >

< "FENETRE"    tmin  tmax  >

( "COURBE"      . . . )

( "SCOURBE"     . . . )

( "RCOURBE"     . . . )

( "DCOURBE"     . . . )

( "PCOURBE"     . . . )

( "TRACE"       . . . )

( "XMGR"        . . . )

( "K2000"       . . . )

( "LISTE"       . . . )
```

coef

The time values are multiplied by `coef` (this e.g. enables the unit of measure to be changed).

`'nom_axe_0x'`

Name of the time axis (at most 16 characters), enclosed in apostrophes.

MINMAX

Print on the EUROPLEXUS listing the minimum and the maximum values for each curve.

PERFO

The value tables specified in the following **LISTE** directive will be output on an auxiliary file, whose name by default is `<base>.PUN`, where `<base>` is the base name of the current calculation. This directive allows to change the default name into the following `'nom_fic'`.

PERK

The value tables specified in the following **K2000** directive will be output on an auxiliary file, whose name by default is `<base>.PUK`, where `<base>` is the base name of the current calculation. This directive allows to change the default name into the following `'nom_fic'`.

FENETRE

Only the results in a given time interval (time window) are considered.

`tmin`

Minimum time (beginning of the time window).

`tmax`

Maximum time (end of the time window).

COURBE

Define a curve representing the evolution **in time** of a certain variable in the current transient calculation. See below for the full details of this directive.

SCOURBE

Define a curve representing the evolution **in space** of a certain variable in the current transient calculation. The space is a curvilinear abscissa (s) defined by a sequence of nodes. The curve is by default built at the final time of the current calculation. To select a different time, use the **ARRET** directive described on page ED.40. See below for the full details of this directive.

RCOURBE

Read in a curve representing the evolution **in time or in space** of a certain variable in a previously executed EUROPLEXUS calculation. The data are read in from a “punch” file produced by EUROPLEXUS via the **SORT LIST** directive, to be described below. In this way, results from different EUROPLEXUS runs may be compared on the same plot. See below for the full details of this directive.

DCOURBE

Define a curve in the form of a table of (x, y) values. This allows e.g. to build a piecewise analytical solution to be compared with numerical results. It may even be used to input experimental results to be used as a reference solution. See below for the full details of this directive.

PCOURBE

Define a *set of curves* for Pochhammer-Chree post-processing. See below for the full details of this directive.

TRACE

Produce a graph containing one or more of the curves defined above, plotted either versus time, or versus space (curvilinear abscissa), or as a function of another curve (e.g. σ - ϵ type of plot). The graph is produced in the PostScript language on a file.

XMGR

Same as **TRACE** but the graph data are stored on a file which may then be read by the XMGR program (a publicly available software) to produce the actual drawing.

K2000

Same as **TRACE** but the graph data are stored on a file which may then be read by the CASTEM 2000 program to produce the actual drawing.

LISTE

Same as **TRACE** but the graph data are stored on a file which may then be read by a generic external tool to produce the actual drawing. The file format is very simple. This command also allows to store a curve in a certain EUROPLEXUS run and read it in (by the **RCOURBE** directive described above) in a subsequent EUROPLEXUS run, thus opening the way to the production of graphs containing comparisons of results from different EUROPLEXUS calculations, and even analytical curves or experimental data.

Comments:

- The time axis is the same for all drawings produced as a function of time.
- The time window is the same for all drawings produced as a function of time.

Example :

```
"GRAP"  "AXTEMP"  1000.  'TEMPS (MS)'  
"FENETRE"  0.    10E-3  MINMAX  
. . .
```

Curve (Nodal Variables)

Object:

Definition of the variables **relative to nodes** to be drawn or listed.

Syntax :

```
"COURBE" nuco < 'nomcourbe' >

| [ "COOR" ; "DEPL" ; "VITE" ; "ACCE" ; "FINT" ; "FEXT" ;
    "FLIA" ; "ADFT" ; "MCPR" ; "MCRO" ; "MCTE" ; "MCMF" ;
    "MCUX" ; "MCUY" ; "MCUZ" ; "SIGN" ; "ECRN" ; "LFNO" ;
    "LFNV" ; "ILNO" ; "DTNO" ; "VITG" ; "NTLE" ; "MASN" ;
    "FDEC" ] |

| [ "COMP" icomp ; "NORME" ] |

$ [ "NOEU" /LECTURE/ ; "ZONE" /LECTURE/ ] $
```

nuco

Identifier of the curve (reference for TRAC, XMGR, K2000 or LISTE). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc.

COOR

Coordinate.

DEPL

Displacement.

VITE

Velocity.

ACCE

Acceleration.

FINT

Internal force.

FEXT

Total external force.

FLIA

External force due to coupled links (LINK COUP).

ADFT

Advection-diffusion temperature.

MCPR

Finite volume (MC) pressure.

MCRO

Finite volume (MC) density.

MCTE

Finite volume (MC) temperature.

MCMF

Finite volume (MC) component mass fraction.

MCUX

Finite volume (MC) fluid velocity along X computed from the conserved variable ($u_x = (\rho u_x)/\rho$).

MCUY

Finite volume (MC) fluid velocity along Y computed from the conserved variable ($u_y = (\rho u_y)/\rho$).

MCUZ

Finite volume (MC) fluid velocity along Z computed from the conserved variable ($u_z = (\rho u_z)/\rho$).

SIGN

Spectral element stress.

ECRN

Spectral element internal variable.

LFNO

Logarithm in base 2 of the level factor associated with a node in the spatial time step partitioning algorithm.

LFNV

Logarithm in base 2 of the level factor associated with a node, including the neighbours in the spatial time step partitioning algorithm.

ILNO

Flag indicating whether a node is (1) or is not (0) subjected to a link condition, used in the spatial time step partitioning algorithm.

DTNO

Stability time step associated with a node, used in the spatial time step partitioning algorithm.

VITG

Grid velocity (ALE only).

NTLE

Node tree level (only in adaptivity).

MASN

Nodal mass.

FDEC

External force due to decoupled links (LINK DECO).

COMP

Introduces the chosen component.

icom_p

Component number. Default value is 1.

NORM

The norm of the considered vector (where applicable) is drawn.

NOEU /LECTURE/

Number of the node. The procedure /LECTURE/ allows if necessary to read a GIBI object, of which only the first node will be retained.

ZONE /LECTURE/

Set of nodal numbers defining a zone. The contributions of all these nodes are added together. This probably makes sense only for some types of variables (e.g. forces, masses etc.). This can be useful to plot e.g. the total (resultant) force acting on a set of nodes, or the total mass of such nodes. It is an alternative to the use of the **REGI** directive. The difference is that with **REGI** the region must be defined in the main calculation, and it cannot be defined when reading the results file (e.g. an Alice file). The present **ZONE** directive, on the contrary, can be defined “on the fly” when reading any results file (provided this file contains the results of all concerned nodes).

Comments :

The directive **COURBE** can be repeated as many times as desired, but each time with a different identifier.

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

The keyword **FORC** is accepted as a synonym of **FEXT** for backward compatibility, but is obsolescent and should not be used in new input files.

Curve (Element Variables)**Object:**

Definition of the variables **relative to elements** to be drawn or listed.

Syntax:

```
"COURBE"  nuco  < 'nomcourbe' >

| [
  | [ "CONT" ; "ECRO" ; "EPST" ; "ENEL" ; "WAUX" ; "LFEL" ;
    "LFEV" ; "DTEL" ; "ELCE" ; "FAIL" ; "RISK" ; "CERR" ;
    "MAXC" ; "ERRI" ; "CLEN" ; "ILEN" ; "ETLE" ; "MASE" ] |

    "COMP"  comp  $[ "GAUS"  igaus ; "GAUZ"  igauz ]$  ;

  "VCVI"  | [ "COMP"  comp ; "NORM" ] |
] |

$[ "ELEM" /LECTURE/ ; "ZONE" /LECTURE/ ]$
```

nuco

Identifier of the curve (reference for TRAC, XMGR, K2000 or LISTE). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc.

CONT

Stress tensor.

ECRO

Hardening quantity.

EPST

Total deformation tensor.

ENEL

Internal energy.

WAUX

Auxiliary energy terms for the element (see details below).

LFEL

Logarithm in base 2 of the level factor associated with an element in the spatial time step partitioning algorithm.

LFEV

Logarithm in base 2 of the level factor associated with an element including its neighbours in the spatial time step partitioning algorithm.

DTL

Stability time step associate with the element.

ELCE

Coordinates of the barycentre of the element.

FAIL

Failure level of the element.

RISK

Risk level of the element (only if risk is activated). **COMP** must be given to define the kind of risk: **COMP**=1 chooses the risk of eardrum rupture, **COMP**=2 chooses the risk of death. Be aware that when reading results from an Alice file (produced by a previous calculation with risk activation), it is mandatory to (re-)specify the whole **RISK** directive (in particular as concerns the **PROB ...** and **LUNG ...** subdirectives, see page A.30), because the risk is computed with the current values of the optional parameters.

CERR

Constant used in element error indicator calculation (adaptivity), see the **CERR** input keyword of the **ADAP** directive on page B.210.

MAXC

Maximum principal curvature of least-squares fitting function, used for element error indicator calculation (adaptivity).

ERRI

Element error indicator (adaptivity).

CLEN

Current characteristic element length used in element error indicator calculation (adaptivity).

ILEN

Optimal (indicated) characteristic element length resulting from error indicator calculations (adaptivity).

ETLE

Element tree level (adaptivity).

MASE

Element mass.

VCVI

Material or particle velocity (first idim components) in Finite Volumes Cell Centred model. Note that these vectors are not represented at the nodes but at the “elements” (i.e. Finite Volumes) centroids.

COMP

Introduces the component (unused for **ENEL**, **LFEL**, **LFEV** and **DTL**).

icomp

Number of the component.

GAUS

Allows to choose a specific Gauss point index (only for the quantities **CONT**, **EPST** and **ECRO**).

igaus

Number of the Gauss point chosen. The special value 0 means that the average over all Gauss points in the element is taken. The default value is 1, i.e. if neither **GAUS** nor **GAUZ** is specified then the first Gauss Point of the specified element is taken. Note that this default is different from the default in rendering via OpenGL, where 0 (average over all Gauss Points) is assumed.

GAUZ

Allows to choose a specific “lamina” of the (shell) element. The value is the index of the lamina through the thickness (only for the quantities **CONT**, **EPST** and **ECRO**). In this case, the code takes the average value of all Gauss Points belonging to the specified lamina.

igauz

Number of Gauss point through the thickness (i.e. index of the chosen lamina).

NORM

The norm of the VCVI vector is drawn.

ELEM /LECTURE/

Number of the element. The procedure **/LECTURE/** allows if necessary to read a GIBI object, of which only the first element will be retained.

ZONE /LECTURE/

Set of element numbers defining a zone. The contributions of all these elements are added together. This probably makes sense only for some types of variables (e.g. masses). This can be useful to plot e.g. the total (resultant) mass of a set of elements. It is an alternative to the use of the **REGI** directive. The difference is that with **REGI** the region must be defined in the main calculation, and it cannot be defined when reading the results file (e.g. an Alice file). The present **ZONE** directive, on the contrary, can be defined “on the fly” when reading any results file (provided this file contains the results of all concerned elements).

Comments:

The directive **COURBE** can be repeated as many times as desired, but each time with a different identifier.

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

If the keyword **GAUSS** is omitted, the first integration point is considered. If **GAUSS** is set to 0, the average over all integration points is used.

As concerns the auxiliary energy terms for the element (**WAUX**), the following components are available at the moment:

1. Energy dissipated by artificial viscosity (**W_ARD**)
2. Pressure work for fluids $-PdV$ (**W_PDV**)
3. Energy injected or lost at the walls (**W_INJ**)

Curve (Combinations)**Object:**

Definition of **combinations** of the previously defined curves, to be drawn or listed.

Syntax:

```
"COURBE"  nuco  < 'nomcourbe' >

| [  "SOMME"      nbrs*( courbe_i  coef_i )      ;
    "PRODUIT"    pcoef  nbrp*( courbe_k )      ;
    "INTEGRALE"   courbe_i                      ;
    "DISTANCE"    /LECTURE/                    ;
    "LIBR"        ;
    "ADDC"  icou val      ;    "SUBC"  icou val      ;
    "MULC"  icou val      ;    "DIVC"  icou val      ;
    "EXPC"  icou val      ;    "CEXP"  icou val      ;
    "SHIFT" icou val      ;
    "ADD"   icou jcou     ;    "SUB"   icou jcou     ;
    "MUL"   icou jcou     ;    "DIV"   icou jcou     ;
    "EXPF"  icou jcou     ;    "DUP"   icou jcou     ;
    "ABS"   icou          ;    "SEGN"  icou          ;
    "SQRT"  icou          ;    "INV"   icou          ;
    "EXP"   icou          ;    "LN"    icou          ;
    "LOG10" icou          ;    "SIN"   icou          ;
    "COS"   icou          ;    "ASIN"  icou          ;
    "ACOS"  icou          ;    "DIFF"  icou          ;
    "INT"   icou          ;    "AVER"  icou          ;
    "MAX"   icou          ;    "MIN"   icou          ;
    "MEAN"  nc*(icou)     ;    "SMAX"  nc*(icou)     ;
    "SMIN"  nc*(icou)     ;    "JOIN"  nc*(icou)     ;
    "FILT"  "MOYG"  icou nval      ]|
```

nuco

Identifier of the curve (reference for TRAC, XMGR, K2000 or LISTE). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc.

SOMME

The current curve results from the linear combination of **nbrs** curves, among those already defined.

$$\text{result} = \text{coef_1} * \text{courbe_1} \dots + \text{coef_i} * \text{courbe_i} + \dots$$

PRODUIT

The current curve results from the product of **nbrp** curves, among those already defined.

$$\text{result} = \text{pcoef} * \text{courbe_1} \dots * \text{courbe_k} * \dots$$

INTEGRALE

Each point of this curve is the value at time t of the integral between 0 and t of curve number **courbe_i**, supposed already defined.

DISTANCE

The current curve results from the calculation of the distance between the two nodes specified by the following directive **/LECTURE/**.

LIBR

The variable concerned by this curve is computed by the subroutine **GRLIBR**, written by the user.

ADDC

Add to curve **icou** a constant value **val**.

SUBC

Subtract from curve **icou** a constant value **val**.

MULC

Multiply curve **icou** by a constant value **val**.

DIVC

Divide curve **icou** by a constant value **val**.

EXPC

Raise curve **icou** to a constant power **val**.

CEXP

Raise constant **val** to power values in curve **icou** (powers of a constant).

SHIFT

Translate of curve `icou` in its abscissa by a value `val`. Undefined values are set to zero. The abscissa of the generated curve is the same as that of curve `icou`.

ADD

Add curve `jcou` to curve `icou`.

SUB

Subtract curve `jcou` from curve `icou`.

MUL

Multiply curve `icou` by curve `jcou`.

DIV

Divide curve `icou` by curve `jcou`.

EXPF

Raise curve `icou` to power values contained in curve `jcou`.

DUP

Copy of curve `icou` having the abscissa of curve `jcou`. The result is set at zero in the non-overlapping abscissa zones.

ABS

Absolute value of curve `icou`.

SEGN

Sign (unit) function of curve `icou`.

SQRT

Square root of curve `icou`.

INV

Inverse of curve `icou`.

EXP

Exponential of curve `icou`.

LN

Natural logarithm of curve `icou`.

LOG10

Decimal logarithm of curve `icou`.

SIN

Sine of curve `icou`.

COS

Cosine of curve `icou`.

ASIN

Arc sine of curve `icou`.

ACOS

Arc cosine of curve `icou`.

DIFF

Derivative of curve `icou` with respect to its abscissa (usually time).

INT

Integral of curve `icou` with respect to its abscissa (usually time).

AVER

Average value of curve `icou`. This results in a single value, repeated over the whole abscissa.

MAX

Maximum value of a curve `icou`. This results in a single value, repeated over the whole abscissa.

MIN

Minimum value of a curve `icou`. This results in a single value, repeated over the whole abscissa.

MEAN

Arithmetic mean of a set of `nc` curves `icou`.

SMAX

Upper bound of a set of `nc` curves `icou`.

SMIN

Lower bound of a set of `nc` curves `icou`.

JOIN

Union of a set of **nc** curves **icou**. The values from each curve are merged together to form a new curve. This especially makes sense for “curves” consisting of just one point each, or for curves whose definition domains are disjoint.

FILT MOYG

Mobile average on **nval** consecutive values of curve **icou** .

Comments:

The directive **COURBE** can be repeated as many times as desired, but each time with a different identifier.

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

For **SOMME** and **PRODUIT**, the curves starting from which the sum (resp. product) is computed must have identifiers lower than that of the current curve and must have been already defined.

Commands **ADDC** to **SMIN** have been inspired from similar ones present in the **TPLOT** data management system, developed at **JRC** since the 1970's. For these commands, the curves identified by **icou**, **jcou**, etc., must have been already defined. Note also that for any of these commands that involve two or more curves **icou**, **jcou**, etc., with the notable exception of the **DUP** command, the abscissas (i.e. the discrete x -values) of all such curves must be identical, otherwise the combination may not be computed. Note also that, with respect to **TPLOT**, the meanings of **MIN**, **SMIN** and of **MAX**, **SMAX** have been interchanged. Moreover, **MIN** and **MAX** now produce a (uniform-valued) curve rather than the printout of a single value.

The subroutine **GRLIBR** allows to compute a quantity as a function of other quantities defined previously by a directive **COURBE**.

An example of such subroutine is:

Programming example for GRLIBR:

```

      SUBROUTINE GRLIBR(TT,VAL,NT,NTEMAX)
C-----
C
C   CALCUL LIBRE DE GRANDEURS A TRACER EN FONCTION DU TEMPS
C-----
C
C           TT   = TABLEAU DES TEMPS (BANDE ALICE)
C           IT   = NUMERO DU PAS DE TEMPS
C           NT   = NOMBRE DE PAS DE TEMPS TOTAL (BANDE ALICE)
C           ICO  = NUMERO D'UNE COURBE
C   VAL(IT,ICO) = TABLEAU DES GRANDEURS DEFINIES PAR UNE COURBE
C           NTEMAX = NOMBRE MAXIMAL DE POINTS
C
C           REAL TT, VAL
C           DIMENSION TT(NTEMAX),VAL(NTEMAX,*)
C
C-----  EXEMPLE : A = B * C
C   DO 10 IT=1,NT
C 10 VAL(IT,5)=VAL(IT,1)*VAL(IT,3)
C
C-----  EXEMPLE d'INTEGRATION :
C   VAL(1,40)=0.
C   NT1=NT-1
C   DO 10 IT=1,NT1
C 10 VAL(IT+1,40)=VAL(IT,40)+0.5*(VAL(IT+1,22)+VAL(IT,22))
C   *          *(TT(IT+1)-TT(IT))
C           RETURN
C           END

```

Warning: the tables TT and VAL must be in simple precision (R*4).

Curve (Regional Balances)**Object:**

Definition of **quantities related to regions** to be drawn or listed.

Syntax:

```
"COURBE"  nuco  < 'nomcourbe' >
| [ "MASS" ; "VOLU" ; "BARY" ; "VMOY" ; "DIMX" ; "DIMN" ; "VEMX" ;
    "VEMN" ; "IMPU" ; "ECIN" ; "EINT" ; "EEXT" ; "EPDV" ; "EINJ" ;
    "RESU" ; "IRES" ; "ECRG" ; "ECRM" ; "EMAS" ; "FLIR" ] |

$[ "COMP"  icomp ; "NORM" ]$

"REGION"  nureg
```

nuco

Identifier of the curve (reference for TRAC, XMGR, K2000 or LISTE). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc.

MASS

Mass of the region (scalar, computed via XMEL).

VOLU

Volume of the region (scalar).

BARY

Barycenter of the region (vector).

VMOY

Mean velocity of the region (vector).

DIMX

Maximum displacement (absolute) of the region (vector), only components 1 to 3.

DIMN

Minimum displacement (absolute) of the region (vector), only components 1 to 3.

VEMX

Maximum velocity (absolute) of the region (vector), only components 1 to 3.

VEMN

Minimum velocity (absolute) of the region (vector), only components 1 to 3.

IMPU

Impulse (momentum) of the region (vector).

ECIN

Kinetic energy of the region (vector).

EINT

Internal energy of the region (scalar).

EEXT

Work of external forces applied to the region (scalar).

EPDV

Work of pressure forces (PdV) for the region (scalar).

EINJ

Energy injected in the region (scalar).

RESU

Resultant of the external forces applied to the region (vector).

IRES

Impulse due to external forces applied to the region (vector).

ECRG

Sum of the values of ECR on the Gauss points of the region (vector without norm).

ECRM

Average of the ECR over the region.

EMAS

Mass of the region (scalar, computed via the element masses XM0).

FLIR

Resultant of the force due to LINK/LIAI applied at the nodes.

COMP

Introduces the component.

i comp

Index of the component.

NORME

The norm of the chosen vector will be plotted.

nureg

Number of the concerned region in the order of definition.

Comments:

The directive COURBE can be repeated as many times as desired, but each time with a different identifier.

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

The directives COMP and NORM make sense only for vectors: they are possible with BARY, DIMX, DIMN, VMOY, VEMX, VEMN, IMPU, ECIN, RESU, and IRES. Furthermore, NORM does not make sense for ECRG (only COMP is possible).

The directives COMP and NORM make no sense for scalars: MASS, VOLU, EINT, EEXT, EPDV, EINJ and EMAS.

If the keyword COMP is absent, it is the first component that is taken in the case of vectors.

The directive EPDV makes sense only for a stand-alone system, for example the fluid within a reservoir. In the remaining cases, it is suggested to use EEXT, which gives the work of the applied external forces.

The directive EINJ is only valid for material EAU.

Curve (Global Quantities)**Object:**

Definition of some **global quantities** to be drawn or listed, e.g. relative to energy balance or spatial time step partitioning.

Syntax:

```
"COURBE"  nuco  < 'nomcourbe' >

      | [          "WINT"          ;
          "WEXT"          ;
          "WCIN"          ;
          "WTOT"          ;
          "WIMP"          ;
          "WSYS"          ;
          "BILAN"         ;
          "WSUM"    <COMP  icomp> ;
          "DTMI"          ;
          "DTMA"          ;
          "MXSU"          ;
          "DT1"           ;
          "NSPL"          ;
          "NUSP"          ;
          "NSPT"          ;
          "NUSE"          ;
          "NACT"          ;
          "NUSN"          ;
          "LMAX"          ;
          "LMIN"          ] |
```

nuco

Identifier of the curve (reference for TRAC, XMGR, K2000 or LISTE). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc.

WINT

Internal energy.

WEXT

External work.

WCIN

Kinetic energy.

WTOT

Sum of all external energies (see comments below).

WIMP

Energy dissipated during contact/impact calculations.

WSYS

Total energy of the system (see comments below).

BILAN

Energy balance.

WSUM

Sum of the auxiliary energy terms, see below (vector).

DTMI

Minimum time increment in the time spatial step partitioning algorithm. This quantity is available only in calculations with partitioning (**OPTI PART**, see Page H.20).

DTMA

Maximum time increment in the time spatial step partitioning algorithm. This quantity is available only in calculations with partitioning (**OPTI PART**, see Page H.20).

MXSU

Logarithm in base 2 of the maximum depth of the time spatial step partitioning algorithm. This quantity is available only in calculations with partitioning (**OPTI PART**, see Page H.20).

DT1

Time integration step (scalar). This is the time increment that has led to the current time. However, at the initial time of the calculation (step 0, i.e. **NPAS=0**) this quantity does not make sense, so we take **DT2** instead, i.e. the time increment that will lead to the following time.

NSPL

Number of elements which have been split during the current time step.

NUSP

Number of elements which have been unsplit during the current time step.

NSPT

Total number of elements which have been split during the calculation.

NUST

Total number of elements which have been unsplit during the calculation.

NUSE

Number of used elements (active or inactive) at the current time step.

NACT

Number of active elements at the current time step.

NUSN

Number of used (and also of active) nodes at the current time step.

LMAX

Maximum element level among all currently active elements at the current time step.

LMIN

Minimum element level among all currently active elements at the current time step.
Level 0 (unused elements) is not considered in computing this quantity. Also currently used but inactive elements are excluded.

i comp

Index of the chosen component (only for vector quantities).

Comments:

The directive **COUR** can be repeated as many times as desired, but each time with a different identifier.

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

WTOT is the sum of all the “external” energies of the system: the work of external forces, the injected energy, the energy due to oil pyrolysis, etc. WTOT is used in the calculation of the energy balance.

WIMP is the energy dissipated due to contact-impact phenomena. This dissipation may come from the impact model used (soft impact, hard impact) in conjunction with the temporal discretization of the problem.

WSYS is the energy of the system, defined as: $WSYS = WTOT + WIMP$.
In a closed system, WSYS must be conserved.

As concerns the global auxiliary energy terms (WSUM), the following components are available at the moment:

1. Energy dissipated by artificial viscosity (W_ARD)
2. Pressure work for fluids $-PdV$ (W_PDV)
3. Energy injected or lost at the walls (W_INJ)

Note that the global quantities NSPL to LMIN in the above list are available only in calculations with adaptivity and with STAT option activated (OPTI ADAP STAT, see Page H.180).

Curve in space (Nodal Variables)

Object:

Definition of the variables **relative to nodes** to be drawn or listed **as a function of space** and not of time (as by default). The space is here represented by a curvilinear abscissa, which is built up starting by the definition of a sequence of nodes.

Syntax :

```
"SCOURBE"  nuco  < 'nomcourbe' >
            $[ "T" t ; "NPAS" npas ; "NSTO" nsto ]$
            "SAXE" scoe 'nom_saxe' <"INIT"> /LECTURE/

            |[ "COOR" ; "DEPL" ; "VITE" ; "ACCE" ; "FINT" ; "FEXT" ;
              "FLIA" ; "ADFT" ; "MCPR" ; "MCRO" ; "MCTE" ; "MCMF" ;
              "MCUX" ; "MCUY" ; "MCUZ" ; "SIGN" ; "ECRN" ; "LFNO" ;
              "LFNV" ; "ILNO" ; "DTNO" ; "VITG" ; "MASN" ]|

            |[ "COMP" icomp ; "NORME" ]|
```

nuco

Identifier of the curve (reference for TRAC, XMGR, K2000 or LISTE). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc.

t

Time of the desired storage station from which results have to be read in. If option STEP IO is active, then the code looks for the precise time t specified (within a small tolerance) among all stored time stations and, if no such time is found, an error message is issued. If option STEP LIBR is active, the code takes the first stored time station (if any) at a time equal to or greater than the specified time. Again, if no such time is found then an error message is issued.

npas

Time step number of the desired storage station from which results have to be read in.

nsto

Storage index number of the desired storage station from which results have to be read in.

SAXE

Introduces the definition of the curvilinear abscissa to be used as x -axis for the curve.

scoe

Multiplicative coefficient for the values of the curvilinear abscissa used as x -axis for the curve. By default, the abscissa is built up according to the distance between nodes, in the order they are defined in the following **/LECTURE/**.

'nom_saxe'

Name of the curvilinear abscissa. This will appear on plots, etc.

INIT

Build up curvilinear abscissa by using the *initial* nodal positions and not the *current* ones.

/LECTURE/

List of nodes defining the curvilinear abscissa. They are taken in the order given by the user (not re-ordered).

COORD

Coordinate.

DEPL

Displacement.

VITE

Velocity.

ACCE

Acceleration.

FINT

Internal force.

FEXT

Total external force.

FLIA

External force due to liaisons (links).

ADFT

Advection-diffusion temperature.

MCPR

Finite volume (MC) pressure.

MCRO

Finite volume (MC) density.

MCTE

Finite volume (MC) temperature.

MCMF

Finite volume (MC) component mass fraction.

MCUX

Finite volume (MC) fluid velocity along X computed from the conserved variable ($u_x = (\rho u_x)/\rho$).

MCUY

Finite volume (MC) fluid velocity along Y computed from the conserved variable ($u_y = (\rho u_y)/\rho$).

MCUZ

Finite volume (MC) fluid velocity along Z computed from the conserved variable ($u_z = (\rho u_z)/\rho$).

SIGN

Spectral element stress.

ECRN

Spectral element internal variable.

LFNO

Logarithm in base 2 of the level factor associated with a node in the spatial time step partitioning algorithm.

LFNV

Logarithm in base 2 of the level factor associated with a node, including the neighbours in the spatial time step partitioning algorithm.

ILNO

Flag indicating whether a node is (1) or is not (0) subjected to a link condition, used in the spatial time step partitioning algorithm.

DTNO

Stability time step associated with a node, used in the spatial time step partitioning algorithm.

VITG

Grid velocity (ALE only).

MASN

Nodal mass.

COMP

Introduces the chosen component.

i comp

Component number.

NORM

The norm of the considered vector (where applicable) is drawn.

Comments :

The directive **SCOURBE** can be repeated as many times as desired, but each time with a different identifier. Identifiers should of course also be different from those of curves defined by the other curve-definition directives (**COURBE**, **RCOURBE**, **DCOURBE**).

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

If neither **T** nor **NPAS** nor **NSTO** are specified, then the last storage station is taken by default.

The keyword **FORC** is accepted as a synonym of **FEXT** for backward compatibility, but is obsolescent and should not be used in new input files.

Curve in space (Element Variables)

Object:

Definition of the variables **relative to elements** to be drawn or listed **as a function of space** and not of time (as by default). The space is here represented by a curvilinear abscissa, which is built up starting by the definition of a sequence of nodes.

Syntax :

```
"SCOURBE"  nuco  < 'nomcourbe' >
            $[ "T" t ; "NPAS" npas ; "NSTO" nsto ]$
            "SAXE" scoe 'nom_saxe' <"INIT"> /LECTURE/
            < "SUPP" /LECT_ELEM/ >

            |[ "CONT" ; "ECRO" ; "EPST" ; "ENEL" ; "WAUX" ; "LFEL" ;
              "LFEV" ; "DTEL" ; "CERR" ; "MAXC" ; "ERRI" ; "CLEN" ;
              "ILEN" ; "MASE" ]|

            "COMP" icomp |[ "GAUS" igauss ; "GAUZ" igausz ]|
```

nuco

Identifier of the curve (reference for TRAC, XMGR, K2000 or LISTE). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc.

t

Time of the desired storage station from which results have to be read in. By default, the last storage station is taken.

npas

Time step number of the desired storage station from which results have to be read in. By default, the last storage station is taken.

nsto

Storage index number of the desired storage station from which results have to be read in. By default, the last storage station is taken.

SAXE

Introduces the definition of the curvilinear abscissa to be used as x -axis for the curve.

scoe

Multiplicative coefficient for the values of the curvilinear abscissa used as x -axis for the curve. By default, the abscissa is built up according to the distance between nodes, in the order they are defined in the following **/LECTURE/**.

'nom_saxe'

Name of the curvilinear abscissa. This will appear on plots, etc.

INIT

Build up curvilinear abscissa by using the *initial* nodal positions and not the *current* ones.

/LECTURE/

List of nodes defining the curvilinear abscissa. They are taken in the order given by the user (not re-ordered).

SUPP /LECT.ELEM/

The optional keyword **SUPP** allows to specify, via the following **/LECT.ELEM/** directive, the geometrical support (list of the elements) to be considered for the projection onto nodes of the chosen element variable. By default, all elements of continuum, shell or beam type present in the mesh are considered. However, the default behaviour may lead to wrong results, for example in the case of shells whose nodes are merged with continuum fluid elements. If one traces, say, the pressure in the fluid, then also the (unrelated) value in the shell would be considered by default. To avoid the problem, specify **SUPP LECT fluid TERM**, where **fluid** is an object containing only the fluid elements. The **SUPP** directive should also be used in *adaptivity* for the definition of space curves (**SCOU**) involving elemental quantities, even in the absence of merged nodes. This allows to avoid ambiguities in the formation of the curvilinear abscissa starting from the base nodes, which are the only ones declared by the user.

CONT

Stress tensor.

ECRO

Hardening quantity.

EPST

Total deformation tensor.

ENEL

Internal energy.

WAUX

Auxiliary energy terms for the element (see details below).

LFEL

Logarithm in base 2 of the level factor associated with an element in the spatial time step partitioning algorithm.

LFEV

Logarithm in base 2 of the level factor associated with an element including its neighbours in the spatial time step partitioning algorithm.

DTL

Stability time step associate with the element.

CERR

Constant used in element error indicator calculation (adaptivity), see the **CERR** input keyword of the **ADAP** directive on page B.210.

MAXC

Maximum principal curvature of least-squares fitting function, used for element error indicator calculation (adaptivity).

ERRI

Element error indicator (adaptivity).

CLEN

Current characteristic element length used in element error indicator calculation (adaptivity).

ILEN

Optimal (indicated) characteristic element length resulting from error indicator calculations (adaptivity).

MASE

Element mass.

COMP

Introduces the component (unused for ENEL, LFEL, LFEV and DTEL).

icomp

Number of the component.

GAUSS

Introduces the Gauss point (only for the quantities CONT, EPST and ECRO).

igau

Number of Gauss point chosen.

GAUZ

Introduces the Gauss point through the thickness (only for the quantities CONT, EPST and ECRO).

igau

Number of Gauss point through the thickness.

Comments :

The directive **SCOURBE** can be repeated as many times as desired, but each time with a different identifier. Identifiers should of course also be different from those of curves defined by the other curve-definition directives (**COURBE**, **RCOURBE**, **DCOURBE**).

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

If the keyword **GAUSS** is omitted, the first integration point is considered. If **GAUSS** is set to 0, the average over all integration points is used.

As concerns the auxiliary energy terms for the element (**WAUX**), the following components are available at the moment:

1. Energy dissipated by artificial viscosity (W_ARD)
2. Pressure work for fluids $-PdV$ (W_PDV)
3. Energy injected or lost at the walls (W_INJ)

Curve Read In from a File**Object:**

Definition of curves to be read in from a file. The file must have been previously produced by EUROPLEXUS itself by means of the `SORT LIST` command, and is a file of type "PUNCH", see page ED.125.

Syntax :

```
"RCOURBE"  nuco      'nomcourbe'    FICH 'nom_fic'  
           <"RENAME" 'new_name'>  <"FACX"  fx>  <"FACY"  fy>
```

nuco

Identifier of the curve (reference for `TRAC`, `XMGR`, `K2000` or `LISTE`). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc. Unlike for the other curve definitions, this name is mandatory here and must exactly match the name by which the curve has been stored on the punch file during a previous EUROPLEXUS run.

nom_fich

Name of the punch file enclosed in apostrophes.

RENA

Allows to change the name of the curve if so desired.

new_name

New name of the curve enclosed in apostrophes.

FACX

Allows to change the x -scale of the curve if so desired.

fx

Multiplicative factor for the x -values.

FACY

Allows to change the y -scale of the curve if so desired.

fy

Multiplicative factor for the y -values.

Comments :

The directive **RCOURBE** can be repeated as many times as desired, but each time with a different identifier. Identifiers should of course also be different from those of curves defined by the other curve-definition directives (**COURBE**, **SCOURBE**, **DCOURBE**).

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

This directive allows to retrieve curves from different calculations and to compare them by plotting them on the same graph. The time scales and the number of points of the various curves are different in general. The program automatically takes this into account.

Warning :

A certain care should be taken concerning the units of measurement of curves stored and later retrieved for plotting. Note that curves are stored with exactly the x -values and the y -values as they would appear on a drawing. In particular, if the coefficients **AXTE coef**, see page ED.60 and **AXES coef**, see page ED.125, are not unitary, the stored values are multiplied by these coefficients.

When the data are subsequently read in by **RCOU**, the scaling is already included. So, plotting them by re-specifying again **AXTE coef** and/or **AXES coef** would probably not have the desired effect, since the coefficients would be applied twice! The results may be particularly confusing if the curves read from file are plotted together with “normal” curves (for which the coefficients are only applied once).

There is a simple way of avoiding this type of problem: when defining curves to be stored on file for subsequent plottings or comparisons, it is advisable to always specify **AXTE 1.0** and **AXES 1.0**. In this way all curves are saved with their “native” units of measurement. Any scale coefficients may be applied later, during the actual plotting phase.

In case of need, it is possible to assign a new name to a read-in curve by means of the **RENA** directive. This is the name that will appear on the plot legend. However, do not confuse this with the original name of the curve (**nomcourbe**) which must in any case exactly match the name stored in the file in order to select the desired curve.

A mechanism for changing the scales of a read-in curve both in x and in y is offered by means of the **FACX** and **FACY** directives. This is another way of overcoming the difficulties mentioned above concerning the scale factors. However, their use should be avoided whenever possible. The method outlined above of using unit factors at storage is cleaner and much preferable.

Curve Defined By The User

Object:

Definition of curves in the form of tables containing a sequence of $(x-y)$ values. These curves may represent a (piecewise) analytical solution, or even an experimental result, to be compared with numerical solutions by EUROPLEXUS.

The table containing the couples of values may be specified directly within the present directive, or refer to a function previously defined by the directive **FONCTION**, see page E.15, or represent an analytical solution to a perfect gas shock tube problem. The second possibility allows to plot and to visually check the functions which are defined and used in the calculation.

Syntax :

```
"DCOURBE"  nuco  < 'nomcourbe' >
            | [ npt*(x y) ;
              "FONC" ifonc ;
              "SHTU" "GAMM" gamm "ROM" rom "ROP" rop
                  "EINT" eint "LENM" lenm "LENP" lenp "TIME" time
                  "NRAR" nrar "VARI" vari ] |
```

nuco

Identifier of the curve (reference for TRAC, XMGR, K2000 or LISTE). A (unique) integer number, freely chosen by the user, by which the curve may be successively referred to when needed.

'nomcourbe'

Name of the curve (reference for the user). This will appear on plots, etc.

npt

Number of $(x-y)$ couples defining the curve (i.e. number of points). In this case the values table is specified directly.

x

Value of the abscissa.

y

Corresponding value of the ordinate.

ifonc

Index of a function previously defined by the directive **FONCTION**, see page E.15.

SHTU

Introduces the parameters of the perfect gas shock tube problem for which the analytical solution (space curve of a chosen variable along the tube length) has to be generated. The high-pressure zone is assumed to be in the left part of the tube, of length **lenm**. The low-pressure zone is assumed to be in the right part of the tube, of length **lenp**. The initial specific energy (and hence the initial temperature) is the same in both parts. The initial density, and hence the initial pressure, is higher in the left part than in the right part of the tube.

gamm

Ratio γ between the specific heat C_p at constant pressure and the specific heat C_v at constant volume of the perfect gas.

rom

Initial density ρ_m in the left part of the tube (high-pressure zone).

rop

Initial density ρ_p in the right part of the tube (low-pressure zone).

eint

Initial specific energy i_0 of the perfect gas.

lenm

Length l_m of the left part of the tube (high-pressure zone).

lenp

Length l_p of the right part of the tube (low-pressure zone).

time

Time t at which the analytical solution should be produced.

nrar

Number of spatial intervals n_r at which the analytical solution has to be computed in the rarefaction zone.

vari

Desired output variable for the analytical solution: 1 means pressure, 2 means density, 3 means specific internal energy, 4 means sound speed, 5 means velocity.

Comments :

The directive **DCOURBE** can be repeated as many times as desired, but each time with a different identifier. Identifiers should of course also be different from those of curves defined by the other curve-definition directives (**COURBE**, **SCOURBE**, **RCOURBE**).

Curve identifiers may be freely chosen by the user, and the order in which they are given is irrelevant.

This directive allows to define arbitrary curves and to compare them with curves built up from EUROPLEXUS results by plotting them on the same graph. The time scales (or more generally the abscissas) and the number of points of the various curves are different in general. The program automatically takes this into account.

If a curve is specified by means of a function previously defined by the directive **FONCTION**, then:

- If the function is of type **TABL**, then the $(x-y)$ values of the table function are directly used for the curve.
- If the function is of type **ROUT** (see user routine **TABANA**) or of type **LSQU** (least-squares fitting), then the x -values are not specified in the function. The code will try to use the stored time values for the current calculation as the x -values and to compute the corresponding y -values by calling the specified function.
- Other types of functions are not accepted at the moment and an error message is issued.

Set of Pochhammer-Chree curves**Object:**

Automatic generation of a *set of* curves for Pochhammer-Chree equation verification. The user must have previously read in results from a .POC file by means of the RESU directive, in addition to reading (global) results from an ordinary results file (typically an ALICE file).

Syntax :

```
"PCOURBE" "YOUN" youn "NU" nu "RHO" rho "R" r  
          "NM" nm "IDOF" idof  
          <"DHAR" dhar "TOL" tol "STEP" step "N1" n1 "AXTE" axte "FREQ" freq>
```

youn

Young's modulus of the bar material.

nu

Poisson's coefficient of the bar material.

rho

Density of the bar material.

r

Radius of the cylindrical bar.

nm

Number of the dispersive modes that will be calculated.

idof

Global dof along which the chosen variable is considered: 1 means radial direction, 2 means axial direction. A 2D axisymmetric calculation (with the bar axis directed vertically along the *y*-axis) is assumed.

dhar

Number of harmonics (or frequencies) that participate in the solution. In the case of the single harmonic excitation it should be 1. By default, 250 harmonics are taken.

tol

Relative error between an analytical and numerical solution. By default, it is 0.05.

step

Number of increments that will be used in the area of the relative error in order to identify a solution. By default, it is 200.

n1

Identifier (number) of the first generated curve. By default, it is 1.

axte

The name of the x-axis that will be used in the plotting of the curves. By default, it is 'RAD/WAVELENGTH'.

freq

The frequency of the excitation load (used only for the case of the single harmonic load).

Comments :

The directive **PCOURBE automatically** generates three sets of curves. The first set (ranging from **n1** to **n1 + nm - 1**) contains the analytical solutions, one for each chosen mode. These curves have the following names: **Mode_1**, **Mode_2** etc. The second set (ranging from **n1 + nm** to **n1 + 2*nm - 1**) contains the numerical solutions, one for each chosen mode. These curves have the following names: **Nume_1**, **Nume_2** etc. The third set (ranging from **n1 + 2*nm** to **n1 + 2*nm + nlines*dhar - 1**) contains the wavenumber spectrum for all the frequencies of interest for all the lines (parallel to the axis of the rod) of the calculation. The peaks on the wavenumber spectrum indicates the specific mode wavenumber for each frequency. These curves have the following name: **LINE_1**, **LINE_2** etc.

Note that any pre-existing curves with the same identifiers will be erased.

The phenomenon of dispersion is the reason why waves with different wavelengths will travel at different speed in the same material. The new module is dealing with the propagation of compressional waves in isotropic cylinders. It calculates the dispersion curves corresponding to each mode of propagation. The dispersion curves for each mode of propagation show the relationship between the phase velocity and the wavelength of a specific material. The procedure of defining those curves is described below in 7 steps

- An axial step function load is imposed in the circular face of the bar.
- Velocity versus time data are calculated and stored at equally spaced points along a pre-defined line in the axial direction of the bar.

- An FFT analysis is performed for each set of these time data in order to obtain the frequency spectrum for each point. This spectrum data are calculated and stored in order to be used in the next step.
- For each frequency, a history in the space domain across the predefined line is calculated and stored. This history can be calculated if the value of spectrum data is used for every point of the line for the desired frequency.
- By performing an FFT analysis on the space domain history across the predefined line, a wavenumber spectrum can be obtained for each frequency. These results corresponds to the third set of curves produced under **PCOURBE** directive.
- Each peak of the wavenumber spectrum corresponds to specific mode wavenumber for each frequency. The identification of the peaks for each frequency, leads to one point on the dispersion curves for each mode (for the modes that appeared in the desired frequency). Each peak indicates a wave number for the desired frequency and from this pair of values (wavenumber and frequency) the phase velocity and the wavelength of the mode can be defined. Lower modes peaks are located in higher wavenumbers.
- Finally the numerical results are compared with the analytical results of Pochhammer-Cree solution.

Drawings (TRACE)**Object:**

This instruction is aimed at defining the drawings to be produced.

Syntax:

```
"TRACE" ( nuco ) $["PS" ; <"TEXT"> ; "MIF" ]$
    "AXES"      coef 'nom_axe_0y'
    <"XAXE" nxax coex 'nom_axe_0x'>
    <"COLO" (co)>
    <"THIC" (th)>
    <"DASH" (da)>
    < $[ "NOLI" ; "LINE" (li) ]$ >
    <"SYMB" <(sy)>> <"SYSC" sysc>
    <"NOXL" (nx)>
    <"NOYL" (ny)>
    <"XZER"> <"YZER">
    <"XGRD"> <"YGRD">
    <"XLOG"> <"YLOG">
    <"XMIN" xmin "XMAX" xmax $[ "DX" dx ; "NX" nx ]$>
    <"YMIN" ymin "YMAX" ymax $[ "DY" dy ; "NY" ny ]$>
```

nuco

Identifiers of the curves to be drawn (at most 12 curves).

PS

Draw on a PostScript file (this is the default).

TEXT

In addition to drawing on a PostScript file, also produce a list of the drawn data in tabular form (*x*-value, *y*-value) on a text file. The name of this file is <base>.txt, where <base> is the base name of the current calculation.

MIF

Draw on a MIF file. MIF is Adobe FrameMaker's language and may be suited to embed the graphics in a FrameMaker document. The drawing remains fully editable in FrameMaker (line style, colors, fonts etc.).

coef

Multiplying coefficient to change the units of the **Oy** axis.

'nom_axe_Oy'

Name of the **Oy** axis (at most 16 characters).

nxax

Optional identifier of a curve to be used for the x -axis. By default, the drawing of the specified curves is done vs. time. However, by specifying the **XAXE** sub-directive, it is possible to produce a combined graph in which one or more quantities are plotted vs. another quantity rather than vs. time. For example, a σ - ϵ graph may be produced.

coex

Multiplying coefficient to change the units of the **Ox** axis.

'nom_axe_Ox'

Name of the **Ox** axis (at most 16 characters).

COLO

Optional keyword that introduces the colors to be used for the various curves. If omitted, all curves are drawn in black.

co

Name of the color for the curve, (**not** enclosed in quotes). This must be repeated exactly as many times as there are curves in the drawing (see **nuco** above). The valid names are those of Cast3m, i.e. **bleu**, **roug**, **rose**, **vert**, **turq**, **jaun**, **blan** or **noir**.

THIC

Optional keyword that introduces the line thicknesses to be used for the various curves, in points. If omitted, all curves are drawn with a line thickness of 0.1 points.

th

Line thickness for the curve, in points. This must be repeated exactly as many times as there are curves in the drawing (see **nuco** above).

DASH

Optional keyword that introduces the dash patterns to be used for the various curves. If omitted, all curves are drawn as solid lines.

da

Code for the curve dash pattern. This must be repeated exactly as many times as there are curves in the drawing (see **nuco** above). Valid dash pattern codes are: 0 for a solid line, 1 for long dashes, 2 for medium dashes, 3 for short dashes, 4 for extra-short dashes, and 5 for long-short dashes.

NOLI

Do not draw any lines connecting points on (all) the curves.

LINE

Choose which curve(s) should be drawn as lines or not.

li

Code for the line connecting the curve points. This must be repeated exactly as many times as there are curves in the drawing (see **nuco** above). Valid line codes are: 0 means no line, 1 means line (with the chosen color, thickness and dash pattern, if any).

SYMB

Draw a symbol at each data point on each of the curves. The symbol is drawn *in addition* to the curve line. To remove the line (leaving only the symbols), use the **NOLI** or the **LINE** (**li**) keywords described above. To selectively choose which curves will get symbols, and/or the symbol used for each curve, specify the following (optional) sequence (**sy**). By default (no (**sy**) specified), symbol types 1 to 12 (see below) are used for curves 1 to 12.

sy

Code for the symbol drawn on each curve data point. If present, this must be repeated exactly as many times as there are curves in the drawing (see **nuco** above). Symbols are drawn with the same color and thickness as the associated curve. Valid symbol codes are: 0 no symbol, 1 plus, 2 cross, 3 square, 4 octagon, 5 triangle north, 6 triangle south, 7 triangle east, 8 triangle west, 9 hourglass, 10 hourglass horizontal, 11 diamond, 12 Y, 13 Z.

SYSC

Introduce a symbol scaling factor **sysc**. By default, the factor is 1.0.

NOXL

Optional keyword that introduces the definition of whether or not the various curves participate in the definition of the x -axis (automatic search of the limits and of the major and minor subdivisions). If omitted, all curves participate in the definition of the x -axis.

nx

Code for the curve participation in the definition of the x -axis. This must be repeated exactly as many times as there are curves in the drawing (see **nuco** above). Valid codes are: 0 means that the curve participates in the definition of the axis, 1 means that the curve is ignored in definition of the axis.

NOYL

Optional keyword that introduces the definition of whether or not the various curves participate in the definition of the y -axis (automatic search of the limits and of the major and minor subdivisions). If omitted, all curves participate in the definition of the y -axis.

ny

Code for the curve participation in the definition of the y -axis. This must be repeated exactly as many times as there are curves in the drawing (see **nucO** above). Valid codes are: 0 means that the curve participates in the definition of the axis, 1 means that the curve is ignored in definition of the axis.

XZER

Draw a vertical dotted line in correspondence of the abscissa $x = 0$.

YZER

Draw a horizontal dotted line in correspondence of the ordinate $y = 0$.

XGRD

Draw vertical grid lines at every major axis tick.

YGRD

Draw horizontal grid lines at every major axis tick.

XLOG

Use a logarithmic (10-base) scale for the x -axis instead of a linear scale. Obviously, all x -values must be strictly positive.

YLOG

Use a logarithmic (10-base) scale for the y -axis instead of a linear scale. Obviously, all y -values must be strictly positive.

XMIN

Use the specified lower limit for the x -axis instead of computing it automatically.

XMAX

Use the specified upper limit for the x -axis instead of computing it automatically.

DX

Use the specified scale increment for the x -axis instead of computing it automatically.

NX

Use the specified number of increments for the x -axis instead of computing it automatically.

YMIN

Use the specified lower limit for the y -axis instead of computing it automatically.

YMAX

Use the specified upper limit for the y -axis instead of computing it automatically.

DY

Use the specified scale increment for the y -axis instead of computing it automatically.

NY

Use the specified number of increments for the y -axis instead of computing it automatically.

Comments:

The instruction TRAC may be repeated as many times as desired.

It is possible to use the same curve (same identifier) for several drawings.

Normally the axes scales are computed automatically. However, the user may take full control of this process by specifying XMAX ... and / or YMAX When specifying a lower bound also the corresponding upper bound and either the increment or the number of increments must be specified as well.

Examples:

```
"TRAC" 1 4 2 "AXES" 1. 'PRESSION (PA) '  
"TRAC" 1 2 "AXES" 1E-6 'PRESSION (MPA) '  
"TRAC" 6 "AXES" 1E-6 'STRESS (MPA)' "XAXE" 5 1.0 'STRAIN'
```

Output on file (XMGR)**Object:**

Definition of the variables to be printed on the auxiliary files directly readable by the XMGR software (Copyright Paul J. Turner). See also the directive PERK on page ED.60, which allows to change the default name of the output file.

Syntax:

```
"XMGR"    (  nuco  )    "AXES"      coef 'nom_axe_0y'  
          <"XAXE" nxax coex 'nom_axe_0x'>
```

nuco

Identifiers of the curves to be printed (at most 12 curves).

coef

Multiplying coefficient to change the units of the 0y axis.

'nom_axe_0y'

Name of the 0y axis (at most 16 characters).

nxax

Optional identifier of a curve to be used for the x -axis. By default, the drawing of the specified curves is done vs. time. However, by specifying the XAXE sub-directive, it is possible to produce a combined graph in which one or more quantities are plotted vs. another quantity rather than vs. time. For example, a σ - ϵ graph may be produced.

coex

Multiplying coefficient to change the units of the 0x axis.

'nom_axe_0x'

Name of the 0x axis (at most 16 characters).

Comments:

The **XMGR** directive may be repeated as many times as needed.

The use of this directive is identical to that of directive **TRACE**. It is possible to combine them by using the same curves:

Example:

```
"TRACE"      1 4 2  "AXES"  1.  'PRESSION (Pa) '  
"XMGR"       4 2  "AXES"  1.  'PRESSION (Pa) '
```

The files created for **XMGR** have names of the form: **<base_xxx>.MGR**, where **<base>** is the base name of the current calculation and **xxx** is a counter. A separate file is produced for each **XMGR** directive. If no base name is available, then the file name becomes **TRACXMGR_xxx.MGR**.

It is possible to use the same curve (same identifier) for more than one list.

Examples:

```
"XMGR"      1 4 2  "AXES"  1.  'PRESSION (Pa) '  
"XMGR"      1 2  "AXES" 1E-6 'PRESSION (MPa) '  
"XMGR" 6 "AXES" 1E-6 'STRESS (MPa)' "XAXE" 5 1.0 'STRAIN'
```

Output on file (K2000)**Object:**

Definition of the variables to be printed on an auxiliary file directly readable by the CASTEM 2000 software. See also the directive **PERF** on page ED.60, which allows to change the default name of the output file.

Syntax:

```
"K2000"    (  nuco  )    "AXES"      coef 'nom_axe_0y'  
              <"XAXE" nxax coex 'nom_axe_0x'>
```

nuco

Identifiers of the curves to be printed (at most 12 curves).

coef

Multiplying coefficient to change the units of the 0y axis.

'nom_axe_0y'

Name of the 0y axis (at most 16 characters).

nxax

Optional identifier of a curve to be used for the x -axis. By default, the drawing of the specified curves is done vs. time. However, by specifying the **XAXE** sub-directive, it is possible to produce a combined graph in which one or more quantities are plotted vs. another quantity rather than vs. time. For example, a σ - ϵ graph may be produced.

coex

Multiplying coefficient to change the units of the 0x axis.

'nom_axe_0x'

Name of the 0x axis (at most 16 characters).

Comments:

The K2000 directive may be repeated as many times as needed.

The use of this directive is identical to that of directive TRACE. It is possible to combine them by using the same curves:

Example:

```
"TRACE"      1 4 2  "AXES"  1.  'PRESSION (Pa)'  
"K2000"      4 2  "AXES"  1.  'PRESSION (Pa)'
```

The formatted file may be directly inserted in the input data for CASTEM 2000. The contained objects are of type "LISTREEL", and the names are "L_TEMPS" for the time and "L_number" for the curves (number is the curve identifier).

It is possible to use the same curve (same identifier) for more than one list.

Examples:

```
"K2000"      1 4 2  "AXES"  1.  'PRESSION (Pa)'  
"K2000"      1 2  "AXES" 1E-6 'PRESSION (MPa)'  
"K2000" 6 "AXES" 1E-6 'STRESS (MPa)' "XAXE" 5 1.0 'STRAIN'
```

Output on file (LIST)**Object:**

Definition of the variables to be printed on an auxiliary file of type "PUNCH" (see also the directive PERF).

Syntax:

```
"LISTE"    (  nuco  )    "AXES"      coef 'nom_axe_0y'  
              <"XAXE" nxax coex 'nom_axe_0x'>
```

nuco

Identifiers of the curves to be printed (at most 12 curves).

coef

Multiplying coefficient to change the units of the 0y axis.

'nom_axe_0y'

Name of the 0y axis (at most 16 characters).

nxax

Optional identifier of a curve to be used for the x -axis. By default, the drawing of the specified curves is done vs. time. However, by specifying the **XAXE** sub-directive, it is possible to produce a combined graph in which one or more quantities are plotted vs. another quantity rather than vs. time. For example, a σ - ϵ graph may be produced.

coex

Multiplying coefficient to change the units of the 0x axis.

'nom_axe_0x'

Name of the 0x axis (at most 16 characters).

Comments:

The LISTE directive may be repeated as many times as needed.

The use of this directive is identical to that of directive TRACE. It is possible to combine them by using the same curves:

Example:

```
"TRACE"      1 4 2  "AXES"  1.  'PRESSION (Pa) '
"LISTE"       4 2  "AXES"  1.  'PRESSION (Pa) '
```

The tables come out as nbco blocks of NT lines with two numbers (x - y values) each each. The first value is the abscissa (by default the time), and the second value is the corresponding ordinate (y -value). Each block therefore fully describes one curve. Blocks are given in the same order as they appear in directive LISTE.

To facilitate the subsequent reading of these tables, each block is preceded by three description lines:

- On the first line, after the word VALEURS there is NT, the number of lines of the block, that is also the number of x - y couples. Then comes the word COMPOSANTES, followed by the number of y -value columns (this number is always 1).
- On the second line, which starts by *, are given the names of the 0x axis and of the 0y axis (string nom_axe_0y of directive AXES).
- On the third line, which also starts by *, are given the names of the curves (nomcourbe) defined in COURBE.

It is possible to use the same curve (same identifier) for more than one list.

Examples:

```
"LISTE"      1 4 2  "AXES"  1.  'PRESSION (Pa) '
"LISTE"       1 2  "AXES" 1E-6 'PRESSION (MPa) '
"LISTE" 6 "AXES" 1E-6 'STRESS (MPa)' "XAXE" 5 1.0 'STRAIN'
```

Warning :

A certain care should be taken concerning the units of measurement of curves stored and later retrieved for plotting. Note that curves are stored with exactly the x -values and the y -values as they would appear on a drawing. In particular, if the coefficients AXTE coef, see page ED.60 and AXES coef, see above, are not unitary, the stored values are multiplied by these coefficients.

When the data are subsequently read in by RCOU, the scaling is already included. So, plotting them by re-specifying again **AXTE coef** and/or **AXES coef** would probably not have the desired effect, since the coefficients would be applied twice! The results may be particularly confusing if the curves read from file are plotted together with “normal” curves (for which the coefficients are only applied once).

There is a simple way of avoiding this type of problem: when defining curves to be stored on file for subsequent plottings or comparisons, it is advisable to always specify **AXTE 1.0** and **AXES 1.0**. In this way all curves are saved with their “native” units of measurement. Any scale coefficients may be applied later, during the actual plotting phase.

16.3.4 VISUALIZATIONS

Object:

To produce, by reading results stored in the results file, (a subset of) the visualizations that are possible during direct execution of the code (see Page A.25). These include graphical rendering interactively in a window or in batch mode on file and production of animations. Not all visualization types and features are available, though (see below for details).

Syntax:

```
( "VISU" $ "T" t ; "NPAS" npas ; "NSTO" nsto $  
  <PLAY>  
  <sequel of interactive commands, see page A.25>  
  <ENDPLAY>  
)
```

t

Time of the desired (initial) storage station from which results have to be read in. Subsequent time stations may then be reached by suitable commands (e.g. **GO** and **FREQ**) in the **PLAY ... ENDPLAY** sequence.

npas

Time step number of the desired (initial) storage station from which results have to be read in. Subsequent time stations may then be reached by suitable commands (e.g. **GO** and **FREQ**) in the **PLAY ... ENDPLAY** sequence.

nsto

Storage index number of the desired (initial) storage station from which results have to be read in. Subsequent time stations may then be reached by suitable commands (e.g. **GO** and **FREQ**) in the **PLAY ... ENDPLAY** sequence.

PLAY

Introduces a sequel of “interactive” commands (see page A.25) that are read subsequently from the input file rather than from the keyboard.

ENDP

Terminates the sequel of “interactive” commands (see page A.25) that are read subsequently from the input file rather than from the keyboard.

Comments:

As indicated by the parentheses in the above syntax, the **VISU** sub-directive may be repeated as many times as needed within the **SORT** directive (see Page ED.40). However, only one **SORT** directive is allowed within each input data set.

Repetition of the **VISU** sub-directive (without repeating **SORT**) may be useful e.g. to **step back** in the ALICE file, i.e. to go to a previously saved time step. To **step forth** in the ALICE file, simply use the **GO** and **FREQ** commands in the **PLAY ... ENDPLAY** sequence, as mentioned above.

The options **T**, **NPAS** and **NST0** are mutually exclusive. Exactly one of them must be specified, in order to position the read cursor of the storage file at the initial storage position of interest. Following storage positions may then be accessed by the “interactive” commands if so desired (e.g. to produce an animation).

So-called “interactive” commands such as **TRAC** may then be issued from the keyboard. Alternatively, they may be embedded in the input file by enclosing them in the pair of keywords **PLAY ... ENDPLAY**.

The read cursor may be advanced by means of the **GO** command. In this case, however, the frequency **FREQ** counts the storage stations rather than the time steps. To terminate the execution of interactive commands (when typing them actually at the keyboard) and to return control to the input file, use the **ENDP** command.

Warnings:

Note that not all the visualization features described in page A.25 for direct execution of the code are available when visualizing results from a results file. Most restrictions come from the fact that the results file (typically an ALICE file) does not contain all the information that is available during direct execution.

For example, the following features will not work:

- Visualization of pinballs.
- Visualization of thicknesses.
- Visualization of material-related data.
- Etc. etc.

Note also that, although the **RESU** directive allows to read data from several types of results files, not all of them are suitable for visualizations. For example, an **ALIC TEMP** results file

typically contains only very limited information (just a few nodes and elements) and therefore it is suitable for the production of graphs (time curves) but not of visualizations involving the whole mesh.

17 GROUP O—INTERACTIVE COMMANDS

Object

This Section describes all the interactive commands which can be issued during a foreground (interactive) execution of the code. To launch EUROPLEXUS in interactive mode, include the `CONV` keyword at the beginning of the input file, as described in Section 6.2 (Page A.25).

When interactive execution is chosen, EUROPLEXUS reads the input data-set as usual, performs step 0 to initialise the computation, then prompts the user for commands from the keyboard with the phrase: `COMMANDE ?`.

The user can then issue various commands and subcommands **typically from the keyboard** in order to pilot the computation. For example, he can ask the program to perform a certain number of steps, then to pause again for further commands. Each time the calculation is paused, the current computational model can be visualized (e.g. by means of the built-in OpenGL-based visualization module) and information concerning the computation (time step, CPU time, etc.) can be printed. Furthermore, the current time step can be varied by the user.

As an alternative to typing commands by hand from the keyboard, such commands may be included in the regular EUROPLEXUS input file by enclosing them into a special directive `PLAY ... ENDPLAY` as described in Section 15.6 (Page I.24) and in Section 16.3.4 (Page ED.140). For example, this may be useful when the “interactive” command have to be saved for later re-execution, or when the command sequence is particularly complex, e.g. for the production of an animated visualization sequence.

17.1 COMMANDS

The following Sections list all the available “interactive” commands:

- Section 17.1.1 lists all primary interactive commands;
- Section 17.1.2 lists all `CALCUL` commands, used to pilot the current time increment;
- Section 17.1.3 lists all `TRACE` commands, used to visualize the current results;
- Section 17.1.4 lists all `MAVI` commands, used to produce an animation AVI file from a sequence of images previously produced by EUROPLEXUS itself;
- Section 17.1.5 lists all `GOTRAC` commands, used to activate a simple looping mechanism, useful e.g. in animation production;
- Section 17.1.6 lists all `CAMERA` commands, used to set the camera used for visualization;
- Section 17.1.7 lists all `SLERP` commands, used to set the motion of the camera used for visualization;

- Section 17.1.8 lists all **SCENE** commands, used to set all the details of a visualization;
- Section 17.1.9 lists all **TITLES** commands, used to insert titles in an animation.

17.1.1 Primary interactive commands

Object

To pilot a calculation interactively. Note, however, that it is also possible to store such commands within the regular EUROPLEXUS input file (rather than typing them at the keyboard) and then to execute them by means of the **PLAY** directive, described on Page I.24. In this way, the unique functionalities offered by the “interactive” commands become available also for unattended code execution, allowing e.g. to automatize the production of graphics or animated sequences.

Syntax

Here is the syntax of interactive commands and subcommands:

```

$ "?"                                     $
$ "GO"                                   $
$ "STOP"                                $
$ "INFO"                                 $
$ "FREQ" npas                            $
$ "TFRE" tfreq                           $
$ <$ HPIN ; NOHP $>                       $
$ "CALC" $ "?" $                         $
$      $ "AUTO" $                       $
$      $ "UTIL" $                       $
$      $ "DT" tstep $                   $
$      $ "R" $                          $
$ "TRAC" $ "?" $                         $ $
$      $ "NORM" $                       $ $
$      $ "ELEM" i1 i2 $                 $ $
$      $ "ZOOM" $ "?" $                 $ $
$      $      $ "POIN" xmin ymin xmax ymax $ $
$      $      $ "RETI" $                 $ $
$      $      $ "R" $                     $ $
$      $ "OEIL" xoeil yoeil zoeil $     $ $
$      $ "CULL" $                       $ $
$      $ "NOCU" $                       $ $
$      $ "NUME" $ "NOEU" $               $ $
$      $      $ "ELEM" $                 $ $
$      $ "NONU" $                       $ $
$      $ "R" $                          $ $
$      $ "CDEP" $                       $ $
$      $ "CDNO" $                       $ $

```

```

$      $ "OBJE" <$ FAIL ; NFAI $> /LECT/ <SURF ; FSIN /LECT/> $ $
$      $ "NOOB" "NOGR" "OMEM" $ $
$      $ "PINB" $ $
$      $ "PINC" $ $
$      $ "DEFO" $ $
$      $ "AMPD" ampd $ $
$      $ "VITE" $ $
$      $ "VITG" $ $
$      $ "AMPV" ampv $ $
$      $ "FEXT" $ $
$      $ "FINT" $ $
$      $ "AMPF" ampf $ $
$      $ "DASH" idsh $ $
$      $ "AVS" | "DEPL" "VITE" "FEXT" "ACCE" "MCXX" $ $
$      $          "VITG" "FINT" "CONT" "EPST" "ECRO" $ $
$      $          "ECRC" /LECT/ | $ $
$      $ "PS" $ $
$      $ "MIF" $ $
$      $ "POVR" $ $
$      $ "P10" $ $
$      $ < <"OFFS" <"SIZE" w h> $ $
$      $          <"ZIP"> <"FICH" $ "BMP" $ $ $
$      $          $ "PPM" $ $ $
$      $          $ "PPMA" $ $ $
$      $          $ "TGA" $ $ $
$      $          $ "EPS" $ $ $
$      $          $ "EPSB" $ $ $
$      $          $ "AVI" <pars> $ <'base'> > $ $
$      $ <"SYMX"> <"SYMY"> <"EXTZ nz dz> $ $
$      $ <"AXIS" na ang> <"NOSY"> $ $
$      $ <"SAVE"> <"REUS"> "REND" > $ $
$ "MAVI" <"DUMP"> <"FROM" 'base'> <"UZIP"> <"RZIP"> <"TO" 'to'> $
$      <"FIRS" firs> <"LAST" last> <"STEP" step"> <pars> "REND" $
$ "CAME" icam < "EYE" ex ey ez > $
$      < $ "Q" qr qx qy qz $ $
$      $ "VIEW" vz vy vz "RIGH" rx ry rz "UP" ux uy uz $ > $
$      < FOV fovy > $
$ "LCAM" $
$ "SLER" "CAM1" ic1 <"CAM2" ic2> <"NFRA" nfra> $
$      <INTE /PROG/> <CENT cx cy cz> $
$ "LSLE" $
$ "SCEN" <spars> $
$ "TITL" <tpars> $
$ "GOTR" <"LOOP" n> <trac_options> trac_terminator $
$ "R" $
$ "TIME" $

```

\$ "BENS"	\$
\$ "NOBE"	\$
\$ "QMS"	\$
\$ "COPY"	\$
\$ "MEAS" <measurement commands> TERM	\$
\$ "ADAP" (\$ "SPLI" iel ; "USPL" jel \$) TERM	\$

?

Lists the available primary interactive commands.

GO

Advances the computation of npas time steps.

STOP

Stops the computation.

INFO

Prints information: current time and step number of the computation, time step increment, stability step and critical time step.

FREQ

Specifies the interval (in time steps) between two successive interruptions of the calculation. Initial value is 1. The computation will halt every npas steps (counted from step 0!) and prompt for commands. This command may be combined with "TFRE", see below.

npas

Computation interval in time steps.

TFRE

Specifies the interval (in time) between two successive interruptions of the calculation. Initial value is 0.0. The computation will halt every tfreq time units (counted from the initial time!) and prompt for commands. This command may be combined with "FREQ", see above.

tfreq

Computation interval in time units.

HPIN

Halt interactive execution whenever pinball contacts are established (passing from a situation of zero contacts to one or more contacts) or completely disappear (passing from one or more contacts to zero contacts), so that the user may e.g. visualize the contacts. This switch has a toggling behaviour (see comments and sample usage below).

NOHP

Do not halt interactive execution whenever pinball contacts appear or disappear. This is the default so normally it does not need to be specified explicitly. However, the keyword is useful to restore the default behaviour after the optional keyword **HPIN** has been specified.

CALC

Allows to change the current time step increment. See options below.

TRAC

Displays (on graphics screens) or plots (on plotting devices) the current, deformed mesh shape. By default the entire mesh is visualized. See options below.

MAVI

Make an animation file (.AVI) starting from a sequence of bitmap images. At the moment, this functionality is available only starting from bitmap files of type BMP. See syntax and options below.

CAME

Defines a camera for OpenGL rendering. See below for the various options.

LCAM

List all the cameras defined so far. Note that on EUROPLEXUS versions implemented on a non-OpenGL platform, this directive is simply ignored. This enhances portability of benchmark tests on the various platforms.

SLER

Defines a spherical linear interpolation (slerp) for OpenGL rendering. See below for the various options.

LSLE

List the currently valid slerp. Note that on EUROPLEXUS versions implemented on a non-OpenGL platform, this directive is simply ignored. This enhances portability of benchmark tests on the various platforms.

SCEN

Define the current scene parameters **<spars>** to be used for OpenGL rendering. See below for the various parameters.

TITL

Define the titles (**<tpars>**) to be used for the production of a titles frame (or AVI sequence) in off-screen OpenGL rendering. See below for the various parameters.

GOTR

Performs a **GO** followed by a **TRAC**. The sequence may be automatically repeated **n** times by using the **LOOP** sub-keyword. This command is useful, among other things, for the automatic preparation of image sequences or animations. See below for a complete description.

R

Repeat the last command issued (this works also if the preceding command was a **GOTR**).

TIME

Prints the current physical and CPU time.

BENS

Activates Benson plotter graphics output instead of screen output.

NOBE

Deactivates Benson plotter output; subsequent graphics are visualized on the screen.

QMS

Pilots a QMS laser printer directly connected to a graphics terminal (Tektronix emulation) to obtain a copy of the graphics appearing on the screen. This command was formerly used at JRC and is now obsolete.

COPY

Used at JRC to redirect Tektronix graphics output from the screen to a file, defined as logical unit number 17. The typical command sequence is "**COPY TRAC NORM**", by which the current mesh plot is added to the contents of the file connected to unit 17 (initially void). This can be later visualized again (under UNIX, by the '**cat file_name**' command) and/or printed.

MEAS ... TERM

Introduces some measurement commands, see the full syntax on page G.105.

ADAP ... TERM

Introduces interactive adaptivity commands. These commands may be repeated any number of times. The end of this directive is marked by the **TERM** keyword.

SPLI iel

Split element **iel**.

USPL jel

Unsplit element **jel**.

Comments

An example of use of the HPIN and NOHP keywords is as follows. Suppose a user wants to visualize contact details in a calculation using pinballs. Normally it is difficult to exactly foresee when a contact will be established and/or it will disappear. Use the following interactive commands:

```
FREQ 1000000  
HPIN  
GO
```

In this way, the code will halt when the first pinball contact is detected and the user will have the possibility of visualizing the contact conditions. Then, type again:

```
GO
```

The execution will continue and will halt again when there are no more pinball contacts (toggling behaviour). This is useful because normally a contact remains for a number of successive time steps after it has first occurred. Then, type again:

```
GO
```

The calculation will halt when a new contact is detected, and so on.

Note that the HPIN keyword is automatically combined with the effects of FREQ, TFRE etc. The first of the specified conditions which occurs determines code halting.

To disable this behaviour and restore the normal behaviour, use the NOHP command.

Example of interactive adaptivity

To pilot adaptivity interactively, proceed as follows (this is useful mainly for debugging purposes). Assume we have a base mesh of quadrilaterals with ten elements. We want to split elements 1 and 3 at step 1, generating descendent elements 11 to 18. Then at step 2 we want to further split element 15. Finally, at step 3 we want to unsplit element 1. At each step we want to dump out the whole adaptivity data structure for debugging, and also plot the adapted mesh.

We can do this either interactively or in batch mode (via the PLAY ... ENDPLAY command).

In the second case, the input would be as follows.

```
Title of test  
CONV win  
<normal input of a test case with adaptivity>  
ECRI ... FREQ 1  
OPTI ADAP DUMP ! to dump out adaptivity data structure at printouts  
CALC ...  
PLAY
```

```
TRAC REND          ! draw base mesh at step 0
ADAP SPLI 1 SPLI 2 TERM ! split elements 1 and 2
GO                 ! compute step 1
TRAC REND          ! draw adapted mesh at step 1
ADAP SPLI 15 TERM   ! further split element 15
GO                 ! compute step 2
TRAC REND          ! draw adapted mesh at step 2
ADAP USPL 1 TERM    ! unsplit element 1
GO                 ! compute step 3
TRAC REND          ! draw adapted mesh at step 3
STOP               ! terminate calculation
ENDPLAY
```

Obviously, the same interactive commands can also be typed from the keyboard, if one removes the `PLAY ... ENDPLAY` block from the input file.

Note that the first time station at which adaptivity commands can be prescribed is step 1 (not step 0), because step 0 is always computed by the code before asking for interactive commands for the first time.

17.1.2 CALCUL options

Object

To pilot current time step.

Syntax:

```
"CALC"    $ "?"          $
           $ "AUTO"       $
           $ "UTIL"       $
           $ "DT" tstep  $
           $ "R"          $
```

?

Lists available subcommands

AUTO

Sets automatic time step calculation (see "OPTI PAS AUTO").

UTIL

Sets fixed time step (see "OPTI PAS UTIL")

DT

Set fixed time step to following value.

tstep

Time step value.

R

Return to primary commands.

17.1.3 TRACE options

Object

To set options for successive mesh visualizations.

Syntax

```

"TRAC" $ "?" $
$ "NORM" $
$ "ELEM" i1 i2 $
$ "ZOOM" $ "?" $
$ $ "POIN" xmin ymin xmax ymax $
$ $ "RETI" $
$ $ "R" $
$ "OEIL" xoeil yoeil zoeil $
$ "CULL" $
$ "NOCU" $
$ "NUME" $ "NOEU" $
$ $ "ELEM" $
$ "NONU" $
$ "R" $
$ "CDEP" $
$ "CDNO" $
$ "OBJE" <$ FAIL ; NFAI $> /LECT/ <SURF ; FSIN /LECT/> $
$ "NOOB" "NOGR" "OMEM" $
$ "PINB" $
$ "PINC" $
$ "DEFO" $
$ "AMPD" ampd $
$ "VITE" $
$ "VITG" $
$ "AMPV" ampv $
$ "FEXT" $
$ "FINT" $
$ "AMPF" ampf $
$ "DASH" idsh $
$ "AVS" | "DEPL" "VITE" "FEXT" "ACCE" "MCXX" $
$ "VITG" "FINT" "CONT" "EPST" "ECRO" $
$ "ECRC" /LECT/ | $
$ "PS" $
$ "MIF" $
$ "POVR" $

```

```

$ "P10"
$ < <"OFFS" <"SIZE" w h>
$          <"ZIP"> <"FICH" $ "BMP"      $
$          $ "PPM"      $
$          $ "PPMA"     $
$          $ "TGA"      $
$          $ "EPS"      $
$          $ "EPSB"     $
$          $ "AVI" <pars> $ <'base'> >$
$ <"SYMX"> <"SYMY"> <"EXTZ nz dz>
$ <"AXIS" na ang> <"NOSY">
$ <"SAVE"> <"REUS">          "REND" > $

```

?

Lists available subcommands.

NORM

Displays current mesh according to current options.

ELEM i1 i2

Chooses elements i1 to i2 for display. To select a non-contiguous set of elements use the OBJE directive, see below.

ZOOM

Activates zoom display mode.

OEIL x y z

Sets position of viewpoint (3D only) for parallel projection.

CULL

Activates backfacing polygon culling (3D only).

NOCU

Deactivates backface polygon culling (default).

NUME

Activates number visualization for nodes and/or elements.

NONU

Deactivates number visualization (default).

R

Returns to primary commands.

CDEP

Represents 3D degenerated shells of type CQDx with their physical thickness (the topological thickness of these elements is zero). All other element types are automatically hidden in the plot.

CDNO

Represents 3D degenerated shells of type CQDx with their topological (zero) thickness. This is the default.

OBJE

Allows to choose non-consecutive elements for display. The list of elements is given in the following `/LECT/` and may be in the form of one or more CASTEM2000 objects. This directive is alternative to the **ELEM** directive, which only allows to specify a range of consecutive elements.

FAIL

Allows to choose for display only the failed elements among those specified in the subsequent `/LECT/`. By default, all elements (both failed and non-failed) are chosen for display. Note that failed elements have a tendency to assume strange forms due to excessive deformation. This is not a problem since they are excluded from the calculation after failure. Note also that for elements of type DEBR (flying debris particle), this keyword has a special meaning: it visualizes only the *idle* debris particles, i.e. those attached to a not-yet failed element (which is not shown, since it is not failed).

NFAI

Allows to choose for display only the non-failed elements among those specified in the subsequent `/LECT/`. By default, all elements (both failed and non-failed) are chosen for display. Note that failed elements have a tendency to assume strange forms due to excessive deformation. This is not a problem since they are excluded from the calculation after failure. Note also that for elements of type DEBR (flying debris particle), this keyword has a special meaning: it visualizes only the *active* debris particles, i.e. those resulting from the fragmentation of a previously failed element (which is not shown, since it is failed).

SURF

Allows to choose only the external surface of the chosen object. This greatly reduces the amount of information to treat in the graphical module with respect to the full 3D case in large and complicated models (but of course it prevents the possibility of visualizing results in the internal parts of the model). This visualization mode makes sense only in 3D and requires the presence of continuum-like fluid elements. This option is only available for the OpenGL-based visualizer (`TRAC ... REND`) and, if specified, it must immediately follow the **OBJE** `/LECT/` directive (so it is mutually exclusive with the **FSIN** keyword described below).

FSIN

Allows to visualize only the fluid-structure interface portions of the fluid part of the chosen object. These appear as fluid element faces “sticking” onto the matching structural parts, if any are specified as well. For optimal visualization in the OpenGL renderer, it is suggested to turn on backface rendering and to apply some shrinking, e.g. by the directive (see the SCEN directive below): SCEN ... GEOM SHRI 0.98 ISOL FACE SBAC The following /LECT/ lists the concerned fluid *nodes*, i.e. the fluid nodes that lie on the fluid-structure interface: in simple cases these are just the same nodes used in the FSA and/or FSR directives. A fluid face is drawn if and only if all its nodes belong to the given /LECT/. This visualization mode makes sense only in 3D and requires the presence of continuum-like fluid elements. This option is only available for the OpenGL-based visualizer (TRAC ... REND) and, if specified, it must immediately follow the OBJE /LECT/ directive (so it is mutually exclusive with the SURF keyword described above).

NOOB

Do not make available object names (either defined by CAST3M or by I-DEAS) in the graphical rendering module. This option may be useful to speed up the rendering operations since the number of defined objects is sometimes very large. By default, object names are made available in on-screen rendering, because the user may decide interactively to use them. In off-screen rendering, they are made available only if they are needed for the visualization of the specified scene.

NOGR

Do not make available element group names (defined by the GROU directive, see page C.61) nor node group names (defined by the NGR0 directive, see page C.62) in the graphical rendering module. By default, group names are made available in on-screen rendering, because the user may decide interactively to use them. In off-screen rendering, they are made available only if they are needed for the visualization of the specified scene.

OMEM

Optimize memory during the graphical rendering, at the expense of some (or may be a lot) more CPU time. This optional keyword should only be activated in extreme cases where the size of the geometrical model is so large that the memory is not sufficient to render it (the graphics-related arrays are too big). In this way the code tries to save memory by computing some big arrays “on the fly” rather than storing them in memory. An example is the representation of iso-surfaces in very large fluid volumes. This is only useful in off-screen rendering, since in on-screen rendering any manipulation of the mesh (e.g. rotation, zoom etc.) would be extremely slow. Furthermore, if there is memory shortage, then off-screen rendering is by far preferable since only the strictly necessary tables are allocated, in contrast to on-screen (interactive) rendering. Another way of saving some memory is to specify also the NOOB and NOGR keywords described above, if objects/groups are not needed.

PINB

Draw the pinballs declared by the LIAI PINB directive. These are represented by circles. If this directive is combined with the TRAC DEFO directive, then the displacement amplification factor (AMPD) must be 1.0.

PINC

Draw the contacting (sub-)pinballs. These are represented by circles. A straight line joins the centers of each couple of contacting (sub-)pinballs. If this directive is combined with the TRAC DEFO directive, then the displacement amplification factor (AMPD) must be 1.0.

DEFO

This directive produces a plot of the initial, undeformed geometry of the model, superposed to the deformed one, which is plotted by default. The initial geometry is traced using a dashed line style (see directive DASH below) in order to distinguish it better from the current one. If this directive is combined with the TRAC PINB or TRAC PINC directive, then the amplification factor AMPD must be set at 1.0.

AMPD

Sets the amplification factor for displacements used to draw the deformed geometry. By default it is 1.0. If this directive is combined with the TRAC PINB or TRAC PINC directive, then the amplification factor must be 1.0.

VITE

This directive produces a plot of the material velocity vectors, superposed to the deformed mesh.

VITG

This directive produces a plot of the grid (mesh) velocity vectors, superposed to the deformed mesh.

AMPV

Sets the amplification factor for velocity vectors. By default it is 1.0.

FEXT

This directive produces a plot of the external force vectors, superposed to the deformed mesh.

FINT

This directive produces a plot of the internal force vectors, superposed to the deformed mesh.

AMPF

Sets the amplification factor for force vectors. By default it is 1.0.

DASH

Sets the line type for plotting the initial geometry, when "DEFO" is specified. There are 4 different styles, so idsh should be from 1 to 4. By default, idsh=3.

AVS

Produce a storage for AVS postprocessing. The variable(s) to be stored (each one on a separate file) are specified next. Before listing the variables, one may optionally specify a deformation factor (1.0 by default) via the DEFO directive and an object via the OBJE directive (by default the entire mesh is stored). Note that output results for AVS may also be produced (in “batch” modality) by means of the ECRI FICH AVS directive, see page G.70.

DEPL

Store displacements for AVS post-processing. In this case the geometry stored is the initial one and displacements are also stored as a nodal field. For all other variables, the stored geometry is the current (deformed) one.

VITE

Store particle velocity for AVS post-processing.

FEXT

Store external forces (including reactions) for AVS post-processing.

ACCE

Store particle acceleration for AVS post-processing.

MCXX

Store multicomponent fluid variables for AVS post-processing.

VITG

Store grid velocity for AVS post-processing.

FINT

Store internal forces for AVS post-processing.

CONT

Store stresses for AVS post-processing.

EPST

Store total strains (still to be implemented for AVS post-processing).

ECRO

Store hardening quantities for AVS post-processing. The relevant components are chosen by ECRC.

ECRC

Select ECR components to be chosen (/LECT/).

PS

Produce output on PostScript file instead of screen.

MIF

Produce output on MIF (FrameMaker) file instead of screen.

POVR

Produce output in the form of a POV-Ray (Persistence of Vision) file instead of screen. Only the geometry is stored.

P10

Produce PLOT-10 (Tektronix) output on file instead of screen (available also on MS-Windows).

<OFFS ...> REND

Start OpenGL rendering (currently available only under MS-Windows or Linux). The rendering process may take place either on-screen (the default) or off-screen (in a file), as specified by the **OFFS** sub-directive (see full description below) which, if present, must precede the **REND** keyword. Note that on EUROPLEXUS versions implemented on a non-OpenGL platform, the **TRAC ... REND** directive is simply (and entirely) ignored. This enhances portability of benchmark tests on the various platforms.

SYMX

Perform a symmetry with respect to the X-axis before rendering. (This option is still under development).

SYMY

Perform a symmetry with respect to the Y-axis before rendering. (This option is still under development).

EXTZ nz dz

Perform an extrusion with respect to the Z-plane before rendering. The extrusion amount is **dz**, subdivided into **nz** increments. (This option is still under development).

AXIS na ang

Perform an axial symmetry around the Y-axis before rendering. The total angle of symmetry is **ang**, subdivided into **na** increments. (This option is still under development).

NOSY

Disable any symmetries previously defined by the **SYMX**, **SYMY**, **EXTZ** and **AXIS** directives.

SAVE

Save for the next rendering action(s) all geometrical quantities computed in the rendering process. Since the computation of these quantities is very CPU time consuming, this option may allow to considerably shorten the time required to produce animations composed by long sequences of frames in which the geometrical quantities stay constant and only the iso field or vector field change from frame to frame. The default is not to save the computed quantities. A typical use of **SAVE/REUS** would be in an Eulerian calculation, to show the time evolution of pressure field and velocity vectors in the fluid domain. The chosen domain (fluid) is always the same in all frames (although possibly the viewpoint may change) and the nodes do not move since they are Eulerian. The same optimization may be obtained also in an ALE calculation, if one visualizes only a completely Eulerian sub-domain: typical is the case of a fluid-structure interaction calculation with the **FLSR** model, where the fluid domain is typically completely Eulerian. For safety, the code verifies that all nodes to be visualized be Eulerian. If this rule is not respected, the geometrical quantities are re-computed and so no (or little) optimization takes place.

REUS

Re-use the geometrical quantities computed and saved in a preceding rendering process (by the **SAVE** option described above) rather than spend CPU time to recompute them. The default is to re-compute these quantities anew each time. Note that **REUS** is not compatible with symmetry commands, i.e. with the **SYM**, **SYMY**, **EXTZ** and **AXIS** directives.

Comments

The **CULL** option performs a very basic hidden surface removal. All element faces (polygons) whose outward normal points away from the observer are eliminated from the plot. This results in hidden surface removal for very simple, basic shapes, but is imperfect for complex, arbitrary geometries.

At each required **AVS** storage (see **AVS** above), one file is produced for each variable, with the name **avs.<VARI>.N.inp**, where **<VARI>** stands for the variable (i.e., **DEPL**, **VITE** etc.), and **N** is an integer counter which is automatically incremented by one each time storage is requested. Such files may be postprocessed interactively by **AVS** while **EUROPLEXUS** is running.

At each required **POVRAY** storage (see **POVR** above), one file is produced containing the current geometry of the model. The file names are **povray.GEOM.N.pov** where **N** is an integer counter starting at 0 and incremented by one each time storage is requested. Such files may be postprocessed interactively by **POVRAY** while **EUROPLEXUS** is running.

Symmetries

Note that these options are still under development.

The **SYMX**, **SYMY**, **EXTZ** and **AXIS** directives are only available in conjunction with OpenGL-based rendering (**REND**).

They may be combined, but the following combinations are invalid:

- **EXTZ** and **AXIS** are mutually exclusive.
- **SYMY** and **AXIS** are mutually exclusive.

Furthermore, the following restrictions apply:

- All nodes of the original mesh must have $z = 0$.
- **SYMX** requires that all nodes of the original mesh have $y \geq 0$.
- **SYMY** requires that all nodes of the original mesh have $x \geq 0$.
- **AXIS** requires that all nodes of the original mesh have $x \geq 0$.

Off-screen rendering

The **REND** directive admits an optional **OFFS** sub-directive that allows to produce OpenGL rendered images off-screen and to prepare animations of the results. By default, when the **OFFS** keyword is specified alone, each frame of the scene is recorded in a bitmap file, having the name **base_nnnn.<ext>**, where **base** is the base name of the run, **nnnn** is a four-digit integer counter that starts at 0001 and is incremented automatically by the program, and **<ext>** is the file extension which depends upon the chosen file type.

Such files are uncompressed by default and may therefore require a large disk space. To save space, the optional **ZIP** keyword may be specified, which automatically compresses the bitmap file after its generation.

The following bitmap file types are currently supported:

- **BMP** indicates a MS Windows bitmap (binary) file, with the extension **.bmp**.
- **PPM** indicates a Portable Pixmap (binary) file, with the extension **.ppm**.
- **PPMA** indicates a Portable Pixmap (ASCII) file, with the extension **.ppm**.
- **TGA** indicates a Targa (binary) file, with the extension **.tga**.
- **EPS** indicates an Encapsulated PostScript file in color, with the extension **.eps**.
- **EPSB** indicates an Encapsulated PostScript file in black and white, with the extension **.eps**.

By specifying the optional keyword **FICH**, users may change the base name mentioned above: for example the sequence **TRAC OFFS FICH BMP 'toto' REND** would produce frame files **toto_0001.bmp**, **toto_0002.bmp**, etc.

In addition (or in alternative), the user may request the production of an AVI animated sequence from the single frames. For example, the directive **TRAC OFFS FICH AVI 'anim' REND** would produce an animation file **anim_01.avi** but no BMP frame files. To produce both the BMP frames and the AVI file, specify both options. The default name of the animation file (i.e. if **'base'** is omitted in the above syntax) is **base_nn.avi** where **base** is the base name of the run and **nn** is a two-digit integer counter that starts at 01 and is incremented automatically by the program.

By default, the size of off-screen generated (**OFFS**) bitmap and AVI files is of 500 pixels (width) by 500 pixels (height). To produce a different size, use the **SIZE w h** sub-directive where **w** is the width in pixels and **h** is the height in pixels. Both these quantities should be multiples of 4.

For on-screen generated images, the *initial* size as the window is popped up is of 500 pixels (width) by 500 pixels (height). This may then be changed interactively by the user.

The production of AVI files may be piloted by a sequence of parameters **<pars>** that are described below.

ZOOM suboptions

Syntax:

```
"ZOOM" $ "?" $
        $ "POIN" xmin ymin xmax ymax $
        $ "RETI" $
        $ "R" $
```

?

Lists available subsubcommands.

POIN

Sets zoom window using coordinates of lower left and upper right corner.

xmin ymin

Coordinates of lower left window corner.

xmax ymax

Coordinates of upper right window corner.

RETI

Activates crosshair cursor for the definition of zoom window. Position the cursor with direction keys on lower left corner, then type <CR>, repeat for upper right corner.

R

Return to primary commands.

NUME suboptions

Syntax:

```
"NUME" $ "NOEU" $  
        $ "ELEM" $
```

NOEU

Activates node number display.

ELEM

Activates element number display.

Direct AVI file generation

Object

To generate directly an animated AVI file without using an external utility program that generates the AVI starting from a sequence of still frames (bitmaps).

A serious drawback for the use of this command, in particular during the (direct) calculation of a transient solution, is that the total number of frames in the animation (see **NFTO** below) must be set exactly. If for some reason the application stops before having written all the frames and having closed properly the AVI file, this file is unusable. The problem may be circumvented by producing bitmaps (frames) during the direct calculation, and then by making the AVI file separately starting from the frames sequence, as described in the **MAVI** directive below. In fact, in that case the program is able to determine the total number of frames automatically, if needed, because all the frames are available when the animation production is started. At the moment, the **MAVI** command is only available for bitmap images of type BMP.

Note that this functionality is based upon the Microsoft Video for Windows library and therefore it is currently available only on MS-Windows based platforms. If the following commands are issued on a different platform, they are simply ignored by EUROPLEXUS.

Syntax:

```
TRAC    <OFFS <FICH AVI <pars> $ <'base'> > > REND
```

where <pars> represents the following syntax:

```
<CONT> <NOCL>  
<NFTO nfto>  
<FPS fps> <COMP comp> <KFRE kfre> <CQUA cqua>
```

CONT

The AVI scene is a continuation of the AVI file created with a previous `TRAC OFFS FICH AVI` command *during the same EUROPLEXUS run*. This optional keyword allows to build up a complex animation as a series of simple sequences (scenes), each one produced by a separate command. By default (i.e. in the absence of the `CONT` keyword) a new AVI file is started. Note that, if present, the keywords `CONT` and/or `NOCL` must immediately follow the keyword `AVI` and come before the other optional keywords.

NOCL

Do not close the AVI file after writing the current scene. This allows to add further scenes by subsequent commands. By default, i.e. in the absence of the `NOCL` keyword, the AVI file is closed after writing the current scene. Note that, if present, the keywords `CONT` and/or `NOCL` must immediately follow the keyword `AVI` and come before the other optional keywords.

nfto

The total number of frames forming the AVI file animated sequence. Note that, unlike the following ones, this parameter is mandatory but only when the scene being defined is the first one of a multi-scene AVI file, i.e. when the `CONT` keyword is not present but the `NOCL` keyword is specified. When both `CONT` and `NOCL` are omitted, then the AVI file contains just one scene (the current one) and the (total) number of frames needs not be specified, since it may be obtained as the value given for the currently valid slerp (see `SLER` below).

nfto

The total number of frames forming the AVI file animated sequence. Note that, unlike the following ones, this parameter is mandatory.

fps

Number of frames per second for the visualization of the AVI file. If omitted, the default value of 5 frames per second is used.

comp

Compression type of the produced AVI file. The value `-1` indicates that the Microsoft Video 1 codec has to be used. This codec is somewhat obsolete for realistic films, but perfectly adequate for the type of technical graphics produced by EUROPLEXUS, and has the advantage of being present on virtually any MS-Windows based computer. The value `1` produces a popup dialog box that allows the user to interactively choose (and somewhat configure) the desired codec from the list of those available on his platform. Obviously, this is adequate only for interactive execution of EUROPLEXUS (while the previous value `-1` is the normal choice for unattended AVI file creation, i.e. batch execution). If omitted, the default value of `0` (no compression) is used. Note, however, that without compression the produced AVI file size grows very rapidly since it is simply the sum of the size of its (uncompressed) frames. An advantage of this choice is that the AVI file may be compressed *a posteriori* by means of an external utility (e.g. Virtualdub).

kfre

Key frame frequency. This parameter is only used when the Microsoft Video 1 codec is chosen (see `comp` above). The default value is `0`, meaning that every frame is a key frame. This somewhat increases the file size but it simplifies navigation through it during playback.

cqua

Compression quality in %. This parameter is only used when the Microsoft Video 1 codec is chosen (see `comp` above). The default value is `100`, meaning full (loss-less) quality.

17.1.4 AVI file generation from a sequence of bitmaps (MAVI)

Object

To generate an animated AVI file starting from a sequence of still frames (bitmaps). At the moment, the only type of bitmap images that are recognized by the MAVI command are BMP images.

Note that this functionality is based upon the Microsoft Video for Windows library and therefore it is currently available only on MS-Windows based platforms. If the following commands are issued on a different platform, they are simply ignored by EUROPLEXUS.

Syntax:

```
MAVI <DUMP> <FROM 'base'> <UZIP> <RZIP> <TO 'to'>  
      <FIRS first> <LAST last> <STEP step> <pars> REND
```

DUMP

Dump out verbose information about the bitmaps that are being read in during the AVI file generation (only for debugging).

'base'

Base name of the sequence of bitmap images, in quotes. If omitted, the base name of the test case is used. For example, by specifying FROM 'toto' the program looks for files of the form toto_0001.bmp, toto_0002.bmp etc. in the current directory, if uncompressed bitmaps (as by default) are used. If compressed bitmaps are used (see the next keyword UZIP), then the expected file names are toto_0001.bmp.gz, toto_0002.bmp.gz etc. in the current directory.

UZIP

Unzip (decompress) the bitmap files before using them to produce the AVI file. The bitmap files have either been compressed by hand, or they have been produced by the TRAC OFFS ZIP ... directive as explained above. By default, uncompressed bitmaps are expected.

RZIP

Re-zip (re-compress) the bitmap files after using them to produce the AVI file. This saves a lot of disk space. By default, the bitmap files are left uncompressed after use.

'to'

Base name of the AVI file to be produced, in quotes. If omitted, the base name of the test case is used. For example, by specifying T0 'tata' the program generates AVI file(s) of the form tata_01.avi, tata_02.avi etc. in the current directory.

FIRS

Index of the first bitmap file (frame) to be used for the animation. By default, the first found file in alphabetical order is used.

LAST

Index of the last bitmap file (frame) to be used for the animation. By default, the last found file in alphabetical order is used, so the total number of frames in the animation is in this case determined automatically by the program.

STEP

Increment in the index of the bitmap file (frame) to be used for the animation. By default, all files are used, in alphabetical order.

REND

This keyword terminates the MAVI sequence and triggers its execution. If omitted, no animation file is produced!

In the above MAVI directive, the sequence <pars> represents the following syntax (similar to the one already described above for the direct AVI file creation):

<FPS fps> <COMP comp> <KFRE kfre> <CQUA cqua>

fps

Number of frames per second for the visualization of the AVI file. If omitted, the default value of 5 frames per second is used.

comp

Compression type of the produced AVI file. The value -1 indicates that the Microsoft Video 1 codec has to be used. This codec is somewhat obsolete for realistic films, but perfectly adequate for the type of technical graphics produced by EUROPLEXUS, and has the advantage of being present on virtually any MS-Windows based computer. The value 1 produces a popup dialog box that allows the user to interactively choose (and somewhat configure) the desired codec from the list of those available on his platform. Obviously, this is adequate only for interactive execution of EUROPLEXUS (while the previous value -1 is the normal choice for unattended AVI file creation, i.e. batch execution). If omitted, the default value of 0 (no compression) is used. Note, however, that without compression the produced AVI file size grows very rapidly since it is simply the sum of the size of its (uncompressed) frames. An advantage of this choice is that the AVI file may be compressed *a posteriori* by means of an external utility (e.g. Virtualdub).

kfre

Key frame frequency. This parameter is only used when the Microsoft Video 1 codec is chosen (see `comp` above). The default value is 0, meaning that every frame is a key frame. This somewhat increases the file size but it simplifies navigation through it during playback.

cqua

Compression quality in %. This parameter is only used when the Microsoft Video 1 codec is chosen (see `comp` above). The default value is 100, meaning full (loss-less) quality.

17.1.5 GOTRAC: a simple looping mechanism

Object

To perform a **GO** in order to advance the solution to the next desired time step or time value, directly followed by a **TRAC** operation to display the results. This sequence may be automatically repeated a given number of times, if so desired.

Syntax:

```
"GOTR" <"LOOP" n> <trac_options> trac_terminator
```

n

An integer used to specify the number of times the **GOTRAC** sequence has to be repeated. By default, the sequence is executed just once.

trac_options

Any valid sequence of sub-commands of the **TRAC** command, see above.

trac_terminator

A valid terminator of the **TRAC** command, which actually produces the drawing or visualization. The possible values are **NORM** for vector-graphics based (on-screen or on file) drawing or **REND** for OpenGL-based rendering. See the above description of the **TRAC** command for further details.

17.1.6 CAMERA parameters and options

Object

To define a camera for OpenGL rendering. Repeat this command any number of times to define as many cameras as needed (with different identifiers `icam`, see below). The orientation of the camera in space may be defined in two alternative ways: either via a quaternion, or via a triplet of versors defining a right-handed reference frame.

Note that on EUROPLEXUS versions implemented on a non-OpenGL platform, this directive is simply (and entirely) ignored. This enhances portability of benchmark tests on the various platforms.

Syntax:

```
CAME icam < EYE ex ey ez >
          < $ Q qr qx qy qz          $
          $ VIEW vz vy vz RIGH rx ry rz UP ux uy uz    $ >
          < FOV fovy >
```

`icam`

An integer used to identify the camera later on. It must be a positive number and it is mandatory (no default value is provided). Typically, use 1, 2, 3, etc. Best efficiency is obtained by starting the definition of cameras with the highest index. By repeating the definition of an existing camera (same index), the old one is replaced by the new one and is no longer available.

EYE

Position of the camera in space, i.e. position of the observer's eye. If omitted, the program assumes the position (0,0,1).

Q

Quaternion defining the orientation of the camera in space. Here `qr` is the real part while `qx`, `qy`, `qz` are the components of the imaginary part. Its norm must be unitary, so that the quaternion represents a rigid-body rotation in space, with respect to a default orientation. This default orientation is assumed such that the x -axis points to the right of the picture, the y -axis points upwards and the negative z -axis points "inside". The default value of this parameter is the identity quaternion (1,0,0,0).

VIEW RIGH UP

Triplet of unit versors that may be used, in alternative to the quaternion form described above, to define the orientation in space of the camera. The **VIEW** vector points from the camera position (**EYE**) to the observed object. The **RIGHT** vector defines the right-hand orientation (horizontally in the picture) and the **UP** vector defines the upright direction (vertically in the picture). These vectors must be unitary in length and be mutually orthogonal so as to define a left-handed reference frame. The vector product of **RIGHT** times **VIEW** must equal **UP** (and cyclic permutations thereof). The default values are: (0,0,-1) for **VIEW**, (1,0,0) for **RIGHT** and (0,1,0) for **UP**.

FOV

Angle representing the field of view of the camera, in degrees. Smaller angles produce a zoom-in effect while larger ones produce a zoom-out effect. The default value is 60 degrees.

17.1.7 SLERP parameters and options

Object

To define a slerp (spherical linear interpolation) of camera positions for OpenGL rendering.

Note that on EUROPLEXUS versions implemented on a non-OpenGL platform, this directive is simply (and entirely) ignored. This enhances portability of benchmark tests on the various platforms.

Syntax:

```
SLER CAM1 ic1 <CAM2 ic2> <NFRA nfra>  
      <INTE /PROG/> <CENT cx cy cz>
```

CAM1

Identifier of the first (initial) camera for the slerp. This camera must of course have been previously defined by the **CAME** directive. This value is mandatory, and thus no default value is provided.

CAM2

Identifier of the second (final) camera for the slerp. This camera must of course have been previously defined by the **CAME** directive. This value may be omitted, and in that case 0 is assumed. This means that the scene is still: the first camera defined above is used for the whole sequence.

NFRA

Number of frames of the slerp sequence. If omitted, 1 is assumed: the scene consists of a single frame, produced by **CAM1**. If greater than 1, then there are two cases: if **CAM2** is not defined, then all frames are produced with the first camera (**CAM1**), i.e. the sequence is still. If **CAM2** is defined, then the camera is interpolated between **CAM1** and **CAM2**. Note, however, that in the case of linear interpolation (missing **INTE**, see below) the first interpolated value is **not** **CAM1** but the first non-zero value going from **CAM1** to **CAM2**. The last interpolated camera, however, coincides with **CAM2**. This convention allows to chain successive sequences one after the other without obtaining double (repeated) frames at the intermediate camera values.

INTE

Interpolation values for the calculation of parameters for the intermediate frames. If omitted, linear equidistant values are used. Linear interpolation is applied to the camera **FOV** while slerp interpolation is used for the camera orientation (**Q** or **VIEW**, **RIGHT**, **UP**). The camera **EYE** is interpolated as described below (see **CENT**).

CENT

Centre of rotation for the interpolation of the camera eye positions. If omitted, or if its position coincides with the eye position for the first camera (**CAM1**), the eye position is interpolated linearly between the initial and final (if relevant) specified positions. Thus, the observer moves along a straight line (while at the same time possibly rotating around the eye). When present and different from the eye position for the first camera (**CAM1**), it represents the centre of a circle along which the camera eye moves. The circle passes through the initial and final camera eye positions. Therefore, the given point must be equidistant from these two points (but of course not aligned between them, so that the three points define a unique plane).

17.1.8 SCENE options

Object

To define a set of parameters (globally indicated above as **<spars>**) for the definition of the characteristics of the current scene, to be used during OpenGL rendering.

These parameters parallel as closely as possible the menu items that are available for interactive OpenGL visualization. For more details on the parameters and options, see the reference manual of the interactive OpenGL renderer.

Once defined by a **SCEN** directive, a set of scene parameters (the **current scene**) remains active for any following **TRAC** directive(s), until a new set of parameters is defined by a new **SCEN** directive.

If no **SCEN** directive is given, some reasonable default values are assumed.

In order to restore the default scene values during a calculation *after* a scene has been defined, use an empty **SCEN** directive, i.e. **SCEN** followed by no other sub-keywords or parameters.

Note that on EUROPLEXUS versions implemented on a non-OpenGL platform, this directive is simply (and entirely) ignored. This enhances portability of benchmark tests on the various platforms.

Syntax:

```
SCEN
  <OBJE ( <SELM /LECT/> <USLM /LECT/>
          <SELG /LECT/> <USLG /LECT/>
          <SELP /LECT/> <USLP /LECT/> )
          <SELV | FLSR ; FLSW ; HANG ; BHAN | >
          <DHAS $ OUTL ; CGLA ; BGLA ; GGLA ; GLAS ; FADE ffac $> >
  <GEOM <NAVI FREE>
    <NPTO npto>
    <PROJ ORTH>
    <REFE <FRAM> <BBOX> <CENT> >
    <FACE <HFRO> <SBAC> <SINT> <HBIS> <SHOW /LECT/> >
    <LINE <HEOU> <SSHA> <SFRE> <SPER> <SISO> <ANTI> <SBOU> <SIU> >
    <POIN $ DOT dsiz ; SPHE ssiz ; SPHP <FACT fact> $>
    <SHRI sh <GROU> <NOUT> <ISOL> <HFAC> <PINS> >
    <PINB <PARE> <CDES> <CPOI> <NORM> <JOIN> <NASN> <PASN> <DASN> >
    <INIT $ ASIS ; CGLA ; WIRE ; OUTL $ >
    <DEBR $ TRAJ ; TRCO $ >
```

```

    <FLSR <DOMA> <SPHE> <CONE> <PRIS> <HEXA> <NORM> <COUP> <BLOQ> >
    <FLSW <DOMA> <SPHE> <CONE> <PRIS> <HEXA> <NORM> <COUP> <BLOQ> >
<VECT $ SCAV ; COLO ; SCCO $
    <vec_field> <SUPP /LECT/>
    <SCAL $ A6 ; A14 ; USER /PROG/ $>
    <$ LENG fac ; SFAC sfac $ >
    <COSC $ COLS ; GRAY ; ICOL ; IGRA $ >
    <SIVE> >
<ISO $ LINE ; FILL ; FILI ; FELE ; SMOO ; SMLI ; SMEL ; SURF ; SULI $
    <SHIN> <FADE ffac>
    <iso_field> <SUPP /LECT/> <GAUS igauss | GAUZ igauss>
    <SCAL $ A1 ; A6 ; A14 ; USER /PROG/ $>
    <COSC $ COLS ; GRAY ; ICOL ; IGRA $ > >
<TEXT <NODE> <ELEM> <OBJE> <VSCA> <ISCA> <HINF> <CAME> <DEBU> <PCON> >
<COLO ( SELE $ RED ; GREE ; BLUE ; CYAN ; MAGE ;
        YELL ; BLAC ; WHIT ; GR05 ; GR10 ;
        GR15 ; GR20 ; GR25 ; GR30 ; GR35 ;
        GR40 ; GR45 ; GR50 ; GR55 ; GR60 ;
        GR65 ; GR70 ; GR75 ; GR80 ; GR85 ;
        GR90 ; GR95 $
        APPL $ BGRN ; CENT ; BBOX ; IFAC ; ELOU ;
        SHAR ; FRED ; PERP ; VECT ; ISOE ;
        ISOL ; POIN ; NNUM ; ENUM ; ONAM ;
        TEXT ; INWI ; INOU ; TRAJ ; ISOD $ )
    <PAPE> >
<LIMA <ON>
    <LIGX ligx> <LIGY ligy> <LIGZ ligz>
    <LAMB $ LOW ; MEDI ; HIGH >
    <LDIF $ LOW ; MEDI ; HIGH >
    <LSPE $ LOW ; MEDI ; HIGH >
    <LSHI $ LOW ; MEDI ; HIGH >
    <LMAM $ LOW ; MEDI ; HIGH >
    ( SELE $ BRAS ; BRON ; PBRO ; CHRO ; COPP ;
        PCOP ; GOLD ; GOL2 ; PGOL ; PEWT ;
        SILV ; PSIL ; EMER ; JADE ; OBSI ;
        PEAR ; RUBY ; TURQ ; BLAP ; CYAP ;
        GREP ; REDP ; WHIP ; YELP ; BLAR ;
        BLR2 ; CYAR ; GRER ; REDR ; WHIR ;
        YELR $
        APPL $ MESH ; SELO ; /LECT/ $ ) >

```

Objects Menu Parameters

OBJE

Introduces the parameters relative to the **Objects** menu.

SELM

Choice of the elements (objects of mesh type) to be visualized.

USLM

Choice of the elements (objects of mesh type) to be hidden.

SELG

Choice of the groups (of elements) to be visualized.

USLG

Choice of the groups (of elements) to be hidden.

SELP

Choice of the points (objects of points type) to be visualized.

USLP

Choice of the points (objects of points type) to be hidden.

SELV

Select some “variable” objects, i.e. objects (of elements or of points type) whose composition varies in time rather than being topologically constant. At the moment, only the keywords **FLSR**, **FLSW**, **HANG** and **BHAN** are available. This directive is useful only in the generation of graphics in ‘batch’ mode. In fact, when using the graphics interactively one may access the same objects from the **SELG** or **SELP** menus. For example, the fluid nodes currently subjected to **FLSR** coupling conditions are available interactively in a special node group named “**FLSR**”.

FLSR

Select for visualization the fluid nodes currently subjected to **FLSR** coupling conditions. These nodes are drawn according to the selected drawing mode for points (see **GEOM POIN** directive).

FLSW

Select for visualization the fluid elements currently subjected to **FLSW** coupling conditions.

HANG

Select for visualization the currently hanging nodes in adaptivity. These nodes are drawn according to the selected drawing mode for points (see **GEOM POIN** directive).

BHAN

Select for visualization the currently boundary-hanging nodes in adaptivity. These nodes are drawn according to the selected drawing mode for points (see **GEOM POIN** directive).

DHAS

Allows to choose the way to draw the hidden (non-visualized) portions of the mesh. If omitted, they are ‘drawn’ as hidden (i.e., not drawn at all).

OUTL

Draw hidden mesh portions as element outlines (wireframe representation).

CGLA

Draw hidden mesh portions as colored glass.

BGLA

Draw hidden mesh portions as blue glass.

GGLA

Draw hidden mesh portions as green glass.

GLAS

Draw hidden mesh portions as colored glass (variation of **CGLA** that produces better results in some circumstances).

FADE

Draw hidden mesh portions as fading out objects.

ffac

Fading out factor (between 1.0 i.e. fully visible and 0.0 i.e. fully hidden).

Geometry Menu Parameters**GEOM**

Introduces the parameters relative to the **Geometry** menu.

NAVI

Introduces the parameters relative to the *Navigation* sub-menu.

FREE

Choose the free camera navigation mode. By default, the rotating camera navigation mode is used.

NPTO

Introduces the parameters relative to the *Near Plane Tolerance* sub-menu.

npto

Near plane tolerance. By default, a value of 1.E-4 is used. During navigation inside bodies, it may be useful to increase this value (e.g. to 1.E-2).

PROJ

Introduces the parameters relative to the *Projection* sub-menu.

ORTH

Choose the orthogonal projection. By default, the perspective projection is used.

REFE

Introduces the parameters relative to the *References* sub-menu.

FRAM

Show the global reference frame.

BBOX

Show the bounding box.

CENT

Show the centre.

FACE

Introduces the parameters relative to the *Faces* sub-menu.

HFRO

Hide the front faces.

SBAC

Show the back faces.

SINT

Show the internal faces.

HBIS

Hide the back iso surfaces.

SHOW /LECT/

Force visualization of front and back faces belonging to the elements specified in the following `/LECT/`. This may be useful e. g. in a calculation with both a fluid (volumetric mesh) and a structure when the user wants to display both the structure and some iso-surfaces in the fluid. When `ISO SURF` or `ISO SULI` is selected, the code automatically disables the view of front faces (as a global setting), therefore the structure would not be drawn. By specifying `SHOW LECT stru TERM` the structural faces (both front and back faces) are forced to be drawn, thus obtaining the desired effect.

LINE

Introduces the parameters relative to the *Lines* sub-menu.

HEOU

Hide element outlines.

SSHA

Show sharp corners.

SFRE

Show free edges.

SPER

Show perpendicular contours.

SISO

Show iso surface outlines.

ANTI

Antialias lines.

SBOU

Show backface outlines (even when backfaces are not shown).

SIOU

Show internal face outlines (even when internal faces are not shown). This is a way to show the internal part of the mesh in wireframe representation.

POIN

Introduces the parameters relative to the *Points* sub-menu.

DOT

Render points as dots of size `psiz`. By default, points are rendered as dots of size 2.

SPHE

Render points as spheres of size **ssiz**.

SPHP

Render points as spheres of “physical” size (it must be possible to determine this size from some physical parameter associated with the point, e.g. the radius of a material particle).

FACT

Optional factor by which the physical radius of each sphere is multiplied for visualization purposes. By default it is 1.0.

SHRI

Introduces the parameters relative to the *Shrinkage* sub-menu.

sh

Shrink by a factor **sh**. This parameter is mandatory and must immediately follow the **SHRI** keyword.

GROU

Shrinkage will occur by element groups (which must have been defined), rather than element-by-element. If a group-related centerpoint has been specified in the **GROU** directive (see page C.61), then it is used for the shrinkage operation, otherwise shrinkage occurs around the average (unweighted) of the center points of the elements contained in the group. If a group-related shrink factor has been specified (see page C.61), it overrides the scene’s generic **sh** factor. Finally, if a group-related shift has been specified (see page C.61), it is applied as well during rendering.

NOUT

Do not shrink outlines (they are shrunken by default if some shrinkage is activated).

ISOL

Shrink isolines.

HFAC

Shrink hidden faces.

PINS

Shrink pinballs.

PINB

Introduces the parameters relative to the *Pinballs* sub-menu.

PARE

	Show parent pinballs.
CDES	
	Show contacting descendents.
CPOI	
	Show contact points.
NORM	
	Show contact normals.
JOIN	
	Show contact joints.
NASN	
	Show nodal ASNs (assembled surface normals).
PASN	
	Show pinball ASNs (assembled surface normals) for the parent pinballs.
DASN	
	Show pinball ASNs (assembled surface normals) for the contacting descendent pinballs.
INIT	
	Render the initial geometry of the model besides the current one.
ASIS	
	Render the initial geometry in the same way as the current one.
CGLA	
	Render the initial geometry as colored glass.
WIRE	
	Render the initial geometry as wireframe.
OUTL	
	Render the initial geometry as outline.
DEBR	
	Render the flying debris besides the current geometry.
TRAJ	

Render the flying debris trajectories.

TRCO

Render the flying debris trajectories in shades of color. The color is related to the local debris velocity.

FLSR

Introduce rendering of quantities related to FLSR domains besides the current geometry.

DOMA

Render *all* the FLSR domains themselves (i.e. spheres, cones, prisms and tetrahedra).

SPHE

Render the spherical FLSR domains.

CONE

Render the conical FLSR domains.

PRIS

Render the prisms FLSR domains.

HEXA

Render the hexahedra FLSR domains.

NORM

Render the FLSR domain normal(s). First normals are rendered in blue, second normals (if any) in green and third normals (if any) in red.

COUP

Render the FLSR couplings.

BLOQ

Render the FLSR blocked MC fluxes.

FLSW

Introduce rendering of quantities related to FLSW domains besides the current geometry.

DOMA

Render *all* the FLSW domains themselves (i.e. spheres, cones, prisms and tetrahedra).

SPHE

Render the spherical FLSW domains.

CONE

Render the conical FLSW domains.

PRIS

Render the prisms FLSW domains.

HEXA

Render the hexahedra FLSW domains.

NORM

Render the FLSW domain normal(s). First normals are rendered in blue, second normals (if any) in green and third normals (if any) in red.

COUP

Render the FLSW couplings.

BLOQ

Render the FLSW blocked MC fluxes.

Vectors Menu Parameters**VECT**

Introduces the parameters relative to the **Vectors** menu.

SCAV

Show scaled vectors.

COLO

Show colored vectors.

SCCO

Show scaled colored vectors.

SUPP

Define, by means of the following `/LECT/`, the list of the nodes that form the geometric support of the vector field. By default, the support is the entire mesh.

`<vec_field>`

Choose the vector field to be represented (see below).

SCAL

Introduces the parameters relative to the *Scale* sub-menu.

A6

Use an automatic scale with 6 values. This is the default.

A14

Use an automatic scale with 14 values. By default, an automatic scale with 6 values is used.

USER

Use the fixed, user-specified scale given by */PROG/*.

LENG

Introduces the parameters relative to the *Length* sub-menu.

fac

Scaling factor with respect to the default vector length, which is 10% of the geometric model size.

SFAC

May be use in alternative to the **LENG** directive to set the absolute length or the maximum length of the drawn vectors. This is useful e.g. in animations when the size of the geometric model may vary considerably during a transient calculation.

sfac

The meaning of this value depends upon the chosen vector type representation. For scaled vectors (colored or not) **sfac** is the scale factor by which the physical vector norm is multiplied to obtain the length of the drawn vector. Thus, the length of a drawn vector may be associated with a physical value of the represented quantity, independently from the geometric model size and its variations, and on the chosen scale. For colored vectors of uniform length (non-scaled) **sfac** represents the length of the drawn vectors.

COSC

Introduces the color scheme to be used for the visualization of vectors.

COLS

Use colors (this is the default).

GRAY

Use a scale of grays.

ICOL

Use colors but invert the colors set (blue indicates the maximum value insted of minimum value).

IGRA

Use a scale of grays but invert the colors set (dark gray indicates the maximum value insted of minimum value).

SIVE

Show internal vectors in 3D models. By default, vectors are only traced on the visible faces, i.e. typically just on the envelope of 3D models.

Iso Menu Parameters**ISO**

Introduces the parameters relative to the **Iso** menu.

LINE

Show iso lines.

FILL

Show iso fill.

FILI

Show iso fill lines.

FELE

Show iso fill elements.

SMOO

Show iso smooth.

SMLI

Show iso smooth lines.

SMEL

Show iso smooth elements.

SURF

Show iso surfaces.

SULI

Show iso surfaces lines.

SHIN

Render iso surfaces as shiny surfaces. By default, iso surfaces are rendered as dull surfaces.

FADE

Draw iso surfaces as fading out objects (this is only applicable to **SURF** or **SULI**. Useful to see “through” the iso surfaces, e.g. in case of pressure waves in a blast.

ffac

Fading out factor (between 1.0 i.e. fully visible and 0.0 i.e. fully hidden).

SUPP

Define, by means of the following **/LECT/**, the list of the elements that form the geometric support of the iso field. By default, the support is the entire mesh.

GAUS

Allows to choose a specific Gauss point index (only for the quantities **CONT**, **EPST** and **ECRO**).

igauss

Number of the Gauss point chosen. The special value 0 means that the average over all Gauss points in the element is taken. This is the default, i.e. if neither **GAUS** nor **GAUZ** is specified. Note that this default is different from the default in curve plotting (Page ED.80) where 1 is assumed, i.e. the first Gauss Point is plotted.

GAUZ

Allows to choose a specific “lamina” of the (shell) element. The value is the index of the lamina through the thickness (only for the quantities **CONT**, **EPST** and **ECRO**). In this case, the code takes the average value of all Gauss Points belonging to the specified lamina.

igauz

Number of Gauss point through the thickness (i.e. index of the chosen lamina).

<iso_field>

Choose the iso field to be represented (see below).

SCAL

Introduces the parameters relative to the *Scale* sub-menu.

A1

Use an automatic scale with just one value. This value is the average of the data extremes. By default, an automatic scale with 6 values is used.

A6

Use an automatic scale with 6 values. This is the default.

A14

Use an automatic scale with 14 values. By default, an automatic scale with 6 values is used.

USER

Use the fixed, user-specified scale given by `/PROG/`.

COSC

Introduces the color scheme to be used for the visualization of iso values.

COLS

Use colors (this is the default).

GRAY

Use a scale of grays.

ICOL

Use colors but invert the colors set (blue indicates the maximum value insted of minimum value).

IGRA

Use a scale of grays but invert the colors set (dark gray indicates the maximum value insted of minimum value).

Text Menu Parameters**TEXT**

Introduces the parameters relative to the **Text** menu.

NODE

Show node numbers.

ELEM

Show element numbers.

OBJE

Show object names.

VSCA

Show vectors scale.

ISCA

Show iso scale.

HINF

Hide info.

CAME

Show camera values.

DEBU

Show debug info.

PCON

Show pinball contacts.

Colors Menu Parameters

COLO

Introduces the parameters relative to the **Colors** menu. This menu allows to choose the colors of many graphical elements of the rendered scene, such as the background, the element outlines etc. To apply special colors to the model itself, or to parts of it, see the parameters relative to the **Lights/Mats** menu below. To apply a color, first it is selected by means of the **SELE** keyword, then it is applied to the desired graphical element by means of the **APPL** keyword. This sequence may be repeated as many times as necessary.

SELE

Introduces the selection of a color by means of the *Select color* sub-menu. The available colors are listed in the following Table. For greys, the number indicates the luminosity, i.e. GR05 is almost black, while GR95 is almost white.

Name	Color	Name	Color	Name	Color
RED	Red	GREE	Green	BLUE	Blue
CYAN	Cyan	MAGE	Magenta	YELL	Yellow
BLAC	Black	WHIT	White	GR05	Grey 05%
GR10	Grey 10%	GR15	Grey 15%	GR20	Grey 20%
GR25	Grey 25%	GR30	Grey 30%	GR35	Grey 35%
GR40	Grey 40%	GR45	Grey 45%	GR50	Grey 50%
GR55	Grey 55%	GR60	Grey 60%	GR65	Grey 65%
GR70	Grey 70%	GR75	Grey 75%	GR80	Grey 80%
GR85	Grey 85%	GR90	Grey 90%	GR95	Grey 95%

APPL

Introduces the application of the selected color by means of the *Apply it to* sub-menu. The available items to which a color may be applied are listed in the following Table.

Name	Item	Name	Item	Name	Item
BGRN	Background	CENT	Centre	BBOX	Bounding box
IFAC	Internal faces	ELOU	Element outlines	SHAR	Sharp corners
FRED	Free edges	PERP	Perpendicular contours	VECT	Vectors
ISOE	Iso surface edges	ISOL	Iso surface outlines	POIN	Points
NNUM	Node numbers	ENUM	Element numbers	ONAM	Object names
TEXT	Text	INWI	Initial wireframe	INOU	Initial outline
TRAJ	Debris trajectories	ISOD	Iso default color		

ISOD

Introduces the default color for isovalues. When iso-values are drawn and the user has selected only part of the mesh by the SUPP directive, the non-selected parts of the mesh are drawn in this color. The default value of this color is GR50 (i.e. 50% grey).

PAPE

Select default colors for paper. By default, colors are set for the screen. For example, by default the background is black and element outlines are white, while by the PAPE option the background is white and the element outlines are black.

Lights and Materials Menu Parameters

LIMA

Introduces the parameters relative to the **Lights/Mats** menu. This menu allows to switch the light on, to choose some parameters relative to the lighting model and to apply special materials (colors) to the model, or to parts of it. To apply a material, first it is selected by means of the SELE keyword, then it is applied to the desired object by means of the APPL keyword. This sequence may be repeated as many times as necessary.

ON

Switches the light on. The light must be on to see the special material effects properly.

LIGX

Introduces the x -position of the light **ligx**. By default, **ligx=-1**, i.e. the light comes from the left.

LIGY

Introduces the y -position of the light **ligy**. By default, **ligy=1**, i.e. the light comes from the top.

LIGZ

Introduces the z -position of the light **ligz**. By default, **ligz=1**, i.e. the light comes from the front.

LAMB

Introduces the ambient light, which may be **LOW**, **MEDIUM** or **HIGH**. By default, the ambient light is **MEDIUM** for dull surfaces, **LOW** for shiny surfaces. The values for dull or shiny surfaces are somewhat different.

LDIF

Introduces the diffuse light, which may be **LOW**, **MEDIUM** or **HIGH**. By default, the diffuse light is **HIGH** for dull surfaces, **HIGH** for shiny surfaces. The values for dull or shiny surfaces are somewhat different.

LSPE

Introduces the specular light, which may be **LOW**, **MEDIUM** or **HIGH**. By default, the specular light is **LOW** for dull surfaces, **HIGH** for shiny surfaces. The values for dull or shiny surfaces are somewhat different.

LSHI

Introduces the light shininess, which may be **LOW**, **MEDIUM** or **HIGH**. By default, the light shininess is **LOW** for dull surfaces. This quantity is unused for shiny surfaces.

LMAM

Introduces the model ambient light, which may be **LOW**, **MEDIUM** or **HIGH**. By default, the model ambient light is **MEDIUM** for shiny surfaces. This quantity is unused for dull surfaces.

SELE

Introduces the selection of a material by means of the *Select material* sub-menu. The available materials are listed in the following Table.

Name	Material	Name	Material	Name	Material
BRAS	Brass	BRON	Bronze	PBR0	Polished bronze
CHRO	Chrome	COPP	Copper	PCOP	Polished copper
GOLD	Gold	GOL2	Gold 2	PGOL	Polished gold
PEWT	Pewter	SILV	Silver	PSIL	Polished silver
EMER	Emerald	JADE	Jade	OBSI	Obsidian
PEAR	Pearl	RUBY	Ruby	TURQ	Turquoise
BLAP	Black plastic	CYAP	Cyan plastic	GREP	Green plastic
REDP	Red plastic	WHIP	White plastic	YELP	Yellow plastic
BLAR	Black rubber	BLR2	Black rubber 2	CYAR	Cyan rubber
GRER	Green rubber	REDR	Red rubber	WHIR	White rubber
YELR	Yellow rubber				

APPL

Introduces the application of the selected material by means of the *Apply it to* sub-menu.

MESH

Apply the selected material to the whole mesh.

SELO

Apply the selected material to the currently selected objects.

/LECT/

Apply the selected material to the specified objects.

Choice of a vector field**Object**

To select the vector field to be represented by the **VECT** directive described above.

Syntax:

```
VECT . . . <FIEL $ VITE ; VITG ; ACCE ; DEPL ; FINT ; FEXT ; FLIA ;  
          MASS ; VCVI ; FDEC $> . . .
```

VITE

Material or particle velocity (first **idim** components). This is the vector field represented by default, if the **FIEL** directive is omitted.

VITG

Mesh velocity in an ALE calculation (first **idim** components).

ACCE

Material or particle acceleration (first **idim** components).

DEPL

Displacement (first **idim** components).

FINT

Internal force (first **idim** components).

FEXT

Total external force (first **idim** components).

FLIA

External force due to liaisons (coupled links) (first **idim** components).

MASS

Nodal mass (first **idim** components).

VCVI

Material or particle velocity (first **idim** components) in Finite Volumes Cell Centred model. Note that these vectors are not represented at the nodes but at the “elements” (i.e., Finite Volumes) centroids.

FDEC

External force due to decoupled links (first **idim** components).

Choice of an iso field**Object**

To select the iso field to be represented by the **ISO** directive described above.

Syntax:

```
ISO . . . <FIEL $ CONT icon ; EPST iepst; ECRO iecr ;
          VITE <icom> ; VITG <icom> ; ACCE <icom> ;
          DEPL <icom> ; FINT <icom> ; FEXT <icom> ;
          MASS <icom> ; FLIA <icom> ; FDEC <icom> ;
          MCPR ; MCRO ; MCTE ; MCCS ; MCMF icom ;
          MCP1 ; MCP2 ;
          SIGN isig ; ECRN iecr ;
          ADFT ;
          FAIL ;
          RISK irsk ;
          LFEL ; LFEV ;
          LFNO ; LFNV ; ILNO ; DTNO ;
          VCVI <icom> ;
          CERR ; MAXC ; ERRI ; CLEN ; ILEN          $> . . .
```

CONT icon

The icon-th component of the stress tensor.

EPST iepe

The iepe-th component of the cumulated strain.

ECRO iecr

The iecr-th component of the hardening parameters.

VITE <icom>

Material or particle velocity: icom-th component if specified, else norm of the first idim components. This is the iso field represented by default, if the FIEL directive is omitted.

VITG <icom>

Mesh velocity in an ALE calculation: icom-th component if specified, else norm of the first idim components.

ACCE <icom>

Material or particle acceleration: icom-th component if specified, else norm of the first idim components.

DEPL <icom>

Displacement: icom-th component if specified, else norm of the first idim components.

FINT <icom>

Internal force: icom-th component if specified, else norm of the first idim components.

FEXT <icom>

External force: icom-th component if specified, else norm of the first idim components.

MASS <icom>

Nodal mass: icom-th component if specified, else norm of the first idim components.

FLIA <icom>

Liaison (coupled links) force: icom-th component if specified, else norm of the first idim components.

FDEC <icom>

Decoupled links force: icom-th component if specified, else norm of the first idim components.

MCPR

Finite volumes pressure (defined at nodes).

MCRO

Finite volumes density (defined at nodes).

MCTE

Finite volumes temperature (defined at nodes).

MCCS

Finite volumes sound speed (defined at nodes).

MCMF *icom*

Finite volumes mass fraction of the *icom*-th component. (defined at nodes).

MCP1

Finite volumes minimum pressure during the transient (defined at nodes).

MCP2

Finite volumes maximum pressure during the transient (defined at nodes).

SIGN *isig*

Stress (*isig*-th component) in spectral elements (defined at nodes).

ECRN *iecr*

Hardening parameter (*iecr*-th component) in spectral elements (defined at nodes).

ADFT

Advection-diffusion temperature (defined at nodes).

FAIL

Failure level of the element which has been reached: 0 means virgin element, 1 means completely failed element and an intermediate values indicates a partially failed element.

RISK *irsk*

Risk due to the effects of an explosion. Risk values go from 0 (no risk) to 1 (full risk). Risk is estimated in the fluid field, and at the moment, it is only computed in JRC's FLxx elements and the cell centred finite volumes (VFCC). To activate this calculation, it is necessary to specify the **RISK** keyword in the calculation type (see page A.30). The *irsk* parameter indicates the "component" (i.e. the type) of risk considered: 1 means eardrum rupture risk, 2 means death risk. Be aware that when reading results from an Alice file (produced by a previous calculation with risk activation), it is mandatory to (re-)specify the whole **RISK** directive (in particular as concerns the **PROB ...** and **LUNG ...** subdirectives, see page A.30), because the risk is computed with the current values of the optional parameters.

LFEL

Logarithm in base 2 of the level factor associated with elements in the spatial time step partitioning algorithm.

LFEV

Logarithm in base 2 of the level factor associated with elements including the neighbours in the spatial time step partitioning algorithm.

LFNO

Logarithm in base 2 of the level factor associated with nodes in the spatial time step partitioning algorithm (defined at nodes).

LFNV

Logarithm in base 2 of the level factor associated with nodes including the neighbours in the spatial time step partitioning algorithm (defined at nodes).

ILNO

Flag indicating whether a node is (1) or is not (0) subjected to a link condition, used in the spatial time step partitioning algorithm (defined at nodes).

DTNO

Stability time step associated with nodes, used in the spatial time step partitioning algorithm (defined at nodes).

VCVI <icom>

Material or particle velocity in Finite Volumes Cell Centred model: **icom**-th component if specified, else norm of the first **idim** components.

CERR

Constant used in element error indicator calculation (adaptivity), see the **CERR** input keyword of the **ADAP** directive on page B.210.

MAXC

Maximum principal curvature of least-squares fitting function, used for element error indicator calculation (adaptivity).

ERRI

Element error indicator (adaptivity),

CLEN

Current characteristic element length used in error indicator calculations.

ILEN

Optimal (indicated) characteristic element length resulting from error indicator calculations.

The MCPR, MCRO, MCTE, MCVI and MCMF keywords are available only in calculations with finite volumes.

The SIGN and ECRN keywords are available only in calculations with spectral elements.

The ADFT keyword is available only in calculations with advection-diffusion.

The FAIL keyword is available only in calculations with the element erosion algorithm, see Page A.30.

The LFEL, LFEV, LFNO, LFNV, ILNO and DTNO keywords are available only in calculations with spatial time step partitioning (OPTI PART, see Page H.20).

The CERR, MAXC, ERRI,
tt CLEN, ILEN keywords are available only in calculations with “true” adaptivity (see the ADAP directive on page B.210).

17.1.9 TITLES options

Object

To define a set of parameters (globally indicated above as **<tpars>**) for the definition of a frame or AVI sequence containing titles, to be used during OpenGL rendering.

Once defined by a **TITL** directive, the first following **TRAC** directive will produce a titles frame (or AVI sequence) instead of the normal rendering of geometrical objects.

The color of the background and of the text used for the titles may be set by means of the **SCEN** directive described above (background color and text color, respectively).

Note that on EUROPLEXUS versions implemented on a non-OpenGL platform, this directive is simply (and entirely) ignored. This enhances portability of benchmark tests on the various platforms.

Syntax:

TITL

```
<TIT1 'text1'>  
<TIT2 'text2'>  
<TIT3 'text3'>
```

TIT1

Introduces the text of the first title (**text1**), enclosed in quotes. This text appears centered in the upper part of the frame or sequence.

TIT2

Introduces the text of the second title (**text2**), enclosed in quotes. This text appears centered in the central part of the frame or sequence, and is therefore the “main” title.

TIT3

Introduces the text of the third title (**text3**), enclosed in quotes. This text appears centered in the lower part of the frame or sequence.

Comments:

If none of the titles is given, the **TITL** directive desactivates the production of title frames (i.e. the next **TRAC** directive will produce regular geometry rendering).

Any omitted title is not represented in the titles frame.

18 GROUP SR—SAVING AND RESTART

The following directives allow to save the data during a computation and to the successive restart of the computation.

18.1 SAVING

Object:

To produce a saving file for successive restart, the directive **ECRI FICH SAUV** has to be inserted in group G of the EUROPLEXUS input data (as part of the **ECRI** directive). See page G.110 for details on the **SAUV** directive.

18.2 RESTART

Object:

The REPR keyword, to be inserted in group A, enables to restart a computation which was previously saved.

Syntax:

```
... title ...

"REPRISE"  nbanrep      "POSI" numer      < "PROT"  'maclef' >

... Instructions of the groups C,D,E,F,G that are modified ...
... (in particular, if a further saving is desired, repeat
    the ECRI FICH SAUV directive :)                               ...

< ECRI ... FICH  SAUV <ndsauv> <PROT 'maclef'> <LAST> </CTIM/> >

... Instructions of the group H that are modified ...

"CALCUL"   . . .

"FIN"
```

The REPR instruction is described in detail below, Page SR.30.

Warning:

During a restart the structure of the data is very different. In general, only the instructions of the groups C,D,E,F,G,H which are modified have to be repeated. However, note that the directive ECRITURE must be repeated even if it is not changed. All other instructions are useless. Especially, the geometry must NEVER be repeated.

During a restart run, the directive FICH ALIC reads the file produced during the previous run and adds the new results like if a single calculation would have been performed.

Comments:

The various possibilities are described in the following pages.

During the first run, the directive `ECRI FICH SAUV` `FREQ n` causes data to be saved once every `n` time steps, on the default saving file.

During a restart run, the directive `REPR` `'myfile.sau'` `j` reads the previously created saving file and re-starts the calculation from the `j`-th saved data station.

If saving is requested during a restart run, be aware that a saving is not performed at the restart time itself.

During a restart run, all frequencies `nf` or `tf` in directives of the type:

- `ECRI FICH SAUV iu FREQ nf`
- `ECRI FREQ nf`
- `ECRI TFRE tf`
- `ECRI FICH ALIC ju FREQ nf`
- `ECRI FICH ALIC ju TFRE tf`
- ...

are considered starting from the initial time (or the initial step) of the first run, not of the restart run.

During a restart, if a directive similar to `CALC ... NMAX nsteps ...` is used, be aware that `nsteps` is the total number of time steps, not the number of steps during the restart run.

Files:

The listing file is always produced anew during a restart, beginning at the restart time.

The saving file is specified by the instruction `ECRI FICH SAUV`.

The restart one is specified by the directive `REPRISE`.

For postprocessing files, see the description of `REPRISE` below.

18.3 DIRECTIVE “SAUVE” (OBSOLETE FORM)

This is the obsolete form of the **SAUVER** directive, which produces a saving file for subsequent restart of the calculation. It is only included here for compatibility with old input files. For new input files, please use the **ECRI ... FICH SAUV** directive, described on page G.110.

Object:

This keyword creates a saving file and, in conjunction with the keyword **REPR** (to be used in a subsequent run), allows splitting a computation in two or more parts.

The results are saved on a file (saving file) at times specified by the user. Each saving corresponds to a number or position on the file (1, 2, 3 etc.), from which a restart of the computation can be carried out (see directive **REPR** on page SR.30).

Syntax:

```
SAUV nbansav < FREQ > ifreq < DER > < PROT 'maclef' >
```

nbansav

Number of the saving file or name of the file in quotes. If completely omitted, the code will assume the default file name **<basename>.sau** where **<basename>** is the root of the input file name (i.e. without extension **.epx**). However, note that in this case the following keyword **FREQ** becomes mandatory to introduce the frequency.

ifreq

Frequency of the savings, in time steps. The results are saved each **ifreq** computation steps. Note that the code always saves the last step of the calculation (if the run is terminated normally), irrespective of the frequency chosen. Therefore, if one is only interested in getting the possibility to continue the calculation further on, a very large frequency may be chosen, larger than the total number of steps expected in the present run.

DER

This keyword indicates that the saving file should contain just one saving station, corresponding to the last saved time station in the present calculation. In other words, each new saving station replaces the former one, if any. This allows to obtain a saving file of the smallest possible size. However, restarting from an intermediate time is obviously not possible in this case: the only possibility to restart the calculation will be **REPR ... POSI 1** (see page SR.30).

PROT

Keyword entering a protection on the saving file.

'maclef'

Key of up to 8 characters, enclosed in apostrophees. In order to restart the computation from that file, the instruction **REPR** must contain the keyword **PROT** with an identical key.

Comments:

The keyword **FREQ** introduces a fixed frequency in time steps and has been maintained for backward compatibility (also, if **FREQ** is missing, and a number is read instead, the program assumes that it indicates a step frequency **ifreq**). This form of the directive (e.g. **SAUV 30 1000**) is still accepted for backwards compatibility only but is deprecated.

The keyword **PROT** is not compulsory. If it is not used, there is no protection (this is equivalent to a key of 8 blanks). If specified, the keyword **PROT** must be at the end of the **SAUV** directive (i.e., after **DER**, if any).

If a unit number is used for **nbansav**, the saving file and its number must have been defined before on the control cards.

A first saving station (position number 1) containing some header data is always produced at the initial time (step 0 of the calculation). Of course, it is normally meaningless to restart from this time station, unless the **DER** keyword has been specified (see above), because it would be the same as starting the calculation anew from the initial time. On the contrary, if **DER** has been specified, the only possibility for restart is to use the first time station which, in this case, will contain the data of the last saving performed (not the first one in general).

Examples:

Assume a calculation performs 4994 time steps to arrive at its final time. The following saving directives are accepted:

- **SAUV 'myfile.sau' FREQ 1000** saves data for restart on the indicated file every 1000 steps. The following six saving stations are produced: 1 (step 0), 2 (step 1000), 3 (step 2000), 4 (step 3000), 5 (step 4000) and 6 (step 4994, i.e. the last step).
- **SAUV FREQ 1000:** same as above but the saving file has the default name **<basename>.sau**.
- **SAUV 'myfile.sau' FREQ 1000 DER** saves data for restart on the indicated file every 1000 steps, by always re-writing the previous saving. Only one saving station is thus present on the saving file (assuming that the run terminates normally): 1 (step 4994, i.e. the last step). If the run would fail, say, at step 2500, the saving file would also contain one station (step 2000).

- **SAUV FREQ 10000**: the saving file has the default name **<basename>.sau**. Only one saving station is produced (assuming that the run terminates normally): 1 (step 4994, i.e. the last step). If the run would fail, say, at step 2500, no (useful) saving time station would be available (there is still one saving station, but it is at step 0).

The following saving directives are still accepted for backwards compatibility but are strongly deprecated:

- **SAUV 30 1000** saves data for restart on unit 30. Under Windows, this produces a file named **fort.30** on the current directory. The saving frequency is 1000, so the following six saving stations are produced: 1 (step 0), 2 (step 1000), 3 (step 2000), 4 (step 3000), 5 (step 4000) and 6 (step 4994, i.e. the last step).

18.4 DIRECTIVE “REPRISE”

Object:

This option enables a computation to be restarted from a time which was previously saved on a ‘saving’ file. This file now becomes the restart file and contains, besides the computed values at the saved time stations, a certain amount of data of the preceeding computation which **must not** be defined again in the restart input file (see page SR.40).

Syntax:

```
REPR  nbanrep  POSI numer  < PROT 'maclef' >
```

nbanrep

Number of the restart unit, or name of the restart file in quotes.

numer

Number of the saving station which determines the restart (1, 2, 3 etc.). The restart will occur from the **numer**-th saved time station.

PROT

Compulsory keyword if there is a protection on the saving file.

'maclef'

Key protecting the saving file.

Comments:

If a unit number is used for **nbanrep**, the restart file and its unit number must have been defined first in the control cards. The restart file corresponds to the former saving file (i.e., it has the same file name).

During a restart, the user can save data for a further restart. In this case, the logical numbers of the saving and restart files must be different.

As far as data storage for postprocessing purposes (FICH ALIC, TPLOT, XPLOT, K2000 etc.) is concerned, two strategies may be followed during a restart. The first one consists in producing a single results file that after the restart corresponds to the file which would have been

produced by a single run. To obtain this behaviour, during the restart use the same storage file which was defined in the previous run. The program will read the previous results file, position the pointer at the correct time (restart time) and then continue writing the data on the file.

The second possibility is that of splitting the results file in several pieces, one for the first run, another for the first restart, and so on. This may be useful e.g. in very large computations, to keep the file size acceptable. To obtain this behaviour, simply change the name of the results file each time you restart the calculations. The program will then produce a new data set containing also the necessary header information (e.g., geometry, etc.).

Examples of both techniques are available in the example files.

18.5 DATA NECESSARY FOR RESTARTS

Warning:

The data structure for restarts is different from that of a normal run.

All the EUROPLEXUS data relative to the preceeding computation are written on the restart file.

By explicitly re-defining a directive, the former one is cancelled and replaced. The user can also add some new directives.

The following directives must be repeated:

- Title card;
- Dimension (or keyword `TERM`);
- Directives `CALCUL` and `FIN`.

The following directive is NOT repeated:

- Geometry (with mesh).

The following directives may be defined again:

- Dimensions;
- Masses;
- Thicknesses;
- Materials;
- Connections;
- Loads;
- Printouts;
- Options.

Comments:

DIMENSIONS: they can be larger (if for example loads are added), or smaller (if too much space had been provided) than the dimension of the preceeding computation. If it is not modified, the word **TERM** at least must be used.

MATERIALS: the density of the material and the stress-strain law can be changed, but the value of the initial stresses and strains (final value of the saving computation) must be compatible with the new law for all elements.

CONNECTIONS: the user can for example add or suppress a blocked displacement; in this case, do not forget to define again the whole instruction. If he wants to cancel all the connections of the preceeding computation, only the keyword **LIAISON** is necessary, without any other sub-instruction.

LOADS: the loads may be completely re-defined. Sometimes this is necessary, if the final time defined in the arrays becomes lower than the final time of the instruction **CALCUL**.

COMPUTATION: the start time corresponds to the time of restart. If it does not, EUROPLEXUS uses the time written on the restart file. The number of time steps takes the preceeding computation into account. If the user stops after 1000 steps, but wants to continue for a further 500 steps, the total number of time steps will be 1500.

19 GROUP RM—CHANGE OF TOPOLOGY ("REMAILLAGE")

Object:

This directive enables "remeshing", i.e. changing the topology of a part of the mesh previously defined by directive "GEOM".

Note that this is very different from mesh "rezoning", i.e. prescribing the motion of a fluid grid in ALE computations. In fact, rezoning operations just move the nodes, without changing the topology (number of nodes and elements, composition of the elements).

For remeshing, two options are possible:

- automatic;
- manual.

With the option "AUTOMATIQUE", the user has to supply only the boundary points of the zone submitted to remeshing, then EUROPLEXUS builds a new and more regular mesh based on the boundary.

With the option "MANUEL", the user must enumerate all the elements which are to be eliminated and supply the new mesh.

At the present time, only 3-noded triangles ("TRIA") can be taken into account.

Syntax:

```
"REMAILLAGE"
$  "AUTOMATIQUE"  . . .  $
$  "MANUEL "      . . .  $
```

Comments:

The suppressed elements must belong to the same zone (defined by the instruction "GEOMETRIE").

19.1 AUTOMATIC REMESHING

Object:

The new sub-mesh is created by EUROPLEXUS. Therefore, only the boundary points are necessary. As a matter of fact, these points should be common to the old and the new mesh.

Syntax:

```
"AUTOMATIQUE"    "CONTOUR"    /LECTURE/  
                  "AJOUTER"    npts*( xi , yi )  
                  < "ELEM"    ideb ifin >  < "ZONE"    nzone >  
                  < "MAIL"    < nmail > >  < "STOP" >
```

/LECTURE/

Numbers of the existing points making up the boundary.

npts

Number of points to be added.

xi , yi

Coordinates of the added points (abscissa then ordinate for each of the npts points).

ideb,ifin

The numbers of the elements of the zone concerned are lying between ideb and ifin.

nzone

The elements of the zone to be remeshed belong to the zone number nzone.

nmail

Number of the logical unit (by default nmail = 7), where the data set available for the procedure "LECTURE" of "COCO" is stored.

Comments:

If the boundary point is an existing node, it is defined by its number.

If it is an added point, it is defined by a negative number. In this case, at the end to the procedure /LECTURE/, the user must enter the coordinates of these points in the same order as they are mentioned in the procedure, by the means of the instruction "AJOUTER".

The directives "ELEM" and "ZONE" enable the elements submitted for remeshing to be limited.

By means of the directive "MAIL", the user can obtain a data set for "COCO" in order to plot the results of the remeshing.

The key-word "STOP" stops EUROPLEXUS when the mesh is created and enables the user to check his mesh.

19.2 MANUAL REMESHING

Object:

This option enables a new partial mesh to be explicitly entered.

Syntax :

```
"MANUEL"   ize   nelim nelaj npaj
            "ANCIEN" /LECTURE/
            "NOUVEAU"

            . . .   COCO data set . . .

            "IDENTIFIER" npts*( numl numg )
```

ize

Number of the zone of elements which must be eliminated.

nelim

Total number of the elements to be eliminated.

nelaj

Total number of the elements to be added and belonging to the new sub-mesh.

npaj

Total number of points belonging to the new sub-mesh.

ANCIEN

Keyword defining the old sub-mesh.

LECTURE

List of the elements belonging to the sub-mesh to be eliminated.

NOUVEAU

Keyword introducing the "COCO" data set of the new sub-mesh.

npts

Number of pairs (local number, global number) to be identified.

numl

Local number of the node in the sub-mesh

numg

Number of that same node in the global mesh.

Comments:

The "COCO" data set with its title is composed of the coordinates of the new sub-mesh (in format 6E12.5) and of the numbers of the mesh elements (in format 18I4).

The key-word "IDENTIFIER" enables the integration of the new sub-mesh into the global mesh to be defined.

19.3 CHECKING THE REMESHING

Object:

To edit (verify) the result of a remeshing operation.

Syntax:

```
< "PERFO" nuperf > < "TRACE" nutrac > < "SAUVE" nusauv >
```

During a restart the user can save data for another restart.

nuperf

Number of the logical unit which will contain the "COCO" data necessary to plot a new sub-mesh.

nutrac

Necessary to write the results on the file "TRACE" number nutrac.

nusauv

To save the results on the file nusauv, so as to enable a further restart.

Comments:

The user is advised to use these commands, because the numbers of certain elements may have changed after a remeshing. That is why it is necessary to save that time step (considered by EUROPLEXUS as step zero). Do not forget to check that the new mesh is correct before the computation continues by means of a restart ("REPRISE").

20 GROUP EX—EXAMPLES

On the following pages the user may find some examples of input files. These are actually taken from the PLEXIS-3C manual, since a separate example manual for EUROPLEXUS is still being prepared. Therefore, the examples are purely indicative of the 'flavour' of the program.

The user can find many actual EUROPLEXUS examples (including input, pre-treatment and post-treatment files) in some of the publications listed in the bibliography at the end of the present manual (see in particular PLEXIS-3C reports from 1992 on).

Unless it is specified otherwise, the mesh is created by means of GIBI. The GIBI data is given so as to define the objects without any ambiguity.

Remark:

The semicolon (;) indicates the end of the card, and the dollar (\$) at the beginning of the card announces a commentary. The sequence (;\$) enables comments on the same line as the data to be entered.

20.1 BENDING OF A BEAM

Object:

This is a 2-D elastic computation.

A beam is subjected to an uniform stress (pressure).

Geometry and meshing :

$L = 24.0$ mm : half length of the beam;

$e = 1.0$ mm : thickness;

The mesh is entered in free format;

There are 12 "COQU" shell elements.

Physical properties:

$\rho = 8000$ Kg/m³ : density;

$\nu = 0.3$: Poisson's ratio;

$E = 200$ GPa : Young's modulus.

Boundary conditions:

- The pressure increases from 0 to 2 MPa in 0.1 millisecond, then remains constant.
- Clamped boundary and symmetry conditions are applied at the center.

Computation:

The step is automatic and the computation concerns 5000 time steps.

In order to visualize the results more easily, the computation is followed by the drawing of the time dependant displacement of the center of the beam.

List of the input file:

```
----- VIBRATION OF A BEAM -----
DPLA LINE
PT3L 13 ZONE 1
COQU 12
BLOQ 4 PRES 12 TABLE 1 3
TERM
GEOM LIBRE POINT 13 COQU 12 TERM
  0 0  2 0  4 0  6 0  8 0 10 0 12 0
14 0 16 0 18 0 20 0 22 0 24 0
1 2 2 3 3 4 4 5 5 6 6 7 7 8 8 9 9 10 10 11 11 12 12 13
EPAIS 1. TOUS
MATE LINE RO 8E-9 YOUNG 2E5 NU 0.3 TOUS
LIAISON BLOQ 23 LECT 1  TERM
          13 LECT 13  TERM
CHARGE 1 FACTO 2 PRES COQU -2. TOUS
          TABLE 3 0 0  1E-4 1  1 1
IMPRE FREQ 1000 TRACE ALICE 10 50
OPTION NOTEST
CALCUL 0 1E-7 1E-6 5000 1
SUITE
----- VIBRATION OF A BEAM -----
RESUL 10
TEMPS 120 COURBES 4 TERM
SORTIES GRAPHIQUES
AXTEMPS 1E3 'TEMPS (MS)'
COURBE  1 'COQU'  DEPLA COMP 2 NOEU 13
DESSIN 1  1  AXES 1. 'FLECHE (MM)'
FIN
```

20.2 IMPACT ON A CIRCULAR PLATE

Object:

This is a 2-D plastic computation.

A clamped plate is submitted to the impact of a rigid missile.

Geometry and meshing :

$D = 465$ mm : diameter of the plate

$e = 6$ mm : thickness

The missile is a flat nose cylinder, 90 mm in diameter. It falls straight in the middle of the plate.

The user wants to follow the displacements of point P1 to P4 whose location has been imposed.

GIBI data:

```
TITRE 'IMPACT SUR DES PLAQUES CIRCULAIRES' ;
OPTIO DIME 2 ELEM SEG2 ;
DPROJ=90; RPROJ=DPROJ / 2;
DENS 7;
CENTR= 0 0 ; BORD=RPROJ 0 ; BORD2=DPROJ 0 ; ENCAS=233.5 0 ;
  P1=60 0 ;   P2=80 0   ;   P3=120 0   ;   P4=160 0   ;
  P0= 0 0 ;
LIG1=CENTR DROIT BORD ; LIG1=LIG1 COUL ROUG ;
LIG2=BORD D P1 D P2 D BORD2 D P3 D P4 D ENCAS ;
LIG2=LIG2 COUL VERT ;
PLAQ=LIG1 ET LIG2 ;
PROJ=MANUEL POI1 P0;
TOUT=PROJ ET PLAQ;
SORTIE TOUT;
FIN;
```

Physical properties:

$\rho = 7800$ Kg/m³ : density;

$\nu = 0,3$: Poisson's ratio;

$E = 230$ GPa : Young's modulus;

$Y = 188$ MPa : elastic limit.

Boundary conditions:

The missile has a mass of 257 Kg, it falls at a speed of 11.38 m/s (be careful, the axisymmetric computation concerns ONE radian).

- Clamped boundary and symmetry conditions.

Computation:

The step is automatic and the computation concerns the first 12 milliseconds.

In order to visualize the results more easily, the computation is followed by three drawings:

- impulse of the missile
- displacements of the missile and the nodes submitted to the impact
- displacements of the center and the remarkable points

List of the input file:

```
--- IMPACT SUR DES PLAQUES CIRCULAIRES (E=6 D=90 M=257 VI=11.38)
ECHO
GIBI 9 TOUT
AXIS
DIMENSION
      BLOQ 5 TRAC 1 13 IMPA 1 PSIMPA 7
TERM
GEOM COQU PLAQ PMAT PROJ TERM
COMPLEMENTS
      EPAIS 6 LECT PLAQ TERM
MATE VMIS ISOT RO 7.8E-9 YOUN 230E3 NU .3 ELAS 188
      TRAC 13 188 .0817E-2
              261 .2000E-2
              288 .3000E-2
              318 .4900E-2
              339 .7500E-2
              354 1.07E-2
              377 1.94E-2
              423 4.75E-2
              497 9.54E-2
              585 18.20E-2
              649 26.20E-2
              693 33.70E-2
```

```

      710 37.20E-2
    LECT PLAQ TERM
    MASS 40.903E-3    LECT PROJ TERM
INIT VITE 2 -11380 LECT PROJ TERM
LIAIS BLOQ 123 LECT ENCAS TERM
      13 LECT CENTR TERM
    IMPACT DDL 2 COTE -1 PROJ LECT PROJ TERM
              CIBLE LECT LIG1 TERM

IMPR TFREQ 1E-3
    TRACE 10 TFREQ .1E-3
OPTION NOTEST
CALCUL 0 3E-7 1E-6 60000 12.01E-3
SUITE
--- IMPACT SUR DES PLAQUES CIRCULAIRES (E=6 D=90 M=257 VI=11.38)
RESUL 10
DIME
TEMPS 130 COURBES 10 TERM
SORTIES GRAPHIQUES
AXTEMPS 1000 'TEMPS (MS)'
COURBE 1 'IMPULSION' ECROU COMP 1 ELEM 31
      DESSIN 1 1          AXES 1. 'IMPUL. (N*S)'
COURBE 2 'D-PROJ'      DEPLA COMP 2 NOEUD 32
COURBE 3 'D-CENTRE'    DEPLA COMP 2 NOEUD 31
COURBE 4 'D-BORD'      DEPLA COMP 2 NOEUD 25
COURBE 5 'D-60 (J1)'   DEPLA COMP 2 NOEUD 23
COURBE 6 'D-80 (D1)'   DEPLA COMP 2 NOEUD 21
COURBE 7 'D-120 (D2)'  DEPLA COMP 2 NOEUD 16
COURBE 8 'D-160 (D3)'  DEPLA COMP 2 NOEUD 11
      DESSIN 3 2 3 4          AXES 1. 'DEPLA (MM)'
      DESSIN 6 3 4 5 6 7 8    AXES 1. 'DEPLA (MM)'
FIN
```


20.3 EXPLOSION IN A TANK

Object:

This is a 2-D computation in A.L.E.

A cylindrical tank is filled with water. At its center a micro charge of T.N.T. explodes. the development of a gas bubble (supposed perfect), and deformations of the cylinder may be observed.

Geometry and meshing:

Only one quarter of the tank is meshed. Water and gas are modelled by quadrilaterals and the cylinder by thin shell elements. There are also elements of fluid-structure interactions.

$R = 0.19$ m : radius of the cylinder;

$H = 0.19$ m : half height;

$r = 0.299$ m : initial radius of the bubble.

GIBI data:

```
TITRE 'MAILLAGE MANON';
OPTION DIME 2 ELEM QUA4;
RBUL=0.029947;RAY=0.19;
CB=0 0 ; FBR=RBUL 0; FLR=RAY 0;
FBZ=0 RBUL; FLZ=0 RAY;TOP=RAY RAY;
LBR=CB D 5 FBR;LLR=FBR D 12 FLR;
LBZ=CB D 5 FBZ;LLZ=FBZ D 12 FLZ;
AUX=RBUL*(SIN 45);P45=AUX AUX;
FB1=C 5 FBR CB P45;FB2=C 5 P45 CB FBZ;
FBUL=FB1 ET FB2;
BULLE=LBR FB1 FB2 LBZ DALLER PLAN;
TOIT=FLZ D 5 TOP;
SUR =FLR D 5 TOP;
SEP =P45 D 12 TOP;
LIQ1 =LLR SUR SEP FB1 DALLER PLAN;
LIQ2 =SEP TOIT LLZ FB2 DALLER PLAN;
EAU = LIQ1 ET LIQ2;
VA=0 0;CQ1=FLR PLUS VA;CQ2=TOP PLUS VA;
COQ=CQ1 D 5 CQ2;
RAC=RACCOR 0.001 SUR COQ;
PBUL=BULLE CHANGE POI1;
```

```
PBUL=PBUL DIFF ((FBUL CHANG POI1) ET CB);
PLIQ1=LIQ1 CHANGE POI1;
PLIQ1=PLIQ1 DIFF ((FB1 ET SEP ET SUR) CHANG POI1);
PLIQ2=LIQ2 CHANGE POI1;
PLIQ2=PLIQ2 DIFF (((FB2 ET SEP) CHANG POI1) ET FLZ);
PSEP=(SEP CHANG POI1) DIFF (TOP ET P45);
ZALE=PBUL ET PLIQ1 ET PLIQ2 ET PSEP;
TOUT=BULLE ET EAU ET RAC ET COQ ET ZALE;
SORTIE TOUT;
FIN;
```

Physical properties:**Water:**

$\rho = 1000 \text{ Kg/m}^3$: density;
 $c = 1500 \text{ m/s}$: velocity of sound.

Bubble:

$\rho = 482 \text{ Kg/m}^3$: density;
 $p_{ini} = 288 \text{ MPa}$: initial pressure;
 $\gamma = 1.535$: C_p/C_v ratio.

Cylinder:

$\rho = 7900 \text{ Kg/m}^3$: density;
 $E = 190 \text{ GPa}$: Young's modulus;
 $\nu = 0.3$: Poisson's ratio;
 $\sigma_{elas} = 265 \text{ MPa}$: elastic limit.

Boundary conditions:

- The upper part of the tank is very rigid, therefore displacements have been embedded along z.

Computation :

The step is automatic and the computation is done during the first two milliseconds.

In order to visualize the results more easily, the computation is followed by the drawing of the time dependant displacements of the generating line of the cylinder.

List of the input file:

```
TEST MANON 11 (1/4) ( IN A.L.E. WITH CAR4 ) !titre
```

```
echo
```

```
GIBI 9 TOUT
```

```
$TRAC
```

```
AXIS ALE
```

```
DIME
```

```
    BLOQ 50 RELA 1 2 NALE 200
```

```
TERM
```

```
GEOM CAR4 BULLE EAU COQU COQ FS2D RAC TERM
```

```
COMPL
```

```
    EPAIS 1.25E-3 LECT COQ TERM
```

```
GRILLE
```

```
    LAGRANGE LECT COQ FBUL TERM
```

```
    ALE      LECT BULLE EAU TERM
```

```
$ POUR LE CONTACT "FS"
```

```
    FS      LECT RAC TERM
```

```
$ POUR L'EAU
```

```
    LIGNE BASE LECT CQ2 P45 TERM
```

```
        LIST LECT PSEP TERM
```

```
    PLAN   BASE LECT CQ1 CQ2 P45 FBR TERM
```

```
        LIST LECT PLIQ1 TERM
```

```
    PLAN   BASE LECT CQ2 FLZ FBZ P45 TERM
```

```
        LIST LECT PLIQ2 TERM
```

```
$ POUR LA BULLE
```

```
    PLAN   BASE LECT CB FBR P45 FBZ TERM
```

```
        LIST LECT PBUL TERM
```

```
MATERIAUX
```

```
    VMIS ISOT RO 7900. YOUNG 190E9 NU .3 ELAS 265E6
```

```
        TRAC 5 265E6      .00139
```

```
            352E6      .0202
```

```
            481E6      .105
```

```
            559E6      .2214
```

```
            600E6      .349
```

```
        LECT COQ TERM
```

```
FLUI RO 1000. C 1500. PINI 1E5 PREF 1E5 PMIN 0
LECT EAU TERM
GAZP RO 482. GAMMA 1.535 PINI 2.88E8 PREF 1E5
LECT BULLE TERM
LINK RENUM
BLOQ 1 LECT LBZ LLZ CQ2 TERM
2 LECT TOIT LBR LLR TERM
23 LECT CQ1 CQ2 TERM
ECRITURE
TFREQ .25E-3 TRACE ALICE 10 TFREQ .5E-4
OPTION AUTO
NOTEST
CALCUL tini 0 nmax 1000 tfin 1.005E-3
SUITE
TEST MANON 11 (1/4) ( EN A.L.E. AVEC CAR4 )
RESULT 10
TEMPS 100 COURBE 5 TERM
SORTIE GRAPHIQUE
AXTEMPS 1E3 'T (MILLISEC.)'
COURBE 1 'DR-00 ' DEPLA COMP 1 NOEUD 5
COURBE 2 'DR-38 ' DEPLA COMP 1 NOEUD 6
COURBE 3 'DR-76 ' DEPLA COMP 1 NOEUD 4
COURBE 4 'DR-114 ' DEPLA COMP 1 NOEUD 3
COURBE 5 'DR-152 ' DEPLA COMP 1 NOEUD 2
DESSIN 5 1 2 3 4 5 AXES 1000 'DR-COQUE (MM)'
FIN
```

20.4 MODELLING OF PERFORATED PLATES

Object:

This is a 2-D computation in A.L.E.

A plane wave is propagating in a cylindrical tube. It meets a perforated plate that generates partial reflections and head losses. The walls of the cylinder are supposed rigid and the plate flexible.

Geometry and meshing:

Only the fluid in the tube is meshed. Elements with absorbant boundary conditions shall avoid reflected waves. At one end, a range of CL2D elements enables pressure source to be input in the form of a slope. These elements are superimposed to the absorbant elements.

The clamped plate is meshed with shell elements. The fluid is meshed in a continuous way to make it pass through the plate. The connecting elements of the fluid-structure junction ensure the coupling between the plate and the neighbouring nodes of the fluid. Another range of CL2D elements enables the characteristics of the grid to be input for the computation of the head losses.

$R = 500$ mm : radius of the cylinder;

$H = 1000$ mm : half height.

GIBI data:

```
TITRE 'MAILLAGE D'UNE PLAQUE PERFOREES';
OPTI DIME 2 ELEM QUA4 NIVEAU 1;
PA=0 0 ; PB=500 0 ; PC=500 2000 ; PD=0 2000;
PE=0 1000 ; PF=500 1000;
ENT=PA D 5 PB ; SOR=PC D 5 PD ;
GRI=PF D 5 PE ; ENT2=ENT ;
LBF=PB D 10 PF ; LFC=PF D 10 PC ;
LDE=PD D 10 PE ; LEA=PE D 10 PA ;
LIQ1=ENT LBF GRI LEA DALLER PLAN ;
LIQ2=(INVE GRI) LFC SOR LDE DALLER PLAN ;
LIQ=LIQ1 ET LIQ2 ;
PCE=0 1000 ; PCF=500 1000 ;
COQ=PCF D 5 PCE;
FSE=RACC 0.00001 LIQ1 COQ ;
PLIQ=LIQ CHAN POI1;
```

```
PLIQ=PLIQ DIFF (GRI CHAN POI1);  
AXE=LDE ET LEA ET PCE ;  
BOR=LBF ET LFC ET PCF ;  
LIQ=LIQ COUL BLEU;COQ=COQ COUL ROUG;  
FSE=FSE COUL TURQ;ENT2=ENT2 COUL VERT;  
TOUT=LIQ ET COQ ET FSE ET PLIQ ET AXE ET BOR ET ENT2;  
SORT TOUT;  
fin;
```

Physical properties:**Fluid:**

$\rho = 1000 \text{ Kg/m}^3$: density
 $c = 1000 \text{ m/s}$: velocity of sound

Plate:

$\rho = 7800 \text{ Kg/m}^3$: density
 $E = 190 \text{ GPa}$: Young's modulus
 $\nu = 0.3$: Poisson's ratio

Computation:

The step is automatic and the computation is carried out for the first 6 milliseconds.

In order to visualize the results more easily, the computation is followed by two drawings. The first one describes the evolution of the pressures upstream and downstream, and the second describes the displacement of the center of the plate

List of the input file:

```
PLAQUE PERFOREE SOUPLE (EULER) ALP=1 TAU=0 E=15  
ECHO  
GIBI 9 TOUT  
AXIS EULER  
DIMENSION  
    BLOQ 60 NALE 100  
    TERM  
GEOM CAR1 LIQ CL2D ENT SOR GRI ENT2 COQU COQ FS2D FSE TERM
```

```

GRIL LAGR LECT COQ TERM
  ALE LECT LIQ TERM
  FS LECT FSE TERM
COMPLEMENT EPAIS 15 LECT COQ TERM
MATE LINE RO 7.8E-9 YOUNG 190E3 NU 0.3
  LECT COQ TERM
  FLUI RO 1E-9 C 1E6
  LECT LIQ TERM
  IMPE PIMP RO 1E-9 PRES -0.02
    TABP 3 0 0 0.002 1 1 1
  LECT ENT TERM
  IMPE ABSO RO 1E-9 C 1E6
  LECT SOR ENT2 TERM
  IMPE GRIL RO 1E-9 C 1E6 ALP 1 TAU 0
  LECT GRI TERM
LIAI RENUM
  BLOQ 1 LECT AXE TERM
    1 LECT BOR TERM
    2 LECT PCF TERM
    3 LECT PCE PCF TERM
  FS LECT FSE TERM
IMPR FREQ 500
  TRAC ALIC 10 10
OPTI PAS AUTO
  NOTEST
CALCUL 0. 1E-5 1E-4 500 6E-3
SUITE
PLAQUE PERFOREE SOUPLE (EULER) ALP=1 TAU=0 E=15
RESULT 10
TEMPS 60 COURBE 7 TERM
SORTIE GRAPHIQUE
AXTEMPS 1E3 'T (MILLISEC.)'
COURBE 1 'P-3 ' ECROU COMP 1 ELEM 3
COURBE 2 'P-28 ' ECROU COMP 1 ELEM 28
COURBE 3 'P-48 ' ECROU COMP 1 ELEM 48
COURBE 4 'P-53 ' ECROU COMP 1 ELEM 53
COURBE 5 'P-73 ' ECROU COMP 1 ELEM 73
COURBE 6 'P-98 ' ECROU COMP 1 ELEM 98
COURBE 7 'DZ-64 ' DEPLA COMP 2 NOEU 64
DESSIN 6 1 2 3 4 5 6 AXES 10 'PRES. (BARS)'
DESSIN 1 7 AXES 1. 'DEPLA. (MM)'
FIN

```

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This section provides a list of the bibliography related to EUROPLEXUS (from December 1999 on) and its ancestors: PLEXIS-3C (including its own ancestors, the codes of the EURDYN series), and CASTEM-PLEXUS.

The section is subdivided into four sub-sections:

- Some selected references of the EURDYN series of codes,
- References pertaining to PLEXIS-3C and EUROPLEXUS development performed at JRC,
- References pertaining to CASTEM-PLEXUS and EUROPLEXUS developments performed at CEA,
- References pertaining to PLEXUS and EUROPLEXUS developments performed at Samtech.

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It is also recalled that an updated on-line version of the EUROPLEXUS User manual is available on the Internet and may be consulted interactively from a workstation or terminal.

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22 Keywords Index

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